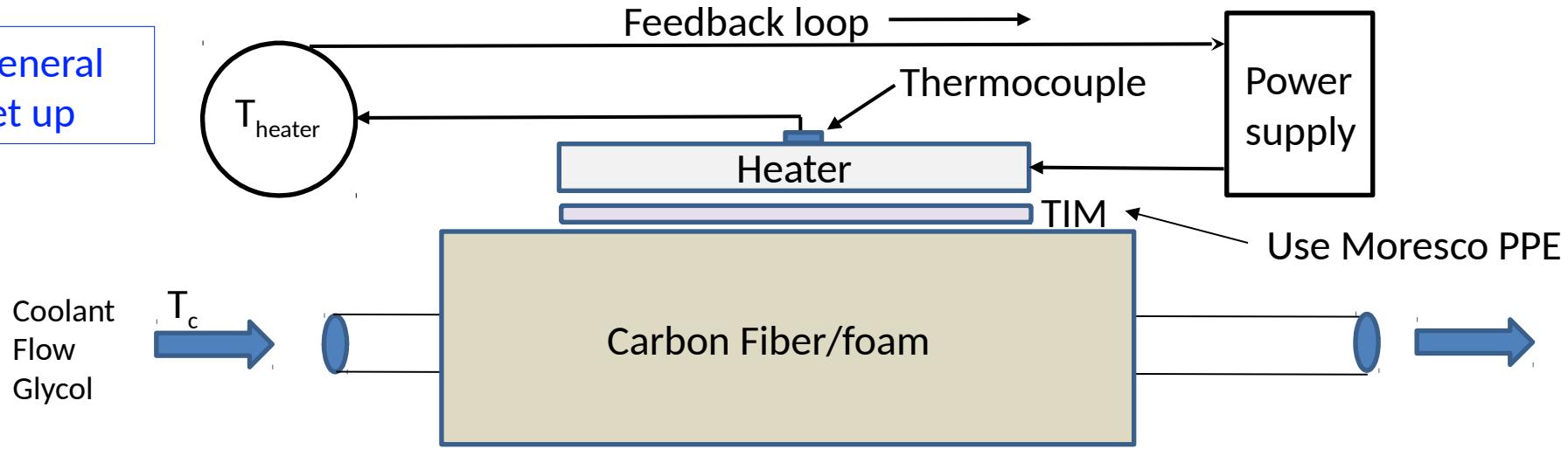


Strategy for runaway emulation

TFPIX Cornell U. team

Strategy for runaway emulation

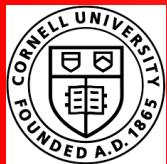
General set up



Specific goal

Measure the difference between the heater temperature (T_h) and coolant temperature (T_c) as a function of the coolant temperature, i.e.

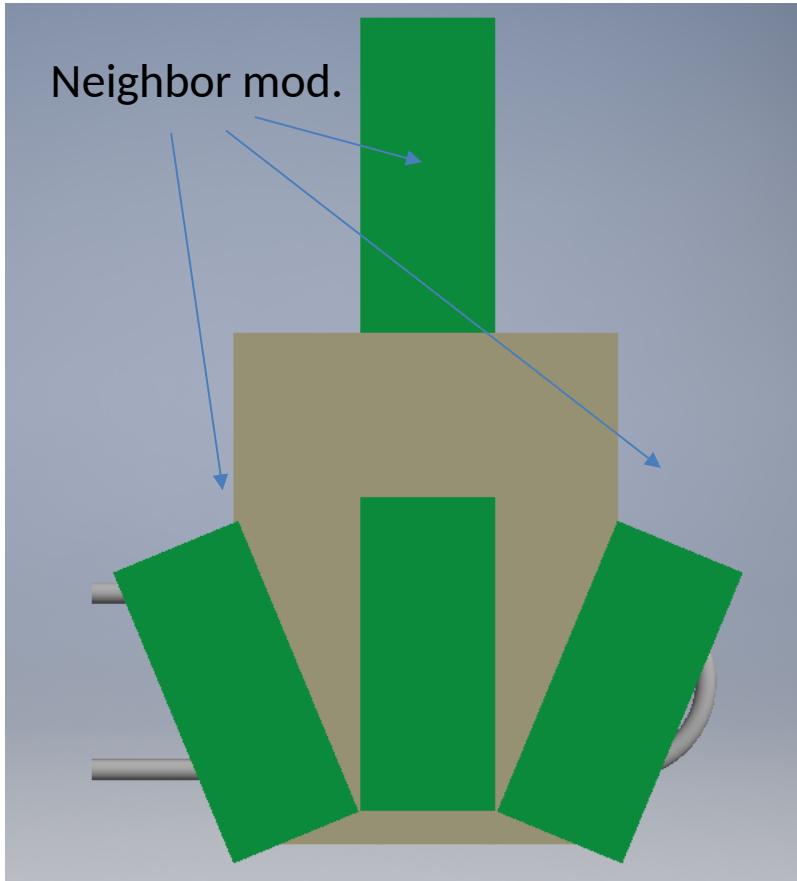
$$T_h - T_c \text{ vs } T_c$$



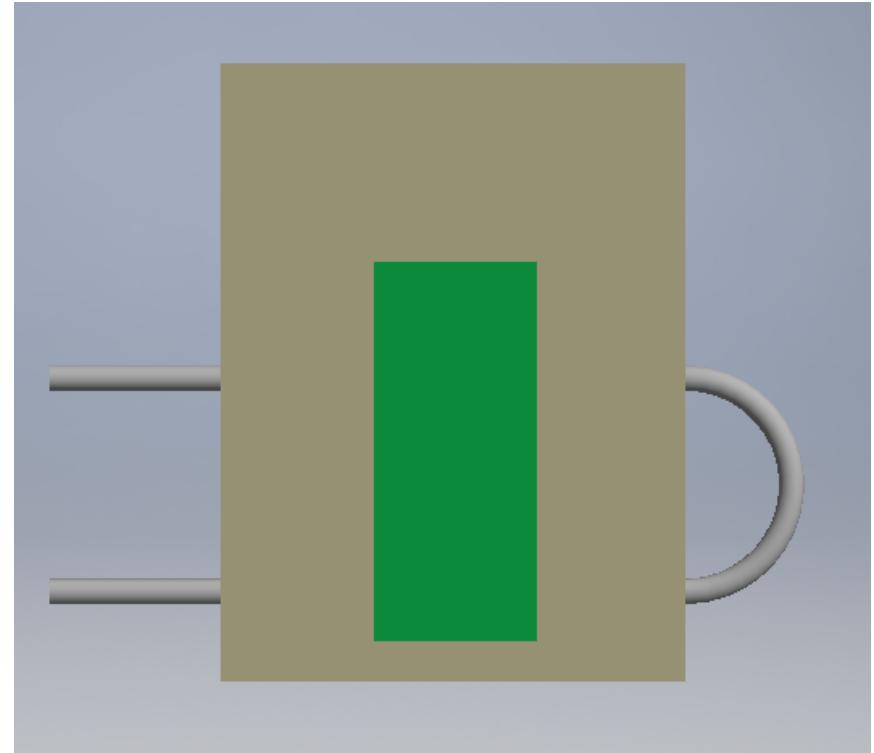
Unit cell sample (ucs)



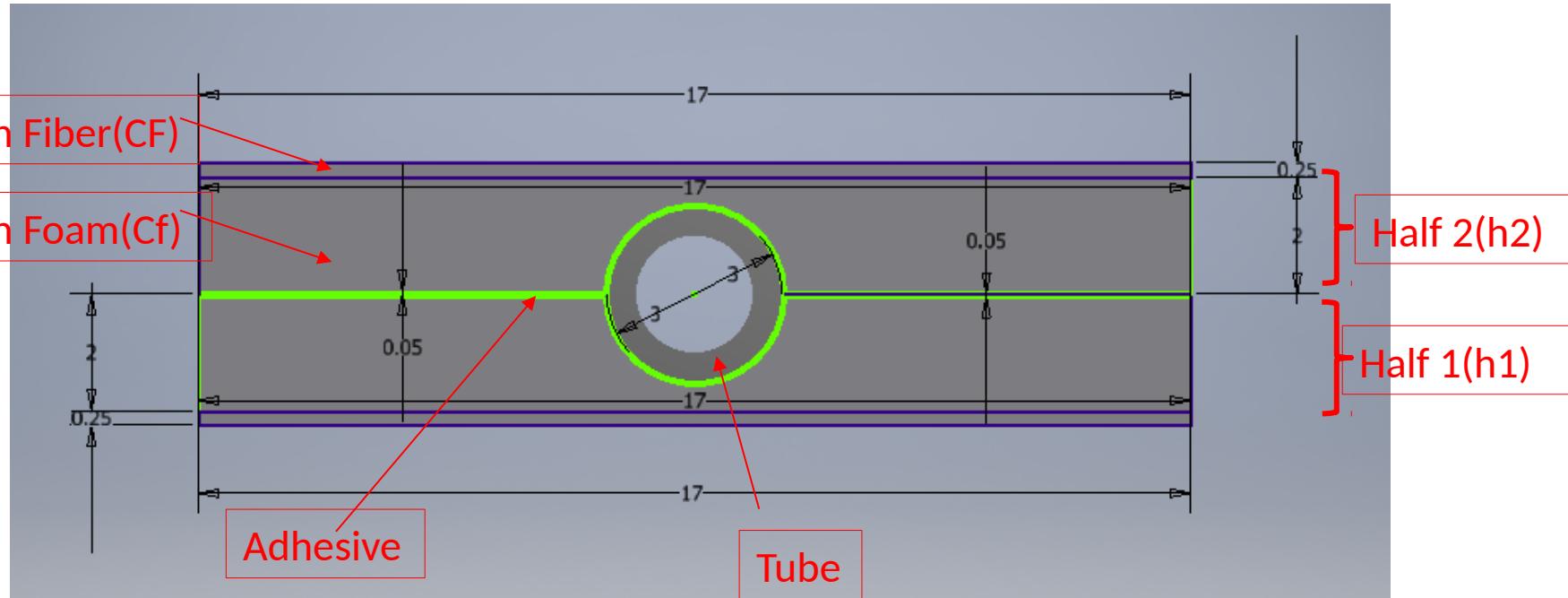
“One module space-region”



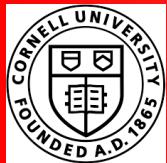
- Dimensions according to location of Neighbor modules, Cooling circuit.
- Gluing according to single tube testing



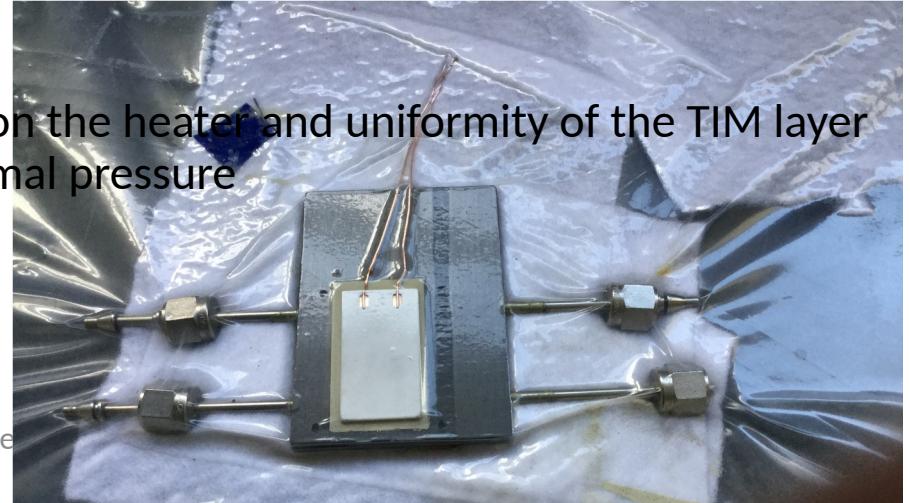
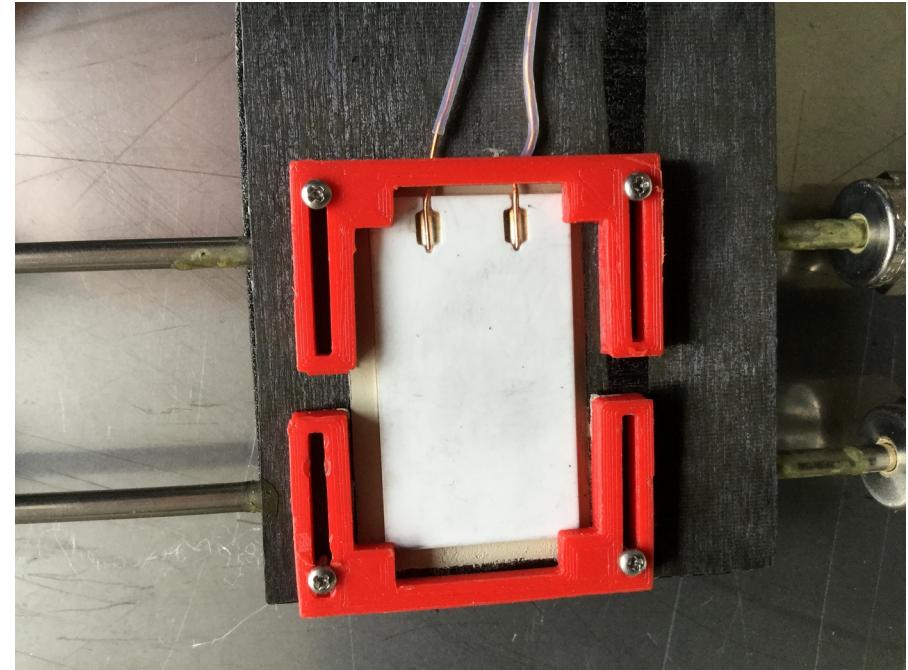
Single tube sample view



- Adhesive: Hysol EA 9396 (**Phase I BPix**), CVD loaded, Graphite loaded, others.
- Tubing: Stainless steel => OD=3mm, ID 2.5 mm



Sample preparation



- Vacuum -> aiming for uniform pressure on the heater and uniformity of the TIM layer
- Red clip-> hold in place the heater. Minimal pressure

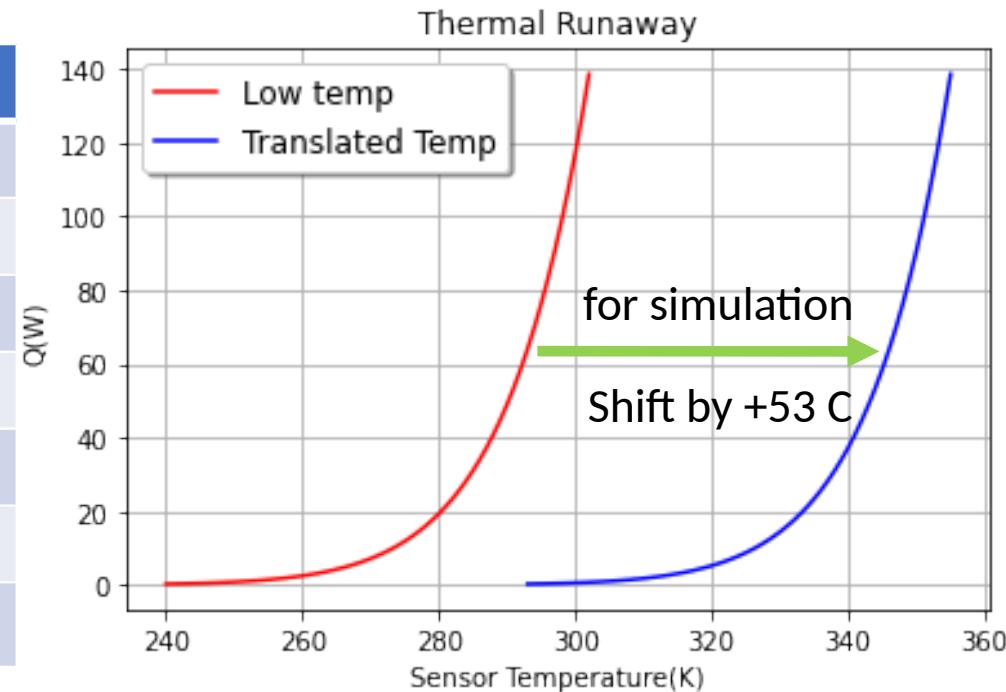
Thermal runaway(TRA) simulation

$$Q_{sens}(T) = I(T_{ref}) V_{bias} \left(\frac{T}{T_{ref}} \right)^2 e^{T_A \left(\frac{1}{T} - \frac{1}{T_{ref}} \right)}$$

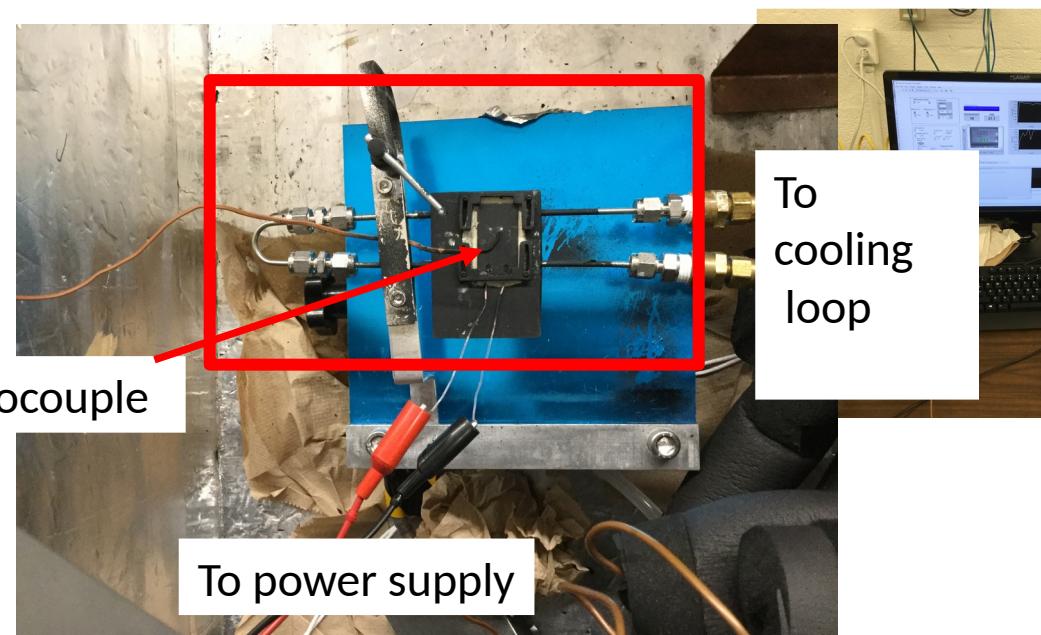
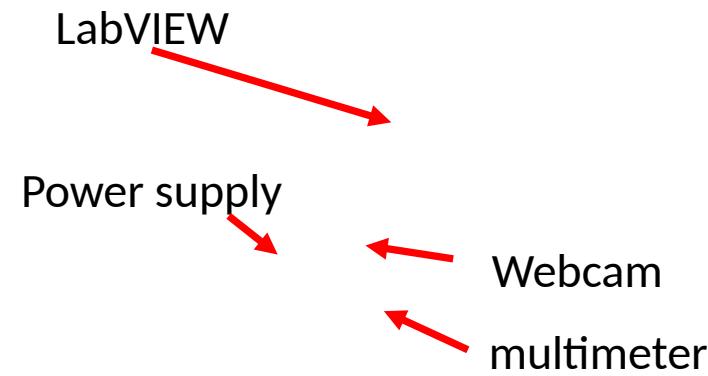
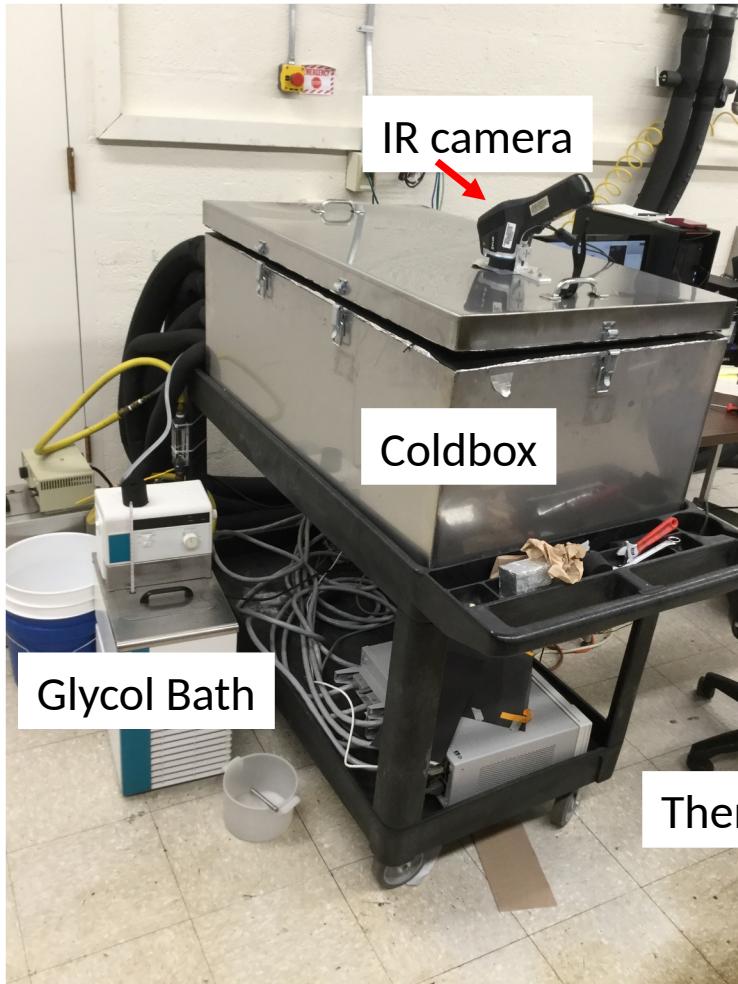
with

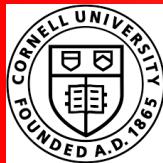
$$I(T_{ref} = 20^\circ C) = 4.28 \times 10^{-17} \left(\frac{A}{cm} \right) \times \Phi \left(cm^{-2} \right) \times V_{sens} \left(cm^3 \right)$$

Value	Parameter
$2.1 \times 10^{16} \text{ neq/cm}^2$	Fluence
$h=4.4 \text{ cm}$	length of the sensor
$l=1.8 \text{ cm}$	width of the sensor
$dd=100 \times 10^{-4} \text{ cm}$	depletion depth
$V_b=800 \text{ V}$	bias voltage
$T_0=293 \text{ C}$	T_{ref}
$T_a=7000 \text{ C}$	T_A



TRA data taking- setup





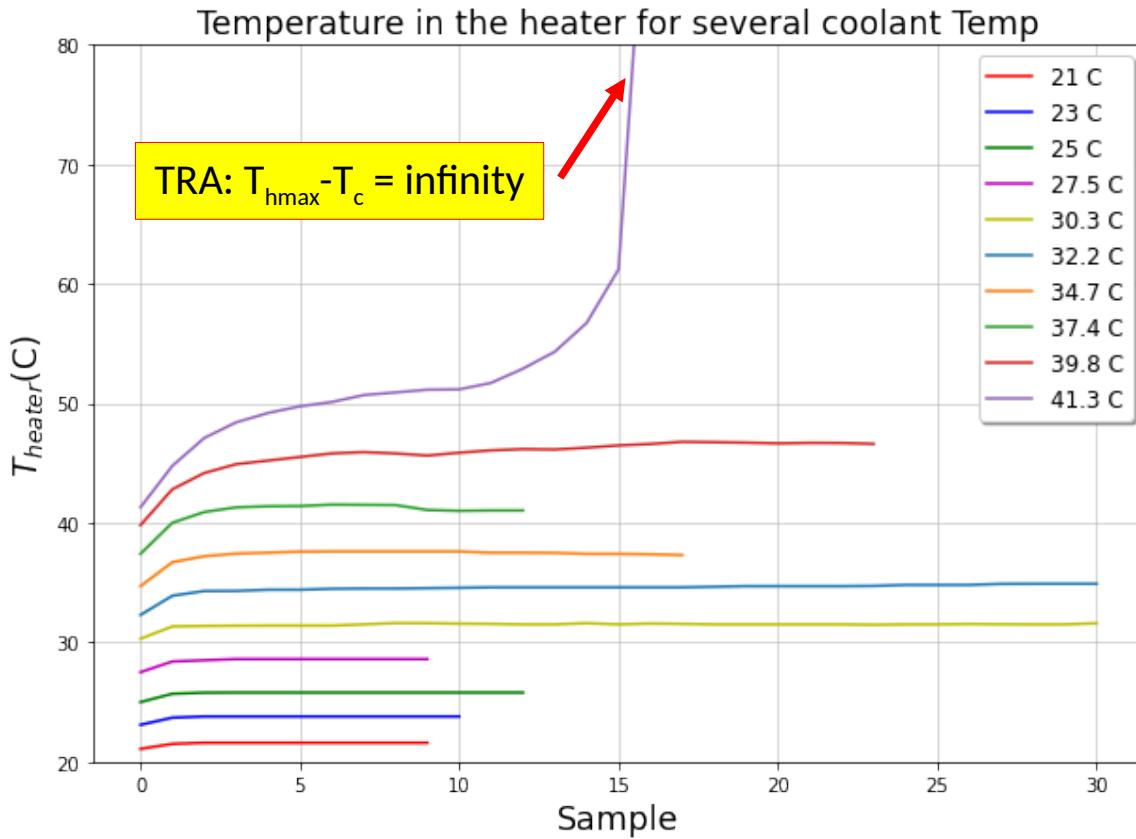
TRA data taking-procedure



- • Set coolant temperature T_c
 - Set $P=0$ W, wait 3 min. then read T_h (which should be = T_c).
 - Initiate the feedback loop to control power supply by setting $P(T_h)$.
-
- • Wait 3 minutes. Then read device temperature T_h .
 - Next iteration: Raise power level to $P(T_h)$. T_h will increase...
-
- Continue iterating until temperature plateaus ...or thermal runaway occurs

Reminder: testing is done at $20 \text{ degC} < T < 110 \text{ degC}$ while the effects we are trying to simulate actually will take place at around -30 degC. (the 53 degC shift would be reversed).

TRA data taking- Th vs Tc

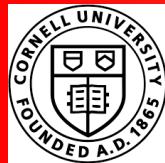


Increasing T_c would induce TRA* ; we are interested maximum change in temperature, i.e.

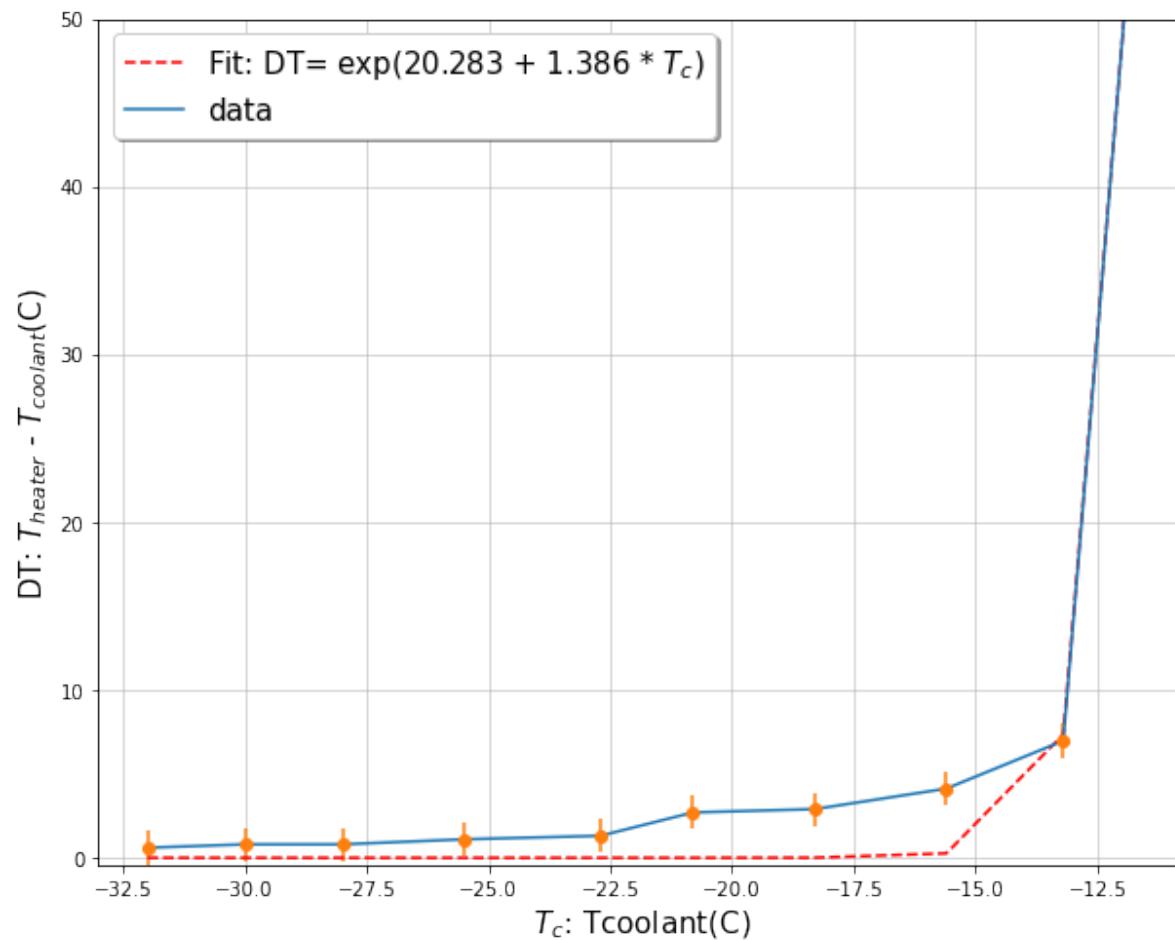
$$DT = T_{hmax} - T_c$$

If no TRA then $T_{hmax} = T_{plateau}$

* Increase in fluence and/or TIM performance will also induce TRA but we concentrate in one effect at a time.

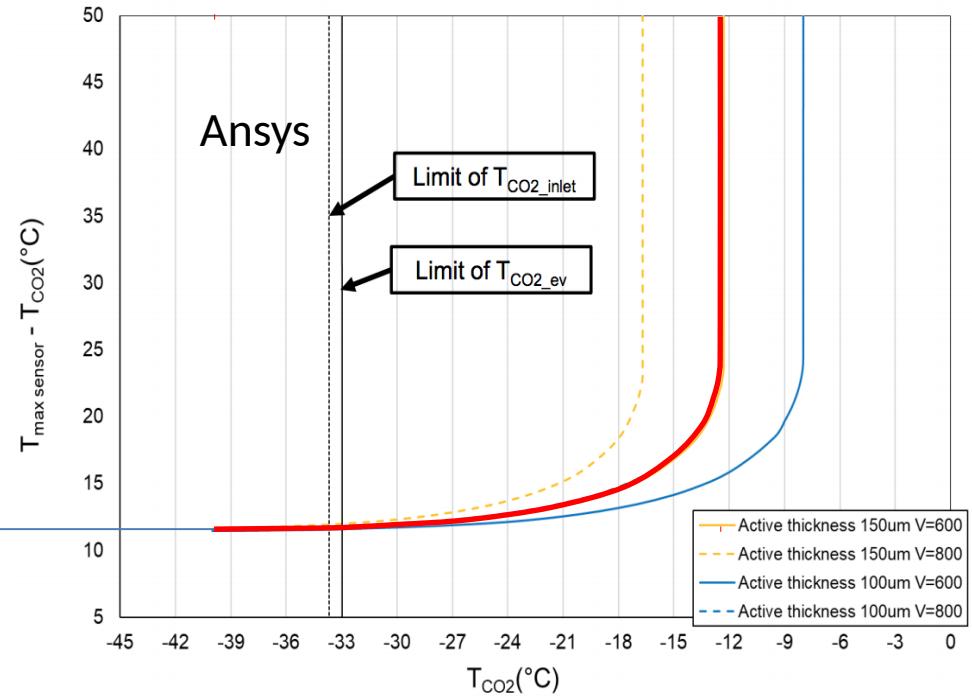
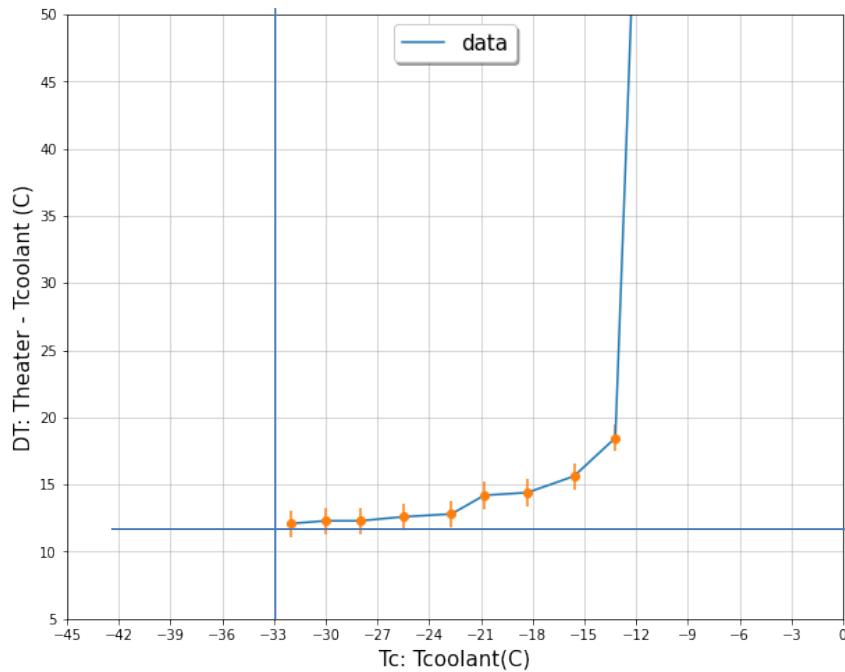


TRA: DT vs Tc



Note the Tc scale was shifted back to low temperature range.
Error bars corresponds to an estimate of 1 degC

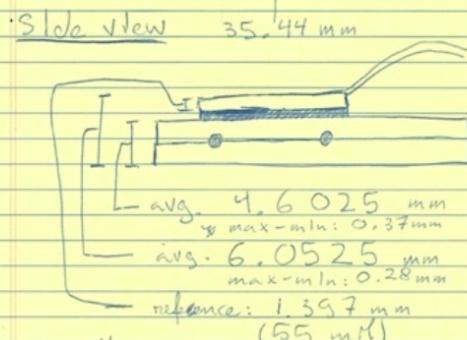
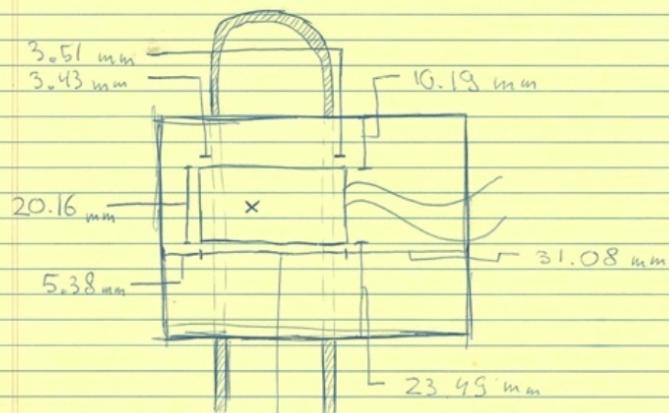
TRA Lab simulation vs Ansys



We are interested in the Ansys thermal simulation of our sample

Sample features and schematic I

Plaquette: top view



plaquette area:

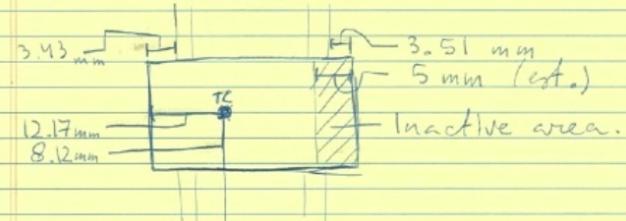
$$\frac{1}{2} (5.38 + 35.44 + 31.08) \times 23.49 = 71.30 \text{ mm}^2$$

$$10.19 + 20.16 + 23.49 = 53.84 \text{ mm}$$

measured total:	72.83 mm
	53.58 mm

(1)

Heaters: top view



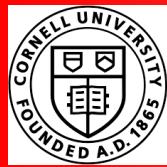
Thermocouple

$$\text{Heater area: } 35.44 \text{ mm} \times 20.16 \text{ mm} = 714.97 \text{ mm}^2$$

However, the bottom part of the heater does not contribute to the heating. Based on ~~the~~ thermal images of the heater at various power settings, this inactive area was estimated to be 0.5 mm wide.

The area used for the 'active heater' is $20 \text{ mm} \times 30 \text{ mm} = 600 \text{ mm}^2$.

One might actually want to discount this strips along the other edges; however, I do not think this necessary.



Sample features and schematic II

