

#### Lecture 4

Solutions of the characteristic equation

$$S_{1,2} = -\frac{c}{2m} \pm \left(\frac{c^2}{4m^2} - \frac{k}{m}\right)^{1/2}$$

Underdamped vibration

$$s_{1,2} = -\zeta \omega_0 \pm i \omega_0 \sqrt{1 - \zeta^2}$$

$$x(t) = C e^{-\zeta \omega_0 t} \cos(\underline{\omega_D} t - \phi)$$

$$x(t) = e^{-\zeta \omega_0 t} (A \cos(\underline{\omega_D} t) + B \sin(\underline{\omega_D} t))$$

New dynamic parameters

$$\omega_D = \omega_0 \sqrt{1 - \zeta^2}$$

$$\zeta = \frac{c}{2\sqrt{mk}}$$

Logarithmic decrement and damping

$$\zeta_{\text{exp}} \approx \frac{1}{2\pi N} \ln \left( \frac{x(t_1)}{x(t_1 + NT_D)} \right)$$



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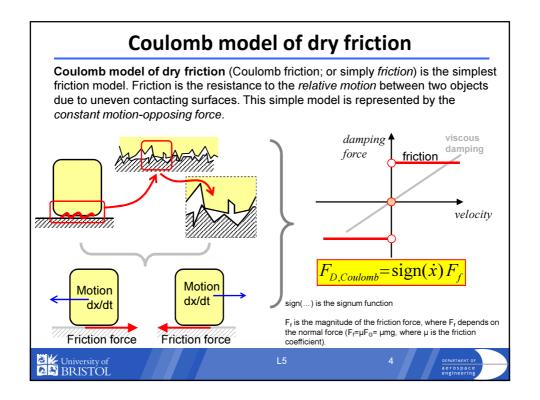
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#### **Lecture 5**

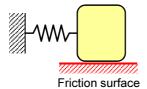
- · Coulomb model of dry friction
- · Free vibration with friction
- Solved example

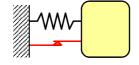




#### **Coulomb friction**

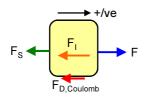
We can use alternative ways to indicate the presence of friction in our problems or model sketches:





Discrete friction element

Equation of motion:



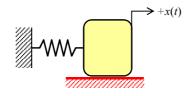
$$F_I + \overline{F_{D,Coulomb}} + F_S = F$$

$$m\ddot{x} + \frac{\operatorname{sign}(\dot{x})F_f}{\operatorname{sign}(\dot{x})F_f} + kx = F$$

Numerical algorithms don't like sign(.) function!

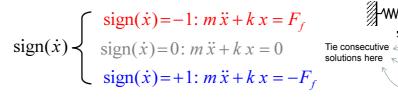
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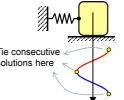
## Free vibration with Coulomb friction



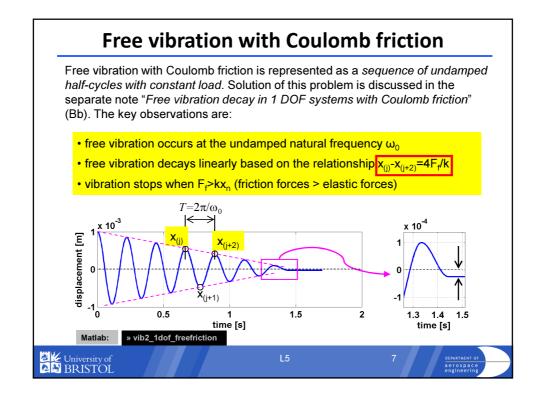
$$m\ddot{x} + \operatorname{sign}(\dot{x})F_f + kx = 0$$

Function sign has the three possible discrete values -1, 0 and 1. Thus, all three cases have to be considered independently:

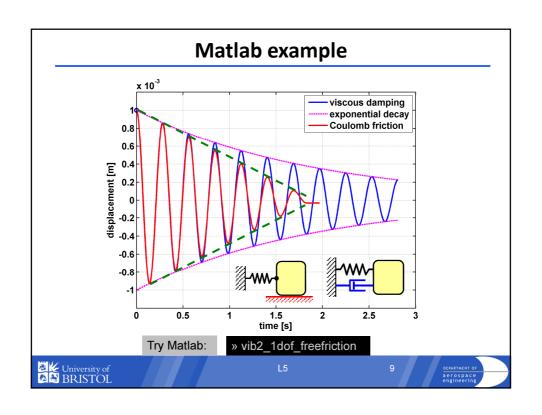




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| Friction versus viscous damping       |  |   |
|---------------------------------------|--|---|
|                                       | Viscous damping                          | Coulomb friction  |
| Natural<br>frequency                  | $\omega_0 \sqrt{1-\zeta^2}$              | $\omega_0$  |
| Vibration decay                       | exponential $\exp(-\zeta\omega_0t)$      | $-\frac{4F_f/k}{2\pi/\omega_0}t = -\frac{2F_f\omega_0t}{\pi k}$ |
| Energy<br>dissipated per<br>one cycle | $\int_0^T f_D v dt = \pi \omega c X_0^2$ | $\int_0^T f_D v dt = 4 F_f X_0$                                 |
| Theoretical duration of motion        | Infinite                                 | Finite  |



# **Example: free vibration with friction**

The mass block shown in Fig. 1 is displaced 10 mm and released. How many cycles of motion will be executed? Assume friction due to the normal force  $F_N = mg, \, m = 1 \, kg, \, g = 9.81 \, m.s^{-2},$  and the friction coefficient is  $\mu = 0.12$ . Stiffness of the spring is 10 kN/m.

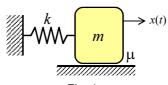


Fig. 1

Equation of motion:  $m\ddot{x} + kx = \pm F_f \implies \ddot{x} + \frac{\omega_0^2}{\omega_0^2}x = \pm F_f/m$ 

Friction force:

$$F_f = \mu F_N = \mu (mg) = 0.12 \times (1 \, kg) \times (9.81 \, ms^{-2}) = 1.18 \, N$$

Amplitude decrease per one full cycle:

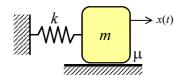
$$x_{(j)} - x_{(j+2)} = 4 F_f/k = 4 \times (1.18 N)/(10000 N/m) = 0.47 mm$$



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## **Example**



Motion stops when the spring force cannot overcome the friction force:

$$F_f \ge F_K = k \, x_{stop} \implies x_{stop} = F_f/k = (1.18 \, N)/(10000 \, N/m) = 0.118 \, mm$$

Condition for the number of cycles from the initial 10 mm to the displacement when the system is "locked":

$$\frac{x_0 - N_{cyc} (x_{(j)} - x_{(j+2)}) \le x_{stop}}{x_{(j)} - x_{(j+2)}} \Rightarrow N_{cyc} \ge \frac{x_0 - x_{stop}}{x_{(j)} - x_{(j+2)}}$$

$$N_{cyc} \ge \frac{x_0 - x_{stop}}{x_{(j)} - x_{(j+2)}} = \frac{10 \, mm - 0.118 \, mm}{0.47 \, mm} = 21.02 \Rightarrow N_{cyc} = 22 \, cycles$$



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## **Summary**

- · Coulomb friction model
- Free vibration with friction:
  - decay rate and decay trend
  - natural frequency
  - convergence to the equilibrium position
- · Comparison with viscous damping



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