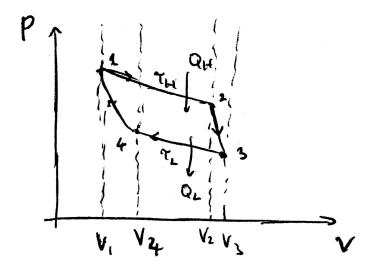
Phys 112: Lecture 24

Notes for the Fall 2017 Physics 112 Course taught by Professor Holzapfel prepared by Joshua Lin (email: joshua.z.lin@gmail.com)

November 16, 2017

1 PV Diagrams

Remember from last lecture, we were expressing physical engines that exchange between heat and work as closed curves in some phase space - before we were working in the Temperature Entropy space, but we can also work in the Pressure Volume case:



Now, we can calculate all our quantites of interest for this particular cycle. For the 1 \rightarrow 2 transition, the heat input Q_H has to equal the work done by the system $W = \int p dV$ since the internal energy doesn't change (set by the temperature). So we have:

$$Q_{H} = \int_{V_{1}}^{V_{2}} p dV = \int_{V_{1}}^{V_{2}} \frac{N\tau}{V} dV = N\tau_{H} \ln \left(\frac{V_{2}}{V_{1}}\right)$$

Now, from point $2 \to \text{point } 3$ on the phase diagram, we don't have any heat flow, so the work done by the system is equal to negative of the change of internal entropy, so we have that:

$$W = -\Delta U = -C_V(\tau_3 - \tau_2) = \frac{3}{2}N(\tau_H - \tau_L)$$

Recall that for adiabatic processes, we derived in a past lecture that:

$$PV^{\gamma} = \text{Constant}; \qquad TV^{\gamma-1} = \text{Constant}; \qquad \gamma = \frac{C_P}{C_V}$$

and we have that $\gamma = 5/3$ for monotomic gases, and 7/5 for the diatomic gases. Now it turns out that the particular value of γ doesn't matter; our calculations always hold since the gammas vanish. But for the sake of simplicity, we will only treat the case where we have a monomic gas. So we have that:

$$au_L V_3^{2/3} = au_H V_2^{2/3}; \qquad \frac{V_3}{V_2} = \left(\frac{ au_H}{ au_L}\right)^{3/2}$$

Now, from point $3 \to \text{point } 4$ on the phase diagram, we have the same situation as in $1 \to 2$, but in reverse in some sense. So we have to do some work on the system, so we have:

$$W = N\tau_L \ln \left(\frac{V_3}{V_4}\right) = Q_L$$

Finally, the last step, from point $4 \to \text{point } 1$, we have the same situation as point $2 \to \text{point } 3$:

$$W = C_V(\tau_1 - \tau_4) = \frac{3}{2}N(\tau_H - \tau_L)$$

so we have:

$$\frac{V_4}{V_1} = \left(\frac{\tau_H}{\tau_L}\right)^{3/2}$$

so using our equation from before, we have:

$$\frac{V_4}{V_1} = \frac{V_3}{V_2}; \qquad \frac{V_3}{V_4} = \frac{V_2}{V_1}$$

Now, the total work is going to be:

$$W = W_{12} + W_{23} - W_{34} - W_{41}$$

where W_{ij} is the absolute value of the work done going from i to j (as described above in equations, since the Professor wants everything to be positive). Notice that $W_23 = W_41$ so those two terms cancel. So we have:

$$W = N \left[\tau_H \ln \left(\frac{V_2}{V_1} \right) - \tau_L \ln \left(\frac{V_3}{V_4} \right) \right] = N(\tau_H - \tau_L) \ln \left(\frac{V_2}{V_1} \right)$$

Now, we also have that:

$$Q_H = N\tau_H \ln \left(\frac{V_2}{V_1}\right)$$

so the efficiency of the system shown; we have:

$$\eta = \frac{W}{Q_H} = \frac{\tau_H - \tau_L}{\tau_H} = \eta_c$$

the carnot efficiency!!

2 Reversible and Irreversible Processes

¹ If we are thinking about a system in Energy Entropy phase space, when we go from state U_1, σ_1 to U_2, σ_2 , then there must be a reversible process that does this process for me, which costs dW_{rev} amount of work, and dQ_{rev} amount of heat. For any irreversible proces, we have that $dW_{\text{irrev}} > dW_{\text{rev}}$ (we need to do more work), but since we have:

$$dU = dW_{irrev} + dQ_{irrev} = dW_{rev} + dQ_{rev}$$

so we have that in this case, $dQ_{irrev} < dQ_{rev}$, i.e. there is less heat flow in the irreversible case

As a model problem, we can firs consider the case of an isothermal expansion:

$$dW_{\text{rev}} = -\int_{V_1}^{V_2} p dV = -N\tau \ln \left(\frac{V_2}{V_1}\right) < 0$$

is the work done on the system, and we have:

$$dU = 0;$$
 $dQ_{\text{rev}} = -dW_{\text{rev}} > 0$

$$d\sigma = \sigma_2 - \sigma_1 = \frac{dQ_{\text{rev}}}{\tau} = N \ln \left(\frac{V_2}{V_1}\right)$$

We could start with the same initial state and end with the same final state if we just did a Free expansion, in which case:

$$dU = 0;$$
 $dW_{\text{irrev}} = dQ_{\text{irrev}} = 0$

So we have verified that:

$$dW_{\text{irrev}} > dW_{\text{rev}}; \qquad dW_{\text{irrev}} < dQ_{\text{rev}}$$

As another model problem, we can consider two systems of temperature τ_1, τ_2 , placed in thermal contact. We have that:

$$dU_{1} = dQ_{1} = \tau_{1}d\sigma_{1}; dU_{2} = dQ_{2} = \tau_{2}d\sigma_{2}$$
$$dQ_{1} + dQ_{2} = 0$$
$$d\sigma_{\text{irrev}} = \frac{dQ_{1}}{\tau_{1}} + \frac{dQ_{2}}{\tau_{2}} = \frac{dQ_{1}(\tau_{2} - \tau_{1})}{\tau_{2}\tau_{1}}$$

As another model problem, we can consider isothermal work. In this case, we have that $d\tau = 0$. So our equations are given by:

$$dW_{\rm rev} = \Delta F; \qquad F = U - \tau \sigma$$

$$dQ = \tau d\sigma = d(\tau \sigma)$$

$$dW = dU - dQ = dU - d(\tau \sigma) = dF$$

Now, since an ideal gas internal energy only depends on temperature, we have that:

$$dU = 0;$$
 $dW = -dW = dF$

¹From now on, the convention is that work refers to work done on the system, and heat refers to heat flowing into the system