



INDIAN INSTITUTE OF TECHNOLOGY KHARAGPUR
Mid-Autumn Semester Examination 2023-24

Date of examination: Sep. 20, 2023

Session: AN

Duration: 2 hrs

Full Marks: 60

Subject No.: ME21203

Subject: Mechanics of Solids

Department: Mechanical Engineering

Specific charts, graph paper, log book, etc. required: NO

Special Instructions (if any): Answer all the parts of a question together.

1. Consider a situation where the coordinate axes are oriented along the principal directions. Then the state of stress is given by (you don't have to explain or prove this):

$$\sigma = \begin{bmatrix} \sigma_{p1} & 0 & 0 \\ 0 & \sigma_{p2} & 0 \\ 0 & 0 & \sigma_{p3} \end{bmatrix}$$

A plane which is equally inclined to these coordinate axes is referred to as an octahedral plane, and the unit outward normal to this plane $\hat{\mathbf{n}} = [n_1 \ n_2 \ n_3]^T$ is such that $|n_1| = |n_2| = |n_3| = 1/\sqrt{3}$ (again, you don't have to prove this).

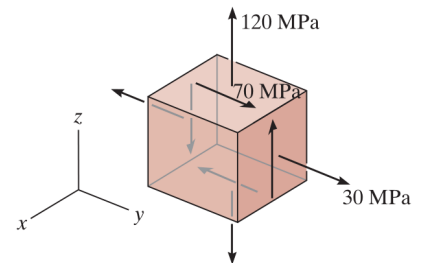
- (a) Show that the expression for the normal stress component on this plane is $\sigma_{nn} = \frac{1}{3}I_1$.
- (b) Show that the expression for the shear stress component (i.e. the traction component lying in the same plane that contains the traction vector and the unit outward normal) is $\sigma_{ns} = \frac{\sqrt{2}}{3} (I_1^2 - 3I_2)^{1/2}$.

Here, $I_1 = \text{trace}(\sigma)$ and $I_2 = \sigma_{p1}\sigma_{p2} + \sigma_{p2}\sigma_{p3} + \sigma_{p3}\sigma_{p1}$.

[3 + 4 = 7 marks]

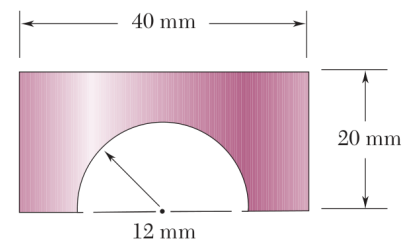
2. The state of stress at a point is depicted on a stress element as shown in the figure. Determine the principal stresses and the absolute maximum shear stress. [5 marks]

NOTE: The direction of the arrows represent the *actual physical* direction of the stress components. It is *your* responsibility to take the correct signs following the positivity convention in your calculations.



3. A beam has the cross-section as shown in the figure. The beam material has an allowable stress of 80 MPa in tension and 100 MPa in compression.

- (a) What is the location of the neutral axis (in mm) with respect to the bottom of the cross-section?
- (b) What is the area moment of inertia (or, second moment of area) (in mm^4) of the cross-section about the neutral axis?
- (c) Determine the largest bending moment (in N·m) that can be sustained by the beam.



[2 + 3 + 5 = 10 marks]

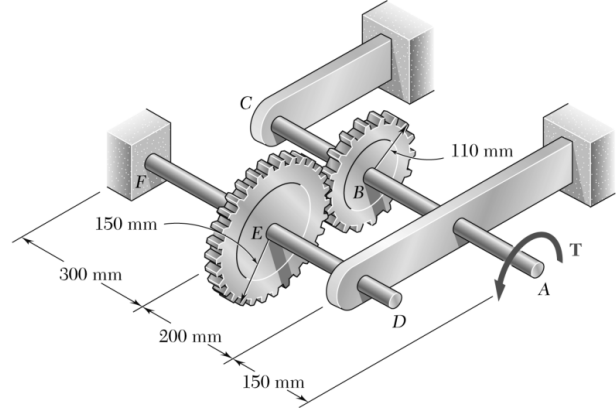
4. Consider a case of plane stress (in the xy -plane) for a material with Young's modulus, E , and Poisson's ratio, ν , and which follows the generalised Hooke's law.

(a) Show that: $\sigma_{xx} = \frac{E}{1-\nu^2} (\varepsilon_{xx} + \nu\varepsilon_{yy})$ and $\sigma_{yy} = \frac{E}{1-\nu^2} (\varepsilon_{yy} + \nu\varepsilon_{xx})$.

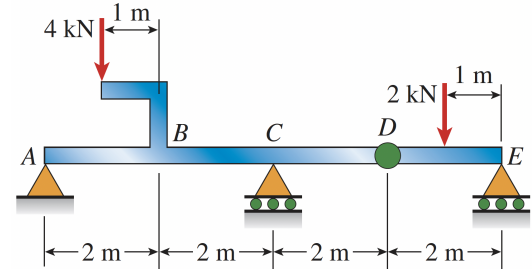
- (b) Derive the strain transformation equations from the stress transformation equations.

[3 + 5 = 8 marks]

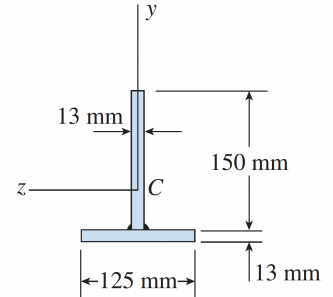
5. Two shafts, each of 22 mm diameter and made of the same material are connected by gears as shown in the figure. Given that $G = 77$ GPa and the shaft at F is fixed, determine the angle through which the end A rotates when a torque $T = 130$ N·m is applied at A . The slender beam-like supports through which the shafts pass do not provide any resistance to their rotation. Hint: When two gears mesh, the contact forces are equal (not the torques!); additionally the arcs of contact are equal. [10 marks]



6. The compound beam $ABCDE$ shown in the figure consists of two beams (AD and DE) joined by a hinged connection at D . The hinge can transmit a shear force but not a bending moment. The loads on the beam consist of a 4 kN force at the end of a bracket attached at point B , and a 2 kN force at the midpoint of beam DE . Draw the shear-force and bending-moment diagrams for this compound beam. [10 marks]



7. The T-beam shown in the figure is fabricated by welding together two steel plates. If the allowable load for each weld is 400 kN/m in the longitudinal direction (out of the plane of the paper), what is the maximum allowable shear force V ? [10 marks]



END OF QUESTION PAPER

List of useful formulae

- Torsion: $\tau = \frac{Tr}{J}$; $\phi = \frac{TL}{GJ}$

- Bending: $\frac{dV}{dx} = -w$ (for w pointed downward);

$$\frac{dM}{dx} = V$$

- Flexure formula: $\sigma = -\frac{My}{I}$

- Shear formula: $\tau = -\frac{VQ}{It}$

- Stress transformation:

$$\sigma_{x'x'} = \frac{\sigma_{xx} + \sigma_{yy}}{2} + \frac{\sigma_{xx} - \sigma_{yy}}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\tau_{x'y'} = -\frac{\sigma_{xx} - \sigma_{yy}}{2} \sin 2\theta + \tau_{xy} \cos 2\theta$$

- Centroid of a semi-circle of radius R is located $\frac{4R}{3\pi}$ from horizontal base