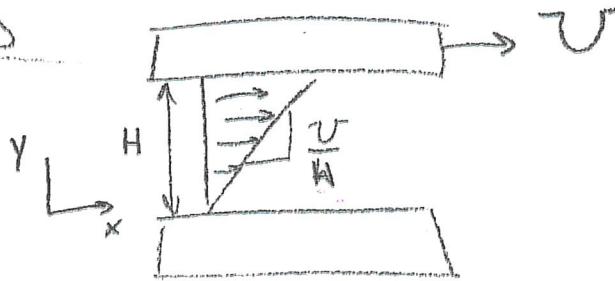


Materials Processing

Couette Flow

$$\frac{\partial u_x}{\partial x} + \frac{\partial u_y}{\partial y} + \frac{\partial u_z}{\partial z} = 0$$



$$\rho \left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} \right) = - \frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} + \frac{\partial^2 u_x}{\partial z^2} \right) + f_x$$

$$\mu \frac{\partial^2 u_x}{\partial y^2} = 0$$

$$\frac{\partial^2 u_x}{\partial y^2} = 0$$

$$\frac{\partial u_x}{\partial y} = C_1$$

$$u_x = C_1 y + C_2$$

$$y(0), u_x = 0$$

$$0 = 0 + C_2, C_2 = 0$$

$$u_x = \frac{U}{H} y$$

$$y(H), u_x = U$$

$$U = H C_1, C_1 = \frac{U}{H}$$

diameter

$$\frac{[\text{mm}]^2 [\text{mm}] [\text{rev}]}{[\text{rev}] [\text{min}]}$$

Material Removal Rate

volume/min

$$\frac{\pi D^2}{4} f N$$

D = diameter (mm)

f = feed $\left[\frac{\text{mm}}{\text{rev}} \right]$ 每转一周沿轴向走动的距离

N = rotations/min $[\text{rps}] [\text{rev}/\text{min}]$

for turning (lathe) = $\pi D f d N$

$$\frac{[\text{mm}] [\text{mm}] [\text{mm}] [\text{rev}]}{[\text{rev}] [\text{min}]}$$

d = depth of cut.

Reynolds number $R_D = \frac{\rho D V}{\mu}$ initial force / density * diameter / velocity

格拉晓夫数

Grashof: $g \beta (T_s - T_\infty) D^3 / \nu \mu$

surface bulk pipe diameter
buoyancy force viscous force

Reynolds number $Re = \frac{D}{\alpha} \frac{\rho V}{\mu}$ specific heat / momentum viscosity

viscous force / diffusion thermal diffusivity

Similar to Reynolds num

Brinkman: $\frac{v}{\mu}$

flow viscous heating conduction

Pr = $\frac{C_p \mu}{K}$

thermal diffusivity

Prandtl num

Eckert num

$B_r = Pr Ec$

$Ec = \frac{v^2}{C_p \Delta T}$

Thermal conductivity

$T_w - T_o$

bulk fluid temp

wall temp

flow viscous heating

conduction

adhesive transport

Heat dissipation potential

manufacturing types for # of parts

$$\frac{P}{\mu} = \frac{1}{N}$$

ΔT diff between wall and local temp.

a) 1

快速原型制造

rapid proto

hard machinery

手工加工

b) 5

CNC

rapid prototyping

Computer Numerical Control

c) 100

investment casting

投资铸造

d) 10,000

die casting

压铸

Materials processing

Chips:
切屑

Continuous chips: ductile material
high rake angle
or high cutting speed
韧性材料

Built up edge: adverse effects on surface finish

reduce by increasing cutting speed

Reduce depth of cut, increase rake angle
金属卷刃粘附在工具上(粘着性)

Serrated chips: metals w/ low k (thermal conductivity)
锯齿状(振动) or low strength (Breaks like glass)
低热导率 热性能差

锯齿状(振动) or low strength (Breaks like glass)
低强度

or due to chatter

discontinuous chips: brittle workpiece, workpiece w/ inclusions
very high or low cutting speed 脆性材料

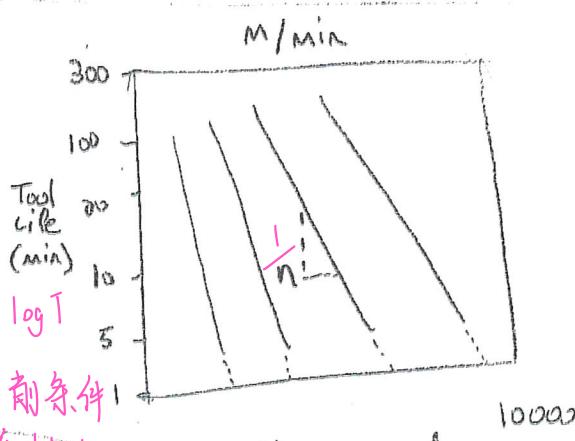
$$\text{Taylor's tool life: } V T^n = C$$

V = cutting speed 切削速度, m/min

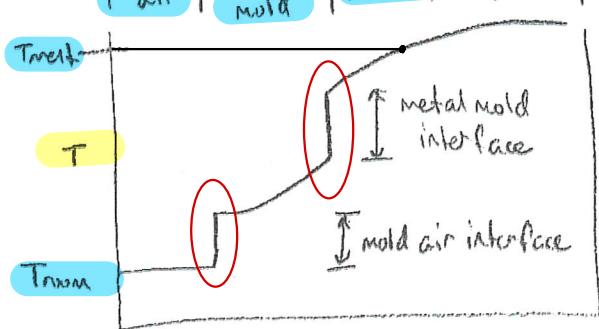
T = tool life 刀具寿命, min

n = slope Tool life index

C = constant 1 min tool life
constant Machining index, 取决于切削条件

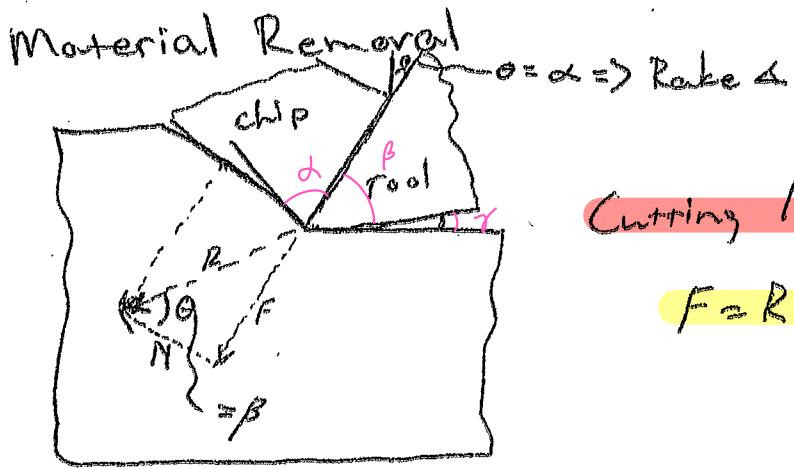


Mold casting Temperature profile 和材料



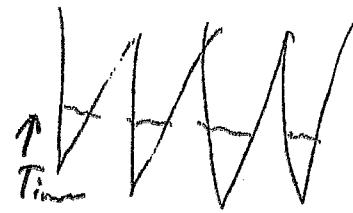
$$\log T = \frac{1}{n} (\log C - \log V)$$

铸造中心



Cutting Force

$$F = R \sin \beta$$



Taylor tool-wear $V T^n = C$

V = cutting speed, T = time in minutes to develop a flank wear land; n depends on cutting conditions, C is a constant

MRR - material Removal Rate, Volume of mtl removed per unit time

* In Turning, $MRR = \pi D_{avg} df N$

Cutting time $t = \frac{l}{fN}$

$$V = \pi D_0 N = \text{turning speed.}$$

切削速度 (m/min)

f = feed or distance tool travels in one revolution

N = # rev per min

l = workpiece length.

D_{avg} gives depth of cut

$$D_{avg} = (D_0 + D_{final})/2$$

* In Milling

$$V = \pi D N$$

$$t_c = 2f \sqrt{\frac{d}{\pi}}$$

$$f = \frac{v}{Nn}$$

$$t = \frac{l + l_c}{v}$$

$$MMR = \frac{Lwd}{t}$$

t_c = chip thickness, d = depth of cut

D = Diameter of cutter

N = rotational speed, f = feed per tooth of cutter, v = linear speed, n = # teeth

t = cutting time, l = length of workpiece

l_c = extent of cutters first contact

w = width of cut

l: 刀具接触工件的长度

w: 切削宽度

d: 切削深度

$$\eta = \frac{\tau}{\dot{\gamma}}, \text{ viscosity, 粘度}$$

Shear VS. Shear rate

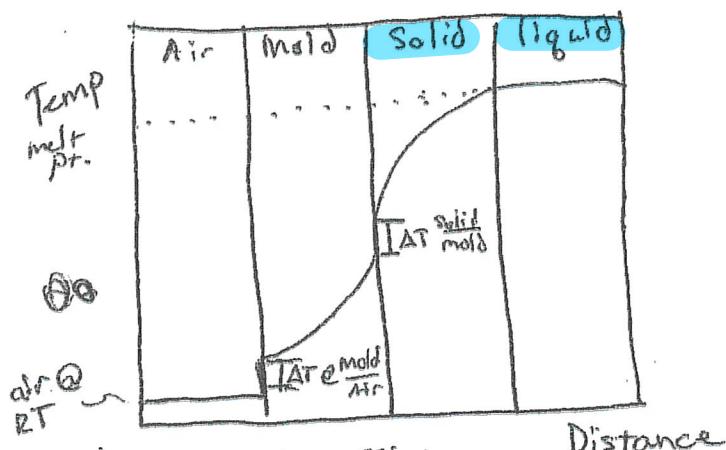


Metal Casting. 流动性 船模

- Viscosity: fluidity \downarrow as viscosity \uparrow
- Surface tension: fluidity \downarrow w/ high surface tension
- Inclusions: insoluble 不溶的 颗粒, 阻碍流动 particles that can effect fluidity
- Solidification pattern of the alloy: how a material solidifies can influence fluidity. Fluidity is inversely prop. to the freezing range: Thus pure metals $= \uparrow$ fluidity. Alloys $= \downarrow$ fluidity
- Mold design: mold components can influence fluidity
- Mold Mat + Surface: higher the thermal conductivity & rougher the surface, \downarrow the fluidity. Heating mold improves fluidity, but increases grain size which lowers strength.
- Degree of Super-Heat: \uparrow fluidity by delaying solidification - Temp above meltin point
- Rate of pouring: \downarrow pouring rate, \downarrow fluidity b/c faster cooling.
- Heat Transfer: affects viscosity \therefore fluidity.

Testing for fluidity - not universal, but a molten mt/ can be made to flow through a channel. The distance the mt gets to b4 solidification can help define its fluidity.

Heat Transfer in A Mold



凝固时间公式

$$\text{Solidification time} = C \left(\frac{\text{Volume}}{\text{Surf. Area}} \right)^n$$

$C = \text{constant dependent on mold mat}$
 $n = \text{normally } (2) \text{ (but is actually between } 1.5 \text{ to } 2)$

热传导方程

Heat eq. for conduction

$$q'' = -k \frac{\partial T}{\partial x} \quad C_p [J/kg.c]$$

$P \rightarrow \text{density}$

$$\frac{\partial T}{\partial x} = P C_p \frac{\partial T}{\partial t} \quad T = P C_p \frac{\partial T}{\partial t} \frac{x^2}{2} + C_1 x + C_2$$

Material Selection

Know mechanical properties

strength, toughness, ductility, hardness, resistance

+ fatigue, creep + impact.

Know physical properties

density, melting pt., specific heat, conductivity, thermal expansion, magnetic properties.

Know chem. properties

oxidation, corrosion

Mtls selected should

- exceed min requirements + specs.
- can mtl be replaced w/ cheaper mtl.
- have appropriate manufacturing characteristics.
- Raw mtls have standard shapes, tolerances, dimensions, surface finish
- reliable supply

材料选择标准

在选择材料时，需要确保材料能满足以下几个标准：

1. • **满足设计要求和规格**: 选择的材料需要达到设计中对强度、耐久性、重量等方面的要求。
2. • **成本效益**: 评估是否有更便宜的材料可以替代，同时达到相同的性能。
3. • **制造特性**: 材料应适合预定的制造过程，如铸造、焊接、机加工等。
4. • **标准形状和尺寸**: 选用的原材料应有标准的形状和公差，以减少制造成本和复杂性。
5. • **可靠的供应链**: 确保所选材料的供应稳定，以免影响生产计划。

2. 物理特性:

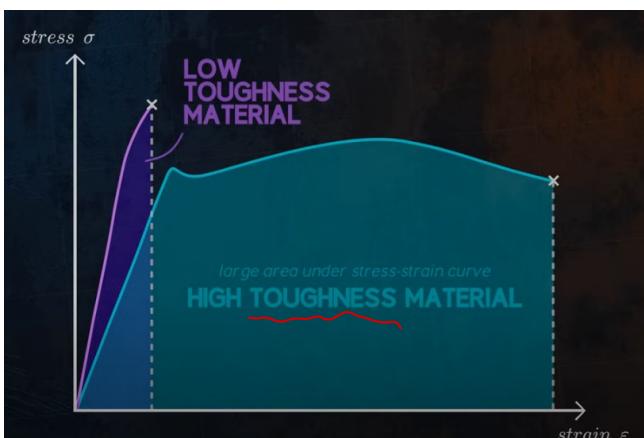
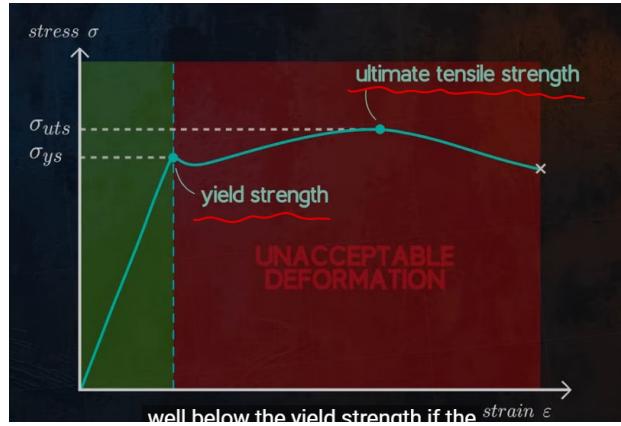
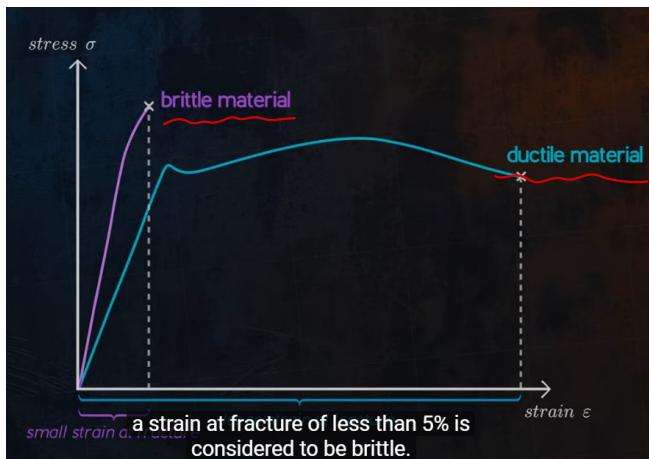
- **密度**: 单位体积的质量。
- **熔点**: 材料从固态转变为液态的温度。
- **比热容**: 单位质量的材料升高单位温度所需的热量。
- **导热性**: 材料传导热能的能力。
- **热膨胀性**: 温度变化时，材料体积或长度变化的性质。
- **磁性特性**: 材料响应磁场的特性。

3. 化学特性:

- **抗氧化性**: 材料抵抗在氧化环境中反应的能力。
- **耐腐蚀性**: 材料抵抗环境因素，如湿度、酸、盐等引起的腐蚀的能力。

1. **强度 (Strength)**: 材料在受到力时抵抗形变和破坏的能力。
2. **韧性 (Toughness)**: 材料吸收能量并且在受到冲击时抵抗断裂的能力。
3. **延展性 (Ductility)**: 材料在受力后能够拉长的程度。 *plastic range ↑*
4. **硬度 (Hardness)**: 材料抵抗硬物压入或划伤的能力。
5. **耐疲劳性 (Fatigue Resistance)**: 材料抵抗反复应力或应变而不发生疲劳破坏的能力。
6. **抗蠕变性 (Creep Resistance)**: 在高温下材料抵抗持续变形的能力。
7. **抗冲击性 (Impact Resistance)**: 材料抵抗突然冲击或撞击的能力。

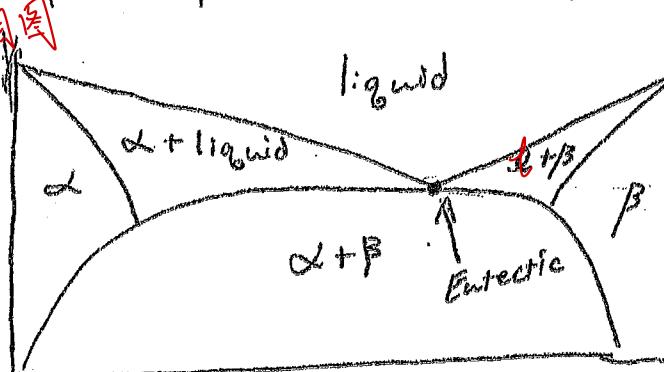
- Strength:** This is a measure of the maximum amount of stress a material can withstand while being stretched or pulled before failing or breaking. Strength is typically measured in terms of tensile strength (the resistance to being pulled apart) and compressive strength (the resistance to deformation under compression).
- Ductility:** Ductility refers to a material's ability to deform under tensile stress. Highly ductile materials can be stretched into a wire. Materials with high ductility, such as gold and copper, are typically capable of being shaped or molded without breaking, which is a crucial attribute for many manufacturing processes.
- Toughness:** Toughness is the ability of a material to absorb energy and plastically deform without fracturing. It combines strength and ductility, providing an indication of the energy a material can absorb before it breaks. A tough material can withstand impact and sudden forces well, making it valuable in applications where shock resistance is important.



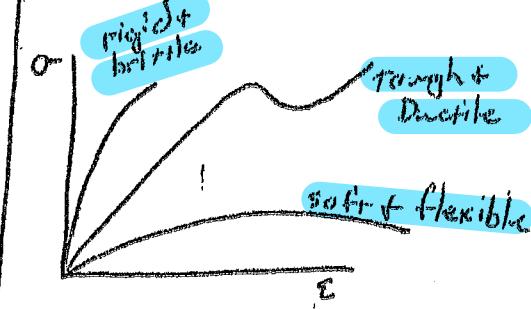
Phase diagrams

Eutectic point - point where liquid phase can transfer directly to solid

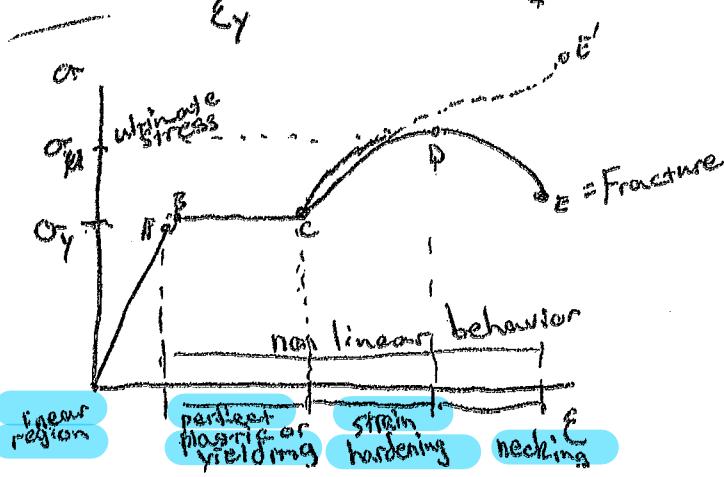
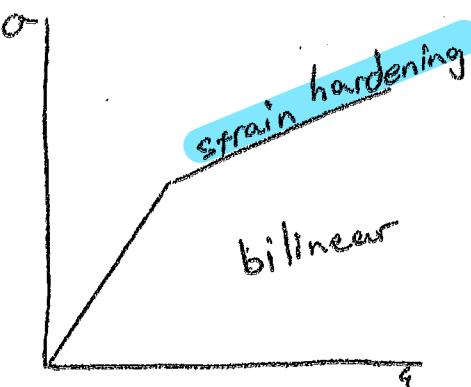
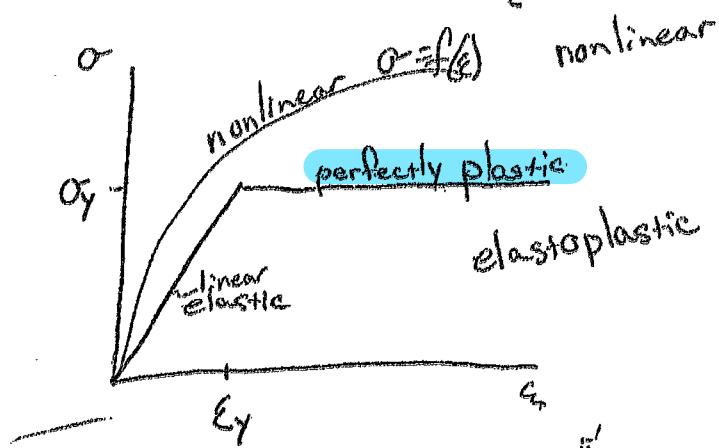
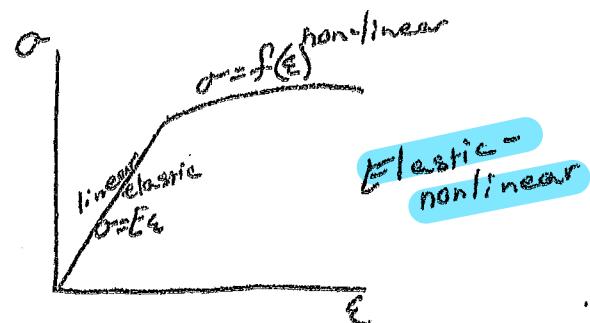
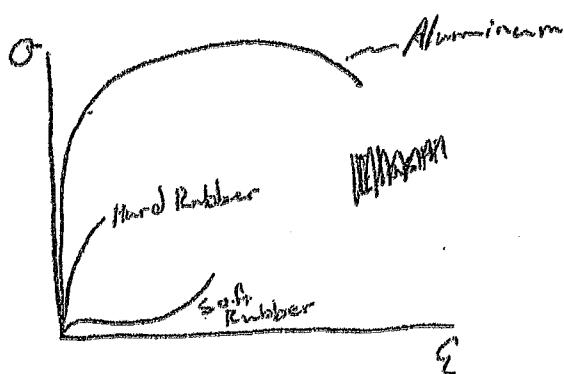
二元合金相图



Stress-Strain, types of int'l



Stress-Strain Curves



↑ strain rate (like during tensile testing under variable speeds), ↑ strength.
And the sensitivity of strength to strain rate ↑ as temp ↑.

1. 共晶反应

■ 共晶 (eutectic) : α

与 β 同时自液相结晶;

■ 共晶点E: 液相线AE

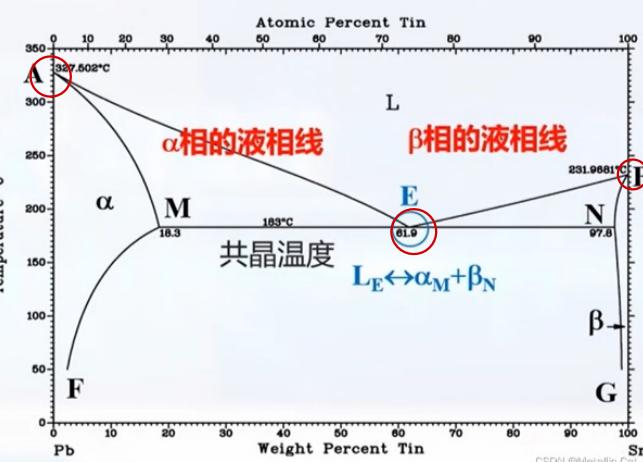
与BE相交点;

■ 二元共晶为不变反应;

■ 共晶转变线MEN: 发

生共晶相变的合金; 包

含三条共轭线。



~~flow~~ flow (simple)

EOM: $\rho \left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} \right) = -\frac{\partial P}{\partial x} + \left(\frac{\partial T_{xx}}{\partial x} + \frac{\partial T_{yx}}{\partial y} + \frac{\partial T_{zx}}{\partial z} \right)$

in x component
X is flow direction

$$\therefore 0 = \frac{\partial T_{yx}}{\partial y} \quad T_{xx} = C$$

for Newtonian fluids, $C = \eta \Delta = \eta \frac{\partial u_x}{\partial y}$ ← velocity gradient
b/c of b.c. 1+2
viscosity

$$u_x = \frac{V_y}{h} \quad \frac{U_y}{V} = \frac{y}{h} \quad \text{so, dimensionless velocity } \propto \text{dimensionless position}$$

$Q = \text{volumetric flow rate} \Rightarrow Q = (U_x) A_{\text{cross}}$
 $A = U_x(h \cdot w)$

Tube flow continuity eq. $\frac{1}{r} \frac{\partial}{\partial r} (r u_r) + \frac{1}{r} \frac{\partial u_x}{\partial x} + \frac{\partial u_z}{\partial z} = 0$



$\therefore U_r = u_r = 0$, f.d.f.

$\frac{1}{r} \frac{\partial}{\partial r} (r u_r) + \frac{\partial u_z}{\partial z} = 0$ if $u_r = 0$ then $\frac{\partial u_z}{\partial z} = 0$ f.d.f.

$z(\text{eom}) \quad \rho \left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_z \frac{\partial u_x}{\partial z} \right) = -\frac{\partial P}{\partial z} + \left(\frac{1}{r} \frac{\partial}{\partial r} (r u_z) + \frac{\partial T_{rz}}{\partial z} \right)$

θ -sym

+ Pg_z

$$\frac{\partial P}{\partial z} = \frac{1}{r} \frac{\partial}{\partial r} (r T_{rz}) \quad \frac{\partial P}{\partial z} = \frac{\Delta P}{L} \quad T_{rz} = -C \left(\frac{\partial u_z}{\partial r} \right)$$

neglect stress for shear flow

$$\frac{\partial u_z}{\partial r} = \frac{\Delta P r}{2 \mu L} \quad u_z = \frac{\Delta P r^2}{4 \mu L} - \frac{\Delta P R^2}{4 \mu L}$$

$$Q = \int_0^R u_z (2\pi r) dr$$

$$Q = \frac{\Delta P}{4 \mu L} (2\pi) \int_0^R [r^2 - R^2] r dr$$

Pressure Driven Slit flow

$$0 = -\frac{\partial P}{\partial z} + \left(\frac{\partial T_{xz}}{\partial x} + \frac{\partial T_{yz}}{\partial y} + \frac{\partial T_{xz}}{\partial z} \right) + Pg_z$$

$$\frac{\partial P}{\partial z} = \frac{\partial T_{yz}}{\partial y}$$

$$P = \frac{\partial T_{yz}}{\partial y}(z) + C_1$$

Machining

Cutting ⑥

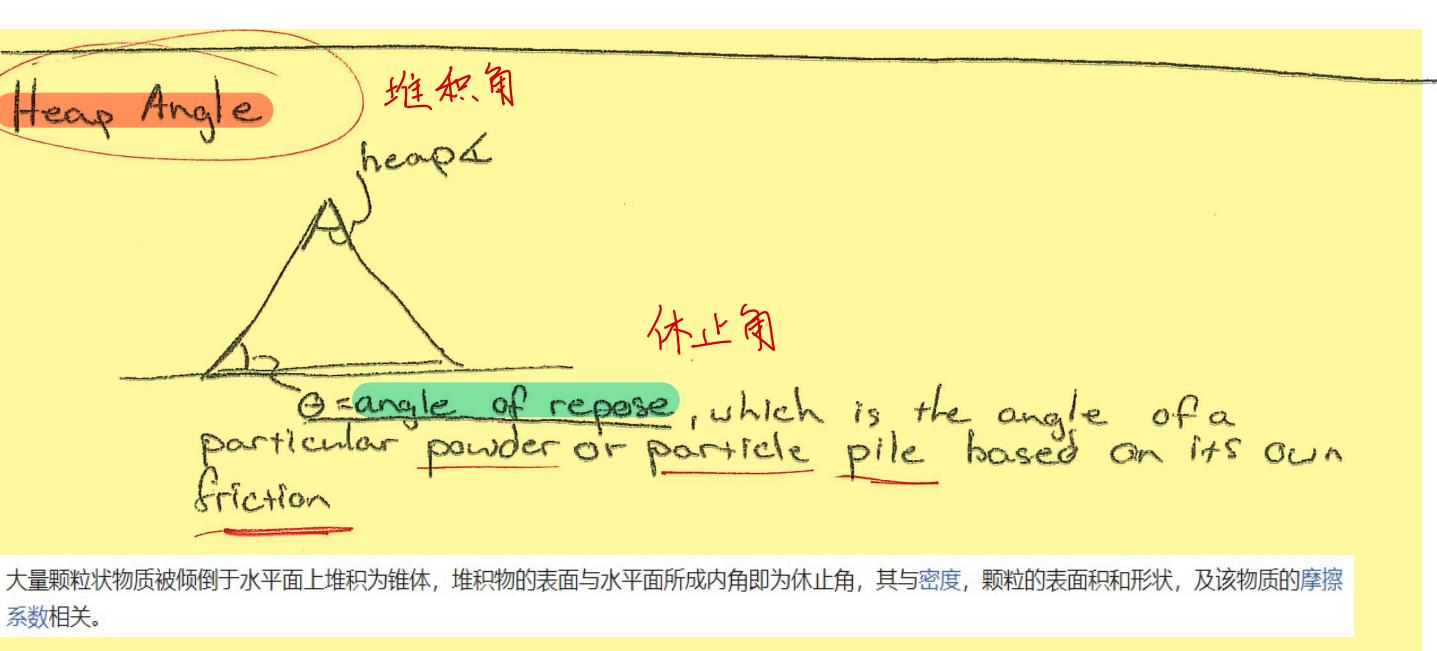
grinding ⑤

milling ③

polishing ①

sanding ④

turning ②



大量颗粒状物质被倾倒于水平面上堆积为锥体，堆积物的表面与水平面所成内角即为休止角，其与密度，颗粒的表面积和形状，及该物质的摩擦系数相关。

1. 锯割 (Sawing)

- **应用:** 初步将材料切割成大致需要的尺寸和形状，通常是加工过程的第一步，用于准备材料块。
- **排序:** 首步，因为通常先要将原材料切割成合适的工作尺寸。

GPC

2. 车削 (Turning)

- **应用:** 用于生成圆柱形或其他旋转体形状的部件，可以迅速去除较大的材料量。
- **排序:** 紧随锯割，尤其是在需要处理圆柱形或轴向对称的零件时。

3. 铣削 (Milling)

- **应用:** 进一步形状加工，适用于创建复杂的几何形状、槽口、孔和平面。
- **排序:** 在车削之后，用于在多个平面上或在不规则形状上进行精细加工。

STEM

4. 磨削 (Grinding)

- **应用:** 提供非常高的表面精度和光洁度，用于完成加工，确保零件符合严格的尺寸容差和表面光洁度要求。
- **排序:** 在大部分形状加工完成后进行，用于最终精加工表面。

5. 抛光 (Polishing)

- **应用:** 最终增强工件表面的美观和光滑度，通常作为最后一步进行，用以提高外观质量和减少表面缺陷。
- **排序:** 最后步骤，用于完成表面处理。

6. 切割 (Cutting)

- **应用:** 这里的“切割”可能特指某种特殊的切割方式，如激光切割、等离子切割等，用于精确切割或在需要非常精细的外形定义时使用。
- **排序:** 可能在任何需要精确形状定义的阶段使用，具体位置取决于具体的制造流程和零件要求。

M+s.

Chap(1) metals atomic structures of metals: bcc, fcc + hcp, or body-centered cubic, face-centered-cubic, hexagonal-close-packed.
More than one type of crystal in a metal is allotropism or polymorphism 多态性

plastic deformation; slipping due to shear stress & twinning
shear stress - ratio of shear force to x-sect. area being sheared
slip in single-crystal occurs along planes of max atomic density

- Defects + impurities in crystal structures are the root cause of discrepancy in strength values between actual & theoretical calculations

- Point Defects - vacancies, interstitial atom, & impurities

- Linear Defects - dislocations (1D)

- Planer Impurities (2D) - grain boundaries & phase boundaries

- Volume Impurities (3D) - voids, inclusions, cracks

- Dislocations: defects in the orderly arrangement of a metal's atomic structure

- Slip plane containing a dislocation requires less shear stress to allow slip.

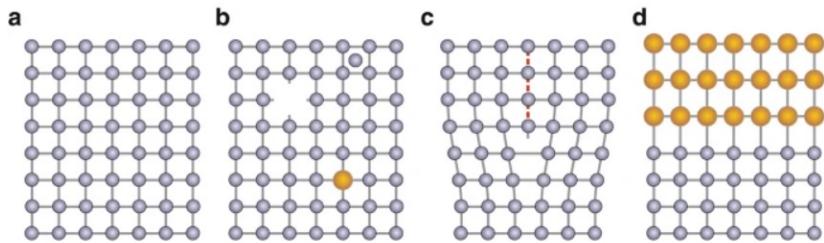
- Entanglements + Impediments: of dislocations increase shear stress required to cause slip. which results in overall ↑ in strength & hardness (strain or work hardening). ↑ deformation \Rightarrow ↑ # of entanglements \Rightarrow ↑ metal strength

- Work hardening: cold rolling sheetmetal for auto + aircraft bodies, forging bolt head, x sec reduction to wire to increase strength called drawing

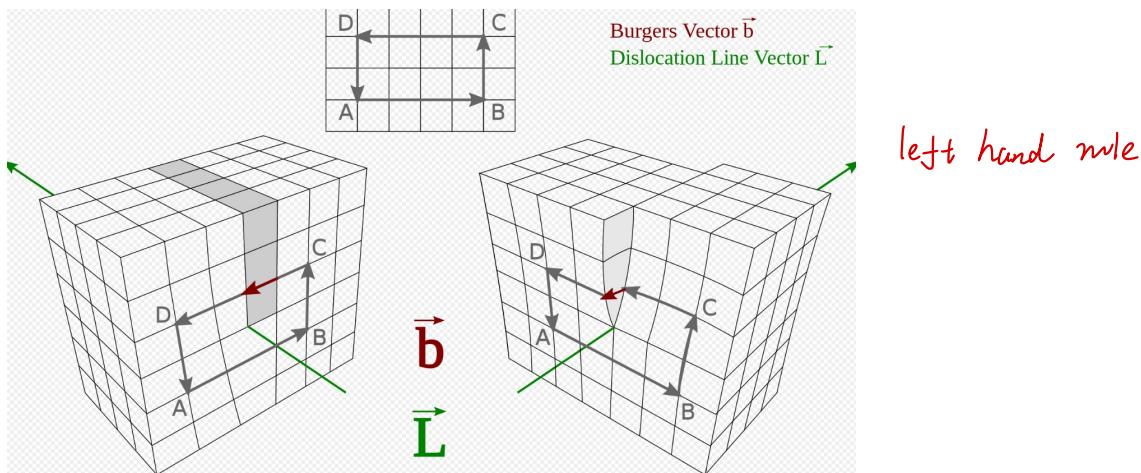
- Rapid cooling produces small grains (quench) & slow cooling, make large grains (rough surface & low strength, hardness)

- ↑# Grain boundaries \Rightarrow ↑ strength & ductility of metals as they interfere w/ the movement of dislocation & influence strain hardening

Fig. 1



(a) Two-dimensional representation of an ideal cubic lattice. (b) Point defects: a vacancy, an interstitial, and an impurity atom. (c) Line defect: cross section perpendicular to a dislocation line located at the end of the inserted lattice plane marked in red. (d) Planar defect: interface at the heterojunction of two dissimilar solids



位错的增加确实会降低单个位错滑移所需的局部剪切应力，但从宏观角度来看，材料的硬度和强度会提高。这是由于以下几个因素的相互作用：

- 1. 位错的相互作用：**当位错的密度（位错的数量和分布）增加时，它们之间的相互作用也增加。位错之间的相互阻碍会使得它们的移动更加困难。位错移动是材料塑性变形的主要机制，因此位错移动受阻意味着更大的外力（应力）是必需的才能使材料发生形变。
- 2. 位错与晶界的相互作用：**位错在穿越晶粒界时会遇到阻力。晶粒越小，晶界越多，位错在通过这些晶界时受到的阻碍也越多。因此，增加位错密度通常与提高晶界阻力相结合，可以有效提高材料的强度和硬度。
- 3. 应变硬化 (Work Hardening)：**在塑性变形过程中，由于位错的增加和重新排列，材料的硬度和强度逐渐增加。这种现象称为应变硬化或加工硬化。位错密度的增加导致位错网络更复杂，任何进一步的变形都需要更大的应力。

总结来说，尽管位错的增加在局部降低了单个位错所需的剪切应力，但在宏观上，由于位错的相互作用和位错与晶界的相互阻碍，整体材料的硬度和强度实际上会提高。这是因为整体上位错的运动更加困难，需要更大的力来推动这些位错穿越材料。

Which depends on temp., deformation rate + type & amt of impurities along the grain boundaries

- Grain boundaries are more eff reactive than the grains themselves, b/c atoms along the boundaries are packed less efficiently $\Rightarrow \downarrow$ energy than atoms in the mxt & can be easily removed & chemically bonded to another atom.

- GB embrittlement: a normally ductile & strong metal wetted w/ a brittle material causing the entire mxt to crack when subjected to \downarrow external stresses.

- During Plastic Deformation (tension or compression), The material exhibits \uparrow strength b/c of the entanglements of dislocations w/ boundaries. The \uparrow in strength depends on the degree of deformation (strain) ie, \uparrow deformation $\Rightarrow \uparrow$ strength of the mtl.

- Plastic Deformation is caused by grain boundary sliding (creep)

- As strength \uparrow \Rightarrow ductility \downarrow

- Recrystallization: the process in which (within a certain temp range) new equiaxed & strain free grains are formed (replacing old grains) 加热到再结晶温度

在冷加工后

- Cold working results in \uparrow strength $\Rightarrow \downarrow$ ductility + causes anisotropy (preferred orientation or mechanical fibering). Properties are different in different directions.

- Annealing - 特定环节 温镍通常高再结晶温度 Heating the metal in a certain temp range for a period of time + allows successive process of recovery (recrystallization) & grain growth to take place

- Effects of cold working can be reversed by annealing the metal

chap ② Mechanical behavior

- Mechanical properties extracted from simple tension test: strength, ductility, toughness, E, strain hardening capability

- Ductility - plastic deformation that the mtl undergoes b/4 fracture Total area under stress strain curve

- Measure of duct. total elongation / reduction in area

Toughness - area under stress-strain curve from ultimate strength to fracture

Yield stress - stress where elastic deformation ends & plastic deformation begins. In soft ductile mtl's, yield stress is defined as the pt on the stress-strain curve that is offset by a strain of 0.2% elongation

$$\sigma = E \epsilon$$

- E Young's - ratio of stress to strain in elastic region, Hooke's law - stiffness, $\uparrow E \Rightarrow \uparrow$ load to stretch $\Rightarrow \uparrow$ stiffness

横向应变 / 纵向应变

- ν Poisson's - ratio of lateral strain to longitudinal strain

- \uparrow temp \Rightarrow \uparrow ductility & toughness \Rightarrow \downarrow yield stress & ultimate tensile strength & E

- Deformation rate: speed @ which tension test occurs

- Strain rate - $\frac{\Delta \epsilon}{\Delta t}$

- Tension test 抗拉测试 to determine shear properties of a mtl.

- G (Shear Modulus) ratio of shear stress / shear strain in elastic range.

- \uparrow # twists in a round shaft in torsion prior to failure \Rightarrow better the forgeability of mtl.

- flexural test 抗弯强度测试 conducted on brittle mtl's to measure the mod of rupture or transverse rupture strength on other mtl's to get flexural strength.

- Hardness: 维氏硬度测验 gives a general indication of the strength of the material & its resistance to scratch & wear

- Hardness is not a fundamental property b/c it depends on the shape of the indenter & applied load.

- Fatigue - progressive & localized structural damage that occurs when a mtl is subjected to cyclic loading which is below yield limit, Ultimate Tensile Stress

S-N curves - plot stress amplitude (σ) + # of cycles (N_f) + are associated w/ fatigue.

stress amplitude - max stress (tension + comp) to which the specimen is subjected.

endurance limit - max stress to which the matl can be subjected w/out fatigue failure.

creep - permanent elongation of a component under a static load maintained over a period of time. creep @ elevated temps is attributed to grain-boundary sliding. 在恒定的静态载荷下，随时间发生蠕变。

- resistance to creep ↑ as melt temp ↑ 在持续固态形变的条件下，

stress relaxation - ↓ in magnitude of stresses resulting from loading over a period of time, even though the dimensions of the component remains constant. 材料内部之力随时间逐渐减小而观察

Impact test: consists of placing a notched specimen in an impact tester & breaking it w/ a pendulum (Tod & Charpy)

- **Types of failure**: brittle + ductile fracture (due to external cracks + buckling (compressive loading))

- **Ductile fracture**: plastic deformation that precedes failure & takes place along planes of max shear stress.

- In simple shear, failure occurs due to excessive slip along slip planes w/in the grains.

factors affecting void formation - strength of bond between an inclusion + matx + hardness of inclusions

mechanical fibering: alignment of inclusion during plastic deformation.

strain aging - a phenomenon where carbon in steels segregate to dislocations + pinning them down + ↑ resistance to dislocation movement

Brittle fracture - no plastic deformation, fracture along crystal plane

improve fatigue strength - induced comp. residual stresses on surfaces, case hardening, fine surface finish

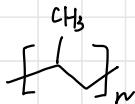
stress corrosion cracking - ductile matl fails brittly by cracks over time

Annealing - could be used to help stress-relieve parts to avoid stress corrosion, however annealing can ↓ strength

Hydrogen embrittlement: presence of hydrogen can reduce ductility + cause failure

Residual Stress: stresses that remain w/in a part after being formed + all external forces removed. reduced by annealing

Polypropylene (PP)



Co-polymers

two or more monomer types

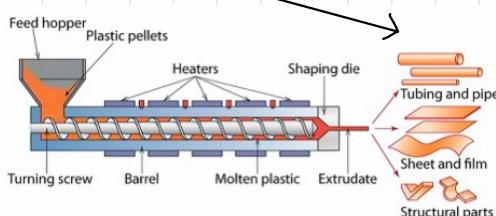
Polymerization

Polymer Blends

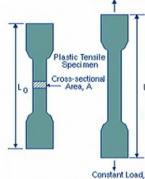
Mixing or blending two or more polymers

Polymer composites

Injection Mold — thermoplastic materials



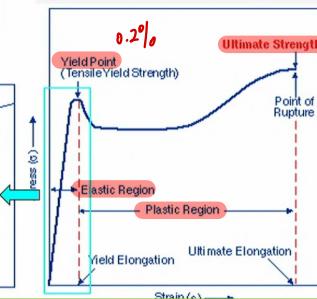
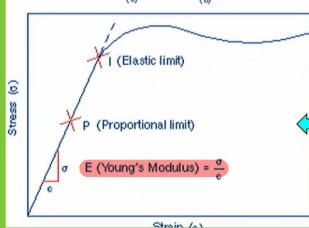
Tensile Properties and Young's Modulus



$$\text{stress } (\sigma) \equiv \frac{\text{loading force } (F)}{\text{cross-sectional area } (A)}$$

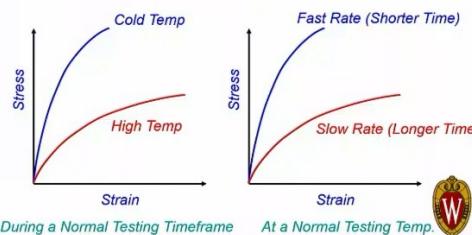
$$\text{strain } (\epsilon) \equiv \frac{L - L_0}{L_0}$$

dimensionless



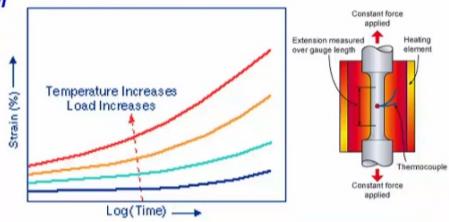
toughness —
area under curve

Short-term Tensile Test (Effects of Temperature and Strain Rate)

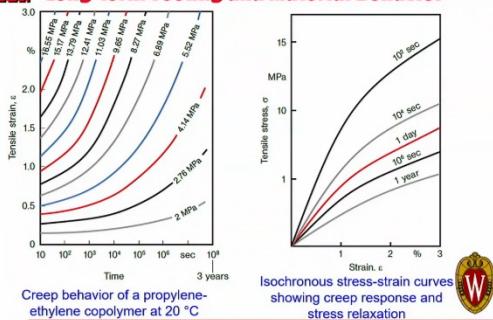


Long-term Testing: Creep

Long-term Testing



Long-term Testing and Material Behavior



Repeated Loading/Testing: Fatigue

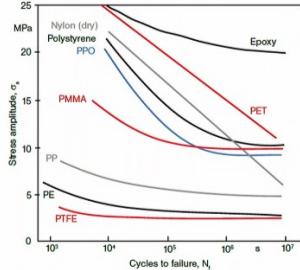
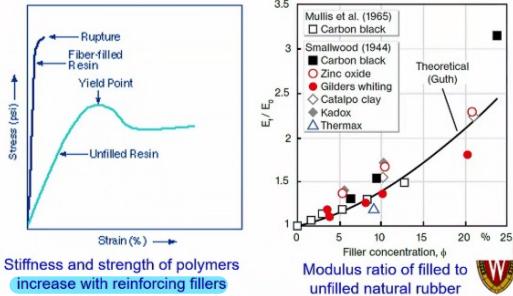


Figure 2.20 Stress-life (S-N) curves for several thermoplastic and thermoset polymers tested at a 30 Hz frequency about a zero mean stress

Mechanical Property of Filled Polymers



Rheology Example 3: Deborah Number

Deborah No. (De):

$$De = \frac{\text{Material's relaxation time}}{\text{Processing (Observation) time}}$$

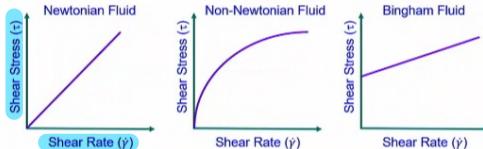
0 ← De → ∞
Viscous fluid ← De → Elastic solid



Silly putty

What Is Rheology Anyway?

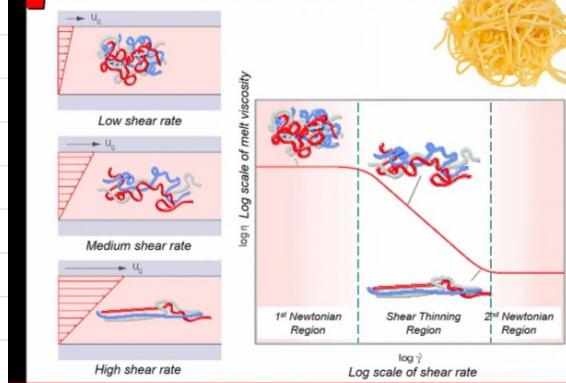
Greek words *rhéō*, "flow" and *-logia*, "study of"



"Rheology" or "Theology"?

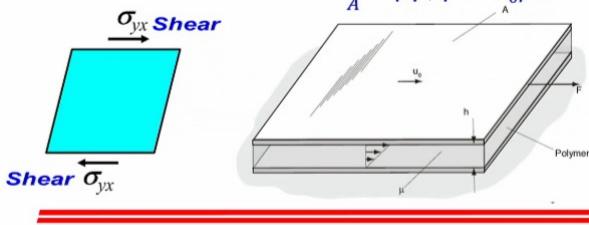


Rheological Behavior



Some Definitions

- Viscosity – Material's Resistance to Flow
- Shear Viscosity = Shear Stress / Shear Rate ($\mu = \tau / \dot{\gamma}$)
- Simple Shear Flow

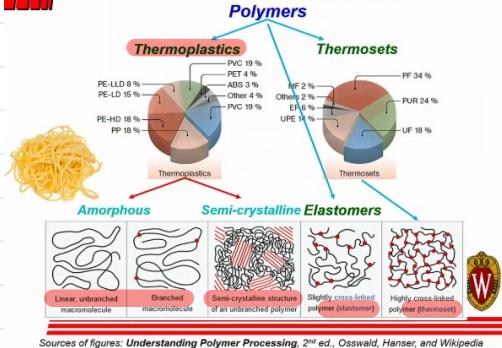


τ – shear stress

μ – 流体层加速度

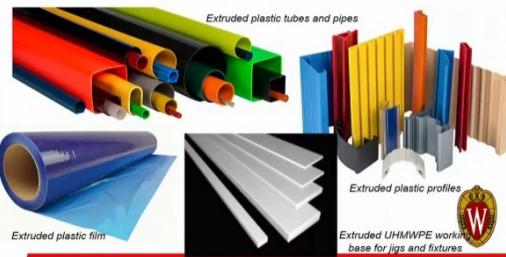
$\dot{\gamma}$ – 不同层之间加速度梯度

Classification of Polymers

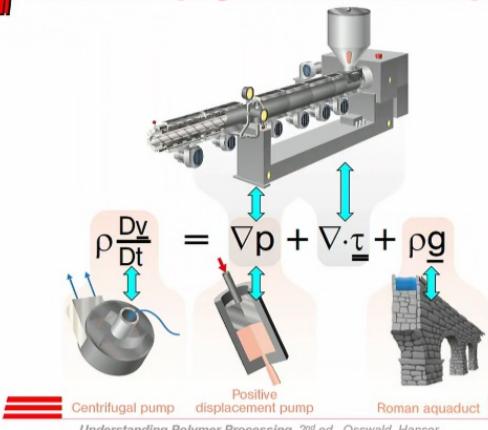


What Is Extrusion?

Extrusion: pumping a polymer melt through a shaping die to form a profile, e.g., a plate, a film, a tube, or a profile with any cross-sectional shape.



■■■ Various Pumping Mechanisms (Principles)



■■■ Single Screw Extruder and Three Screw Zones

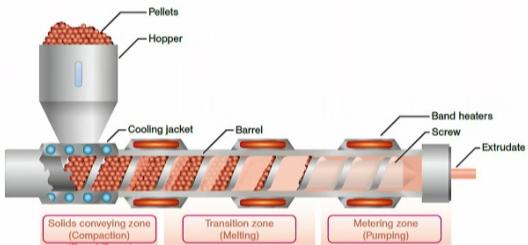
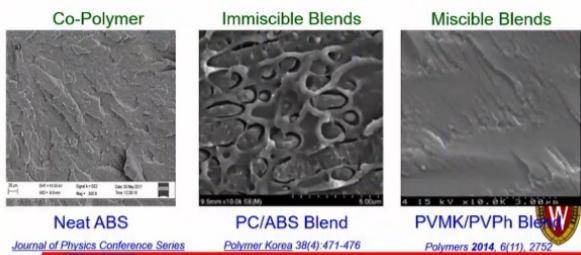


Figure 4.5 Schematic of a plasticating single screw extruder

■■■ Polymer Blends vs. Co-Polymers

"Co-polymers contain two or more monomer types within a single polymer chain (this occurs during synthesis), whereas polymer blends are made via mixing techniques (combination of multiple polymers, which generally occurs after synthesis)"
www.sciencedirect.com



Journal of Physics Conference Series
1002(1) 012010

■■■ Distributive vs Dispersive Mixing

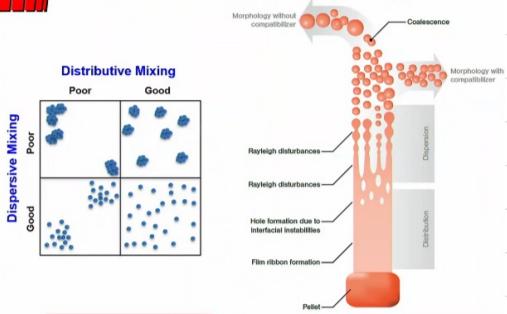
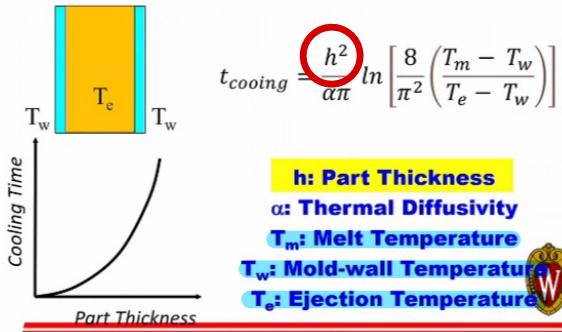


Figure 5.1 Mechanism for morphology development in polymer blends

■■■ Part Thickness and Cooling Time

Averaged cooling time for flat plates:



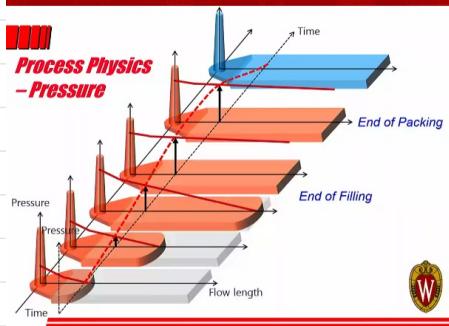
h: Part Thickness

α: Thermal Diffusivity

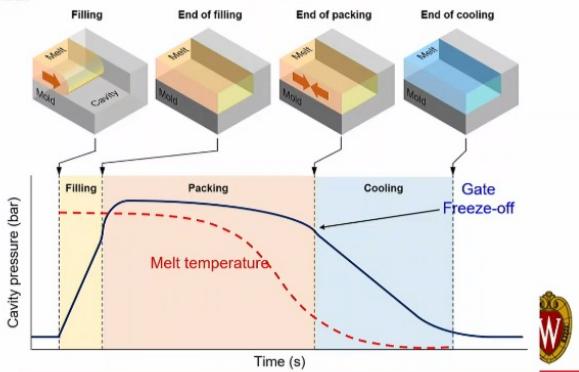
T_m: Melt Temperature

T_w: Mold-wall Temperature

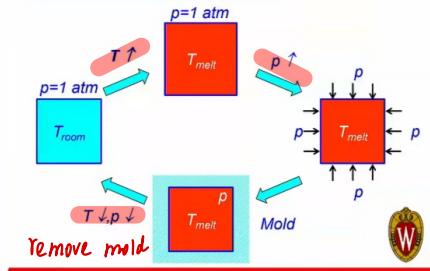
T_e: Ejection Temperature



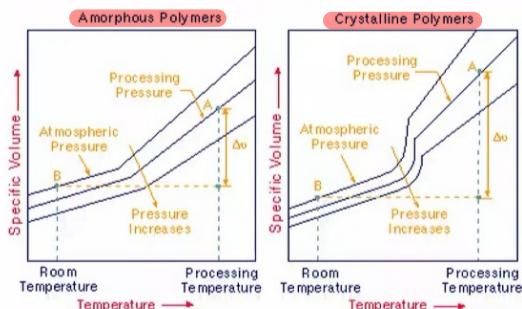
Process Physics – Pressure



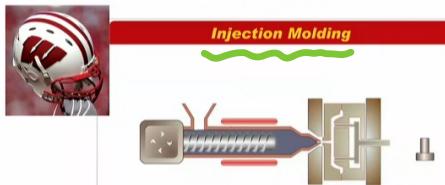
Relationship among Pressure, Volume, and Temperature



Schematic pVT Diagram



Manufacturing of Football Helmets



Polymer Engineering Center
University of Wisconsin - Madison

Modeling for Materials Processing



A stainless steel turbine wheel

- What manufacturing method?
 - CNC
 - Die casting
 - Investment casting
 - Sand casting + machining
 - Additive manufacturing/
3D printing
 - **Injection molding (really?)**
- It all depends
 - Quantity, quality, & time
 - Resources and facility

(1) Analytical Approach

Cartesian Coordinates (x, y, z):

$$\rho \left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} \right) = - \frac{\partial p}{\partial x} + \left[\frac{\partial}{\partial x} \tau_{xx} + \frac{\partial}{\partial y} \tau_{xy} + \frac{\partial}{\partial z} \tau_{xz} \right] + \rho g_x$$

Body force neglected

Cartesian Coordinates (x, y, z):

$$\rho \sqrt{\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z}} = - \frac{\partial p}{\partial x} + \mu \left[\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} + \frac{\partial^2 u_x}{\partial z^2} \right] + \rho g_x$$

Steady state

$$u_x = 0$$

u_x independent of z

$$\rho \left(u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} \right) = - \frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} \right)$$

Inlet Uniform velocity

Impenetrable wall

$\frac{\partial^2 u_x}{\partial y^2} = \frac{1}{\mu} \frac{dp}{dx}$

$u_x(y) = \frac{h^2}{8\mu} \frac{dp}{dx} \left[1 - \left(\frac{2y}{h} \right)^2 \right]$

$= \frac{h^2 \Delta p}{8\mu L} \left[1 - \left(\frac{2y}{h} \right)^2 \right]$

$Q = hW\bar{u}_x = \frac{Wh^3 \Delta p}{12\mu L}$

Source: www.sciencedirect.com

If focus on fully developed flow...

"fully developed region"是指在流体力学或传热学中，流动或温度场已经达到平衡的区域。在这个区域内，流体的速度或温度分布不再随流动方向变化。具体来说，这意味着：

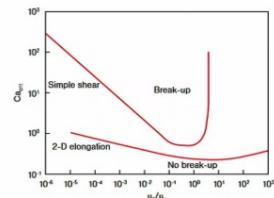
- 1. 流体流动：**在管道或通道内，流体进入的最初阶段，流动会受到入口效应的影响，速度分布会随着距离逐渐变化，直达到达某一段距离后，速度分布稳定下来，即进入“充分发展区域”(fully developed region)。在这一区域，流体的速度分布不再随流动方向变化。
- 2. 传热过程：**类似地，在传热过程中，进入通道的流体温度分布会随着距离变化，直达到达某一段距离后，温度分布也会稳定下来，不再随流动方向变化，这一段距离之后的区域也称为“充分发展区域”。



- T (F) A plastic component under a constant load will deform with time, a phenomenon called **stress relaxation**. **Creep**
- T (F) Thermoplastics can be repeatedly softened (or hardened) by an increase (or decrease) in temperature.
- T (F) Plastics exhibit viscoelastic behavior. Therefore, their mechanical properties, such as the stress-strain curves, depend on the temperature and duration, but not on the rate at which the load is applied.
- T (F) In thermoplastics injection molding, if one doubles the part thickness, the cooling time will also increase by approximately 100% (i.e., the cooling time will be twice as long).
- T (F) For macromolecules like polymers, their weight-average molecular weight will always be lower than their number-average molecular weight.



The graph below shows the critical capillary number for drop break-up as a function of viscosity ratio in a simple shear flow and a 2-D elongational flow.



T (F) This graph shows that the simple shear flow is more effective in breaking up the droplets than the 2-D elongation flow.

T (F) The capillary number, Ca , can be considered as the ratio of flow stresses to droplet surface stresses, and it has a unit of $[g \cdot cm/sec^2]$.

Dimensionless



2. Connect the polymers with their corresponding microstructure at the solid state. (3 points)

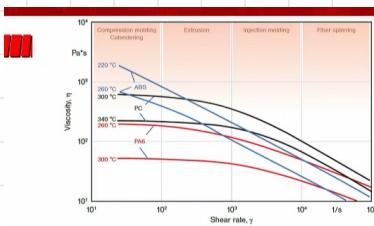
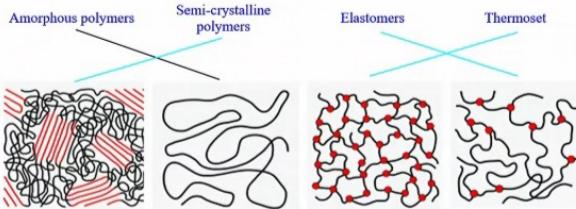


Figure 3.2 Viscosity curves for a selected number of thermoplastics

3(a) Which polymer exhibits the strongest shear thinning behavior at low shear rates?

ABS

3(b) Which polymer has the highest shear viscosity at 300 °C and a shear rate of 100 s⁻¹?

PC

3(c) When one doubles the extrusion rate, which material allows the least torque % increase?

ABS

III 4. Which one of the following rheometers can generate the highest shear rates within the sample?

- Melt flow indexer
- Capillary rheometer
- Cone and plate rheometer
- Parallel plate rheometer

5. Which one of the following rheometers generates a constant-shear-rate throughout the sample?

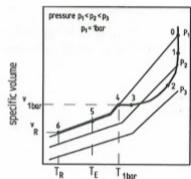
- Melt flow indexer
- Capillary rheometer
- Cone and plate rheometer
- Parallel plate rheometer

6. Which rheometer is often used to characterize a polymer and provide a simple and quick means of quality control?

- Melt flow indexer
- Capillary rheometer
- Cone and plate rheometer
- Parallel plate rheometer

7. Shrinkage is a major concern for injection molding. Please select the one effective strategy for reducing part shrinkage.

- Increasing the part thickness
- Increasing the pack/hold pressure
- Decreasing the pack/hold time
- Decreasing the melt temperature



10. Below are four plastic products that were produced using various polymer processing methods. Fill in the blanks below with the most suitable process used. (6 points)



Milk jug from molded preform
Blow molding



Clamshell package
Thermoforming



PVC elbow
Injection molding

Fill in the circle of all correct answers. There may be more than one correct answer. (4 points)

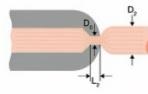
8. What are the potential solutions to fix an incomplete fill problem for an injection molded tensile test sample?

- Increase mold temperature
- Use polymer materials with lower viscosities
- Enhance cooling in critical zones
- Lower holding pressure and shorten holding time
- Increase gate size or shorten the runner length



9. During the extrusion process

- The extrudate swell parameter (γ) can be calculated as D_o/D_i
- The extrudate swell can be reduced by increasing melt temperature
- The extrudate swell can be reduced by increasing die length, L_d
- The extrudate swell can be reduced by increasing extrusion rate
- The extrudate swell can be reduced by modifying die design

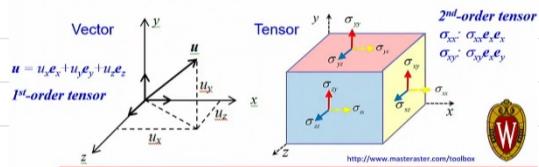


III Basic Concept

Scalar: a value (no direction), e.g., pressure (p), temperature (T), speed, etc.

Vector: both magnitude and direction, e.g., velocity, force, gradient of a scalar function (e.g., pressure gradient), etc.

Tensor: "a multi-dimensional vector", e.g., 2nd-order surface traction (or stress tensor)



<http://www.masterster.com/toolbox>

III Basic Concept: Operators

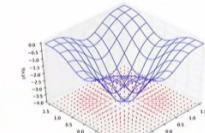
Gradient operator: $\nabla(\cdot)$

Pressure gradient (∇p), temperature gradient (∇T), etc.
 p and T are scalars, but ∇p and ∇T are vectors

$$\nabla = \frac{\partial}{\partial x} \mathbf{e}_x + \frac{\partial}{\partial y} \mathbf{e}_y + \frac{\partial}{\partial z} \mathbf{e}_z$$

$$\nabla p = \frac{\partial p}{\partial x} \mathbf{e}_x + \frac{\partial p}{\partial y} \mathbf{e}_y + \frac{\partial p}{\partial z} \mathbf{e}_z$$

$$\nabla T = \frac{\partial T}{\partial x} \mathbf{e}_x + \frac{\partial T}{\partial y} \mathbf{e}_y + \frac{\partial T}{\partial z} \mathbf{e}_z$$



\mathbf{e}_x , \mathbf{e}_y , and \mathbf{e}_z : unit vectors

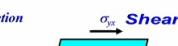


Elongation



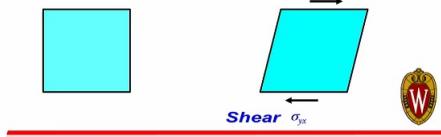
σ_{xx} σ_{yy}

\rightarrow x direction



σ_{xx} σ_{yy}

Shear σ_{xy}



Basic Concept: Operators (Cont'd)

$$\nabla \cdot \sigma = \left(\frac{\partial}{\partial x} e_x + \frac{\partial}{\partial y} e_y + \frac{\partial}{\partial z} e_z \right) \cdot$$

A vector A 2nd order operator tensor

$$(\sigma_{xx} e_x e_x + \sigma_{xy} e_x e_y + \sigma_{xz} e_x e_z +$$

$$\sigma_{yx} e_y e_x + \sigma_{yy} e_y e_y + \sigma_{yz} e_y e_z +$$

$$\sigma_{zx} e_z e_x + \sigma_{zy} e_z e_y + \sigma_{zz} e_z e_z)$$

$$= \sigma_{xx} e_x + \sigma_{xy} e_y + \sigma_{xz} e_z +$$

$$\sigma_{yx} e_x + \sigma_{yy} e_y + \sigma_{yz} e_z +$$

$$\sigma_{zx} e_x + \sigma_{zy} e_y + \sigma_{zz} e_z$$

A vector

$$= (\sigma_{xx} + \sigma_{yy} + \sigma_{zz}) e_x + (\sigma_{xy} + \sigma_{yz} + \sigma_{zx}) e_y + (\sigma_{xz} + \sigma_{zy} + \sigma_{zz}) e_z$$

- Dot product reduces the "dimensional rank" of each of its two operands by one.
- Example here: Dot product of a vector and a 2nd order tensor yields a vector



Transient Derivative Term ($\frac{\partial(\cdot)}{\partial t}$)

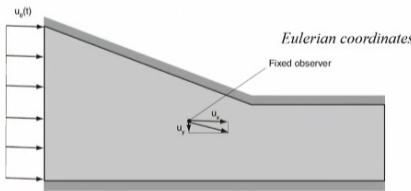


Figure 2.2: Flow system with a fixed observer.

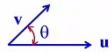
In transient, $\frac{\partial u_x}{\partial t}, \frac{\partial u_y}{\partial t}, \frac{\partial u_z}{\partial t}, \frac{\partial T}{\partial t}$, etc.



Basic Concept: Operators

- "Dot" product operator for vectors/tensors: ":"

$\mathbf{u} \cdot \mathbf{v} = \mathbf{v} \cdot \mathbf{u} = |\mathbf{u}| \times |\mathbf{v}| \times \cos \theta$



$\mathbf{u} \cdot \mathbf{v} = (u_x e_x + u_y e_y + u_z e_z) \cdot (v_x e_x + v_y e_y + v_z e_z)$

$= u_x v_x + u_y v_y + u_z v_z \leftarrow \text{A scalar}$

Note: $e_x \cdot e_x = e_y \cdot e_y = e_z \cdot e_z = 1$

$e_x \cdot e_y = e_x \cdot e_z = e_y \cdot e_z = 0$



Dot product reduces the "dimensional rank" of each of its two operands by one.

Note: scalars have a rank of zero, vectors have a rank of one, second order tensors have a rank of two, etc.

Example here: Dot product of two vectors yields a scalar

Total, Material, or Substantial Derivative ($\frac{D(\cdot)}{Dt}$)

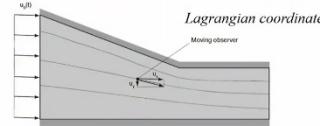


Figure 2.3: Flow system with an observer moving with a fluid particle on a given streamline.

$$\frac{D(\cdot)}{Dt} = \frac{\partial(\cdot)}{\partial t} + \mathbf{u} \cdot \nabla (\cdot)$$

Material, Substantial, or $\frac{Du}{Dt} =$

Total Derivative $\left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} \right) \mathbf{e}_x$

Transient term Convention term $\left(\frac{\partial u_y}{\partial t} + u_x \frac{\partial u_y}{\partial x} + u_y \frac{\partial u_y}{\partial y} + u_z \frac{\partial u_y}{\partial z} \right) \mathbf{e}_y$

$\left(\frac{\partial u_z}{\partial t} + u_x \frac{\partial u_z}{\partial x} + u_y \frac{\partial u_z}{\partial y} + u_z \frac{\partial u_z}{\partial z} \right) \mathbf{e}_z$

1. 固定观察者与随动观察者的区别

- 固定观察者 (Eulerian描述)：观察者固定在空间中的某一个点，记录流体在这个固定点的变化情况。例如，一个温度计放在房间的某个角落，记录这个点的温度变化。
- 随动观察者 (Lagrangian描述)：观察者跟随流体质点一起移动，记录流体质点随时间的变化情况。例如，一个漂浮在河流中的叶子，记录它随水流移动过程中所经历的温度变化。

2. 全导数的物理意义

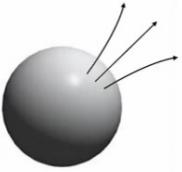
- 全导数 ($\frac{D}{Dt}$) 也叫做物质导数或实质导数，是流体力学中用于描述随动观察者看到的物理量变化的工具。
- 它综合了两部分变化：时间变化和空间变化。
 - 时间变化 ($\frac{\partial}{\partial t}$)：在固定位置上，物理量随时间的变化。
 - 空间变化 ($\mathbf{u} \cdot \nabla$)：由于流体质点在空间中移动，物理量随空间位置的变化。

Basic Concept: Flux

- Flux: Net outflow of a quantity through surface, e.g., volume flux, mass flux, momentum flux, energy flux, etc.
- $\nabla \cdot (\text{Vector}, \text{Tensor})$

For example:

Mass flux ($\nabla \cdot \rho \mathbf{u}$)



Momentum flux ($\nabla \cdot \sigma$)

Energy flux ($\nabla \cdot q$)

ρu : mass flow rate per unit area

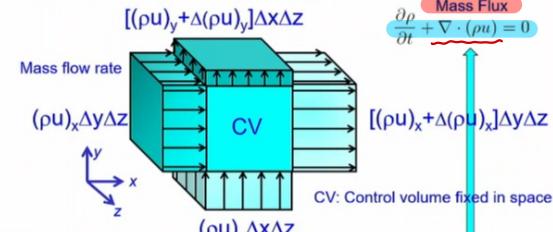
$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$$



www.eXorithm.com

The Continuity Equation - Cont'd

Compressible fluid (i.e., density, ρ , is a variable)

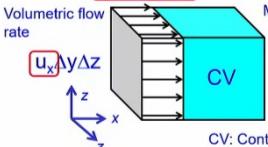


$$\frac{\text{Net mass flow rate}}{\text{Unit volume}} = \frac{\partial(\rho u)_x}{\partial x} + \frac{\partial(\rho u)_y}{\partial y} + \frac{\partial(\rho u)_z}{\partial z} = \nabla \cdot (\rho \mathbf{u})$$

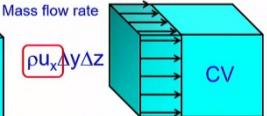
The Continuity Equation

Incompressible fluid
(i.e., density, ρ , is a constant)

$$(\nabla \cdot \mathbf{u}) = 0$$



Compressible fluid
(i.e., density, ρ , is a variable)



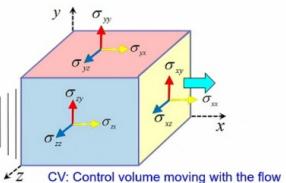
1. ρ (rho) :

• 密度: ρ 表示流体的密度, 即单位体积的质量。它通常以千克每立方米 (kg/m^3) 为单位。

2. \mathbf{u} (\mathbf{u}) :

• 速度向量: \mathbf{u} 表示流体的速度向量, 描述了流体在空间中的移动速度和方向。它的单位通常 是米每秒 (m/s)。

The Momentum Balance or Equation of Motion



$$\Sigma \mathbf{F} = m \mathbf{a}$$



http://www.masterster.com/toolbox

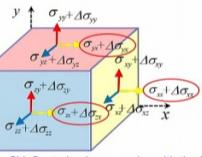
The Equation of Motion - Cont'd

In x-direction,

$$\sum \mathbf{f} = m \frac{D u_x}{D t} \quad \text{where } m = \rho \Delta x \Delta y \Delta z$$

x forces that act on the surfaces of the differential fluid element,

1. $\bar{\sigma}_{xx} \Delta y \Delta z$
2. $(\sigma_{xx} + \Delta \sigma_{xx}) \Delta y \Delta z$
3. $-\sigma_{yz} \Delta x \Delta z$
4. $(\sigma_{yz} + \Delta \sigma_{yz}) \Delta x \Delta z$
5. $-\sigma_{zx} \Delta x \Delta y$
6. $(\sigma_{zx} + \Delta \sigma_{zx}) \Delta x \Delta y$
7. $\rho g_x \Delta x \Delta y \Delta z$ Body forces



|||| The Equation of Motion - Cont'd

After adding the forces, dividing by the element's volume, and letting the volume go to zero, the force balance results in

$$\frac{\rho \Delta x \Delta y \Delta z}{\text{Volume}} \frac{D u_x}{D t} = \sum F = \Delta \sigma_{xx} \Delta y \Delta z + \Delta \sigma_{yz} \Delta x \Delta z + \Delta \sigma_{zx} \Delta x \Delta y + \rho g_x \Delta x \Delta y \Delta z$$

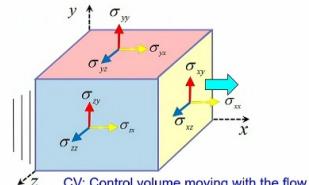
$$\frac{D u_x}{D t} = \frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{yx}}{\partial y} + \frac{\partial \sigma_{zx}}{\partial z} + \rho g_x$$

Inertia Surface forces Body forces

x-dir component of the momentum flux ($\nabla \bullet \sigma$)



|||| The Momentum Balance (Equation of Motion)

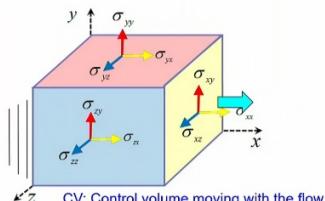


$$\frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{yx}}{\partial y} + \frac{\partial \sigma_{zx}}{\partial z} = \nabla \bullet (\sigma)$$



<http://www.masterster.com/toolbox>

|||| The Momentum Balance (Equation of Motion)



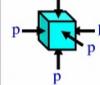
$$\frac{\rho D u}{D t} = \rho a = m a / V = \sum F / V = \nabla \bullet (\sigma) + \rho g$$



|||| Momentum Equation (Conservation of Momentum)

compress

$$\sigma = -p I + \tau$$



$$\rho \frac{D u}{D t} = \nabla \bullet (\sigma) + \rho g$$

$$\rho \frac{D u}{D t} = -\nabla p + \nabla \bullet (\tau) + \rho g$$

$\tau = \eta \dot{\gamma}$ Constitutive equation

Note my notion:

$$\eta = \eta(T, \dot{\gamma}, p) \text{ or } \eta(x, y, z)$$

$$\text{Non-Newtonian Fluids} \quad \rho \frac{D u}{D t} = -\nabla p + \nabla \bullet (\eta \dot{\gamma}) + \rho g$$

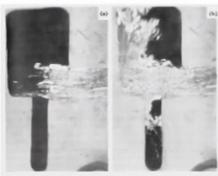
$$\mu = \text{A constant}$$

$$\text{Newtonian Fluids} \quad \rho \frac{D u}{D t} = -\nabla p + \mu \nabla^2 u + \rho g$$

$$\text{Navier-Stokes equations}$$



Die Casting vs. Injection Molding



Die Casting w/ Liquid Metals

$$\rho \frac{D u}{D t} = -\nabla p + \cancel{\nabla \cdot \tau} + \rho g$$

Pressure Body forces
Inertia Surface forces w/o pressure



Injection Molding w/ Plastic Melts

$$\rho \cancel{\frac{D u}{D t}} = -\nabla p + \nabla \cdot \tau + \cancel{\rho g}$$

Pressure Body forces
Inertia Surface forces w/o pressure

C: 268 ms

D: 794 ms

压铸 (Die Casting with Liquid Metals)

涉及液态金属的压铸过程，公式如下：

$$\rho \frac{D u}{D t} = -\nabla p + \cancel{\nabla \cdot \tau} + \rho g$$

- $\rho \frac{D u}{D t}$: 表示惯性力。

- $-\nabla p$: 表示压力梯度。

- $\cancel{\nabla \cdot \tau}$: 表面法向力被忽略，因为液态金属的流动主要受到压力和重力影响。

- ρg : 表示重力影响。

The Energy Balance or Equation of Energy

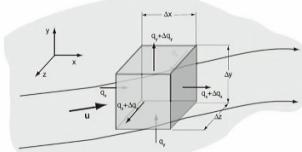


Figure 2.7: Heat flux across a differential fluid element during flow.

An energy balance around a moving fluid element can be written as,

$$\rho C_p \frac{DT}{Dt} = -\frac{\partial q_x}{\partial x} - \frac{\partial q_y}{\partial y} - \frac{\partial q_z}{\partial z} + \dot{Q} + \dot{Q}_{\text{viscous heating}}$$

↳ leaving

where the left hand term represents the transient and convective effects and the right hand the conduction terms, arbitrary heat source (\dot{Q}), and viscous dissipation ($\dot{Q}_{\text{viscous heating}}$).

Continuity Equation (Conservation of Mass)

$$\frac{\partial \rho}{\partial t} + \nabla \bullet (\rho \mathbf{u}) = 0$$

Momentum Equation (Conservation of Momentum)

$$\rho \frac{D\mathbf{u}}{Dt} = \nabla \bullet (\sigma) + \rho \mathbf{g}$$

Energy Equation (Conservation of Energy)

$$\rho C_p \frac{DT}{Dt} = -\nabla \bullet (\mathbf{q}) + \dot{Q}_{\text{viscous heating}} + \dot{Q}$$



The Equation of Energy – Cont'd

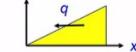
Derivation of heat conduction term. $-\nabla \bullet (\mathbf{q})$

Using Fourier's law for heat conduction

$$q_i = -k_i \frac{\partial T}{\partial x_i}$$

and assuming an isotropic material, $k_x = k_y = k_z = k$, we can write

$$\rho C_p \frac{DT}{Dt} = k \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right) + \dot{Q} + \dot{Q}_{\text{viscous heating}}$$



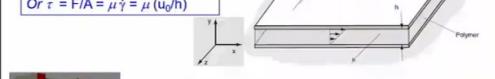
where k_i is heat conduction coefficient.

$$\rho C_p \frac{DT}{Dt} = -\nabla \bullet (\mathbf{q}) + \dot{Q} + \dot{Q}_{\text{viscous heating}}$$



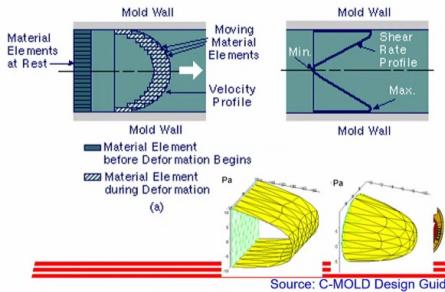
Simple Shear Flow and Examples

$$\begin{aligned} \text{Shear stress } (\tau, \sigma_{xy}) &= F/A \\ \text{Shear rate } (\dot{\gamma}, \dot{\gamma}_s) &= u_0/h \\ \text{Shear viscosity } (\mu, \eta) &= \tau / \dot{\gamma} \\ \text{Or } \tau &= F/A = \mu / h = \mu (u_0/h) \end{aligned}$$



Sources of images: blenderartists.org, baldor.com, substech.com, pinterest.com

Flow between Plates or inside a Tube



The Equation of Energy - Cont'd

$$\rightarrow \frac{F}{A} = \mu \frac{u_0}{h}$$

In the system, the rate of energy input is given by

$$F u_0 = \mu \frac{u_0}{h} A u_0$$

and the rate of energy input per unit volume is represented by

$$\frac{F u_0}{A h} = \mu \left(\frac{u_0}{h} \right) \left(\frac{u_0}{h} \right)$$

or

$$\dot{Q}_{\text{viscous heating}} = \mu \left(\frac{\partial u_x}{\partial y} \right) \left(\frac{\partial u_x}{\partial y} \right)$$



Continuity Equation (Conservation of Mass)

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$$

Table 2.1: Continuity Equation

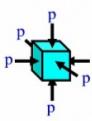
$$\text{Cartesian Coordinates } (x, y, z): \quad \nabla = \frac{\partial}{\partial x} \mathbf{e}_x + \frac{\partial}{\partial y} \mathbf{e}_y + \frac{\partial}{\partial z} \mathbf{e}_z$$

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} (\rho u_x) + \frac{\partial}{\partial y} (\rho u_y) + \frac{\partial}{\partial z} (\rho u_z) = 0$$

$$\text{Cylindrical Coordinates } (r, \theta, z): \quad \frac{\partial \rho}{\partial t} + \frac{1}{r} \frac{\partial}{\partial r} (\rho r u_r) + \frac{1}{r} \frac{\partial}{\partial \theta} (\rho u_\theta) + \frac{\partial}{\partial z} (\rho u_z) = 0$$

$$\text{Spherical Coordinates } (r, \theta, \phi): \quad \frac{\partial \rho}{\partial t} + \frac{1}{r^2} \frac{\partial}{\partial r} (\rho r^2 u_r) + \frac{1}{r \sin \theta} \frac{\partial}{\partial \theta} (\rho u_\theta \sin \theta) + \frac{1}{r \sin \theta} \frac{\partial}{\partial \phi} (\rho u_\phi) = 0$$

Momentum Equation (Conservation of Momentum)



$$\sigma = -p \mathbf{I} + \boldsymbol{\tau} \quad \sigma_{ij} = \begin{pmatrix} -p + \tau_{xx} & -\tau_{xy} & \tau_{xz} \\ -\tau_{yx} & -p + \tau_{yy} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & -p + \tau_{zz} \end{pmatrix}$$

$$\rho \frac{D\mathbf{u}}{Dt} = \nabla p + \nabla \cdot (\boldsymbol{\tau}) + \rho \mathbf{g}$$

$$\rho \frac{D\mathbf{u}}{Dt} = -\nabla p + \nabla \cdot (\boldsymbol{\tau}) + \rho \mathbf{g}$$

$\boldsymbol{\tau} = \eta \dot{\gamma}$ Constitutive equation

Note my notion:

$$\eta = \eta(x, y, z) \quad \text{Non-Newtonian Fluids} \quad \rho \frac{D\mathbf{u}}{Dt} = -\nabla p + \nabla \cdot (\eta \dot{\gamma}) + \rho \mathbf{g}$$

$$\mu = \text{A constant} \quad \text{Newtonian Fluids} \quad \rho \frac{D\mathbf{u}}{Dt} = -\nabla p + \mu \nabla^2 \mathbf{u} + \rho \mathbf{g}$$

Navier-Stokes equations



Momentum Equation (Conservation of Momentum)

$$\rho \frac{D\mathbf{u}}{Dt} = -\nabla p + \nabla \cdot (\boldsymbol{\tau}) + \rho \mathbf{g}$$

Table 2.2: Momentum Equation in terms of τ

$$\text{Cartesian Coordinates } (x, y, z):$$

$$\rho \left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} \right) = -\frac{\partial p}{\partial x} + \left[\frac{\partial}{\partial x} \tau_{xx} + \frac{\partial}{\partial y} \tau_{yx} + \frac{\partial}{\partial z} \tau_{zx} \right] + \rho g_x$$

$$\rho \left(\frac{\partial u_y}{\partial t} + u_x \frac{\partial u_y}{\partial x} + u_y \frac{\partial u_y}{\partial y} + u_z \frac{\partial u_y}{\partial z} \right) = -\frac{\partial p}{\partial y} + \left[\frac{\partial}{\partial x} \tau_{xy} + \frac{\partial}{\partial y} \tau_{yy} + \frac{\partial}{\partial z} \tau_{zy} \right] + \rho g_y$$

$$\rho \left(\frac{\partial u_z}{\partial t} + u_x \frac{\partial u_z}{\partial x} + u_y \frac{\partial u_z}{\partial y} + u_z \frac{\partial u_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \left[\frac{\partial}{\partial x} \tau_{xz} + \frac{\partial}{\partial y} \tau_{yz} + \frac{\partial}{\partial z} \tau_{zz} \right] + \rho g_z$$

For non-Newtonian fluids like polymer melts, ketchup, silly putty, and paints



Energy Equation (Conservation of Energy)

$$\rho C_p \frac{DT}{Dt} = -\nabla \cdot (\mathbf{q}) + \dot{Q}_{\text{viscous heating}} + \dot{Q}_{\text{heat loss}}$$



Table 2.5: Energy Equation for a Newtonian Fluid with Constant Properties

$$\text{Cartesian Coordinates } (x, y, z):$$

$$\rho C_p \left(\frac{\partial T}{\partial t} + u_x \frac{\partial T}{\partial x} + u_y \frac{\partial T}{\partial y} + u_z \frac{\partial T}{\partial z} \right) = k \left[\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right] + \mu \Phi_v + \dot{Q}$$

Table 2.6: Viscous Dissipation Function Φ_v for Incompressible Newtonian Fluids

$$\text{Cartesian Coordinates } (x, y, z):$$

$$\Phi_v = 2 \left[\left(\frac{\partial u_x}{\partial x} \right)^2 + \left(\frac{\partial u_y}{\partial y} \right)^2 + \left(\frac{\partial u_z}{\partial z} \right)^2 \right] + \left[\frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right]^2 + \left[\frac{\partial u_x}{\partial z} + \frac{\partial u_z}{\partial x} \right]^2 + \left[\frac{\partial u_y}{\partial z} + \frac{\partial u_z}{\partial y} \right]^2$$



Simplifying the Continuity Equation

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$$

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} (\rho u_x) + \frac{\partial}{\partial y} (\rho u_y) + \frac{\partial}{\partial z} (\rho u_z) = 0$$

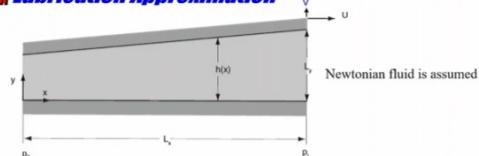
$$\text{Steady State} \rightarrow \frac{\partial u_x}{\partial t} = \frac{\partial u_y}{\partial t} = \frac{\partial u_z}{\partial t} = \frac{\partial T}{\partial t} = \text{etc.} = 0$$

$$\text{Constant Material Properties} \rightarrow \frac{\partial \rho}{\partial t} = 0 \quad \frac{\partial u_z}{\partial z} + \frac{\partial u_y}{\partial y} + \frac{\partial u_x}{\partial x} = \frac{\partial u_i}{\partial x_i} = 0 \quad \nabla \cdot \mathbf{u} = 0$$

$$\text{Fully Developed Flow} \rightarrow \frac{\partial u_x}{\partial z} = 0$$

$$\text{Dimensions of Geometry} \rightarrow \text{If } u_x = u_x(x, y), \text{ then } \frac{\partial u_x}{\partial z} = 0$$

Lubrication Approximation



Continuity Equation (Conservation of Mass)

$$\text{Cartesian Coordinates } (x, y, z):$$

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} (\rho u_x) + \frac{\partial}{\partial y} (\rho u_y) + \frac{\partial}{\partial z} (\rho u_z) = 0$$

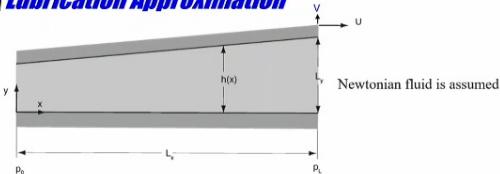
Steady state

Incompressible fluid

$$\frac{\partial u_x}{\partial z} + \frac{\partial u_y}{\partial y} + \frac{\partial u_x}{\partial x} = 0 \rightarrow \frac{\partial u_x}{\partial x} + \frac{\partial u_y}{\partial y} = 0$$



Lubrication Approximation



Momentum Equation (Conservation of Momentum)

$$\rho \left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} \right) = -\frac{\partial p}{\partial x} + \mu \left[\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} + \frac{\partial^2 u_x}{\partial z^2} \right] + \rho g_x$$

Steady state $\frac{\partial u_x}{\partial t} = 0$

$$\rho \left(\frac{\partial u_y}{\partial t} + u_x \frac{\partial u_y}{\partial x} + u_y \frac{\partial u_y}{\partial y} + u_z \frac{\partial u_y}{\partial z} \right) = -\frac{\partial p}{\partial y} + \mu \left[\frac{\partial^2 u_y}{\partial x^2} + \frac{\partial^2 u_y}{\partial y^2} + \frac{\partial^2 u_y}{\partial z^2} \right] + \rho g_y$$

Steady state $\frac{\partial u_y}{\partial t} = 0$

Lubrication Approx. – Order of Magnitude

Continuity equation: $\frac{\partial u_x}{\partial x} + \frac{\partial u_y}{\partial y} = 0$ $\because L_x \gg L_y \Rightarrow U \gg V$

Momentum equations:

$$\rho \left(u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} \right) = -\frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} \right) \quad (1)$$

$$\rho U \frac{U}{L_x} \approx \rho V \frac{U}{L_y} \quad (2)$$

$$\frac{U}{L_x} \ll \frac{U}{L_y} \text{ if } Re \frac{L_y}{L_x} \ll 1 \text{ then (1)} \gg (1)' \quad (3)$$

$$\rho \left(u_x \frac{\partial u_y}{\partial x} + u_y \frac{\partial u_y}{\partial y} \right) = -\frac{\partial p}{\partial y} + \mu \left(\frac{\partial^2 u_y}{\partial x^2} + \frac{\partial^2 u_y}{\partial y^2} \right) \quad (4)$$

$$\rho U \frac{V}{L_x} \approx \rho V \frac{V}{L_y} \quad (5)$$

$$\frac{V}{L_x} \ll \frac{V}{L_y} \text{ if } Re \frac{L_x}{L_y} \ll 1 \text{ then (5)} \gg (1)' \quad (5')$$

Lubrication Approx. – Order of Magnitude

Continuity equation: $\frac{\partial u_x}{\partial x} + \frac{\partial u_y}{\partial y} = 0$ $\Rightarrow U \gg V$

Momentum equations:

$$\rho \left(u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} \right) = -\frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} \right) \quad (1) \quad (2) \quad (3) \quad (4) \quad (5) \quad (5') \quad \because U \gg V \Rightarrow (5) \gg (5')$$

$$\rho \left(u_x \frac{\partial u_y}{\partial x} + u_y \frac{\partial u_y}{\partial y} \right) = -\frac{\partial p}{\partial y} + \mu \left(\frac{\partial^2 u_y}{\partial x^2} + \frac{\partial^2 u_y}{\partial y^2} \right) \quad (1)' \quad (2)' \quad (3)' \quad (4)' \quad (5)' \quad (5'') \quad \frac{\Delta p}{L_y} \ll \frac{\Delta p}{L_x} \quad (3)$$

$$\rho \left(u_x \frac{\partial u_y}{\partial x} + u_y \frac{\partial u_y}{\partial y} \right) = -\frac{\partial p}{\partial y} + \mu \left(\frac{\partial^2 u_y}{\partial x^2} + \frac{\partial^2 u_y}{\partial y^2} \right) \quad (1)' \quad (2)' \quad (3)' \quad (4)' \quad (5)' \quad (5'') \quad \frac{\Delta p}{L_y} \ll \frac{\partial p}{\partial x} \quad (3)$$

Pressure Driven Flow of a Newtonian Fluid Through a Slit

* come out of the plane

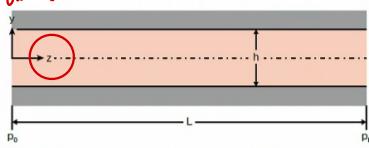


Figure 9.18 Schematic diagram of pressure flow through a slit

Assume incompressible Newtonian fluid; unidirectional and steady fully developed flow (a flow where the entrance effects are ignored).



Pressure Driven Flow Through a Slit – Cont'd

Continuity equation: $u_x = 0 \quad u_z = 0$

$$\frac{\partial u_z}{\partial z} + \frac{\partial u_y}{\partial y} + \frac{\partial u_x}{\partial x} = 0 \quad \frac{du_z}{dz} = 0$$

Momentum equations:

Cartesian Coordinates (x, y, z): $u_x = 0$

$$\rho \left(\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} \right) = -\frac{\partial p}{\partial x} + \mu \left[\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} + \frac{\partial^2 u_x}{\partial z^2} \right] + \rho g_x \quad \frac{\partial p}{\partial x} = 0$$

$u_x = 0$ $u_y = 0$ $u_z = 0$ $u_x \text{ independent of } x$

$$\rho \left(\frac{\partial u_y}{\partial t} + u_x \frac{\partial u_y}{\partial x} + u_y \frac{\partial u_y}{\partial y} + u_z \frac{\partial u_y}{\partial z} \right) = -\frac{\partial p}{\partial y} + \mu \left[\frac{\partial^2 u_y}{\partial x^2} + \frac{\partial^2 u_y}{\partial y^2} + \frac{\partial^2 u_y}{\partial z^2} \right] + \rho g_y \quad \frac{\partial p}{\partial y} = 0$$

$u_x = 0$ $u_y = 0$ $u_z = 0$ $u_y \text{ independent of } x$

$$\rho \left(\frac{\partial u_z}{\partial t} + u_x \frac{\partial u_z}{\partial x} + u_y \frac{\partial u_z}{\partial y} + u_z \frac{\partial u_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{\partial^2 u_z}{\partial x^2} + \frac{\partial^2 u_z}{\partial y^2} + \frac{\partial^2 u_z}{\partial z^2} \right] + \rho g_z \quad \frac{\partial p}{\partial z} = 0$$

$u_x = 0$ $u_y = 0$ $u_z = 0$ $u_z \text{ independent of } x$

Steady state $u_x \text{ independent of } x$ $u_y \text{ independent of } x$ $u_z \text{ independent of } x$

Fully developed flow $u_x \text{ independent of } z$ $u_y \text{ independent of } z$ $u_z \text{ independent of } z$

Pressure Driven Flow Through a Slit – Cont'd

Continuity equation:

$$-\frac{du_z}{dz} = 0$$

$$F(z) \quad G(y) \Rightarrow F(z) = G(y) = \text{a constant}$$

Momentum equations:

$$-\frac{\partial p}{\partial x} = 0 \quad -\frac{\partial p}{\partial y} = 0 \quad -\frac{\partial p}{\partial z} + \mu \frac{\partial^2 u_z}{\partial y^2} = 0$$

The total pressure is a function of z alone. Additionally, since u does not vary with z , the pressure gradient, $\partial p / \partial z$, must be a constant. Therefore,

$$\frac{dp}{dz} = \frac{\Delta p}{L}$$

The momentum equation can now be written as,

$$-\frac{1}{\mu} \frac{\Delta p}{L} = \frac{\partial^2 u_z}{\partial y^2}$$



$$\frac{\partial^2 u_z}{\partial y^2} = -\frac{\Delta p}{\mu L}$$

$$\frac{d^2 u_z}{dy^2} = -\frac{\Delta p}{\mu L}$$

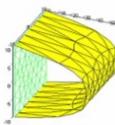
$$\int \frac{d^2 u_z}{dy^2} dy = \int -\frac{\Delta p}{\mu L} dy$$

$$\frac{du_z}{dy} = -\frac{\Delta p}{\mu L} y + C_1$$

Since $\frac{du_z}{dy} = 0$ at $y = 0$ (symm. condition), hence $C_1 = 0$

$$\int \frac{du_z}{dy} dy = \int -\frac{\Delta p}{\mu L} y dy$$

$$u_z(y) = -\frac{\Delta p}{2\mu L} y^2 + C_2$$



Pressure Driven Flow Through a Slit – Cont'd

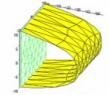
Since $u_z(y) = 0$ at $y = \pm h/2$ (no slip boundary condition), hence $C_2 = 0$

Boundary conditions, two no-slip conditions given by, $u_z(y) = -\frac{\Delta p}{2\mu L} y^2 + C_2$
 $u_z(\pm h/2) = 0$

Integrating twice and evaluating the two integration constants with the boundary conditions gives,

$$u_z(y) = \frac{h^2}{8\mu L} \frac{dp}{dz} \left[1 - \left(\frac{2y}{h} \right)^2 \right]$$

$$= \frac{h^2}{8\mu L} \frac{\Delta p}{L} \left[1 - \left(\frac{2y}{h} \right)^2 \right]$$



Note that the same profile will result if one of the non-slip boundary conditions is replaced by a symmetry condition at $y = 0$, namely $du_z/dy = 0$.

Pressure Driven Flow Through a Slit – Cont'd

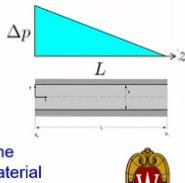
The mean velocity in the channel is obtained integrating the above equation,

$$\bar{u}_z = \frac{2}{h} \int_0^{h/2} u_z(y) dy = \frac{h^2}{12\mu L} \frac{dp}{dz} \quad \frac{dp}{dz} = -\frac{\Delta p}{L}$$

and the volumetric flow rate,

$$Q = hW\bar{u}_z = \frac{Wh^3}{12\mu L} \Delta p$$

where W is the width of the channel.



Can you see the relationship among the required pressure, Δp , flow rate, Q , material viscosity, μ , and part dimensions, W , h , L ?