

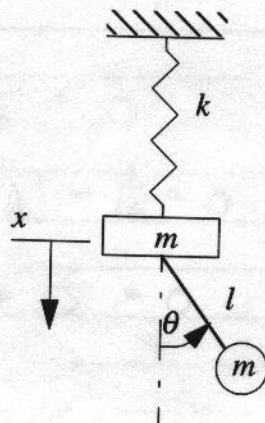
ME 460/660 Final, Spring '96

One equation sheet. Front and back. No examples. No derivations. It must be turned in with the exam. Each problem is worth 25 points.

- 1) Find the spectral matrix, Λ , and eigenvector matrix, P , given the following:

$$M = \begin{bmatrix} 4 & 0 \\ 0 & 36 \end{bmatrix}, K = \begin{bmatrix} 3 & -3 \\ -3 & 27 \end{bmatrix}$$

- 2) Choose a suspension damping coefficient (c) for a 1000 kg car such that the settling time is less than 3 sec and the displacement transmissibility is less than 0.5 at 3 Hz ($r = 2$).
- 3) The force exerted by an eccentric ($e = 0.22$ mm) flywheel of 1000 kg is $600 \cos(52.4t)$ N. Design a mounting to reduce the amplitude of the force exerted on the floor to 1% of the force generated. Also, use the suspension system to ensure that the maximum force transmitted to ground never exceeds twice the generated force (at any speed).
- 4) Derive the equations of motion for the following system using Lagrange's equations with x and θ as the generalized coordinates. The block can only move in the vertical direction. (Hint: Set the datums to be at the unstretched spring length and $\theta = 0$.)



$$1) \quad M^{1/2} = \begin{bmatrix} 2 & 0 \\ 0 & 6 \end{bmatrix}$$

$$M^{-1/2} = \begin{bmatrix} \frac{1}{2} & 0 \\ 0 & \frac{1}{6} \end{bmatrix}$$

$$M^{-1/2} K M^{-1/2} = \tilde{K} = \begin{bmatrix} \frac{1}{2} & 0 \\ 0 & \frac{1}{6} \end{bmatrix} \begin{bmatrix} 3 & -3 \\ -3 & 27 \end{bmatrix} \begin{bmatrix} \frac{1}{2} & 0 \\ 0 & \frac{1}{6} \end{bmatrix}$$

$$\tilde{K} = \begin{bmatrix} \frac{3}{2} & -\frac{3}{2} \\ -\frac{1}{2} & \frac{9}{2} \end{bmatrix} \begin{bmatrix} \frac{1}{2} & 0 \\ 0 & \frac{1}{6} \end{bmatrix}$$

$$\tilde{K} = \begin{bmatrix} \frac{3}{4} & -\frac{1}{4} \\ -\frac{1}{4} & \frac{3}{4} \end{bmatrix}$$

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$$\det(\tilde{K} - \lambda I) = 0$$

$$\left(\frac{3}{4} - \lambda\right)^2 - \frac{1}{16} = 0$$

$$\frac{9}{16} - \frac{3}{2}\lambda + \lambda^2 - \frac{1}{16} = 0$$

$$\lambda^2 - \frac{3}{2}\lambda + \frac{1}{2} = 0$$

$$\lambda = \frac{1}{2}, 1$$

$$L = \begin{bmatrix} \frac{1}{2} & 0 \\ 0 & 1 \end{bmatrix}$$

For $\lambda_1 = \frac{1}{2}$:

$$\begin{bmatrix} \frac{3}{4} - \frac{1}{2} & -\frac{1}{4} \\ -\frac{1}{4} & \frac{3}{4} - \frac{1}{2} \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\frac{1}{4} v_1 - \frac{1}{4} v_2 = 0$$

$$-\frac{1}{4} v_1 + \frac{1}{4} v_2 = 0$$

$$\underline{v_1} = \frac{1}{\sqrt{2}} \begin{bmatrix} 1 \\ 1 \end{bmatrix}$$

For $\lambda_2 = 1$:

$$\begin{bmatrix} \frac{3}{4} - 1 & -\frac{1}{4} \\ -\frac{1}{4} & \frac{3}{4} - 1 \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\underline{v_2} = \frac{1}{\sqrt{2}} \begin{bmatrix} 1 \\ -1 \end{bmatrix}$$

$$K = m\omega^2 = 1000 \cdot 9\pi^2 = 88826 \text{ N/m}$$

$$C = 5 \sqrt{K} = 3770 \text{ kg/s}$$

$$2) \ln .05 = -\xi \omega t_s$$

$$3 = \xi \omega t_s$$

$$t_s = 3/\xi \omega < 3$$

$$\xi \omega > 1$$

Since $r = 2 @ 3 \text{ Hz}$, $F = 1.5 \text{ Hz}$

$$\omega = 2\pi F = 3\pi \text{ rad/s} \quad (5)$$

$$\xi > \frac{1}{\omega} = \frac{1}{3\pi}$$

$$\xi > \frac{1}{3\pi} \approx \frac{1}{9.42}$$

$$\text{Pick } \xi = .2$$

Does this meet $TR < .5 @ r = 2$?

$$TR = \left[\frac{1 + (2\zeta r)^2}{(1-r^2)^2 + (2\zeta r)^2} \right]^{\frac{1}{2}} \stackrel{?}{<} .5$$

$$\frac{1 + .8^2}{9 + .8^2} \stackrel{?}{<} .25$$

$$.1061 < \xi < .322$$

$$2000 < C < 6070$$

$$C = 2.17 < .25$$

+10

$$K = m\omega^2 = 1000 \cdot 9\pi^2 = 88826 \text{ N/m}$$

$$C = \xi 2\sqrt{Km} = 3770 \text{ kg/s}$$

$$3) a) TR = .01 = \left[\frac{1 + (25r)^2}{(1-r^2)^2 + (25r)^2} \right]^{1/2}$$

$$b) \text{ At resonance, } TR = 2$$

$$2 = \left(\frac{1 + (25)^2}{(25)^2} \right)^{1/2}$$

$$4 \cdot 45^2 = 1 + 45^2 \quad \times 10$$

$$125^2 = 1$$

$$5 \geq .2887$$

$$\text{pick } \xi = .3$$

$$.0001 = \frac{1 + (.65)^2}{(1-r^2)^2 + (.65)^2}$$

$$.0001(1 - 2r^2 + r^4 + .36r^2) = 1 + .36r^2 \quad \times 10$$

$$.0001r^2 \approx .36$$

$$r = 60 = \frac{52.4}{\omega} \quad \omega = .873 \text{ rad/s}$$

$$K = m\omega^2 = 1000 \cdot .873^2 = 762.7 \text{ N/m}$$

$$C = 2 \xi \sqrt{Km} = 524 \text{ kg/s} \quad \times 5$$

$$\frac{dx}{dt} = 2m\ddot{x} - m\ddot{\theta} \sin\theta - m\ddot{\theta} \cos\theta$$

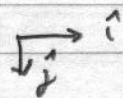
$$\frac{dx}{dt} = 0$$

$$\frac{dx}{dt} = Kx - 2mg$$

$$2m\ddot{x} - m\ddot{\theta} \sin\theta - m\ddot{\theta} \cos\theta + Kx - 2mg = 0$$

4) The position of the pendulum mass is

$$x_p = x\hat{j} + l\cos\theta\hat{j} + l\sin\theta\hat{i}$$



Its velocity is then

$$v_p = \dot{x}\hat{j} - \dot{\theta}l\sin\theta\hat{j} + \dot{\theta}l\cos\theta\hat{i}$$

The total kinetic energy is then

$$T = \frac{1}{2}m\dot{x}^2 + \frac{1}{2}m(\dot{x} - \dot{\theta}l\sin\theta)^2 + \frac{1}{2}m\dot{\theta}^2l^2\cos^2\theta$$

The potential energy is

$$U = \underbrace{\frac{1}{2}Kx^2 - mgx}_{\text{Top mass}} - \underbrace{mgx + mgl(1-\cos\theta)}_{\text{pendulum mass}}$$

$$U = \frac{1}{2}Kx^2 - 2mgx + mgl(1-\cos\theta)$$

Lagrange's Equation

$$\frac{d}{dt}\left(\frac{\partial T}{\partial \dot{q}_i}\right) - \frac{\partial T}{\partial q_i} + \frac{\partial U}{\partial q_i} = 0$$

For $q_i = x$

$$\frac{\partial T}{\partial \dot{x}} = m\dot{x} + m(\dot{x} - \dot{\theta}l\sin\theta)$$

$$\frac{d}{dt}\left(\frac{\partial T}{\partial \dot{x}}\right) = 2m\ddot{x} - m\ddot{\theta}l\sin\theta - m\dot{\theta}^2l\cos\theta$$

$$\frac{\partial T}{\partial x} = 0$$

$$\frac{\partial U}{\partial x} = Kx - 2mg$$

$$\textcircled{1} \quad 2m\ddot{x} - m\ddot{\theta}l\sin\theta - m\dot{\theta}^2l\cos\theta + Kx - 2mg = 0$$

For $q_i = \theta$

$$\frac{\partial T}{\partial \dot{\theta}} = -m l \sin \theta (\dot{x} - l \dot{\theta} \sin \theta) + m \dot{\theta} l^2 \cos^2 \theta$$

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\theta}} \right) = -m l \dot{\theta} \cos \theta (\dot{x} - l \dot{\theta} \sin \theta) - m l \sin \theta (\ddot{x} - l \ddot{\theta} \sin \theta - l \dot{\theta}^2 \cos \theta) + m \ddot{\theta} l^2 \cos^2 \theta - 2m \dot{\theta}^2 l^2 \cos \theta \sin \theta$$

$$\frac{\partial T}{\partial \theta} = -m \dot{\theta} l \cos \theta (\dot{x} - \dot{\theta} l \sin \theta) - m \dot{\theta}^2 l^2 \sin \theta \cos \theta$$

$$\frac{\partial U}{\partial \theta} = m g l \sin \theta$$

$$\begin{aligned} & \cancel{-m l \dot{\theta} \cos \theta (\dot{x} - l \dot{\theta} \sin \theta)}^{(1)} - m l \sin \theta (\dot{x} - l \dot{\theta} \sin \theta) \\ & \quad - \cancel{l \dot{\theta}^2 \cos \theta}^{(2)} + m \ddot{\theta} l^2 \cos^2 \theta - \cancel{2m \dot{\theta}^2 l^2 \cos \theta \sin \theta}^{(2)} \\ & \quad + \cancel{m \dot{\theta} l \cos \theta (\dot{x} - \dot{\theta} l \sin \theta)}^{(1)} + \cancel{m \dot{\theta}^2 l^2 \sin \theta \cos \theta}^{(2)} \\ & \quad + m g l \sin \theta = 0 \end{aligned}$$

$$-m l \sin \theta \ddot{x} + m l^2 \sin^2 \theta \ddot{\theta} + m \ddot{\theta} l^2 \cos^2 \theta + m g l \sin \theta = 0$$

Derive the equations of motion for the following using Lagrange's equations with x and θ as the generalized coordinates. The block can only move in the vertical direction. (Hint: Set

$$(2) \quad m l \ddot{\theta} - \sin \theta \ddot{x} + g \sin \theta = 0$$

