

Advanced Bending and Torsion

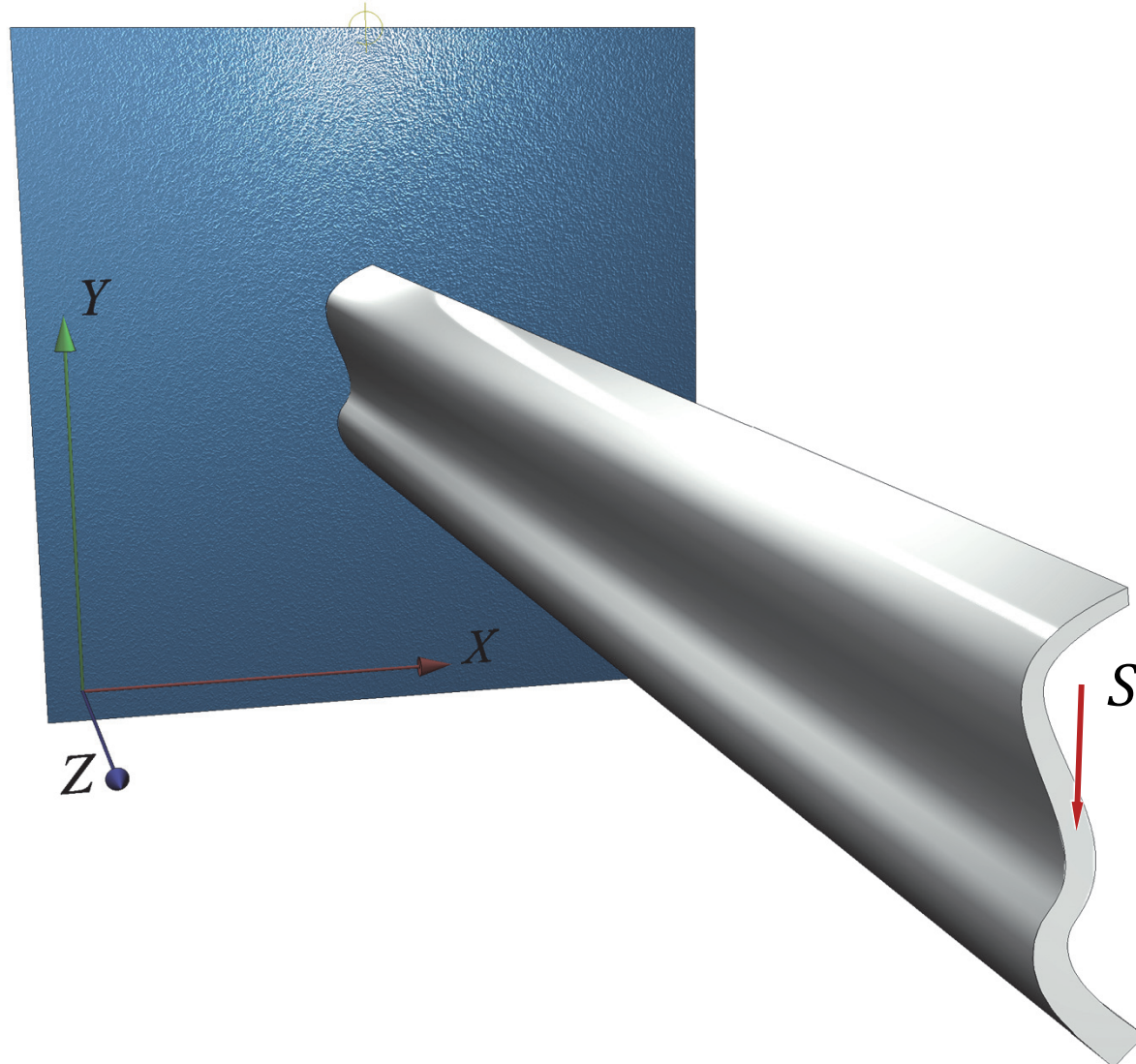
Shear Stresses in Open Thin-Walled Sections

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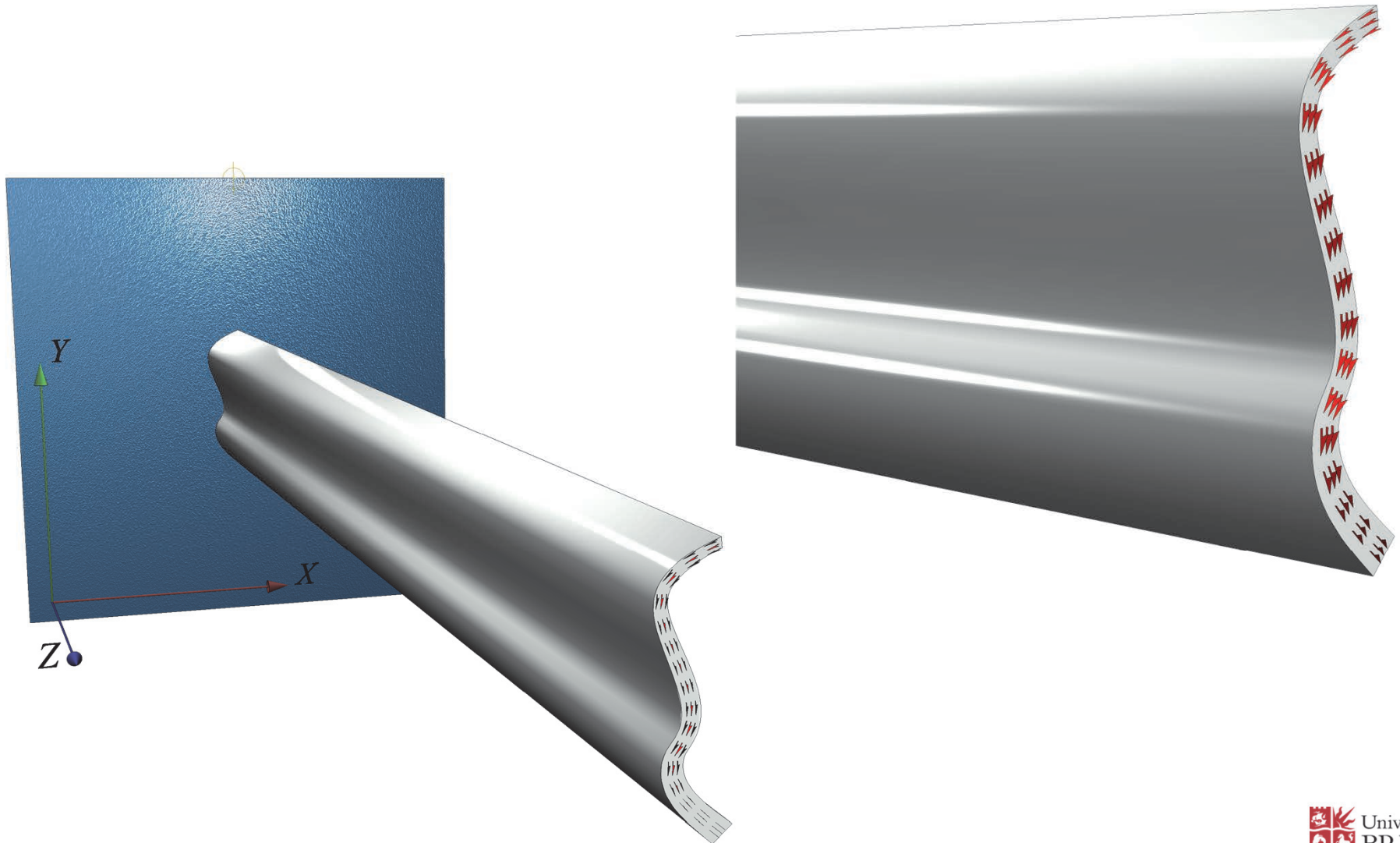
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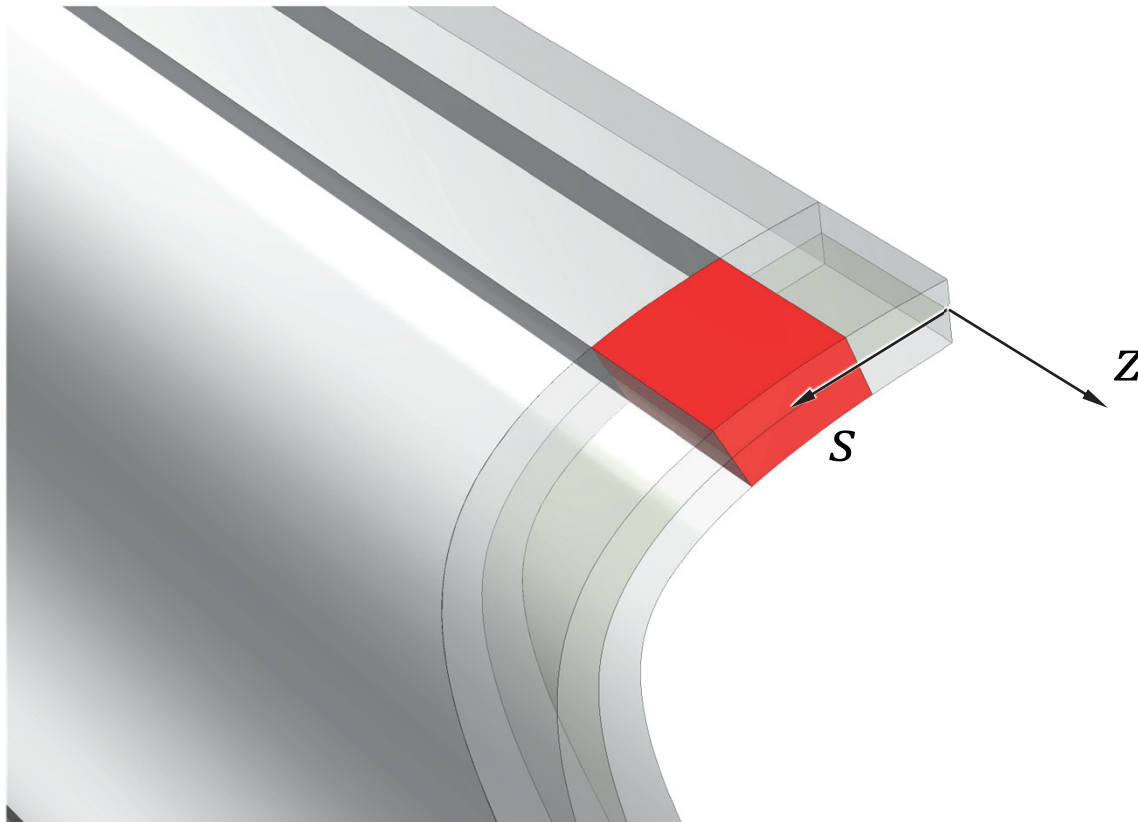
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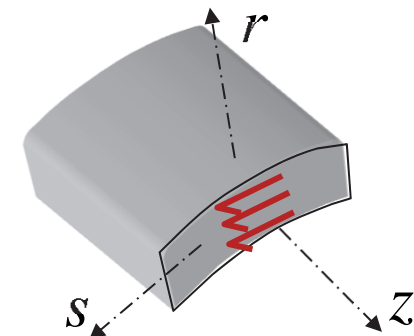
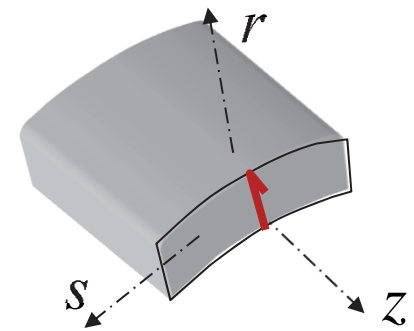
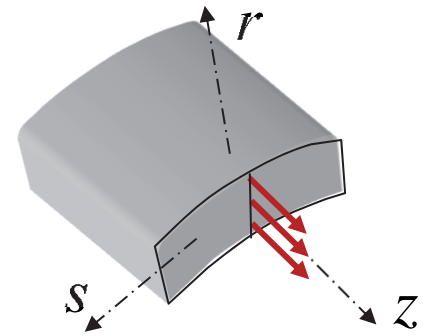


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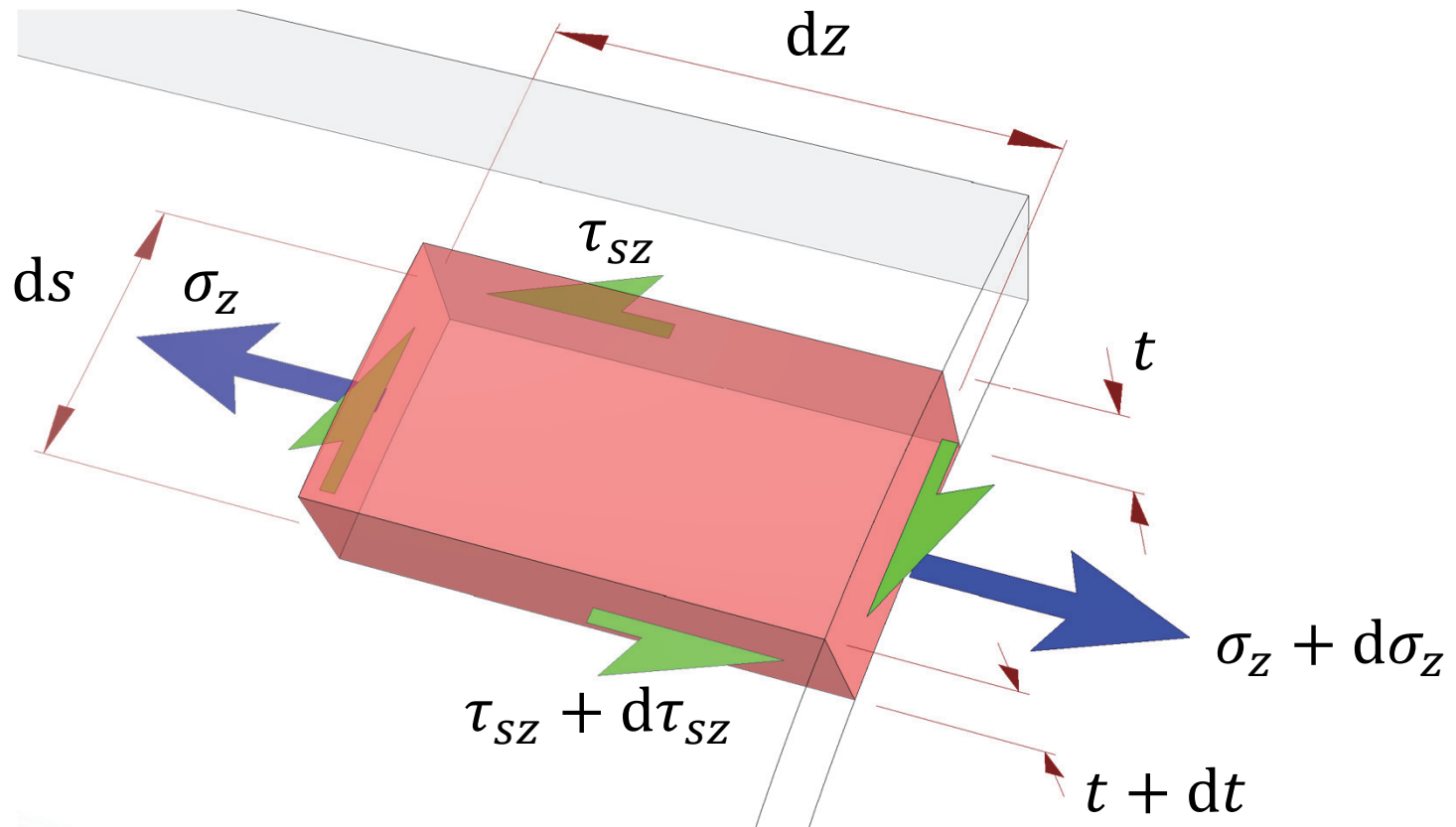


- Thin-wall assumptions
 - Direct stresses are constant through the thickness:
 - Through-thickness direct and shear stresses are negligible:
 - In-plane shear stresses are constant through the thickness:
- The **shear flow** (shear force per unit arclength) is defined as:

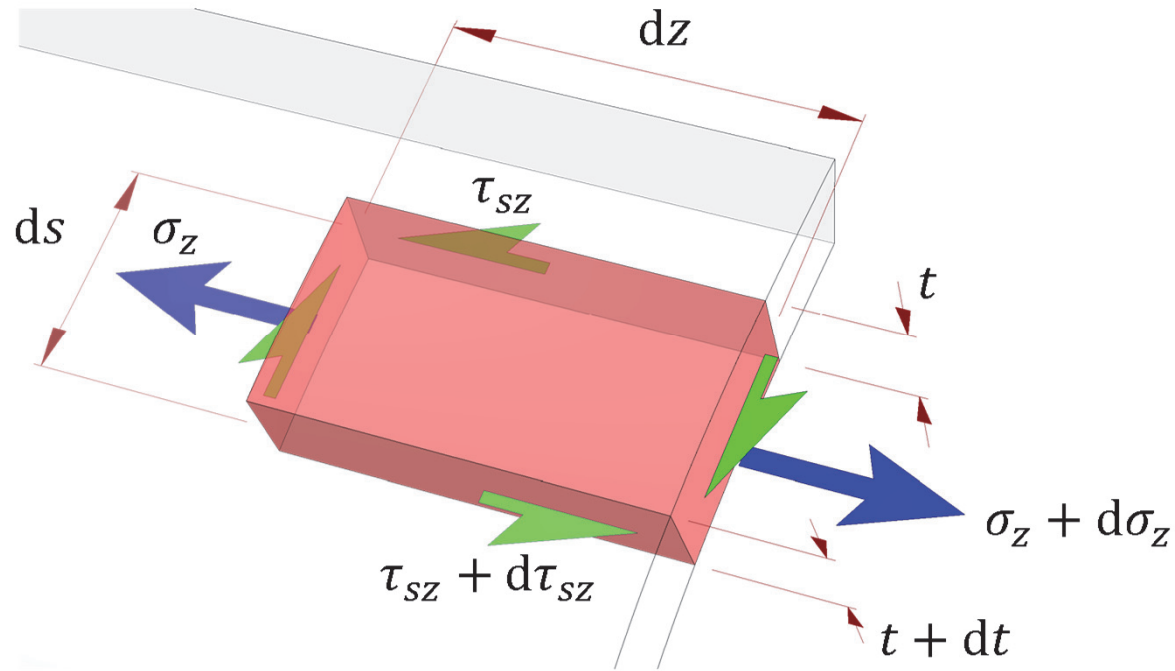
$$q_{zs} = t \tau_{zs}$$



- We consider a small element along the arc length s
- There are two major stresses acting: σ_z and τ_{sz}



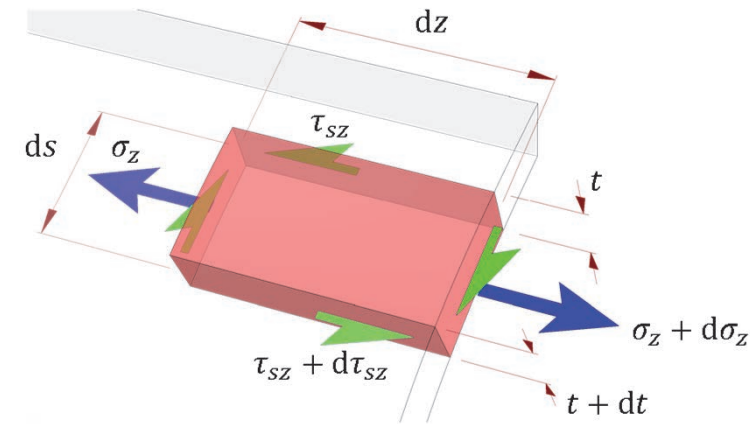
- Force equilibrium in the z direction:



$$\begin{aligned}
 & (\sigma_z + d\sigma_z) \left(t + \frac{dt}{2} \right) (ds) - (\sigma_z) \left(t + \frac{dt}{2} \right) (ds) \\
 & + (\tau_{sz} + d\tau_{sz})(t + dt)(dz) - (\tau_{sz})(t)(dz) = 0
 \end{aligned}$$

- Complementarity of shear: $\tau_{sz} = \tau_{zs}$
- Therefore:

$$(\sigma_z + d\sigma_z) \left(t + \frac{dt}{2} \right) (ds) - (\sigma_z) \left(t + \frac{dt}{2} \right) (ds) \\ + (\tau_{zs} + d\tau_{zs})(t + dt)(dz) - (\tau_{zs})(t)(dz) = 0$$



- Neglecting higher order terms (*e.g.* containing $d\sigma_z \times d\tau_{zs} \times ds$)

$$t d\sigma_z ds + \tau_{zs} dt dz + t d\tau_{zs} dz = 0$$

- Dividing by $(ds dz)$

$$t \frac{d\sigma_z}{dz} + \tau_{zs} \frac{dt}{ds} + t \frac{d\tau_{zs}}{ds} = 0$$

$$t \frac{d\sigma_z}{dz} + \frac{d(t \tau_{zs})}{ds} = 0$$

- For constant t :

$$t \frac{d\sigma_z}{dz} + \frac{d(\tau_{zs})}{ds} = 0$$

- Making $q_s = q_{zs} = t \tau_{zs}$:

$$t \frac{d\sigma_z}{dz} + \frac{dq_s}{ds} = 0$$

- Replacing with our stress equation:

$$\sigma_z = \frac{M_y I_{xx} + M_x I_{xy}}{I_{xy}^2 - I_{xx} I_{yy}} x + \frac{M_x I_{yy} + M_y I_{xy}}{I_{xx} I_{yy} - I_{xy}^2} y$$

- Gives finally:

$$-q_s = \left(\frac{S_x I_{xx} + S_y I_{xy}}{I_{xy}^2 - I_{xx} I_{yy}} \right) \int_0^s x t ds + \left(\frac{S_y I_{yy} + S_x I_{xy}}{I_{xx} I_{yy} - I_{xy}^2} \right) \int_0^s y t ds$$

