

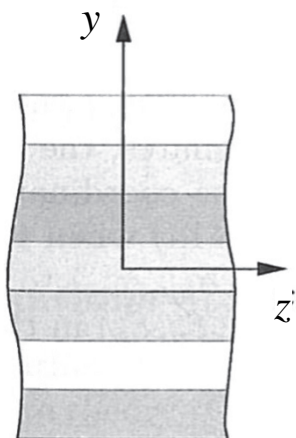
Advanced Bending and Torsion Composite Beams

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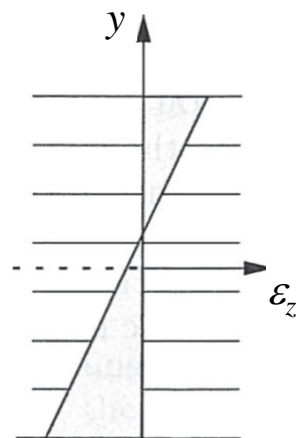
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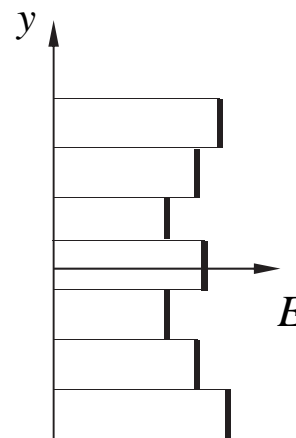
- Beams made of two or more different materials (with different Young's moduli)
- Assumptions:
 - 'Plane sections remain plane', *i.e.* linear strain distribution through the thickness (Euler-Bernoulli beams)
 - However, different moduli will cause stress discontinuities or 'jumps'



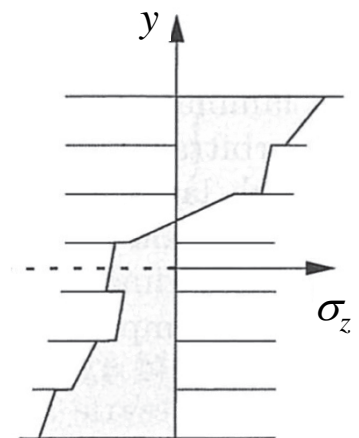
**composite
beam**



strains



moduli



stresses

- Method:

1. Choose one Young’s modulus to be used as reference: E_{ref}
2. Find the area scaling factors n_i for each material section i : $n_i = \frac{E_i}{E_{\text{ref}}}$
3. Conduct the analysis as before, utilising ‘effective’ (scaled) properties:

$$A_i^{\text{eff}} = A_i n_i$$

$$I_i^{\text{eff}} = I_i n_i$$

4. Finally, calculate global deflections (and hence strains) based on E_{ref} alone

- Note:

- Only the ‘local width’ variable (usually denoted b_i) is scaled - limits of integration and centroids **remain unchanged**

$$A_i^{\text{eff}} = \underbrace{(n_i b_i)}_{\text{scaled width}} \int_{-\frac{h_i}{2}}^{+\frac{h_i}{2}} dy$$

original height

$$I_i^{\text{eff}} = \underbrace{(n_i b_i)}_{\text{scaled width}} \int_{-\frac{h_i}{2}}^{+\frac{h_i}{2}} y^2 dy$$

original height

- We now make the flange (section B) of steel with $E = 210$ GPa

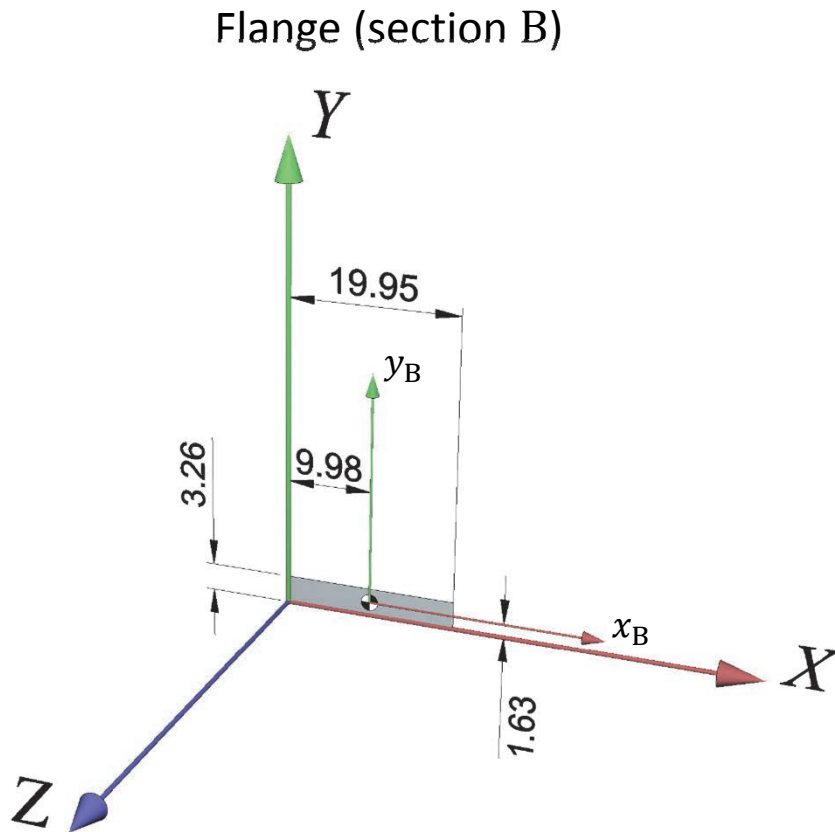
$$E_{\text{ref}} = 70 \text{ GPa}$$

$$n_B = \frac{210 \text{ GPa}}{70 \text{ GPa}} = 3$$

$$A_B = n_B(19.95)(3.26) = 195.11 \text{ mm}^2$$

$$\bar{X}_B = 9.98 \text{ mm}$$

$$\bar{Y}_B = 1.63 \text{ mm}$$



- We can now find the centroid of the compound section:

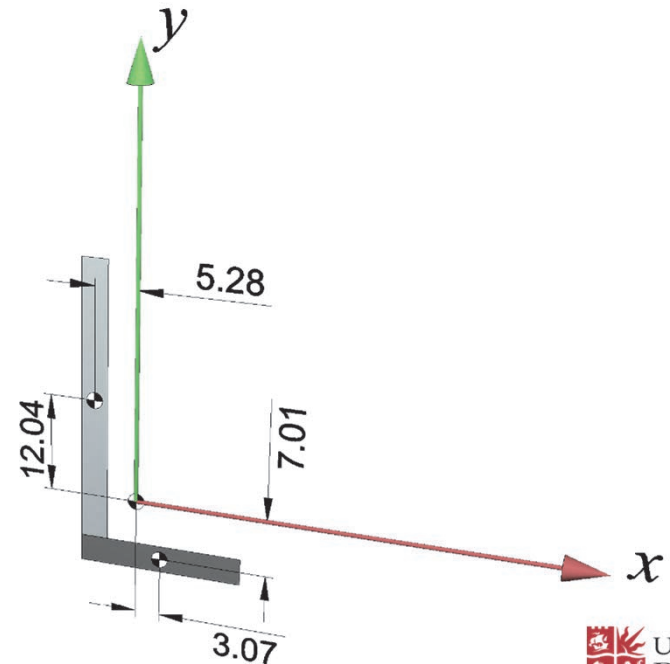
$$\bar{X} = \frac{\sum \bar{X}_i A_i}{\sum A_i} = \frac{\bar{X}_A A_A + \bar{X}_B A_B}{A_A + A_B} = \frac{(1.63)(113.58) + (9.98)(195.11)}{(113.58) + (195.11)}$$

$$\bar{X} = 6.91 \text{ mm}$$

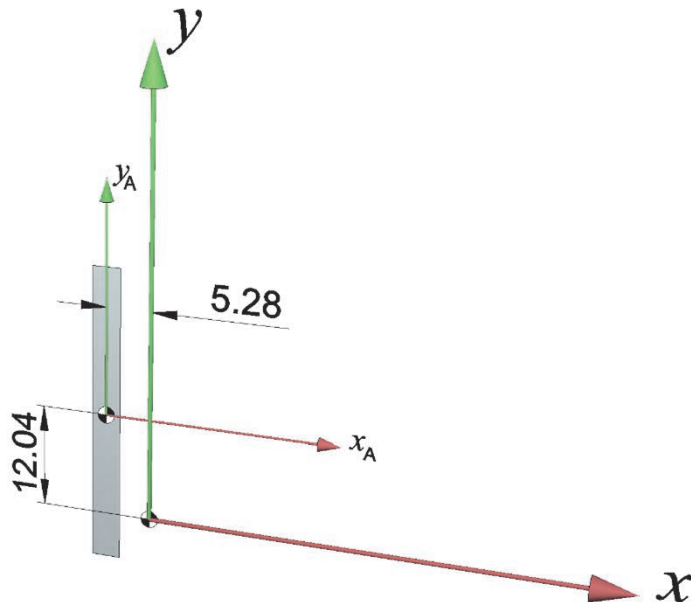
$$\bar{Y} = \frac{\sum \bar{Y}_i A_i}{\sum A_i} = \frac{\bar{Y}_A A_A + \bar{Y}_B A_B}{A_A + A_B} = \frac{(20.68)(113.58) + (1.63)(195.11)}{(113.58) + (195.11)}$$

$$\bar{Y} = 8.64 \text{ mm}$$

- Plotting on the cross-section:



- We now place the origin of (x, y) at the compound centroid and apply the parallel axes theorem for section A:



$$I_{x_A x_A} = \frac{b h^3}{12} = \frac{(3.26)(34.84)^3}{12} = 11,488.70 \text{ mm}^4$$

$$\bar{y}_A = \bar{Y}_A - \bar{Y} = 20.68 - 8.64 = 12.04 \text{ mm}$$

$$I_{xx}^A = I_{x_A x_A} + A_A (\bar{y}_A)^2$$

$$I_{xx}^A = (11,488.70) + (113.58)(12.04)^2$$

$$I_{xx}^A = 27,955.37 \text{ mm}^4$$

$$I_{y_A y_A} = \frac{b h^3}{12} = \frac{(34.84)(3.26)^3}{12} = 100.58 \text{ mm}^4$$

$$\bar{x}_A = \bar{X}_A - \bar{X} = 1.63 - 6.91 = -5.28 \text{ mm}$$

$$I_{yy}^A = I_{y_A y_A} + A_A (\bar{x}_A)^2$$

$$I_{yy}^A = (100.58) + (113.58)(-5.28)^2$$

$$I_{yy}^A = 3,264.24 \text{ mm}^4$$

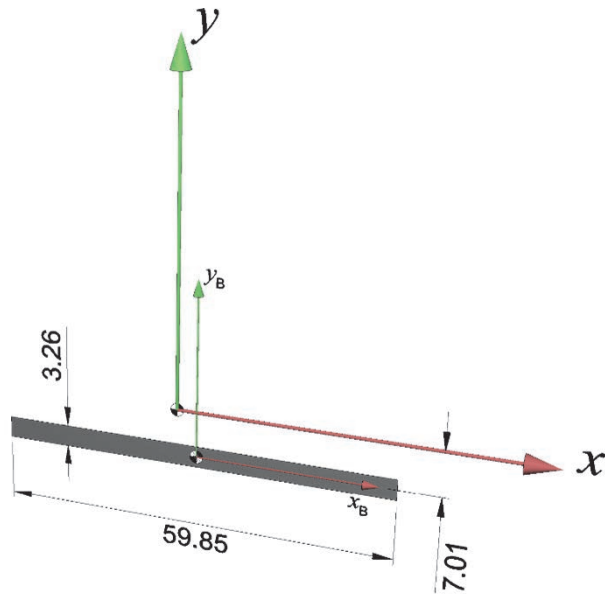
$$I_{x_A y_A} = 0 \text{ (symmetric cross-section)}$$

$$I_{xy}^A = I_{x_A y_A} + A_A (\bar{x}_A \bar{y}_A)$$

$$I_{xy}^A = 0 + (113.58)(-5.28)(12.04)$$

$$I_{xy}^A = -7,217.67 \text{ mm}^4$$

- Now applying the parallel axis theorem for section B:



$$I_{x_B x_B} = \frac{b h^3}{12} = \frac{(3)(19.95)(3.26)^3}{12} = 172.80 \text{ mm}^4$$

$$\bar{y}_B = \bar{Y}_B - \bar{Y} = 1.63 - 8.64 = -7.01 \text{ mm}$$

$$I_{xx}^B = I_{x_B x_B} + A_B (\bar{y}_B)^2$$

$$I_{xx}^B = (172.80) + (3)(65.04)(-7.01)^2$$

$$I_{xx}^B = 9,758.41 \text{ mm}^4$$

$$I_{y_B y_B} = \frac{b h^3}{12} = \frac{(3)(3.26)(19.95)^3}{12} = 6,471,22 \text{ mm}^4$$

$$\bar{x}_B = \bar{X}_B - \bar{X} = 9.98 - 6.91 = 3.07 \text{ mm}$$

$$I_{yy}^B = I_{y_B y_B} + A_B (\bar{x}_B)^2$$

$$I_{yy}^B = (6,471,22) + (3)(65.04)(3.07)^2$$

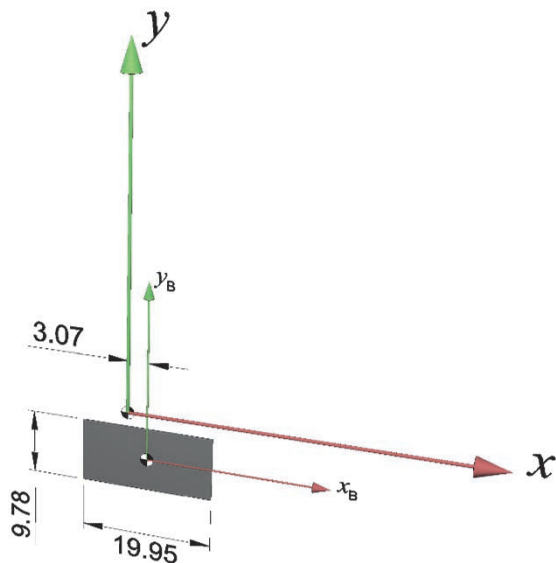
$$I_{yy}^B = 8,312.85 \text{ mm}^4$$

$$I_{x_B y_B} = 0 \text{ (symmetric cross-section)}$$

$$I_{xy}^B = I_{x_B y_B} + A_B (\bar{x}_B \bar{y}_B)$$

$$I_{xy}^B = 0 + (3)(65.04)(3.07)(-7.01)$$

$$I_{xy}^B = -4,201.57 \text{ mm}^4$$



- Finally, for the **compound composite** section:

$$I_{xx} = I_{xx}^A + I_{xx}^B$$

$$I_{xx} = (27,955.37) + (9,758.41)$$

$$I_{xx} = 37,713.78 \text{ mm}^4$$

$$I_{yy} = I_{yy}^A + I_{yy}^B$$

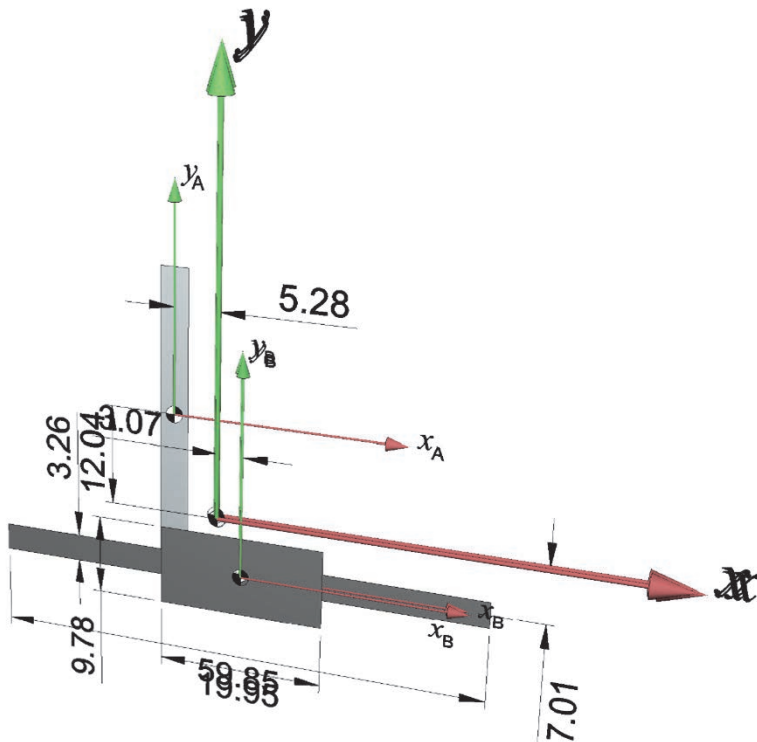
$$I_{yy} = (3,264.24) + (8,312.85)$$

$$I_{yy} = 11,577.10 \text{ mm}^4$$

$$I_{xy} = I_{xy}^A + I_{xy}^B$$

$$I_{xy} = (-7,217.67) + (-4,201.57)$$

$$I_{xy} = -11,419.24 \text{ mm}^4$$

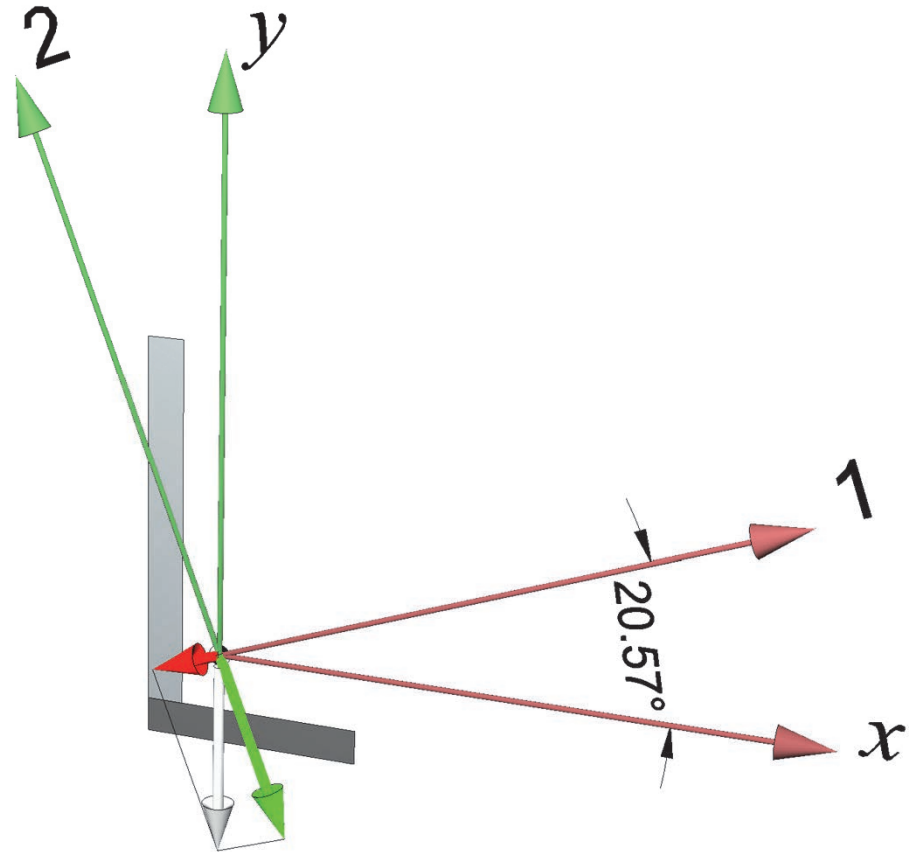


- We can now find the **principal axes**:

$$\theta_p = \frac{1}{2} \arctan \left(\frac{2 I_{xy}}{I_{yy} - I_{xx}} \right)$$

$$\theta_p = \frac{1}{2} \arctan \left[\frac{2 (-11,419.24)}{(11,577.10) - (37,713.78)} \right]$$

$$\theta_p = 20.57^\circ$$



- And the **principal 2nd moments of area** are:

$$\begin{Bmatrix} I_{11} \\ I_{22} \\ I_{12} \end{Bmatrix} = \begin{bmatrix} m^2 & n^2 & -2 m n \\ n^2 & m^2 & 2 m n \\ m n & -m n & m^2 - n^2 \end{bmatrix} \begin{Bmatrix} I_{xx} \\ I_{yy} \\ I_{xy} \end{Bmatrix}$$

- where: $m = \cos \theta_p = 0.936$
 $n = \sin \theta_p = 0.351$

- Using an Excel spreadsheet (see Blackboard) we get:

$$\begin{Bmatrix} I_{11} \\ I_{22} \\ I_{12} \end{Bmatrix} = \begin{Bmatrix} 41,999.99 \\ 7,290.89 \\ \sim 0 \end{Bmatrix} \text{mm}^4$$

- Now we can project the applied load onto our principal axes:

$$\begin{Bmatrix} P_1 \\ P_2 \end{Bmatrix} = \begin{bmatrix} m & n \\ -n & m \end{bmatrix} \begin{Bmatrix} P_x \\ P_y \end{Bmatrix}$$

$$\begin{Bmatrix} P_1 \\ P_2 \end{Bmatrix} = \begin{bmatrix} 0.962 & 0.272 \\ -0.272 & 0.962 \end{bmatrix} \begin{Bmatrix} 0 \\ -19.62 \end{Bmatrix} \text{ N}$$

$$\begin{Bmatrix} P_1 \\ P_2 \end{Bmatrix} = \begin{Bmatrix} -6.90 \\ -18.37 \end{Bmatrix} \text{ N}$$

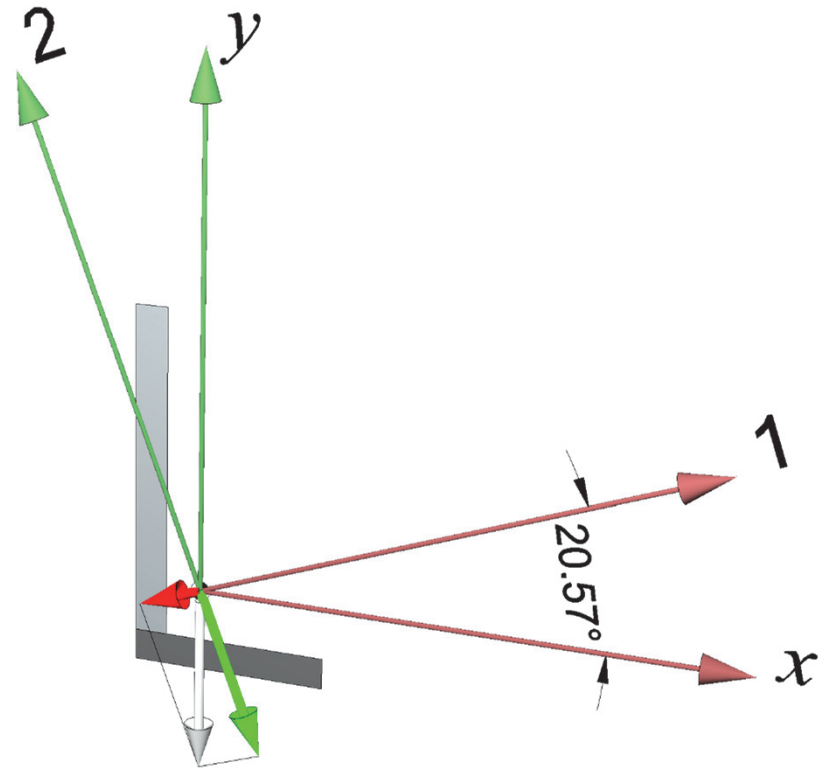
- Applying the tip deflection formula:

$$\delta = \frac{P L^3}{3 E I}$$

$$\delta_1 = \frac{(-6.90)(1000)^3}{3 (70,000)(7,290.88)} \quad \delta_2 = \frac{(-18.37)(1000)^3}{3 (70,000)(41,999.99)}$$

$$\delta_1 = -4.50 \text{ mm}$$

$$\delta_2 = -2.08 \text{ mm}$$

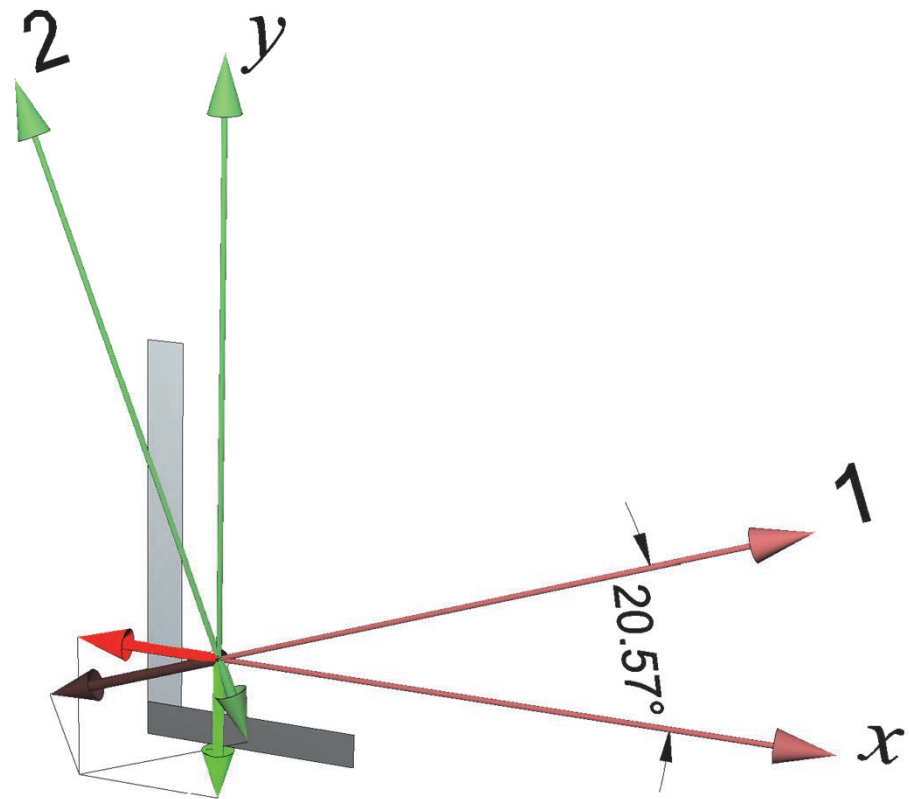


- Finally, transform deflections from principal axes (1,2) back to our reference axes (x, y) through rotation by $(-\theta_p)$:

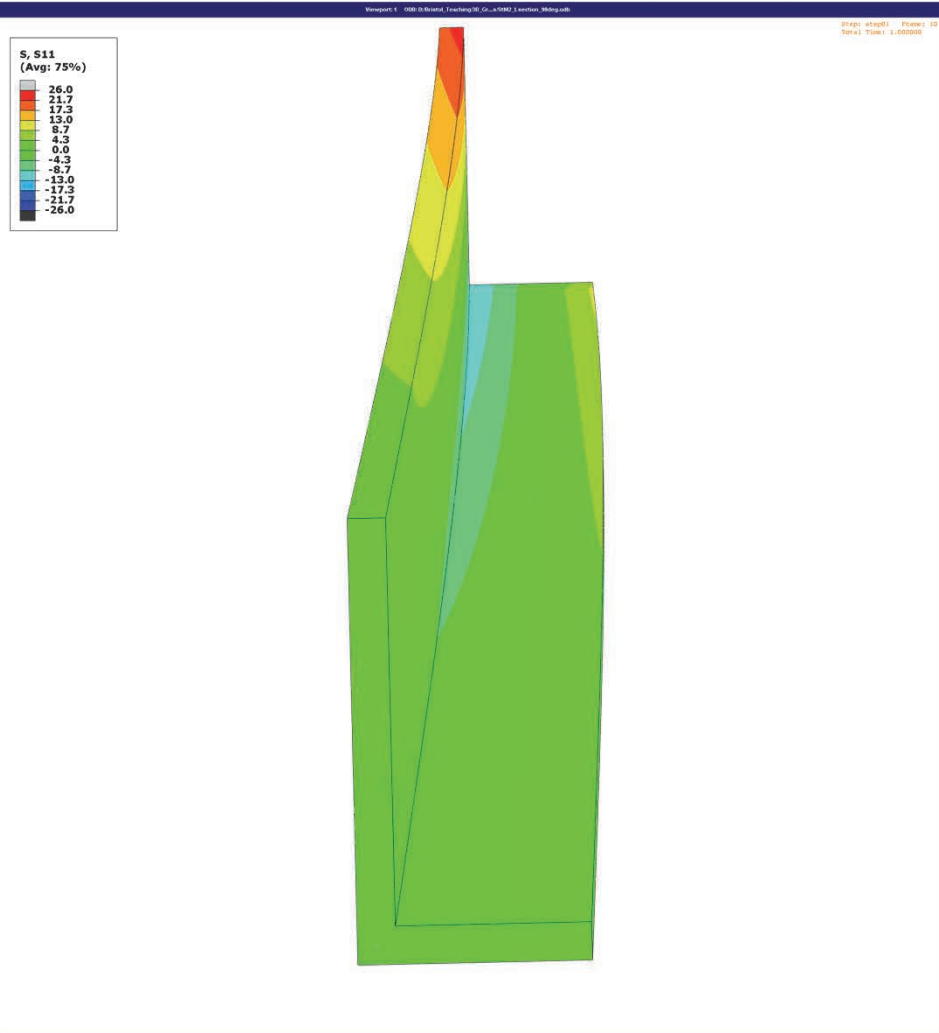
$$\begin{Bmatrix} \delta_x \\ \delta_y \end{Bmatrix} = \begin{bmatrix} m & n \\ -n & m \end{bmatrix} \begin{Bmatrix} \delta_1 \\ \delta_2 \end{Bmatrix} \quad \begin{aligned} m &= \cos(-\theta_p) \\ n &= \sin(-\theta_p) \end{aligned}$$

$$\begin{Bmatrix} \delta_x \\ \delta_y \end{Bmatrix} = \begin{bmatrix} 0.962 & -0.272 \\ 0.272 & 0.962 \end{bmatrix} \begin{Bmatrix} -4.50 \\ -2.08 \end{Bmatrix} \text{ mm}$$

$$\begin{Bmatrix} \delta_x \\ \delta_y \end{Bmatrix} = \begin{Bmatrix} -3.48 \\ -3.53 \end{Bmatrix} \text{ mm}$$



Aluminium Alloy



Aluminium Alloy + Steel

