PSI plane-strain example

This notebook is a companion to "Phase-space iterative solvers", it reproduces the results in section "Plate with hole".

```
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```

Parameters of this study

Material

```
Intact Material: E = 200^9 \text{Pa} and v = 0.33

In[1]:= youngModIntact = 200 \cdot * \cdot 10^9;

nu = 0.33;

(*function*)

p = 2.0 \cdot * \cdot 10^{-4};

c = (1/p) \cdot (1/(p-1));

youngMod[x ] = p * youngModIntact * ((Abs[x] + c)<sup>p-1</sup>);

youngModTan[x ] = p * (p - 1) * youngModIntact * ((Abs[x] + c)<sup>p-2</sup>);
```

Convergence

Convergence tolerance: the step has converged once

```
|f_{\text{ext}} - f_{\text{int}}| / |f_{\text{ext}}| < \text{tol}
```

```
ln[7] = tol = 1 * 10^{-2}; (*in terms of eq.*)
     Phase-space convergence tolerance: the step has converged once
      |z^{(n+1)} - z^{(n)}| / |z^{(n)}| < \text{tolPS}
ln[8]:= tolPS = 1*10^{-3}; (*in terms of phase-space distance*)
     PSI
     At this point I define the parameters of the phase-space solver.
In[9]:= (*method*)
     approach = "minimization";(* "E-L" or "minimization"*)
     optimalC = "False";
     (*method in the case of minimization*)
     methodMin = "ConjugateGradient";
     (*"ConjugateGradient", "Newton", "QuasiNewton",
     "PrincipalAxis"*)
     (*number of tests PSI*)
     Ntests = 1;
     (*Constant for the distance function*)
     Cd = 0.1*youngModIntact;
     (*Number of CPUs*)
     numKernels = 4;
     NR
In[15]:= (*number of tests NR*)
     NtestsNR = 1;
In[16]:= numLoadSteps = 1;
```

(*number of steps we use to derive the load*)

Pre-processing

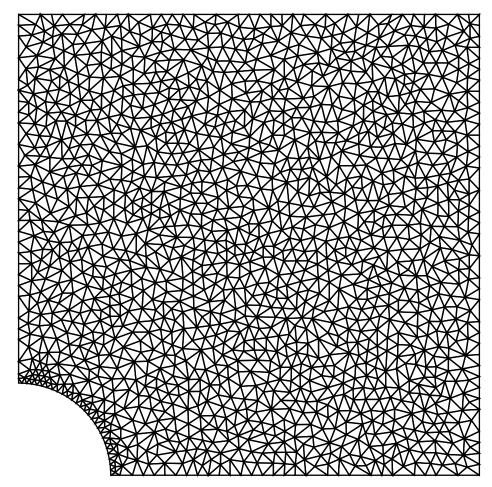
Create mesh. Begin by loading FEM tools:

```
In[17]:= Needs["NDSolve`FEM`"];
```

Define region (exploiting symmetry), see Fig.1a in the paper.

```
In[18]:= r = 0.02; (*hole radius*)
```

```
In[19]:= region = RegionDifference[
         Polygon[{{0, 0}, {0., 0.1}, {0.1, 0.1}, {0.1, 0}}],
         Disk[-{0.0, 0.0}, r]];
     mesh = ToElementMesh[region,
         MaxCellMeasure → {"Length" → 0.005}, "MeshOrder" → 1,
         AccuracyGoal → 5];
     Show[
      mesh["Wireframe"]
       , Background → White]
```



Mesh properties

Use Mathematica's built-in tools to extract information

```
In[22]:= nodes = mesh["Coordinates"];(*nodes in the mesh*)
     numNodes = Length@nodes;(*number of nodes*)
     connectivity = mesh["MeshElements"][[1]][[1]];
     (*element connectivity*)
     numElements = Length@connectivity;
     (*number of elements*)
     numDOFs = 2;(*dofs per node: u and v*)
     (*for each element,
     to what dofs contributes to w/ forces
      this is consistent w/ the way in which the B
      matrives are constructed*)
     locDOFs =
       Table
         Flatten[{connectivity[[ii]],
           numNodes + connectivity[[ii]]], {ii, 1, numElements}];
     Identify fixed nodes and loaded (to apply BCs)
```

```
In[28]:= lowerEdge = {};
     Do[
        If[nodes[[ii]][[2]] ≤ 0.0005, AppendTo[lowerEdge, ii]]
        , {ii, 1, numNodes}];
     upperEdge = {};
     Do[
        If[nodes[[ii]][[2]] == 0.1, AppendTo[upperEdge, ii]]
        , {ii, 1, numNodes}];
     leftEdge = {};
     ]od
        If[nodes[[ii]][[1]] ≤ 0.001, AppendTo[leftEdge, ii]]
        , {ii, 1, numNodes}];
     2D problem: define a thickness
```

In[34]:= thickness = 1.;

Plane strain operator for Mathematica pre-processing

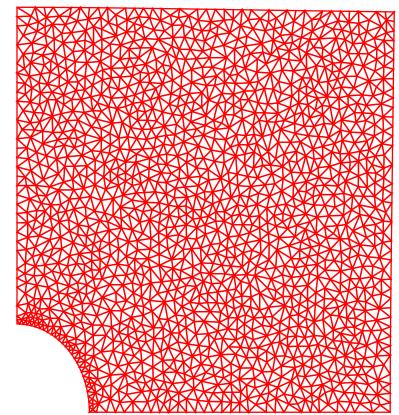
Loading: constant traction at the upper edge

```
In[35]:= ps = 10<sup>8</sup>;(*Pa/m (all units SI)*)
      tractions = NeumannValue[ps, y == 0.1];
```

Plane-strain operator for linear-elastic homogeneous isotropic material:

```
In[37]:= planeStrainOperator =
                               {Inactive[Div][
                                                   \{\{\{0, -((Y v)/((1-2 v) (1+v)))\}, \{-(Y/(2 (1+v))), 0\}\}\}
                                                             Inactive[Grad][v[x, y], \{x, y\}]), \{x, y\}] +
                                               Inactive[Div][
                                                   \{\{-((Y(1-v))/((1-2v)(1+v))), 0\}, \{0, -(Y/(2(1+v)))\}\}.
                                                             Inactive[Grad][u[x, y], \{x, y\}], \{x, y\}],
                                          Inactive[Div][
                                                   \{\{0, -(Y/(2(1+v)))\}, \{-((Yv)/((1-2v)(1+v))), 0\}\}.
                                                             Inactive[Grad][u[x, y], \{x, y\}]), \{x, y\}] +
                                               Inactive[Div][
                                                   \{\{-(Y/(2(1+v))), 0\}, \{0, -((Y(1-v))/((1-2v)(1+v)))\}\}.
                                                             Inactive[Grad][v[x, y], \{x, y\}]), \{x, y\}]} /.
                                    \{Y \rightarrow 200.*10^9, v \rightarrow 33./100\};
                      BCs:
In[38]:= (* held fixed at left *)
                      bcs = {
                                     DirichletCondition[u[x, y] == 0, x \le 0],
                                     DirichletCondition[v[x, y] == 0, y \le 0]};
                      Define PDE:
In[39]:= pde2D = planeStrainOperator == {0, 1.*tractions};
                      Mathematica linear-elastic solution (shown for completeness for
                     other users to play with if interested, not necessary):
ln[40]:= \{usol, vsol\} = NDSolveValue[\{pde2D, bcs\}, \{u, v\}, \{
                                    \{x, y\} \in mesh];
```

```
In[41]:= mesh = usol["ElementMesh"];
     Show[
      {mesh["Wireframe"["MeshElementStyle" → EdgeForm[White]]],
       ElementMeshDeformation[mesh, {usol, vsol},
          "ScalingFactor" → 100][
        "Wireframe"["ElementMeshDirective" →
           Directive[EdgeForm[Red], FaceForm[]]]]},
      ImageSize → 300]
```



Pre-processing: solve linear-elastic problem

Define intact material

In[43]:= (*Intact plane-strain tensor*)
$$\begin{pmatrix}
\sigma_{11} \\ \sigma_{22} \\ \tau_{12}
\end{pmatrix} = \begin{pmatrix}
\lambda + 2\mu & \lambda & 0 \\ \lambda & \lambda + 2\mu & 0 \\ 0 & \mu
\end{pmatrix} \begin{pmatrix}
\epsilon_{11} \\ \epsilon_{22} \\ \gamma_{12}
\end{pmatrix} *)$$

$$matCIntact = \left\{ \left\{ 2 + \frac{2 \text{ nu}}{1 - 2 \text{ nu}}, \frac{2 \text{ nu}}{1 - 2 \text{ nu}}, 0 \right\},$$

$$\left\{ \frac{2 \text{ nu}}{1 - 2 \text{ nu}}, 2 + \frac{2 \text{ nu}}{1 - 2 \text{ nu}}, 0 \right\},$$

$$\left\{ 0, 0, 1 \right\} * \frac{\text{youngModIntact}}{2 (1 + \text{nu})};$$

Preprocessing: take advantage of Mathematica to distribute the load to the nodes

```
In[44]:= nr = ToNumericalRegion[mesh];
     vd = NDSolve`VariableData[
        {"DependentVariables", "Space"} \rightarrow {{u, v}, {x, y}}];
     sd = NDSolve`SolutionData[{"Space"} → {nr}];
     (*We use NDSolve as a pre-processor:*)
     {state} =
      NDSolve ProcessEquations[{pde2D, bcs}, {u, v},
        \{x, y\} \in mesh];
     (*Extract the finite element data:*)
     femdata = state["FiniteElementData"];
     initBCs = femdata["BoundaryConditionData"];
     methodData = femdata["FEMMethodData"];
     initCoeffs = femdata["PDECoefficientData"];
     (*discretize*)
     discretePDE = DiscretizePDE[initCoeffs, methodData,
        sd, "SaveFiniteElements" → True,
        "AssembleSystemMatrices" → True];
     discreteBCs = DiscretizeBoundaryConditions[initBCs,
        methodData, sd];
     (*Extract the system matrices:*)
     load = discretePDE["LoadVector"];
     stiffness = discretePDE["StiffnessMatrix"];
     stiffnessBeforeBCs = stiffness;
     DeployBoundaryConditions[{load, stiffness},
       discreteBCs];
```

Construct matrices B

I need to do this manually although most probably there is a way to get the same using **NDSolve**

```
In[58]:= elCentroids = RegionCentroid[Polygon[nodes[[#]]]] & /@
         connectivity;
     elAreas = ConstantArray[0., {numElements, 1}];
     elMatB = ConstantArray[0., {numElements, 1}];
     Do
      (*relevant nodal coordinates
        (to compute the coefficients of B)*)
       node1 = nodes[[connectivity[[ii]][[1]]]];
       node2 = nodes[[connectivity[[ii]][[2]]]];
       node3 = nodes[[connectivity[[ii]][[3]]]];
       (*Compute area*)
       elArea = Area@Polygon[nodes[[connectivity[[ii]]]]];
       elAreas[[ii]] = elArea;
       (*compute the B matrix of the
         element----*)
       matB = ConstantArray[0., {3, 6}];
      matB[[1, 1]] = \frac{1}{2 * elArea} (Last@node2 - Last@node3);
      matB[[1, 2]] = \frac{1}{2 * elArea} (Last@node3 - Last@node1);
      matB[[1, 3]] = \frac{1}{2 * elArea} (Last@node1 - Last@node2);
      matB[[2, 4]] = \frac{1}{2 * elArea} (First@node3 - First@node2);
      matB[[2, 5]] = \frac{1}{2 * elArea} (First@node1 - First@node3);
```

Construct the K matrix

Element-wise contributions:

```
In[62]:= elMatK =
        Table[Transpose[elMatB[[ii]]].matCIntact.elMatB[[ii]]*
          elAreas[[ii]], {ii, 1, numElements}];
```

Assemble:

```
In[63]:= totalList = ConstantArray[0., {numElements, 1}];
     Do[
        (*If[Mod[kk, 100]==0, Print[kk]];*)
        subList = Table[
          {If[ii ≤ 3, connectivity[[kk]][[ii]],
              connectivity[[kk]][[ii - 3]] + numNodes],
             If [jj \leq 3, connectivity[[kk]][[jj]],
              connectivity[[kk]][[jj - 3]] + numNodes]}
           → elMatK[[kk]][[ii, jj]],
          {ii, 1, 3*numDOFs}, {jj, 1, 3*numDOFs}];
        totalList[[kk]] = Flatten[subList, 1];
        , {kk, 1, numElements}];
     SetSystemOptions[
        "SparseArrayOptions" →
         {"TreatRepeatedEntries" → Total}];
     globalK = SparseArray[Flatten[totalList, 1],
         {numDOFs*numNodes, numDOFs*numNodes}];
     SetSystemOptions["SparseArrayOptions" →
         {"TreatRepeatedEntries" → 0}];
     Apply BCs
     First, the horizontal symmetry plane:
In[68]:= restrainedDOFsY =
        Sort[Flatten[{# + numNodes} & /@ lowerEdge]];
     activeDOFs = DeleteCases[Range[numDOFs*numNodes],
         Alternatives @@ restrainedDOFsY];
     Next the vertical symmetry plane:
```

```
In[70]:= restrainedDOFsX = Sort[Flatten[{#} & /@ leftEdge]];
     activeDOFs = DeleteCases[activeDOFs,
         Alternatives @@ restrainedDOFsX];
```

Solve linear-elastic solution using matrices that will later be used PSI

Just for sanity check purposes: this yields the same than using **NDSolve** above

```
In[72]:= forceVecExt = load;
     intactU = SparseArray[{}, {numNodes*numD0Fs, 1}];
     intactU[[activeDOFs]] =
       LinearSolve[globalK[[activeDOFs, activeDOFs]],
         forceVecExt[[activeDOFs]]];
```

Solve using Newton-Raphson method

The reason to use NR:

it is the main reference to compare PSI against

Mean stress function (necessary to define the threshold):

Introducing the mean strain

I use the mean strain to gauge how much the stiffness changes as strain accumulates (recall, plane strain -> ϵ_{33} =0)

$$\ln[75]:= \mathsf{epsilonMean}[\epsilon] := \frac{\epsilon[[1, 1]] + \epsilon[[2, 1]]}{3};$$

The loop

Initialize the rest

```
In[76]:= timeListNR = Table[, NtestsNR];
    Do
      (*initialize material stiffness to the intact
       stiffness*)
      matCList = Table[matCIntact, {ii, 1, numElements}];
      matT = SparseArray[{},
        {numNodes*numDOFs, numNodes*numDOFs}];
      Unr = SparseArray[{}, {numNodes*numD0Fs, 1}];
          ----*)
      timeNR = AbsoluteTiming
        Do
          forceVecExt = \frac{ff}{numLoadSteps} load;
          normForceExt = Norm[forceVecExt[[activeDOFs]]];
          Print("*----*"):
          Print["Load step # "<> ToString[ff] <>
            " out of "<> ToString[numLoadSteps]];
          Print["*----*"];
          (*prepare for the iterations at that load level*)
          steps = 1;
          resError = 1.;
          done = False;
          While done == False,
```

```
forceVecInt = SparseArray[{},
   {numNodes*numDOFs, 1}]; (*initialize*)
matT = 0.*matT;
(*build stiffness matric fot this
stressList = ConstantArray[0., {numElements, 1}];
strainList = ConstantArray[0., {numElements, 1}];
Do
 (*displacements in the relevant nodes*)
 Ue = Unr[[locD0Fs[[ii]]]];
 matB = elMatB[[ii]];
 (*element strains*)
 strain = matB.Ue;
 strainList[[ii]] = strain;
 (*material stiffnesses*)
 matC = matCList[[ii]];
 (*element stresses*)
 stress = matC.strain;
 (*update element tangent matrix for next
   iteration*)
 matCList[[ii]] = \frac{youngMod\left[\frac{epsilonMean@strain}{1.00}\right]}{1.00}
    \left\{\left\{2+\frac{2 \text{ nu}}{1-2 \text{ nu}}, \frac{2 \text{ nu}}{1-2 \text{ nu}}, 0\right\}\right\}
      \left\{\frac{2 \text{ nu}}{1-2 \text{ nu}}, 2+\frac{2 \text{ nu}}{1-2 \text{ nu}}, 0\right\}, \{0, 0, 1\}\right\};
 (*matCList[[ii]]= youngMod[sigmaMean@stress]
1-nu<sup>2</sup>
```

```
(\{\{1, nu, 0\}, \{nu, 1, 0\}, \{0, 0, \frac{1-nu}{2}\}\}); *)
 (*take the average betwen two steps,
 this boosts convergence*)
 matC = 0.5 (matC + matCList[[ii]]);
 stress = matC.strain;
 stressList[[ii]] = stress;
 (*save history*)
 (*internal force contribution*)
 forceVecInt[[locD0Fs[[ii]]]] =
  Transpose[elMatB[[ii]]].stress*elAreas[[ii]]*
     thickness + forceVecInt[[locD0Fs[[ii]]]];
 (*contribution to the stiffness matrix*)
 matT[[locD0Fs[[ii]], locD0Fs[[ii]]]] =
  matT[[locD0Fs[[ii]], locD0Fs[[ii]]]] +
   Transpose[elMatB[[ii]]].matC.elMatB[[ii]]*
    elAreas[[ii]];
 , {ii, 1, numElements}];
deltaF = SparseArray[forceVecExt - forceVecInt];
                       ----*)
resError = Norm[deltaF[[activeDOFs]]] /
  normForceExt;
deltaU = SparseArray[{}, Dimensions@Unr];
deltaU[[activeDOFs]] =
 LinearSolve[matT[[activeDOFs, activeDOFs]],
  deltaF[[activeDOFs]]];
Unr[[activeDOFs]] = Unr[[activeDOFs]] +
```

```
deltaU[[activeDOFs]];
       If[Mod[steps, 25] == 0,
         Print["Step #" <> ToString[steps] <>
                Log Res. Error (Force):" <>
             ToString[Log10@resError]];]
        If[steps > 500,
         Print["Failed to converge in "<>
           ToString[steps] <> " steps"]; Break[]];
       If[resError < tol, (*done yet?*)</pre>
        Print["Converged in " <> ToString[steps] <>
          " steps"];
        done = True, (*yes*)
        steps = steps + 1;(*next step*)
       (*done!*)
     ];,
     {ff, 1, numLoadSteps}];];
 timeListNR[[kk]] = First@timeNR;
 , {kk, 1, NtestsNR}
Load step # 1 out of 1
```

Converged in 12 steps

PSI

Solve the same project using phase-space iterations

Preliminaries

```
In[78]:= methodC = matCIntact;
     methodCinv = Inverse[methodC];
     methodK = globalK;
     elMatC = elMatK;
     (*elementary K matrix -- before assembling*)
```

Define material

A bit redundant, but I define the constitutive law again here. This is because, due to my lack of consistency, the strain is taken as a matrix in NR and as vector in PSI.

```
\sigma = material(\epsilon)
ln[82]:= epsilonMean[\epsilon] := \frac{\epsilon[[1]] + \epsilon[[2]]}{3};
```

This seems clogged but it is just Eq. (18) in the paper

In[83]:= material[eps_] :=
$$\frac{\text{youngMod}\left[\frac{\text{epsilonMean@eps}}{1.}\right]}{2(1 + \text{nu})}$$

$$\left\{ \left\{ 2 + \frac{2 \text{ nu}}{1 - 2 \text{ nu}}, \frac{2 \text{ nu}}{1 - 2 \text{ nu}}, 0 \right\}, \left\{ \frac{2 \text{ nu}}{1 - 2 \text{ nu}}, 2 + \frac{2 \text{ nu}}{1 - 2 \text{ nu}}, 0 \right\}, \\ \left\{ 0, 0, 1 \right\} \right\}. \text{eps};$$

Iterate

Initialize kernels for parallel projections into the material law

In[84]:= CloseKernels[];
 LaunchKernels[numKernels];

Looping time

The commented lines below are to change some parameters of the method

w/o having to go all the way to the top of the notebook.

Iterate

In[89]:= timeListPSI = Table[, {Ntests}];

```
Do
 (*initialize displacements and Lagrange multipliers*)
 methodU = ConstantArray[0., {numNodes*numD0Fs, 1}];
 methodEtas = ConstantArray[0., {numNodes*numD0Fs, 1}];
 (*auxiliary srrays to save information about
  the iterations*)
 internalForcesListIPS = {};
 (*force resdiual across iterations*)
 psDistList = {};
 (*distance after iterations*)
 elemN = 10;
 (*particular element number to monitor phase
  space points and visualize convergence -- see
  figure 3 in the paper*)
 psListE = {};
 (*to save phase-space points in E visited by
   that element*)
 psListD = {};
 (*to save phase-space points in D visited by
   that element*)
 timeProjsE = {}; (*to save time invested in
  projecting onto E*)
 timeProjsD = {}; (*to save time invested in
  projecting onto D*)
 (*iterations*)
 (*just initialization*)
 (*initial point (origin),
 this is actually a material-admissible state*)
 sigmaPrime = ConstantArray[{0., 0., 0.}, numElements];
 epsilonPrime = ConstantArray[{0., 0., 0.}, numElements];
 sigma = ConstantArray[{0, 0, 0}, numElements];
```

```
epsilon = ConstantArray[{0, 0, 0}, numElements];
zetaPrev = Table[Join[epsilon[[ii]], sigma[[ii]]],
  {ii, 1, numElements}];
cont = True;
q = 0;
Print["Solving with approach "<> approach <>
  ", with method "<> methodMin];
(*loop*)
timePSI = AbsoluteTiming
  While cont,
   q = q + 1;
   (*----- PROJECTION
     ONTO E ----*)
   tictoc = AbsoluteTiming[
     rhsU = ConstantArray[0., {numD0Fs*numNodes, 1}];
     (*initialize*)
     forceVecInt = ConstantArray[0.,
       {numDOFs*numNodes, 1}]; (*initialize*)
     (*build eqns' rhs for this
            ----*)
     (*K u = Sum[w B<sup>T</sup> C \epsilon*]=rhsU i.e.,
     right-hand side of the equation to compute U*)
     (*K \eta = F_{ext} - Sum[w B^T \sigma^*]*)
     Dol
      (*strain contribution*)
```

```
rhsU[[locD0Fs[[ii]]]] =
  elAreas[[ii]] * thickness *
     Transpose[elMatB[[ii]]].methodC.
      epsilonPrime[[ii]] + rhsU[[locD0Fs[[ii]]]];
 (*stress contribution*)
 forceVecInt[[locD0Fs[[ii]]]] =
  elAreas[[ii]] * thickness *
     Transpose[elMatB[[ii]]].sigmaPrime[[ii]] +
   forceVecInt[[locD0Fs[[ii]]]];
 , {ii, 1, numElements}];
(*Solve linear system*)
methodU[[activeDOFs]] =
 LinearSolve[methodK[[activeDOFs, activeDOFs]],
  rhsU[[activeD0Fs]]];
(*no need to condense forces,
no forced displacement*)
methodEtas[[activeDOFs]] =
 LinearSolve[methodK[[activeDOFs, activeDOFs]],
  forceVecExt[[activeD0Fs]] -
   forceVecInt[[activeD0Fs]]];
(*update stress and strain*)
lod
 (*\sigma_e = \sigma_e' + C B_e \eta*)
 sigma[[ii]] = sigmaPrime[[ii]] +
   Flatten[methodC.elMatB[[ii]].
      methodEtas[[locD0Fs[[ii]]]]];
 (*\sigma_e = \sigma_e^* + C B_e \eta_e *)
 (*\epsilon_e = B_e u*)
```

```
epsilon[[ii]] =
     Flatten[elMatB[[ii]].methodU[[locD0Fs[[ii]]]]];
   (*\epsilon_e = B_e u_e*)
    , {ii, 1, numElements}];
 ];
AppendTo[timeProjsE, tictoc];
AppendTo[psListE, {epsilon[[elemN]], sigma[[elemN]]}];
AppendTo[internalForcesListIPS, forceVecInt];
resError =
 Norm[methodK[[activeDOFs, activeDOFs]].
     methodEtas[[activeDOFs]]] /
  Norm[forceVecExt[[activeDOFs]]];
zeta = Table[Join[epsilon[[ii]], sigma[[ii]]],
  {ii, 1, numElements}];
         Norm[Flatten[zeta] - Flatten[zetaPrev]];
                   Norm[Flatten[zeta]]
AppendTo[psDistList, psDist];
                         *)
Which[
 q > 2000, Print["Solver failed to converge"];
 Break[], (*surpassed the limit of iterations*)
 resError < tol,
```

```
Print["Solver converged in " <> ToString[q] <>
   " iterations (equilibrium tolerance
     satisfied)."]; Break[],
 psDist < tolPS,</pre>
 Print["Solver converged in "<> ToString[q] <>
   " iterations; phase-space distance
     converged (eq.res=" <>
   ToString[DecimalForm[100*resError, {3, 2}]] <>
   "%)"]; Break[],
 (*-----*)
 True,
 Print["Residual: "<>
   ToString[DecimalForm@resError] <>
   ", phase-space percentual increment: "<>
   ToString[DecimalForm@psDist]]
];
zetaPrev = zeta;
(*----- PROJECTION
  ONTO D -----*)
If approach == "minimization",
 (*YES, go use minimization*)
 If[optimalC == "False",
   (*---- Using distance
   minimization ----*)
   tictoc = First@AbsoluteTiming
     epsilonPrime = ParallelTable
```

```
{a, b, y} /. Last@Quiet@
            FindMinimum
             Cd*Norm[{a, b, y} - epsilon[[ii]]]^2 +
               Norm[material[{a, b, y}] - sigma[[ii]]]<sup>2</sup>,
             {{a, epsilon[[ii]][[1]]},
              {b, epsilon[[ii]][[2]]},
              {y, epsilon[[ii]][[3]]}},
             (*initial guess*)
             Method -> methodMin (*method*)
        , {ii, numElements}];];
 (*Project stresses*)
 sigmaPrime = material/@ epsilonPrime;,
 (*---- Using distance minimization
  w/ optimal C -----
 tictoc = First@AbsoluteTiming[
     epsilonPrime = ParallelTable[
        {a, b, y} /. Last@Quiet@FindMinimum[
             (material[{a, b, c}] - sigma[[ii]]).
              (material[{a, b, y}] - sigma[[ii]]),
             {{a, epsilon[[ii]][[1]]},
              {b, epsilon[[ii]][[2]]},
              {y, epsilon[[ii]][[3]]}},
             (*initial guess*)
             Method -> methodMin](*method*)
        , {ii, numElements}];];
 (*Project stresses*)
 sigmaPrime = sigma;
;, (*NO, use Euler-Lagrange equations*)
```

```
(*Break[];*)
     (*---- Using distance E-
      L w/ optimal C ----*)
     tictoc = First@AbsoluteTiming[
         epsilonPrime = ParallelTable[
            {a, b, y} /. Last@Quiet@FindMinimum[
                 (material[{a, b, y}] - sigma[[ii]]).
                  (material[{a, b, y}] - sigma[[ii]]),
                 {{a, epsilon[[ii]][[1]]},
                  {b, epsilon[[ii]][[2]]},
                  {y, epsilon[[ii]][[3]]}}, (*initial
                  guess*)
                 Method -> methodMin](*method*)
            , {ii, numElements}];];
     (*Project stresses*)
     sigmaPrime = sigma;
                            ----- save
     time*)
    AppendTo[psListD, {epsilonPrime[[elemN]],
      sigmaPrime[[elemN]]}];
    AppendTo[timeProjsD, tictoc];
   (*end while*)
 timeListPSI[[tt]] = timePSI;
 , {tt, 1, Ntests}
Solving with approach minimization, with method QuasiNewton
Residual: 1., phase-space percentual increment: 1.
```

Residual: 0.0993757, phase-space percentual increment: 0.0303667 Residual: 0.0731393, phase-space percentual increment: 0.0196742 Residual: 0.0563818, phase-space percentual increment: 0.015241 Residual: 0.0432433, phase-space percentual increment: 0.0120636

Residual: 0.0333261, phase-space percentual increment: 0.00949006

Residual: 0.0259936, phase-space percentual increment: 0.00748172

Residual: 0.0206514, phase-space percentual increment: 0.00596663

Residual: 0.0168229, phase-space percentual increment: 0.00484044

Residual: 0.0141308, phase-space percentual increment: 0.00400283

Residual: 0.0122708, phase-space percentual increment: 0.00337154

Residual: 0.0110004, phase-space percentual increment: 0.00288712

Residual: 0.0101307, phase-space percentual increment: 0.00250725

Solver converged in 14 iterations (equilibrium tolerance satisfied).

ln[91] = sf = 100;(*----*) deformedShape =

```
Table
   nodes[[ii]] + sf * First /@ {Unr[[ii]], Unr[[ii + numNodes]]},
   {ii, 1, numNodes}];
edgeList = {};
lod
 list = DeleteDuplicates@
   Flatten[deformedShape[[#]] & /@
     (Sort @ Permutations[connectivity[[ii]], {2}]), 1];
 AppendTo[edgeList, Line@AppendTo[list, First@list]];
 , {ii, 1, numElements}]
(*----*)
deformedShapeM =
  Table
   nodes[[ii]]+
    sf*First/@{methodU[[ii]], methodU[[ii + numNodes]]},
   {ii, 1, numNodes}];
edgeListM = {};
]oQ
 list = DeleteDuplicates@
   Flatten[deformedShapeM[[#]] & /@
     (Sort /@ Permutations[connectivity[[ii]], {2}]), 1];
AppendTo[edgeListM, Line@AppendTo[list, First@list]];
 , {ii, 1, numElements}]
(*----*)
Show
 ElementMeshDeformation[mesh, {usol, vsol},
   "ScalingFactor" → 0][
  "Wireframe"["ElementMeshDirective" →
    Directive[EdgeForm[Red], FaceForm[]]]],
 Graphics[{
   {Lighter@Gray, Thickness[0.01], edgeList},
```

```
{Dashed, Black, Thickness[0.0015], edgeListM}
 }],
Axes → True,
PlotRange → All,
PlotRangeClipping -> True,
ImageSize → Large,
Background -> White]
```

