

## StM3 – Composite Laminate Analysis

Lecture 2 :

- macromechanics of uni-directional lamina

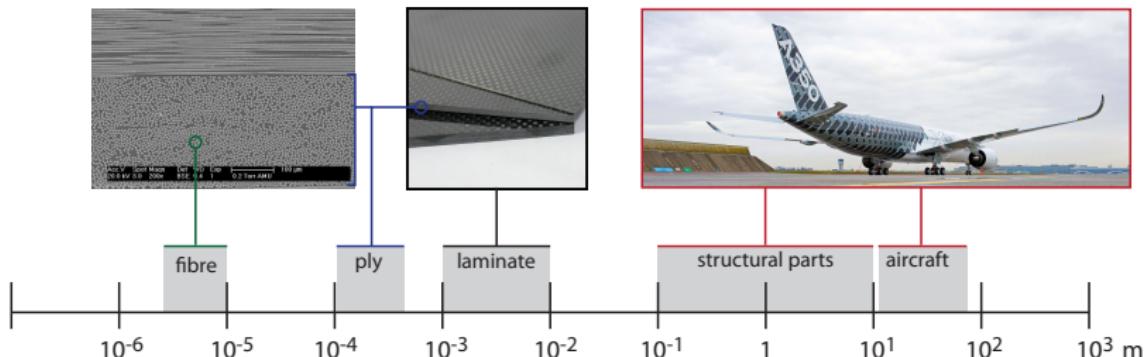
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Mark Schenk – M.Schenk@bristol.ac.uk

# Unit Content

## Composite Laminate Analysis (CLA)

learning outcome: able to describe and analyse *structural* properties of a fibre reinforced composite laminate material

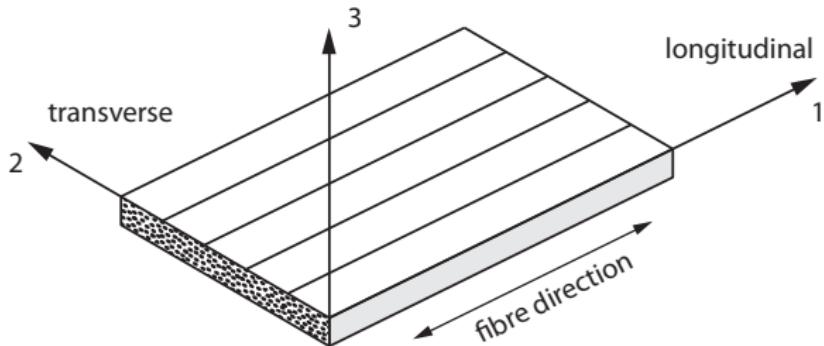


## Lecture Outline

- derive material model for a specially orthotropic ply
- revise stress/strain transformation equations (StM2)
- derive material model for an angled ply

## Composite Lamina: Plane Stress – I

a unidirectional composite lamina is **anisotropic**: mechanical properties differ along direction of fibres and perpendicular to fibres



formulate constitutive equations in natural/material axes (123)

## Composite Lamina: Plane Stress – II

each individual lamina is assumed to be loaded under *plane stress*

constitutive equations reduce to *specially orthotropic* material:

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{bmatrix} = \begin{bmatrix} Q_{11} & Q_{12} & 0 \\ Q_{12} & Q_{22} & 0 \\ 0 & 0 & Q_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix}$$

$Q_{ij}$  components are known as **reduced stiffnesses**

## Composite Lamina: Plane Stress – III

find  $\mathbf{Q}$  by inverting the plane stress **compliance** matrix  $\mathbf{S}$ :

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} S_{11} & S_{12} & 0 \\ S_{12} & S_{22} & 0 \\ 0 & 0 & S_{66} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{bmatrix}$$

to give:

$$Q_{11} = \frac{S_{22}}{S_{11}S_{22} - S_{12}^2} \quad Q_{22} = \frac{S_{11}}{S_{11}S_{22} - S_{12}^2}$$

$$Q_{12} = -\frac{S_{12}}{S_{11}S_{22} - S_{12}^2} \quad Q_{66} = \frac{1}{S_{66}}$$

## Composite Lamina: Plane Stress – IV

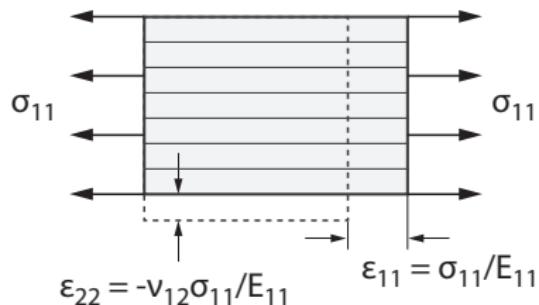
compliance matrix is constructed by considering three load cases:

1. **uni-axial stress**  $\sigma_{11}$  (with  $\sigma_{22} = \tau_{12} = 0$ ):

$$\varepsilon_{11} = \frac{\sigma_{11}}{E_{11}}$$

$$\varepsilon_{22} = -\nu_{12}\varepsilon_{11} = -\nu_{12}\frac{\sigma_{11}}{E_{11}}$$

$$\gamma_{12} = 0$$



definition of the Poisson's ratio  $\nu_{ij}$

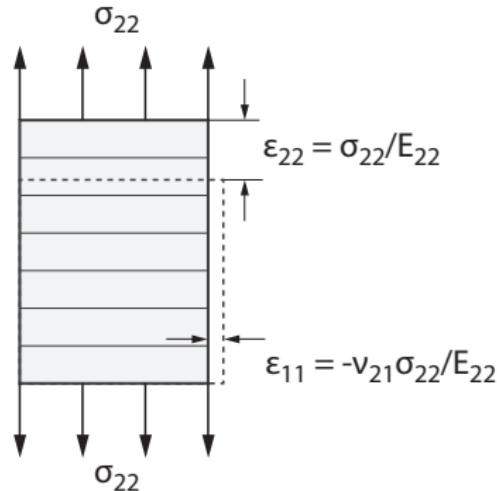
## Composite Lamina: Plane Stress – V

2. uni-axial stress  $\sigma_{22}$  (with  $\sigma_{11} = \tau_{12} = 0$ ):

$$\varepsilon_{11} = -\nu_{21}\varepsilon_{22} = -\nu_{21} \frac{\sigma_{22}}{E_{22}}$$

$$\varepsilon_{22} = \frac{\sigma_{22}}{E_{22}}$$

$$\gamma_{12} = 0$$



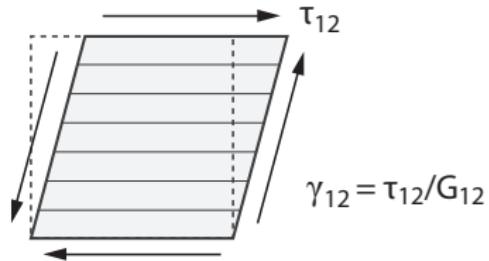
## Composite Lamina: Plane Stress – VI

3. pure shear  $\tau_{12}$  (with  $\sigma_{11} = \sigma_{22} = 0$ ):

$$\varepsilon_{11} = 0$$

$$\varepsilon_{22} = 0$$

$$\gamma_{12} = \frac{\tau_{12}}{G_{12}}$$



## Composite Lamina: Plane Stress – VII

each load case provides column of  $\mathbf{S}$

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} \frac{1}{E_{11}} & -\frac{\nu_{21}}{E_{22}} & 0 \\ -\frac{\nu_{12}}{E_{11}} & \frac{1}{E_{22}} & 0 \\ 0 & 0 & \frac{1}{G_{12}} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{bmatrix}$$

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symmetry of compliance matrix ( $S_{12} = S_{21}$ ) gives:

$$\frac{\nu_{21}}{E_{22}} = \frac{\nu_{12}}{E_{11}}$$

specially orthotropic material in plane stress:  $E_{11}$ ,  $E_{22}$ ,  $\nu_{12}$ ,  $G_{12}$

## Summary: Specially Orthotropic Lamina – I

*specially orthotropic:* coordinate system aligned with natural axes  
compliance matrix  $\mathbf{S}$

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} S_{11} & S_{12} & 0 \\ S_{12} & S_{22} & 0 \\ 0 & 0 & S_{66} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{bmatrix}$$

with

$$S_{11} = \frac{1}{E_{11}} \qquad \qquad S_{22} = \frac{1}{E_{22}}$$

$$S_{12} = -\frac{\nu_{12}}{E_{11}} = -\frac{\nu_{21}}{E_{22}} \qquad S_{66} = \frac{1}{G_{12}}$$

## Summary: Specially Orthotropic Lamina – II

reduced stiffness matrix  $\mathbf{Q}$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{bmatrix} = \begin{bmatrix} Q_{11} & Q_{12} & 0 \\ Q_{12} & Q_{22} & 0 \\ 0 & 0 & Q_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix}$$

with

$$Q_{11} = \frac{E_{11}}{1 - \nu_{12}\nu_{21}}$$

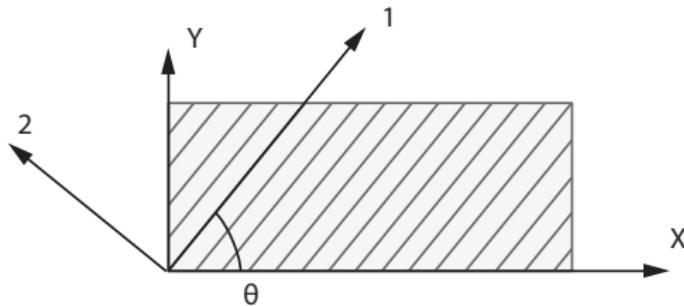
$$Q_{22} = \frac{E_{22}}{1 - \nu_{12}\nu_{21}}$$

$$Q_{12} = \frac{\nu_{12}E_{22}}{1 - \nu_{12}\nu_{21}} = \frac{\nu_{21}E_{11}}{1 - \nu_{12}\nu_{21}}$$

$$Q_{66} = G_{12}$$

## Angled Plies

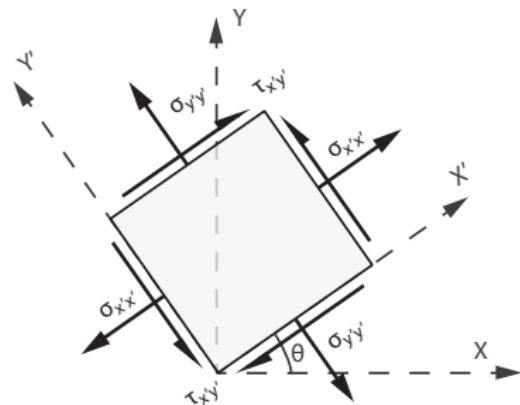
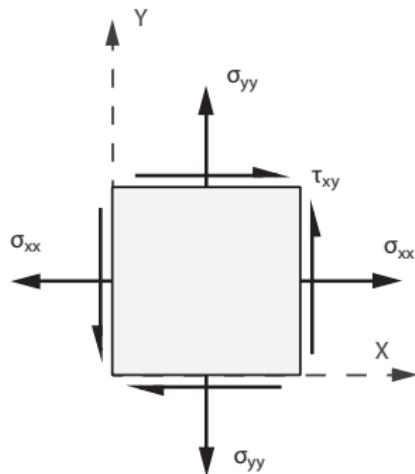
angled plies: generally, in composites the material axes (123) are not aligned with structural axes (xyz)



transform stress/strain between natural and structural axes

# Revision: Stress/Strain Transformation – I

stresses in  $x'y'$  coordinate system, at CCW angle  $\theta$  to  $xy$  axes



## Revision: Stress/Strain Transformation – II

calculate using transformation matrix  $T$ :

$$\begin{bmatrix} \sigma_{x'x'} \\ \sigma_{y'y'} \\ \tau_{x'y'} \end{bmatrix} = \begin{bmatrix} \cos^2 \theta & \sin^2 \theta & 2 \sin \theta \cos \theta \\ \sin^2 \theta & \cos^2 \theta & -2 \sin \theta \cos \theta \\ -\sin \theta \cos \theta & \sin \theta \cos \theta & \cos^2 \theta - \sin^2 \theta \end{bmatrix} \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{bmatrix}$$

coordinate transformation matrix for second-rank tensor

## Revision: Stress/Strain Transformation – III

remarkably, in-plane strain transforms identically:

$$\begin{bmatrix} \varepsilon_{x'x'} \\ \varepsilon_{y'y'} \\ \varepsilon_{x'y'} \end{bmatrix} = \begin{bmatrix} \cos^2 \theta & \sin^2 \theta & 2 \sin \theta \cos \theta \\ \sin^2 \theta & \cos^2 \theta & -2 \sin \theta \cos \theta \\ -\sin \theta \cos \theta & \sin \theta \cos \theta & \cos^2 \theta - \sin^2 \theta \end{bmatrix} \begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{xy} \end{bmatrix}$$

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convert *stress* and *strain* between **structural axes** (defined by applied loads) and **material axes** (defined by fibre orientations)

## Revision: Stress/Strain Transformation – IV

NB: tensor transformation for *tensorial* strain  $\varepsilon_{xy} = \gamma_{xy}/2$

introduce Reuter's matrix  $R$

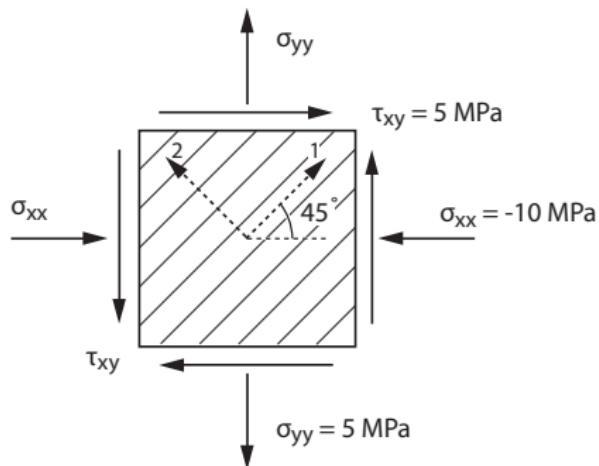
$$R = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 2 \end{bmatrix}$$

*engineering* strain transformation becomes:

$$\begin{bmatrix} \varepsilon_{x'x'} \\ \varepsilon_{y'y'} \\ \gamma_{x'y'} \end{bmatrix} = RTR^{-1} \begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{bmatrix}$$

## Example 1.1 - Lamina Stress/Strain – I

angled composite ply with  $E_{11} = 180 \text{ GPa}$ ,  $E_{22} = 10 \text{ GPa}$ ,  $\nu_{12} = 0.2$ ,  $G_{12} = 5 \text{ GPa}$ , is subject to the following loads:



**Q:** what are the resulting strains?

## Example 1.1 - Lamina Stress/Strain – II

**A:** first, transform applied stresses from structural into material coordinate system:

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{bmatrix} = \boldsymbol{T} \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{bmatrix} = \begin{bmatrix} 2.5 \\ -7.5 \\ 7.5 \end{bmatrix} \text{ MPa}$$

using transformation matrix  $\boldsymbol{T}$  with  $\theta = 45^\circ$

$$\boldsymbol{T} = \begin{bmatrix} 0.5 & 0.5 & 1 \\ 0.5 & 0.5 & -1 \\ -0.5 & 0.5 & 0 \end{bmatrix}$$

## Example 1.1 - Lamina Stress/Strain – III

next, using compliance matrix  $\mathbf{S}$  and Reuter's matrix  $\mathbf{R}$

$$\mathbf{R} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 2 \end{bmatrix} \quad \mathbf{S} = \begin{bmatrix} 1/E_{11} & -\nu_{21}/E_{22} & 0 \\ -\nu_{12}/E_{11} & 1/E_{22} & 0 \\ 0 & 0 & 1/G_{12} \end{bmatrix}$$

calculate strains in material coordinate system:

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \mathbf{S} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{bmatrix} = \begin{bmatrix} 22 \\ -753 \\ 1500 \end{bmatrix} \mu\varepsilon$$

## Example 1.1 - Lamina Stress/Strain – IV

convert strains back into structural reference frame:

$$\begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{bmatrix} = \mathbf{R} \mathbf{T}^{-1} \mathbf{R}^{-1} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{bmatrix} = \begin{bmatrix} -1115 \\ 385 \\ 775 \end{bmatrix} \mu\varepsilon$$

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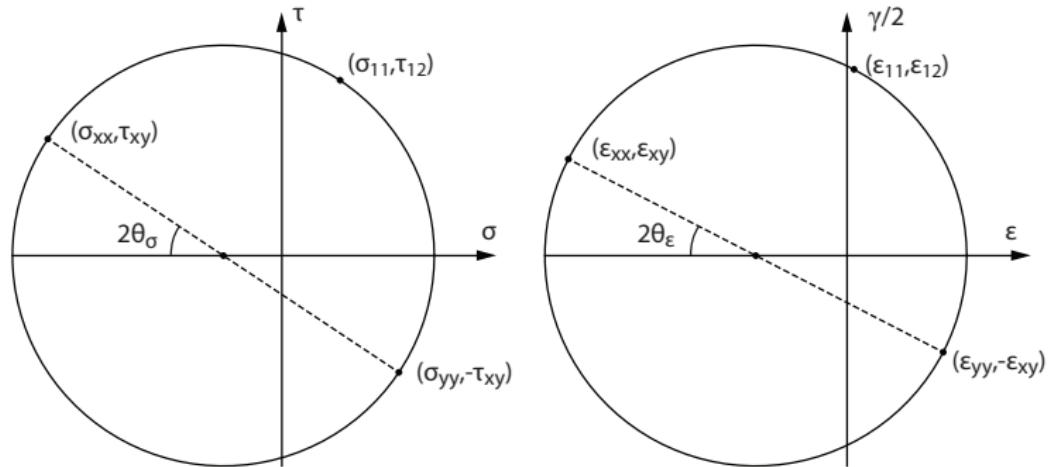
NB: calculate  $\mathbf{T}^{-1}$  as a CW  $\theta$  rotation – do not invert!

$$\mathbf{T} = \begin{bmatrix} c^2 & s^2 & 2sc \\ s^2 & c^2 & -2sc \\ -sc & sc & c^2 - s^2 \end{bmatrix}; \quad \mathbf{T}^{-1} = \begin{bmatrix} c^2 & s^2 & -2sc \\ s^2 & c^2 & 2sc \\ sc & -sc & c^2 - s^2 \end{bmatrix}$$

where  $s = \sin \theta$  and  $c = \cos \theta$

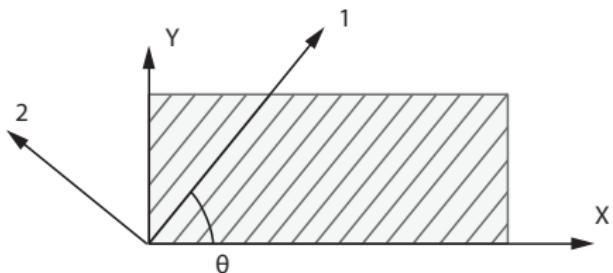
## Example 1.1 - Lamina Stress/Strain – V

Mohr's circle for stress *and* strain are insightful



principal directions for stress and strain are not aligned!

## Generally Orthotropic Material – I



constitutive equations in structural axes: *generally orthotropic*

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{bmatrix} = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{26} \\ \bar{Q}_{16} & \bar{Q}_{26} & \bar{Q}_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{bmatrix}$$

$$\bar{\mathbf{Q}} = \mathbf{T}^{-1} \mathbf{Q} \mathbf{R} \mathbf{T} \mathbf{R}^{-1}$$

## Generally Orthotropic Material – II

with components  $\bar{Q}_{ij}$

$$\bar{Q}_{11} = Q_{11} \cos^4 \theta + 2(Q_{12} + 2Q_{66}) \sin^2 \theta \cos^2 \theta + Q_{22} \sin^4 \theta$$

$$\bar{Q}_{22} = Q_{11} \sin^4 \theta + 2(Q_{12} + 2Q_{66}) \sin^2 \theta \cos^2 \theta + Q_{22} \cos^4 \theta$$

$$\bar{Q}_{12} = (Q_{11} + Q_{22} - 4Q_{66}) \sin^2 \theta \cos^2 \theta + Q_{12} (\sin^4 \theta + \cos^4 \theta)$$

$$\bar{Q}_{66} = (Q_{11} + Q_{22} - 2Q_{12} - 2Q_{66}) \sin^2 \theta \cos^2 \theta + Q_{66} (\sin^4 \theta + \cos^4 \theta)$$

$$\bar{Q}_{16} = (Q_{11} - Q_{12} - 2Q_{66}) \sin \theta \cos^3 \theta - (Q_{22} - Q_{12} - 2Q_{66}) \cos \theta \sin^3 \theta$$

$$\bar{Q}_{26} = (Q_{11} - Q_{12} - 2Q_{66}) \cos \theta \sin^3 \theta - (Q_{22} - Q_{12} - 2Q_{66}) \sin \theta \cos^3 \theta$$

## Generally Orthotropic Material – III

ply stiffness matrix  $\bar{\mathbf{Q}}$  is symmetric and fully populated:

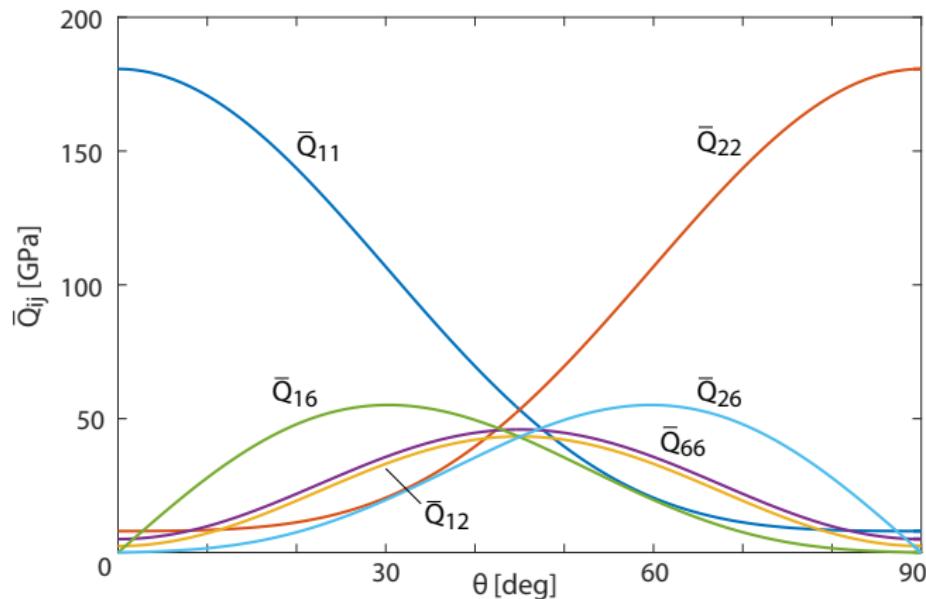
$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{bmatrix} = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} \\ \bar{Q}_{12} & \bar{Q}_{22} & \bar{Q}_{26} \\ \bar{Q}_{16} & \bar{Q}_{26} & \bar{Q}_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{bmatrix}$$

where  $\bar{Q}_{16}$  and  $\bar{Q}_{26}$  provide direct/shear coupling

NB:  $\bar{\mathbf{Q}}$  matrix contains only four independent mechanical constants of a specially orthotropic material ( $E_{11}$ ,  $E_{22}$ ,  $G_{12}$  and  $\nu_{12}$ )

## Example 1.2 - Stiffness Transformation

carbon/epoxy composite with reduced stiffness  $\bar{Q}_{11} = 180.7$  GPa,  
 $\bar{Q}_{12} = 2.41$  GPa,  $\bar{Q}_{22} = 8$  GPa and  $\bar{Q}_{66} = 5$  GPa



## Generally Orthotropic Material – I

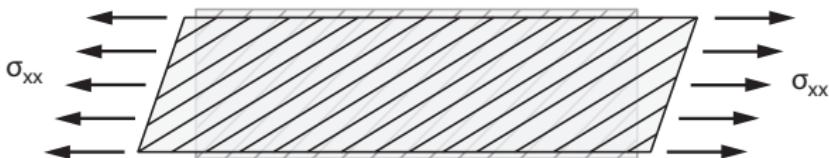
similar transformation for compliance matrix:

$$\begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{bmatrix} = \begin{bmatrix} \bar{S}_{11} & \bar{S}_{12} & \bar{S}_{16} \\ \bar{S}_{12} & \bar{S}_{22} & \bar{S}_{26} \\ \bar{S}_{16} & \bar{S}_{26} & \bar{S}_{66} \end{bmatrix} \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{bmatrix}$$

$$\bar{\mathbf{S}} = \mathbf{R}\mathbf{T}^{-1}\mathbf{R}^{-1}\mathbf{S}\mathbf{T}$$

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due to non-zero  $\bar{S}_{12}$  and  $\bar{S}_{16}$  direct stress results in shear strain:



## Generally Orthotropic Material – II

with components  $\bar{S}_{ij}$

$$\bar{S}_{11} = S_{11} \cos^4 \theta + (2S_{12} + S_{66}) \sin^2 \theta \cos^2 \theta + S_{22} \sin^4 \theta$$

$$\bar{S}_{22} = S_{11} \sin^4 \theta + (2S_{12} + S_{66}) \sin^2 \theta \cos^2 \theta + S_{22} \cos^4 \theta$$

$$\bar{S}_{12} = (S_{11} + S_{22} - S_{66}) \sin^2 \theta \cos^2 \theta + S_{12} (\sin^4 \theta + \cos^4 \theta)$$

$$\bar{S}_{66} = 2(2S_{11} + 2S_{22} - 4S_{12} - S_{66}) \sin^2 \theta \cos^2 \theta + S_{66} (\sin^4 \theta + \cos^4 \theta)$$

$$\bar{S}_{16} = (2S_{11} - 2S_{12} - S_{66}) \sin \theta \cos^3 \theta - (2S_{22} - 2S_{12} - S_{66}) \cos \theta \sin^3 \theta$$

$$\bar{S}_{26} = (2S_{11} - 2S_{12} - S_{66}) \cos \theta \sin^3 \theta - (2S_{22} - 2S_{12} - S_{66}) \sin \theta \cos^3 \theta$$

## Engineering Constants – I

compliance matrix  $\bar{\mathbf{S}}$  for angled ply

$$\begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{bmatrix} = \begin{bmatrix} \bar{S}_{11} & \bar{S}_{12} & \bar{S}_{16} \\ \bar{S}_{12} & \bar{S}_{22} & \bar{S}_{26} \\ \bar{S}_{16} & \bar{S}_{26} & \bar{S}_{66} \end{bmatrix} \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{bmatrix}$$

discussing properties of ply in terms of  $\bar{S}_{11}$ ,  $\bar{S}_{12}$ , etc. not intuitive

use effective engineering constants:

$$\begin{aligned} \varepsilon_{xx} &= \frac{\sigma_{xx}}{E_{xx}} - \nu_{yx} \frac{\sigma_{yy}}{E_{yy}} + m_{x,xy} \frac{\tau_{xy}}{G_{xy}} \\ &= \bar{S}_{11}\sigma_{xx} + \bar{S}_{12}\sigma_{yy} + \bar{S}_{16}\tau_{xy} \end{aligned}$$

## Engineering Constants – II

effective engineering constants:

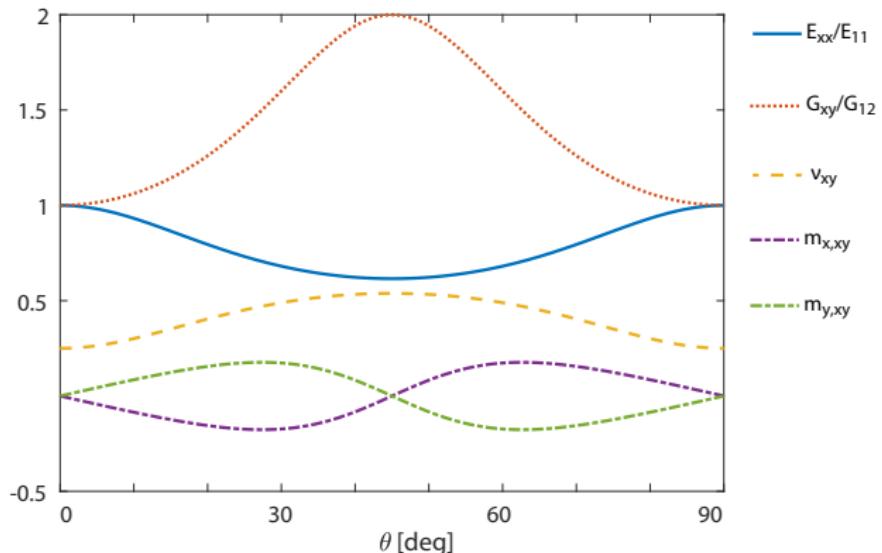
$$E_{xx} = \frac{1}{\bar{S}_{11}} \quad E_{yy} = \frac{1}{\bar{S}_{22}}$$

$$G_{xy} = \frac{1}{\bar{S}_{66}} \quad \nu_{xy} = -\frac{\bar{S}_{12}}{\bar{S}_{11}}$$

coefficients of mutual influence  $m$  describe direct/shear coupling

## Example 1.3 – Engineering Constants Transformation – I

$E_{11} = E_{22} = 100$  GPa, and  $\nu_{12} = \nu_{21} = 0.25$ , and  $G_{12} = 20$  GPa



Does not behave isotropically! Why?

## Example 1.3 – Engineering Constants Transformation – II



## Summary

- derived reduced stiffness matrix  $\mathbf{Q}$  and compliance matrix  $\mathbf{S}$
- revised plane stress and strain coordinate transformation
- derived generally orthotropic material model ( $\bar{\mathbf{Q}}$  and  $\bar{\mathbf{S}}$ )
- able to calculate stress/strain for an angled composite lamina

# Revision Objectives

## Revision Objectives:

- recognise and use composite subscript conventions ( $i, j = 1 \dots 6$ );
- derive lamina compliance matrix  $\mathbf{S}$  in terms of  $E_{11}$ ,  $E_{22}$ ,  $G_{12}$ , and  $\nu_{12}$ ;
- recall relationship between Poisson's ratio:  $\nu_{12}/E_{11} = \nu_{21}/E_{22}$ ;
- recall reduced stiffness matrix  $\mathbf{Q}$  in terms of 4 elastic constants;
- apply tensor transformation  $\mathbf{T}$  for plane stress and strain;
- derive transformed reduced stiffness matrix:  $\bar{\mathbf{Q}} = \mathbf{T}^{-1} \mathbf{Q} \mathbf{R} \mathbf{T}^{-1}$ ;
- derive transformed compliance matrix:  $\bar{\mathbf{S}} = \mathbf{R} \mathbf{T}^{-1} \mathbf{R}^{-1} \mathbf{S} \mathbf{T}$ ;
- calculate stiffness/compliance in non-fibre directions using  $\bar{\mathbf{Q}}$  and  $\bar{\mathbf{S}}$ ;
- calculate effective engineering constants from ply stiffness matrix;