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```
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% MAE 231 - HW6.2  
% 02/25/15
```

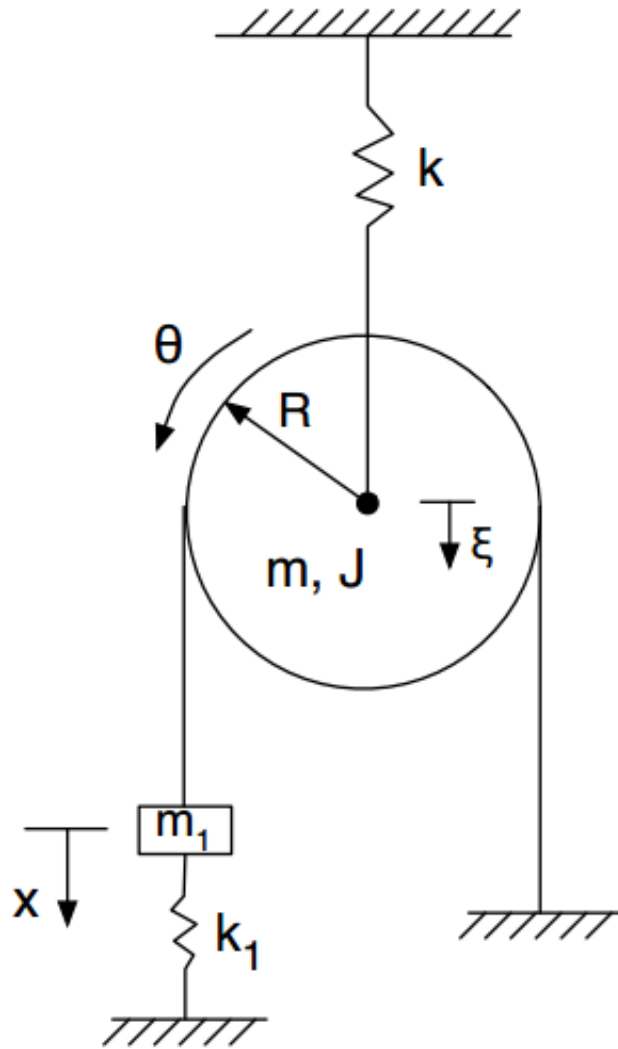
```
clear all  
close all  
clc
```

Problem 2:

For the system sketched on the next page, use the Euler-Lagrange equations to derive the (single) equation of motion in terms of one general coordinate x . Feel free to ignore the static deflection that it will cancel out as we've seen in class (and to not include Δ in the expression of the spring potential when you do). Some helpful reminders:

- Due to the pulley, $\xi = \frac{x}{2}$ and $x = 2R\theta$
- In terms of polar moment of inertia, consider the pulley as a disk (What is J in terms of m and R ?)

For $m_1 = 1\text{kg}$, $m = 2\text{kg}$, $k = 2\text{N/m}$, $k_1 = 5\text{N/m}$, $R = 0.3\text{m}$, $x_0 = 0.1\text{m}$ and $v_0 = 0$, plot the response of this system. From the figure, estimate the natural frequency of the system. Separately, calculate ω_n from the coefficients in the equation of motion. How do they compare?



Find: Response of system, ω_n (Estimate/Calculate)

Known

$m_1, m, k, k_1, R, x_0, v_0$

```

mass1      = 1;    % kg
mass       = 2;    % kg
stiffness  = 2;    % N/m
stiffness1 = 5;    % N/m
radius     = 0.3;  % m
xInitial   = 0.1;  % m
vInitial   = 0;    % m/s
  
```

Calculations

$$\xi = \frac{x}{2}, \quad x = 2R\theta$$

$$T = \frac{1}{2}m_1\dot{x}^2 + \frac{1}{2}J\dot{\theta}^2 = \frac{1}{2}m_1\dot{x}^2 + \frac{1}{16}m\dot{x}^2$$

$$U = \frac{1}{2}k\xi^2 + \frac{1}{2}k_1x^2 = \frac{1}{8}kx^2 + \frac{1}{2}k_1x^2$$

$$L = T - U = \frac{1}{2}m_1\dot{x}^2 + \frac{1}{16}m\dot{x}^2 - \frac{1}{8}kx^2 - \frac{1}{2}k_1x^2$$

$$\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{x}}\right) - \frac{\partial L}{\partial x} = 0$$

$$\left(\frac{1}{8}m + m_1\right)\ddot{x} + \left(\frac{1}{4}k + k_1\right)x = 0$$

$$\ddot{x} + \left(\frac{\frac{1}{4}k + k_1}{\frac{1}{8}m + m_1}\right)x = 0$$

$$\omega_n = \sqrt{\frac{\frac{1}{4}k + k_1}{\frac{1}{8}m + m_1}}$$

```
frequencyNaturalEoM = sqrt((0.25 * stiffness + stiffness1) / ...
                        (mass / 8 + mass1))
time                = linspace(0, 10, 1000);
x                   = xUndamped(xInitial, vInitial, frequencyNaturalEoM, time);
[peak1, indexPeak1] = max(x(200:400));
indexPeak1          = 199 + indexPeak1;
[peak2, indexPeak2] = max(x(500:700));
indexPeak2          = 499 + indexPeak2;
frequencyNaturalEst = (2 * pi) / (time(indexPeak2) - time(indexPeak1))
```

```
frequencyNaturalEoM =
```

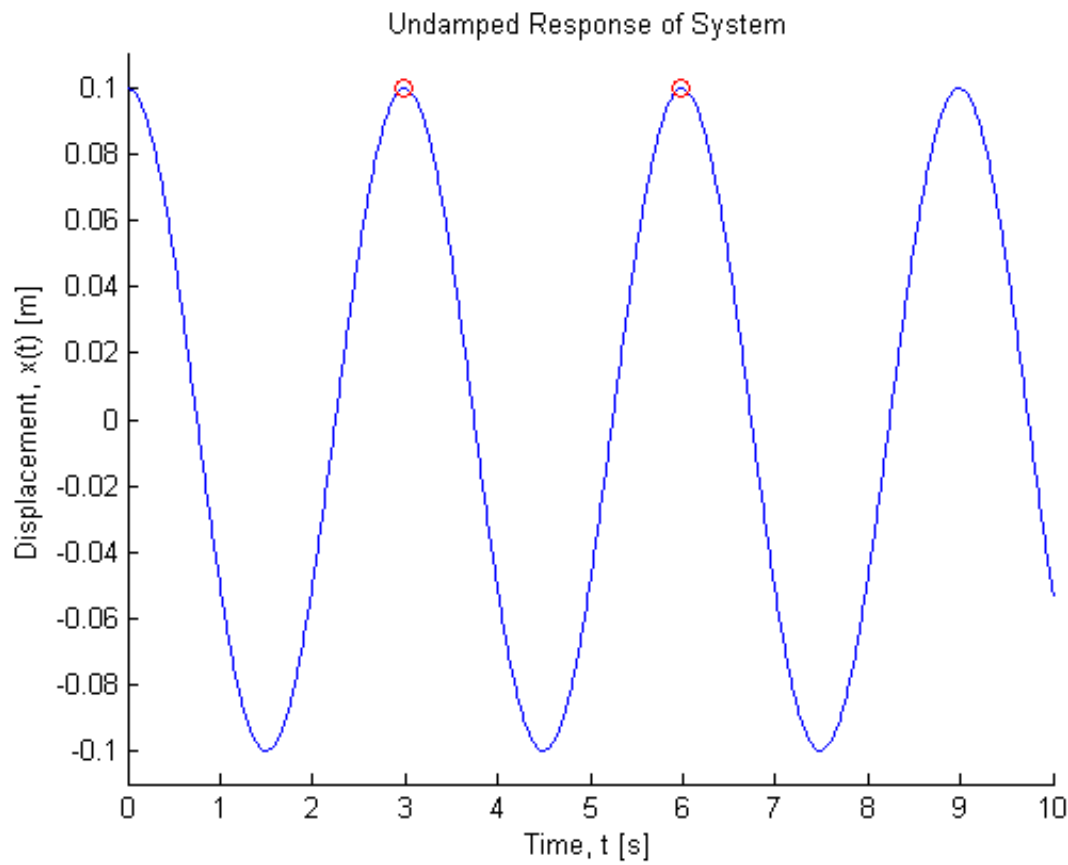
```
2.0976
```

```
frequencyNaturalEst =
```

```
2.0993
```

Plot

```
figure(1)
hold on
axis([0 10 -0.11 0.11])
title('Undamped Response of System')
xlabel('Time, t [s]')
ylabel('Displacement, x(t) [m]')
plot(time, x)
plot(time(indexPeak1), x(indexPeak1), 'o', 'color', [1 0 0])
plot(time(indexPeak2), x(indexPeak2), 'o', 'color', [1 0 0])
```



Results

The natural frequency obtained from coefficients of the equation of motion is $\omega_n = 2.0976 \text{ rad/s}$ and the natural frequency estimated from the plot of the response is $\omega_n = 2.0993 \text{ rad/s}$. These numbers are very close, and theoretically should be the same. The difference in the values can be accounted for by error caused by a time step that is too large in the plot.