

ME 397- ASBR Week 4-Lecture 2



a Curiosity NASA/JPLCaltech;
 b Savioke Relay;
 c self driving car, Oxford Univ.;
 d Cheetah legged robot, Boston Dynamics

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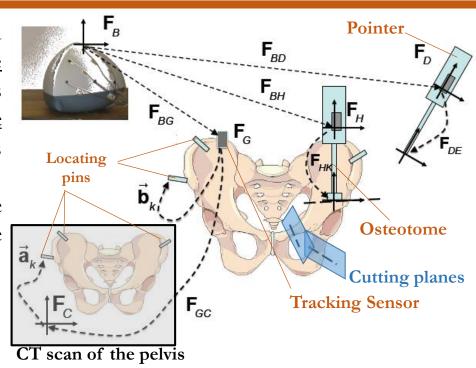
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Example: Computer-Assisted Osteotomy

Consider the pelvic osteotomy situation illustrated in the figure. Here we assume that a **three locating pins** have been inserted into the patient's pelvis, and that a CT scan of the pelvis with the pins inserted has been produced. The patient has been placed onto the operating table.

- A magnetic navigation system (here, the Northern Digital Aurora) is present in the room.
- > Two surgical tools are available:
 - ✓ A probe/pointer device
 - An osteotome (essentially a fancy chisel) CT scan of the pelvis that will be used to cut the pelvis.



https://www.youtube.com/watch?v=N8rfMzU4siQ

- ➤ 6 DOF Aurora tracking sensors have been attached to the **handle of each tool** and an additional 6 DOF sensor has been **affixed rigidly to the pelvis**. The <u>Aurora is capable of determining the position and orientation of each sensor relative to the Aurora base unit</u>.
- ightharpoonup Let $\mathbf{p}_{up} = \mathbf{p}_{GE}$ be the position of the tip of the pointer tool relative to the reference marker coordinate system \mathbf{F}_G .
- \triangleright Give a formula for computing \mathbf{p}_{tip} , based on the available tracking system measurements

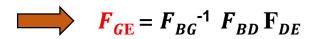
$$\mathbf{F}_{Bx} = (\mathbf{R}_{Bx}, \mathbf{p}_{Bx})$$

Example is from the Computer Integrated Surgery course, Russell H. Taylor, JHU

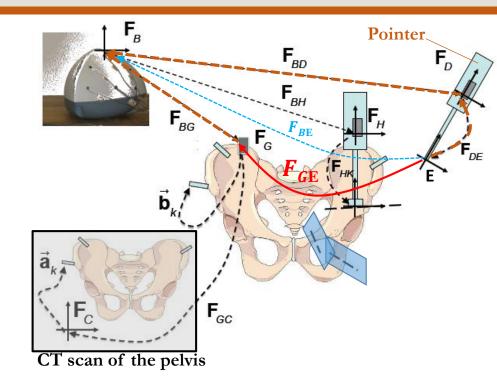
Example: Computer-Assisted Osteotomy

$$F_{BE}$$
 F_{BE}

$$F_{BG}F_{GE}=F_{BD}F_{DE}$$



 p_{GE} is the last column of F_{GE}



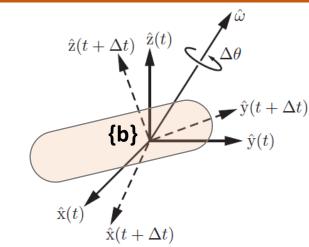
Angular Velocities

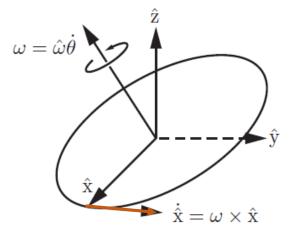
- Suppose that a **body frame** with **unit** axes $\{\hat{x}, \hat{y}, \hat{z}\}$ is attached to a **rotating body**.
 - If we examine the body frame at times \mathbf{t} and $\mathbf{t}+\Delta\mathbf{t}$, the change in frame orientation can be described as a rotation of angle $\Delta\theta$ about some unit axis $\widehat{\boldsymbol{\omega}}$ passing through the origin.
 - The axis $\widehat{\omega}$ is **coordinate-free**; it is not **yet** represented in any particular reference frame.
 - As t approaches zero, the ratio $\Delta\theta/\Delta t$ becomes the rate of rotation $\dot{\theta}$, and $\hat{\omega}$ is the <u>instantaneous</u> axis of rotation. Hence, angular velocity W is:

$$\mathbf{w} = \hat{\mathbf{w}}\dot{\theta}$$
.

 $\hat{\mathbf{x}} = \mathbf{w} \times \hat{\mathbf{x}},$ $\mathbf{\hat{x}} = \mathbf{w} \times \hat{\mathbf{x}},$ $\mathbf{\hat{y}} = \mathbf{w} \times \hat{\mathbf{y}},$ $\mathbf{\hat{y}} = \mathbf{w} \times \hat{\mathbf{y}},$

$$\dot{\hat{z}} = w \times \hat{z}$$





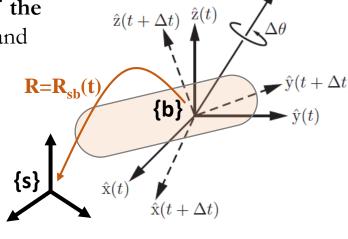
To express these equations in coordinates, we have to choose a reference frame in which to represent w typically the fixed frame {s} or the body frame {b}.

Fixed-Frame Angular Velocities

- Let R(t) be the rotation matrix describing the **orientation of the body frame with respect to the fixed frame {s}** at time t and
 R(t) is its time rate of change.
- Let $R(t) = [r_1(t); r_2(t); r_3(t)]$ where $\mathbf{r_i}$ is the representation of the corresponding **body frame axis** in the fixed frame $\{s\}$.
- At a specific time t, let $\omega_s \in \mathbb{R}^3$ be the angular velocity ω expressed in **fixed-frame** then we have:

$$\dot{r}_i = \omega_s \times r_i, \qquad i = 1, 2, 3.$$

OR
$$\dot{R} = [\omega_s \times r_1 \ \omega_s \times r_2 \ \omega_s \times r_3]$$



$$\mathbf{w} = \hat{\mathbf{w}}\dot{\theta}$$
.

$$[x] = \begin{bmatrix} 0 & -x_3 & x_2 \\ x_3 & 0 & -x_1 \\ -x_2 & x_1 & 0 \end{bmatrix}$$
$$a \times b = (a)^b.$$

We can rewrite $\omega_s \times r_i$ as $[\omega_s]R$, where $[\omega_s]$ is a 3×3 skewsymmetric matrix representation of $\omega_s \in \mathbb{R}^3$. Hence:

Skew-Symmetric angular velocity of ω represented in the fixed frame

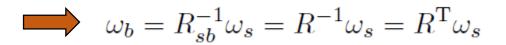
$$[\omega_s]R = \dot{R}$$
$$[\omega_s] = \dot{R}R^{-1}$$

Time rate of change of the orientation of the body frame with respect to the fixed frame {s}

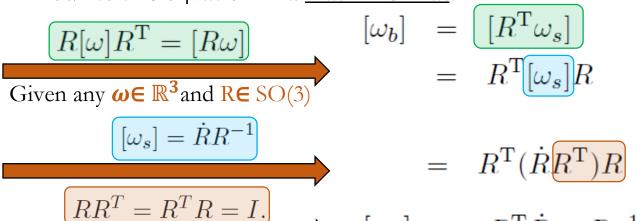
Body-Frame Angular Velocities

Let ω_s and ω_b be two different vector representations of the same angular velocity w expressed in the fixed and body-frame coordinates, respectively. Hence: $\omega_s = R_{sb}\omega_b$.

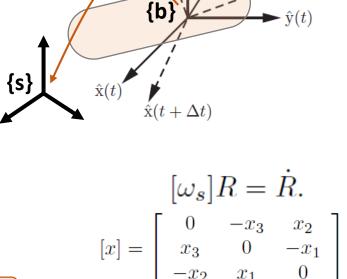
R=R_{sb}(t)



Let us now use the skew-symmetric operator [.] to rewrite this equation in a matrix format:



Skew-Symmetric angular velocity of ω represented in the **body frame**



Time rate of change of the orientation of the body frame with respect to the fixed frame {s}

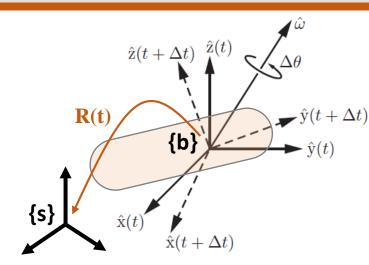
 $= R^{\mathrm{T}}\dot{R} = R^{-1}\dot{R}.$

Angular Velocities

Let R(t) denote the <u>orientation of the rotating</u> frame as seen from the fixed frame. Denote by w the angular velocity of the rotating frame. Then

$$\dot{R}R^{-1} = [\omega_s]$$

$$R^{-1}\dot{R} = [\omega_b]$$



where $\omega_s \in \mathbb{R}^3$ is the **fixed-frame vector** representation of \mathbf{w} and $[\omega_s] \in \mathbf{so}(3)$ is its 3×3 matrix representation, and where $\omega_b \in \mathbb{R}^3$ is the **body-frame vector** representation of \mathbf{w} and $[\omega_b] \in \mathbf{so}(3)$ is its 3×3 matrix representation.

- \triangleright It is important to note that the **fixed-frame angular velocity** ω_s does **not** depend on the choice of body frame.
- \triangleright <u>Similarly</u>, the body-frame angular velocity ω_b does not depend on the choice of fixed frame.
- An angular velocity expressed in an arbitrary frame {d} can be represented in another frame {c} if we know the rotation that takes {c} to {d}:

$$\omega_c = R_{cd}\omega_d$$

Twists

We now consider both the linear and angular velocities of a moving frame. Let

$$T_{sb}(t) = T(t) = \begin{bmatrix} R(t) & p(t) \\ 0 & 1 \end{bmatrix}$$

 \triangleright Let us pre-multiply $\dot{\mathbf{T}}$ by \mathbf{T}^{-1} :

$$T^{-1}\dot{T} = \begin{bmatrix} R^{T} & -R^{T}p \\ 0 & 1 \end{bmatrix} \begin{bmatrix} \dot{R} & \dot{p} \\ 0 & 0 \end{bmatrix}$$

$$= \begin{bmatrix} R^{T}\dot{R} & R^{T}\dot{p} \\ 0 & 0 \end{bmatrix}$$

$$= \begin{bmatrix} \omega_{b} & v_{b} \\ 0 & 0 \end{bmatrix}. \quad v_{b} = R^{T}\dot{p} \text{ is the linear velocity of the origin of } \{b\} \text{ expressed in } \{b\}.$$

T⁻¹ T represents the linear and angular velocities of the moving frame relative to the stationary frame {b} currently aligned with the moving frame (i.e., <u>current body</u> <u>frame</u>).

{b}

Body Twist

 \triangleright We define the **spatial velocity in the body frame**, or simply the **body twist** ν_b as:

$$\mathcal{V}_b = \left[\begin{array}{c} \omega_b \\ v_b \end{array} \right] \in \mathbb{R}^6$$

As it is convenient to have a skew-symmetric matrix representation of an angular velocity vector, it is convenient to have a matrix representation of a twist as:

$$T^{-1}\dot{T} = [\mathcal{V}_b] = \begin{bmatrix} [\omega_b] & v_b \\ 0 & 0 \end{bmatrix} \in se(3) \qquad [x] = \begin{bmatrix} 0 & -x_3 & x_2 \\ x_3 & 0 & -x_1 \\ -x_2 & x_1 & 0 \end{bmatrix}$$

where $[\omega] \in so(3)$ and $v_b \in \mathbb{R}^3$.

- \triangleright The set of all 4×4 matrices of this form is called se(3).
- > se(3) comprises the matrix representations of the twists associated with the rigid-body configurations SE(3).

Spatial Twist

 \triangleright let us now evaluate TT^{-1} :

$$\dot{T}T^{-1} = \begin{bmatrix} \dot{R} & \dot{p} \\ 0 & 0 \end{bmatrix} \begin{bmatrix} R^{\mathrm{T}} & -R^{\mathrm{T}}p \\ 0 & 1 \end{bmatrix} \\
= \begin{bmatrix} \dot{R}R^{\mathrm{T}} & \dot{p} - \dot{R}R^{\mathrm{T}}p \\ 0 & 0 \end{bmatrix} \\
= \begin{bmatrix} \omega_s & v_s \\ 0 & 0 \end{bmatrix}.$$

$$\begin{array}{ccc}
\dot{R}R^{-1} & = & [\omega_s] \\
R^{-1}\dot{R} & = & [\omega_b]
\end{array}$$

where $[\omega_s] \in so(3)$ and $v_s \in \mathbb{R}^3$.

 \triangleright We can assemble ω_s and v_s into a six-dimensional twist:

$$\mathcal{V}_s = \begin{bmatrix} \omega_s \\ v_s \end{bmatrix} \in \mathbb{R}^6, \quad \begin{bmatrix} [\mathcal{V}_s] = \begin{bmatrix} [\omega_s] & v_s \\ 0 & 0 \end{bmatrix} = \dot{T}T^{-1} \in se(3)$$

- \triangleright We call ν_s the spatial velocity in the space frame, or simply the spatial twist.
- We can obtain ν_b from ν_s as follows:

$$[\mathcal{V}_b] = T^{-1} \dot{T} \qquad [\mathcal{V}_s] = T [\mathcal{V}_b] T^{-1}$$
$$= T^{-1} [\mathcal{V}_s] T.$$

Adjoint Representation

$$[\mathcal{V}_s] = T \left[\mathcal{V}_b \right] T^{-1} \longrightarrow \left[\mathcal{V}_s \right] = \left[\begin{array}{c} R[\omega_b] R^{\mathrm{T}} \\ 0 \end{array} \right] - \left[R[\omega_b] R^{\mathrm{T}} \right] + Rv_b$$

$$\begin{array}{c}
R[\omega]R^{\mathrm{T}} = [R\omega] \\
[\omega]p = -[p]\omega \text{ for } p, \omega \in \mathbb{R}^{3}
\end{array}$$

$$\begin{bmatrix}
\omega_{s} \\
v_{s}
\end{bmatrix} = \begin{bmatrix}
R & 0 \\
[p]R & R
\end{bmatrix}
\begin{bmatrix}
\omega_{b} \\
v_{b}
\end{bmatrix}$$

$$\begin{matrix}
\nu_{b}
\end{matrix}$$

Math tricks to factor out the \omega_b

Figure $T = (R; p) \in SE(3)$, its **adjoint** representation [Ad_T] is

$$[\mathrm{Ad}_T] = \left[\begin{array}{cc} R & 0 \\ [p]R & R \end{array} \right] \in \mathbb{R}^{6 \times 6}$$

For any $\nu \in \mathbb{R}^6$, the adjoint map associated with T is

$$\mathcal{V}' = [\mathrm{Ad}_T]\mathcal{V} \equiv \mathcal{V}' = \mathrm{Ad}_T(\mathcal{V})$$

 \triangleright In terms of the matrix form $[\nu] \in \text{se}(3)$ of $\nu \in \mathbb{R}^6$: $[\mathcal{V}'] = T[\mathcal{V}]T^{-1}$

Adjoint Properties

$$ightharpoonup$$
 Let T_1 ; $T_2 \in SE(3)$ and $v = (\omega; v)$ and $[Ad_T] = \begin{bmatrix} R & 0 \\ [p]R & R \end{bmatrix} \in \mathbb{R}^{6 \times 6}$ then,

$$\operatorname{Ad}_{T_1}\left(\operatorname{Ad}_{T_2}(\mathcal{V})\right) = \operatorname{Ad}_{T_1T_2}(\mathcal{V}) \quad or \quad [\operatorname{Ad}_{T_1}][\operatorname{Ad}_{T_2}]\mathcal{V} = [\operatorname{Ad}_{T_1T_2}]\mathcal{V}.$$

 \triangleright Also, for any T \in SE(3) the following holds:

$$[\mathrm{Ad}_T]^{-1} = [\mathrm{Ad}_{T^{-1}}]$$

Proof: Choosing $T_1 = T^{-1}$ and $T_2 = T$,

$$\operatorname{Ad}_{T^{-1}}(\operatorname{Ad}_T(\mathcal{V})) = \operatorname{Ad}_{T^{-1}T}(\mathcal{V}) = \operatorname{Ad}_I(\mathcal{V}) = \mathcal{V}.$$

$$[\mathrm{Ad}_{T_1}][\mathrm{Ad}_{T_2}]\mathcal{V} = [\mathrm{Ad}_{T_1T_2}]\mathcal{V}.$$

Summary of Twist Results

Given a fixed (space) frame $\{s\}$, a body frame $\{b\}$, and a differentiable $T_{sb}(t) \in SE(3)$

1. Matrix representation of the body twist

$$T_{sb}(t) = \begin{bmatrix} R(t) & p(t) \\ 0 & 1 \end{bmatrix}$$

$$\begin{array}{|c|c|}
\hline
T_{sb}^{-1}\dot{T}_{sb} = [\mathcal{V}_b] \\
\hline
0 & 0
\end{array} \in se(3)$$

2. Matrix representation of the spatial twist

$$\begin{bmatrix}
\dot{T}_{sb}T_{sb}^{-1} = [\mathcal{V}_s] \\
0 & 0
\end{bmatrix} \in se(3)$$

3. The twists ν_s and ν_b are related by

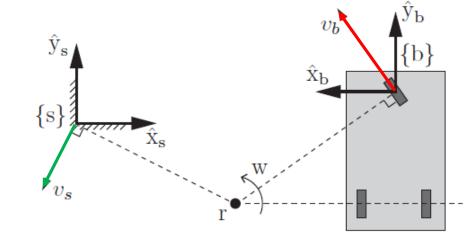
$$\mathcal{V}_{s} = \begin{bmatrix} \omega_{s} \\ v_{s} \end{bmatrix} = \begin{bmatrix} R & 0 \\ [p]R & R \end{bmatrix} \begin{bmatrix} \omega_{b} \\ v_{b} \end{bmatrix} = \begin{bmatrix} \operatorname{Ad}_{T_{sb}} \end{bmatrix} \mathcal{V}_{b},
\mathcal{V}_{b} = \begin{bmatrix} \omega_{b} \\ v_{b} \end{bmatrix} = \begin{bmatrix} R^{T} & 0 \\ -R^{T}[p] & R^{T} \end{bmatrix} \begin{bmatrix} \omega_{s} \\ v_{s} \end{bmatrix} = \begin{bmatrix} \operatorname{Ad}_{T_{bs}} \end{bmatrix} \mathcal{V}_{s}.$$

4. For any two frames {c} and {d}, a twist represented as v_c in {c} is related to its representation v_d in {d} by: $v_c = [\mathrm{Ad}_{T_{cd}}] v_d$, $v_d = [\mathrm{Ad}_{T_{dc}}] v_c$

Example

Figure shows a top view of a car, with a single steerable front wheel, driving on a plane. The $\hat{\mathbf{z}}_b$ -axis of the body frame $\{b\}$ is into the page and the $\hat{\mathbf{z}}_s$ -axis of the fixed frame $\{s\}$ is out of the page.

The angle of the front wheel of the car causes the car's motion to be a pure angular velocity w = 2 rad/s about an axis out of the page at the point r in the plane.



If $\mathbf{r_s} = (2;-1;0)$ or $\mathbf{r_b} = (2;-1.4;0)$, calculate twists $\mathbf{v_s}$ and $\mathbf{v_b}$ and verify them using corresponding adjoints.

Example

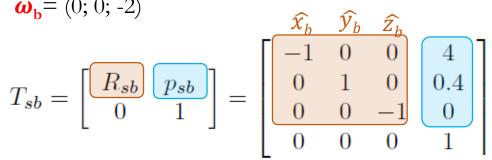
$$\mathbf{r_s} = (2;-1;0)$$

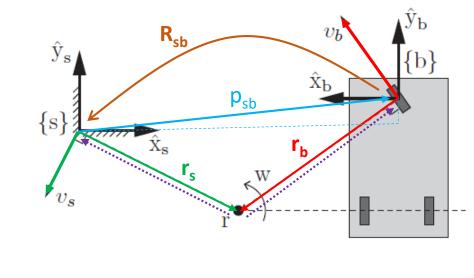
 $\mathbf{r_b} = (2;-1.4;0)$

$$\boldsymbol{\omega}_{\mathbf{s}} = (0; 0; 2)$$

$$\omega_{b} = (0; 0; -2)$$

$$T_{sb} = \begin{bmatrix} R_{sb} & p_{sb} \\ 0 & 1 \end{bmatrix} = \begin{bmatrix} R_{sb} & P_{sb} \\ 0 & 1 \end{bmatrix}$$





$$\mathcal{V}_s = \begin{bmatrix} \omega_s \\ v_s \end{bmatrix} = \begin{bmatrix} \mathbf{R} & 0 \\ \mathbf{p}\mathbf{R} & \mathbf{R} \end{bmatrix} \begin{bmatrix} \omega_b \\ v_b \end{bmatrix} = \begin{bmatrix} \operatorname{Ad}_{T_{sb}} \end{bmatrix} \mathcal{V}_b$$

$$[x] = \begin{bmatrix} 0 & -x_3 & x_2 \\ x_3 & 0 & -x_1 \\ -x_2 & x_1 & 0 \end{bmatrix}$$

[p] is the **skew-symmetric matrix** representation of vector p_{sb}

References

- Murray, R.M., Li, Z., Sastry, S.S., "A Mathematical Introduction to Robotic Manipulation.", Chapter 2.
- Corke, Peter. "Robotics, vision and control: fundamental algorithms in MATLAB®" second, completely revised. Vol. 118. Springer, 2017, **Chapter 2.**
- Lynch and Park, "Modern Robotics," Cambridge U. Press, 2017, Chapter 3.