Introduction to Automatic Controls

Stability

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Lesson Objectives

- Formally define stability
- For linear systems
 - We have already seen a strong pattern between a system's eigenvalues. In this lesson, we will:
 - Review system response with respect to stability
 - Provide additional insight for determining which systems are stable (or under what conditions).
- For nonlinear systems
 - Linearization and stability
 - Lyapunov functions for evaluating stability

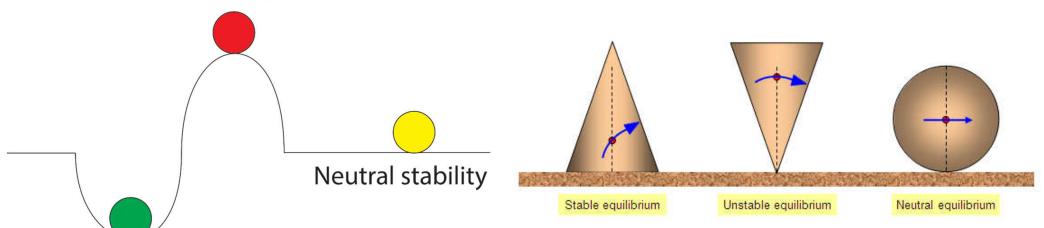
Formal definition of stability

A system is <u>stable</u> if that system's response stays arbitrarily near some value, \mathbf{z}_a , for all of time greater than some value, t_f .

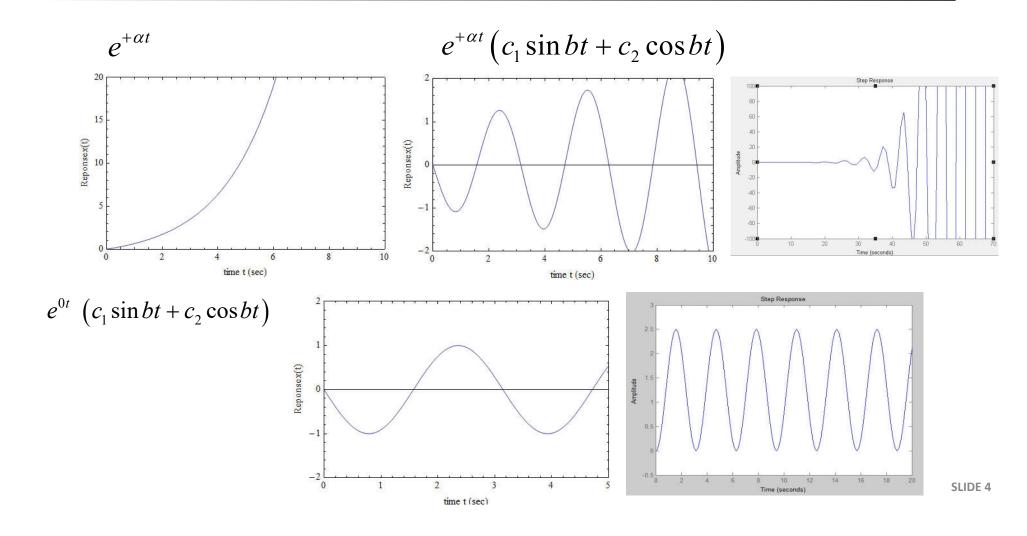
$$\|\mathbf{z}_a - \mathbf{z}_b\| < \delta \Rightarrow \|\mathbf{z}(t; \mathbf{z}_b) - \mathbf{z}(t; \mathbf{z}_a)\| < \varepsilon \text{ for all } t > 0$$

Unstable

Stable



Unstable, neutrally stable responses



Common 2nd order example

Given:

$$\ddot{x} + a_1 \dot{x} + a_2 x = bu(t)$$

Where:

$$x(0) = w b = 1$$

$$\dot{x}(0) = v u = \begin{cases} 0 \\ 1 \end{cases} t < t_0$$

$$u = \begin{cases} 1 \\ t \ge t_0 \end{cases}$$

$$\frac{d}{dt} \begin{bmatrix} z_1 \\ z_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -a_2 & -a_1 \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$

$$y = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} z_1 \\ z_2 \end{bmatrix}$$

Find:

what happens as a_1 and a_2 vary?

Solve:

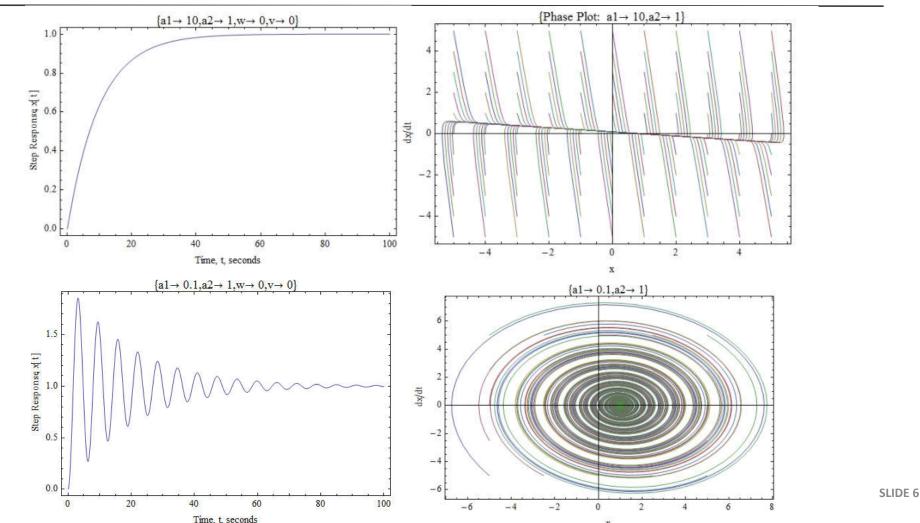
Using methods from previous lessons:

$$y(t) = \frac{1}{a_2} + C_1 e^{\frac{1}{2}(-a_1 - \sqrt{a_1^2 - 4a_2})t} + C_2 e^{\frac{1}{2}(-a_1 + \sqrt{a_1^2 - 4a_2})t}$$

Which is used to generate the following examples for a variety of system parameters and initial conditions that illustrate common stability modalities.

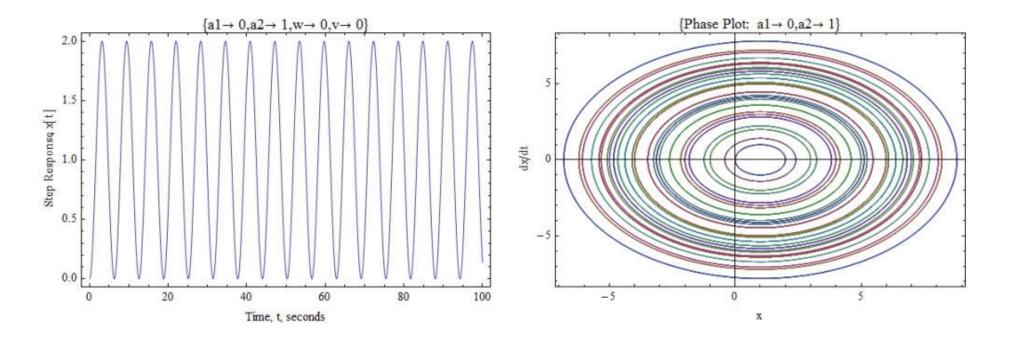
Asymptotically Stable Examples $x(t) = \frac{1}{a_2} + C_1 e^{\frac{1}{2}(-a_1 - \sqrt{a_1^2 - 4a_2})t} + C_2 e^{\frac{1}{2}(-a_1 + \sqrt{a_1^2 - 4a_2})t}$

$$x(t) = \frac{1}{a_2} + C_1 e^{\frac{1}{2}(-a_1 - \sqrt{a_1^2 - 4a_2})t} + C_2 e^{\frac{1}{2}(-a_1 + \sqrt{a_1^2 - 4a_2})t}$$



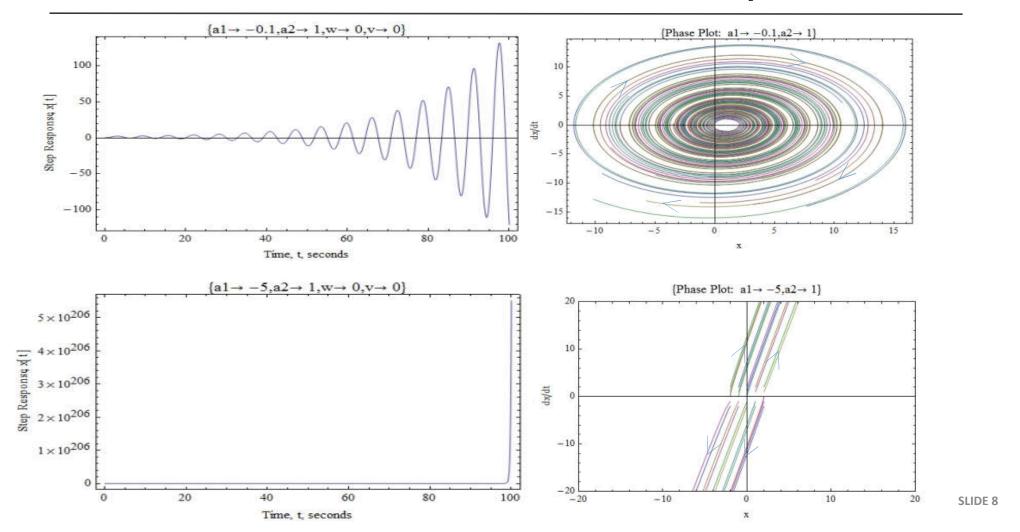
Neutrally stable examples

$$x(t) = \frac{1}{a_2} + C_1 e^{\frac{1}{2}(-a_1 - \sqrt{a_1^2 - 4a_2})t} + C_2 e^{\frac{1}{2}(-a_1 + \sqrt{a_1^2 - 4a_2})t}$$



Unstable examples

$$x(t) = \frac{1}{a_2} + C_1 e^{\frac{1}{2}(-a_1 - \sqrt{a_1^2 - 4a_2})t} + C_2 e^{\frac{1}{2}(-a_1 + \sqrt{a_1^2 - 4a_2})t}$$



Stability in higher order systems

Example: For what values of α (if any) is the following system stable?

$$\frac{d\mathbf{z}}{dt} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & \alpha & -2 \end{bmatrix} \mathbf{z} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} u$$

$$y = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \mathbf{z} + \begin{bmatrix} 4 \end{bmatrix} u$$

Solve:

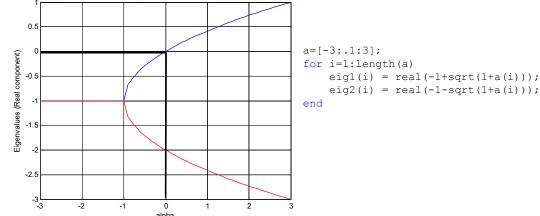
$$\det(\lambda \mathbf{I} - \mathbf{A}) = \begin{bmatrix} \lambda & -1 & 0 \\ 0 & \lambda & -1 \\ 0 & -\alpha & \lambda + 2 \end{bmatrix} \mathbf{z}$$

$$\lambda \left(\lambda^2 + 2\lambda - \alpha \right) + 0 = 0$$

Therefore,

$$\lambda_{1} = 0 \qquad \lambda_{2,3} = \frac{-2 \pm \sqrt{4 + 4\alpha}}{2} = \frac{-b \pm \sqrt{b^{2} - 4ac}}{2a}$$
$$= \frac{-2 \pm 2\sqrt{1 + \alpha}}{2}$$
$$= -1 \pm \sqrt{1 + \alpha}$$

The system is – at best – neutrally stable. Is there a range where the system is unstable?



So the system becomes unstable if $\alpha > 0$.

So far...

- Presented formal definition of stability
- For Linear Systems
 - We have seen many examples.
 - Stability can be determined with respect to system parameters.
 - But method can get burdensome.
 - Note that "all coefficients of the Characteristic Equation must be nonzero and have the same sign" in order for the system to be stable.
 - This is a <u>necessary</u>, but NOT <u>sufficient</u> condition for stability.

$$6\lambda^{5} - 5\lambda^{4} + 3\lambda^{3} + 2\lambda^{2} + 2\lambda + 3 = 0 \leftarrow NOT_stable$$

$$6\lambda^{5} + 5\lambda^{4} + 3\lambda^{3} + 2\lambda^{2} + 3 = 0 \leftarrow NOT_stable$$

$$6\lambda^{5} + 5\lambda^{4} + 3\lambda^{3} + 2\lambda^{2} + 2\lambda + 3 = 0 \leftarrow MAYBE_stable$$

Still need to deal with stability of nonlinear systems.

Nonlinear Systems: Multiple Options

- Determining stability for nonlinear systems using linearization.
- Exploit assumption that a system is properly controlled.
 - This allows us to treat some nonlinear systems as linear.

Apply Lyapunov stability analysis to determine if a solution to a

nonlinear dynamical system is stable.



Alexandr Lyapunov (1857-1918)

Previous example

In our example...

(0,0) and (-3,0) are equilibrium points;

We can linearize about the equilibrium points and examine stability in near z_{ρ}

Find the Jacobian for the system....

$$\mathbf{J} = \begin{bmatrix} \frac{\partial f_1}{\partial z_1} & \frac{\partial f_1}{\partial z_2} \\ \frac{\partial f_2}{\partial z_1} & \frac{\partial f_2}{\partial z_2} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -3 - 2z_1 & -0.6 \end{bmatrix}$$

Like a gradient, but for multiple variables

Find the Jacobian for the system....
$$\mathbf{J} = \begin{bmatrix} \frac{\partial f_1}{\partial z_1} & \frac{\partial f_1}{\partial z_2} \\ \frac{\partial f_2}{\partial z_1} & \frac{\partial f_2}{\partial z_2} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -3 & -0.6 \end{bmatrix}$$

$$\mathbf{J} = \begin{bmatrix} 0 & 1 \\ -3 & -0.6 \end{bmatrix} \Rightarrow \lambda = \begin{bmatrix} -0.3000 + 1.7059i \\ -0.3000 - 1.7059i \end{bmatrix}$$
so stable near (0,0)
$$\mathbf{J} = \begin{bmatrix} 0 & 1 \\ -3 & -0.6 \end{bmatrix} \Rightarrow \lambda = \begin{bmatrix} 1.4578 \\ -2.0578 \end{bmatrix}$$
ut for multiple variables

so unstable saddle point at (-3,0)

Stability in the 'region' of an equilibrium point.

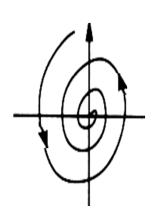
$$\mathbf{J} = \begin{bmatrix} 0 & 1 \\ -3 & -0.6 \end{bmatrix} \Rightarrow \lambda = \begin{bmatrix} -0.3000 + 1.7059i \\ -0.3000 - 1.7059i \end{bmatrix}$$

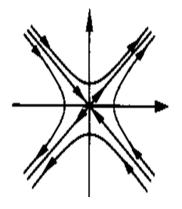
At (-3,0)

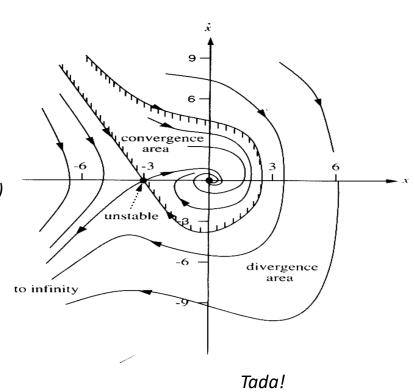
so stable near (0,0)

$$\mathbf{J} = \begin{bmatrix} 0 & 1 \\ 3 & -0.6 \end{bmatrix} \Rightarrow \lambda = \begin{bmatrix} 1.4578 \\ -2.0578 \end{bmatrix}$$

so unstable saddle point at (-3,0)

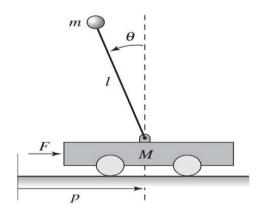






Inverted Pendulum Example

Given: A inverted pendulum on a moving cart:



Determine: If the inverted pendulum system shown is <u>stable</u> if the pendulum is initially perpendicular to the ground.

Solution:

$$\sum F_{i} = (M + m)\ddot{x}$$

$$\sum \tau_{i} = I\ddot{\theta}$$

$$(M + m)\ddot{x} = ml\cos(\theta)\ddot{\theta} - c\dot{x} - ml\sin(\theta)\dot{\theta}^{2} + F$$

$$(J + ml^{2})\ddot{\theta} = ml\cos(\theta)\ddot{x} - \gamma\dot{\theta} + mgl\sin(\theta)$$

F is the input, linearize at $\theta = 0^{\circ}$ (i.e. $\cos(\theta) = 1 \& \sin(\theta) = \theta$.)

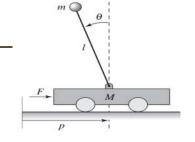
$$(M + m)\ddot{x} = m l(1)\ddot{\theta} - (0)\dot{x} - m l\theta \dot{\theta}^2 + u$$

$$(J + m l^2)\ddot{\theta} = m l(1)\ddot{x} - (0)\dot{\theta} + m g l\theta$$

Put in matrix form...

$$\begin{bmatrix} (M+m) & -ml \\ -ml & (J+ml^2) \end{bmatrix} \begin{bmatrix} \ddot{x} \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} -ml\theta\dot{\theta}^2 \\ mgl\theta \end{bmatrix} + \begin{bmatrix} u \\ 0 \end{bmatrix}$$

Inverted pendulum example



$$\begin{bmatrix} (M+m) & -ml \\ -ml & (J+ml^2) \end{bmatrix} \begin{bmatrix} \ddot{x} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} -ml\theta\dot{\theta}^2 \\ mgl\theta \end{bmatrix} + \begin{bmatrix} u \\ 0 \end{bmatrix}$$

When <u>controlled</u>, the angular velocity should be close to zero, so we can ignore terms quadratic and higher angular velocity terms.

$$\begin{bmatrix} \ddot{x} \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} (M+m) & -ml \\ -ml & (J+ml^2) \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ mgl\theta \end{bmatrix} + \begin{bmatrix} u \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} \ddot{x} \\ \dot{\theta} \end{bmatrix} = \frac{1}{(M+m)(J+ml^2) - m^2l^2} \begin{bmatrix} (J+ml^2) & ml \\ ml & (M+m) \end{bmatrix} \begin{bmatrix} 0 \\ mgl\theta \end{bmatrix} + \begin{bmatrix} u \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} \ddot{x} \\ \dot{\theta} \end{bmatrix} = \frac{1}{\mu} \begin{bmatrix} (J+ml^2) & -ml \\ ml & (M+m) \end{bmatrix} \begin{bmatrix} 0 \\ mgl\theta \end{bmatrix} + \begin{bmatrix} u \\ 0 \end{bmatrix}$$

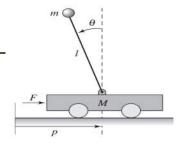
Let's define the states as.

$$\mathbf{z} = \begin{bmatrix} x & \theta & \dot{x} & \dot{\theta} \end{bmatrix}^T$$

Inverted Pendulum Example

Note, in this case that:

$$\mathbf{y} = \mathbf{C} \, \mathbf{z} + \mathbf{D} \, u = \begin{bmatrix} 0 & 0 & 1 & 0 \end{bmatrix} \mathbf{z}$$



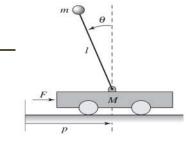
And our system is....

$$\mathbf{z} = \begin{bmatrix} x & \theta & \dot{x} & \dot{\theta} \end{bmatrix}^{T} \qquad \begin{bmatrix} \ddot{x} \\ \ddot{\theta} \end{bmatrix} = \frac{1}{\mu} \begin{bmatrix} (J+ml^{2}) & -ml \\ ml & (M+m) \end{bmatrix} \begin{bmatrix} 0 \\ mgl\theta \end{bmatrix} + \begin{bmatrix} u \\ 0 \end{bmatrix}$$

$$\dot{\mathbf{z}} = \mathbf{A} \mathbf{z} + \mathbf{B} u = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & \frac{m^{2}l^{2}g}{\mu} & 0 & 0 \\ 0 & \frac{(M+m)mgl}{\mu} & 0 & 0 \end{bmatrix} \begin{bmatrix} x \\ \theta \\ \dot{x} \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \frac{J+ml^{2}}{\mu} \\ \frac{lm}{\mu} \end{bmatrix} u$$

This system is linearized at θ =0 assuming that the angular velocity is small. So is the system stable?

Inverted Pendulum Example



$$\dot{\mathbf{z}} = \mathbf{A} \, \mathbf{z} + \mathbf{B} \, u = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & \frac{m^2 l^2 g}{\mu} & 0 & 0 \\ 0 & \frac{(M+m) m g l}{\mu} & 0 & 0 \end{bmatrix} \begin{bmatrix} x \\ \theta \\ \dot{x} \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \frac{J+m l^2}{\mu} \\ \frac{lm}{\mu} \end{bmatrix} u$$

This system is linearized at $\theta=0$ assuming that the angular velocity is small. So is the system stable?

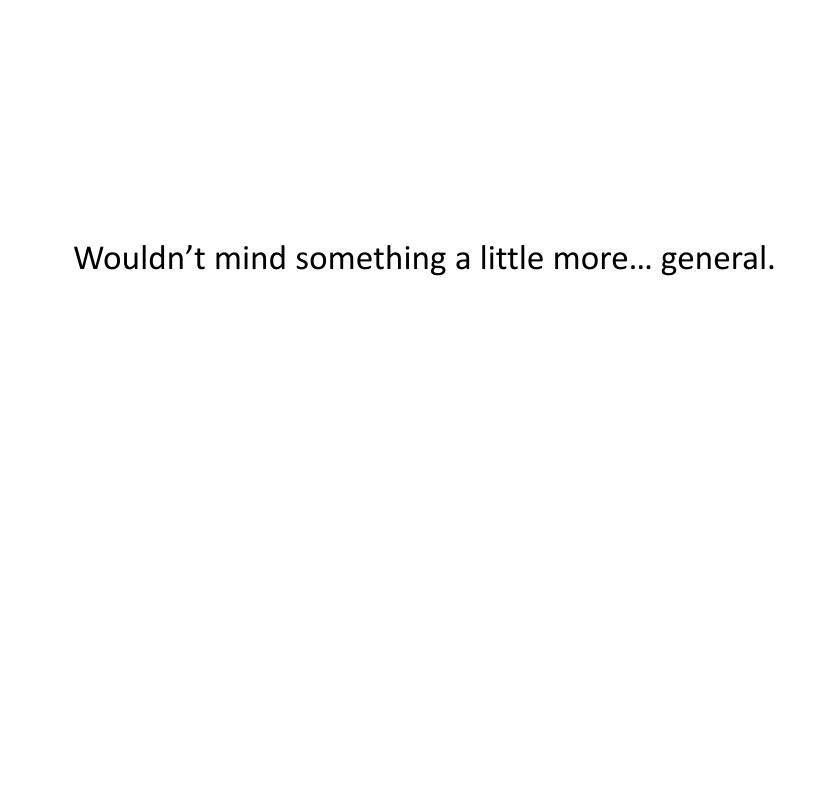
$$\det (\lambda \mathbf{I} - \mathbf{A}) = \det \begin{bmatrix} \lambda & 0 & -1 & 0 \\ 0 & \lambda & 0 & -1 \\ 0 & -\frac{m^2 l^2 g}{\mu} & \lambda & 0 \\ 0 & -\frac{(M+m) m g l}{\mu} & 0 & \lambda \end{bmatrix}$$

$$CE = \lambda \left(\lambda \left(\lambda^{2} \right) - 1 \left(-\frac{(M+m)mgl}{\mu} \lambda \right) \right) - 1 \left(0 \right)$$

$$= \lambda^{4} - \lambda^{2} \frac{(M+m)mgl}{\mu}$$

From this we get that the system's eigenvalues at this equilibrium point are:

$$\lambda = 0, 0, \pm \sqrt{\frac{(M + m) mgl}{\mu}}$$
 Therefore the system is unstable for any mechanical system qualifying as a pendulum!



Lyapunov Functions

Lyapunov Function: $V(\mathbf{x}): \mathbb{R}^n \to \mathbb{R}^1$ is an "energy-like" function that is <u>nonnegative</u> and always decreasing along trajectories of a given system. If this is true, then we can conclude that the minimum of the function V is a stable equilibrium point.

Before presenting a more formal definition, we need a few definitions.

 $V(\mathbf{x}) > 0$ for all $\mathbf{x} \neq 0$ AND $V(\mathbf{0}) = 0$ A function is *positive definite* if:

 $V(\mathbf{x}) < 0$ for all $\mathbf{x} \neq 0$ AND $V(\mathbf{0}) = 0$ A function is *negative definite* if:

 $V(\mathbf{x}) \ge 0$ for all $\mathbf{x} \ne 0$ AND $V(\mathbf{0}) = 0$ A function is *positive semi-definite* if:

(i.e. the function can be 0 at pts other than x=0.)

Examples where
$$\mathbf{x} \in \mathbb{R}^2$$

$$V_1 = x_1^2$$

Is only positive semi-definite since its value can be 0 when $\mathbf{x}=(0,a)$

$$V_1 = x_1^2 \qquad V_2 = x_1^2 + x_2^2$$

Is only positive definite since its value can only be 0 when $\mathbf{x}=(0,0)$

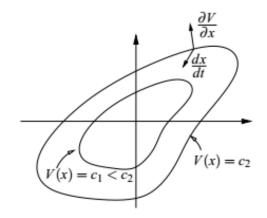
Lyapunov Stability Theorem

Let V be a nonnegative function on \mathbb{R}^n and let \dot{V} represent the time derivative of V along trajectories of the system dynamics.

$$\dot{V} = \frac{\partial V}{\partial \mathbf{x}} \frac{d\mathbf{x}}{dt} = \frac{\partial V}{\partial \mathbf{x}} F(\mathbf{x})$$

Let $B_r=B_r(0)$ be a ball of radius r around the origin. If there exists an r>0 such that V is positive definite and \dot{V} is negative semi-definite for all $\mathbf{X}\in B_r$

Then, \mathbf{x} =0 is locally stable in the "sense of Lyapunov." If V is positive definite and \dot{V} is negative definite, in B_n , then \mathbf{x} =0 is locally asymptotically stable.



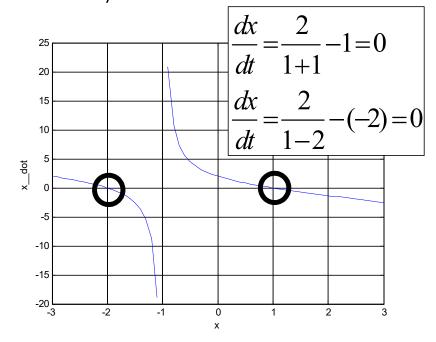
If we are interested in examining an area B_r around an equilibrium point that is *not* at the origin, then we can can rewrite the equations in a new set of coordinates (i.e. $z=x-x_\rho$)

Example

Given: Simple scalar nonlinear system

$$\frac{dx}{dt} = \frac{2}{1+x} - x$$

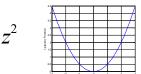
Note, this system has two equilibrium points (at 1 and -2)

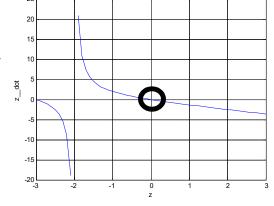


Let's consider the equilibrium point x=1 by rewriting the dynamic equations in a coordinate frame that moves this equilibrium point to the origin (i.e. z=x-1 thus x=z+1)

$$\frac{dz}{dt} = \frac{2}{2+z} - z - 1$$

Let's try the candidate Lyapunov function





Which is globally positive definite. The derivative of *V* along the trajectories is given by:

$$\dot{V}(z) = \frac{\partial V}{\partial z} \dot{z} = z \left(\frac{2}{2+z} - z - 1 \right) = \frac{2z}{2+z} - z^2 - z$$

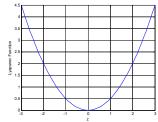
Consider an interval (1 dimensional region) B_r where r < 2.

Example cont'd

$$\frac{dx}{dt} = \frac{2}{1+x} - x$$

Candidate Lyapunov function

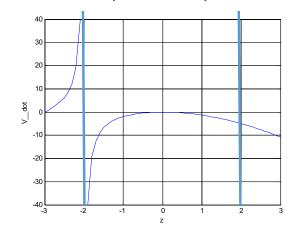
$$V(z) = \frac{1}{2}z^2$$



...is globally positive definite.

A its derivative is negative definite over B_r if r < 2.

$$\dot{V}(z) = \frac{\partial V}{\partial z} \dot{z} = z \left(\frac{2}{2+z} - z - 1 \right) = \frac{2z}{2+z} - z^2 - z$$



Lyapunov's method requires that one <u>choose</u> a positive definite Lyapunov function (candidate) and then <u>prove</u> that its derivative is negative (semi) definite in the region of interest.

For proof in this case, let us consider all values of z>-2, or z+2>0. Since z+2>0 and V_{dot} must be negative, then

$$(z+2)\left(\frac{2z}{2+z} - z^2 - z\right) < 0$$

$$2z - (z^2 + z)(z+2) < 0$$

$$2z - z^3 + z^2 + 2z^2 + 2z < 0$$

$$-z^3 + 3z^2 < 0$$

$$-z^2(z+3) < 0$$

Which provides more formal proof than the graph to the left. So we proved that our original function is asymptotically stable at x_e =1.

Pendulum Example

Given: Normalized model for a hanging pendulum

$$\frac{dx_1}{dt} = x_2$$

$$\frac{dx_2}{dt} = -\sin x_1$$

Where x_1 is the angle between the pendulum and the vertical and a positive value occurring with a clockwise rotation.

There is an equilibrium when

$$x_1 = x_2 = 0$$

when the pendulum is at rest and hanging straight down.

One candidate Lyapunov function is

$$V(x) = 1 - \cos x_1 + \frac{1}{2}x_2^2 \approx \frac{1}{2}x_1^2 + \frac{1}{2}x_2^2$$

Where the Taylor series approximation is positive definite.

Next we find the derivative of the Lyapunov Function...

$$V(x) = \dot{x}_1 \sin x_1 + \dot{x}_2 x_2$$
$$= x_2 \sin x_1 - x_2 \sin x_1$$
$$= 0$$

Which is negative semi-definite, proving the system is stable, but not asymptotically stable. (which make sense since there is no frictional term in our dynamic equations)

Summary

- We formally defined stability
- We can determine the stability of linear systems including as a function of a system's parameters.
- We can use linearization to ascertain the stability of a nonlinear system around a given equilibrium point
 - The analysis required to evaluate stability can be extensive.
- Lyapunov Functions can be utilized the determine the stability of nonlinear systems.
 - This approach can provide additional insight into the region of stability about an equilibrium (more so than linearization)
 - Provides a method to discuss the stability for nonlinear controllers (not a subject we will have time for in this course)
 - But functions can be hard to identify and/or prove their validity.