

# Stress, Strain and Deformation

## **Axially Loaded Members**

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- Consider a bar subjected to an axial force:

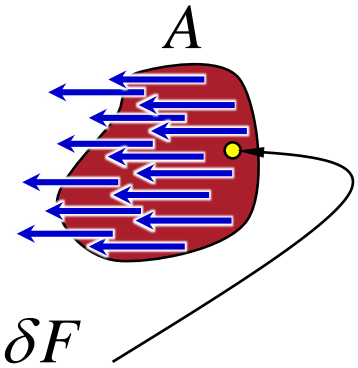


- We need to know:
  - the distribution of the force in the material
    - i.e. the “**stress**”
  - the extension of the bar per unit length
    - i.e. the “**strain**”

- Normal stress = Force/Area  $\rightarrow \sigma = F/A$   $\sigma$  = “sigma”
  - where the force is normal (perpendicular) to the area

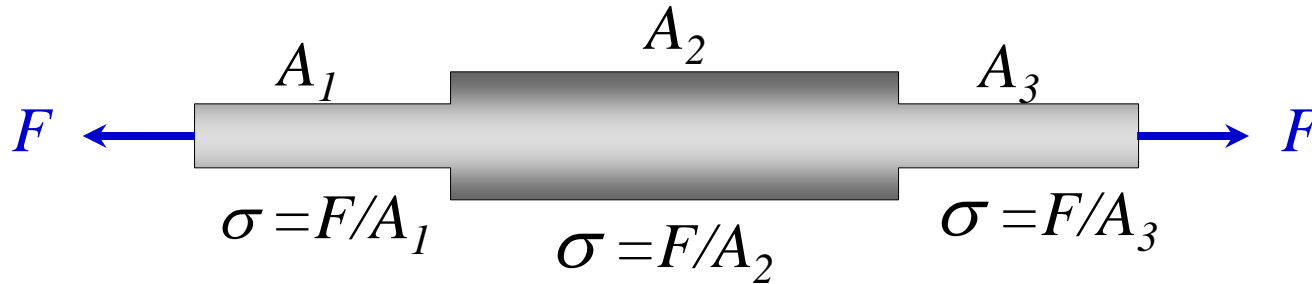


- Can be considered at a **point**
- Can be **tensile** or **compressive**
- Units of stress = force per unit area
  - e.g.  $1 \text{ N/m}^2 = 1 \text{ Pa}$  (Pascal)
  - or  $1 \text{ N/mm}^2 = 10^6 \text{ N/m}^2 = 10^6 \text{ Pa} = 1 \text{ MPa}$

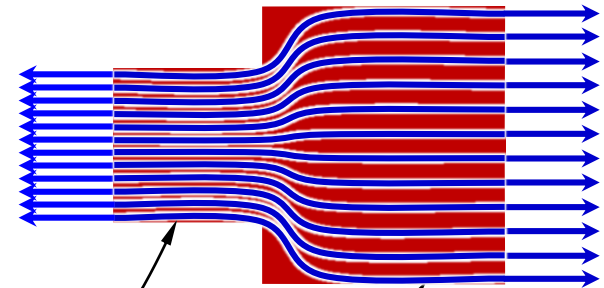


$$\sigma = \lim_{\delta A \rightarrow 0} \frac{\delta F}{\delta A}$$

- Consider a change in  $x$ -section area, *e.g.*:



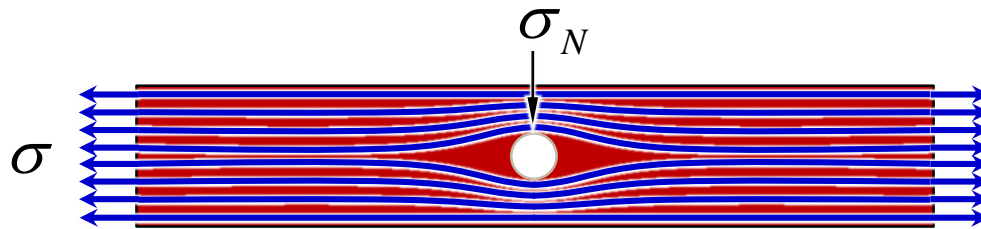
- Average stress =  $F/A$ 
  - Valid away from discontinuity
- But at the discontinuity?
  - Consider “lines of internal loading”
  - Note areas of close spacing: higher stresses
  - and areas of wide spacing: lower stresses



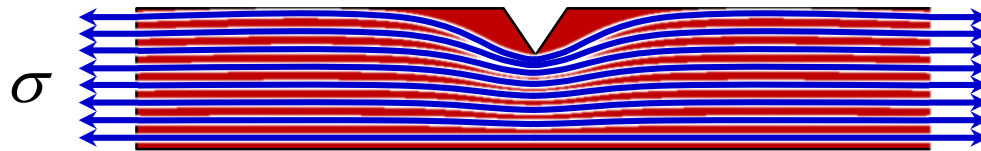
- For example at a hole, notch or crack:

$$\sigma_N = K_T \cdot \sigma$$

(max. local stress) = (stress concentration factor) · (far field stress)



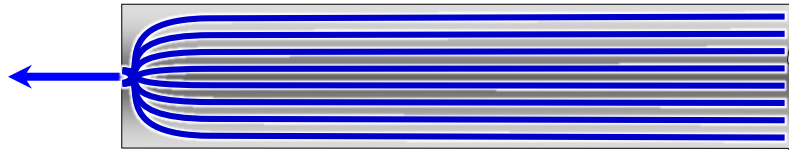
$K_T = 3$  for round hole



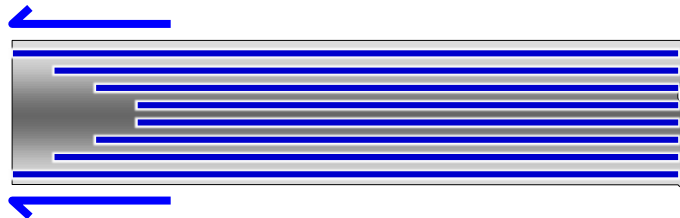
$K_T > 3$  for sharp notch or corner

- Tabulated solutions for standard cases usually account for the area reduction plus the stress concentration effect.

- Local stress distribution will depend on the **details of load introduction**
- Consider a bar where the load is introduced (or reacted) at:
  - a point on its end

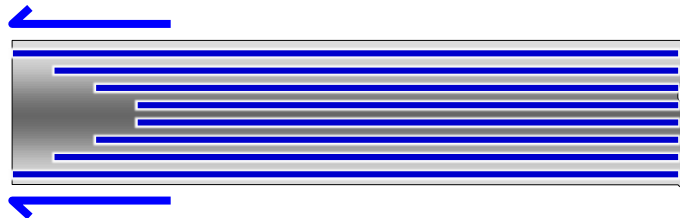
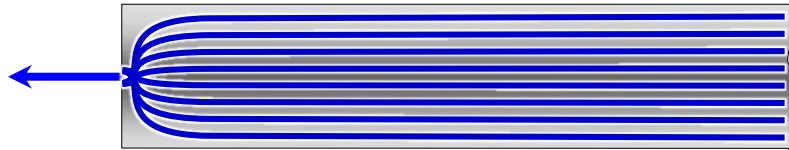


- by gripping the sides near the end

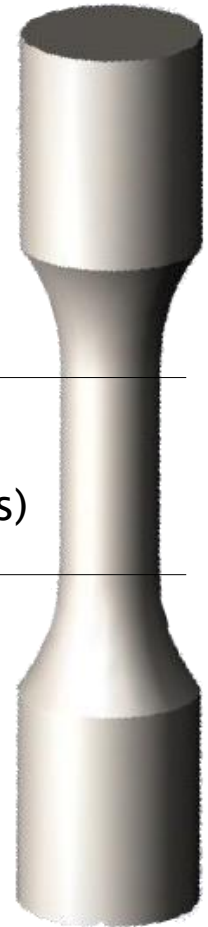


→ Stress fields tend to become uniform as we move away from the discontinuity

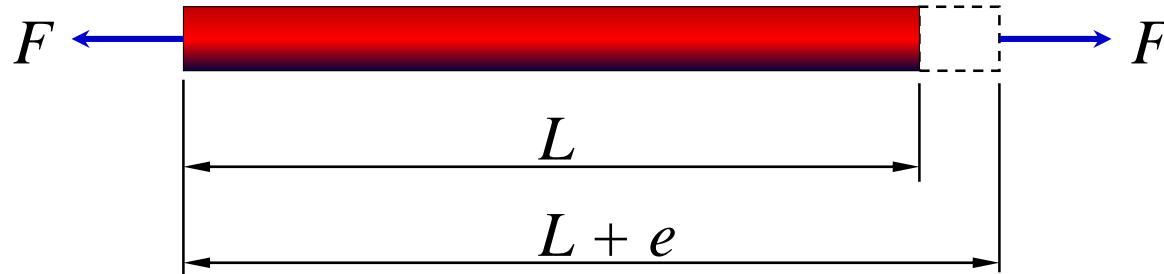
- For example, a 'dogbone' test specimen:



Gauge length  
(uniform stress)



- E.g. loaded bar:



$$\varepsilon = \frac{e}{L}$$

$$\text{strain} = \frac{\text{extension}}{\text{original length}}$$

- Note that strain is non-dimensional, but is commonly stated as:
  - Percentage strain:
  - Micro strain:

$$\% \varepsilon = \varepsilon \cdot 100\%$$

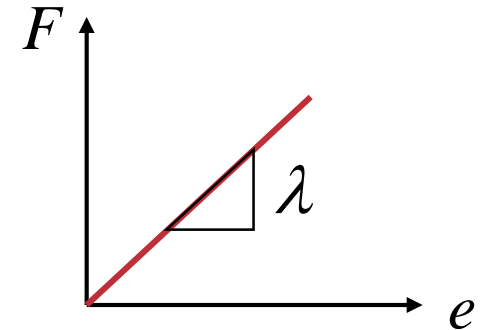
$$\mu\varepsilon = \varepsilon \cdot 10^{-6}$$



Linear elastic behaviour:

$$F = \lambda e$$

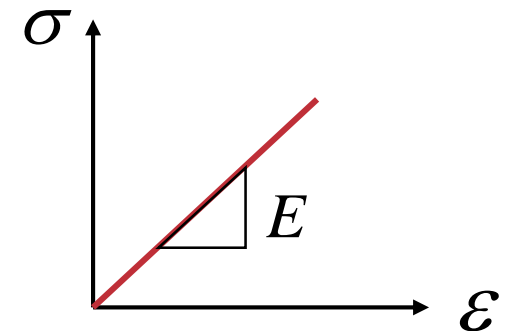
- where  $\lambda$  = “stiffness”  $\lambda$  = “lambda”
- units: force per unit displacement
- *e.g.* N/m, N/mm



In terms of stress and strain:

$$\sigma = E \varepsilon$$

- where  $E$  = “Young’s modulus”
- units: stress per unit strain
- *i.e.* same as stress since strain is dimensionless
- *e.g.* N/mm<sup>2</sup> or N/m<sup>2</sup> or MPa or GPa



- Note,  $E$  is constant for a particular material and does not vary much within an alloy family, e.g.:

## Steel:

$$\begin{aligned} E &= 200,000 \text{ N/mm}^2 &= 200,000 \text{ MPa} \\ &= 200 \text{ kN/mm}^2 &= \mathbf{200 \text{ GPa}} \end{aligned}$$

## Titanium alloy:

$$E = 110,000 \text{ N/mm}^2 \quad = \mathbf{110 \text{ GPa}}$$

## Aluminium alloy:

$$E = 70,000 \text{ N/mm}^2 \quad = \mathbf{70 \text{ GPa}}$$

(Note:  $1 \text{ kN/mm}^2 = 1 \text{ GPa}$ )

- Relationship between stiffness and modulus:

$$\lambda = \frac{F}{e}$$
$$= \frac{\sigma A}{\varepsilon L} \quad \text{where } \sigma / \varepsilon = E$$

$$\lambda = \frac{E A}{L}$$