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Safe delay-adaptive control of strict-feedback nonlinear systems with application in vehicle platooning*



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ABSTRACT

This paper presents a safe delay-adaptive control for a strict-feedback nonlinear ODE with a delayed actuator, whose dynamic is also a strict-feedback nonlinear ODE and the delay length is unknown. By formulating the delay as a transport PDE, the plant becomes a sandwich configuration consisting of nonlinear ODE-transport PDE-nonlinear ODE, where the transport speed in the PDE is unknown. We propose a predictor-based nonovershooting backstepping transformation to build the nominal safe delay-compensated control, guaranteeing that the output of the distal ODE safely tracks the target trajectory from one side without undershooting. To address the uncertainty in the delay, we incorporate recent delay-adaptive and safe adaptive technologies to build a safe adaptive-delay controller. The adaptive closed-loop system ensures (1) the exact identification of the unknown delay in finite time; (2) the output state stays in the safe region all the time, especially in the original safe region, instead of a subset, after a finite time; (3) all states are bounded, and moreover, they will converge to zero if the target trajectory is identically zero. In the simulation, the proposed control design is verified in the application of safe vehicle platooning. It regulates the spacing between adjacent vehicles to converge to a small distance and avoids collisions by ensuring they do not breach the safe distance at any time, even in the presence of large unknown delays and at a relatively high speed.

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1. Introduction

Vehicle platooning (Ploeg, Van De Wouw, & Nijmeijer, 2013), as an automatic vehicle-following control system that ensures the vehicles in the queue follow each other with a small constant spacing, has gained widespread attention due to its benefits in improving traffic capacity, reducing congestion, and saving fuel. An effective platooning strategy should ensure not only string stability but also safety, i.e., keeping the spacing between adjacent vehicles at a preset safe distance and not breaching this safe distance all the time for the purpose of avoiding collision (Axelsson, 2016). Besides, delay compensation is an important issue in vehicle platooning, considering that delays, which have an impact on the system's stability, widely appear in practice, and moreover, its length cannot always be known exactly. The results of safe control design for vehicle platooning under unknown delays are still rare

because of the technical challenges in combining delay-adaptive design and CBF-based safe control.

1.1. Delay-adaptive control

Since the emergence of "Smith predictor" (Smith, 1959), various control schemes have been developed to reduce the effects of time delays (Artstein, 1982; Bekiaris-Liberis & Krstic, 2010), particularly in nonlinear systems with state delays (Jankovic, 2001; Mazenc & Bliman, 2006) or input delays (Mazenc, Malisoff, & Lin, 2008; Mazenc, Mondie, & Francisco, 2004). In Krstic (2009a), Krstic and Smyshlyaev (2008), a backstepping-based technique was proposed on the basis of representing the time delay as a transport PDE. Utilizing this technique, the problem of delay compensation in nonlinear systems has been addressed in Krstic (2009a, 2009b). This approach has also been extended to compensate for various types of delays, including time-varying delays (Bekiaris-Liberis & Krstic, 2011) and state-dependent delays (Bekiaris-Liberis & Krstic, 2012; Diagne, Bekiaris-Liberis, & Krstic, 2017; Diagne, Bekiaris-Liberis, Otto, & Krstic, 2017). In addition to the ODE systems, this approach has also been applied in PDE delay compensation, such as in Koga, Bresch-Pietri, and Krstic (2020), Lhachemi, Prieur, and Trelat (2020), Qi, Wang, Fang, and Diagne (2019), where the plant becomes a cascade of PDEs after representing the delay as a transport PDE. More results about

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nonlinear delay-compensated control are included in Bekiaris-Liberis (2024), Bekiaris-Liberis and Krstic (2013), Karafyllis and Krstic (2017).

The aforementioned results consider a known delay. When the exact value of the delay is unknown, a delay-adaptive approach is required to compensate for it. A Lyapunov-based adaptive delay controller was developed for ODE plants (Bekiaris-Liberis, Jankovic, & Krstic, 2013; Bresch-Pietri & Krstic, 2014; Krstic & Bresch-Pietri, 2009; Zhu, Krstic, & Su, 2016, 2020; Zhu, Su, & Krstic, 2015). It provides better transient performance than the traditional adaptive methods such as swapping or passive identifiers (Krstic, 2009a) and has been further developed for PDE systems in Wang, Qi, and Diagne (2021), Wang, Qi, and Krstic (2023). Recently, a delay-adaptive controller for coupled hyperbolic PDE subject to an unknown input delay has been proposed in Wang and Diagne (2024), where a delay estimator is built based on batch least-square identifier (BaLSI) that was introduced in Karafyllis, Kontorinaki, and Krstic (2019), Karafyllis and Krstic (2018) for nonlinear ODEs and extended to PDEs in Karafyllis, Krstic, and Chrysafi (2019), Wang and Krstic (2021, 2022), Wang, Krstic, and Karafyllis (2021). By this delay estimator, the unknown delay can be exactly estimated in the finite time, which contributes to better transient performance, enabling exponential regulation of the plant states.

1.2. CBF safe control

The current delay-adaptive control designs do not consider the safety issue. In many engineering applications, like autonomous driving, robotics, and UAV, the safety for avoiding collision is vital (Ames, Grizzle, & Tabuada, 2014; He & Orosz, 2018; Molnar, Kiss, Ames, & Orosz, 2022; Nubert, Kohler, Berenz, Allgower, & Trimpe, 2020). One way to constrain the state in a safe region is the Barrier Lyapunov Function (BLF) method, whose major limitation is that it enforces invariance of every level set, resulting in relatively large conservatism (Ames et al., 2019). Control Barrier Functions (CBFs) introduced in Ames et al. (2019, 2014), Ames, Xu, Grizzle, and Tabuada (2016) have been demonstrated as an effective approach to guarantee safety. The CBF-based safe design constrains the focused system state in the safe region by ensuring the nonnegativity of CBFs and then building a safety filter to override the control law. In addition to the above one relative degree CBFs, the high relative degree CBF design was reported in Nguyen and Sreenath (2016), Xiao and Belta (2021), whose root is the nonovershooting control design in Krstic and Bement (2006) for a class of strict-feedback nonlinear systems. Utilizing this tool (Krstic & Bement, 2006), some advanced safe control designs were proposed for the stochastic nonlinear systems (Li & Krstic, 2020), the PDE model (Koga & Krstic, 2023; Wang & Krstic, 2024, 2025), or in the prescribed-time safety task (Abel, Steeves, Krstic, & Jankovic, 2023) where safety is only enforced within a preset finite time determined by the user. In the case of unknown parameters, guaranteeing safety has attracted the attention of many scholars due to its practical and theoretical significance. The representative work is Taylor and Ames (2020), which introduces the adaptive Control Barrier Functions (aCBFs), on the basis of the adaptive Control Lyapunov Functions (aCLFs), to ensure adaptive safety. However, it has the conservatism that the plant states are constrained in a subset of the original safe set. The study in Lopez, Slotine, and How (2020) alleviates this conservatism by leveraging the parameter adaption and data-driven model estimation. Some extended safe-adaptive control results can be found in Cohen and Belta (2022), Lopez and Slotine (2023). Recently, an adaptive-safe control scheme was proposed in Wang and Krstic (2025) based on the nonovershooting control design in Krstic and Bement (2006) and the BaLSI, which reduces the

conservatism of the current safe-adaptive schemes, constraining the plant states in the original safe set after a finite time, and achieves the exponential regulation of 2 \times 2 hyperbolic PDE-ODE cascade, in the presence of the uncertainties in both PDE and ODE subsystems.

1.3. Contributions

This paper presents the safe delay-adaptive control design for strict-feedback nonlinear ODEs subject to an unknown state delay, ensuring that the output state safely tracks the target trajectory from one side without undershooting. Main contributions are:

- (1) To our knowledge, this is the first safe delay-adaptive control design for a nonlinear system. It incorporates the safety goal into the existing delay-adaptive control results like Bresch-Pietri and Krstic (2014), Wang, Diagne, and Qi (2021), Zhu and Krstic (2020), and removes the known delay requirement in the current safe delay-compensated control (Jankovic, 2018; Molnar, Singletary, Orosz, & Ames, 2020; Orosz & Ames, 2019).
- (2) Compared with the recent safe adaptive control design for sandwich systems (Wang & Krstic, 2025) where the distal ODE to be safely regulated is linear, this paper focuses on the safe regulation of the nonlinear ODE and extends the safe stabilization control in Wang and Krstic (2025) to safe trajectory tracking.

1.4. Organization

The problem is formulated in Section 2. We present the nominal predictor-based safe control design in Section 3. Further, the safe adaptive design for this system, where the delay length is unknown, is proposed in Section 4. The effectiveness of the proposed design scheme is verified in the application of vehicle platooning with avoiding collisions in Section 5. Conclusion and future work are presented in Section 6.

1.5. Notation

- Let $U \in \mathbb{R}^n$ be a set with non-empty interior and let $\Omega \in \mathbb{R}$ be a set. By $\mathbb{C}^0(U;\Omega)$, we denote a class of continuous mappings on U, which takes values in Ω . By $\mathbb{C}^k(U;\Omega)$, where $k \geq 1$, we denote the class of continuous functions on U, which have continuous derivatives of order k on U and take values in Ω .
- The notation $f^{(i)}(t)$ denote i times derivatives of f, $u_t^{(i)}(x, t)$, $u_x^{(i)}(x, t)$ denote i times derivatives with respect to x and with respect to t of u(x, t) respectively.
- Define $y_i(t) := [y_1(t), y_2(t), \dots, y_i(t)]^T$, and $\underline{s}^{(i)}(t) := [s(t), s^{(1)}(t), \dots, s^{(i)}(t)]^T$
- For *n*-vector, the norm $|\cdot|$ denotes the usual Euclidean norm. For square-integrable, measurable functions $u:[0,1]\times\mathbb{R}\to\mathbb{R}$, the norm $\|u(t)\|:=(\int_0^1 u(x,t)^2\,\mathrm{d}x)^{\frac{1}{2}}<+\infty$.
- The symbol e_i denotes that n-dimensional unit vector with i th entry as 1 and other entries are zero, i.e., $e_i := [\underbrace{0,\ldots,0}_{i-1},1,0,\ldots,0]_{1\times n}$.

For ease of presentation, we omit or simplify the arguments of functions and functionals when no confusion arises. Besides, if a > b happens in $\sum_{i=a}^{b}$ of this paper, it means that the result is zero.

2. Problem formulation

We consider the following n+m relative order strict-feedback nonlinear system with an unknown state delay D, whose position and length are arbitrary:

$$\dot{y}_i(t) = y_{i+1}(t) + \psi_i(y_i), \ i = 1, \dots, n-1$$
 (1)

$$\dot{y}_n(t) = \psi_n(y_n) + bx_1(t-D),$$
 (2)

$$\dot{x}_j(t) = x_{j+1}(t) + \varphi_j(\underline{x}_j), j = 1, \dots, m-1$$
(3)

$$\dot{\mathbf{x}}_m(t) = \varphi_m(\mathbf{x}_m) + U(t),\tag{4}$$

where $Y^T(t) = [y_1, y_2, \dots, y_n] \in \mathbb{R}^n$ is the state of the "post-delay" subsystem and $X^T(t) = [x_1, x_2, \dots, x_m] \in \mathbb{R}^m$ is the state of the "pre-delay" subsystem.

Assumption 1. The bound of the unknown parameter is known and arbitrary, i.e., $0 < \underline{D} \le D \le \overline{D}$, where positive constants \underline{D} , \overline{D} are arbitrary and known.

The unknown delay D>0 between the two subsystems satisfies Assumption 1. The signal $y_1(t)$ is the output of the overall plant, and the scalar U(t) is the control input to be designed. The nonzero constant b is arbitrary. Physically, the X-system (3), (4) driven by the control input describes a nonlinear actuator, i.e., X-actuator, whose actuation reaches the nonlinear Y-plant subject to an unknown delay.

Besides, we make the following assumption to ensure that the nonlinear functions ψ_i , φ_j are sufficiently smooth, considering a high relative degree plant is dealt with, which contributes to the well-posedness of the closed system.

Assumption 2. The nonlinearity terms $\psi_i(\underline{y}_i)$ in (1), (2) are n+m-i times continuously differentiable and $\varphi_j(\underline{x}_j)$ in (3), (4) are m-j times continuously differentiable in all their arguments, and $\psi_i(\mathbf{0})=0$, $\varphi_j(\mathbf{0})=0$.

Control objective: Under the unknown delay D, design a controller U(t) to exponentially regulate the output state $y_1(t)$ to track the target trajectory s(t) and ensure

$$y_1(t) - s(t) \ge 0, \quad \forall t \ge 0 \tag{5}$$

i.e., safety defined in this paper, while ensuring that all plant states are bounded. Moreover, when $s(t) \equiv 0$, the exponential convergence to zero of all states in the overall plant is guaranteed.

Because the plant is n+m relative order, we impose the following assumption regarding the required smoothness of the target trajectory s(t).

Assumption 3. The given target trajectory s(t) is n + m times continuously differentiable.

For reducing the reading burden in the design process, we denote the distal Y ODE (1), (2) as

$$\dot{Y}(t) = f(Y(t), bx_1(t-D)). \tag{6}$$

Because there is no control actuation on the Y-subsystem before t=D, we require the following initial condition assumptions ensuring the boundedness and safety of Y-subsystem on this no control period $t\in[0,D]$, which is necessary for safe delay-compensated control in nonlinear systems.

Assumption 4. The distal system (1), (2) with the virtual input $x_1(t)$, i.e., (6), is forward-complete.

Assumption 4 ensures the boundedness of the states before the delayed control action kicks in, i.e., $t \in [0, D]$.

Assumption 5. The initial output state $y_1(0)$ and initial virtual input histories $x_1(\theta)$, $\theta \in [-D, 0)$ are such that $y_1(t) - s(t) \ge 0$ for all $t \in [0, D]$.

Assumption 5 is the sufficient and necessary condition for the situation that $y_1(t)$ is kept in the safe region during the uncontrolled period: $t \in [0, D]$.

Besides, we require the following assumption that restricts the actuator state beginning within the region of safe regulation.

Assumption 6. The initial value of the transport actuator state $x_1(t)$ satisfies $bx_1(0) > h_n(Y(D), \underline{s}^{(n-1)}(t+D)) + s^{(n)}(t+D)$, where the value h_n will be given by (13) in Section 3.1.

Defining

$$u(x,t) = bx_1(t - D + Dx), \tag{7}$$

the delay can be modeled as a transport PDE, and (1), (2) is rewritten as

$$\dot{Y}(t) = f(Y(t), u(0, t)), \tag{8}$$

$$Du_t(x,t) = u_x(x,t), \tag{9}$$

$$u(1,t) = bx_1(t),$$
 (10)

for $x \in [0, 1]$, $t \in [0, +\infty)$. Now the overall plant is (8)–(10) with (3), (4), on the basis of which the control design and stability analysis will be conducted.

3. Safe delay-compensated controller

3.1. Nonundershooting backstepping transformation for the Y part

Following Krstic and Bement (2006), we introduce the safe backstepping transformation

$$z_i(t) = y_i(t) - h_{i-1} - s^{(i-1)}(t), \tag{11}$$

$$h_0 = 0, \ h_1 = -k_1 z_1(t) - \psi_1,$$
 (12)

$$h_i(y_i(t),\underline{s}^{(i-1)}(t)) = -k_i z_i(t) - \psi_i(y_i)$$

$$+\sum_{k=1}^{i-1}\left[\frac{\partial h_{i-1}}{\partial y_k}\left(y_{k+1}(t)+\psi_k\right)+\frac{\partial h_{i-1}}{\partial s^{(k-1)}}s^{(k)}(t)\right],$$

$$i = 1, ..., n$$
 (13)

where the positive design parameters k_1, \ldots, k_n are to be determined later to ensure safety. We then arrive at the transformed target Z-system given by

$$\dot{z}_i(t) = -k_i z_i(t) + z_{i+1}(t), \quad i = 1, \dots, n-1$$
(14)

$$\dot{z}_n(t) = -k_n z_n(t) - h_n - s^{(n)}(t) + u(0, t). \tag{15}$$

The transformed states $Z(t) = [z_1(t), \dots, z_n(t)]^T$ are indeed high-relative-degree CBFs in Wang and Krstic (2025). Considering the control action begins to regulate the *Y*-system from t = D, the CBFs Z(t) need to be kept nonnegative for $t \in [D, \infty]$ in the control design to ensure the safety.

3.2. Predictor-based nonundershooting backstepping transformation for the \boldsymbol{X} part

According to Bekiaris-Liberis and Krstic (2013), we introduce predictor states $P^{T}(t) = [P_1(t), \dots, P_n(t)]$, which denote the D time units ahead predictor of Y(t), i.e.,

$$P(t) = Y(t+D). (16)$$

These predictor states can be generated by integrating (6) from t to t+D, as follows:

$$P(t) = Y(t) + \int_{t}^{t} f(P(\sigma), bx_1(\sigma)) d\sigma, \qquad (17)$$

with the initial condition

$$P(\theta) = Y(0) + \int_{-D}^{\theta} f(P(\sigma), bx_1(\sigma)) d\sigma, \theta \in [-D, 0].$$
 (18)

The nonlinear predictor (17) can be solved via the numerical approximation that is illustrated in Chapter 4 of Karafyllis and Krstic (2017).

Based on the predictor (17), (18), we define an auxiliary system as

$$Dp_t(x,t) = p_x(x,t), (19)$$

$$p(1,t) = P(t), \tag{20}$$

$$p(x,0) = P(D(x-1)), x \in [0,1]$$
(21)

whose solution $p^{T}(x, t) = [p_1(x, t), \dots, p_n(x, t)]$ is

$$p(x,t) = P(t - D + Dx), \ x \in [0,1] \times t \in [0,\infty).$$
 (22)

Relying on the auxiliary system, applying the following transformation

$$w(x,t) = u(x,t) - h_n(p(x,t), \underline{s}^{(n-1)}(t+Dx)) - s^{(n)}(t+Dx)$$
 (23)

for $x \in [0, 1]$ and the nonundershooting backstepping transformation

$$r_j(t) = bx_j(t) - \tau_{j-1} - \Delta^{(j-1)}(t),$$
 (24)

$$\tau_0 = 0, \ \tau_1 = -c_1 r_1(t) - b \varphi_1,$$
 (25)

$$\tau_i(\underline{x}_i(t),\underline{\Delta}^{(j-1)}(t)) = -c_i r_i(t) - b\varphi_i(\underline{x}_i)$$

$$+\sum_{k=1}^{j-1}\left[\frac{\partial\tau_{j-1}}{\partial x_k}\left(x_{k+1}(t)+\varphi_k\right)+\frac{\partial\tau_{j-1}}{\partial\Delta^{(k-1)}}\Delta^{(k)}(t)\right],$$

$$j=1,\ldots,m \tag{26}$$

where c_1, \ldots, c_m are some positive design parameters to be determined later, and where

$$\Delta(t) = h_n(P(t), s^{(n-1)}(t+D)) + s^{(n)}(t+D), \tag{27}$$

we arrive at the target system

$$\dot{z}_i(t) = -k_i z_i(t) + z_{i+1}(t), i = 1, \dots, n-1$$
(28)

$$\dot{z}_n(t) = -k_n z_n(t) + w(0, t), \tag{29}$$

$$Dw_t(x,t) = w_x(x,t), \tag{30}$$

$$w(1,t) = r_1(t), (31)$$

$$\dot{r}_i(t) = -c_i r_i(t) + r_{i+1}(t), j = 1, \dots, m-1$$
(32)

$$\dot{r}_m(t) = -c_m r_m(t),\tag{33}$$

with choosing the control law as

$$U(t) = \frac{1}{b} \left(\tau_m(\underline{x}_m(t), \underline{\Delta}^{(m-1)}(t)) + \underline{\Delta}^{(m)}(t) \right)$$

:= $\mathcal{U}(t, D)$. (34)

The above nominal control law U(t) is constructed using the plant states X(t), the predictor states P(t), and the derivatives of the trajectory function s(t) up to (n+m) order at the future moment t+D, i.e., $\underline{s}^{(n+m)}(t+D)$. We write D as an argument of the function \mathcal{U} (34) because the predictor P(t) and the trajectory functions $s^{(n+m)}(t+D)$ depend on it.

3.3. Inverse transformations

In this subsection, we derive the inverse transformations of the transformations in Sections 3.1, 3.2, i.e., (11)–(13), (23), (24)–(26), converting the target system back to the original system. First, we show the predictors $\delta^T(x, t) = [\delta_1(x, t), \ldots, \delta_n(x, t)]$ of the transformed states Z(t) in (28), (29) as follows:

$$\delta(x, t) = Z(t + Dx)$$

$$= e^{DAx}Z(t) + D \int_{0}^{x} e^{DA(x-y)}Bw(y,t) dy$$
 (35)

for $x \in [0, 1] \times t \in [0, \infty)$, where

$$A = \begin{pmatrix} -k_1 & 1 & 0 & 0 & \cdots & 0 & 0 \\ 0 & -k_2 & 1 & 0 & \cdots & 0 & 0 \\ & & \ddots & & \vdots & \vdots & \\ 0 & 0 & 0 & 0 & \cdots & -k_{n-1} & 1 \\ 0 & 0 & 0 & 0 & \cdots & 0 & -k_n \end{pmatrix}, B = \begin{pmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{pmatrix}, B = \begin{pmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{pmatrix}_{n \times 1}$$
(36)

which will be used in building the inverse transformations.

Proposition 1. The inverse transformations of (11)–(13) are

$$y_i(t) = z_i(t) + \bar{h}_{i-1} + s^{(i-1)}(t), \tag{37}$$

$$\bar{h}_0 = 0, \ h_1 = -k_1 z_1(t) - \bar{\psi}_1,$$
 (38)

$$\bar{h}_i(z_i(t), s^{(i-1)}(t)) = -k_i z_i(t) - \bar{\psi}_i(z_i, s^{(i-1)})$$

$$+\sum_{k=1}^{i-1} \left(\frac{\partial \bar{h}_{i-1}}{\partial z_k} \left(-k_k z_k + z_{k+1}\right) + \frac{\partial \bar{h}_{i-1}}{\partial s^{(k-1)}} s^{(k)}(t)\right),\,$$

$$i=1,\ldots,n \tag{39}$$

where the functions $\bar{h}_i(\underline{z}_i, \underline{s}^{(i-1)})$, $\bar{\psi}_i(\underline{z}_i, \underline{s}^{(i-1)}) = \psi_i(\underline{y}_i)$ are continuously differentiable in all their arguments and satisfy $\bar{h}_i(0,0) = 0$, $\bar{\psi}_i(0,0) = 0$. The inverse of the transformation (23) is

$$u(x,t) = w(x,t) + \bar{h}_n(\delta(x,t), s^{(n-1)}(t+Dx)) + s^{(n)}(t+Dx), \tag{40}$$

where $\delta(x, t)$ is given by (35). The inverse transformations of (24)–(26) are

$$x_{j}(t) = \frac{1}{h}r_{j}(t) + \bar{\tau}_{j-1} + \frac{1}{h}\bar{\Delta}^{(j-1)}(t), \tag{41}$$

$$\bar{\tau}_0 = 0, \ \bar{\tau}_1 = -\frac{c_1}{h}r_1(t) - \bar{\varphi}_1,$$
 (42)

$$\bar{\tau}_j(\underline{r}_j(t), \underline{\bar{\Delta}}^{(j-1)}(t)) = -\frac{c_j}{h}r_j(t) - \bar{\varphi}_j(\underline{r}_j, \underline{\bar{\Delta}}^{(j-1)})$$

$$+\sum_{k=1}^{j-1}\left[\frac{\partial\bar{\tau}_{j-1}}{\partial r_k}\left(-c_kr_k+r_{k+1}\right)+\frac{\partial\bar{\tau}_{j-1}}{\partial\bar{\Delta}^{(k-1)}}\bar{\Delta}^{(k)}(t)\right],$$

$$j = 1, \dots, m \tag{43}$$

where

$$\bar{\Delta}(t) = \bar{h}_n(\delta(1, t), s^{(n-1)}(t+D)) + s^{(n)}(t+D), \tag{44}$$

and the functions $\bar{\tau}_j(\underline{r}_j, \underline{\bar{\Delta}}^{(j-1)})$, $\bar{\varphi}_j(\underline{r}_j, \underline{\bar{\Delta}}^{(j-1)}) = \varphi_j(\underline{x}_j)$ are continuously differentiable in all their arguments and satisfy $\bar{\tau}_j(0,0) = 0$, $\bar{\varphi}_j(0,0) = 0$.

Proof. The proof is shown in Appendix A. \Box

According to (37)–(43), we have built the invertible transformation between the (Y, u, X)-original system with the predictor depending on Y, u and the (Z, w, R)-target system with the predictor δ depending on Z, w, which is illustrated in Fig. 1. The

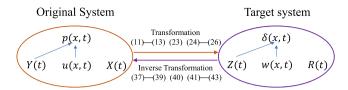


Fig. 1. The transformation between the Y, u, X-original and Z, w, R-target systems with the predictors p, δ .

invertibility built here will be used to prove the stability of the closed-loop system.

3.4. Selection of design parameters

We need to select the design parameters under certain conditions to ensure safety, as shown in this subsection. We select the parameters k_1, \ldots, k_n to satisfy

$$k_i > \max\{2, \check{k}_i\}, i = 1, \dots, n - 1, \quad k_n > 1,$$
 (45)

where

$$\check{k}_{i} = \frac{1}{P_{i}(0) - h_{i-1}(\underline{P}_{i-1}(0), \underline{s}^{(i-2)}(D)) - s^{(i-1)}(D)}
\times \left[-P_{i+1}(0) - \psi_{i}(\underline{P}_{i}(0)) + s^{(i)}(D) \right]
+ \sum_{k=1}^{i-1} \left(\frac{\partial h_{i-1}}{\partial P_{k}} \left(P_{k+1}(0) + \psi_{k} \right) + \frac{\partial h_{i-1}}{\partial s^{(k-1)}} s^{(k)}(D) \right) \right],
i = 1, \dots, n-1$$
(46)

and where $P_i(0) = e_i P(0)$ is the initial condition of the predictor, which is determined by the initial condition Y(0) and initial virtual input histories $x_1(\theta)$, $\theta \in [-D,0)$ via (18). The purpose of this design parameter selection is to ensure that the CBF states $z_i(t)$, $i=1,\ldots,n$ are positive at the initial time t=D for control action at the distal ODE (1), (2). This will be seen clearly in the proof of Lemma 4.

The design parameters c_1, \ldots, c_m are selected as:

$$c_j > \max\{2, \check{c}_j\}, j = 1, \dots, m - 1, \quad c_m > 1,$$
 (47)

where

$$\check{c}_{j} = \frac{1}{bx_{j}(0) - \tau_{j-1}(\underline{x}_{j-1}(0), \underline{\Delta}^{(j-2)}(0)) - \underline{\Delta}^{(j-1)}(0)}
\times \left[-bx_{j+1}(0) - b\varphi_{j}(\underline{x}_{j}(0)) + \underline{\Delta}^{(j)}(0)
+ \sum_{k=1}^{j-1} \left(\frac{\partial \tau_{j-1}}{\partial x_{k}} (x_{k+1}(0) + \varphi_{k}) + \frac{\partial \tau_{j-1}}{\partial \underline{\Delta}^{(k-1)}} \underline{\Delta}^{(k)}(0) \right) \right],
j = 1, ... m - 1$$
(48)

where $\Delta(0)$ given by (27) depends on the initial predicted value P(0) that is determined by the initial condition Y(0) and initial virtual input histories $x_1(\theta)$, $\theta \in [-D,0)$ via (18), as mentioned before. This selection is to make the values of CBF states $r_j(t)$, $j=1,\ldots,m$ positive at the initial time t=0, which will be seen clearly in the proof of Lemma 3. Additionally, it also contributes to the exponential regulation of all plant states, which will be shown in Lyapunov analysis (B.5).

3.5. Result with the nominal safe delay-compensated control

Theorem 1. For initial data $Y(0) \in \mathbb{R}^n$, $X(0) \in \mathbb{R}^m$, the initial input history $x_1(t) \in \mathbb{C}^{m-1}([-D, 0])$ satisfying Assumptions 4–6, and a target trajectory s(t) satisfying Assumption 3, choosing the design

parameters $k_1, \ldots, k_n, c_1, \ldots, c_m$ satisfying (45)–(48), the closed-loop system (1)–(4) with the nominal controller (34) and (17), (18) has the following properties

1. The output $y_1(t)$ exponentially tracks the target trajectory s(t) in the sense that $|y_1(t) - s(t)|$ exponentially converges to zero, and all plant states, i.e.,

$$\Psi(t) = |X(t)| + |Y(t)| + \sup_{t - D \le \tau \le t} |x_1(\tau)|$$
(49)

is bounded, and the ultimate bound depends on the target trajectory. If the target trajectory $s(t) \equiv 0$, then $\Psi(t)$ is exponentially convergent to zero.

2. The safety is enforced in the sense that $y_1(t) - s(t) \ge 0$ holds on t > 0.

Before presenting the proof, we propose the following lemmas regarding the exponential stability and non-negative CBFs in the target system (28)–(33), which will be used in establishing the theorem.

Lemma 1. The exponential stability of the target system (28)–(33) is achieved in the sense that there exist positive constants Υ_{Ω} and σ_{Ω} such that

$$\Omega(t) \le \Upsilon_{\Omega} \Omega(0) e^{-\sigma_{\Omega} t} \tag{50}$$

where

$$\Omega(t) = \sum_{i=1}^{n} z_i(t)^2 + \sum_{i=1}^{m} r_i(t)^2 + \sum_{i=0}^{m} \|w_x^{(i)}(\cdot, t)\|^2.$$
 (51)

Proof. The proof is shown in Appendix B. \Box

The inclusion of the higher spatial derivatives of w in (51) is to show the exponential regulation of the overall system including the actuator dynamics, which will be seen clearly later.

Lemma 2. For the predictor δ (35) of the transformed state Z(t) in the target system, the signals $|\delta(x,t)|^2$, $\forall x \in [0,1]$ are exponentially convergent to zero, and also $|\delta_t^{(i)}(1,t)|^2$, $i=0,\ldots,m$ are exponentially convergent to zero.

Proof. The proof is shown in Appendix C. \Box

We show that the CBFs are ensured non-negative in the following two lemmas.

Lemma 3. The high-relative-degree CBFs $r_j(t)$, $j=1,\ldots,m$ are nonnegative all the time under the selection of design parameters (47), (48), i.e., $r_j(t) \ge 0$, $j=1,\ldots,m$, for time $t \ge 0$.

Proof. Firstly, we claim that all the initial values of $r_i(t)$ are positive. Recalling Assumption 6 and (7), (23), (27), (31), we have the initial condition $r_1(0) = w(1,0) > 0$. Setting t = 0 in (24), we have

$$r_{j+1}(0) = bx_{j+1}(0) - \tau_{j}(\underline{x}_{j}(0), \underline{\Delta}^{(j-1)}(0)) - \Delta^{(j)}(0)$$

$$= bx_{j+1}(0) - \Delta^{(j)}(0) + c_{j}r_{j}(0) + b\varphi_{j}(\underline{x}_{j}(0))$$

$$- \sum_{k=1}^{j-1} \left(\frac{\partial \tau_{j-1}}{\partial x_{k}}(x_{k+1}(0) + \varphi_{k}) + \frac{\partial \tau_{k-1}}{\partial \Delta^{(k-1)}}\Delta^{(k)}\right)$$
(52)

for $j=1,\ldots,m-1$. For $r_j(0)>0$, using the design parameter selection of c_j in (47), (48), we obtain from (52) that $r_{j+1}(0)>0$. Based on this induction result, starting from $r_1(0)>0$, it can recursively infer that all the initial conditions $r_j(0)>0$, $j=1,\ldots,m$. The solution of (32), (33) is $r_m(t)=r_m(0)e^{-c_mt}$,

 $r_j(t)=r_j(0)e^{-c_jt}+\int_0^t e^{-c_j(t-y)}r_{j+1}(y)\,\mathrm{d}y, j=1,\ldots,m-1$, together with the above positive initial condition, the lemma is then obtained. \square

Lemma 4. The high-relative-degree CBFs $z_i(t)$, $i=1,\ldots,n$ are nonnegative under the selection of design parameters (45), (46), i.e., $z_i(t) \ge 0$, $i=1,\ldots,n$, for time $t \ge D$.

Proof. Setting t = D and replacing Y(D) with P(0) in (11), then one gets

$$z_{i+1}(D) = P_{i+1}(0) - h_i(\underline{P}_i(0), \underline{s}^{(i-1)}(D)) - s^{(i)}(D)$$

$$= P_{i+1}(0) - s^{(i)}(D) + k_i z_i(D) + \psi_i(\underline{P}_i(0))$$

$$- \sum_{k=1}^{i-1} \left(\frac{\partial h_{i-1}}{\partial P_k} \left(P_{k+1}(0) + \psi_k\right) + \frac{\partial h_{i-1}}{\partial s^{(k-1)}} s^{(k)}(D)\right).$$
(53)

Under the selection of parameters k_i in (45), (46), we also obtain the induction step: if $z_i(D) > 0$, then $z_{i+1}(D) > 0$ according to (53). Because of the base case that $z_1(D) = y_1(D) - s(D) > 0$ ensured in Assumption 5, applying the induction step obtained above, we thus have that all $z_i(D) > 0$, $i = 1, \ldots, n$. Considering the structure of (28), (29), together with the initial conditions $z_i(D) > 0$, and $w(0, t) = r_1(t - D) \ge 0$, $t \ge D$ according to Lemma 3 as well as (30), (31), we obtain that $z_i(t) > 0$, $i = 1, \ldots, n$ on $t \in [D, \infty]$. \square

Now, we are ready to show the proof of Theorem 1:

Proof. (1) The stability results of the target system in Lemmas 1 and 2 indicate that $|Z(t)|^2$, $|R(t)|^2$, $\sum_{i=0}^m \|w_x^{(i)}(\cdot,t)\|^2$, $|\delta(x,t)|^2$, $\forall x \in [0,1]$, and $|\delta_t^{(i)}(1,t)|^2$, $i=0,\ldots,m$ are exponentially convergent to zero.

Considering the tracking error $|z_1(t)|$ exponentially converges to zero, we have that $|y_1(t) - s(t)|$ exponentially converges to zero.

Recalling Proposition 1, applying Cauchy–Schwarz inequality, it follows (37)–(39) and exponential convergence to zero of $z_i(t)^2$ that $y_i(t)^2$ are bounded and the ultimate bound depends on $\sum_{j=0}^{i-1} s^{(j)}(t)^2$ for $i=1,\ldots,n$. According to (40) in Proposition 1, it is then obtained that $\sup_{x\in[0,1]}|u(x,t)|$ is bounded and the ultimate bound depends on the functions $\int_0^1 |\underline{s}^{(n)}(t+Dx)| \, \mathrm{d}x$. Recalling (7), we thus have that

$$\sup_{t-D \le \tau \le t} |x_1(\tau)| = |b| \sup_{x \in [0,1]} |u(x,t)| \tag{54}$$

is also ultimately bounded by a function depending on $\int_0^1 |\underline{s}^{(n)}(t+Dx)| \, \mathrm{d}x$.

Considering the forward-completeness of the distal system (1), (2) defined in Assumption 4 and the bounded initial condition of the virtual input state $x_1(\theta)$, $\theta \in [-D, 0)$, it follows that

$$P(\theta) < \infty. \tag{55}$$

Plugging the predicted states p(1, t), $\delta(1, t)$ into the inverse transformation (37), we have

$$p_{i}(1,t) = \delta_{i}(1,t) + \bar{h}_{i-1}(\underline{\delta}_{i-1}(1,t), \underline{s}^{(i-2)}(t+D)) + s^{(i-1)}(t+D), \quad i = 1, \dots, n.$$
(56)

Recalling the continuous differentiability of \bar{h}_i in (39) and the exponential convergence to zero of $\delta_i(x,t)$ proven above, we have that $e_i P(t) = p_i(1,t)$ is bounded for t>0 and the ultimate bound depends on $\underline{s}^{(i-1)}(t+D)$ for $i=1,\ldots,n$. Recalling (22) and (55), we know $|p(x,t)|^2$ are bounded and the ultimate bound depends on $\underline{s}^{(i-1)}(t+Dx)^2$.

Recalling (44), one obtains that $|\underline{\bar{\Delta}}^{(i)}(t)|^2$ is bounded with the ultimate bound depending on $\sum_{j=0}^n s^{(j)}(t+D)^2$, because

 $|\delta_t^{(i)}(1,t)|^2$, $i=0,\ldots,m$ are exponentially convergent to zero and the h_n in (39) is continuously differentiable. Recalling Proposition 1, applying Cauchy–Schwarz inequality, it follows (41)–(43) and exponential convergence to zero of $r_i(t)^2$, $i=1,\ldots,m$ that $x_i(t)^2$ are bounded and the ultimate bound depends on $\sum_{j=0}^{i-1} s^{(n+j)}(t+D)^2$ for $i=1,\ldots,m$. Therefore, we conclude that $\Psi(t)$ defined by (49) is bounded, and the ultimate bound depends on the given target trajectory s(t).

Next, we show that $\Psi(t)$ (49) is exponentially convergent to zero if the target trajectory $s(t) \equiv 0$. Considering the exponential convergence to zero of |Z(t)| and $\bar{h}_n(0,0)=0$, applying Cauchy– Schwarz inequality for (37) in Proposition 1, we know that |Y(t)|is exponentially convergent to zero. Recalling the inverse transformation of u(x, t) (40) in Proposition 1, it is obtained from the exponential convergence to zero of $|\delta(x,t)|$, $||w(\cdot,t)||$, and $h_n(0,0) = 0$ that $\sup_{x \in [0,1]} |u(x,t)|$ is exponentially convergent to zero. It follows (54) that $\sup_{t-D \le \tau \le t} |x_1(\tau)|$ is also exponentially convergent to zero. It is obtained from (56) that p(1, t) = P(t)are exponentially convergent to zero considering $\bar{h}_i(0,0) = 0$ in (39), and the exponential convergence to zero of $|\delta(x, t)|$ obtained before. Moreover, according to (22) and (55) with Assumption 4, the exponential convergence to zero of |p(x, t)|, $\forall x \in [0, 1]$ is obtained. Then, recalling (44), we have that $|\bar{\Delta}^{(i)}(t)|, i = 0, \dots, m$ are exponentially convergent to zero because of the exponential convergence to zero of $|\delta_t^{(1)}(1,t)|$ obtained above and continuous differentiability of \bar{h}_n . Finally, it is obtained from (41)–(43) in Proposition 1 that |X(t)| is exponentially convergent to zero, recalling the exponential convergence to zero of |R(t)|, $|\bar{\Delta}^{(i)}(t)|$, i = $0, \ldots, m, \text{ and } \bar{\tau}_i(0, 0) = 0.$

The Property 1 is obtained.

(2) Over the time period $t \in [0, D]$, i.e., when no control action reaches, the Y-system (1), (2) is only actuated by the initial input history signal $x_1(t)$, $t \in [-D, 0]$. Under Assumption 5, we get $y_1(t)-s(t)>0$, $t \in [0, D]$, i.e., the safety is ensured on $t \in [0, D]$. From Lemma 4, it holds that $z_1(t)=y_1(t)-s(t)\geq 0$, $t \in [D, \infty)$. Consequently, the safety is ensured all the time. The Property 2 is thus proved.

The proof of Theorem 1 is complete. \Box

Based on safe infinite-dimensional backstepping transformations, a nominal safe delay-compensated controller is proposed in this section. Next, given the uncertainty of the delay, we will propose a safe delay-adaptive controller.

4. Safe delay-adaptive controller

4.1. Delay-adaptive control design

Following the delay-adaptive design in Wang and Diagne (2024), we obtain the delay identifier:

$$\hat{D}(t_{i+1}) = \arg\min \left\{ |\ell - \hat{D}(t_i)|^2 : \ell \in D_0,
G_n(t_{i+1}, \mu_{i+1})\ell = F_n(t_{i+1}, \mu_{i+1}), n = 1, 2, \ldots \right\},$$
(57)

where the set $D_0 := \{\ell \in \mathbb{R} : \underline{D} \leq \ell \leq \overline{D}\}$ uses the known bounds of the unknown delay given in Assumption 1, and where $\{t_i \geq 0\}_{i=0}^{\infty}, i = 0, 1, 2, \dots$ is a sequence of time instants for identification, defined as

$$t_{i+1} = t_i + T. (58)$$

The free positive design parameter T denotes the dwell time between two adjacent triggering times. A larger T can improve the robustness of the delay identifier against sensor measurement

errors because it allows more measured data to join the parameter identification, at the cost of increased computation time. The time instant μ_{i+1} is defined as

$$\mu_{i+1} := \min\{t_g : g \in \{0, \dots, i\}, t_g \ge t_{i+1} - \tilde{N}T\},$$
 (59)

where the positive integer $\tilde{N} \geq 1$ is a free design parameter, which determines the size of the data set used in the delay identification at t_{i+1} , reflecting a trade-off between the identifier's robustness and computation cost.

The functions G_n , F_n in (57) are given as $G_n(t_{i+1}, \mu_{i+1}) = \int_{\mu_{i+1}}^{t_{i+1}} g_n(t)^2 dt$, $F_n(t_{i+1}, \mu_{i+1}) = \int_{\mu_{i+1}}^{t_{i+1}} g_n(t) f_n(t) dt$, where $f_n(t) = \pi n \int_0^t \int_0^1 \cos(x\pi n) u(x, \tau) dx d\tau$, $g_n(t) = -\int_0^1 \sin(x\pi n) u(x, t) dx$. The detailed design process and the proof of exact delay identifiation of the proof of cation can be found in Wang and Diagne (2024). We only consider the scenario where $x_1(t - D) = 0$ on $t \in [0, D)$, i.e., there is no signal reaching Y-subsystem before t = D according to (2), in the adaptive control. For the case that $x_1(t)$ is not identically zero on $t \in [-D, 0)$, a slight modification is needed in the formulation of the delay identifier, and some expanded analysis is required in the proof.

Now, using the proposed estimate $\hat{D}(t_i)$ to replace the unknown delay D in the nominal controller (34), we construct a delay-adaptive controller

$$U_d(t) = \mathcal{U}(t, \hat{D}(t_i)), \ t \in [t_i, t_{i+1}). \tag{60}$$

The safety ensured by the nominal control cannot be guaranteed here because of the delay identification error. Following the safeadaptive control design in Wang and Krstic (2025), we introduce a QP safety filter (67) to override the potentially unsafe adaptive controller (60) to enforce the safety in the adaptive control.

4.2. Safe delay-adaptive control design

First, considering the unknown D, we select the design parameters k_i , c_i (45), (47) as

$$k_i > \max_{\mathcal{D} \in \mathcal{D}_n} \{2, \check{k}_i(\mathcal{D})\}, i = 1, \dots, n - 1, k_n > 1,$$
 (61)

$$k_{i} > \max_{\mathcal{D} \in \mathcal{D}_{0}} \{2, \check{k}_{i}(\mathcal{D})\}, i = 1, \dots, n - 1, k_{n} > 1,$$

$$c_{j} > \max_{\mathcal{D} \in \mathcal{D}_{0}} \{2, \check{c}_{j}(\mathcal{D})\}, j = 1, \dots, m - 1, c_{m} > 1,$$
(62)

where $\dot{k}_i(\mathcal{D})$, $\check{c}_i(\mathcal{D})$ are obtained by replacing the unknown delay D in (46), (48) by $\mathcal{D} \in [D, \overline{D}]$, where the bounds D, \overline{D} are known according to Assumption 1. Because the condition (61), (62) is a subset of the one (45), (47), the positive initialization about $r_i(0)$ and $z_i(D)$, as shown in the proofs of Lemmas 3, 4, still hold here. Then recalling the target system (28)-(33) and the analysis about the safety in the proof of Property 2 in Theorem 1, we know the safety objective $z_1(t) = y_1(t) - s(t) \ge 0$ is achieved as long as $r_m(t) > 0$ for all time t > 0, of which a sufficient condition is

$$\dot{r}_m(t) \ge -\overline{c}r_m(t),\tag{63}$$

where the positive parameter \bar{c} is free. A safe region for the control action is then obtained from (63) as $S(t) = \{u \in \mathbb{R} : t \in \mathbb{R}$ $bu \geq b\mathcal{U}^*(t, D)$ } where

$$U^{*}(t,D) = \frac{1}{b} \left[(c_{m} - \overline{c}) r_{m}(t) + \tau_{m} + \Delta^{(m)}(t) \right].$$
 (64)

Considering the unknown D, by replacing the unknown delay by \mathcal{D} , a conservative safe region of the adaptive control input is

$$C(t) = \left\{ u \in \mathbb{R} : bu \ge \max_{\mathcal{D} \in D_0} b\mathcal{U}^*(t, \mathcal{D}) \right\}.$$
 (65)

By using a QP safety filter to constrain the input signal within this safe region (65) before exact identification is achieved, we build

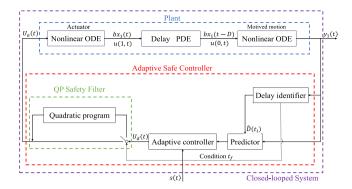


Fig. 2. The diagram of the safe delay-adaptive control.

the following safe adaptive controller:

$$U_{a}(t) = \begin{cases} \arg\min_{u \in \mathbb{R}} \{|u - U_{d}|\}^{2} \\ s.t. & u \in C(t), \\ U_{d}, & t \in [t_{f}, \infty) \end{cases}$$
(66)

whose explicit solution is

$$U_{a}(t) = \begin{cases} \max\{U_{d}, \max_{\mathcal{D} \in D_{0}} \mathcal{U}^{*}(t, \mathcal{D})\}, & \text{if } b > 0; \\ \min\{U_{d}, \min_{\mathcal{D} \in D_{0}} \mathcal{U}^{*}(t, \mathcal{D})\}, & \text{if } b < 0, \\ 0 \leq t \leq t_{f} \\ U_{d}(t). & t > t_{f} \end{cases}$$
(67)

The switching time t_f is the triggering time when the delay D is exactly identified, determined by

$$t_f = \min\{t_i : \exists t \in [0, t_i), u(\cdot, t) \neq 0\},\tag{68}$$

according to the proof of exact identification of delay shown in Section 5 of Wang and Diagne (2024). The diagram of the proposed safe delay-adaptive control system is depicted in Fig. 2. The practical implementation of this safe delay-adaptive control law can refer to Remarks 3, 4 in the simulation.

4.3. Result with safe delay-adaptive control

Comparing the safe nominal controller (34) with the safe adaptive controller (67), we define their difference as

$$\gamma(t) = bU_a(t) - bU(t). \tag{69}$$

Then, implementing $U_a(t)$ as the input into the original system (1)–(4), the target system becomes (28)–(32) with

$$\dot{r}_m(t) = -c_m r_m(t) + \gamma(t). \tag{70}$$

Remark 1. In the control input (34), the delay D, which exists in the predictor state $P_i(t)$ of the distal Y-system as shown in (17), is associated with $\Delta(t)$, i.e., the parts related to states of the *Y*-system, while independent of the signals from the *X*-system. Thus, the function $\gamma(t)$ given by (69) does not contain the signals from the R-system (in the form of states of the target system), that is, $\gamma(t)$ can be regarded as an external signal to (70).

Proposition 2. For every $(u(\cdot,0),X(0),Y(0)) \in \mathbb{C}^{m-1}([0,1]) \times$ $\mathbb{R}^m \times \mathbb{R}^n$, there exist a unique solution $(u, X, Y) \in \mathbb{C}^{m-1}([0, \infty) \times$ [0,1]) $\times \mathbb{C}^0([0,\infty);\mathbb{R}^m)\times \mathbb{C}^0([0,\infty);\mathbb{R}^n)$ to the system (1)–(4) with control input (67).

Proof. It is obtained from $(u(\cdot,0),X(0),Y(0)) \in \mathbb{C}^{m-1}([0,1]) \times$ $\mathbb{R}^m \times \mathbb{R}^n$ and the transformations (11)–(13), (23), (24)–(26) that $(w(\cdot,0),R(0),Z(0)) \in \mathbb{C}^{m-1}([0,1]) \times \mathbb{R}^m \times \mathbb{R}^n$. According to

Remark 1 and [Proposition 1, Wang and Krstic (2025)], for the target system consisting of (28)–(32) and (70), we have that $(w(\cdot,t),R(t),Z(t))\in\mathbb{C}^{m-1}([0,\infty)\times[0,1])\times\mathbb{C}^0([0,\infty);\mathbb{R}^m)\times\mathbb{C}^0([0,\infty);\mathbb{R}^n)$ in the weak sense. Recalling the inverse transformations (37)–(43), this proposition is then obtained. \square

The result of safe delay-adaptive control is presented as follows.

Theorem 2. For the initial condition $Y(0) \in \mathbb{R}^n$, $X(0) \in \mathbb{R}^m$ satisfying Assumptions 4–6, the history input signal $x_1(t) = 0$ on $t \in [-D, 0)$, and a target trajectory s(t) satisfying Assumption 3, choosing the design parameters $k_1, \ldots, k_n, c_1, \ldots, c_m$ satisfying (61), (62), the closed-loop system (1)–(4) with the safe delay-adaptive controller (67) has the following properties

- 1. The delay estimation $\hat{D}(t)$ is bounded and reaches the true value in finite time t_f .
- 2. The output $y_1(t)$ exponentially tracks the target trajectory s(t) in the sense that $|y_1(t) s(t)|$ exponentially converges to zero, and all plant states, i.e., $\Psi(t)$ given by (49), are bounded, and the ultimate bound depends on the target trajectory. If the target trajectory $s(t) \equiv 0$, then $\Psi(t)$ is exponentially convergent to zero.
- 3. The safety is ensured in the sense that $y_1(t) s(t) \ge 0$ hold on $t \ge 0$.

Proof. (1) The proof of Property 1 can be found in Sec. IV-C of Wang and Diagne (2024).

(2) Recalling the Lyapunov function V(t) (B.1), choose the analysis parameters as $a_0 > \overline{D}$, $a_i > 0$, $i = 1, \ldots, m$, i.e., replacing the unknown D in the condition (B.6) by the known bounds \underline{D} , \overline{D} . Recalling (70) in the target system of the adaptive case, the inequity (B.4) now becomes $\sum_{i=1}^m a_i e D^{i-1} r_1^{(i)}(t)^2 \le \sum_{i=1}^m b_i r_i(t)^2 + \overline{b} \gamma^2(t)$, where b_i , \overline{b} depend on the upper bound \overline{D} , the design parameters c_i in (62), and the analysis parameters a_i . Implementing the process similar to (B.3)–(B.5) and choosing the design parameters ρ (B.7) as $\rho > \max\left\{\frac{a_0 e}{2} + \frac{1}{3}b_1, b_m, \frac{1}{2}b_j\right\} + 1, j = 2, \ldots, m-1$. The time derivative of $\overline{V}(t)$ in (B.8) becomes

$$\dot{V}(t) \le -\varrho V(t) + \bar{b}\gamma^2(t) + \rho r_m(t)\gamma(t), \tag{71}$$

where $\varrho=\frac{1}{\theta_2}\min\left\{1,\frac{a_i}{2\overline{D}}\right\}, i=0,\ldots,m$. For the time period $t\in[t_f,\infty]$, recalling Property 1 in Theorem 2 and (67), we know that $U_a(t)=U(t)$, i.e., $\gamma(t)=0$. Thus, for (71), we obtain $V(t)\leq V(t_f)e^{-\varrho(t-t_f)}$, $\forall t\geq t_f$. From Remark 1 and transformations (11)–(13), (23)–(27), one obtains $\gamma^2(t)\leq \Upsilon\Omega(t)$ for some positive Υ , where $\Omega(t)$ is defined in (51). Thus it follows (B.2), (71) that $\dot{V}(t)\leq -\varrho V(t)+\varrho_0 V(t), t\in [0,t_f)$ for some positive ϱ_0 . We then get that $V(t)\leq V(0)e^{|\varrho_0-\varrho|t_f}, t\in [0,t_f)$. Considering the continuity of V(t) by recalling Proposition 2, we have that $V(t_f)\leq V(0)e^{|\varrho_0-\varrho|t_f}$. From the above relation, it further implies that

$$V(t) \le V(0)e^{|\varrho_0 - \varrho|t_f + \varrho t_f}e^{-\varrho t} \tag{72}$$

for $t\in [0,\infty)$. According to (50), we have that $\Omega(t)\leq \frac{\theta_2}{\theta_1} \gamma_0 \Omega(0) e^{-\varrho t}$, where $\gamma_0=e^{|\varrho_0-\varrho|t_f+\varrho t_f}$. Through the following process in the proof of Property 1 in Theorem 1, we obtain Property 2 in this theorem.

(3) Implementing $U_a(t)$ as the input into the original system (1)–(4) and recalling the transformation (11)–(13), (24)–(26), we have

$$\dot{r}_m(t) = -\bar{c}r_m(t) + \overline{\gamma}(t), \ t \in [0, t_f)$$
(73)

where $\overline{\gamma}(t) = bU_a(t) - b\mathcal{U}^*(t, D) \ge 0$ recalling (34), (63)–(67). Since $z_i(D)$, $r_j(0)$ are positive under the choice of the design parameters k_i , c_j in (61), (62), we have that $r_m(t) > 0$, $t \in [0, t_f)$

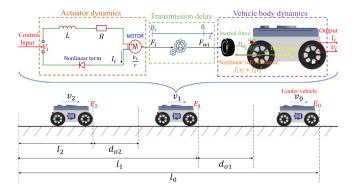


Fig. 3. Vehicle platooning with leader E_0 , and the followers E_i , i = 1, 2, where the safe distances to be maintained are d_{oi} .

from the structure of (73). Considering the continuity of $r_m(t)$ in Proposition 2, we obtain $r_m(t_f) = r_m(t_f^-) > 0$. When the adaptive input signal $U_a(t)$ is equal to $U_a(t)$ at the moment $t = t_f$, the solution $r_m(t)$ is $r_m(t) = r_m(t_f)e^{-c_m(t-t_f)}$. Thus, the nonnegativity of $r_m(t)$ can be ensured during $t \in [0, \infty)$. It implies that all states $r_j(t)$, $z_i(t)$ are all nonnegative from the same process in the proof of Lemmas 3,4 and Property 2 in Theorem 1. The safety, i.e., $z_1(t) = y_1(t) - s(t) \geq 0$, t > 0, is then proved.

The proof of this theorem is complete. \Box

5. Application in safe vehicle platooning

5.1. Physical model

In this section, we validate the effectiveness of the designed controller in the practical application of safe vehicle platooning by simulations. Considering the scenario described in Fig. 3: an electric vehicle E_0 is the leader of the vehicle string and is traveling at speed $v_0(t)$. The vehicle E_1 is running behind the vehicle E_0 at speed $v_1(t)$, and the third vehicle E_2 is traveling behind the vehicle E_1 at speed $v_2(t)$. The control task is to achieve a safe formation of these three vehicles while avoiding collisions, i.e., regulating the distances between adjacent vehicles to converge to the safe distances, denoted by d_{o1} , d_{o2} , but not to breach the safe distances all the time, i.e., $d_1(t) = l_0(t) - l_1(t) \ge d_{o1}$, $d_2(t) = l_1(t) - l_2(t) \ge d_{o2}$ and $d_1(t)$, $d_2(t)$ converge to d_{o1} , d_{o2} respectively, where $l_i(t)$ is the measurable displacement of the each vehicle E_i .

In *i*th electric vehicle, the control input is the voltage of the electric motor, shown in Fig. 3, whose dynamics are described by nonlinear ODEs:

$$F_{i} = \frac{k_{t}}{r} I_{i}, \quad \dot{I}_{i} = -\frac{R}{I} I_{i} - a I_{i}^{2} + \frac{1}{I} V_{i}, \tag{74}$$

where k_t is the torque constant of the DC motor, r is the length of moment arm, I_i is the motor current, R is the resistance of the motor, L is the inductance and \mathcal{V}_i is the input voltage. To reduce the modeling error between the mathematical model and the practical model, we introduce an unmodeled nonlinear term aI_i^2 to approximate the nonlinear elements in the drive circuit where the coefficient a is to be calibrated in practice by matching the mathematical and practical models. The output force F_i of the motor is transmitted to the wheel, generating the wheel drive force F_{wi} , through a set of transmissions, like a gearbox. There always exists a delay D_i , whose length is not easy to know exactly in advance, in such the transmission between the motor and the wheel, i.e.,

$$F_{wi}(t) = F_i(t - D_i). \tag{75}$$

Physical parameters in the vehicle platooning model.

Parameters(units)	Values
Linear damping coefficient: $f_1(\frac{N s}{m})$	5
Aerodynamic drag coefficient $f_2(\frac{N s^2}{m})$	0.25
Vehicle mass: M_{vi} (kg)	4
Target safe distance between E_0 and E_1 : d_{01} (m)	0.5
Target safe distance between E_1 and E_2 : d_{02} (m)	0.5
Transmission delay in E_1 : D_1 (s)	2.5
Transmission delay in E_2 : D_2 (s)	1.5
Torque constant of the DC motor: k_t (N m/A)	0.8
Length of moment arm: r (m)	0.1
Resistance of the motor: $R(\Omega)$	5
Inductance of motor drive circuit: L (H)	0.05
Calibrated coefficient of drive circuit nonlinearity: a	1

Based on the wheel drive forces F_{wi} , according to Ames et al. (2014), Bekiaris-Liberis (2024), the dynamic of ith vehicle is modeled as the following nonlinear ODE:

$$M_{vi}\frac{dv_i}{dt} = F_{wi} - f_1 v_i - f_2 v_i^2, (76)$$

where M_{vi} is the vehicle mass, and $f_2v_i^2$ describes the nonlinear damping force. The physical parameters used in the simulation are given in Table 1, where we choose delays that are greater than those generally encountered in practice to evaluate the controller under much challenging circumstances and showcase its performance.

5.2. Matching the physical model and the plant (1)–(4)

For ith vehicle, setting $y_{i1}(t) = -l_i(t)$, $y_{i2}(t) = -v_i(t)$, $x_{i1}(t) =$ F_i (i.e., $x_{i1}(t-D_i) = F_{wi}$), and $U_i = \frac{k_t}{r!} \mathcal{V}_i$, the physical model (74)–(76) is rewritten as

$$\dot{y}_{i1}(t) = y_{i2}(t), \tag{77}$$

$$\dot{y}_{i2}(t) = \frac{-f_1 y_{i2}(t) + f_2 y_{i2}(t)^2}{M_{vi}} - \frac{x_{i1}(t - D_i)}{M_{vi}},$$

$$\dot{x}_{i1}(t) = -\frac{R}{L} x_{i1} - a \frac{r}{k_t} x_{i1}^2 + U_i(t),$$
(78)

$$\dot{x}_{i1}(t) = -\frac{R}{L}x_{i1} - a\frac{r}{k}x_{i1}^2 + U_i(t),\tag{79}$$

which is covered by the considered plant (with the states $Y_i(t)$ = $[y_{i1}(t), y_{i2}(t)]^T, X_i(t) = x_{i1}(t)$, where i denotes ith considered plants). Considering the physical safety constraint $l_{i-1}(t) - l_i(t)$ $d_{oi} \ge 0$, we set the target trajectory of the *i*th vehicle as $s_i(t) =$ $-l_{i-1}(t)+d_{oi}$. We set that the initial position of the leader vehicle E_0 is $l_0(0) = 10$, with the speed given by $v_1(t) = 4 + \sin(t)$ and thus the target trajectory for E_1 is $s_1(t) = -4t + \cos(t) - 11 + d_{01}$. The target trajectory for vehicle E_2 is $s_2(t) = y_{11}(t) + d_{02}$. The unknown delays of vehicles E_1, E_2 are given in Table 1, with known bounds as D = 0.2, $\overline{D} = 4$. Assuming the initial values of vehicles' speed and distance between them as $v_1(0) = 1$, $v_2(0) = 2$, $d_1(0) = 5$, $d_2(0) = 5$, then the initial conditions of $Y_i(t)$ are given as $y_{11}(0) = -5$, $y_{12}(0) = -1$, $y_{21}(0) = 0$, $y_{22}(0) = -2$. Besides, we take $x_{11}(0) = 2$, $x_{21}(0) = 1$ with $x_{11}(t) \equiv 0$, $x_{21}(t) \equiv 0$ during the period $t \in [-D, 0)$. They satisfy Assumptions 4-6 regarding the initial conditions.

5.3. Controller

According to (34), the nominal controller for this three-order

$$U_i(t) = -c_{i1}x_{i1} - \varphi_{i1} + \frac{c_{i1}}{b}\Delta_i(t) + \frac{1}{b}\Delta_i^{(1)}(t), \tag{80}$$

where
$$\Delta_i(t) = -k_{i1}k_{i2}P_{i1} - (k_{i1} + k_{i2})(P_{i2} + \psi_{i1}(P_{i1})) + k_{i1}k_{i2}s_i(t + D_i) + (k_{i1} + k_{i2})s_i^{(1)}(t + D_i) + s_i^{(2)}(t + D_i) - \psi_{i1}^{(1)}(P_{i1}) - \psi_{i2}(\underline{P_{i2}}).$$

Remark 2. The target trajectory $s_i(t + D_i)$ used in (80) depends on the predicted state of the preceding vehicle E_{i-1} , i.e., P_{i-1} . In our simulation, recalling (17), $P_{i-1}(t)$ is computed on the ego vehicle E_{i-1} by the current state $Y_{i-1}(t)$, the historical predictions $P_{i-1}(t_P), t_P \in [t - D_{i-1}, t)$, and the virtual input $x_{i-1,1}(t)$, and then transmitted to the following vehicle E_{i+1} . Upon receiving the prediction information from vehicle E_{i-1} , vehicle E_i can simply calculate that $s_i(t + D_i) = P_{i-1}(t - D_{i-1} + D_i) + d_{oi}$. We only add a restriction $D_1 \ge D_2$ in this application when implementing the obtained theoretical results, considering that the controller of the vehicle E_2 requires the predicted value of the displacement y_{11} of the vehicle E_1 in a time horizon of D_2 , while the predictor time span in E_1 controller is D_1 .

Considering the unknown delay, according to (67), the safe delay-adaptive controller is derived from the nominal safe delaycompensated controller (80) based on the control design in Section 4. According to (45)-(48), (61), (62), the design parameters for the nominal or adaptive controller are chosen as $k_{11} = k_{21} =$ $3, k_{12} = k_{22} = 2, c_{11} = c_{12} = 2$. The initial value of the estimator (57) is defined as $\hat{D}_i(t_0) = 0.2$. The delay-adaptive controller $U_d(t)$ (60) is constructed by replacing the unknown delay with the estimate $\hat{D}(t_i)$ in (80), and the safety filter is built by choosing $\bar{c} = 2$ in (64), where it is required to seek the maximum or minimum of the signal $\mathcal{U}^*(t, \mathcal{D})$ with respect to the delay variable D. The implementation of seeking the maximum or minimum of $\mathcal{U}^*(t,\mathcal{D})$ and that of the delay estimator are described in the following two remarks.

Remark 3 (Seeking the maximum or minimum of $\mathcal{U}^*(t, \mathcal{D})$). We divide the known range [D, D] by the interval of d_D , i.e., each possible delay $\mathcal{D}(i) = \underline{D} + i d_D$, $i = 0, ..., N_D$, where the number N_D is a freely chosen positive integer (the larger N_D is accompanied with higher accuracy but larger computation source). The predictor values under all possible delays $\mathcal{D}(i)$, $i = 0, 1, ..., N_D$ should be computed for each moment before t_f . We set a N_D row data matrix to record the predictor values under all $\mathcal{D}(i)$. Taking the discretization step d, there are $\lceil \frac{\mathcal{D}(i)}{d} \rceil$ predictor values in the row corresponding to the delay $\mathcal{D}(i)$, in the process of computing the prediction for $\mathcal{D}(i)$ (where [x] is defined as the ceiling function: taking the least integer that is greater than or equal to x). By taking all the possible predictor values in (67) and selecting the maximum or minimum value as a control input, it is ensured that the current control input is in a subset of the safe control region of the nominal safe controller. After t_f , i.e. the time when the exact delay identification is achieved, only the values in the row corresponding to the certain delay $\mathcal{D}(i)$ that is closest to the output of the delay estimator are reserved, to continue calculating the predictor values and the control signal along this delay. Then there is no need to calculate the predicted values of other rows anymore after t_f , and the control input stays in the original safe control region of the nominal control.

Besides, when seeking the maximum or minimum of $\mathcal{U}^*(t, \mathcal{D})$ for vehicle E_2 , the predicted displacements of vehicle E_1 under all possible delays are required to join in this seeking process. Once the identification time t_f is reached, only the predictor value $P_1(t-\hat{D}_1(t_f)+\hat{D}_2(t_f))$ based on $\hat{D}_1(t_f)$ is used in the design of $U_a(t)$ (67).

Remark 4 (Implementation of the delay estimator). The delay estimate $\hat{D}(t)$ is constructed using the finite difference method to approximate the integration with respect to the space variable, which may cause some errors between the identified value and true value. The smaller space step dx and larger design parameter T can reduce this approximation error, but they also increase the computational burden and time. If the difference between

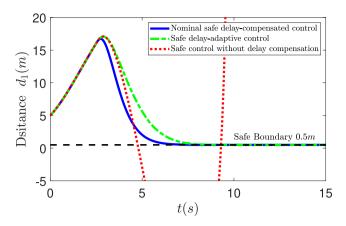


Fig. 4. Results for the distance between vehicle E_0 and E_1 , i.e., $d_1(t) = l_0(t) + y_{11}(t)$.

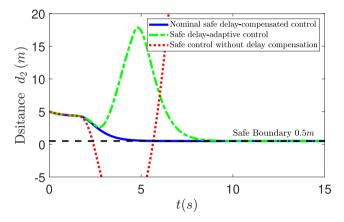


Fig. 5. Results for the distance between vehicle E_1 and E_2 , i.e., $d_2(t) = y_{21}(t) - y_{11}(t)$.

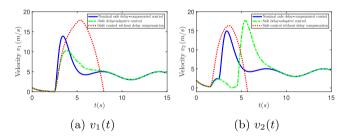


Fig. 6. Results for the velocities of followers E_i , i.e., $v_i = -y_{i2}$, i = 1, 2.

the estimates from the identifier at two adjacent updating times is smaller than 2% of the true value, we consider that the approximation error causes this difference, and thus, we keep the estimated value the same as it was at the previous updating time. Additionally, we set an upper limit of $\overline{n}=3$ for n in (57) to save computation time for estimation. Other design parameters in the estimator (57)–(59) are selected as $\tilde{N}=5$, T=3.

5.4. Simulation results

The simulation, including the implementation of the predictor (17) and identifier (57), is performed using the finite difference method with a time step $\mathrm{d}t=0.001$ and a space step $\mathrm{d}x=0.02$. As mentioned in Remark 3, the interval of $\mathcal{D}(i)$ is taken as $\mathrm{d}_D=0.01$. In addition to the nominal safe delay-compensated

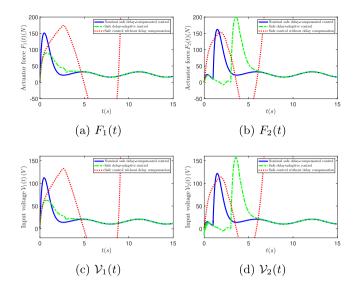


Fig. 7. Results for the output force of the actuator $F_i = x_{i1}(t)$ and the input voltage $v_i(t) = \frac{rtU_i(t)}{k}$.

controller and the safe delay-adaptive controller, as a comparison, we also apply a safe controller without delay compensation, i.e., replacing the predictor states in (80) with the states Y(t).

The simulation results are shown in Figs. 4-7, where the blue line represents the results under nominal safe delay-compensated control, the green dot-dashed line denotes the results under safe delay-adaptive control, and the red dotted line shows the results under safe control without delay compensation. The results regarding the output states $y_{i1}(t)$, i = 1, 2 of the ith plant considered in this paper, i.e., the distances $d_i = y_{i1} + l_{i-1}$ between vehicles E_i and E_{i-1} , i = 1, 2 in practice, are illustrated in Figs. 4, 5. We can see that the vehicle distances are convergent to the pre-set safe values d_{01} , d_{02} respectively, and never exceed the safe boundary in the entire control process under the nominal and adaptive controllers. For the safe controller without delay compensation, the distances between vehicles undergo large oscillations, breaching the safety constraint and ultimately diverging due to the effects of the delay and nonlinearity. Compared with the nominal control, even though the results under the safe delay-adaptive controller exhibit greater conservatism with respect to safety in the process of delay identification, they have similar behavior ultimately, after the effective estimate of the unknown is obtained. Due to the uncertainty of the predicted information of vehicle E_1 , which is required in the controller of E_2 , the adaptive results of E_2 exhibit greater conservatism with respect to safety than E_1 in the process of delay identification, as shown in Figs. 4, 5. We also know from Figs. 6(a), 6(b) that the velocities of follower vehicles E_1 , E_2 converge to the target speed (i.e., $v(t) = 4 + \sin t$) of the leader vehicle E_0 under adaptive and nominal control, while they diverge in the case without delay compensation. In Figs. 4-6, the results under the three controllers are identical before the delay time $t = D_i$ because the vehicle behaviors only depend on the initial data on $t \in [0, D)$ and there is no control action. The responses of the actuator output F_i $x_{i1}(t)$, i = 1, 2, are shown in Figs. 7(a), 7(b), where they converge to constant values under adaptive and nominal control. The input voltages (i.e., control signals) of the two vehicles are depicted in Figs. 7(c), 7(d) respectively. Additionally, Fig. 8 presents the delay estimates from the delay identifier, where the blue line shows the estimate $D_1(t)$ of the delay D_1 and the green dot-dashed line denotes the delay estimate $\hat{D}_2(t)$. Starting from the initial delay estimate $\hat{D}_i(t_0) = \underline{D} = 0.2$, the successful identification

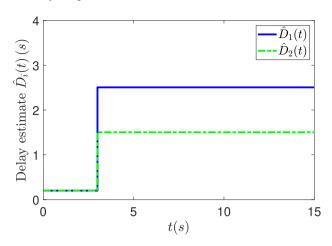


Fig. 8. Estimates of the unknown delays D_i , i = 1, 2 under the initial estimates $\hat{D}_i(t_0) = 0.2.$

of the unknown delay D_i is achieved at the first triggering time $t_f = 3$. Tiny differences exist between the delay estimates and the true values due to the errors in approximating integration as summation in the use of the finite difference method.

6. Conclusion and future work

In this paper, we design a safe delay-adaptive controller for a strict-feedback nonlinear system under a delayed nonlinear actuator, where the arbitrarily long delay D between two nonlinear subsystems is unknown, based on the safe predictor-based backstepping transformation and a QP safety filter with BaLSI. Finally, we achieve exponential regulation of system states with a safety guarantee. The effectiveness of our design is verified in the application of safe vehicle platooning, ensuring vehicle string stability with a small gap and avoiding collisions at a relatively high speed in the presence of unknown delays. Considering external disturbances, and measurement or prediction errors, which often occur in practice, improving the robustness of the adaptive safe controller will be dealt with in our future work. We will also try to apply neural operators to improve the proposed controller's realtime efficacy in implementation by approximating the nonlinear ODEs as an open-loop flow map in the predictor (Bhan, Shi, & Krstic, 2024; Krstic, 2023).

Appendix A. Proof of Proposition 1

A.1. Inverse transformation (37)–(39)

Considering $y_1(t) = z_1(t) + s(t)$ given by (11) at i = 1, we have $\psi_1(y_1(t)) = \psi_1(z_1(t) + s(t)) := \bar{\psi}_1(z_1(t), s(t))$ which is continuously differentiable and satisfies $\bar{\psi}_1(0,0)=\psi_1(0)=0$ according to Assumption 2. Considering (1) in the original system and (28) in the target system, recalling (11), for i = 1, we have

$$y_2(t) = \dot{y}_1(t) - \psi_1 = \dot{z}_1(t) + s^{(1)}(t) - \bar{\psi}_1(z_1(t), s(t))$$

= $z_2(t) + \bar{h}_1(z_1(t), s(t)) + s^{(1)}(t),$ (A.1)

where $\bar{h}_1(z_1(t), s(t)) = -k_1 z_1(t) - \bar{\psi}_1(z_1(t), s(t))$. From (A.1) and $y_1(t) = z_1(t) + s(t)$, we have $\psi_2(y_1(t)) := \overline{\psi}_2(\underline{z}_2(t), \underline{s}^{(1)}(t))$ by replacing y_1, y_2 with z_1, z_2 and $\underline{s}^{(1)}(t)$. It is obvious that $\bar{h}_1, \bar{\psi}_2$ is continuously differentiable and $\bar{h}_1(0,0)=0, \bar{\psi}_2(0,0)=0$ from (A.1), Assumption 2, and $\bar{\psi}_1(0,0) = 0$ shown above. Similarly, from (1), (28) at i = 2 and (A.1), one gets

$$y_{3}(t) = \dot{y}_{2}(t) - \psi_{2} = z_{3}(t) - k_{2}z_{2}(t) + s^{(2)}(t) - \bar{\psi}_{2}$$

$$+ \frac{\partial \bar{h}_{1}}{\partial z_{1}} \left(-k_{1}z_{1} + z_{2} \right) + \frac{\partial \bar{h}_{1}}{\partial s} s^{(1)}(t)$$

$$= z_{3}(t) + \bar{h}_{2}(\underline{z}_{2}(t), \underline{s}^{(1)}(t)) + s^{(2)}(t), \tag{A.2}$$

where $\bar{h}_2=-k_2z_2(t)-\bar{\psi}_2+\frac{\partial\bar{h}_1}{\partial z_1}\left(-k_1z_1+z_2\right)+\frac{\partial\bar{h}_1}{\partial s}s^{(1)}(t)$. We now prove the induction step: if all the inverse transfor-

mations from $z_i(t)$ to $y_i(t)$, j = 2, ..., i for $i \le n - 1$ are given as

$$y_j(t) = z_j(t) + \bar{h}_{j-1}(\underline{z}_{j-1}, \underline{s}^{(j-2)}) + s^{(j-1)}(t),$$
 (A.3)

where $\bar{h}_{j} = -k_{j}z_{j}(t) - \bar{\psi}_{j} + \sum_{k=1}^{j-1} \left(\frac{\partial \bar{h}_{j-1}}{\partial z_{k}} \left(-k_{k}z_{k} + z_{k+1}(t)\right) + \right)$ $\frac{\partial \bar{h}_{j-1}}{\partial s^{(k-1)}} s^{(k)}(t)$, $\bar{\psi}_j(\underline{z}_j(t),\underline{s}^{(j-1)}(t)) = \psi_j(\underline{y}_i(t))$, $j=1,\ldots,i-1$ and for all j, $\bar{\psi}_j$, \bar{h}_j are continuously differentiable function satisfying $\bar{h}_i(0,0) = 0$, $\bar{\psi}_i(0,0) = 0$, thus we have $y_{i+1}(t) = z_{i+1}(t) + \bar{h}_i + 1$ $s^{(i)}(t)$ with continuously differentiable function $\bar{h}_i, \ \bar{\psi}_i$ satisfying $\bar{h}_i(0,0) = 0, \, \bar{\psi}_i(0,0) = 0$ as well. The proof of the induction step is given as follows. Substituting the induction hypothesis (A.3) at i = i into the original system (1) at i + 1, recalling (28) in the target system, one obtains $y_{i+1}(t) = \dot{y}_i(t) - \psi_i(y_i) = z_{i+1}(t) +$ $\bar{h}_{i}(\underline{z}_{i}(t), \underline{s}^{(i-1)}(t)) + s^{(i)}(t) \text{ where } \bar{h}_{i}(\underline{z}_{i}(t), \underline{s}^{(i-1)}(t)) = -k_{i}z_{i} - \bar{\psi}_{i} + \sum_{k=1}^{i-1} \left(\frac{\partial \bar{h}_{i-1}}{\partial z_{k}} \left(-k_{k}z_{k} + z_{k+1}\right) + \frac{\partial \bar{h}_{i-1}}{\partial s^{(k-1)}} s^{(k)}(t)\right), \text{ and } \bar{\psi}_{i}(\underline{z}_{i}(t), \underline{s}^{(i-1)}(t)) = \psi_{i}(\underline{y}_{i}(t)) \text{ by replacing } \underline{y}_{i}(t) \text{ in } \psi \text{ with } \underline{z}_{i}(t), \underline{s}^{(i-1)}(t) \text{ using the}$ relations from z_j to y_j , $\vec{j}' = 2, ..., i$, given by (A.3). The function $\bar{\psi}$ is continuously differentiable because all $\bar{h}_i, j=1,\ldots,i-1$ are continuously differentiable, and satisfy $\bar{\psi}_i(0,0) = \psi_i(0) = 0$ because of (A.3), $\bar{h}_i(0,0) = 0, j = 1, \dots, i-1$, and Assumption 2. It implies that \bar{h}_i is continuously differentiable and $\bar{h}_i(0,0)=0$. The proof of the induction step is complete.

Starting from the base cases $y_1(t) = z_1(t) + s(t)$, (A.1), (A.2), and applying the induction step proved above, the inverse transformation (37)–(39) is verified.

A.2. Inverse transformation (40)

Recalling (2), (7), we have

$$\dot{y}_n(t + Dx) = \psi_n(y_n(t + Dx)) + u(x, t). \tag{A.4}$$

Taking the time derivative of (37) at i = n, replacing the current states y_n, z_i by the predictor states p_n, δ_i , applying $\dot{p}_n(x, t) =$ $\psi_n(p_n(x,t)) + u(x,t)$ obtained from (A.4), we then have

$$\psi_{n}(\underline{p}_{n}(x,t)) + u(x,t) = \frac{\partial \delta_{n}(x,t)}{\partial t} + s^{(n)}(t+Dx)
+ \sum_{k=1}^{n-1} \left[\frac{\partial \bar{h}_{n-1}(\underline{\delta}_{n-1}(x,t),\underline{s}^{(n-2)}(t))}{\partial \delta_{k}(x,t)} \left(-k_{k}\delta_{k}(x,t) \right) \right]
+ \delta_{k+1}(x,t) + \frac{\partial \bar{h}_{n-1}}{\partial s^{(k-1)}} s^{(k)}(t+Dx) \right].$$
(A.5)

Recalling $\psi_n(p_n(x,t)) = \bar{\psi}_n(\underline{\delta}_n(x,t),\underline{s}^{(n-1)}(t+Dx))$, plugging $\frac{\partial \delta_n(x,t)}{\partial t} = -k_n \delta_n(x,t) + w(x,t)$ obtained from (29) into (A.5), one gets $u(x, t) = w(x, t) + \left(-k_n \delta_n(x, t) - \bar{\psi}_n + \sum_{k=1}^{n-1} \frac{1}{\partial \delta_k(x, t)} \left(-k_k \delta_k(x, t) + \delta_{k+1}(x, t)\right) + \frac{\partial \bar{h}_{n-1}}{\partial s^{(k-1)}} s^{(k)}(t + Dx)\right]\right) + s^{(n)}(t + Cx)$ $\bar{D}x$). Recalling the definition of \bar{h}_n in (39), the inverse of the transformation (23) is obtained as (40).

A.3. Inverse transformation (41)–(43)

According to (40), (10), (31), we have $x_1 = \frac{1}{b}r_1 + \frac{1}{b}\bar{h}_n(\delta(1,t), \underline{s}^{(n-1)}(t+D)) + \frac{1}{b}s^{(n)}(t+D)$, i.e., (41) at j=1 with (44). Through

the recursive process similar to Appendix A.1, recalling (3), (4), (32) and (33), as well as Assumption 2, the inverse of transformation (24)–(26) is obtained as (41)–(43), with continuously differentiable $\bar{\varphi}_i(r_i(t), \Delta^{(i-1)}(t)) = \varphi_i(x_i)$ satisfying $\bar{\varphi}_i(0, 0) = 0$.

Appendix B. The proof of Lemma 1

We construct the following Lyapunov function for the target system (28)–(33),

$$V(t) = \frac{1}{2} \sum_{i=1}^{n} z_i(t)^2 + \frac{\rho}{2} \sum_{i=1}^{m} r_i(t)^2 + \frac{1}{2} \sum_{i=0}^{m} \int_0^1 a_i e^x w_x^{(i)}(x, t)^2 dx,$$
(B.1)

where ρ , a_1 , ..., a_m are positive analysis parameters that will be determined later. According to (51), we have

$$\theta_1 \Omega(t) \le V(t) \le \theta_2 \Omega(t)$$
 (B.2)

for some positive constants θ_1 , θ_2 . Taking the time derivative of V(t) in (B.1), one obtains

$$\dot{V}(t) = -\sum_{i=1}^{n} k_{i} z_{i}(t)^{2} + \sum_{i=1}^{n-1} z_{i}(t) z_{i+1}(t) + w(0, t) z_{n}(t)
- \rho \sum_{i=1}^{m} c_{i} r_{i}(t)^{2} + \rho \sum_{i=1}^{m-1} r_{i}(t) r_{i+1}(t) + \frac{a_{0}e}{2D} w(1, t)^{2}
- \sum_{i=0}^{m} \frac{a_{i}}{2D} w_{x}^{(i)}(0, t)^{2} + \sum_{i=1}^{m} \frac{a_{i}e}{2D} w_{x}^{(i)}(1, t)^{2}
- \sum_{i=0}^{m} \frac{a_{i}}{2D} \int_{0}^{1} e^{x} w_{x}^{(i)}(x, t)^{2} dx,$$
(B.3)

where integration by parts has been used. Recalling (31)–(33), we conclude that there exist positive constants b_1, \ldots, b_m , determined by delay time D, design parameters c_1, \ldots, c_m , and analysis parameters a_1, \ldots, a_m , such that

$$\sum_{i=1}^{m} \frac{a_i e}{D} w_{\chi}^{(i)}(1, t)^2 = \sum_{i=1}^{m} a_i e D^{i-1} r_1^{(i)}(t)^2 \le \sum_{i=1}^{m} b_i r_i^2.$$
 (B.4)

Thus applying Young's inequality and inserting (B.4) into (B.3) yield that

$$\dot{V}(t) \leq -(k_1 - \frac{1}{2})z_1(t)^2 - \sum_{i=2}^n (k_i - 1)z_i(t)^2 \\
- \left[\rho(c_1 - \frac{1}{2}) - \frac{a_0e}{2D} - \frac{1}{2}b_1\right]r_1(t)^2 \\
- \sum_{i=2}^{m-1} [\rho(c_i - 1) - \frac{1}{2}b_i]r_i(t)^2 \\
- \left[\rho(c_m - \frac{1}{2}) - \frac{1}{2}b_m\right]r_m(t)^2 - (\frac{a_0}{2D} - \frac{1}{2})w(0, t)^2 \\
- \sum_{i=2}^m \frac{a_i}{2D} \int_0^1 e^x w_x^{(i)}(x, t)^2 dx - \sum_{i=2}^m \frac{a_i}{2D} w_x^{(i)}(0, t)^2. \tag{B.5}$$

Under the conditions of the design parameters $c_1, \ldots, c_m, k_1, \ldots, k_n$ in (45)–(48), and choosing the analysis parameters a_0, \ldots, a_m, ρ as

$$a_0 \ge D, \quad a_i > 0, \ i = 1, \dots, m$$
 (B.6)

$$\rho > \max \left\{ \frac{a_0 e}{3D} + \frac{1}{3} b_1, b_m, \frac{1}{2} b_j \right\} + 1, j = 2, \dots, m - 1$$
 (B.7)

we get

$$\dot{V}(t) \le -\sum_{i=1}^{n} z_{i}(t)^{2} - \sum_{i=1}^{m} r_{i}(t)^{2} - \sum_{i=0}^{m} \frac{a_{i}}{2D} \int_{0}^{1} e^{x} w_{x}^{(i)}(x, t)^{2} dx \le -\varrho V(t),$$
(B.8)

where $\varrho=\frac{1}{\theta_2}\min\left\{1,\frac{a_i}{2D}\right\}>0, i=0,\ldots,m$. Recalling (B.2), we thus have (50), where $\varUpsilon_{\Omega}=\frac{\theta_2}{\theta_1}$ and $\sigma_{\Omega}=\varrho$. The lemma is obtained.

Appendix C. The proof of Lemma 2

Applying Cauchy–Schwarz inequality for (35), it is obtained that $|\delta(x,t)|^2$, $\forall x \in [0,1]$, are exponentially convergent to zero from the exponential convergence to zero of $|Z(t)|^2$, $\|w(\cdot,t)\|^2$ in Lemma 1. Moreover, taking i order time derivatives of (35) at x=1, one obtains

$$\begin{split} &\delta_{t}^{(i)}(1,t) = e^{DA}Z^{(i)}(t) + D\int_{0}^{1} e^{DA(1-y)}Bw_{t}^{(i)}(y,t)\,\mathrm{d}y\\ &= e^{DA}\left(A^{i}Z(t) + \sum_{j=0}^{i-1}A^{i-1-j}Bw_{t}^{(j)}(0,t)\right) + \frac{1}{D^{(i-1)}}\\ &\times \left[\sum_{j=0}^{i-1}(DA)^{j}\left(Bw_{x}^{(i-1-j)}(1,t) - e^{DA}Bw_{x}^{(i-1-j)}(0,t)\right)\right.\\ &+ \int_{0}^{1}(DA)^{i}e^{DA(1-y)}Bw(y,t)\,\mathrm{d}y\right], \end{split} \tag{C.1}$$

where integration by parts and (30) have been used. According to (30)–(33), applying Cauchy–Schwarz inequality, we also have $w_x^{(i)}(1,t)^2+w_x^{(i)}(0,t)^2 \leq \varUpsilon_w\left(\|w_x^{(i+1)}(\cdot,t)\|^2+\varUpsilon_{i+1}(t)^2\right), \ i=1,\ldots,m-1$ for some positive \varUpsilon_w . Then applying (C.1), and recalling the exponential convergence to zero of $\sum_{i=0}^m\|w_x^{(i)}(\cdot,t)\|^2$, $|Z(t)|^2$, $|R(t)|^2$ in Lemma 1, and that of $|\delta(x,t)|^2$, $\forall x\in[0,1]$ obtained above, we have that $|\delta_t^{(i)}(1,t)|^2$, $i=0,\ldots,m$ are exponentially convergent to zero. The lemma is then obtained.

References

Abel, I., Steeves, D., Krstic, M., & Jankovic, M. (2023). Prescribed-time safety design for strict-feedback nonlinear systems. *IEEE Transactions on Automatic Control*, 69(3), 1464–1479.

Ames, A. D., Coogan, S., Egerstedt, M., Notomista, G., Sreenath, K., & Tabuada, P. (2019). Control barrier functions: Theory and applications. In 18th European control conference (pp. 3420-3431). IEEE.

Ames, A. D., Grizzle, J. W., & Tabuada, P. (2014). Control barrier function based quadratic programs with application to adaptive cruise control. In *53rd IEEE conference on decision and control* (pp. 6271–6278). IEEE.

Ames, A. D., Xu, X., Grizzle, J. W., & Tabuada, P. (2016). Control barrier function based quadratic programs for safety critical systems. *IEEE Transactions on Automatic Control*, 62(8), 3861–3876.

Artstein, Z. (1982). Linear systems with delayed controls: A reduction. *IEEE Transactions on Automatic Control*, 27(4), 869–879.

Axelsson, J. (2016). Safety in vehicle platooning: A systematic literature review. *IEEE Transactions on Intelligent Transportation Systems*, 18(5), 1033–1045.

Bekiaris-Liberis, N. (2024). Nonlinear predictor-feedback cooperative adaptive cruise control of vehicles with nonlinear dynamics and input delay. *International Journal of Robust and Nonlinear Control*, 34(10), 6683–6698.

Bekiaris-Liberis, N., Jankovic, M., & Krstic, M. (2013). Adaptive stabilization of LTI systems with distributed input delay. *International Journal of Adaptive Control and Signal Processing*, 27(1–2), 46–65.

Bekiaris-Liberis, N., & Krstic, M. (2010). Stabilization of linear strict-feedback systems with delayed integrators. *Automatica*, 46(11), 1902–1910.

Bekiaris-Liberis, N., & Krstic, M. (2011). Compensation of time-varying input and state delays for nonlinear systems. *Journal of Dynamic Systems, Measurement and Control*, 134(1), Article 011009.

Bekiaris-Liberis, N., & Krstic, M. (2012). Compensation of state-dependent input delay for nonlinear systems. *IEEE Transactions on Automatic Control*, 58(2), 275–289

- Bekiaris-Liberis, N., & Krstic, M. (2013). Nonlinear control under nonconstant delays. SIAM.
- Bhan, L., Shi, Y., & Krstic, M. (2024). Neural operators for bypassing gain and control computations in PDE backstepping. *IEEE Transactions on Automatic Control*, 69(8), 5310–5325.
- Bresch-Pietri, D., & Krstic, M. (2014). Delay-adaptive control for nonlinear systems. IEEE Transactions on Automatic Control, 59(5), 1203–1218.
- Cohen, M. H., & Belta, C. (2022). High order robust adaptive control barrier functions and exponentially stabilizing adaptive control lyapunov functions. In 2022 American control conference (pp. 2233–2238). IEEE.
- Diagne, M., Bekiaris-Liberis, N., & Krstic, M. (2017). Compensation of input delay that depends on delayed input. *Automatica*, 85, 362–373.
- Diagne, M., Bekiaris-Liberis, N., Otto, A., & Krstic, M. (2017). Control of transport PDE/nonlinear ODE cascades with state-dependent propagation speed. *IEEE Transactions on Automatic Control*, 62, 6278–6293.
- He, C. R., & Orosz, G. (2018). Safety guaranteed connected cruise control. In 21st international conference on intelligent transportation systems (pp. 549–554). IEEE.
- Jankovic, M. (2001). Control Lyapunov-razumikhin functions and robust stabilization of time delay systems. *IEEE Transactions on Automatic Control*, 46(7), 1048–1060.
- Jankovic, M. (2018). Control barrier functions for constrained control of linear systems with input delay. In 2018 American control conference (pp. 3316–3321). IEEE.
- Karafyllis, I., Kontorinaki, M., & Krstic, M. (2019). Adaptive control by regulationtriggered batch least squares. *IEEE Transactions on Automatic Control*, 65(7), 2842–2855.
- Karafyllis, I., & Krstic, M. (2017). Predictor feedback for delay systems: Implementations and approximations. Springer.
- Karafyllis, I., & Krstic, M. (2018). Adaptive certainty-equivalence control with regulation-triggered finite-time least-squares identification. *IEEE Transactions* on Automatic Control, 63(10), 3261–3275.
- Karafyllis, I., Krstic, M., & Chrysafi, K. (2019). Adaptive boundary control of constant-parameter reaction-diffusion PDEs using regulation-triggered finite-time identification. *Automatica*, 103, 166–179.
- Koga, S., Bresch-Pietri, D., & Krstic, M. (2020). Delay compensated control of the stefan problem and robustness to delay mismatch. *International Journal of Robust and Nonlinear Control*, 30(6), 2304–2334.
- Koga, S., & Krstic, M. (2023). Safe PDE backstepping QP control with high relative degree CBFs: Stefan model with actuator dynamics. *IEEE Transactions on Automatic Control*, 68(12), 7195–7208.
- Krstic, M. (2009a). Delay compensation for nonlinear, adaptive, and PDE systems. Springer.
- Krstic, M. (2009b). Input delay compensation for forward complete and strict-feedforward nonlinear systems. IEEE Transactions on Automatic Control, 55(2), 287–303
- Krstic, M. (2023). Machine learning: Bane or boon for control? In 62nd IEEE conference on decision and control. (pp. 5987–5987). IEEE.
- Krstic, M., & Bement, M. (2006). Nonovershooting control of strict-feedback nonlinear systems. *IEEE Transactions on Automatic Control*, 51(12), 1938–1943.
- Krstic, M., & Bresch-Pietri, D. (2009). Delay-adaptive full-state predictor feedback for systems with unknown long actuator delay. In 2009 American control conference (pp. 4500–4505). IEEE.
- Krstic, M., & Smyshlyaev, A. (2008). Backstepping boundary control for first-order hyperbolic PDEs and application to systems with actuator and sensor delays. Systems & Control Letters, 57(9), 750–758.
- Lhachemi, H., Prieur, C., & Trelat, E. (2020). PI regulation of a reaction-diffusion equation with delayed boundary control. *IEEE Transactions on Automatic* Control, 66(4), 1573–1587.
- Li, W., & Krstic, M. (2020). Mean-nonovershooting control of stochastic nonlinear systems. *IEEE Transactions on Automatic Control*, 66(12), 5756–5771.
- Lopez, B. T., & Slotine, J.-J. E. (2023). Unmatched control barrier functions: Certainty equivalence adaptive safety. In 2023 American control conference (pp. 3662–3668). IEEE.
- Lopez, B. T., Slotine, J.-J. E., & How, J. P. (2020). Robust adaptive control barrier functions: An adaptive and data-driven approach to safety. *IEEE Control Systems Letters*, 5(3), 1031–1036.
- Mazenc, F., & Bliman, P.-A. (2006). Backstepping design for time-delay nonlinear systems. *IEEE Transactions on Automatic Control*, 51(1), 149–154.
- Mazenc, F., Malisoff, M., & Lin, Z. (2008). Further results on input-to-state stability for nonlinear systems with delayed feedbacks. *Automatica*, 44(9), 2415–2421.
- Mazenc, F., Mondie, S., & Francisco, R. (2004). Global asymptotic stabilization of feedforward systems with delay in the input. *IEEE Transactions on Automatic Control*, 49(5), 844–850.
- Molnar, T. G., Kiss, A. K., Ames, A. D., & Orosz, G. (2022). Safety-critical control with input delay in dynamic environment. *IEEE Transactions on Control Systems Technology*, 31(4), 1507–1520.

Molnar, T. G., Singletary, A. W., Orosz, G., & Ames, A. D. (2020). Safety-critical control of compartmental epidemiological models with measurement delays. *IEEE Control Systems Letters*, 5(5), 1537–1542.

- Nguyen, Q., & Sreenath, K. (2016). Exponential control barrier functions for enforcing high relative-degree safety-critical constraints. In 2016 American control conference (pp. 322–328). IEEE.
 Nubert, J., Kohler, J., Berenz, V., Allgower, F., & Trimpe, S. (2020). Safe and fast
- Nubert, J., Kohler, J., Berenz, V., Allgower, F., & Trimpe, S. (2020). Safe and fast tracking on a robot manipulator: Robust MPC and neural network control. *IEEE Robotics and Automation Letters*, 5(2), 3050–3057.
- Orosz, G., & Ames, A. D. (2019). Safety functionals for time delay systems. In 2019 American control conference (pp. 4374–4379). IEEE.
- Ploeg, J., Van De Wouw, N., & Nijmeijer, H. (2013). Lp string stability of cascaded systems: Application to vehicle platooning. *IEEE Transactions on Control Systems Technology*, 22(2), 786–793.
- Qi, J., Wang, S., Fang, J.-A., & Diagne, M. (2019). Control of multi-agent systems with input delay via PDE-based method. *Automatica*, 106, 91–100.
- Smith, O. J. (1959). A controller to overcome dead time. ISA, 6(2), 28-33.
- Taylor, A. J., & Ames, A. D. (2020). Adaptive safety with control barrier functions. In 2020 American control conference (pp. 1399–1405). IEEE.
- Wang, J., & Diagne, M. (2024). Delay-adaptive boundary control of coupled hyperbolic PDE-ODE cascade systems. *IEEE Transactions on Automatic Control*, 69(12), 8156–8171. http://dx.doi.org/10.1109/TAC.2024.3399629.
- Wang, S., Diagne, M., & Qi, J. (2021). Delay-adaptive predictor feedback control of reaction-advection-diffusion PDEs with a delayed distributed input. *IEEE Transactions on Automatic Control*, 67(7), 3762–3769.
- Wang, J., & Krstic, M. (2021). Regulation-triggered adaptive control of a hyperbolic PDE-ode model with boundary interconnections. *International Journal of Adaptive Control and Signal Processing*, 35(8), 1513–1543.
- Wang, J., & Krstic, M. (2022). Event-triggered adaptive control of coupled hyperbolic PDEs with piecewise-constant inputs and identification. *IEEE Transactions on Automatic Control*, 68(3), 1568–1583.
- Wang, J., & Krstic, M. (2024). Safe control of hyperbolic PDE-ODE cascades. In 2024 American control conference (pp. 2533–2538). IEEE.
- Wang, J., & Krstic, M. (2025). Output-positive adaptive control of hyperbolic PDE-ODE cascades. *IEEE Transactions on Automatic Control*, 1–8. http://dx.doi. org/10.1109/TAC.2025.3568557.
- Wang, J., Krstic, M., & Karafyllis, I. (2021). Adaptive regulation-triggered control of hyperbolic PDEs by batch least-squares. In 2021 American control conference (pp. 4057–4062). IEEE.
- Wang, S., Qi, J., & Diagne, M. (2021). Adaptive boundary control of reaction-diffusion PDEs with unknown input delay. *Automatica*, 134, Article 109909
- Wang, S., Qi, J., & Krstic, M. (2023). Delay-adaptive control of first-order hyperbolic PIDEs. arXiv preprint arXiv:2307.04212.
- Xiao, W., & Belta, C. (2021). High-order control barrier functions. *IEEE Transactions on Automatic Control*, 67(7), 3655–3662.
- Zhu, Y., & Krstic, M. (2020). Delay-adaptive linear control. Princeton University Press.
- Zhu, Y., Krstic, M., & Su, H. (2016). Adaptive output feedback control for uncertain linear time-delay systems. *IEEE Transactions on Automatic Control*, 62(2), 545–560.
- Zhu, Y., Krstic, M., & Su, H. (2020). Delay-adaptive control for linear systems with distributed input delays. *Automatica*, *116*, Article 108902.
- Zhu, Y., Su, H., & Krstic, M. (2015). Adaptive backstepping control of uncertain linear systems under unknown actuator delay. *Automatica*, 54, 256–265.



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