## **Chapter 1**

## Demo problem: Buckling of a clamped cylindrical shell under pressure loading

In this document, we discuss the solution of the buckling of a cylindrical shell using oomph-lib's Kirchhoff $\leftarrow$  LoveShell elements.

## [No documentation yet: Here's the driver code.]

```
//LIC// This file forms part of oomph-lib, the object-oriented,
//LIC// multi-physics finite-element library, available
//LIC// at http://www.oomph-lib.org.
            Version 1.0; svn revision $LastChangedRevision$
//T.TC//
//LIC//
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//LIC// The authors may be contacted at oomph-lib@maths.man.ac.uk.
//Driver function for a simple test shell problem:
//Calculate the deformation of an elastic tube approximated
//using Kirchoff--Love shell theory
//Standard system includes
#include <iostream>
#include <fstream>
#include <cmath>
#include <typeinfo>
#include <algorithm>
#include <cstdio>
//Include files from the finite-element library
#include "generic.h"
#include "shell.h"
#include "meshes/rectangular_quadmesh.h"
using namespace std;
using namespace oomph;
/// Global variables that represent physical properties
namespace Global_Physical_Variables
 /// Prescribed position of control point
```

```
double Prescribed_y = 1.0;
 /// \short Pointer to pressure load (stored in Data so it can
 /// become an unknown in the problem when displacement control is used
Data* Pext_data_pt;
 /// Perturbation pressure
 double Pcos=1.0;
 /// \ short Return a reference to the external pressure
 /// load on the elastic tube.
double external_pressure()
  {return (*Pext_data_pt->value_pt(0))*pow(0.05,3)/12.0;}
 /// Load function, normal pressure loading
const Vector<double> &N,
                Vector<double>& load)
  //std::cout « N[0] « " " « N[1] « " " « N[2] « std::endl; //std::cout « xi[0] « " " « xi[1] « std::endl;
  for(unsigned i=0;i<3;i++)</pre>
    load[i] = (external_pressure() -
              Pcos*pow(0.05,3)/12.0*cos(2.0*xi[1]))*N[i];
   }
 }
//----
/// A 2D Mesh class. The tube wall is represented by two Lagrangian
/// coordinates that correspond to z and theta in cylindrical polars.
/// The required mesh is therefore a 2D mesh and is therefore inherited
/// from the generic RectangularQuadMesh
template <class ELEMENT>
class ShellMesh : public virtual RectangularQuadMesh<ELEMENT>,
                 public virtual SolidMesh
public:
 ///Constructor for the mesh
ShellMesh (const unsigned &nx, const unsigned &ny,
           const double &lx, const double &ly);
 /// \ short In all elastic problems, the nodes must be assigned an undeformed,
 /// or reference, position, corresponding to the stress-free state
 /// of the elastic body. This function assigns the undeformed position
 /// for the nodes on the elastic tube
 void assign undeformed positions (GeomObject* const & undeformed midplane pt):
};
//=====
/// Mesh constructor
/// Argument list:
/// nx : number of elements in the axial direction
/// ny : number of elements in the azimuthal direction
/// lx : length in the axial direction
/// ly : length in theta direction
template <class ELEMENT>
ShellMesh<ELEMENT>::ShellMesh(const unsigned &nx,
                             const unsigned &ny,
                             const double &lx,
                             const double &ly) :
RectangularQuadMesh<ELEMENT>(nx,ny,lx,ly)
 //Find out how many nodes there are
unsigned n_node = nnode();
 //Now in this case it is the Lagrangian coordinates that we want to set,
 //so we have to loop over all nodes and set them to the Eulerian
 //coordinates that are set by the generic mesh generator
 for(unsigned i=0;i<n_node;i++)</pre>
  node_pt(i)->xi(0) = node_pt(i)->x(0);
node_pt(i)->xi(1) = node_pt(i)->x(1);
 //Assign gradients, etc for the Lagrangian coordinates of
 //hermite-type elements
 //Read out number of position dofs
 unsigned n_position_type = finite_element_pt(0)->nnodal_position_type();
 //If this is greater than 1 set the slopes, which are the distances between
 //nodes. If the spacing were non-uniform, this part would be more difficult
 if(n_position_type > 1)
```

```
\label{eq:control_double_xstep} $$ double xstep = (this->Xmax - this->Xmin)/((this->Np-1)*this->Nx);$$ double ystep = (this->Ymax - this->Ymin)/((this->Np-1)*this->Ny);$$
   for (unsigned n=0;n<n_node;n++)</pre>
    {
     //The factor 0.5 is because our reference element has length 2.0
    node_pt(n)->xi_gen(1,0) = 0.5*xstep;
node_pt(n)->xi_gen(2,1) = 0.5*ystep;
  }
/// Set the undeformed coordinates of the nodes
template <class ELEMENT>
void ShellMesh<ELEMENT>::assign_undeformed_positions(
GeomObject* const &undeformed_midplane_pt)
 //Find out how many nodes there are
unsigned n_node = nnode();
 //Loop over all the nodes
 for (unsigned n=0;n<n_node;n++)</pre>
   //Get the Lagrangian coordinates
   Vector<double> xi(2);
   xi[0] = node_pt(n)->xi(0);
xi[1] = node_pt(n)->xi(1);
   //Assign memory for values of derivatives, etc
   Vector<double> R(3);
   DenseMatrix<double> a(2,3);
   RankThreeTensor<double> dadxi(2,2,3);
   //Get the geometrical information from the geometric object
   undeformed\_midplane\_pt->d2position\,(\texttt{xi,R,a,dadxi})\,;\\
   //Loop over coordinate directions
   for (unsigned i=0;i<3;i++)</pre>
     //Set the position
     node_pt(n) \rightarrow x_gen(0,i) = R[i];
     //Set the derivative wrt Lagrangian coordinates
     //Set the mixed derivative
     //(symmetric so doesn't matter which one we use)
     node_pt(n) \rightarrow x_gen(3,i) = 0.25*dadxi(0,1,i);
  }
//Problem class to solve the deformation of an elastic tube
template<class ELEMENT>
class ShellProblem : public Problem
public:
 /// Constructor
ShellProblem(const unsigned &nx, const unsigned &ny,
              const double &lx, const double &ly);
 /// Overload Access function for the mesh
 ShellMesh<ELEMENT>* mesh_pt()
  {return dynamic_cast<ShellMesh<ELEMENT>*>(Problem::mesh_pt());}
/// Actions after solve empty
void actions_after_newton_solve() {}
 /// Actions before solve empty
 void actions_before_newton_solve() {}
 //A self_test function
void solve();
private:
 /// Pointer to GeomObject that specifies the undeformed midplane
GeomObject* Undeformed_midplane_pt;
 /// First trace node
Node* Trace node pt;
 /// Second trace node
Node* Trace_node2_pt;
/// Constructor
```

```
template<class ELEMENT>
ShellProblem<ELEMENT>::ShellProblem(const unsigned &nx, const unsigned &ny,
                                                                                   const double &lx, const double &ly)
  //Create the undeformed midplane object
 Undeformed_midplane_pt = new EllipticalTube(1.0,1.0);
  //Now create the mesh
  Problem::mesh_pt() = new ShellMesh<ELEMENT>(nx,ny,lx,ly);
  //Set the undeformed positions in the mesh
 {\tt mesh\_pt()->} assign\_undeformed\_positions ({\tt Undeformed\_midplane\_pt)};\\
  //Reorder the elements, since I know what's best for them...
 mesh_pt()->element_reorder();
  //Apply boundary conditions to the ends of the tube
  unsigned n_ends = mesh_pt()->nboundary_node(1);
  //Loop over the node
  for (unsigned i=0;i<n_ends;i++)</pre>
      //Pin in the axial direction (prevents rigid body motions)
      mesh_pt()->boundary_node_pt(1,i)->pin_position(2);
      mesh_pt()->boundary_node_pt(3,i)->pin_position(2);
       //Derived conditions
      mesh\_pt() -> boundary\_node\_pt(1,i) -> pin\_position(2,2);
      mesh_pt()->boundary_node_pt(3,i)->pin_position(2,2);
//-----CLAMPING CONDITIONS------
       //-----Pin positions in the transverse directions-----
       // Comment these out to get the ring case
       mesh_pt()->boundary_node_pt(1,i)->pin_position(0);
       mesh_pt()->boundary_node_pt(3,i)->pin_position(0);
       //Derived conditions
      mesh_pt()->boundary_node_pt(1,i)->pin_position(2,0);
      mesh_pt()->boundary_node_pt(3,i)->pin_position(2,0);
       mesh_pt()->boundary_node_pt(1,i)->pin_position(1);
       mesh_pt()->boundary_node_pt(3,i)->pin_position(1);
       //Derived conditions
      mesh_pt()->boundary_node_pt(1,i)->pin_position(2,1);
      mesh_pt()->boundary_node_pt(3,i)->pin_position(2,1);
       // Set the axial gradients of the transverse coordinates to be
       // zero --- need to be enforced for ring or tube buckling
       //Pin dx/dz and dy/dz
      mesh_pt()->boundary_node_pt(1,i)->pin_position(1,0);
      mesh_pt()->boundary_node_pt(1,i)->pin_position(1,1);
      mesh_pt()->boundary_node_pt(3,i)->pin_position(1,0);
      mesh_pt()->boundary_node_pt(3,i)->pin_position(1,1);
       //Derived conditions
       mesh_pt()->boundary_node_pt(1,i)->pin_position(3,0);
      mesh_pt()->boundary_node_pt(1,i)->pin_position(3,1);
      mesh_pt()->boundary_node_pt(3,i)->pin_position(3,0);
      mesh_pt()->boundary_node_pt(3,i)->pin_position(3,1);
  //Now loop over the sides and apply symmetry conditions
  unsigned n_side = mesh_pt()->nboundary_node(0);
  for (unsigned i=0;i<n_side;i++)</pre>
      //At the side where theta is 0, pin in the y direction
      mesh_pt()->boundary_node_pt(0,i)->pin_position(1);
       //Derived condition
       mesh_pt()->boundary_node_pt(0,i)->pin_position(1,1);
       //Pin dx/dtheta and dz/dtheta
      mesh_pt()->boundary_node_pt(0,i)->pin_position(2,0);
      mesh_pt()->boundary_node_pt(0,i)->pin_position(2,2);
       //Pin the mixed derivative
       mesh_pt()->boundary_node_pt(0,i)->pin_position(3,0);
       mesh_pt()->boundary_node_pt(0,i)->pin_position(3,2);
       //At the side when theta is 0.5pi % \left( 1\right) =\left( 1\right) +\left( 1\right) +
       mesh_pt()->boundary_node_pt(2,i)->pin_position(0);
       //Derived condition
      mesh_pt()->boundary_node_pt(2,i)->pin_position(1,0);
       //Pin dy/dtheta and dz/dtheta
       mesh_pt()->boundary_node_pt(2,i)->pin_position(2,1);
      mesh_pt()->boundary_node_pt(2,i)->pin_position(2,2);
       //Pin the mixed derivative
      mesh_pt()->boundary_node_pt(2,i)->pin_position(3,1);
      mesh_pt()->boundary_node_pt(2,i)->pin_position(3,2);
   //Set an initial kick to make sure that we hop onto the
              //non-axisymmetric branch
//
//
//
              if((i>1) && (i<n_side-1))
                 mesh\_pt() \rightarrow boundary\_node\_pt(0,i) \rightarrow x(0) += 0.05;
                 mesh_pt()->boundary_node_pt(2,i)->x(1) -= 0.1;
 // Setup displacement control
      //Setup displacement control
// //Fix the displacement at the mid-point of the tube in the "vertical"
```

```
//(y) direction.
    //Set the displacement control element (located halfway along the tube)
// Disp_ctl_element_pt = dynamic_cast<ELEMENT*>(mesh_pt()->element_pt(3*Ny-1));
   //The midpoint of the tube is located exactly half-way along the element
   Vector<double> s(2); s[0] = 1.0; s[1] = 0.0; //s[1] = 0.5 //Fix the displacement at this point in the y (1) direction
// Disp_ctl_element_pt->fix_displacement_for_displacement_control(s,1);
    //Set the pointer to the prescribed position
// Disp_ctl_element_pt->prescribed_position_pt() = &Prescribed_y;
 // Choose element in which displacement control is applied: This
 // one is located about halfway along the tube -- remember that
 // we've renumbered the elements!
 unsigned nel_ctrl=0;
Vector<double> s_displ_control(2);
// Even/odd number of elements in axial direction if (nx%2==1)
  nel_ctrl=unsigned(floor(0.5*double(nx))+1.0)*ny-1;
  s_displ_control[0]=0.0;
  s_displ_control[1]=1.0;
else
  nel_ctrl=unsigned(floor(0.5*double(nx))+1.0)*ny-1;
  s_displ_control[0]=-1.0;
  s_displ_control[1]=1.0;
 // Controlled element
SolidFiniteElement* controlled_element_pt=
 dynamic_cast<ELEMENT*>(mesh_pt()->element_pt(nel_ctrl));
 // Fix the displacement in the y (1) direction...
unsigned controlled_direction=1;
 // Pointer to displacement control element
DisplacementControlElement* displ_control_el_pt;
 // Build displacement control element
displ_control_el_pt=
 new DisplacementControlElement(controlled_element_pt,
                                 s_displ_control,
                                 controlled_direction,
                                 &Global_Physical_Variables::Prescribed_v);
 // Doc control point
 Vector<double> xi(2);
Vector<double> x(3);
 controlled_element_pt->interpolated_xi(s_displ_control,xi);
 controlled_element_pt->interpolated_x(s_displ_control,x);
 std::cout « std::endl;
 std::cout « "Controlled element: " « nel_ctrl « std::endl;
std::cout « "Displacement control applied at xi = (
           \ll xi[0] \ll ", " \ll xi[1] \ll ")" \ll std::endl;
// adjustable load: Use this Data object in the load function:
Global_Physical_Variables::Pext_data_pt=displ_control_el_pt->
 displacement_control_load_pt();
 // Add the displacement-control element to the mesh
mesh_pt()->add_element_pt(displ_control_el_pt);
 // Complete build of shell elements
 //Find number of shell elements in mesh
unsigned n_element = nx*ny;
 //Explicit pointer to first element in the mesh
 ELEMENT* first_el_pt = dynamic_cast<ELEMENT*>(mesh_pt()->element_pt(0));
 //Loop over the elements
 for(unsigned e=0;e<n_element;e++)</pre>
   //Cast to a shell element
  ELEMENT *el_pt = dynamic_cast<ELEMENT*>(mesh_pt()->element_pt(e));
   //Set the load function
   el_pt->load_vector_fct_pt() = & Global_Physical_Variables::press_load;
   //Set the undeformed surface
  el_pt->undeformed_midplane_pt() = Undeformed_midplane_pt;
   //The external pressure is external data for all elements
  el_pt->add_external_data(Global_Physical_Variables::Pext_data_pt);
   //Pre-compute the second derivatives wrt Lagrangian coordinates
   //for the first element only
   if(e==0)
```

```
el_pt->pre_compute_d2shape_lagrangian_at_knots();
   //Otherwise set the values to be the same as those in the first element
   //this is OK because the Lagrangian mesh is uniform.
  else
   {
    el_pt->set_dshape_lagrangian_stored_from_element(first_el_pt);
 //Set pointers to two trace nodes, used for output
Trace_node_pt = mesh_pt()->finite_element_pt(2*ny-1)->node_pt(3);
Trace_node2_pt = mesh_pt()->finite_element_pt(ny)->node_pt(1);
 // Do equation numbering
cout « std::endl;
cout « "# of dofs " « assign_eqn_numbers() « std::endl;
cout « std::endl;
// /Define the solve function, disp ctl and then continuation
template<class ELEMENT>
void ShellProblem<ELEMENT>::solve()
//Increase the maximum number of Newton iterations.
//Finding the first buckled solution requires a large(ish) number
 //of Newton steps -- shells are just a bit twitchy
Max_newton_iterations = 40;
Max_residuals=1.0e6;
 //Open an output trace file
ofstream trace("trace.dat");
 //Gradually compress the tube by decreasing the value of the prescribed
 //position
 for (unsigned i=1;i<11;i++)</pre>
  Global_Physical_Variables::Prescribed_y -= 0.05;
  // Solve
  newton_solve();
  //Position of first trace node
« Trace_node_pt->x(0) « " " « Trace_node_pt->x(1) « " "
        //Position of second trace node
« Trace_node2_pt->x(0) « " " « Trace_node2_pt->x(1) « std::endl;
  // Reset perturbation
Global_Physical_Variables::Pcos=0.0;
 //Close the trace file
trace.close();
 //Output the tube shape in the most strongly collapsed configuration
ofstream file("final_shape.dat");
mesh_pt()->output(file,5);
 file.close();
/// Driver
//=====
int main()
 //Length of domain
double L = 10.0;
double L_phi=0.5*MathematicalConstants::Pi;
 //Set up the problem
ShellProblem<StorableShapeSolidElement<DiagHermiteShellElement> >
 problem(5,3,L,L_phi);
 //Solve the problem
problem.solve();
```

## 1.1 PDF file

A pdf version of this document is available.