

Structural Loads in Beams

Introduction

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11 October 2017

2.1 Beam element definition

2.2 Idealisations and assumptions

2.3 Supports and loads

2.4 Sign convention for beams

2.5 Bending moment and shear force diagrams

2.5.1 Simply-supported beam with a concentrated load

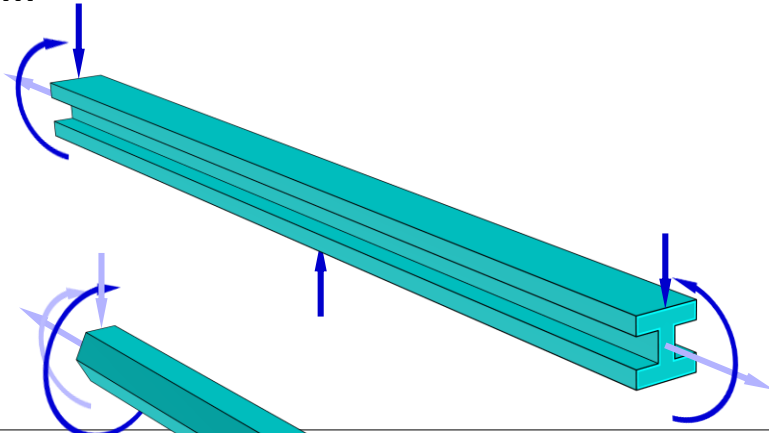
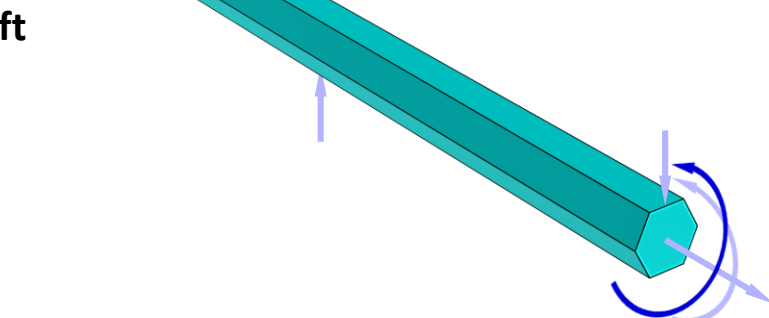
2.5.2 Cantilever beam with a concentrated load

2.5.3 Simply-supported beam with a (constant) distributed load

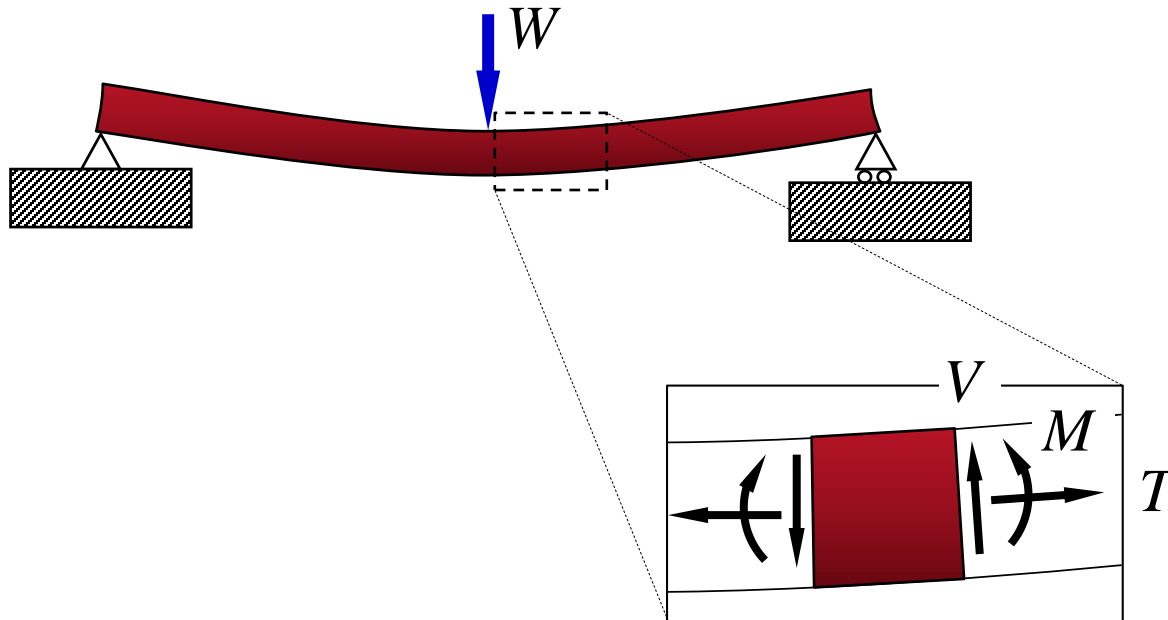
2.5.4 Cantilever beam with a (constant) distributed load

2.5.5 Simply-supported beam with an arbitrary load distribution

2.6 The principle of superposition

| Names | Supported Loads | Stresses & Strains |
|---|--|---|
| Truss (‘axial member’, ‘bar’, ‘rod’, ‘tie’ or ‘strut’) | <ul style="list-style-type: none">• Axial force | <ul style="list-style-type: none">• Direct (tensile OR compressive) |
| Beam  | <ul style="list-style-type: none">• Transverse (shear) force• Bending moment• (Axial force)• (Torque) | <ul style="list-style-type: none">• Direct• ‘Bending’ (tensile AND compressive)• Shear (through the thickness) |
| Shaft  | <ul style="list-style-type: none">• Torque• (Axial force)• (Transverse shear force)• (Bending moment) | <ul style="list-style-type: none">• Shear (torsion)• Direct (tensile OR compressive)• ‘Bending’ (tensile AND compressive)• Shear (through the thickness) |

- A structural member capable of carrying **transverse forces**, **bending moments** and **axial loads**
 - We will leave axial loads aside for now → ‘simple bending’
- A beam subjected to transverse loads will deflect until:
 - the **internal forces** and **moments** generated at **any section** in the beam balance the external loading

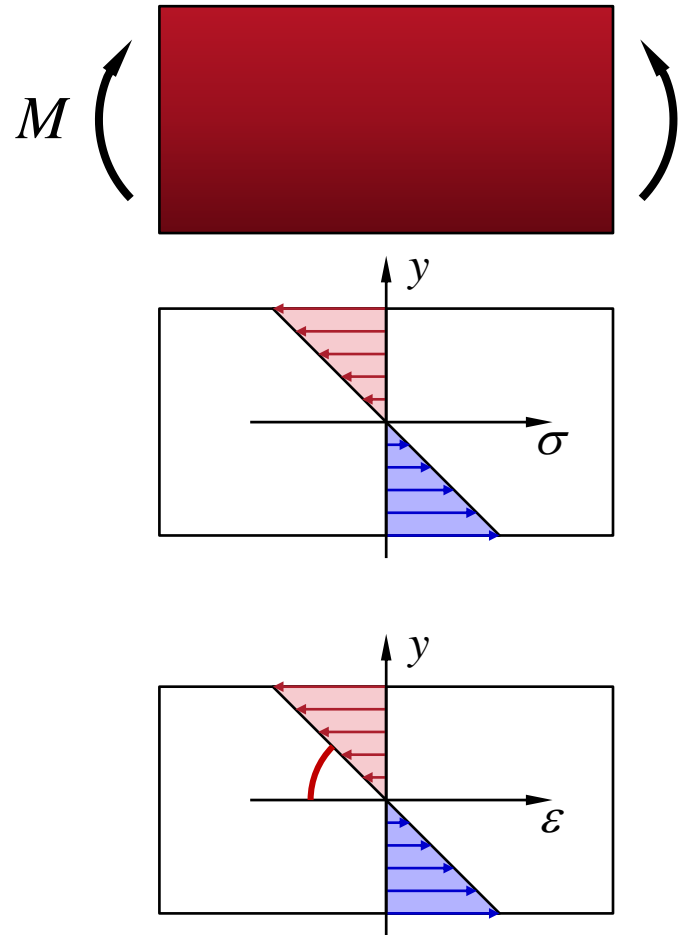


T = Axial force

M = Bending moment

V = Shear force

- Bending moments are related to:
 - Stresses: longitudinal **direct stresses**
 - Strains: longitudinal **direct strains** and **bending curvature**
- Bending stresses will be studied in detail soon



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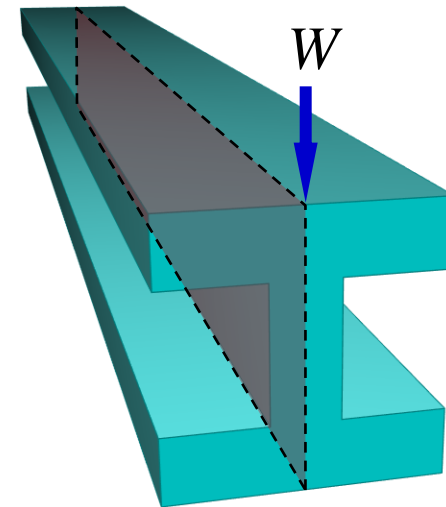
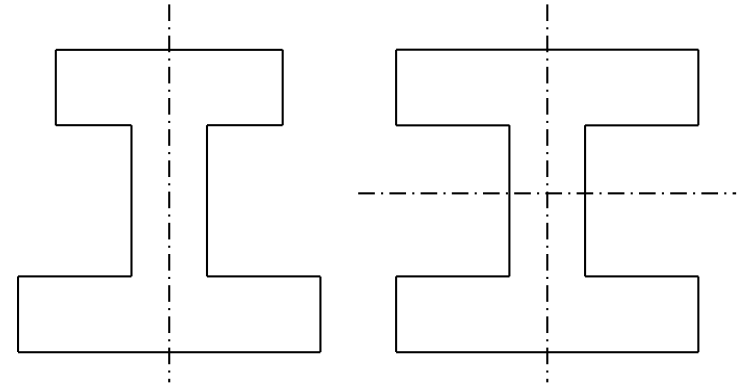
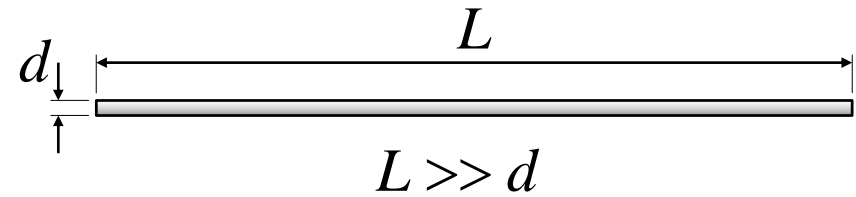
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- Beams are **straight** and **slender**
- Cross-sections are 'singly' or 'doubly' symmetric
- Loads and moments are applied in one symmetric plane of the cross-section
- 'Simple bending' assumption:
 - Neglecting **shear deformation** (but NOT shear forces!)
 - Neglecting **axial forces** (for now)



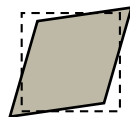
Bending deformation:

- **Plane sections remain plane**
 - Sections rotate about a neutral axis of the cross-section subtending a line through the **centre of curvature** of the beam:
- Only **pure bending deformation** is considered

– Bending



– Shearing

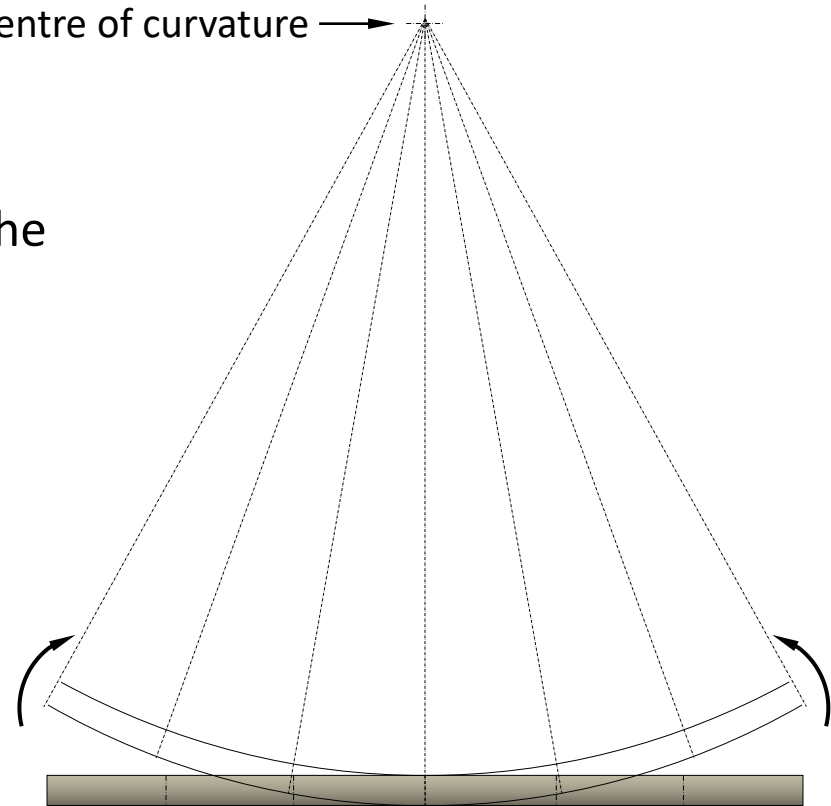


– Extension



These **deformation modes** are neglected here

Centre of curvature →



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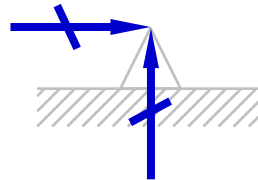
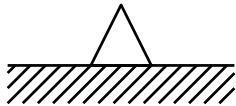
- Similar to trusses, with the addition of the 'built-in' support:

Support

Possible reactions

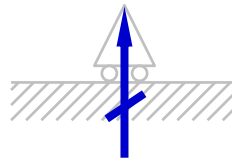
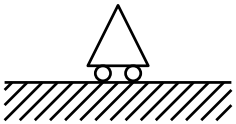
Degrees of Freedom

'Pinned'
'Simple'



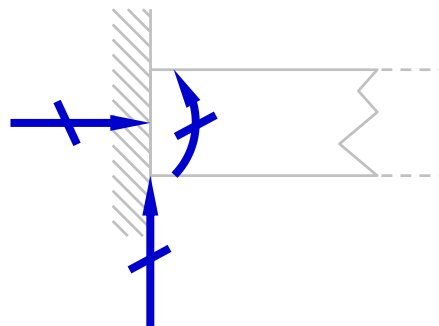
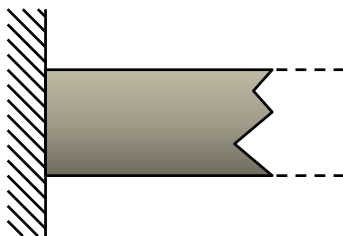
Rotational freedom but **no translational** freedom

'Roller'
'Simple'



Rotational freedom
+ **translational** freedom
along the **surface**

'Built-in'
'Encastre'
'Fixed'



No translational freedom and
no rotational freedom

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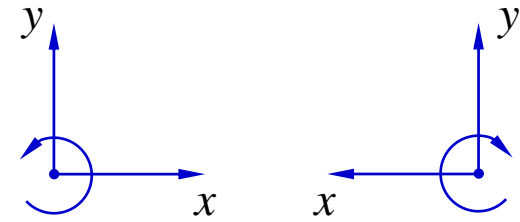
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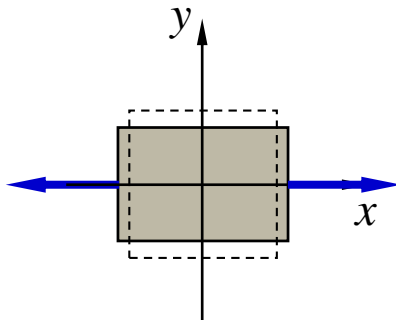
2.4 Sign Convention for Beams

- External sign convention
 - y positive up, but x can be inverted if convenient:
 - For moments, positive sense follows the 'right hand rule'



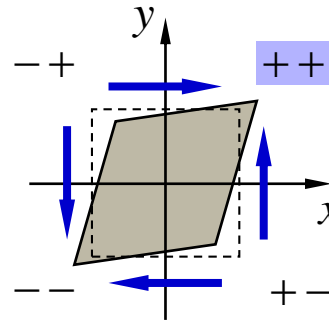
- Internal sign convention

Extension



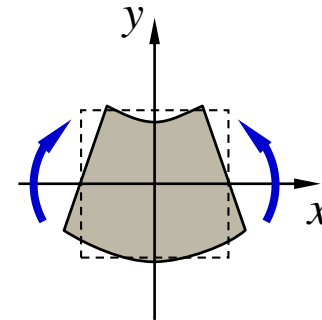
Positive:
'Tension'

Shear



Positive:
'Pinching
the ++ quadrant'

Bending



Positive:
'Sagging'
(or the 'smiley face')



- Always assume unknown moments and shear forces to be **positive** according to the **internal sign convention** above!!

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Diagrams are derived using the **method of sections** - similar to the case of a slender truss!

1. Analyse the **global FBD**

- Resolve reaction forces & moments

2. Choose appropriate origin and sense for the **x coordinate**

- Consider loads, boundary conditions, symmetries etc.

3. Section the beam at a given x and **expose internal forces**

- Always follow the internal sign convention!
- Write equations of equilibrium and solve unknowns at current x : $\begin{cases} V(x) \\ M(x) \end{cases}$

4. **Move along** the x coordinate and repeat

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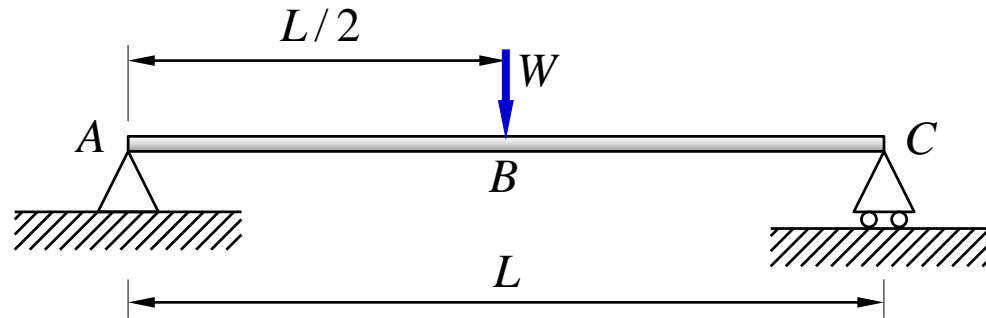
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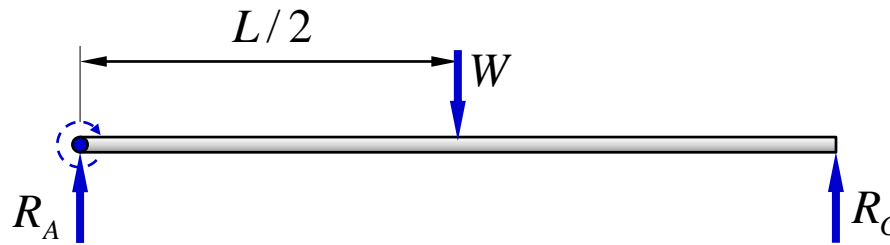
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- Consider the centrally loaded simply-supported beam:



- Global FBD:

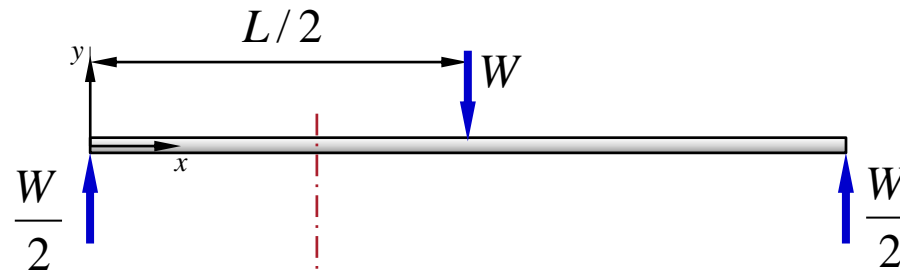


Note: there are **no horizontal forces or reactions**, so we omit the subscripts x and y

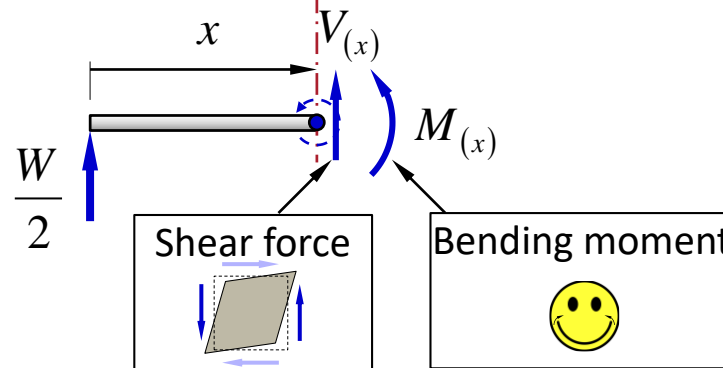
$$\sum M_{@A}^{\text{cw}} = 0 \quad \therefore \quad W \left(\frac{L}{2} \right) - R_C (L) = 0 \quad \therefore \quad R_C = \frac{W}{2}$$

$$\sum F = 0 \quad \therefore \quad R_A + R_C - W = 0 \quad \therefore \quad R_A = \frac{W}{2}$$

- Putting the origin of x at point A, from left to right:



- Sectioning the beam at an arbitrary point $0 < x < L/2$:



IMPORTANT!

Always assume unknowns to be **positive** following the **internal sign convention!**

$$\sum M_{@x}^{\text{ccw}} = 0 \quad \therefore \quad M_{(x)} - \left(\frac{W}{2} \right) x = 0 \quad \therefore \quad M_{(x)} = \frac{W}{2} x$$

$$\sum F_{@x} = 0 \quad \therefore \quad V_{(x)} + \frac{W}{2} = 0 \quad \therefore \quad V_{(x)} = -\frac{W}{2}$$

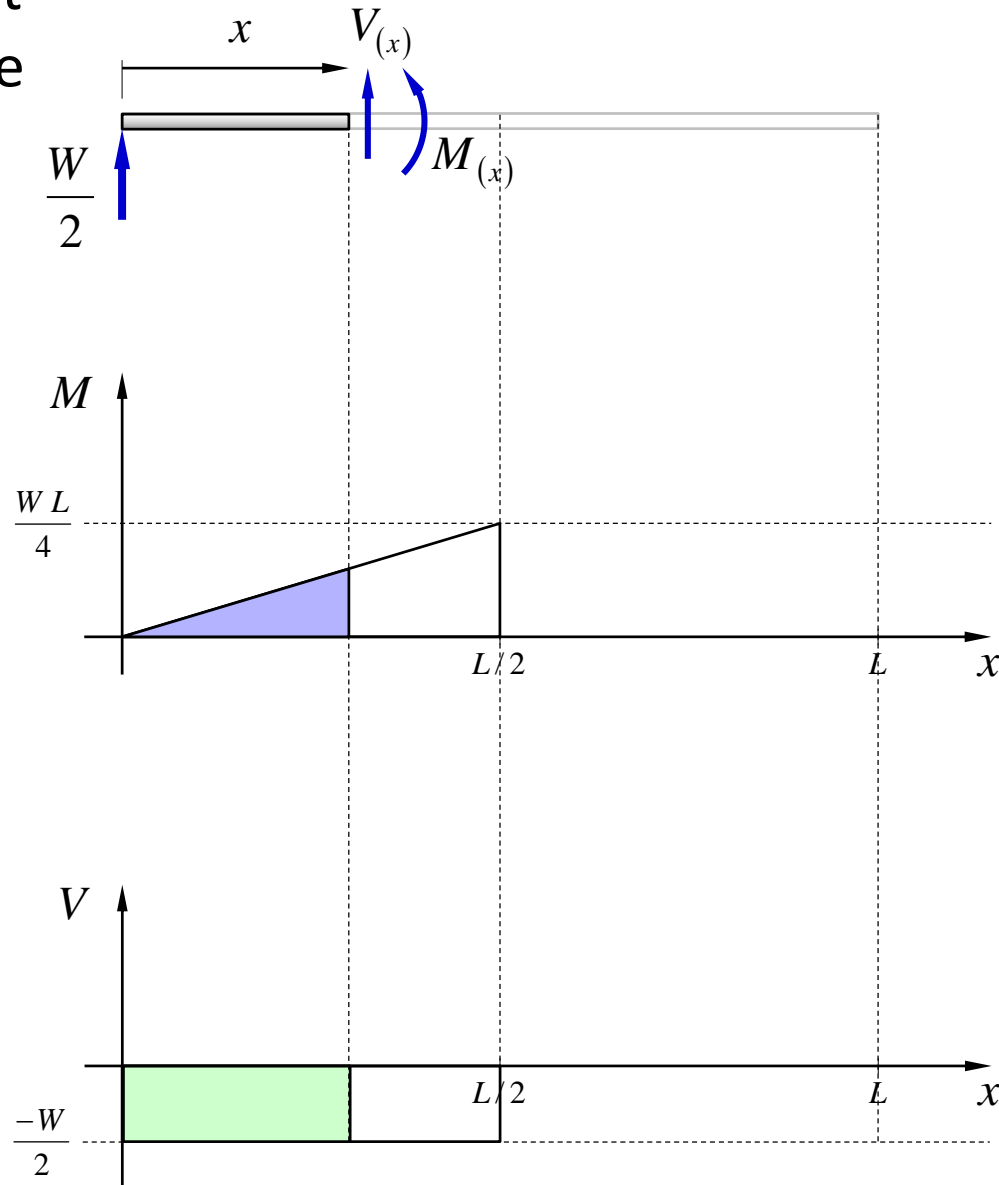
- Note that the bending moment **varies linearly** with x , while the shear force is a **constant**:

$$M_{(x)} = \frac{W}{2}x$$

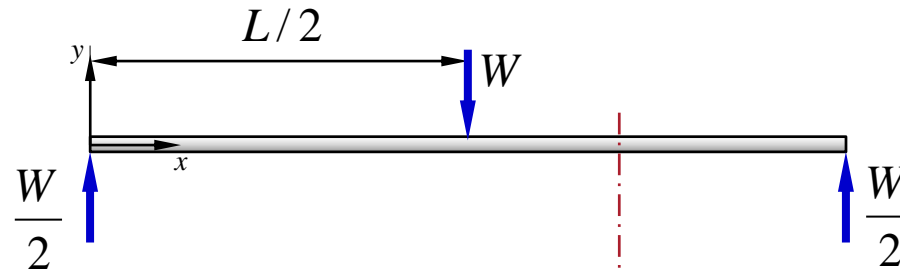
$$V_{(x)} = -\frac{W}{2}$$

- So we can plot the diagrams:

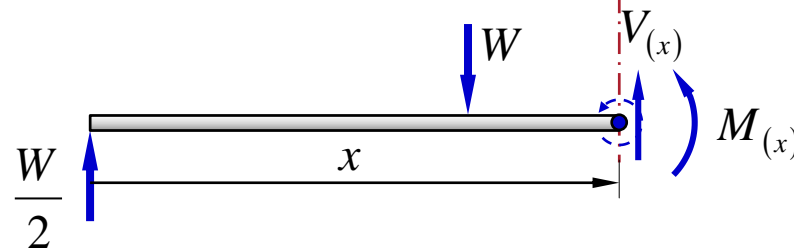
These solutions are valid for
any $0 < x < L/2$.
But what happens after that?



- Now sectioning the beam at $L/2 < x < L$:



- The load W suddenly 'appears' in our FBD:



$$\sum M_{@x}^{\text{ccw}} = 0 \quad \therefore M_{(x)} + (W)\left(x - \frac{L}{2}\right) - \left(\frac{W}{2}\right)x = 0 \quad \therefore M_{(x)} = \left(\frac{W}{2}\right)(L - x)$$

$$\sum F = 0 \quad \therefore V_{(x)} - W + \frac{W}{2} = 0 \quad \therefore V_{(x)} = \frac{W}{2}$$

2.5.1 Simply-Supported Beam with Concentrated Load

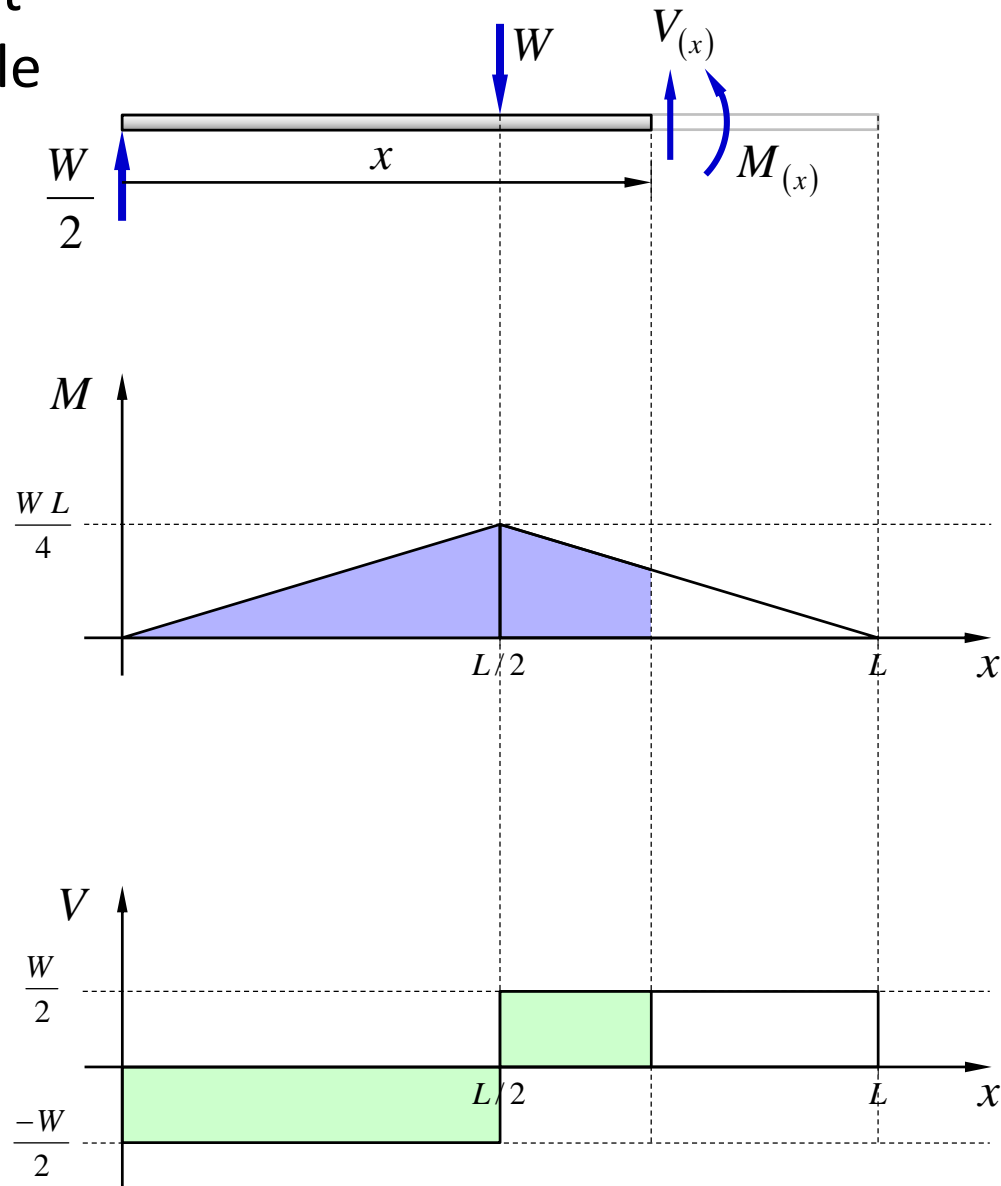
80

- Note that the bending moment now has a **negative slope**, while the shear force has **changed sign**:

$$M_{(x)} = \left(\frac{W}{2}\right)(L - x)$$

$$V_{(x)} = \frac{W}{2}$$

- And the diagrams become:



2.5.1 Simply-Supported Beam with Concentrated Load

81

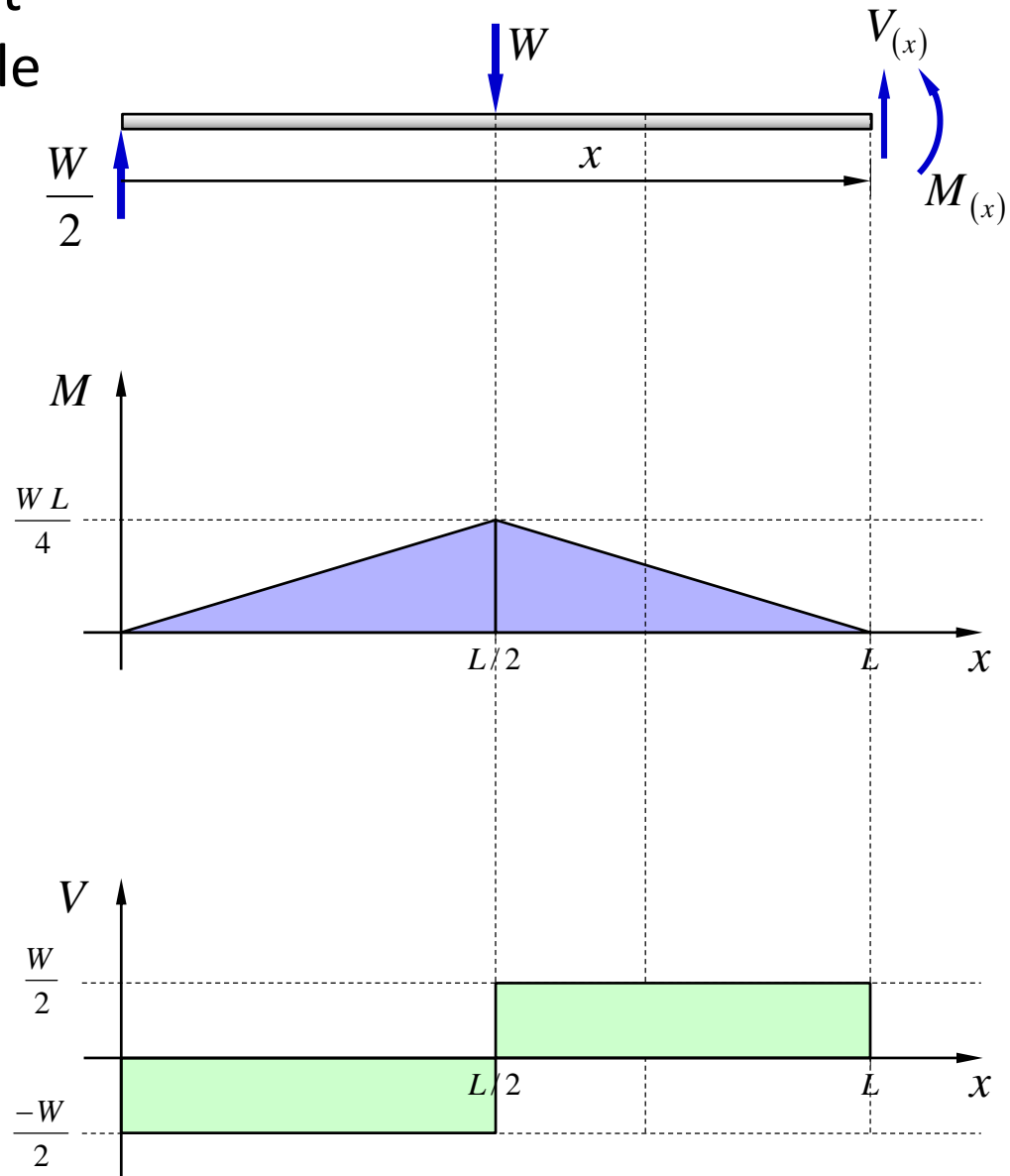
- Note that the bending moment now has a **negative slope**, while the shear force has **changed sign**:

$$M_{(x)} = \left(\frac{W}{2}\right)(L - x)$$

$$V_{(x)} = \frac{W}{2}$$

- And the diagrams become:

And at $x = L$ the bending moment finally vanishes



- Shear force is the first derivative of the bending moment:

$$V_{(x)} = -\frac{dM_{(x)}}{dx}$$

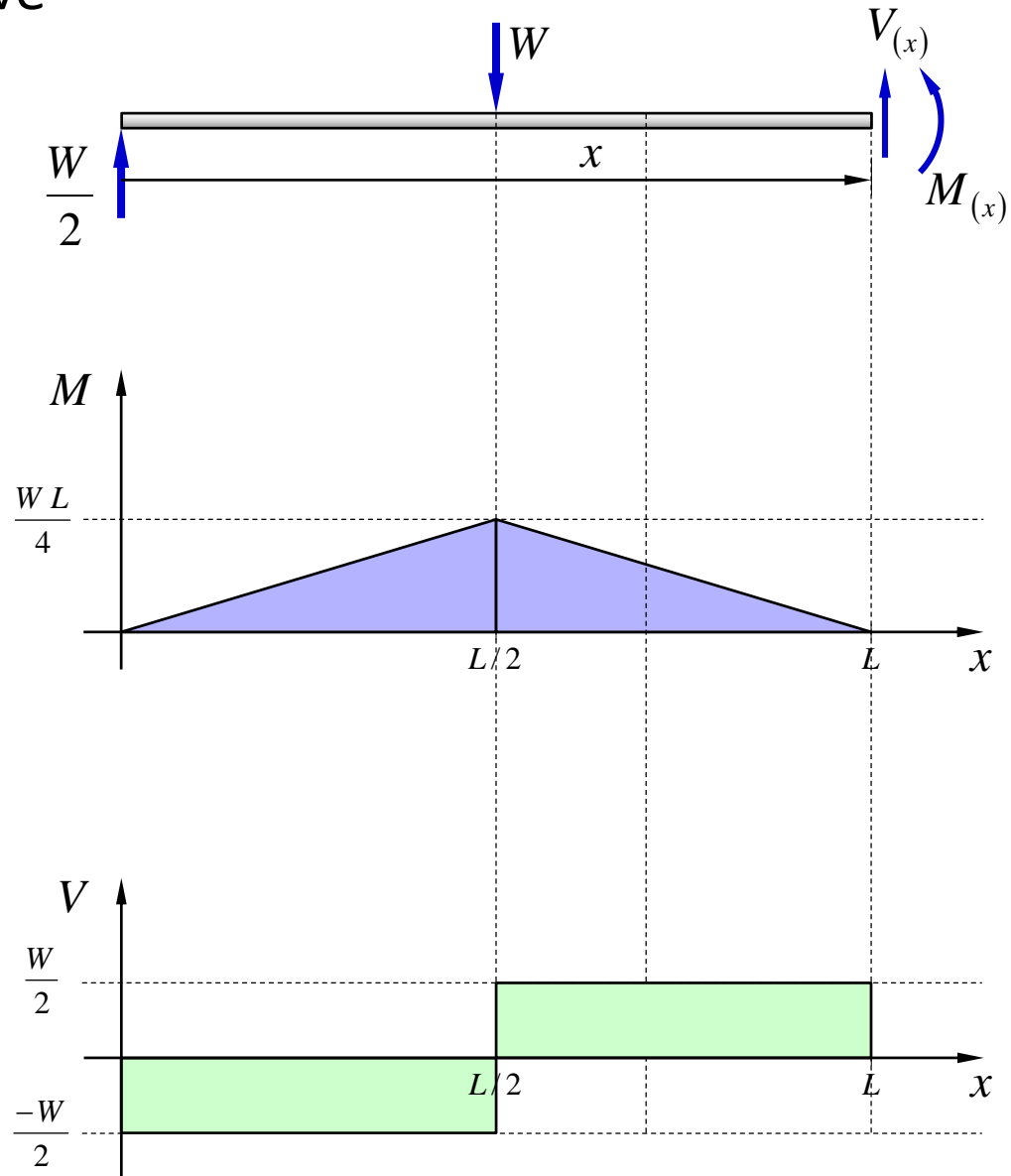
- i.e. $-V_{(x)}$ is the 'slope' of $M_{(x)}$

- Conversely, bending moments are the integral of the shear forces:

$$M_{(x)} = -\int_0^x V_{(x)} dx$$

- i.e. $-M_{(x)}$ is the 'cumulative area' under the graph of $V_{(x)}$

Note: the 'minus' signs in the equations above 'appear/disappear' depending on the direction of the local x -axis !



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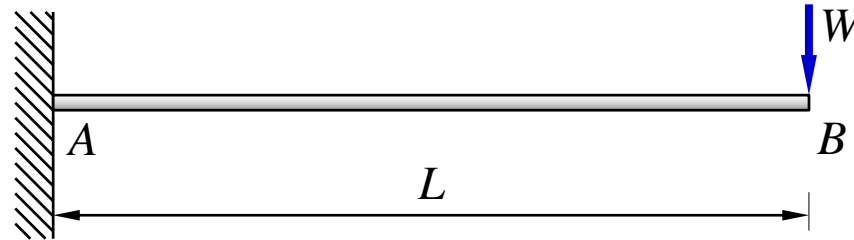
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- The cantilever beam is 'built-in' at the 'root' (A) and loaded at the 'tip' (B):

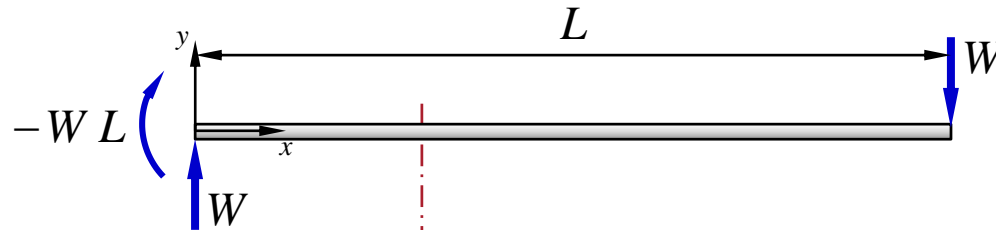


- Global FBD: M_A R_A W L

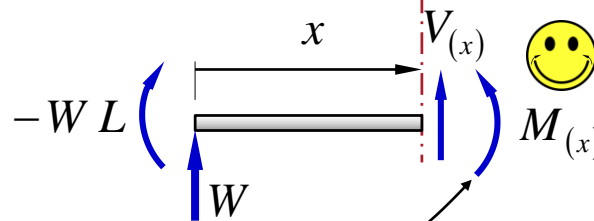
$$\sum M_{@A}^{\text{cw}} = 0 \quad \therefore \quad M_A + (W)(L) = 0 \quad \therefore \quad \boxed{M_A = -W L}$$

$$\sum F = 0 \quad \therefore \quad R_A - W = 0 \quad \therefore \quad \boxed{R_A = W}$$

- Putting the origin of x at point A, from left to right:



- Sectioning the beam at an arbitrary point $0 < x < L$:



$$\sum M_{@x}^{\text{CCW}} = 0 \quad \therefore \quad M_{(x)} - (-WL) - (W)(x) = 0 \quad \therefore \quad \boxed{M_{(x)} = W(x - L)}$$

$$\sum F_{@x} = 0 \quad \therefore \quad W + V_{(x)} = 0 \quad \therefore \quad \boxed{V_{(x)} = -W}$$

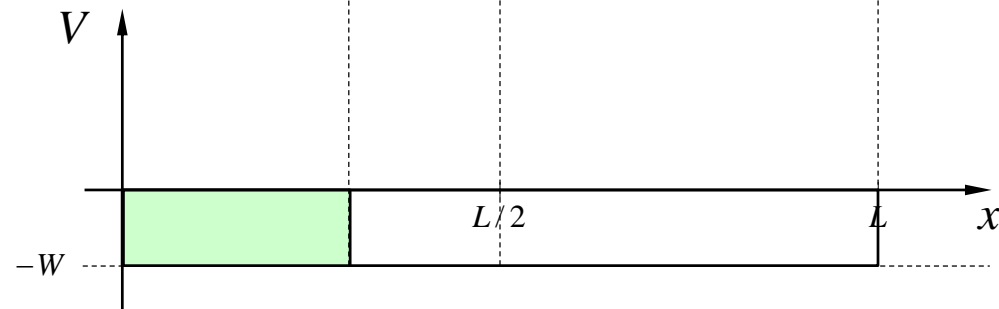
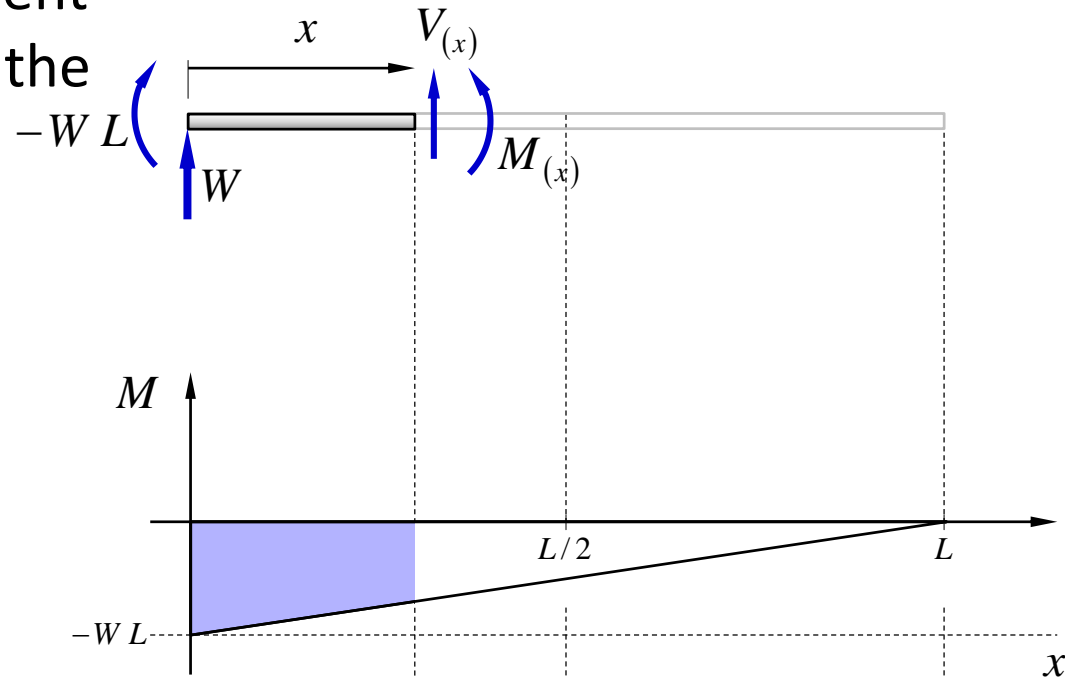
- Note that the bending moment **varies linearly** with x , while the shear force is a **constant**:

$$M_{(x)} = W(x - L)$$

$$V_{(x)} = -W$$

- So we can plot the diagrams:

These solutions are valid for
any $0 < x < L$



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