# TRANSMISSÃO DE CALOR E MASSA

# CAPÍTULO 13 TROCA DE RADIAÇÃO ENTRE SUPERFÍCIES

# Radiation Exchange Between Surfaces: Enclosures with Nonparticipating Media

Chapter 13 Sections 13.1 through 13.4

## **Basic Concepts**

• Enclosures consist of two or more surfaces that envelop a region of space (typically gas-filled) and between which there is radiation transfer. Virtual, as well as real, surfaces may be introduced to form an enclosure.

• A nonparticipating medium within the enclosure neither emits, absorbs, nor scatters radiation and hence has no effect on radiation exchange between the surfaces.

• Each surface of the enclosure is assumed to be isothermal, opaque, diffuse and gray, and to be characterized by uniform radiosity and irradiation.

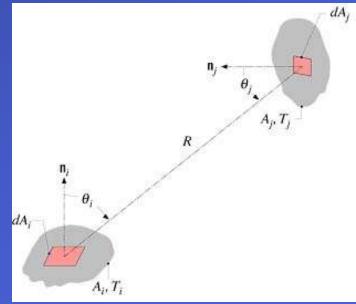
# The View Factor (also Configuration or Shape Factor)

• The view factor,  $F_{ij}$ , is a geometrical quantity corresponding to the fraction of the radiation leaving surface i that is intercepted by surface j.

$$F_{ij} = \frac{q_{i \to j}}{A_i J_i}$$

• The view factor integral provides a general expression for  $F_{ij}$ . Consider exchange between differential areas  $dA_i$  and  $dA_j$ :

$$F_{ij} = \frac{1}{A_i} \int_{A_i} \int_{A_j} \frac{\cos \theta_i \cos \theta_j}{\pi R^2} dA_i dA_j$$



#### View Factor Relations

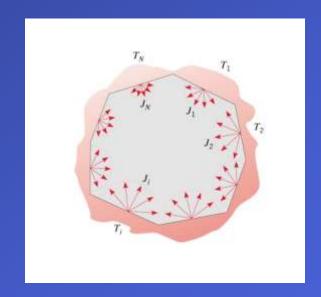
• Reciprocity Relation. With

$$F_{ji} = \frac{1}{A_j} \int_{A_i} \int_{A_j} \frac{\cos \theta_i \cos \theta_j}{\pi R^2} dA_i dA_j$$

$$A_i F_{ij} = A_j F_{ji}$$

• Summation Rule for Enclosures.

$$\sum_{j=1}^{N} F_{ij} = 1$$



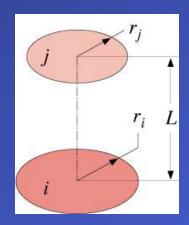
#### • Three-Dimensional Geometries (Table 13.2). For example,

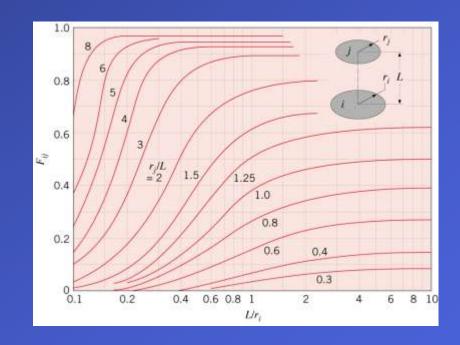
#### Coaxial Parallel Disks

$$F_{ij} = \frac{1}{2} \left\{ S - \left[ S^2 - 4(r_j / r_i)^2 \right]^{1/2} \right\}$$

$$S = 1 + \frac{1 + R_j^2}{R_i^2}$$

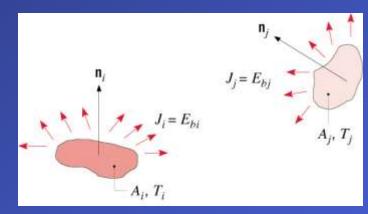
$$R_i = r_i / L$$
  $R_j = r_j / L$ 





# Blackbody Radiation Exchange

- For a blackbody,  $J_i = E_{bi}$ .
- Net radiative exchange between two surfaces that can be approximated as blackbodies net rate at which radiation leaves surface i due to its interaction with j



or net rate at which surface *j* gains radiation due to its interaction with *i* 

$$egin{aligned} q_{ij} &= q_{i
ightarrow j} - q_{j
ightarrow i} \ q_{ij} &= A_i F_{ij} E_{bi} - A_j F_{ji} E_{bj} \ q_{ij} &= A_i F_{ij} \sigma ig( T_i^4 - T_j^4 ig) \end{aligned}$$

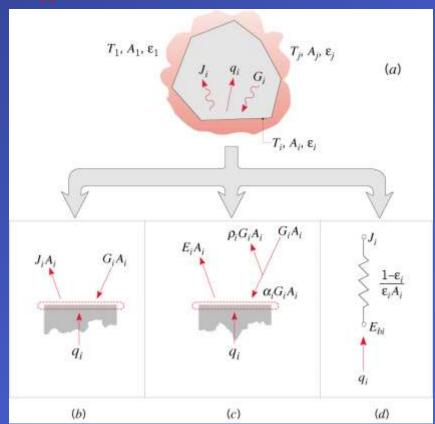
• Net radiation transfer from surface *i* due to exchange with all (*N*) surfaces of an enclosure:

$$q_i = \sum\limits_{i=1}^N A_i F_{ij} \sigma igg(T_i^4 - T_j^4igg)$$

# General Radiation Analysis for Exchange between the N Opaque, Diffuse, Gray Surfaces of an Enclosure $(\varepsilon_i = \alpha_i = 1 - \rho_i)$

• Alternative expressions for net radiative transfer from surface *i*:

$$q_{i} = \frac{E_{bi} - J_{i}}{(1 - \varepsilon_{i})/\varepsilon_{i}A_{i}} \rightarrow \text{Fig. (d)}$$
Suggests a surface radiative resistance of the form:  $(1 - \varepsilon_{i})/\varepsilon_{i}A_{i}$ 



General Enclosure Analysis (cont)

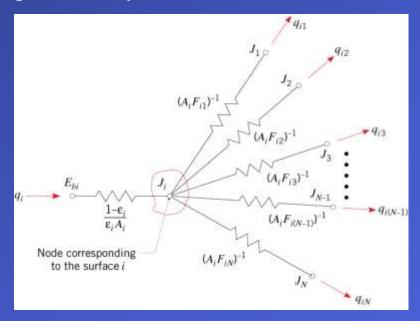
$$q_{i} = \sum_{j=1}^{N} A_{i} F_{ij} \left( J_{i} - J_{j} \right) = \sum_{j=1}^{N} \frac{J_{i} - J_{j}}{\left( A_{i} F_{ij} \right)^{-1}}$$
(4)

Suggests a space or geometrical resistance of the form:  $(A_i F_{ij})^{-1}$ 

• Equating Eqs. (3) and (4) corresponds to a radiation balance on surface *i*:

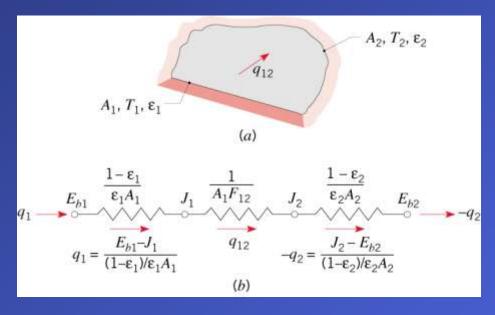
$$\frac{E_{bi} - J_i}{(1 - \varepsilon_i)/\varepsilon_i A_i} = \sum_{j=1}^{N} \frac{J_i - J_j}{(A_i F_{ij})^{-1}}$$
(5)

which may be represented by a radiation network of the form



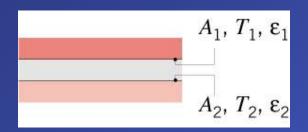
#### Two-Surface Enclosures

• Simplest enclosure for which radiation exchange is exclusively between two surfaces and a single expression for the rate of radiation transfer may be inferred from a network representation of the exchange.



$$q_{1} = -q_{2} = q_{12} = \frac{\sigma(T_{1}^{4} - T_{2}^{4})}{\frac{1 - \varepsilon_{1}}{\varepsilon_{1}A_{1}} + \frac{1}{A_{1}F_{12}} + \frac{1 - \varepsilon_{2}}{\varepsilon_{2}A_{2}}}$$

- Special cases are presented in Table 13.3. For example,
  - ➤ Large (Infinite) Parallel Plates



$$A_1, T_1, \varepsilon_1$$

$$A_1 = A_2 \equiv A$$

$$F_{12} = 1$$

$$q_{12} = \frac{A_1 \sigma \left(T_1^4 - T_2^4\right)}{\frac{1}{\varepsilon_1} + \frac{1}{\varepsilon_2} - 1}$$

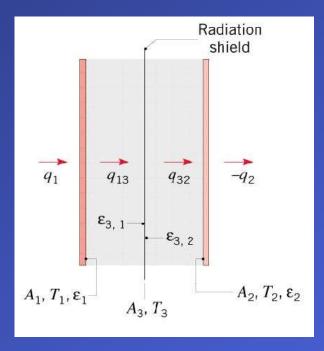
➤ Note result for *Small Convex Object in a Large Cavity*.

#### Radiation Shields

• High reflectivity (low  $\alpha = \varepsilon$ ) surface(s) inserted between two surfaces for which a reduction in radiation exchange is desired.

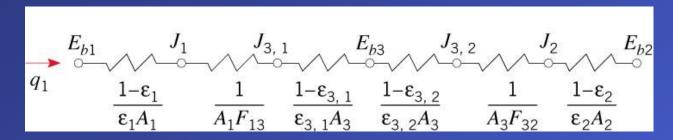
• Consider use of a single shield in a two-surface enclosure, such as that associated with

large parallel plates:



Note that, although rarely the case, emissivities may differ for opposite surfaces of the shield.

• Radiation Network:

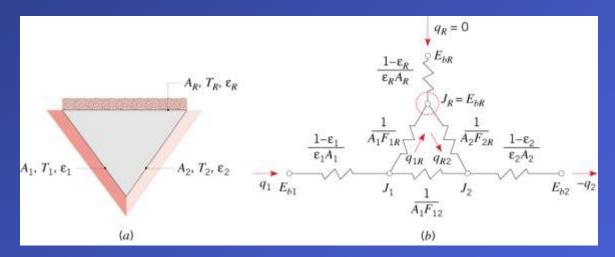


$$q_{12} = q_1 = -q_2 = \frac{\sigma(T_1^4 - T_2^4)}{\frac{1 - \varepsilon_1}{\varepsilon_1 A_1} + \frac{1}{A_1 F_{13}} + \frac{1 - \varepsilon_{3,1}}{\varepsilon_{3,1} A_3} + \frac{1 - \varepsilon_{3,2}}{\varepsilon_{3,2} A_3} + \frac{1}{A_3 F_{32}} + \frac{1 - \varepsilon_2}{\varepsilon_2 A_2}}$$

• The foregoing result may be readily extended to account for multiple shields and may be applied to long, concentric cylinders and concentric spheres, as well as large parallel plates.

### The Reradiating Surface

- An idealization for which  $G_R = J_R$ . Hence,  $q_R = 0$  and  $J_R = E_{bR}$ .
- Approximated by surfaces that are well insulated on one side and for which convection is negligible on the opposite (radiating) side.
- Three-Surface Enclosure with a Reradiating Surface:



$$q_{1} = -q_{2} = \frac{\sigma\left(T_{1}^{4} - T_{2}^{4}\right)}{\frac{1 - \varepsilon_{1}}{\varepsilon_{1}A_{1}} + \frac{1}{A_{1}F_{12} + \left[\left(1/A_{1}F_{1R}\right) + \left(1/A_{2}F_{2R}\right)\right]^{-1}} + \frac{1 - \varepsilon_{2}}{\varepsilon_{2}A_{2}}}$$