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Applied Fluid Mechanics Homework 10

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Problem 12.54

12.54 Air flows from a large tank at $p = 650$ kPa absolute, $T = 550^\circ\text{C}$ through a converging nozzle, with a throat area of 600 mm^2 , and discharges to the atmosphere. Determine the mass rate of flow for isentropic flow through the nozzle.

Solution:

$$\frac{p_t}{p_0} = \frac{101}{650} = 0.155 < 0.528$$

Therefore, the air is choked flow, for which $M_t = 1.0$.

$$\begin{aligned} \frac{T_0}{T_t} &= \left[1 + \frac{k-1}{2} M_t^2 \right] \\ T_t &= \frac{T_0}{\left[1 + \frac{k-1}{2} M_t^2 \right]} = \frac{(550 + 273.15)\text{ K}}{1 + \frac{1.4-1}{2} \times 1^2} \\ &= 685.96\text{ K} \end{aligned}$$

$$\begin{aligned} \frac{p_0}{p_t} &= \left[1 + \frac{k-1}{2} M_t^2 \right]^{\frac{k}{k-1}} \\ p_t &= \frac{p_0}{\left[1 + \frac{k-1}{2} M_t^2 \right]^{\frac{k}{k-1}}} \\ &= \frac{650\text{ kPa}}{\left(1 + \frac{1.4-1}{2} \times 1^2 \right)^{\frac{1.4}{1.4-1}}} \\ &= 343.38\text{ kPa} \end{aligned}$$

$$\begin{aligned} \rho_t &= \frac{p_t}{RT_t} \\ &= \frac{343.38\text{ kPa}}{(287\text{ N} \cdot \text{m/kg} \cdot \text{K}) \times (685.96\text{ K})} \\ &= 1.74\text{ kg/m}^3 \end{aligned}$$

$$\begin{aligned} c_t &= \sqrt{kRT_t} \\ &= \sqrt{1.4 \times (287\text{ N} \cdot \text{m/kg} \cdot \text{K}) \times (685.96\text{ K})} \\ &= 524.99\text{ m/s} \end{aligned}$$

$$\begin{aligned} V_t &= M_t c_t = 1 \times 524.99\text{ m/s} \\ &= 524.99\text{ m/s} \end{aligned}$$

$$\begin{aligned} \dot{m} &= \rho_t V_t A_t = (1.74\text{ kg/m}^3) \\ &\quad \times (524.99\text{ m/s}) \\ &\quad \times (600 \times 10^{-6}\text{ m}^2) \\ &= 0.5494\text{ kg/s} \end{aligned}$$

Problem 12.58

12.58 Air flows isentropically through a converging nozzle attached to a large tank, where the absolute pressure is 171 kPa and the temperature is 27°C . At the inlet section, the Mach number is 0.2 . The nozzle discharges to the atmosphere; the discharge area is 0.015 m^2 . Determine the magnitude and direction of the force that must be applied to hold the nozzle in place.

Solution:

$$\frac{p_t}{p_0} = \frac{101}{171} = 0.6257 > 0.528$$

Therefore, the air is not choked flow.

$$\begin{aligned} \frac{p_0}{p_t} &= \left[1 + \frac{k-1}{2} M_t^2 \right]^{\frac{k}{k-1}} \\ \frac{171\text{ kPa}}{101\text{ kPa}} &= \left[1 + \frac{1.4-1}{2} M_t^2 \right]^{\frac{1.4}{1.4-1}} \\ &\Rightarrow M_t = 0.9010 \\ \frac{T_0}{T_t} &= \left[1 + \frac{k-1}{2} M_t^2 \right] \end{aligned}$$

$$T_t = \frac{T_0}{\left[1 + \frac{k-1}{2} M_t^2\right]}$$

$$= \frac{(27 + 273.15) \text{ K}}{1 + \frac{1.4-1}{2} \times 0.9010^2}$$

$$= 258.2276 \text{ K}$$

$$T_1 = \frac{T_0}{\left[1 + \frac{k-1}{2} M_1^2\right]}$$

$$= \frac{(27 + 273.15) \text{ K}}{\left(1 + \frac{1.4-1}{2} \times 0.2^2\right)}$$

$$= 297.7679 \text{ K}$$

$$c_t = \sqrt{kRT_t}$$

$$c_1 = \sqrt{kRT_1}$$

$$= \sqrt{1.4 \times (287 \text{ N} \cdot \text{m/kg} \cdot \text{K}) \times (258.2276 \text{ K})} = \sqrt{1.4 \times (287 \text{ N} \cdot \text{m/kg} \cdot \text{K}) \times (297.7679 \text{ K})}$$

$$= 322.1116 \text{ m/s} \quad \quad \quad = 345.8947 \text{ m/s}$$

$$V_t = M_t c_t = 0.9010 \times 322.1116 \text{ m/s}$$

$$= 290.2103 \text{ m/s}$$

$$V_1 = M_1 c_1 = 0.2 \times 345.8947 \text{ m/s}$$

$$= 69.1789 \text{ m/s}$$

$$\rho_t = \frac{p_t}{RT_t}$$

$$= \frac{101 \text{ kPa}}{(287 \text{ N} \cdot \text{m/kg} \cdot \text{K}) \times (258.2276 \text{ K})}$$

$$= 1.3628 \text{ kg/m}^3$$

$$\rho_1 = \frac{p_1}{RT_1}$$

$$= \frac{166.2969 \text{ kPa}}{(287 \text{ N} \cdot \text{m/kg} \cdot \text{K}) \times (297.7679 \text{ K})}$$

$$= 1.9459 \text{ kg/m}^3$$

$$\dot{m} = \rho_t V_t A_t = (1.3628 \text{ kg/m}^3)$$

$$\quad \times (290.2103 \text{ m/s})$$

$$\quad \times (0.015 \text{ m}^2)$$

$$= 5.9325 \text{ kg/s}$$

$$A_1 = \frac{\dot{m}}{\rho_1 V_1}$$

$$= \frac{(5.9325 \text{ kg/s})}{(1.9459 \text{ kg/m}^3) \times (69.1789 \text{ m/s})}$$

$$= 0.0441 \text{ m}^2$$

$$\frac{p_0}{p_1} = \left[1 + \frac{k-1}{2} M_1^2\right]^{\frac{k}{k-1}}$$

$$p_1 = \frac{p_0}{\left[1 + \frac{k-1}{2} M_1^2\right]^{\frac{k}{k-1}}}$$

$$= \frac{171 \text{ kPa}}{\left(1 + \frac{1.4-1}{2} \times 0.2^2\right)^{\frac{1.4}{1.4-1}}}$$

$$= 166.2969 \text{ kPa}$$

$$\frac{T_0}{T_1} = \left[1 + \frac{k-1}{2} M_1^2\right]$$

$$\sum F_x = 0 = p_{1,g} A_1 - p_{2,g} A_2 - R_x$$

$$= -V_1(\dot{m}) + V_2(\dot{m})$$

$$R_x = p_{1,g} A_1 - p_{2,g} A_2 + \dot{m}(V_1 - V_2)$$

$$= [(166.2969 \text{ kPa})$$

$$\quad - (101.3 \text{ kPa})]$$

$$\quad \times (0.0441 \text{ m}^2)$$

$$- [(101.3 \text{ kPa})$$

$$\quad - (101.3 \text{ kPa})]$$

$$\quad \times (0.015 \text{ m}^2)$$

$$+ (5.9325 \text{ kg/s})$$

$$\quad \times [(69.1789 \text{ m/s})$$

$$\quad - (290.2103 \text{ m/s})]$$

$$= 1553 \text{ N}$$

Problem 12.59

12.59 Air enters a converging-diverging nozzle at 2 MPa absolute and 313 K. At the exit of the nozzle, the pressure is 200 kPa absolute. Assume adiabatic, frictionless flow through the nozzle. The throat area is 20 cm². What is the area at the nozzle exit? What is the mass flow rate of the air?

Solution:

$$\begin{aligned}\frac{p_0}{p_a} &= \left[1 + \frac{k-1}{2} M_a^2\right]^{\frac{k}{k-1}} \\ \frac{2 \text{ MPa}}{200 \text{ kPa}} &= \left[1 + \frac{1.4-1}{2} M_a^2\right]^{\frac{1.4}{1.4-1}} \\ \Rightarrow M_a &= 2.1572 \\ \frac{A}{A^*} &= \frac{1}{M_a} \left[\frac{1 + \frac{k-1}{2} M_a^2}{\frac{k+1}{2}} \right]^{\frac{k+1}{2(k-1)}} \\ A_e &= A_t \frac{1}{M_a} \left[\frac{1 + \frac{k-1}{2} M_a^2}{\frac{k+1}{2}} \right]^{\frac{k+1}{2(k-1)}} \\ &= (20 \text{ cm}^2) \times \frac{1}{2.1572} \\ &\quad \times \left[\frac{1 + \frac{1.4-1}{2} \times 2.1572^2}{\frac{1.4+1}{2}} \right]^{\frac{1.4+1}{2 \times (1.4-1)}} \\ &= 38.6136 \text{ cm}^2 \\ \dot{m} &= A_e p_0 \sqrt{\frac{k}{RT_0} \left(\frac{2}{k+1} \right)^{\frac{k+1}{2(k-1)}}} \\ &= (38.6136 \text{ cm}^2) \times (2 \text{ MPa}) \\ &\quad \times \sqrt{\frac{1.4}{(287 \text{ N} \cdot \text{m/kg} \cdot \text{K}) \times (313 \text{ K})}} \\ &\quad \times \left(\frac{2}{1.4+1} \right)^{\frac{1.4+1}{2 \times (1.4-1)}} = 17.6432 \text{ kg/s}\end{aligned}$$

Problem 12.64

12.64 Air discharges through a convergent-divergent nozzle which is attached to a large reservoir. At a point in the nozzle a normal shock wave is detected across which the absolute pressure jumps from 69 to 207 kPa. Calculate the pressures in the throat of the nozzle and in the reservoir.

Solution:

$$\begin{aligned}\frac{p_2}{p_1} &= 1 + \frac{2k}{k+1} (M_1^2 - 1) \\ \frac{207 \text{ kPa}}{69 \text{ kPa}} &= 1 + \frac{2 \times 1.4}{1.4+1} (M_1^2 - 1) \\ \Rightarrow M_1 &= 1.6475 \\ \frac{p_0}{p_1} &= \left[1 + \frac{k-1}{2} M_1^2\right]^{\frac{k}{k-1}} \\ p_0 &= p_1 \left[1 + \frac{k-1}{2} M_1^2\right]^{\frac{k}{k-1}} \\ &= (69 \text{ kPa}) \\ &\quad \times \left(1 + \frac{1.4-1}{2} \times 1.6475^2\right)^{\frac{1.4}{1.4-1}} \\ &= 314.7669 \text{ kPa} \\ \frac{p_t}{p_0} &= 0.528 \\ p_t &= 0.528 p_0 = 0.528 \times (314.7669 \text{ kPa}) \\ &= 166.1969 \text{ kPa}\end{aligned}$$



— Christopher King —