Introduction to 2nd-order System Response

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Outline



- 1 Motivation
- 2 Forced Response
- 3 Applied Example: Spring-Mass-Damper
 - Review: System Modeling
 - Derivation: Transfer Function and Step-Response
 - Activity: Response Comparison

2nd-order System Dynamics 2024-02-20

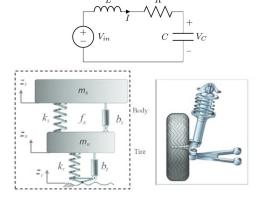
└─Outline

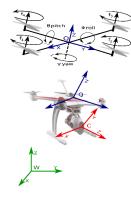
4th-wall break notes

- Lecture Objective: why 2nd-order roots of a dynamical system's can result in more **interesting responses** (i.e.) the 3 cases as a result from the quadratic equation
- Math background/assumptions:
 - Simple ODEs solutions are covered in prereq and explained again in the intro of this
 - Specifically, Laplace transform methods and the inverse-laplace via partial fraction expansion will be well known to students.
 - In a real course I'd spend time in lecture having students walk me through the derivation of the cases instead of leaving as an exercise/assignment.
- Previous lectures:
 - 1st order-system response and how time-constant plays into the system impulse and
 - Solutions to differential equations (w/in time and frequency domains)

Real-World Dynamical Systems







Step Response - 1st vs 2nd order



Step Input: $u(t) \stackrel{\mathcal{L}}{\Rightarrow} U(s) = \frac{1}{2}$

1st-order:

$$Y(s) = \frac{K}{\tau s + 1} \left(\frac{1}{s}\right) \quad \stackrel{\mathcal{L}^{-1}}{\Rightarrow} \quad y(t) = K(1 - e^{-t/\tau})\mathbf{u}(t)$$

2nd-order:

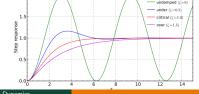
$$Y(s) = \frac{K}{(s+p_1)(s+p_2)} \left(\frac{1}{s}\right) \quad \stackrel{\mathcal{L}^{-1}}{\Rightarrow} \quad y(t) = \left(C_1 + C_2 e^{-p_1 t} + C_3 e^{-p_2 t}\right) \mathbf{u}(t)$$

3 distinct cases:

- Damped: $p_1 \neq p_2$
- Critically Damped: $p_1 = p_2$ Special Case:

$$C_2e^{-p_1t} + C_3e^{-p_2t} \to$$

 $C_2e^{-p_{1,2}t}+C_3te^{-p_{1,2}t}$



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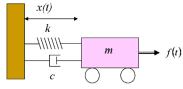
Spring Mass-Damper System Modeling



Newton's 2nd Law:

$$F = m\mathbf{a} = m\frac{\mathrm{d}}{\mathrm{d}t}\mathbf{v} = m\frac{\mathrm{d}}{\mathrm{d}t}\left(\frac{\mathrm{d}}{\mathrm{d}t}\mathbf{x}\right)$$

$$m\frac{\mathrm{d}}{\mathrm{d}t^2}x(t) = \sum F = f(t) - b\frac{\mathrm{d}}{\mathrm{d}t}x(t) - kx(t)$$



Differential Equation: $(\mathbf{x} = x(t), \mathbf{u} = f(t))$

Spring Mass Damper System [1]

 $m\ddot{\mathbf{x}}+c\dot{\mathbf{x}}+k\mathbf{x}=\mathbf{u}$ Activity: https://www.sccs.swarthmore.edu/users/12/abiele1/Linear/examples/simple.html

Transfer Function Derivation



Convert Differential Equation to Laplace: $(x(t) = \dot{x}(t) = 0)$

$$f(t) = m\ddot{x}(t) + b\dot{x}(t) + kx(t) \quad \stackrel{\mathcal{L}}{\Rightarrow} \quad F(s) = ms^2X(s) + bsX(s) + kX(s)$$

Solve for X(s) **in terms of** F(s)

$$X(s) = \frac{1}{ms^2 + bs + k} F(s) = \frac{1}{m(s^2 + \frac{b}{m}s + \frac{k}{m})} \left(\frac{k}{k}\right) F(s) = \left(\frac{1}{k}\right) \frac{\frac{k}{m}}{s^2 + \frac{b}{m}s + \frac{k}{m}} F(s)$$

Transfer Function:

$$H(s) = \frac{X(s)}{F(s)} = \left(\frac{1}{k}\right) \frac{\frac{k}{m}}{s^2 + \frac{b}{m}s + \frac{k}{m}} \Leftarrow \underbrace{F = k\Delta x}_{\text{(Hook's Law)}} \Leftarrow \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}$$
(Standard Form)

Factoring the characteristic polynomial



Apply the quadratic formula to find the roots of the characteristic polynomial:

$$\Delta(s) = ms^2 + bs + k \quad \Rightarrow \quad s = \frac{-b \pm \sqrt{b^2 - 4mk}}{2m}$$

3 Potential cases:

- **1 Damped**: $b^2 > 4mk \Rightarrow p_1 \neq p_2 \Rightarrow (s + p_1)(s + p_2)$
- 2 Critically Damped: $b^2 = 4mk \Rightarrow p_1 = p_2 \Rightarrow (s + p_{1,2})^2$
- **3 Underdamped**: $b^2 < 4mk \Rightarrow p_{1,2} = \sigma \pm j\omega$

Applied Example: Spring-Mass-Damper

Derivation: Transfer Function and Step-Response Factoring the characteristic polynomial

This motivates the standard characteristic polynomial form:

$$s^2 + 2\zeta\omega_0 s + \omega_0^2 \Rightarrow s = \zeta\omega_0 \pm \sqrt{(\zeta\omega_0)^2 - \omega_0^2} = \omega_0 \left(\zeta \pm \sqrt{\zeta - 1}\right)$$

Let
$$2\zeta\omega_n=\sqrt{\frac{b}{m}}$$
 and $\omega_0=\sqrt{\frac{k}{m}}$

$$\Delta(s) = s^2 + \frac{b}{m}s + \left(\sqrt{\frac{k}{m}}\right)^2 \iff \Delta(s) = s^2 + 2\zeta\omega_0s + \omega_0^2$$

In this instance, the three cases are easily seen based on ζ :

- 1. Damped: $\zeta > 1$
- 2. Critically Damped: $\zeta = 1$
- 3. Underdamped: $\zeta \in [0,1)$

Case 1 (Damped)

Distinct real roots: $p_1 \neq p_2 \Rightarrow \Delta(s) = s(s + p_1)(s + p_2)$

$$X(s) = \left(\frac{1}{k}\right) \frac{\frac{k}{m}}{s(s^2 + \frac{b}{m}s + \frac{k}{m})} = \frac{K}{s(s + p_1)(s + p_2)}$$

Activity: Response Compariso

Partial Fraction Expansion:

$$X(s) = \frac{C_1}{s} + \frac{C_2}{s + p_1} + \frac{C_3}{s + p_2}$$

Inverse Laplace:

$$\stackrel{\mathcal{L}^{-1}}{\Rightarrow} x(t) = (C_1 + C_2 e^{-p_1 t} + C_3 e^{-p_2 t}) u(t)$$

2nd-order System Dynamics 2024-02-20 Applied Example: Spring-Mass-Damper

Activity: Response Comparison Case 1 (Damped)

Let
$$a=\frac{b}{2m}+\sqrt{\left(\frac{b}{2m}\right)^2-\left(\sqrt{\frac{k}{m}}\right)^2}$$
 and $b=\frac{b}{2m}-\sqrt{\left(\frac{b}{2m}\right)^2-\left(\sqrt{\frac{k}{m}}\right)^2}$ Evaluate coeficients:
$$(a)(b)=(\frac{b}{2m})^2-((\frac{b}{m})^2-\frac{k}{m})=\frac{k}{m},\quad (a-b)=2\sqrt{\left(\left(\frac{b}{2m}\right)^2-\frac{k}{m}\right)}$$

$$C_1=\frac{(s)}{ms(s+a)(s+b)}\bigg|_{s=0}=\frac{1}{m(a)(b)}\Rightarrow C_1=\frac{1}{k}\text{(Hook's Law @ steady-state)}$$

$$C_2 = \frac{(s+a)}{ms(s+a)(s+b)} \bigg|_{s=-a} = \frac{1}{m(-a)(-a+b)} = \frac{1}{m(a)(a-b)}$$

$$C_3 = \frac{(s+b)}{ms(s+a)(s+b)} \bigg|_{s=-b} = \frac{1}{m(-b)(a-b)} = \frac{-1}{m(b)(a-b)}$$

$$C_{2,3} = \frac{\pm 1}{2m\sqrt{((\frac{b}{2m})^2 - \frac{k}{m})} \left(\frac{b}{2m} \pm \sqrt{(\frac{b}{2m})^2 - \left(\sqrt{\frac{k}{m}}\right)^2}\right)}$$

Case 2 (Critically Damped)



Repeated Roots: $b^2 = 4mk \Rightarrow p_1 = p_2 \Rightarrow \Delta(s) = s(s+p)^2$

$$X(s) = \left(\frac{1}{k}\right) \frac{\frac{k}{m}}{s(s^2 + \frac{b}{m}s + \frac{k}{m})} = \frac{K}{s(s+p)^2}$$

Partial Fraction Expansion:

$$X(s) = \frac{C_1}{s} + \frac{C_2}{s+p} + \frac{C_3}{(s+p)^2}$$

Inverse Laplace:

$$\stackrel{\mathcal{L}^{-1}}{\Rightarrow} x(t) = \left(C_1 + C_2 e^{-pt} + C_3 t e^{-pt}\right) u(t)$$

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Case 3 (Underdamped)

Repeated Roots: $b^2 = 4mk \Rightarrow p_1 = p_2 \Rightarrow \Delta(s) = s(s+p)^2$

$$X(s) = \left(\frac{1}{k}\right) \frac{\frac{k}{m}}{s(s^2 + \frac{b}{m}s + \frac{k}{m})} = \frac{K}{s(s + \sigma \pm j\omega)}$$

Partial Fraction Expansion:

$$X(s) == \frac{C_1}{s} + \frac{C_2}{(s+\sigma+j\omega)} + \frac{C_3}{(s+\sigma-j\omega)}$$

Inverse Laplace:

$$\stackrel{\mathcal{L}^{-1}}{\Rightarrow} x(t) = \left(C_1 + C_2 e^{-\sigma t} e^{j\omega t} + C_3 e^{-\sigma t} e^{-j\omega t} \right) u(t)$$

$$= C_1 u(t) + 2e^{-\sigma t} \left(\frac{C_2 e^{j\omega t} + C_3 e^{-j\omega t}}{2} \right) u(t) \Leftarrow \text{Convert using Euler's Identity}$$

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2nd-order System Dynamic

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2nd-order System Dynamics

Applied Example: Spring-Mass-Damper

Activity: Response Comparison
Case 3 (Underdamped)

 $X(\epsilon) = \left(\frac{1}{\epsilon}\right) \frac{1}{\epsilon(\epsilon^2 + \frac{1}{2\epsilon} + \frac{1}{2\epsilon})} = \frac{K}{\epsilon(\epsilon + \epsilon^2 + \frac{1}{2\epsilon})}$ Partial Fraction Expansion: $X(\epsilon) = \frac{C_1}{\epsilon} \times \frac{C_2}{(\epsilon + \epsilon + \frac{1}{\epsilon})^2} \cdot \frac{C_1}{(\epsilon + \epsilon^2 - \frac{1}{\epsilon})}$ Innoren Laplace: $\sum_{k=0}^{\infty} x(t) \ge \left(\frac{1}{\epsilon} \cdot C_k e^{-kt} e^{kt} + c \cdot C_k e^{-kt} e^{kt}\right) + C_k e^{-kt} e^{kt}$ $\sum_{k=0}^{\infty} x(t) \ge \left(\frac{1}{\epsilon} \cdot C_k e^{-kt} e^{kt} + C_k e^{-kt} e^{kt}\right) + C_k e^{-kt}$

Alternative approach

$$X(s) = \frac{\frac{1}{m}}{s(s^2 + \frac{b}{m}s + \frac{k}{m})} = \frac{C_1}{s} + \frac{C_2}{(s + \frac{b}{2m})^2 + \sqrt{\frac{k}{m} - \sqrt{\frac{b}{m}}}} \stackrel{\mathcal{L}}{\Rightarrow}$$

$$\stackrel{\mathcal{L}}{\Rightarrow} x(t) = \left(C_1 + \frac{C_2}{\sqrt{\frac{k}{m} - \sqrt{\frac{b}{m}}}} \exp\left\{-\frac{b}{2m}t\right\} \cos\left(\sqrt{\frac{k}{m} - \sqrt{\frac{b}{m}}}\right)t\right) u(t)$$

Lecture Overview

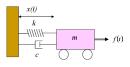


$$m\ddot{x}(t) + b\dot{x}(t) + kx(t) = f(t)$$

$$X(s) = \frac{1}{ms^2 + bs + k} F(s) = \frac{\frac{k}{m}}{s^2 + \frac{b}{m}s + \frac{k}{m}} \frac{1}{k} F(s)$$

$$H(s) = \frac{X(s)}{F(s)} = (K) \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}$$

$$\omega_0 = \sqrt{\frac{k}{m}} \quad \zeta = \sqrt{\frac{b^2}{4mk}} \quad \mathcal{K} = \frac{1}{k}$$



Bibliography I



- Detroit Mercy University of Michigan, Carnegie Mellon. Introduction: System modeling.
- Engineer on a Disk.

 ebook: Dynamic system modeling and control.

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TLDR: Second-Order System Dynamics



Transfer Function

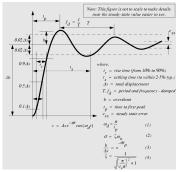
$$H(s) = \frac{Y(s)}{U(s)} = \frac{\omega_0^2}{s^2 + 2\zeta\omega_0 s + \omega_0^2}$$

System Poles

$$s = -\zeta\omega_0 \pm \omega_0 \sqrt{1 - \zeta^2}$$

Spring Mass Damper System Parameters

$$\omega_0 = \sqrt{\frac{k}{m}} \qquad \zeta = \sqrt{\frac{c}{4n}}$$



2nd Order System Response [2]

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