Chapter 1

Developing Vapor Intrusion Models

1.1 Introduction

No models are true representations of reality, but some of them may be useful. Ever since Newton first wrote his laws of motion, mankind has tried to describe reality with an ever increasing number of mathematical statements. With the advent of computation and advancements in numerical methods our capabilities to mathematically describe physical systems has dramatically increased. Even so, real-world systems are too complex to be fully modeled, but mathematical representations may be used to approximate and reveal useful insights of how they function.

This is especially true for vapor intrusion (VI) models. Often it is impossible or difficult to conduct controlled studies of VI sites making models an important tool for understanding these sites and the VI phenomena. The previous chapter is proof of this as it is readily apparent that a multitude of VI models of varying complexity have been developed over the years, and has become an important part of the scientific VI community. From the simple Johnson & Ettinger one-dimensional model to full three-dimensional finite element models (FEM) we see that the increased complexity of the model allowed for a greater number of VI topics and phenomena to be explored.

The processes of VI may be described by partial differential equations (PDEs). Unfortunately, there rarely are any analytical solutions to these (except in the most simple cases) and numerical methods are required to find approximate solutions. One of the most powerful numerical methods for solving PDEs is the finite element method, which not only allows us to find solutions to PDEs but does so for complex three-dimensional geometries.

The purposes of this thesis is not to explain the FEM in any great detail, but there are many great resources available for those who are interested to learn more. There are however, two things that are important to know what makes the FEM unique.

The first is the FEM divides up a complicated geometry into smaller *finite elements*, hence its name. Which elements exactly depend on the dimensionality of the model and the specific problem that one wish to solve. Three-dimensional geometries are usually represented by tetrahedral and two-dimensional ones by triangles.

The second is that the solution to a PDE may be represented by a linear com-

Figure 1.1: Example of a simple conceptual site model of a vapor intrusion site.

bination of a series of *basis functions* with an associated function *coefficient*.

$$u \approx \sum_i u_i \psi_i \tag{1.1}$$

where u is the solution to the PDE, u_i is the coefficient associated with the basis function ψ_i. This approximation allows the PDE to be discretized into a matrix and the u_i coefficients are solved for. Any function may serve as a *basis function*, but typically a simple one is chosen (for simpler computation) like a linear hat function or low-degree polynomial. In certain applications, some basis functions perform better than other, but in most cases linear hat functions or second-degree polynomials are preferable.

The development of the three-dimensional finite element vapor intrusion models begin with a conceptual site model (CSM) of a VI site. In general when one develops models, it is best in the beginning to keep the model as simple as possible, and not to add overly complex features or excessive physics. As such, we begin with a very simple CSM which may be seen in Figure 1.1.

This CSM features a residential building with a 10 by 10 m footprint, with a concrete foundation one meter below ground-surface (bgs). Along the perimeter of foundation there is a one cm wide breach, through the subsurface contaminants enter the house. Three meters below the foundation, there is a contaminated groundwater source, from which contaminants vapor continuously evaporate. The house is assumed to be depressurized relative to the atmosphere which creates a pressure gradient, allowing air to be pulled through the ground-surface, soil, and into the house - carrying some contaminants with it. The indoor air is also exchanged at a constant rate with the outside environment, which is the only way the contaminant leave the house. For simplicity we also assume that the soil is completely homogenous.

To implement this CSM as a finite-element model several steps must be followed.

1. Construct a model geometry (domain).

2. Assign relevant partial differential equations (PDEs) and boundary conditions (BCs) that describe the physics.

3. Mesh the geometry.

4. Configure and choose solvers.

5. Post-processing.

Each step will be carefully explained, beginning with the construction of the domain.

1.2 Geometry

Designing the model geometry is the first step to creating a 3D FEM model. It is one of the most important steps, as the geometry will dictate the model accuracy and astute geometry design will help save computational resources. When designing a geometry the FEM user should have the following goals in mind:

1. Represent the model geometry as accurately as possible.

2. Avoid unnecessarily fine details.

3. Try to leverage symmetry to reduce geometry size.

The first point is somewhat self-explanatory, as we obviously want to create a model geometry that is as similar to what we want to model as possible. The second points can at times run counter to the first and may be more self-evident once meshing is more thoroughly discussed. Tiny details often require a significant number of mesh elements to be fully resolved, disproportionally adding to the total number of mesh element, and may significantly increase computational costs. This is when the skill and judgement of the modeler comes in - choosing which details to omit and which to keep. As a rule-of-thumb one should for the most part try to only model parts of the geometry that is of significant value to the question that one wants answered. In VI modeling, one such obvious area is the crack or breach in the foundation through which contaminant vapors enter the structure, resolving this tiny part of the geometry is of great importance.

The third point is something that the modeler should always be on the lookout for when designing a model geometry - if there are any planes of symmetry in the geometry. Finding a plane of symmetry allows us to reduce the size of the model and save significantly computational costs. A simple example of this is one wants to model a pipe with static mixers inside, then only a sector of the cylinder's face may be necessary to be modeled. Using the simple CSM described by Figure 1.1 only a quarter of the house and surrounding property is necessary to be explicitly modeled, cutting the number of required mesh elements down to just a quarter of what would otherwise be necessary - a huge computational saving!

1.2.1 Geometric Components

Model geometries are typically designed in some sort of computer assisted design (CAD) software. The exact tools and techniques available to the modeler will vary from software to software, with some featuring import options for real-world scanned 3D geometries to combining simple geometric objects through various Boolean operations. The software we use, COMSOL, uses primarily the latter method of combining simple objects to form more complicated ones but more capabilities may be purchased.

To create a model geometry of the CSM in Figure 1.1, only a few simple geometric objects and Boolean operations are required - two cuboids, two rectangles, one Boolean difference operation, and one Boolean join operation. The following steps are needed:

1. Create a block or cuboid that is 15 meter wide and long, and with a height of 4 meter.

2. Create another block that is 5 meter wide and long, with a height of 1 meter.

3. Place the second block 3 meter above zero, so that the top surfaces of the two blocks intersect.

4. Perform a difference operation, removing the smaller block from the first one.

At this point you will see that a quarter soil domain has been created, with an empty space that will represent a house with a foundation slab located 1 meter below ground-surface.

The attentive reader will now of course notice that an entire house is missing from the model geometry. This is intentional, as explicitly modeling the interior of a building is too impractical for two primarily two reasons. First is that house interiors are simply too diverse for any explicit model to truly be representative in any general sense, not to mention how laborious it would be to create such a model interior. Secondly, the computational costs required to solve the air flow inside (necessary for accurate representation of contaminant transport/distribution inside) would be significant. The Navier-Stokes equation would need to be solved, and even using one of the simpler versions of it (large eddy simulation or Reynold's averaged) would impose a significant cost at questionable gain. Therefore the indoor air is modeled implicitly, which will be covered in detailed in section 1.3.

The foundation crack will be modeled as a 1 centimeter wide strip that spans the perimeter of the surface that represents the house foundation. To create the crack do the following:

1. Create a work plane 3 meter above zero.

2. Create two rectangles that are as long as the foundation, with a width of 1 centimeter, rotating one 90 degrees, and making sure that they are place along the foundation perimeter.

3. Join the two rectangles using a Boolean union (do not keep the interior boundaries).

Now that the foundation crack is generated, we have designed a model geometry of the simple CSM and the complete geometry may be seen in Figure 1.2. The next step is to choose and setup the appropriate physics required to model VI, beginning with modeling the indoor environment.

Figure 1.2: The complete geometry of the CSM described in Figure 1.1.

In the appendix, there will be further explanations for additions to the model geometry that will be necessary for modeling various VI scenarios.

1.3   The Indoor Environment

The indoor air space is perhaps the most important part of modeling VI, as the goal of these models ultimately is to predict indoor exposure given external factors. One could therefore assume that most of the effort in modeling VI should be spent to accurately represent the interior. This would be very impractical however, as building interiors are so diverse. Even if one would spend the time to model an interior, this would dramatically increase the number of mesh elements required to solve the model. Additionally, the air flow inside the building must be calculated, and even using a simplified version of Navier-Stokes, like large eddy simulation or Reynolds averaged, the computational cost would be significant.

To overcome this, the indoor environment is instead modeled as a continuously stirred tank reactor (CSTR). The fundamental assumption of a CSTR is that any contaminant or chemical species entering, or inside the indoor air space (control volume), is perfectly mixed, i.e. there are no spatial gradients, and is given by the ordinary differential equation (ODE) (1.2).

$$V\frac{\partial c_{\text{in}}}{\partial t} = n - VA_e c + R \tag{1.2}$$

Here c_{in} is the indoor air contaminant concentration in mol/m^3; n is the contaminant entry rate into the building in mol/s; A_e is the air exchange rate, which determines the which portion of the indoor air is exchanged for a given time period, e.g. if A_e is 0.5 per hour, half of the indoor air is exchanged over one hour; R can be used to simulate sorption of contaminants vapor in the indoor environment, and if this is not of interest it can simply be set to zero; Finally, V is the volume of the building interior in m^3. Typically this is set to only reflect the volume of the floor or rooms on top of the building foundation. Since the indoor environment is does not explicitly "belong" anywhere in the COMSOL model, it is modeled using the "Global ODE" interface.

1.3.1 Contaminant Entry

The most important component of (1.2) is of course determining the contaminant entry rate n, and is the most challenging portion of the modeling effort. The contaminant entry rate has two transport components, advective and diffusive which depends on three factors:

1. The velocity of the contaminant vapors entering or exiting the structure though the foundation crack.

2. The contaminant vapor concentration in the near vicinity of the foundation crack.

3. The indoor air contaminant concentration itself.

Advective Transport The advective transport due to the bulk motion of the contaminant vapors and the flux is given by (1.3).

$$j_{\text{adv}} = \vec{u}c \tag{1.3}$$

The bulk motion of the contaminant vapor is given by the vector quantity $\vec{u}$ in m/s and c is the contaminant vapor concentration.

Diffusive Transport The diffusive transport is due to a concentration gradient, modified by a diffusion coefficient, and is given by Fick's law (1.4)

$$j_{\text{diff}} = \nabla \cdot (D\nabla c) \tag{1.4}$$

Where ∇ is the del operator and D is the diffusion coefficient in m^2/s. In this formulation, D does not have to be a constant and can depend on the coordinate or concentration.

The implication of this is that (1.2) has to be coupled with the equations that describe the contaminant concentration in the soil as their solutions are dependent on each other. How this is achieved will be covered in section 1.4.3 when discussing boundary conditions.

1.3.2 Air Exchange Rate

The air exchange rate, A_e is the parameter that determines the rate at which the contaminant vapors leave the indoor environment. Air infiltrate and exfiltrate through a building primarily via two mechanisms

1. Through breaches and orifices in the building envelope, e.g. windows, slits, or other small opening.

2. Passive or active ventilation.

With the exception of active ventilation, where air is mechanically forced to enter or exit the building, the driving force for the in-/exfiltration is driven by a pressure gradient between the indoor and outdoor environment, $p_{\text{in/out}}$. These pressure gradient are primarily due to differences in indoor and outdoor temperatures and to wind striking the building and quantifying these pressures differences is covered in section 1.3.3.

Quantifying Air Exchange Rate While air exchange rate is largely driven by the indoor and outdoor pressure difference, it is not possible to quantify their relationship for the range of pressures that a building is naturally pressurized; there is too much uncertainty. In Figure 1.3 a two-dimensional kernel density estimation (a way to estimate probability distributions[?]) of indoor/outdoor pressure and air exchange rate from the well-studied ASU house shows the relationship between these[?]. The a darker color signifies are closer association or overlap but by the lack of any visual trends (it is all a blob) combined with a Pearson's r value of x, it is apparent that there is no real correlation between them exist.

Building Leakage Curves   It is possible to quantify air exchange rate if a building is sufficiently pressurized through building leakage curves.

1.3.3 Quantifying Pressure Difference Induction

Wind Effects As the wind strikes a surface its velocity falls to zero, and the change in momentum is directly proportional to the change in pressure:

$$\Delta P = \frac{1}{2}\rho_{\text{air}}u_{\text{wind}}^2 \tag{1.5}$$

where ΔP is the change in pressure; ρ_{air} is the air density; and u_{wind} is the wind speed.

In reality however, the pressure drop is not quite so straightforward due to several factors, e.g. building envelope contains various structures and may be shielded by

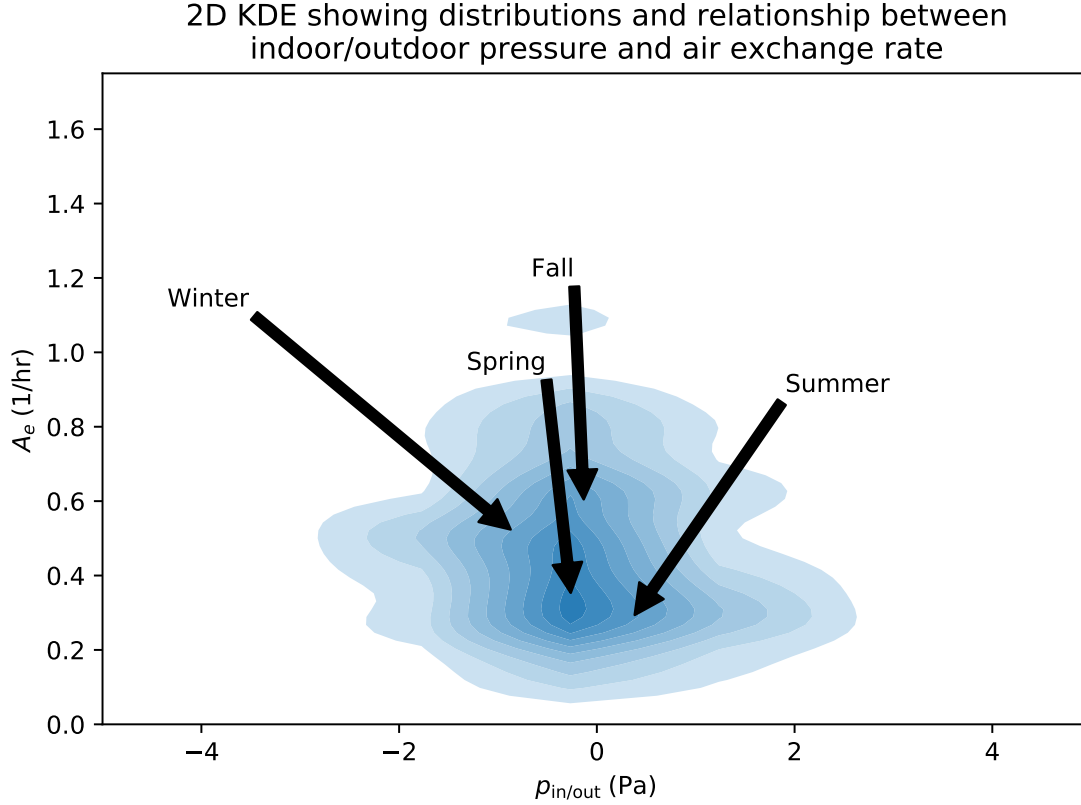## 2D KDE showing distributions and relationship between indoor/outdoor pressure and air exchange rate

Figure 1.3: Distributions of indoor pressurization and air exchange rate and their relationships; seasonal medians indicated by arrows.

other objects. To account for this, a drag or pressure coefficient C_d is introduce to moderate the pressure drop:

$$\Delta P = C_d \frac{1}{2} \rho_{\text{air}} u_{\text{wind}}^2 \tag{1.6}$$

This coefficient is usually determined empirically from e.g. wind tunnel studies.

Temperature Effects   The pressure of any fluid under the influence of gravity varies with elevation and the density of the fluid determines the magnitude of this pressure. Air is a compressible fluid and its density depends on its temperature. Therefore, if you separate two air masses between a wall, with each at a difference temperature, a pressure difference across the wall be induced, i.e. the *stack effect*.

Assuming the ideal gas law applies and that the temperature on either side is constant then

$$P = P_0 \exp\left(\frac{-M_{\text{air}} g z}{RT}\right) \tag{1.7}$$

determines the pressure variation across the wall. Where P is the pressure; P_0 the reference pressure ($z = 0$); M_{air} is the molar weight of air; z is the elevation; g is the acceleration due to gravity; R is the gas constant; and T is the temperature.

This can be further simplified by defining z_0 which is the height at which the pressure on both sides of the wall are equal, i.e. P_0. The pressure variation on either side can now be expressed in terms of distance from z_0, $z - z_0$. Assuming

that the interior and exterior has a constant temperature of T_i and T_o respectively, the difference in pressure between the two sides along the wall is

$$\Delta P = P_0 \left[\exp\left(\frac{-M_{\mathrm{air}}g(z - z_0)}{RT_i} \right) - \exp\left(\frac{-M_{\mathrm{air}}g(z - z_0)}{RT_o} \right) \right] \tag{1.8}$$

Using that $e^{-x} = 1 - x$ for $x \ll 1$ the pressure difference can be expressed as

$$\Delta P = \alpha\left(\frac{1}{T_o} - \frac{1}{T_i}(z - z_0) \right) \tag{1.9}$$

where $\alpha = \frac{P_0 M_{\mathrm{air}}g}{R} \approx 3454 \; \frac{\mathrm{Pa \cdot K}}{\mathrm{m}}\mathrm{Pa}$. Here a negative pressure differnece, $\Delta P < 0$, indicates an inward flow.

Other Effects   Other factors also contribute to the pressure difference between the interior and exterior. Some examples of these are various combustion processes (fireplaces), ventilation fans, or even human factors such as opening and closing doors or windows.

1.3.4   Indoor Sources

There are two types of indoor sources in VI. The first are various products and items that contain the contaminant of concern and continuously emit these. Some examples of these may be various cleaning products, degreasers, gasoline tanks, or even recently dry-cleaned clothing. These are usually not considered to be part of the VI phenomena themselves (even though they can pose health risks) but rather items that need to be removed in a VI investigation to prevent false positives.

The second type are the walls, furniture, objects, or features inside the interior that sorb the contaminant vapors. While these are not *sources* in the strict sense, they may never the less contribute to the overall contaminant exposure. How significant these secondary types of indoor sources are largely depend on the material. Some, such as cinderblock concrete have a very large capacity to hold contaminant vapors while others such as fabrics do not.

Sorption Processes   Modeling the secondary type indoor sources (sorption onto materials) can be done with a simple equilibrium reaction:

$$\mathrm{c}^* \underset{k_2}{\overset{k_1}{\rightleftharpoons}} \mathrm{c}_{\mathrm{in}} \tag{1.10}$$

1.4   Soil Physics Governing Vapor Intrusion

The soil surrounding the structure in our VI model is, unlike the indoor environment, modeled explicitly. In this section we will walk through each physics, the associated governing equation, and boundary conditions required to model the contaminant transport in soil. The following physics and governing equations will be covered:

1. Water flow in unsaturated porous media

2. Vapor transport in unsaturated porous media

3. Mass transport in partially saturated medium

In addition to these, the modeling of the temperature distribution in the soil is covered in Appendix .

The vapor contaminant transport through the soil in VI occurs through the vadose zone - soil that is partially filled with water, giving a three-phase transport system. This partial water content has profound effects on both advective and diffusive transport and modeling the water content is achieved via *Richard's equation*. Since the vapor and mass transport in the soil are so dependent on the soil water content, it is covered first in the following section 1.4.1.

The mass transport in the soil has both an advective and diffusive component. The advective transport in the soil is dictated by the vapor flow in the soil which is described by *Darcy's Law* making it the next logical step to cover in section 1.4.2. The diffusive transport depends on the contaminant vapor concentration itself and accurately modeling this requires coupling with all of the physics discussed so far (including the indoor environment). Therefore the mass transport physics, governed by the *advection-diffusion equation* will be covered last in section 1.4.3.

1.4.1 Water Flow in Unsaturated Porous Media

The vadose zone or unsaturated zone is a region of soil between the top of the ground surface and the water table. In the vadose zone there are two fluid phases, one gas and the other liquid (usually air and water) inside the porous soil matrix giving a three phase system; only one fluid phase (gas or liquid) exist in the saturated zone. As a result, the transport properties in the vadose zone differ from that in a zone saturated where there are only two phases present - water and soil.

The driving force for the pore water in the vadose zone is a negative pressure caused by the surface tension of the water. This phenomena is called *capillary potential* or *matrix potential*.

$$\rho\Big(\frac{C_m}{\rho g} + \mathrm{Se}S\Big)\frac{\partial p}{\partial t} + \nabla \cdot \rho\Big(-\frac{\kappa_s}{\mu}k_r(\nabla p + \rho g\nabla D)\Big) = Q_m \qquad (1.11)$$

Where p is the capillary potential; C_m is the specific moisture capacity; Se is the effective saturation; S is the storage coefficient; κ_s is the saturated permeability of the porous media; μ is the fluid viscosity; k_r is the effective permeability; ρ is the fluid density; g is the acceleration of gravity; D is the elevation or head; and Q_m is a source term, a positive or negative value represent a source or sink respectively.

Soil-Water Potential

Soil-Water Retention Curve

The distribution of soil moisture in the soil matrix has profound implications for the advective and diffusive transport of contaminants. Soil has a limited amount of pore volume available for contaminant transport, and the presence of water restricts this further; decreasing permeability of the soil and subsequently reduces air flow. Diffusivity of the contaminant will also be retarded by the water. The contaminant will dissolve into and evaporate from water and the transport will partially occur through water. Liquid diffusion coefficients are usually around four orders of magnitude smaller than in air.

The soil moisture content of soils can be estimated in many ways, but two common approaches is to use the analytical formulas of *van Genuchten* or *Brooks and Corey*. Both of these formulas give the soil moisture content as a function of the fluid pressure head, H_p. By definition, when the pressure head is equal to or greater than zero, $H_p \geq 0$, the soil is assumed to be 100% saturated with the fluid. In this work, *van Genuchten's* formula is used.

The soil moisture content, θ is given by.

$$\theta = \begin{cases} \theta_r + \text{Se}(\theta_s - \theta_r) & H_p < 0 \\ \theta_s & H_p \geq 0 \end{cases} \tag{1.12}$$

The saturation is given by.

$$\text{Se} = \begin{cases} \frac{1}{(1+|\alpha H_p|^m)^m} & H_P < 0 \\ 1 & H_p \geq 0 \end{cases} \tag{1.13}$$

$$C_m = \begin{cases} \frac{\alpha m}{1-m}(\theta_s - \theta_r)\text{Se}^{\frac{1}{m}}\left(1 - \text{Se}^{\frac{1}{m}}\right)^m & H_p < 0 \\ 0 & H_p \geq 0 \end{cases} \tag{1.14}$$

$$k_r = \begin{cases} \text{Se}^l\left[1 - \left(1 - \text{Se}^{\frac{1}{m}}\right)\right]^2 & H_p < 0 \\ 0 & H_p \geq 0 \end{cases} \tag{1.15}$$

1.4.2 Vapor Transport in Unsaturated Porous Media

Vapor transport in porous media is described by *Darcy's Law*. The vapor velocity depends on the pressure gradient in the soil, is proportional to the permeability of the soil matrix, and is inversely proportional to the viscosity of the fluid.

$$\vec{u} = -\frac{\kappa}{\mu}\nabla p \tag{1.16}$$

For Darcy's Law to be valid, two assumptions must be fulfilled:

1. The fluid must be in the laminar regime, typically Re < 1.

2. The soil matrix must be saturated with the fluid.

In VI-modeling, the first assumption is fulfilled, but the second is not. Most of the contaminant vapor transport takes place in the partially saturated vadose zone and thus, (1.16) needs modification.

In partially saturated soils, a varying portion of the soil pores are available for vapor transport, with the rest being occupied by water, affecting the effective permeability of the soil. To model this, a relative permeability property, k_r, is introduced:

$$\kappa_{\text{eff}} = k_r \kappa_s \tag{1.17}$$

k_r is a dimensionless parameter that varies between 0 and 1, and κ_s is the saturated, or simply the soil matrix permeability.

This gives the modified Darcy's Law used in VI-modeling:

$$\vec{u} = -\frac{k_r \kappa_s}{\mu}\nabla p \tag{1.18}$$

1.4.3 Mass Transport in Unsaturated Porous Media

$$\frac{\partial}{\partial t}(\theta c_i) + \frac{\partial}{\partial t}(\rho_b c_{P,i}) + \frac{\partial}{\partial t}(a_v c_{G,i}) + \vec{u} \cdot \nabla c_i = \nabla \cdot [(D_{D,i} + D_{eff,i}) \nabla c_i] + R_i + S_i \quad (1.19)$$

1.5 Meshing

1.6 Solver Configuration

1.7 References