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Mechanical Design II Homework 13

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2018141521058

Mechanical Design 2

Class Section 01

12/19/2021

Problem 1

Given: The teeth on a pair of mating gears have a 6 diametral pitch, 20° pressure angle AGMA involute form. The pinion has 19 teeth, and the gear has 37 teeth. Find the following parameters:

- a. Gear ratio
- b. Circular pitch
- c. Base pitch
- d. Pitch diameter of pinion and gear
- e. Center distance
- f. Addendum
- g. Dedendum
- h. Whole tooth depth
- i. Clearance
- j. Outside diameter of pinion and gear
- k. Base diameter of pinion and gear
- l. Contact ratio

Solution:

$$P = 6 \text{ teeth/inch}$$

$$\phi = 20^\circ$$

$$N_p = 19 \text{ teeth}$$

$$N_g = 37 \text{ teeth}$$

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- a. Gear ratio: $\frac{N_g}{N_p} = \frac{37}{19}$
- b. Circular pitch: $p = \frac{\pi}{P} = \frac{\pi}{6} \text{ in}$
- c. Base pitch: $p_b = p \cos \phi = \frac{\pi}{6} \times \cos 20^\circ = 0.4920$
- d. Pitch diameter of pinion and gear: $d_p = \frac{N_p}{P} = \frac{19}{6}$, $d_g = \frac{N_g}{P} = \frac{37}{6}$
- e. Center distance: $\frac{d_p + d_g}{2} = \frac{14}{3}$
- f. Addendum: $a = \frac{1}{P} = \frac{1}{6}$
- g. Dedendum: $b = \frac{1.25}{P} = \frac{5}{24}$
- h. Whole tooth depth: $h_t = a_b = \frac{3}{8}$
- i. Clearance: $c = b - a = \frac{1}{24}$
- j. Outside diameter of pinion and gear: $d_{op} = d_p + 2a = \frac{7}{2}$, $d_{og} = d_g + 2a = \frac{13}{2}$
- k. Base diameter of pinion and gear: $d_{bp} = d_p \cos \phi = \frac{19}{6} \times \cos 20^\circ = 2.9757$, $d_{bg} = d_g \cos \phi = \frac{37}{6} \times \cos 20^\circ = 5.7948$

- l. Contact ratio:

$$m_c = \frac{\sqrt{r_{ap}^2 - r_{bp}^2} + \sqrt{r_{ag}^2 - r_{bg}^2} - C \sin \phi}{p_b}$$

$$= \frac{\sqrt{\left(\frac{7}{4}\right)^2 - \left(\frac{2.9757}{2}\right)^2} + \sqrt{\left(\frac{13}{4}\right)^2 - \left(\frac{5.7948}{2}\right)^2} - \frac{14}{3} \sin 20^\circ}{0.4920}$$

$$= 1.6209$$

Problem 2

For a pair of spur gears with gear ratio of 4:1, specify the minimum number of teeth allowed on the pinion to avoid the problem of interference assuming full-depth tooth depth using

- a 20° pressure angle, and
- a 25° pressure angle.

Solution:

a.

Using Eq. 13-11 with $k = 1$, $\phi = 20^\circ$, and $m = 4$,

$$N_p = \frac{2k}{(1 + 2m) \sin^2 \phi} \left(m + \sqrt{m^2 + (1 + 2m) \sin^2 \phi} \right) = 15.44 \text{ teeth}$$

Rounding up,

$$N_p = 16 \text{ teeth}$$

b.

Using Eq. 13-11 with $k = 1$, $\phi = 25^\circ$, and $m = 4$,

$$N_p = \frac{2k}{(1 + 2m) \sin^2 \phi} \left(m + \sqrt{m^2 + (1 + 2m) \sin^2 \phi} \right) = 10.20 \text{ teeth}$$

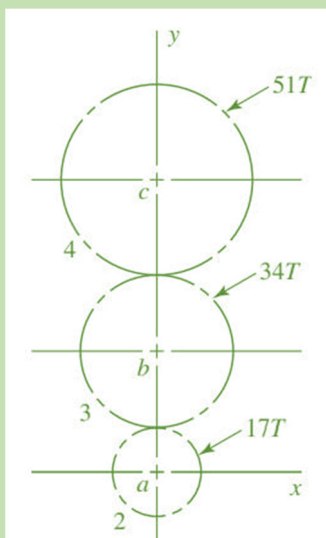
Rounding up,

$$N_p = 11 \text{ teeth}$$

Problem 3

Shaft a in the figure has a power input of 75 kW at a speed of 1000 rev/min in the counterclockwise direction. The gears have a module of 5 mm and a 20° pressure angle. Gear 3 is an idler.

- Find the force F3b that gear 3 exerts against shaft b.
- Find the torque T4c that gear 4 exerts on shaft c.



Solution:

a.

$$\omega = \frac{2\pi n}{60}$$

$$H = T\omega = \frac{2\pi Tn}{60}$$

$$T_a = \frac{60H}{2\pi n} = 398 \text{ N} \cdot \text{m}$$

$$r_2 = \frac{mN_2}{2} = 42.5 \text{ mm}$$

$$F_{32}^t = \frac{T_a}{r_2} = 9.36 \text{ kN}$$

$$F_{3b} = -F_{b3} = 2 \times 9.36 = 18.73 \text{ kN}$$

which is in the positive x -direction.

b.

$$r_4 = \frac{mN_4}{2} = 127.5 \text{ mm}$$

$$T_{c4} = F_{32}^t r_4 = 1193 \text{ N} \cdot \text{m ccw}$$

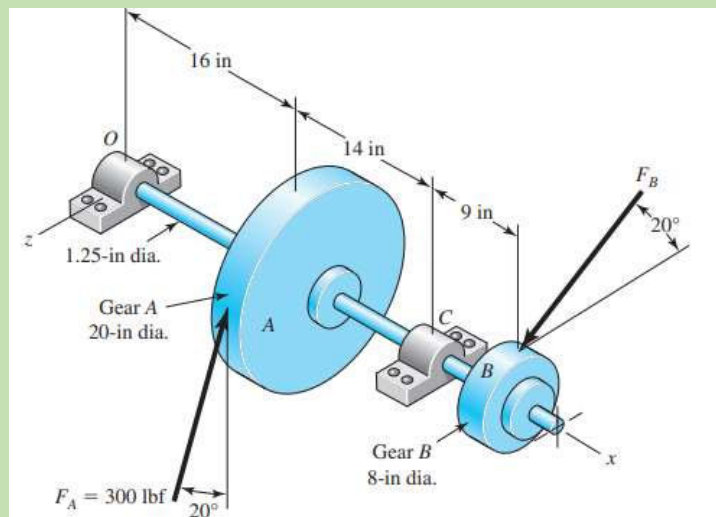
$$T_{4c} = 1193 \text{ N} \cdot \text{m cw}$$

Problem 4

For the countershaft shown below, Gear A receives power from another gear with the transmitted force F_A applied at the 20° pressure angle as shown. The power is transmitted through the shaft and delivered through gear B through a transmitted force F_B at the pressure angle shown.

Assume the gear ratio from gear B to its mating gear is 2 to 1.

- Determine the minimum number of teeth that can be used on gear B without an interference problem in the teeth.
- Using the number of teeth from part (a), what diametral pitch is required to also achieve the given 8-in pitch diameter?
- Suppose the 20° pressure angle gears are exchanged for gears with 25° pressure angle, while maintaining the same pitch diameters and diametral pitch. Determine the new forces F_A and F_B if the same power is to be transmitted.



Solution:

a.

Using Eq. 13-11 with $k = 1$, $\phi = 20^\circ$, and $m = 2$,

$$N_p = \frac{2k}{(1 + 2m) \sin^2 \phi} \left(m + \sqrt{m^2 + (1 + 2m) \sin^2 \phi} \right) = 14.16 \text{ teeth}$$

Rounding up,

$$N_p = 15 \text{ teeth}$$

b.

$$P = \frac{N}{d} = \frac{15}{8} = 1.875 \text{ teeth/in}$$

c.

With $\phi = 20^\circ$

$$W_{tA} = F_A \cos 20^\circ = 281.9 \text{ lbf}$$

With $\phi = 25^\circ$

$$F'_A = \frac{W_{tA}}{\cos 25^\circ} = 311.0 \text{ lbf}$$

And

$$W_{tA} \left(\frac{d_A}{2} \right) = W_{tB} \left(\frac{d_B}{2} \right)$$

$$W_{tB} = 704.75 \text{ lbf}$$

$$F'_B = \frac{W_{tB}}{\cos 25^\circ} = 777.6 \text{ lbf}$$



— Christopher King —