Diagnosis, Fault-tolerant and Robust Control for a Torsional Control System

Mandatory assignment in DTU course 31320

Part A

Dimitrios Papageorgiou

February, 2022

Table 1: Revision history version changes date description 10.02.2022 all pages new new

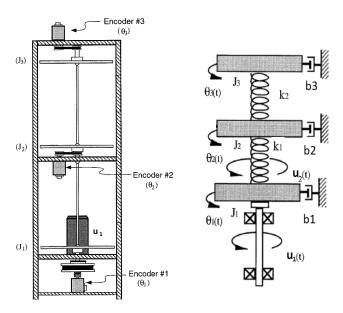
1.a

Assignment part A - 2022

Introduction

This exercise presents a model of the ECP M205a system. It is a three-mass system that comprises three identical disks, connected in parallel via a flexible low-damped shaft. The system is equipped with three incremental position encoders (one for each disk) and two actuators, i.e. a motor applying torques to the bottom disk and one more for the middle disk.

The control goal is to ensure that either of the disks tracks a specified motion profile. The system considered is sketched in Figure 1. The variables are explained in Table 2.



The normal behaviours of the systems are described by the following con-

Table 2: List of variables

variable	\mathbf{unit}	description	
$ heta_1$	rad	angular position of bottom disk	
ω_1	$\rm rads^{-1}$	angular velocity of bottom disk	
$ heta_2$	rad	angular position of middle disk	
ω_2	$\rm rads^{-1}$	angular velocity of middle disk	
$ heta_3$	rad	angular position of top disk	
ω_3	$\rm rads^{-1}$	angular velocity of top disk	
u_1	Nm	torque command for the bottom disk	
u_2	Nm	torque command for the middle disk	
y_1	rad	measured angular position of bottom disk	
y_2	rad	measured angular position of middle disk	
y_3	rad	measured angular position of top disk	

straints:

$$c_{1}: 0 = \dot{\theta}_{1} - \omega_{1}$$

$$c_{2}: 0 = J_{1}\dot{\omega}_{1} - u_{1} + b_{1}\omega_{1} + k_{1} (\theta_{1} - \theta_{2}) + d$$

$$c_{3}: 0 = \dot{\theta}_{2} - \omega_{2}$$

$$c_{4}: 0 = J_{2}\dot{\omega}_{2} - u_{2} + b_{2}\omega_{2} + k_{1} (\theta_{2} - \theta_{1}) + k_{2} (\theta_{2} - \theta_{3})$$

$$c_{5}: 0 = \dot{\theta}_{3} - \omega_{3}$$

$$c_{6}: 0 = J_{3}\dot{\omega}_{3} + b_{3}\omega_{3} + k_{2} (\theta_{3} - \theta_{2})$$

$$d_{7}: 0 = \dot{\theta}_{1} - \frac{d\theta_{1}}{dt}$$

$$d_{8}: 0 = \dot{\omega}_{1} - \frac{d\omega_{1}}{dt}$$

$$d_{9}: 0 = \dot{\theta}_{2} - \frac{d\theta_{2}}{dt}$$

$$d_{10}: 0 = \dot{\omega}_{2} - \frac{d\omega_{2}}{dt}$$

$$d_{11}: 0 = \dot{\theta}_{3} - \frac{d\theta_{3}}{dt}$$

$$d_{12}: 0 = \dot{\omega}_{3} - \frac{d\omega_{3}}{dt}$$

$$m_{13}: 0 = y_{1} - \theta_{1}$$

$$m_{14}: 0 = y_{2} - \theta_{2}$$

$$m_{15}: 0 = y_{3} - \theta_{3}$$

where $d \triangleq T_C(\omega_1)$ is a function of the bottom disk angular velocity and represents the *unknown* Coulomb friction torque on that disk. The control input saturates at 2 Nm, i.e. $u_1 \in [-2,2]$ Nm. The parameters in the forgoing con-

straints are listed in Table 3.

Table 3: List of parameters

Table 9. List of parameters				
${f symbol}$	value	unit	$\operatorname{description}$	
$\overline{J_1}$	0.0025	$\mathrm{kgm^2}$	Bottom disk moment of inertia	
J_2	0.0018	$\mathrm{kgm^2}$	Middle disk moment of inertia	
J_3	0.0018	$\mathrm{kgm^2}$	Top disk moment of inertia	
k_1	2.7	$Nmrad^{-1}$	Stiffness of the bottom shaft	
k_2	2.6	$Nmrad^{-1}$	Stiffness of the middle shaft	
b_1	0.0029	$Nmsrad^{-1}$	Damping/friction on the bottom disk	
b_2	0.0002	$Nmsrad^{-1}$	Damping/friction on the middle disk	
b_3	0.00015	$Nmsrad^{-1}$	Damping/friction on the top disk	

A Simulink model and a parameter initialisation file have been uploaded to DTU Learn in the $File\ share/Matlab\ and\ Simulink\ files/Mandatory\ Assignment\ Part\ A$ folder for your convenience.

Question 1

Make a structural analysis:

- 1. Determine a complete matching on the unknown variables.
- 2. Find the parity relations in symbolic form.
- 3. Investigate other properties you find relevant from a structural analysis.
- 4. Reformulate the parity relations to an analytic form, as functions only of known variables and the system parameters.

Question 2

- 1. Design a set of proper residual generators, i.e. residual generators that are stable and are causal.
- 2. Discretise your residuals with sampling period $T_s = 4$ ms.
- 3. Implement your residual generators using a Simulink model of the system and demonstrate by simulation (in open loop) that your residuals are insensitive to changes of control inputs.

Question 3 (Experimental work)

A single sensor additive fault afflicted the measurements.

- 1. Test the implemented residual generators on the ECP M502a torsional system. Demonstrate their fault-detection properties.
- 2. Which of the three sensors is afflicted by the fault? Justify your answer.
- 3. Are the residuals insensitive to input changes? Comment on the results.

Hint: In order to load the recorded inputs and outputs in Simulink, you can set them in a timeseries structure (Matlab command timeseries()) and use the "From Workspace" block in Simulunk.

Question 4

Faults are possible on any of the sensors or at the actuator.

1. Model these as additive faults and write the system in the standard form

$$\dot{x} = Ax + Bu + E_x d + F_x f$$

 $y = Cx + Du + E_y d + F_y f$

where the matrices need be determined and the vector \boldsymbol{f} contains all actuator and sensor faults.

- 2. Determine the transfer function $H_{rf}(s)$ from faults to residuals in your LTI design.
- 3. Investigate strong and weak detectability of the faults.
- 4. What would change in terms of fault detectability if the Coulomb friction function $T_C(\omega_1)$ were known?

Hints: For this task it is useful to find a basis for the left nullspace by using the Matlab symbolic toolbox in defining the various transfer functions (e.g. syms s; G = 1/(s + 1)). You may find the following functions useful:

- simplify simplifies symbolic expression,
- expand expands symbolic expression,
- numden extracts enumerator and denominator of symbolic fraction,
- sym2poly converts symbolic polynomial to numeric,
- minreal gives a minimal realization of a transfer function,
- zpk expresses a transfer function as a zero-pole-gain product.

Question 5

The sensors have measurement noise. The noise of individual sensors is uncorrelated, i.e.

$$\begin{bmatrix} y_1 \\ y_2 \\ y_3 \end{bmatrix} \sim \mathcal{N} \left(\begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}, \begin{bmatrix} \sigma_1^2 & 0 & 0 \\ 0 & \sigma_2^2 & 0 \\ 0 & 0 & \sigma_3^2 \end{bmatrix} \right) \tag{1}$$

with $\sigma_1 = \sigma_2 = \sigma_3 = 0.0093$ rad.

Choose a residual generator that is sensitive to faults on y_2 .

- 1. Calculate the variance at the output of the residual.
- 2. Design a GLR detector that can detect a sudden offset fault of unknown magnitude in sensor y_2 . Tune the GLR by determining the threshold h and the window size M, such that if the fault on y_2 has magnitude $f_2 = -0.025$ rad, then the false alarm probability will be $P_F = 0.0001$ or lower and the probability of missed detection $P_M = 0.01$ or lower.
- 3. Implement the designed GLR in Simulink and validate the design in simulation.

Question 6 (Experimental work)

Test your GLR detector on the ECP M502a torsional system. Connect the GLR block to the residual generator from Question 5 and demonstrate its robustness properties with respect to false alarms. Comment on the results.

Hint: You can use the "Matlab function" block in Simulunk from the "user-defined functions" library.

Question 7

A discrete linear quadratic regulator (DLQR) is to be designed to ensure that the top disk tracks a given step change θ_{ref} in its position. The resulting closed loop system should have the following form

$$x(k+1) = (F - GK_c)x(k) + GK_cC_{ref}\theta_{ref}(k) + E_xd(k)$$
(2)

$$y(k) = Cx(k) + E_{n}d(k) \tag{3}$$

with $F \in \mathbb{R}^{6 \times 6}$, $G \in \mathbb{R}^{6 \times 2}$ being the discretised system matrices and C_{ref} is a scaling matrix for the scalar reference signal.

1. Discretise the system with sampling period $T_s=4~\mathrm{ms}.$

2. Show that the reference scaling matrix C_{ref} is given by

$$C_{ref} = \left(C_3(I - F + GK_c)^{-1}GK_c\right)^+ \ C_3 = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 \end{bmatrix},$$

where X^+ denotes the pseudoinverse of a matrix X.

4. Implement the controller in Simulink and simulate the closed-loop system for a square wave reference of amplitude $|\theta_{ref}(k)| = \frac{\pi}{2}$ rad (from 0 to $\frac{\pi}{2}$), period 10 s and width 5 s.

Assumption: A Kalman filter with friction estimation is provided to you such that all the states are considered available for feedback.

Practical notes

- The deadline for the report is Wednesday March 16, 2022, at 23:55 hours.
- Please note that max. 20 pages are allowed for part A, excluding front matter. Any pages in excess of 20 (excluding front matter) will be discarded.
- Layout must be reasonable: A4 paper, 20mm margins and font size 11pt.
- The report need be delivered in electronic form, as a .pdf file, via the DTULearn assignment system. Only pdf files are accepted.
- Group-work is encouraged (max 2 persons in one group).
- Please do write your name(s) and student number(s) at the front page and as running heading on each page of your report. In addition, do not forget page numbers.