Air Standard Cycles (Lecture 1/4)

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<u>Objectives</u>

Understand steps in deriving Otto cycle efficiency, and implications.

Air Standard Cycles (Lecture 1/4)

1) Preamble

Engines.

- working fluid ... ideal gas,
- closed cycles ... gas is recirculated
- open cycles ... gas enters and exits

Internal combustion – fuel + air
External combustion - heat transfer

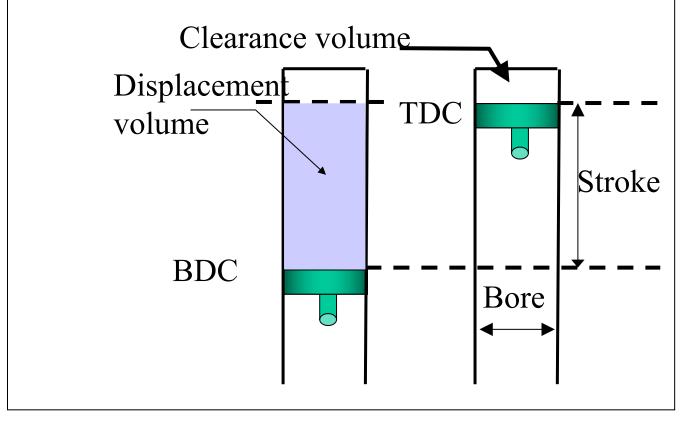
Here ... idealised cycles with no internal irreversibilities

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2) Air standard cycles – assumptions:

- ideal gas with air properties;
- gas circulates in a closed loop;
- heat inputs approximates combustion;
- heat rejection approximates exhaust;
- constant heat capacities (optional)

<u>Piston cylinder -</u>



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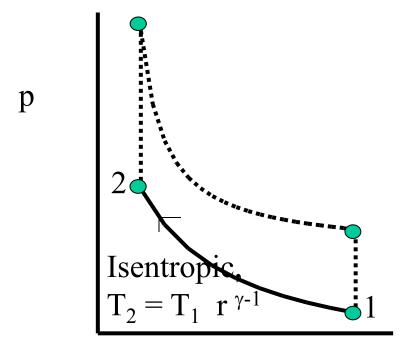
Compression ratio,

$$r = \frac{V_{\text{max}}}{V_{\text{min}}} \tag{1}$$

2) Otto Cycle (4-stroke, Spark ignition)

Compression stroke (1-to-2)

Piston moves BDC to TDC, isentropic compression

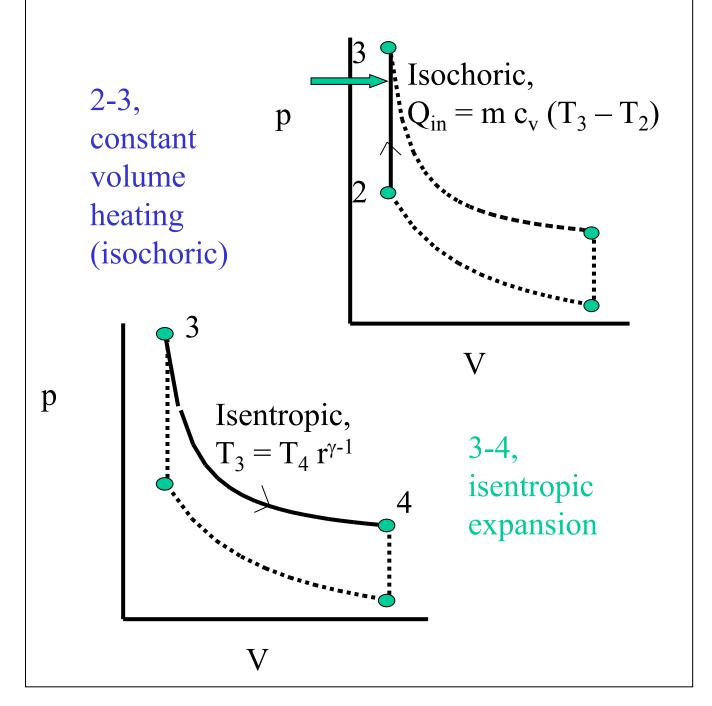


V

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Power stroke (2-to-3 and 3-to-4)

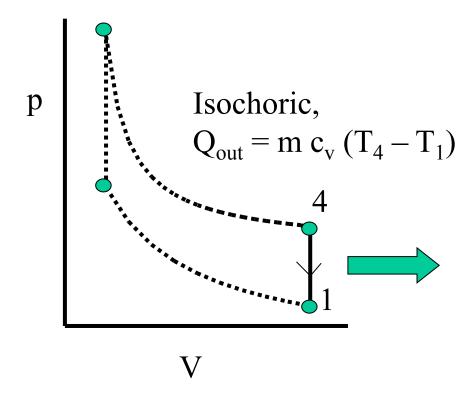
Ignition, then piston moves TDC to BDC



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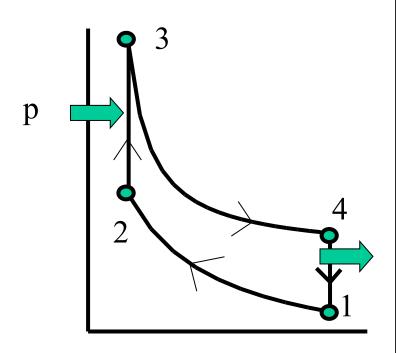
Exhaust stroke and intake stroke (4-to-1)

- Expel combustion gas (BDC to TDC)
- Air induction (TDC to BDC)
- Represent the two strokes with a <u>single</u> isochoric cooling process



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Complete cycle



$$\eta = 1 - \frac{Q_{\text{out}}}{Q_{\text{in}}} = 1 - \frac{m c_v (T_4 - T_1)}{m c_v (T_3 - T_2)} = 1 - \frac{(T_4 - T_1)}{(T_3 - T_2)}$$
(2)

Isentropic eqns yield temps at TDC, T2 & T3

$$\eta = 1 - \frac{T_4 - T_1}{T_3 - T_2} = 1 - \frac{T_4 - T_1}{T_4 r^{\gamma - 1} - T_1 r^{\gamma - 1}} = 1 - \frac{1}{r^{\gamma - 1}}$$
(3)

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The efficiency depends only on the compression ratio, r.

Large compression ratios yield better efficiency, but ... "knocking"

Exam + coursework –often W, Q for each process. Derived efficiency as a check (discussion part)

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Conclusions

Complex engine processes simplified (grossly) with air standard assumptions.

Otto represented by two isentropic and two isochoric processes.