

# Approximate PI via Dynamic Compensation.

$$K \frac{s+z}{s} \rightarrow K \frac{s+z}{s+p} \quad (p < z) \quad \leftarrow \text{lag compensation.}$$

Lag compensation *does not* give perfect tracking (indeed, it does not change system type), but we can get as good a tracking as we want by playing with  $K, z, p$ . On the other hand, unlike PI, lag compensation gives a stable controller.

## Ex1 Pole Placement Via RL $s^2 + 2\zeta\omega_n s + \omega_n^2$

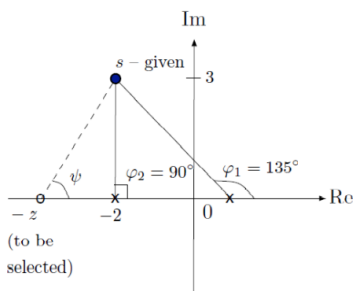
Let  $G_p(s) = \frac{1}{s-1}$ ,  $G_c(s) = K \frac{s+z}{s+p}$

**Problem:** given  $p = 2$ , find  $K$  and  $z$  to place poles at  $-2 \pm 3j$ .

Desired characteristic polynomial:

$$(s+2)^2 + 9 = s^2 + 4s + 13, \quad \text{damping ratio } \zeta = \frac{2}{\sqrt{13}} \approx 0.555$$

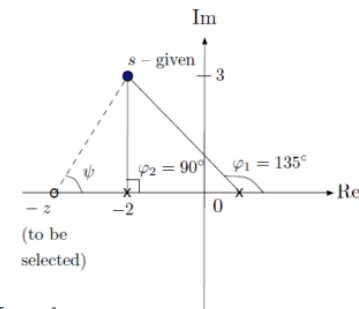
$$\begin{aligned} 2\zeta\omega_n &= 4 \\ \zeta &= \frac{2}{\sqrt{13}} \end{aligned}$$



Must have

$$\underbrace{\psi}_{\text{angle from } s \text{ to zero}} - \sum_i \underbrace{\varphi_i}_{\text{angles from } s \text{ to poles}} = 180^\circ$$

$$\text{So, we want } \psi = 180^\circ + \sum_i \varphi_i$$



We have

$$\varphi_1 = 135^\circ,$$

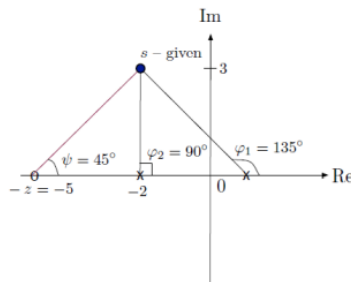
$$\varphi_2 = 90^\circ$$

$$\text{We want } \psi = 180^\circ + \sum_i \varphi_i$$

Must have

$$\begin{aligned} \psi &= 180^\circ + 135^\circ + 90^\circ \\ &= 405^\circ \\ &= 45^\circ \text{ mod } 360^\circ \end{aligned}$$

Thus, we should have  $z = -5$



► resulting characteristic polynomial:

$$\begin{aligned} (s-1)(s+2) + K(s+5) \\ s^2 + (K+1)s + 5K - 2 \end{aligned}$$

► compare against desired characteristic polynomial:

$$s^2 + 4s + 13 \quad \Rightarrow \quad K+1 = 4, \quad 5K-2 = 13$$

so we need  $K = 3$

► compute s.s. tracking error:  $\left| \frac{1}{1 - \frac{Kz}{p}} \right| = \frac{1}{6.5} \approx 15\%$

## Summary: PI & PD

### PD control:

- ▶ provides stability, allows to shape transient response specs
- ▶ replace noncausal D-controller  $Ks$  with a causal, stable lead controller  $K \frac{s+z}{s+p}$ , where  $p > z$
- ▶ this introduces a zero in LHP (at  $-z$ ), pulls the root locus into LHP
- ▶ shape of RL differs depending on how large  $p$  is

### PI control:

- ▶ provides stability and perfect steady-state tracking of constant references
- ▶ replace unstable I-controller  $K/s$  with a stable lag controller  $K \frac{s+z}{s+p}$ , where  $p < z$
- ▶ this does not change the shape of RL compared to PI

## Frequency Response Design Method.

### ① Sinusoidal:

set  $s=j\omega$ .

$$\sin(\omega t) \rightarrow \boxed{G(s)} \rightarrow M \sin(\omega t + \phi).$$

#### Derivation:

1.  $u(t) = e^{st} \mapsto y(t) = G(s)e^{st}$
2. Euler's formula:  $\sin(\omega t) = \frac{e^{j\omega t} - e^{-j\omega t}}{2j}$
3. By linearity,

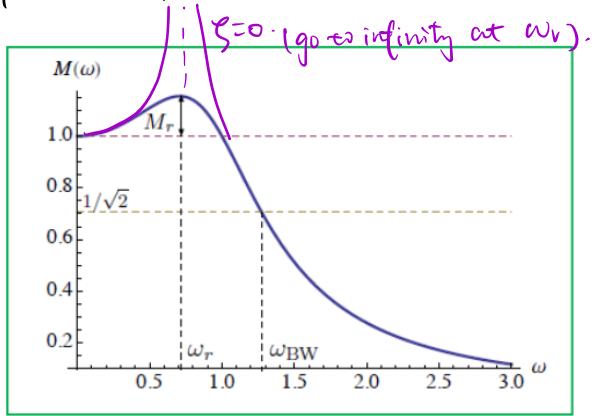
$$\begin{aligned} \sin(\omega t) &\mapsto \frac{G(j\omega)e^{j\omega t} - G(-j\omega)e^{-j\omega t}}{2j} \quad G(j\omega) = M(\omega)e^{j\phi(\omega)} \\ &= \frac{M(\omega)e^{j(\omega t + \phi(\omega))} - M(\omega)e^{-j(\omega t + \phi(\omega))}}{2j} \\ &= M(\omega) \sin(\omega t + \phi(\omega)) \end{aligned}$$

### ② 2nd-order system:

$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$\Rightarrow M(\omega) = |G(j\omega)| = \frac{1}{\sqrt{\left[1 - \left(\frac{\omega}{\omega_n}\right)^2\right]^2 + 4\zeta^2\left(\frac{\omega}{\omega_n}\right)^2}}$$

## Frequency Response Magnitude Plot.



$\omega_r$  - resonant frequency  
 $M_r$  - resonant peak  
 $\omega_{BW}$  - bandwidth

$M_r \downarrow \Rightarrow$  Better Damping

$\omega_{BW} \uparrow \Rightarrow \omega_n \uparrow \Rightarrow tr \downarrow$

We can get the following formulas using calculus:

$$\begin{cases} \omega_r = \omega_n \sqrt{1 - 2\zeta^2} \\ M_r = \frac{1}{2\zeta\sqrt{1 - \zeta^2}} - 1 \end{cases} \quad \left( \text{valid for } \zeta < \frac{1}{\sqrt{2}}; \text{ for } \zeta \geq \frac{1}{\sqrt{2}}, \omega_r = 0 \right)$$

$$\omega_{BW} = \omega_n \sqrt{\underbrace{(1 - 2\zeta^2) + \sqrt{(1 - 2\zeta^2)^2 + 1}}_{=1 \text{ for } \zeta=1/\sqrt{2}}}$$

— so, if we know  $\omega_r, M_r, \omega_{BW}$ , we can determine  $\omega_n, \zeta$  and hence the time-domain specs ( $t_r, M_p, t_s$ )

$\zeta=0 \Rightarrow$  no damping.

## Scale Convention for Bode Plots.

	magnitude	phase
horizontal scale	log	log
vertical scale	log	linear

Bode Form. const. term in each factor = 1, i.e. lump all DC gains into one number in the front  
 e.g.

$$\begin{aligned} KG(s) &= K \frac{s+3}{s(s^2+2s+4)} \\ \text{rewrite as } &\frac{3K \left(\frac{s}{3} + 1\right)}{4s \left(\left(\frac{s}{2}\right)^2 + \frac{s}{2} + 1\right)} \Big|_{s=j\omega} \\ &= \underbrace{\frac{3K}{4}}_{=K_0} \frac{\frac{j\omega}{3} + 1}{j\omega \left(\left(\frac{j\omega}{2}\right)^2 + \frac{j\omega}{2} + 1\right)} \end{aligned}$$