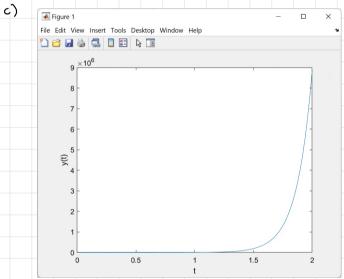
- Fall 2022 16-642 MEC Problem Set 1
- - a) so, inorder to check the system's stability, we have to find the eigenvalues of A. In this case, »eig ([010]) which veturns 7.6690 in matlab. -0.3345 + 0.13612 - 0.3345-0.1361c

Since there is an eigenvalue queator than 0, the system is unstable.

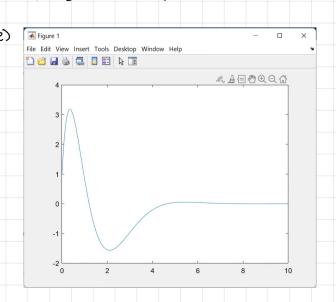
b) To see if the system is controllable, we have to check if [BIABI... Am B]'s rank is n.

$$\begin{bmatrix} 1 & \begin{bmatrix} 0 & 1 & 2 \end{bmatrix} \begin{bmatrix} 1 & 0 & 1 & 2 \end{bmatrix} \begin{bmatrix} 0 & 1 & 2 \end{bmatrix} \begin{bmatrix} 1 & 0 & 1 & 2 \end{bmatrix} \begin{bmatrix} 1 & 0 & 0$$

the system is controllable.



d) K= [11 60 88]



```
2)
   a) Tic-BÖcosO+BÖzsinO+Mic=F
                         α B - β π cos0 - Dsin0 = 0
                                I will put the equation above in a simple form using M = \begin{bmatrix} t & -\beta \cos \theta \\ -\beta \cos \theta & \alpha \end{bmatrix}
                                      \begin{bmatrix} r - \beta \cos \theta \\ -\beta \cos \theta \end{bmatrix} \begin{bmatrix} \ddot{x}_c \\ \ddot{\theta} \end{bmatrix} + \begin{bmatrix} \beta \dot{\theta}^2 \sin \theta + M \dot{x}_c \\ -D \sin \theta \end{bmatrix} = \begin{bmatrix} F \\ 0 \end{bmatrix}
                                          since in the question, it says a is invertible,
                                 \begin{bmatrix} \ddot{x}_c \\ \ddot{b} \end{bmatrix} = \begin{bmatrix} r - \beta \cos \theta \end{bmatrix}^{-1} \begin{pmatrix} \begin{bmatrix} F \\ D \end{bmatrix} - \begin{bmatrix} \beta \dot{\theta}^2 \sin \theta + M \dot{x}_c \\ -D \sin \theta \end{bmatrix}
                                                                      = \frac{1}{[\alpha - \beta^2(\cos\theta)^2]} \left[ \begin{array}{c} \alpha & \beta \cos\theta \\ \beta \cos\theta & \delta \end{array} \right] \left[ \begin{array}{c} F - \beta & \theta^2 \sin\theta - \mu \dot{x} = 0 \end{array} \right]
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                      Let X = \begin{bmatrix} A_1 \\ \theta \\ \dot{A}_2 \\ \dot{A}_4 \end{bmatrix} = \begin{bmatrix} A_1 \\ A_2 \\ A_4 \end{bmatrix}
                                                                       = \frac{1}{\left[ \left[ F d - d \beta \dot{\beta}^{2} \sin \theta - d M \dot{x}_{c} + p D \cos \theta \sin \theta \right] \right]}
\left[ F \beta \cos \theta - \beta^{2} \dot{\theta}^{2} \cos \theta \sin \theta - p M \dot{x}_{c} \cos \theta + P \sin \theta \right]
                                                                               \begin{bmatrix} \dot{x}_c \\ \dot{\theta} \\ \ddot{\alpha}_c \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} \dot{\alpha}_c \\ \dot{\theta} \\ \ddot{\alpha}_c \\ \ddot{\theta} \end{bmatrix}
\frac{\exists \dot{\alpha}_c \\ \dot{\theta} \\ \dot{\alpha}_c \\ \dot{\theta} \end{bmatrix} = \frac{\exists \dot{\alpha}_c \\ \dot{\theta} \\ \dot{\alpha}_c \\ \dot{\theta} \\ \dot{\theta} \end{bmatrix}
\frac{\exists \dot{\alpha}_c \\ \dot{\theta} \\ 
                   50, x= \xc =
                                                                                                                                                                                                                                                                                                                                                                                                                                                                ud - d B dasinds - am da + pD cosdasinda
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                TOY - BECOSAL)2
                                                                                                                                                                                                                                                                                                                                                                                                                                                       UB cos 1/2 - 821/2 cos/2 sin/2 - BN 1/2 cos/2 + Prsinx
                                                                                                                                                  \frac{F_{\beta}\cos\theta - \beta^{2}\theta^{2}\cos\theta\sin\theta - \beta M\dot{\chi}_{c}\cos\theta + \rho \sin\theta}{\Gamma\alpha - \beta^{2}(\cos\theta)^{2}}
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                          \Gamma OY - \beta^2 (COSA)^2
     b) To get the equilibrium points, we have to solve for \dot{x}=0 when F(=u)=0
                                            \frac{Fd-d\beta\dot{\theta}^2\sin\theta-dM\dot{x}_c+\beta D\cos\theta\sin\theta}{[C\alpha-\beta^2(\cos\theta)^2]}
                                                FB cost - p20 costsmo - pMic cost + Prsino
                                                                                       \Gamma Q - \beta^2 (\cos \theta)^2
 naturally, \dot{\alpha}_c = \dot{\theta} = 0 and \dot{\theta} = \partial_{\theta} \dot{\theta}^2 \sin \theta + \partial_{\theta} \dot{\alpha} \dot{\alpha}_c + \rho D \cos \theta \sin \theta
\dot{\theta} = \partial_{\theta} \cos \theta \sin \theta
                                                                                                                                                                           FOR - β<sup>2</sup>(cosθ)<sup>2</sup>

FOR - β<sup>2</sup>(cosθ)<sup>2</sup>
                               \frac{\rho D \cos \theta \sin \theta}{\delta \alpha - \rho^{2} (\cos \theta)^{2}} = 0 \Rightarrow \cos \theta \sin \theta = 0 \qquad \theta = \frac{\pi}{\nu} n \ (n = 0, 1, \dots)
                                                                                                            = 0 = sind =0 0= In (n=0,1,...)
                            TX - B2((050)2
                      Thus, \theta = \pi (n = 0,1,...) and no conditions for \pi (n = 0,1,...)
                         : equilibrium points
                                                                                                                                                             10 for 2001, ...
```

In english, this means that the system has equilibrium points when the pendulum is straight aboun or straight up. At this point, the change rate will be zero and the system will remain the

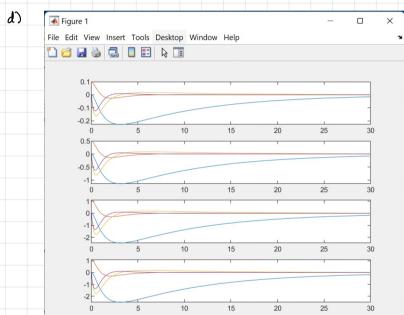
same cundanged) in those two positions.



C

If I use mat lab to calculate the eigenvalues of
$$A$$
, $\Rightarrow eig(ft) = \begin{bmatrix} 0 \\ -3.3901 \\ 1.1284 \end{bmatrix}$

Since for the system to be stable, we need to have Re(k_i)<0 YE. However we have k_3 = 1.1284. Thus the linear system is not stable at the eq. point. Automotically, the original non-linear system is not stable as well at the equilibrium point x=0.



Using mathab, k = [-0.3162 10.2723 -6.7857

Figure 1

File Edit View Insert Tools Desktop Window Help

On the state of the stat

As we are using k that we got trom linearizing the system at 2=0, we can only predict how the system will behave near 2=0. We can see that as a becomes larger than 0, the nonlinear system acts differently than the linearized system. Mostlab even throws this error for the 4th input (not stable)

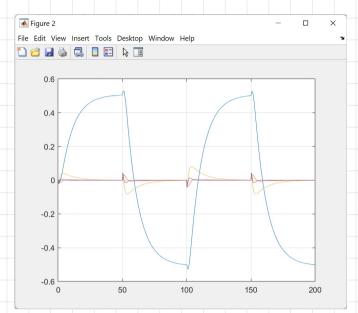
>> Q2_e
Warning: Failure at t=9.061899e+00. Unable to meet
integration tolerances without reducing the step size
below the smallest value allowed (2.842171e-14) at time
t.
> In ode45 (line 352)

> In <u>ode45</u> (<u>line 352</u> In <u>Q2_e</u> (<u>line 30</u>)

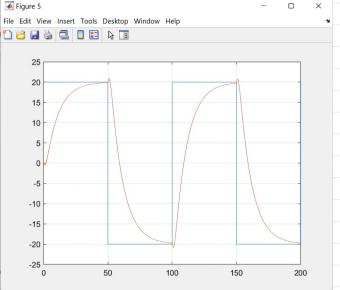
30

From this we can see that linearization of the system and what we can say about its stability as sonly valid for points very close to the equilibrium point used.

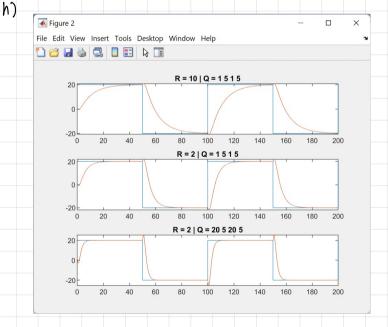
f) Since use only want the cart position in y, C will look something like $C = [c \circ o \circ]$ But since use want to consort meters into inches, $C = \frac{lmeter}{linch} = 39.3700787$ (according to google) So, $C = [39.3700787 \circ o \circ o]$



9)



We can see that there are greatly changing values for each states on 0,50,000,050. This is because the system is using the exponential function to track y_{des} . But overall, the y seems to follow y_{des} fairly well.



So, instead of the initial R=10 and

Q= [1000] I will allow the system to
0500 have a vigger input (u). Meaning
0010 I will reduce R to 2 to not
penalize as much. Then, I get a better tracting
system in the sense that it follows your more
accurately on the curves.

To make it even better, I increased the penalty
for xc, xic in Q so that It will track your faster

Now we use Q= [2000] and R=2.

In the third image, we can see that it follows your better than the first image.