Name: Joseph Specht

Quiz 8 (Total: 30 points)

Due back on Wed. 5 April in the Quiz 8 folder on Canvas

- Follow the problem-solving strategy that you learnt in class. You will be penalized if you do not.
- Save your entire assignment as <u>one</u> PDF document and upload it in the appropriate assignment folder on Canvas.
- Assignments will only be graded if the honor code statement, below, is completed and signed.

Honor Code Statement

ME 200, Quiz 8

Being a student of high standards, I pledge to embody the principles of *academic integrity*.

This ME 200 quiz is my own work. I did not seek (or get) outside help or collaboration with any of the questions and their solutions. I also did not offer my solutions to any other student.

I understand that this quiz is "open book" and "open notes" which means that I was permitted to use my prescribed textbook and lecture notes when addressing any of the questions. I have properly cited any other resources, with full cognizance of the regulations pertaining to plagiarism, copyright infringement, academic cheating, etc., as stipulated in the Student Code.

I acknowledge that academic violations will be dealt with according to the UIUC Student Code, Article 1, Part 4.

ME 200 Student's signature:

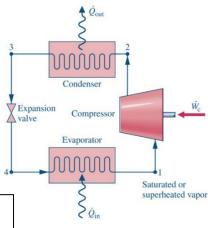
Student's Name: Joseph Specht

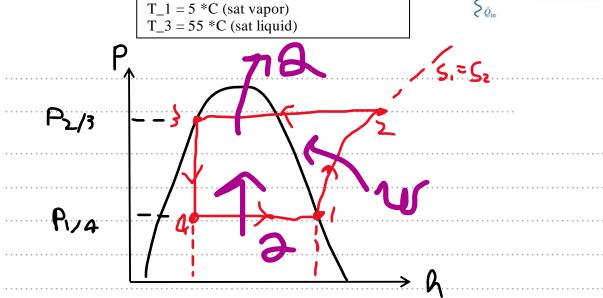
Net-ID: jspecht3

Date: 4/2/23

Consider an air-conditioning system that operates on an ideal vapor-compression cycle. The system uses R-134a as refrigerant. The refrigerant enters the compressor as a saturated vapor at 5°C and leaves the condenser as a saturated liquid at 55°C. The mass flowrate of the refrigerant is 0.7 kg/s.

1. Draw the cycle on a *p-h* diagram and indicate all state points. Clearly show the processes where heat and work transfer take place. Also indicate all known temperatures and pressures. (2)





2. Complete this table *after* you have performed all calculations in Questions 3 – 5. (No credit will be given for completing the table. Non-completion of the table will however result in 5 points being deducted from your quiz total.)

| Process | Specific heat, <i>q</i> (kJ/kg) | Specific work, w (kJ/kg) | Change in specific enthalpy, $h_{\rm exit} - h_{\rm inlet}$ (kJ/kg) |
|---------|------------------------------------|-----------------------------|-----------------------------------------------------------------------|
| 1-2 | 0 | -30.7455 | 29.8675 |
| 2 - 3 | -151.548 | 0 | -151.548 |
| 3 - 4 | 0 | 0 | 0 |
| 4 - 1 | 120.8025 | 0 | 120.8025 |
| Σ: | -30.7455 | -30.7455 | 0 |

3. Calculate the specific enthalpies and specific entropies at state points 1, 2, 3 and 4. Show ALL interpolations! (16)

We know that @ state point 1, we have a saturated refrigerant 134a that is a saturated vapor, so go to table A-10 and interpolate to find the enthalpy and entropy

$$\frac{h_1 - h_g@T = 4^{\circ}C}{5^{\circ}C - 4^{\circ}C} = \frac{h_g@T = 8^{\circ}C - h_g@T = 4^{\circ}C}{8^{\circ}C - 4^{\circ}C}$$

$$\frac{h_1 - 249.53}{5 - 4} = \frac{251.80 - 249.53}{8 - 4}$$

$$h_1 = 250.0975 \frac{kJ}{kg}$$

Now we must repeat the same process to find the entropy!

$$\frac{s_1 - s_{g@T = 4^{\circ}C}}{5^{\circ}C - 4^{\circ}C} = \frac{s_{g@T = 8^{\circ}C} - s_{g@T = 4^{\circ}C}}{8^{\circ}C - 4^{\circ}C}$$
$$\frac{s_1 - .9169}{5 - 4} = \frac{.9150 - .9169}{8 - 4}$$
$$s_1 = .916425 \frac{kJ}{k \, a * K}$$

Before we calculate the values at state point 2, we need to know the pressure at this state, but we are in luck as the pressure at state point 3 and 2 are the same and we know information about state point 3.

@ State point 3, we have a saturated liquid at T=55*C, so using table A-10, we can find all these values

$$\frac{p_3 - p_{sat@55^{\circ}C}}{55^{\circ}C - 52^{\circ}C} = \frac{p_{sat@56^{\circ}C} - p_{sat@55^{\circ}C}}{56^{\circ}C - 52^{\circ}C}$$

$$\frac{p_3 - 13.851}{55^{\circ}C - 52^{\circ}C} = \frac{15.278 - 13.851}{56^{\circ}C - 52^{\circ}C}$$

$$p_3 = p_2 = 14.92125 \ bar$$

After solving state point 3, we can find the enthalpy and entropy at state point 2. We know that state points 1 and 2 have the same entropy and we know state point 2 will be superheated as you do work on a saturated vapor that experiences adiabatic compression. We also know the pressure is equal to p_3.

$$\frac{h_3 - h_{f@p=14bar}}{14.92125bar - 14bar} = \frac{h_{f@p=16bar} - h_{f@p=14bar}}{16bar - 14bar}$$

$$\frac{h_3 - 125.26}{14.92125 - 14} = \frac{134.02 - 125.26}{16 - 14}$$

$$h_3 = 129.295 \frac{kJ}{kg}$$

If we do the same interpolation process for the entropy, we get...

$$\frac{s_3 - s_{f@p=14bar}}{14.92125bar - 14bar} = \frac{s_{f@p=16bar} - s_{f@p=14bar}}{16 - 14}$$

$$\frac{s_3 - .4453}{14.92125 - 14} = \frac{.4714 - .4453}{16 - 14}$$

$$s_3 = .4573 \frac{kJ}{kg * K}$$

Now that we have solved for state point 3, we can realize $s_1 = s_2$ and use table A-12 to solve for h_2 .

$$\frac{h_2 - h_{@p=14,T=60}}{14.92125bar - 14bar} = \frac{h_{@p=16,T=60} - h_{@p=14,T=60}}{16bar - 14bar}$$

$$\frac{h_2 - 283.10}{14.92125 - 14} = \frac{278.20 - 283.10}{16 - 14}$$

$$h_2 = 280.843 \frac{kJ}{kg * K}$$

Since we know $h_3 = h_4$, the only information we need to find is s_4 , so we go to table A-10 as state point 4 lies in the saturated mixture region. We know $p_1 = p_4$, so we must find p_1 first.

$$\frac{p_1 - p_{sat@T=4}}{5^{\circ}C - 4^{\circ}C} = \frac{p_{sat@T=8} - p_{sat@T=4}}{8^{\circ}C - 4^{\circ}C}$$
$$\frac{p_1 - 3.3765}{5 - 4} = \frac{3.8756 - 3.3765}{8 - 4}$$
$$p_1 = p_4 = 3.5013 \ bar$$

Now that have these two properties, we can find the value of s_4 from table A-10. But first, we must find the vapor quality.

 $h_4 = h_{f@n4} + x * h_{fg@n4}$

$$\frac{h_{f@p4} - 55.35}{3.5013 - 3.3765} = \frac{60.73 - 55.35}{3.8756 - 3.3765}$$
$$h_{f@p4} = 56.70$$

$$\frac{h_{fg@p4} - 194.19}{3.5013 - 3.3765} = \frac{191.07 - 194.19}{3.8756 - 3.3765}$$
$$h_{fg@p4} = 193.401$$

Since we know h_4, h_f@p4, and h_fg@p4, we can find x

$$x = \frac{h_4 - h_{f@p4}}{h_{fg@p4}} = .3754$$

If we have this value, we can then interpolate from A-10 to find s_4

$$s_4 = s_{4f} + x * s_{4fg}$$

First, we need to find the values of s_f and s_fg, so

$$\frac{s_{4f} - .2162}{3.5013 - 3.3765} = \frac{.2354 - .2162}{3.8756 - 3.3765}$$
$$s_{4f} = .221$$

Now, we have to find s_g to find s_fg

$$\frac{s_{4g} - .9169}{3.5013 - 3.3765} = \frac{.9150 - .9169}{3.8756 - 3.3765}$$
$$s_{4g} = .916425$$

This means s_fg is...

$$s_{fg} = s_g - s_f = .6954$$

With all the entropies we need, we can find the final value of s_4

$$s_4 = s_{4f} + x * s_{4fg} = .482$$

| State point | H [kJ/kg] | S [kJ/(kg*K)] |
|-------------|-----------|---------------|
| 1 | 250.0975 | .916425 |
| 2 | 280.843 | .916425 |
| 3 | 129.295 | .4573 |
| 4 | 129.295 | .482 |

Now that we have all these values, we just need to calculate the work and heat transfer with the first law

$$(q_{in} - q_{out}) - (w_{in} - w_{out}) = h_2 - h_1$$

For 1-2, we know this is adiabatic and gives no work, so $w_{in} = h_1 - h_2 = 30.7455$

For 2-3, we know this is an isobaric process, so work is zero and this process ejects heat, so q_out = 0 $q_{out} = h_3 - h_2 = -151.548$

For process 3-4, we know it is a valve that does no work or add any heat, so everything is 0.

For process 4-1, we know this is an isobaric process, so work is 0 and this raises the enthalpy, so q_out=0 $q_{in} = h_1 - h_4 = 120.8025$

- 4. Calculate the specific work required at the compressor. (3) For 1-2, we know this is adiabatic and gives no work, so $w_{in} = h_1 h_2 = 30.7455$
- 5. Calculate heat transfer rate (in kW) in the condenser. (3)

For 2-3, we know this is an isobaric process, so work is zero and this process ejects heat, so q_out = 0 $q_{out} = h_3 - h_2 = -151.548$

- 6. Calculate heat transfer rate (in kW) in the evaporator. (3) For process 4-1, we know this is an isobaric process, so work is 0 and this raises the enthalpy, so q_out=0 $q_{in} = h_1 h_4 = 120$.
 - 7. Calculate the coefficient of performance of the cycle. (3)

$$COP = \frac{q_{out}}{w_{in}} = 4.929$$
, which seems high