

Structural Loads in Beams

Cantilever Beam with a Point Load

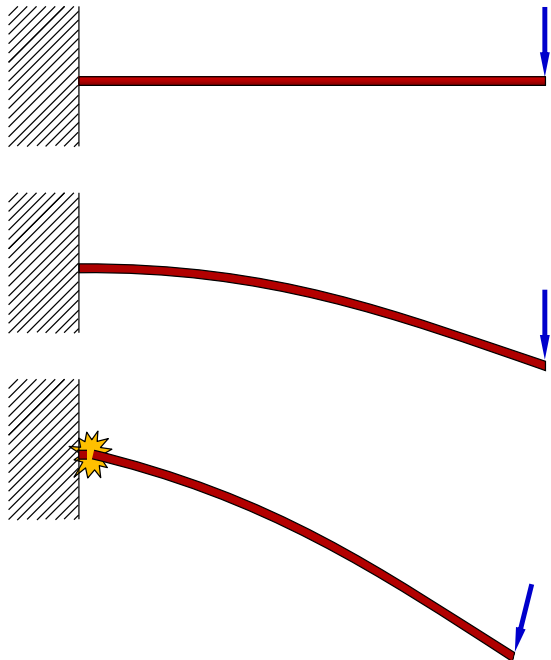
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17 October 2017

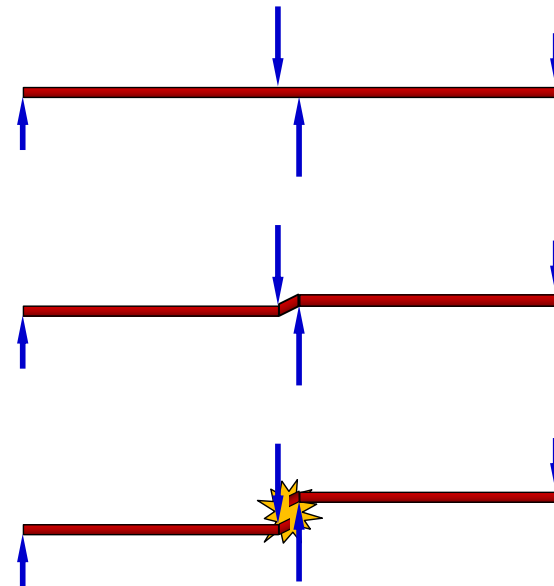
Bending Moment

- Critical for:
 - Long/thin beams, and/or
 - Transverse forces separated by considerable beam length
- Deformation:
 - Curvature
 - Tension/compression on top/bottom surfaces (depending on sign of M)
- Strains: direct strains (tension & compression)
- Failure: 'snapping'



Shear Force

- Critical for:
 - Short/thick beams, and/or
 - Opposing transverse forces close to each other (e.g. when cutting with scissors)
- Deformation:
 - 'Angle changes' (or 'diagonals' stretching or shrinking)
- Strains: shear strains
- Failure: 'shearing off'



2.1 Beam element definition

2.2 Idealisations and assumptions

2.3 Supports and loads

2.4 Sign convention for beams

2.5 Bending moment and shear force diagrams

2.5.1 Simply-supported beam with a concentrated load

2.5.2 Cantilever beam with a concentrated load

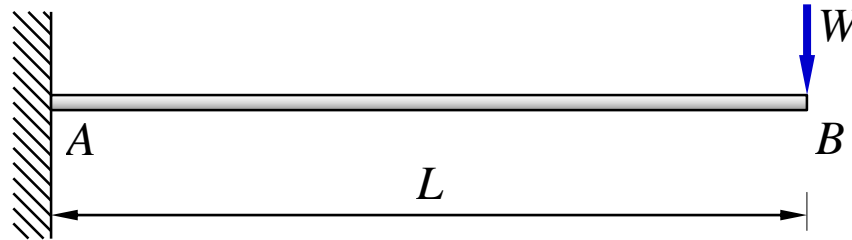
2.5.3 Simply-supported beam with a (constant) distributed load

2.5.4 Cantilever beam with a (constant) distributed load

2.5.5 Simply-supported beam with an arbitrary load distribution

2.6 The principle of superposition

- The cantilever beam is 'built-in' at the 'root' (A) and loaded at the 'tip' (B):

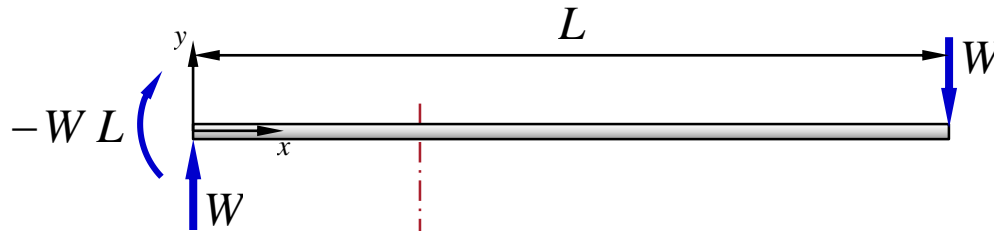


- Global FBD:
 M_A R_A W L

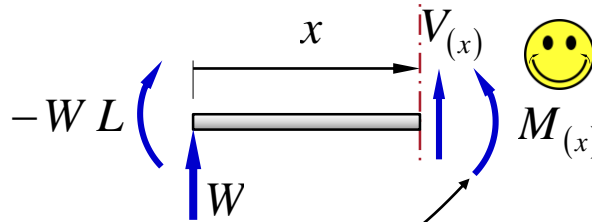
$$\sum M_{@A}^{\text{cw}} = 0 \quad \therefore \quad M_A + (W)(L) = 0 \quad \therefore \quad \boxed{M_A = -W L}$$

$$\sum F = 0 \quad \therefore \quad R_A - W = 0 \quad \therefore \quad \boxed{R_A = W}$$

- Putting the origin of x at point A, from left to right:



- Sectioning the beam at an arbitrary point $0 < x < L$:



$$\sum M_{@x}^{\text{CCW}} = 0 \quad \therefore \quad M_{(x)} - (-WL) - (W)(x) = 0 \quad \therefore \quad \boxed{M_{(x)} = W(x - L)}$$

$$\sum F_{@x} = 0 \quad \therefore \quad W + V_{(x)} = 0 \quad \therefore \quad \boxed{V_{(x)} = -W}$$

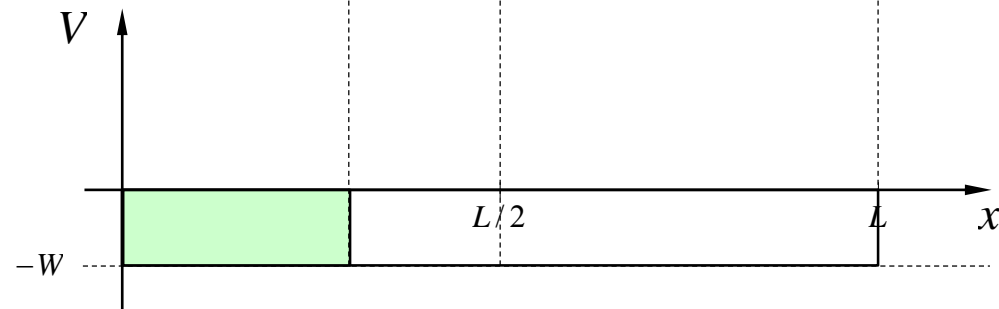
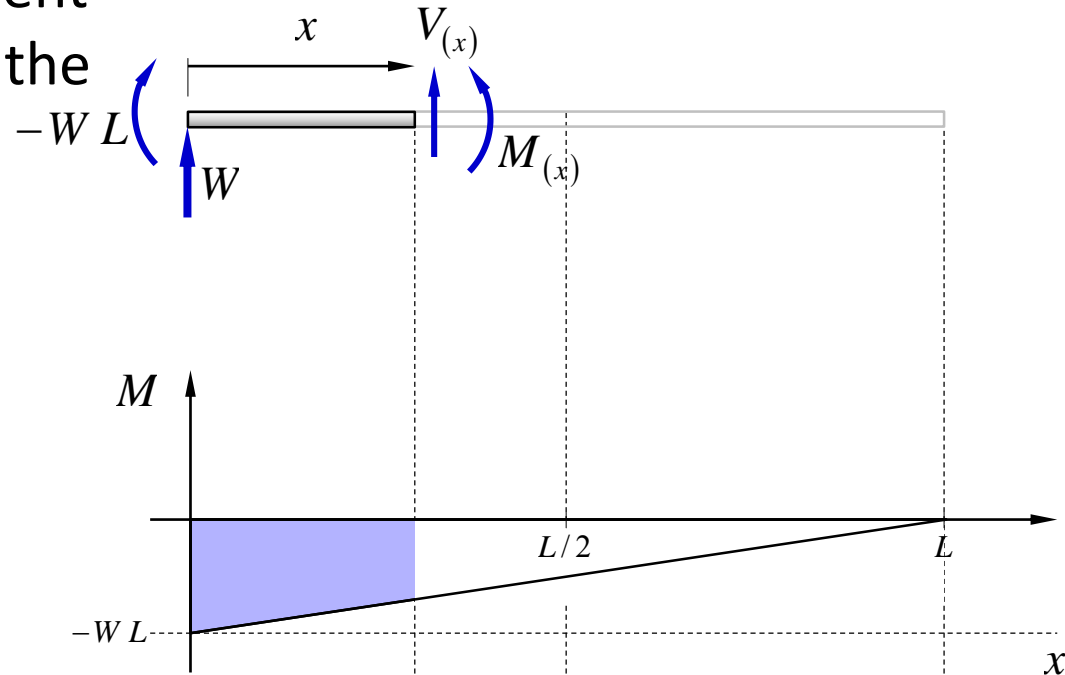
- Note that the bending moment **varies linearly** with x , while the shear force is a **constant**:

$$M_{(x)} = W(x - L)$$

$$V_{(x)} = -W$$

- So we can plot the diagrams:

These solutions are valid for
any $0 < x < L$



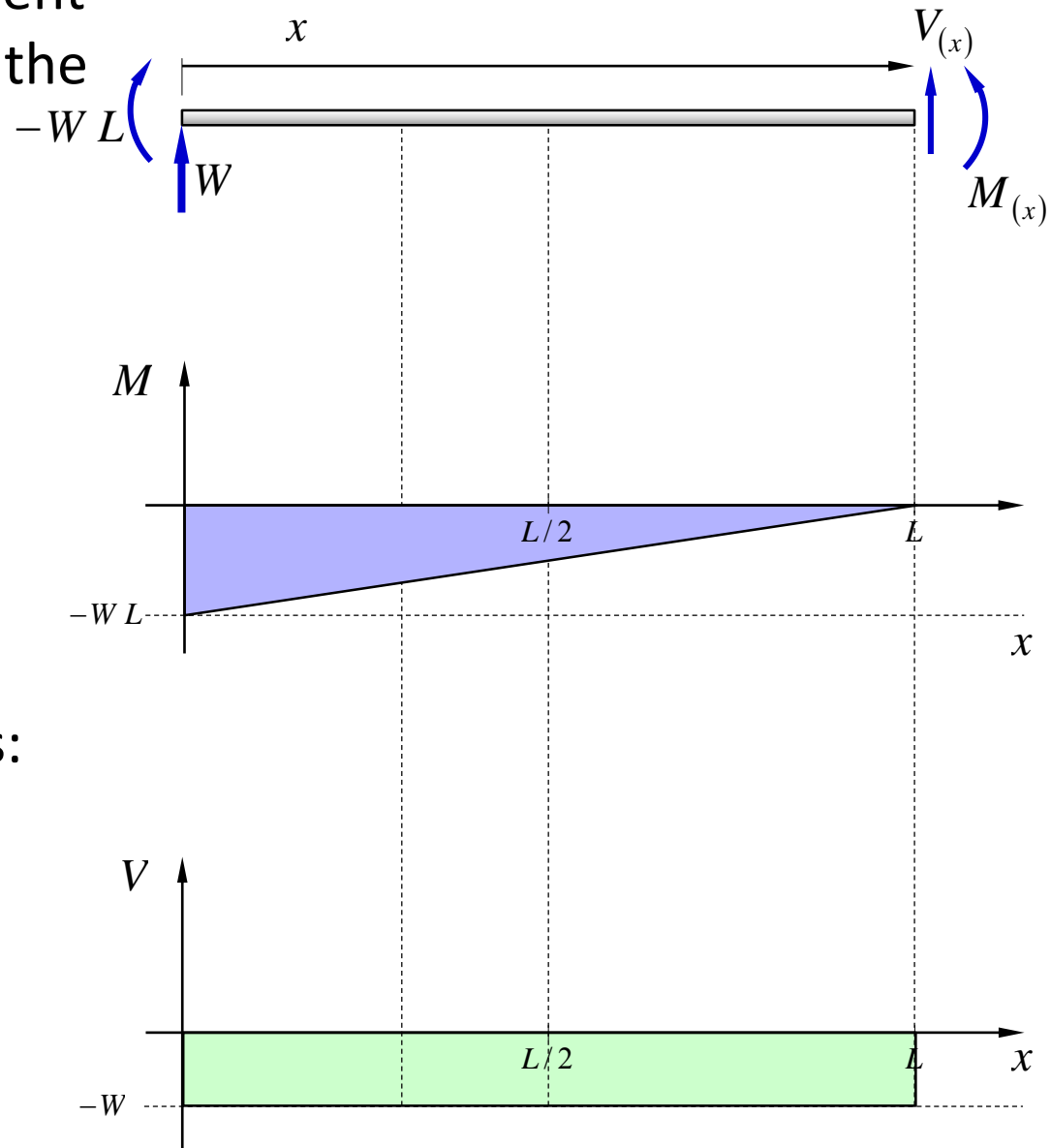
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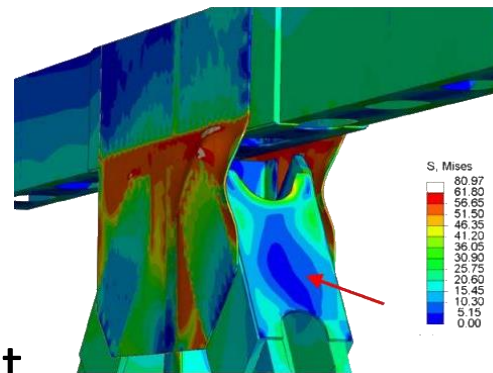
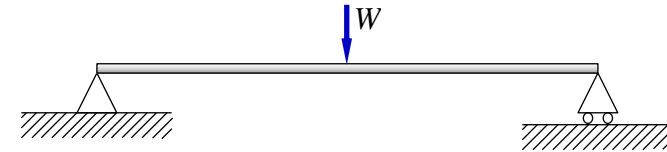
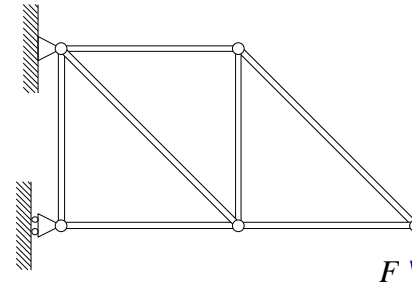
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2.5.4 Simply-supported beam with a (constant) distributed load

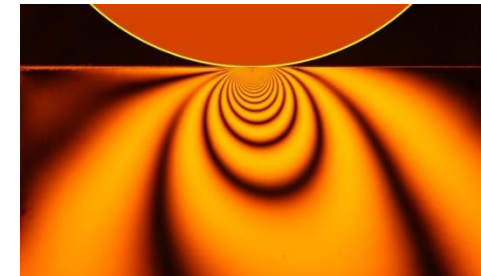
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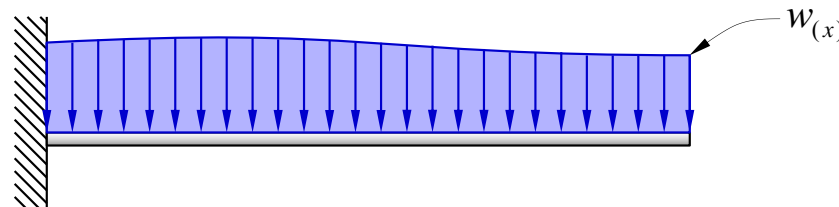
- So far we have been dealing with idealised 'point loads'
 - Forces applied at infinitely small areas \rightarrow not very realistic
- In reality, loads are applied over a **finite area**, *e.g.*:
 - Stresses (N/m^2) in real 3D joints
 - Pressure (N/m^2) due to contact
- In 2D problems we assume 'unit width', and define distributed loads as **force per unit length** (N/m)



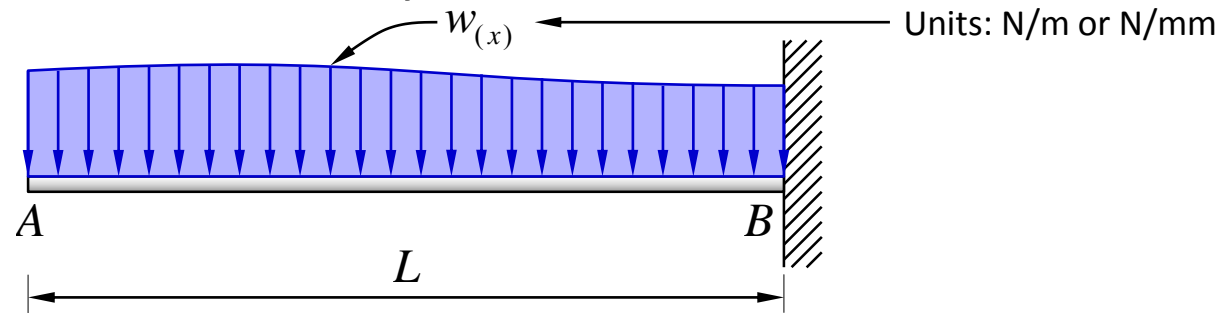
A real truss joint



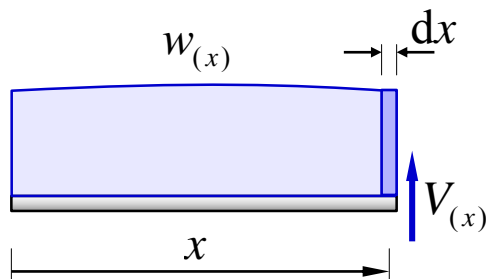
Real contact stresses
(photoelastic effect of a
cylinder on a flat surface)



- Cantilever beam with an arbitrary load distribution:



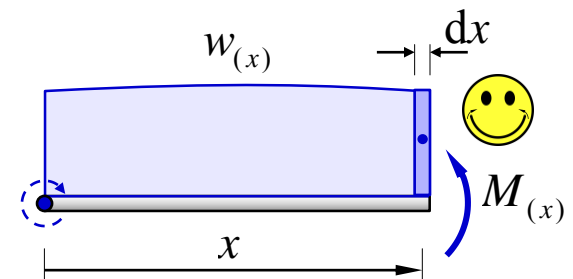
Shear Force



$$V_{(x)} = \oplus \int_0^x w_{(x)} dx$$

- Shear force is the **area** under the curve $w(x)$

Bending Moment

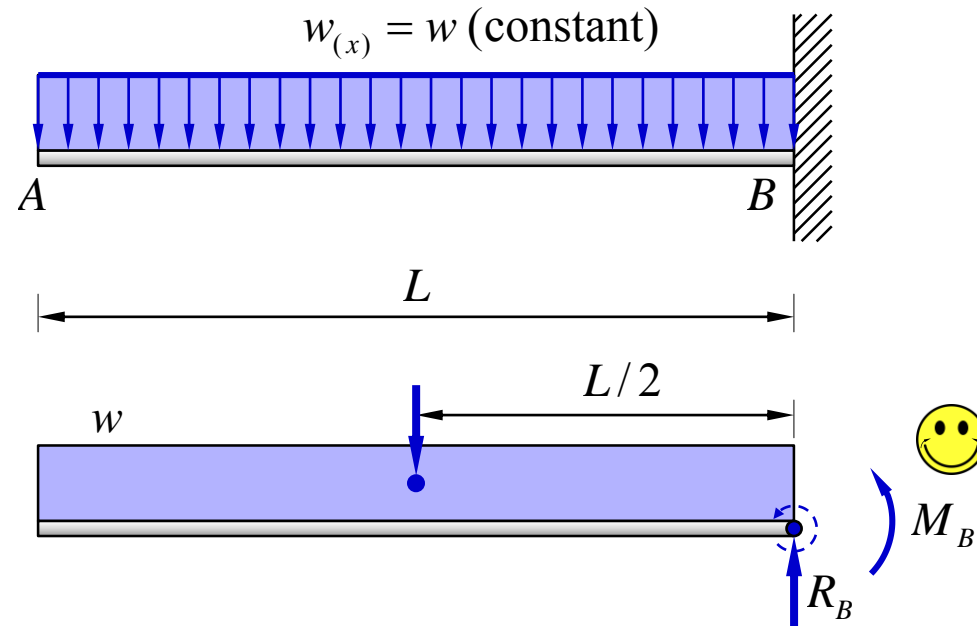


$$M_{(x)} = \ominus \int_0^x w_{(x)} x dx$$

- Bending moment is the **moment of area** of the curve $w(x)$

Signs depend on directions of x and $w(x)$

- Cantilever beam with constant load distribution:

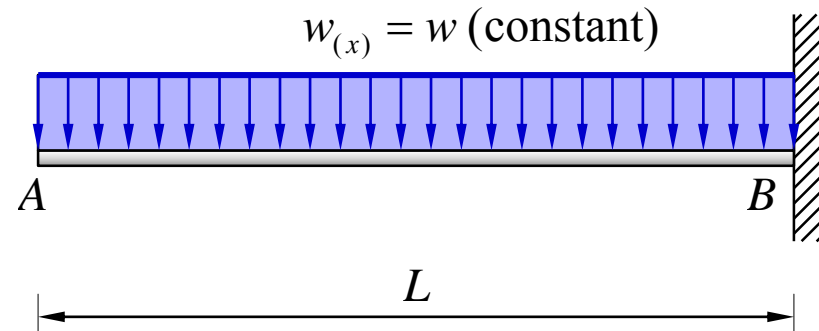


- Global FBD:

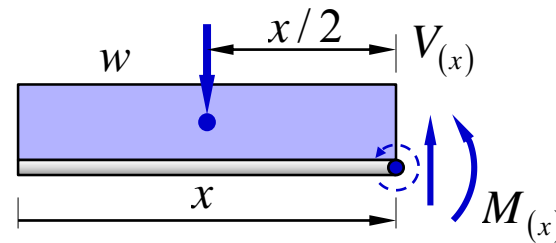
$$\sum F = 0 \quad \therefore \quad R_B - (w)(L) = 0 \quad \therefore \quad \boxed{R_B = wL}$$

$$\sum M_{@B}^{\text{ccw}} = 0 \quad \therefore \quad M_B + (wL)\left(\frac{L}{2}\right) = 0 \quad \therefore \quad \boxed{M_B = -\frac{wL^2}{2}}$$

- Cantilever beam with constant load distribution:



- Section FBD:



$$\sum F = 0 \quad \therefore \quad V_{(x)} - (wx) = 0 \quad \therefore \quad \boxed{V_{(x)} = wx}$$

$$\sum M_{@x}^{\text{ccw}} = 0 \quad \therefore \quad M_{(x)} + (wx) \left(\frac{x}{2} \right) = 0 \quad \therefore \quad \boxed{M_{(x)} = -\frac{wx^2}{2}}$$

- Finally we can plot the shear force and bending moment diagrams

$$R_B = wL$$

$$V_{(x)} = wx$$

$$M_B = -\frac{wL^2}{2}$$

$$M_{(x)} = -\frac{wx^2}{2}$$

