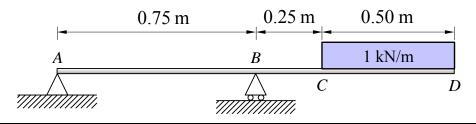
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Example 2.3.3 – Plot the bending moment, slope and deflection diagrams for the beam below, showing the value and location of any maxima and/or minima. The beam has a constant flexural modulus  $EI = 1 \text{ kN m}^2$ .



We start by finding support reactions:

$$\sum M_{@A}^{CW}=0,$$

$$M_A - \left(R_B\right)\left(\frac{3}{4}\text{ m}\right) + \left[\left(1\frac{\text{kN}}{\text{m}}\right)\left(\frac{1}{2}\text{ m}\right)\right]\left(\frac{5}{4}\text{ m}\right) = 0.$$

The extremity A is pinned, therefore  $M_A = 0$  and:

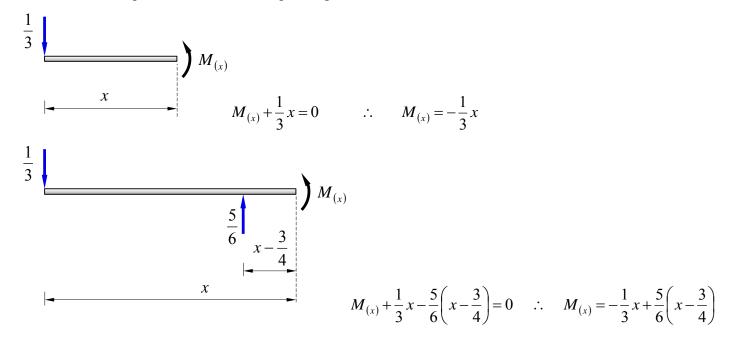
$$(R_B)\left(\frac{3}{4}\mathrm{m}\right) = \left[\left(1\frac{\mathrm{kN}}{\mathrm{m}}\right)\left(\frac{1}{2}\mathrm{m}\right)\right]\left(\frac{5}{4}\mathrm{m}\right)$$
  $\therefore$   $R_B = \frac{5}{6}\mathrm{kN}$ .

Vertical equilibrium gives:

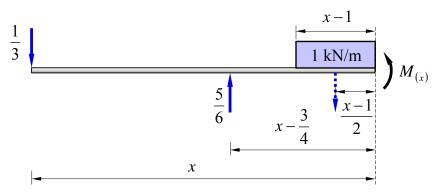
$$\sum F = 0$$

$$R_A + R_B - \left(1\frac{\mathrm{kN}}{\mathrm{m}}\right)\left(\frac{1}{2}\mathrm{m}\right) = 0$$
  $\therefore$   $R_A = -\frac{1}{3}\mathrm{kN}$ 

There are three equations of moment, depending on where we section the beam:



Example 2.3.3



$$M_{(x)} + \frac{1}{3}x - \frac{5}{6}\left(x - \frac{3}{4}\right) + \left[\left(1\right)\left(x - 1\right)\right]\left(\frac{x - 1}{2}\right) = 0$$
 :  $M_{(x)} = -\frac{1}{3}x + \frac{5}{6}\left(x - \frac{3}{4}\right) - \frac{(x - 1)^2}{2}$ 

In order to **combine the three equations into one**, we use the Heaviside step function:

$$M_{(x)} = -\frac{1}{3}x + \left[\frac{5}{6}\left(x - \frac{3}{4}\right)H\left(x - \frac{3}{4}\right)\right] - \left[\frac{(x-1)^2}{2}H(x-1)\right].$$

This is our curvature equation:

$$M_{(x)} = EI \frac{d^2 v}{d x^2} = -\frac{1}{3} x + \left[ \frac{5}{6} \left( x - \frac{3}{4} \right) H \left( x - \frac{3}{4} \right) \right] - \left[ \frac{(x-1)^2}{2} H (x-1) \right]. \tag{1}$$

Integrating once gives the slope:

$$EI \ \phi_{(x)} = EI \frac{dv}{dx} = -\frac{1}{6}x^2 + \left[ \frac{5}{12} \left( x - \frac{3}{4} \right)^2 H \left( x - \frac{3}{4} \right) \right] - \left[ \frac{(x-1)^3}{6} H (x-1) \right] + A. \tag{2}$$

Integrating again gives the deflection:

$$EI \ v_{(x)} = -\frac{1}{18}x^3 + \left[\frac{5}{36}\left(x - \frac{3}{4}\right)^3 H\left(x - \frac{3}{4}\right)\right] - \left[\frac{(x-1)^4}{24}H(x-1)\right] + Ax + B. \tag{3}$$

The first boundary condition is:

$$x = 0$$
,  $v = 0$   $\therefore$   $B = 0$ 

And the second boundary condition is:

$$x = \frac{3}{4}, \ v = 0$$
  $\therefore -\frac{1}{18} \left(\frac{3}{4}\right)^3 + A \left(\frac{3}{4}\right) = 0$   $\therefore A = \frac{1}{32} \text{ kN m}^2$ 

In order to draw the three graphs we need to find some points of interest.

• <u>Point of maximum upward deflection</u>. The span *AB* will 'hog' under the applied load (*i.e.* it will have a 'sad face' type deformation) and the remainder of the beam with deflect downwards. So the maximum upward deflection <u>must</u> occur within span *AB*. Deflection maxima and minima are characterised by a <u>zero slope</u>, therefore we write:

EI 
$$\phi_{x_{(v_{\text{max}})}} = 0 = -\frac{1}{6} x_{(v_{\text{max}})}^2 + \frac{1}{32}$$
  $\therefore$   $x_{(v_{\text{max}})} = \frac{\sqrt{3}}{4} \text{ m} \approx 0.433 \text{ m}.$ 

• Maximum and minimum deflections. These are found by substituting the now known values of x into equation (3):

EI 
$$v_{\text{max}} = -\frac{1}{18} \left( \frac{\sqrt{3}}{4} \right)^3 + \left( \frac{1}{32} \right) \left( \frac{\sqrt{3}}{4} \right)$$
 :  $v_{\text{max}} = 9.021 \text{ mm}$ 



$$EI \ v_{\min} = -\frac{1}{18} \left(\frac{3}{2}\right)^3 + \frac{5}{36} \left(\frac{3}{2} - \frac{3}{4}\right)^3 - \frac{\left(\frac{3}{2} - 1\right)^4}{24} + \left(\frac{1}{32}\right) \left(\frac{3}{2}\right) \quad \therefore \quad v_{\min} = -84.635 \text{ mm}$$

Further relevant points may be found by substituting x values into equations (1), (2) and (3). The final graphs are shown below.

