**This submission is original work and no part is plagiarized** (signed) \_\_\_\_\_\_\_\_\_\_\_\_\_\_\_(Date)\_\_\_\_\_\_\_\_\_\_\_



**Thapar Institute of Engineering and Technology, Patiala**

**MECHANICAL ENGINEERING DEPARTMENT**

**Assignment - 3.**

**Design against failure under *static* actions**

***UTA013 Engineering Design Project-I***



**Assignment - 3.**

**Structural Engineering Component**

**Design against failure under *static* actions**

The following tasks have been based on the lecture by Mr. Kishore Khanna on designing against structural Failure under static loads. Complete the following **individually, copying will be dealt with severely.**

**Notes:**

1. Excel spreadsheets graphs to be created for Q1, Q2 and Q3 and evaluated by end of 2 hour class.
2. The print of this word document with **graphs (with Name and Roll No in text box)** and **hand written conclusion, name and roll number on every page**, stapled together, is to be submitted in next Tutorial class (if it is a holiday, then as instructed).
3. Do not leave this assignment until the last minute to find you have some IT issue.

Despite this list, try and enjoy the assignment and try to think around the subject as much as possible and take from it any tips that you might use with your own Catapult.

**Marking Scheme: Assignment 3 (10 Marks) =10%**

**Evaluation at end of 2 Hours Tutorial: 5**

**Evaluation from printout submission: 5**

**TUTORIAL CLASS EVALUATION Q1-Q3. [5 Marks]**

1. From experiments, you measured experimentally the bending stress at failure of a timber beam (**Calculate in Excel sheet the Average Stress at failure for the experiments**).

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| --- | --- | --- | --- | --- | --- | --- | --- | --- | --- |
|  | **Measured** | | | | **Calculated** | | | | |
| Ex | Span | Width | Depth | Failure load | Failure force | M=PL/4 | y=d/2 | l=bd^3/12 | Strength σ |
|
| No | L mm | b mm | d mm | mass Kg | P N | Nmm | mm | mm^4 | Mpa(N/mm^2) |
| 1 | 300 | 9.60 | 4.73 | 4.25 | 41.6925 | 3126.9375 | 2.365 | 84.65905 | 87.35282138 |
| 2 | 300 | 9.92 | 4.80 | 4.70 | 46.107 | 3458.025 | 2.4 | 91.42272 | 90.7789661 |
| 3 | 300 | 9.64 | 4.56 | 4.85 | 47.5785 | 3568.3875 | 2.28 | 76.17112 | 106.8111376 |
| Av | 300.00000 | 9.72000 | 4.69667 | 4.60000 | 45.12600 | 3384.45000 | 2.34833 | 84.08430 | 94.98098 |

1. Using the average strength of wood from Q1 calculate the theoretical variation in failure load, P, when the span of the beam is varied over the range from 100-700 mm, for the same dimensions in Q1, and **draw a plot** of the relationship. (**Evaluated on laptop**)
2. Using the average strength of wood from Q1 calculate the theoretical variation in failure load, P, when the cross-sectional dimensions of the beam are varied over the range from square of 4-10 mm (for the same span as was used in Q1 and **draw a plot** of the relationship (**Evaluated on laptop**)
3. For Q1. Assuming the square cross section of 6mm and a span of L=200 mm calculate **theoretically** the failure stress (strength) for a simply supported beam which fails due to a 5kg weight. \_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_\_.
4. A second beam of dimensions 9x9 mm and span L=500 mm was tested and found to fail at 7kg. Theoretically, what value should it have failed at? Explain any discrepancy in your result if there is one. What do you learn from this?

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1. What do you observe from the plot of Q2?

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1. What do you observe from the plot of Q3?

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1. **Now let us address the Catapult.** Assume the length of the throwing arm of the Catapult is 240 mm from the axis of rotation to the D-ring. Select the optimum diameter and so that the arm does not fail in bending under static loading. The worst case of static loading is when the arm is fully pulled back and ready to release. You should use a peak static force of 120N in your calculation. Note! The end conditions of the arm are different to those in class experiments, i.e. it is not simply supported! Refer to notes handout to determine which equation is appropriate for this cantilevered condition. Is the diameter of the throwing arm of 28 mm adequate? Comment.

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1. For the conditions in Q8:
   1. Can the nylon cable holding the main arm in place, when cocked, resist the force without breaking? What is the FOS? You may assume that the axial failure stress of the cable is 65MPa (i.e. N/mm2) and that the cable has a circular cross section of diameter 2.4mm.

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* 1. The other end of the cable is attached to a timber dowel 20.5 mm diameter which is held in double shear by the base of the Catapult. Design the minimum diameter of dowel that is required to resist this force without it failing in shear. You may assume the shear stress capacity of the dowel is 15MPa. What is the FOS of the dowel of your Catapult?

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1. Using the library and /or the internet for referencing, compare the strength of timber in bending with a variety of other available materials. Produce a table of the relevant properties and comment on their suitability for use as the main arm in a Catapult. You will use this information as well as the analysis techniques above to help you redesign/optimize the throwing arm.

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| --- | --- | --- | --- |
| Sr No | Material | Strength in Bending | Comment |
|  | Plastic |  |  |
|  | Acrylic |  |  |
|  | Glass |  |  |
|  | Aluminum |  |  |
|  | Stainless Steel |  |  |

**Note!! The end conditions are different in Q1-Q7 (simply supported) from that of the Catapult (cantilever) in Q8 onwards. The equation for the bending stress will therefore be different!**