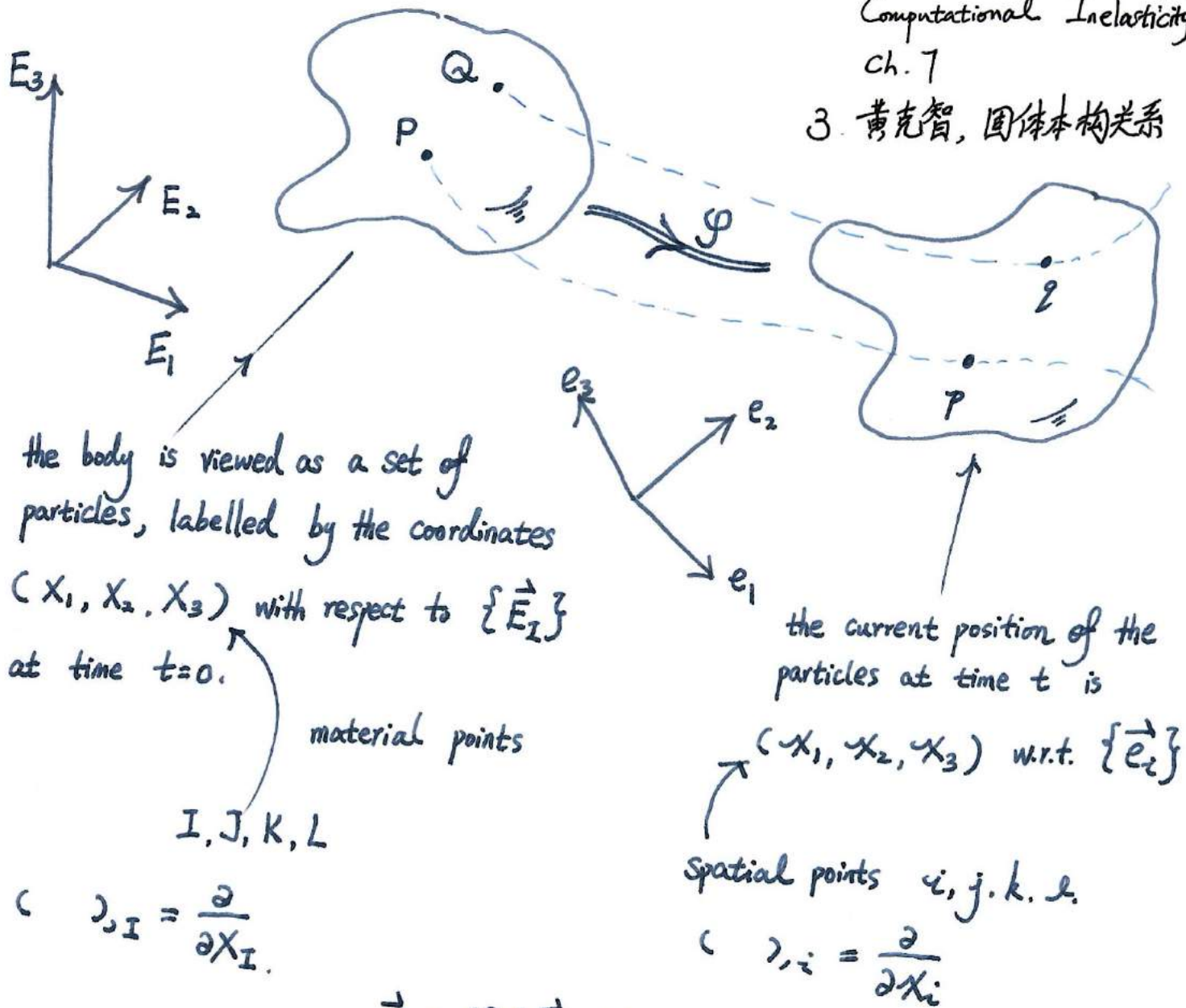


Finite-strain elasticity

- Kinematics: the study of motion and deformation without reference to the cause.

References:

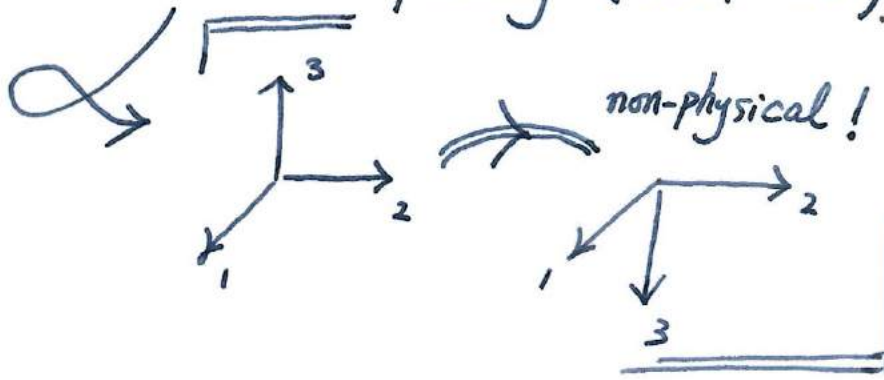
1. G. Holzapfel: Nonlinear Solid Mechanics
2. J. Simo & T.J.R. Hughes: Computational Inelasticity ch. 7
3. 黄克智, 固体本构关系



$$\vec{x} = \varphi(\vec{X}, t)$$

placement: gives the configuration Ω_{t_0} at time t
 Requirement: • smooth (differentiable);

- one-to-one (except possibly at the boundary: contact)
- orientation-preserving ($\det F > 0$).



Displacement: $U_i(X, t) = \varphi_i(X, t) - \varphi_i(X, 0)$

\nearrow
a vector on $\Omega_{\Delta x}$

$$= x_i - \delta_{iI} X_I$$

Note: the ambient space is Cartesian

rigid motion: $x = Q(t) X + c(t)$

\nearrow rigid rotation: Q is proper orthogonal.
rigid translation: $\det(Q) = +1$ $Q^T Q = I$

Deformation gradient:



$$dx_1 = x_{Q_1} - x_p = \varphi_t(X_{Q_1}) - \varphi_t(X_p)$$

$$= \varphi_t(X_p + dx_1) - \varphi_t(X_p) = \frac{\partial \varphi_t}{\partial X}(X_p) dx_1.$$

We call $\frac{\partial \varphi_t}{\partial X} = \frac{\partial x}{\partial X} = F$ the deformation gradient.

$$F = F_{iI} \vec{e}_i \otimes \vec{E}_I$$

↑
current/spatial
Cartesian basis
↑
initial
material
Cartesian basis

$$F^T = F_{iI} \vec{E}_I \otimes \vec{e}_i$$

$$= (F^T)_{Ii} \vec{E}_I \otimes \vec{e}_i$$

$$F^{-1} = \frac{\partial X}{\partial x} = \frac{\partial X_I}{\partial x_i} \vec{E}_I \otimes \vec{e}_i$$

$$\bar{F}^{-T} = \frac{\partial X_I}{\partial x_i} \vec{e}_i \otimes \vec{E}_I$$

Two-point tensors: transform vectors of one configuration to vectors on another configuration

→ more general:

- push-forward $dx = F dX$

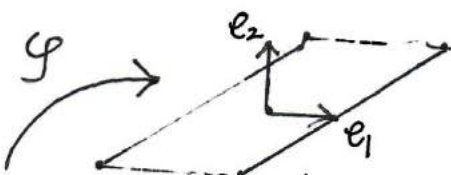
$$= \varphi_{t*}[dX]$$

- pull back $dX = F^{-1} dx$

$$= \varphi_t^{-1*}[dx].$$

$$x_1 = \frac{1}{4}(18 + 4x_1 + 6x_2)$$

$$x_2 = \frac{1}{4}(14 + 6x_2)$$



$$F = \begin{bmatrix} 1 & 1.5 \\ 0 & 1.5 \end{bmatrix}$$

$$\bar{F}^{-1} = \begin{bmatrix} 1 & -1 \\ 0 & \frac{2}{3} \end{bmatrix}$$

$$\varphi_*[E_1] = e_1$$

$$\varphi_*[E_2] = \begin{bmatrix} 1.5 \\ 1.5 \end{bmatrix}$$

$$\varphi_*^{-1} [e_1] = E_1, \quad \varphi_*^{-1} [e_2] = \begin{bmatrix} -1 \\ 2 \\ 3 \end{bmatrix}$$

- Line, Area, and Volume change

$$\underline{dx = F dx} \quad \text{line element}$$

Lemma: $v, w \in \mathbb{R}^3$, $A \in \mathbb{R}^{3 \times 3}$, then
 $(Av) \times (Aw) = (\text{cof } A) (v \times w)$
 if A^{-1} exists, $\text{cof } A = \det A \bar{A}^T$.

Proof:

$$\det A = \sum_{j=1}^n (-1)^{i+j} A_{ij} M_{ij} \quad \text{minor 余子式}$$

Laplace expansion

$$(-1)^{i+j} M_{ij} = (\text{cof } A)_{ij}$$

cofactor 代数余子式.

$$\Rightarrow (\text{cof } A)_{ij} \bar{A}_{jk}^T = (\det A) \delta_{ik}$$

$$\text{or if } A \text{ is invertible, } \text{cof } A = (\det A) \bar{A}^T.$$

Levi-Civita symbol:

$$\det A = \sum_{i=1}^3 \sum_{j=1}^3 \sum_{k=1}^3 \epsilon_{ijk} A_{1i} A_{2j} A_{3k} = \frac{1}{6} \epsilon_{ijk} \epsilon_{pqr} A_{ip} A_{jq} A_{kr}$$

$$v \times w = \sum_{i=1}^3 \sum_{j=1}^3 \sum_{k=1}^3 \epsilon_{ijk} e_i v_j w_k.$$

$$(\text{cof } A)_{ij} \underline{A_{ij}} = 3 \det A = \frac{1}{2} \epsilon_{ipk} \epsilon_{jqr} \underline{A_{ij}} A_{pq} A_{kr}$$

$$\Rightarrow (\text{cof } A)_{ij} = \frac{1}{2} \epsilon_{ipk} \epsilon_{jqr} A_{pq} A_{kr}$$

$$\Rightarrow (\text{cof } A)_{ij} (v \times w)_j = (\text{cof } A)_{ij} \epsilon_{jmn} v_m w_n$$

$$= \frac{1}{2} \epsilon_{ipk} \epsilon_{jqr} \epsilon_{jmn} v_m w_n A_{pq} A_{kr}$$

$$= \frac{1}{2} \epsilon_{ipk} (\delta_{qm} \delta_{rn} - \delta_{qn} \delta_{rm}) v_m w_n A_{pq} A_{kr}$$

$$= \frac{1}{2} \epsilon_{ipk} (\cancel{v_q w_r - v_r w_q}) (v_q w_r A_{pq} A_{kr} - v_r w_q A_{pq} A_{kr})$$

$$= \frac{1}{2} \epsilon_{ipk} v_q w_r A_{pq} A_{kr} - \frac{1}{2} \epsilon_{ipk} v_r w_q A_{pq} A_{kr}$$

$$= \epsilon_{ipk} (A_{pq} v_q) (A_{kr} w_r)$$



Now we may consider an area element:

$$dx_1 \times dx_2 = N dA$$

orientation:
normal vector.

after deformation: $dx_1 \times dx_2 = n da$

$$\parallel$$

$$(F dx_1) \times (F dx_2)$$

$$\parallel$$

$$(\text{cof } F) (dx_1 \times dx_2)$$

$$\parallel$$

$$(\text{cof } F) N dA$$

Jacobian

$$J := \det F.$$

$$\boxed{n da = J \bar{F}^T N dA}$$

Nanson's formula.

F is invertible.

$$\boxed{\int_{\tilde{S}} \dots n da = \int_S \dots J \bar{F}^T N dA.}$$

this is how one change the integration variable for integration on orientable surfaces. (e.g. traction integration).

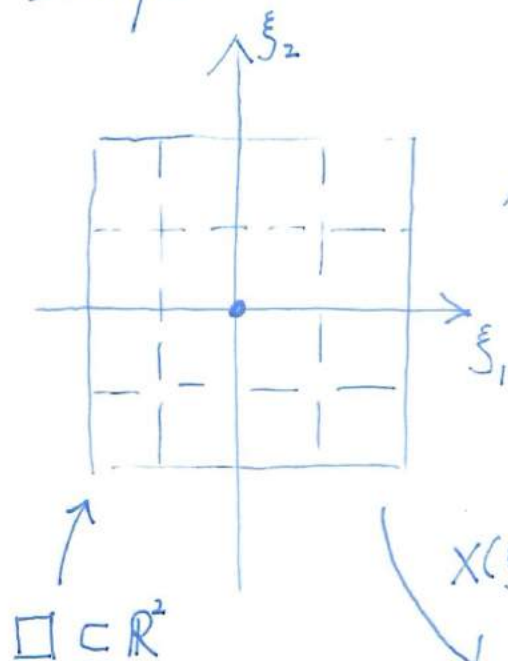
Consider a volume element $dx_1 \cdot (dx_2 \times dx_3)$

after deformation

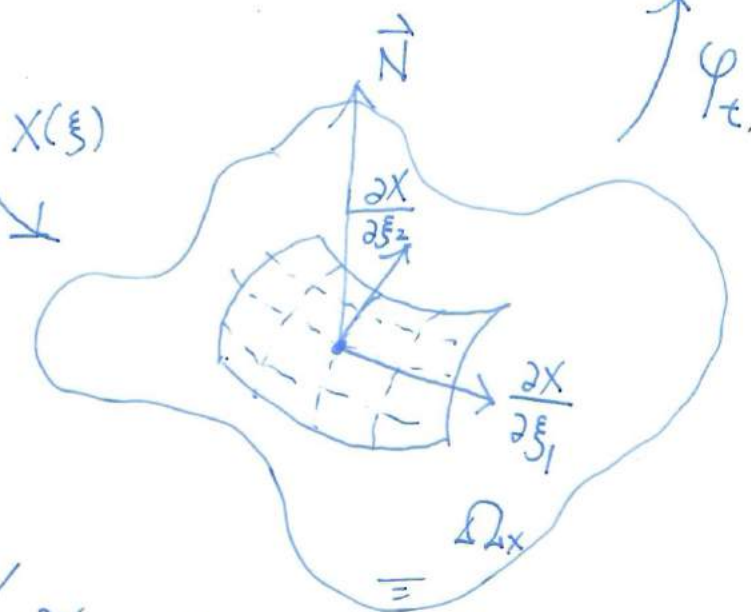
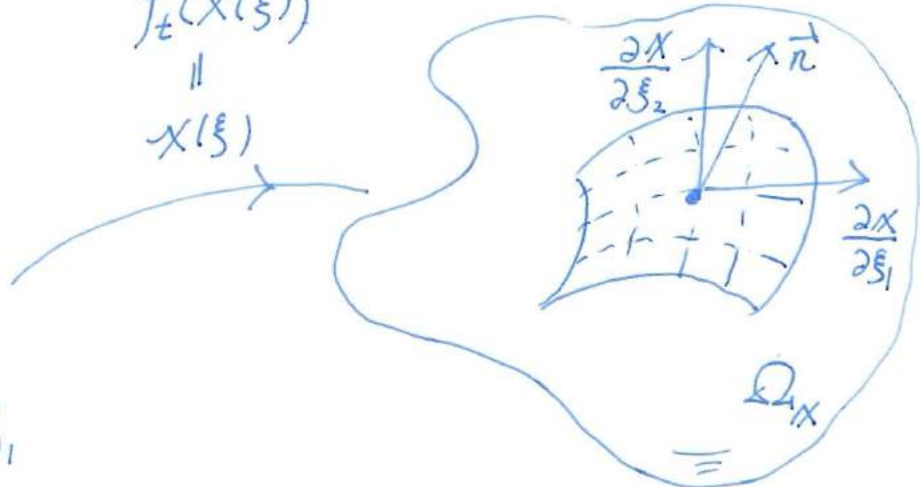
$$\begin{aligned} &\downarrow dx_1 \cdot (dx_2 \times dx_3) \\ &= F dx_1 \cdot (F dx_2 \times F dx_3) \\ &= dx_1 \cdot (F^T \text{cof} F) (dx_2 \times dx_3) \\ &= J dx_1 \cdot (dx_2 \times dx_3) \end{aligned}$$

$$\boxed{\int_{\psi = \varphi_t(V)} \dots d\psi = \int_V \dots J dV}$$

Example:



$$\varphi_t(X(\xi)) \parallel X(\xi)$$



$$\vec{N} = \left(\frac{\partial X}{\partial \xi_1} \times \frac{\partial X}{\partial \xi_2} \right) / \left\| \frac{\partial X}{\partial \xi_1} \times \frac{\partial X}{\partial \xi_2} \right\|$$

$$\vec{n} = \left(\frac{\partial X}{\partial \xi_1} \times \frac{\partial X}{\partial \xi_2} \right) / \left\| \frac{\partial X}{\partial \xi_1} \times \frac{\partial X}{\partial \xi_2} \right\|$$

$$\int_S \dots dA = \int_{\square} \dots \left\| \frac{\partial X}{\partial \xi_1} \times \frac{\partial X}{\partial \xi_2} \right\| d\xi_1 d\xi_2$$

$$\int_S \dots da = \int_{\square} \dots \left\| \frac{\partial X}{\partial \xi_1} \times \frac{\partial X}{\partial \xi_2} \right\| d\xi_1 d\xi_2$$

More, $\boxed{\frac{\partial J}{\partial F_{iI}} = \text{cof } F_{iI} = J \bar{F}_{Ii}^{-1}}$

Very useful!

Strain

measures length & angle.

$$dx_1 \cdot dx_2 = F dx_1 \cdot F dx_2 = dx_1 \cdot \underbrace{(F^T F)}_C dx_2$$

$$C_{IJ} = (F^T)_{Ii} F_{iJ} = F_{iI} F_{iJ}$$

material tensor known as the right Cauchy-Green deformation tensor.

$$\begin{aligned} \text{Alternatively, } dx_1 \cdot dx_2 &= \bar{F}^{-1} dx_1 \cdot \bar{F}^{-1} dx_2 \\ &= dx_1 \cdot \underbrace{(\bar{F}^{-T} \bar{F}^{-1})}_{b^{-1}} dx_2 \end{aligned}$$

$$b_{ij} = \cancel{F_{iI} F_{jI}} \quad (b = FF^T)$$

spatial tensor known as the left Cauchy-Green deformation tensor,
or Finger tensor.

$$\frac{1}{2} (dx_1 \cdot dx_2 - dx_1 \cdot dx_2) = dx_1 \cdot E dx_2$$

$$E = \frac{1}{2} (C - I) = \frac{1}{2} (F^T F - I)$$

material tensor, Green-Lagrange strain tensor

$$\frac{1}{2}(dx_1 \cdot dx_2 - dx_1 \cdot dx_2) = dx_1 \cdot e dx_2$$

$$e = \frac{1}{2}(I - b^{-1}) \quad \text{spatial tensor, Euler-Almansi strain tensor}$$

Remark 1: $2E = F^T F - I = \left[\left(\frac{\partial U}{\partial X} \right) + I \right]^T \left[\left(\frac{\partial U}{\partial X} \right) + I \right] - I$

$$= \underbrace{\left(\frac{\partial U}{\partial X} \right)^T \left(\frac{\partial U}{\partial X} \right)}_{\text{quadratic/nonlinear}} + \underbrace{\left(\frac{\partial U}{\partial X} + \left(\frac{\partial U}{\partial X} \right)^T \right)}_{\text{Linear.}}$$

$$2E = \frac{\partial U}{\partial x} + \left(\frac{\partial U}{\partial x} \right)^T \quad \text{infinitesimal strain / small strain.}$$

linear appr. of E. when deformation is small, we do not differentiate x and X .

For engineering materials, $E \sim 2 \times 10^{11} \text{ Pa}$.

$$\sigma_Y \sim 2 \times 10^8 \text{ Pa} \quad \Rightarrow \quad \epsilon = E \frac{\partial U}{\partial x} < 10^{-3} \ll 1$$

thus, it is acceptable to use E.
as the quadratic term will vanish.

When there is rigid rotations.

$$\frac{\partial U}{\partial x} = Q - I \Rightarrow 2E = Q^T Q - I = 0$$

$$2E = Q + Q^T - 2I.$$

Consider 2D rotations $Q = \begin{bmatrix} \cos \theta & \sin \theta \\ -\sin \theta & \cos \theta \end{bmatrix}$

$E = (\cos \theta - 1) I \approx -\frac{\theta^2}{2} I$. good for small θ .

extreme case $\theta = \frac{\pi}{2}$. $E = -I$ (bad!)

More on the deformation gradient:

Spectrum decomposition:

there exist $\{\vec{N}_a\}$ and $\{\vec{n}_a\}$ $a=1, 2, 3$, and $\{\lambda_a\}$
 \uparrow \uparrow
 material, \uparrow spatial,
 mutually orthogonal mutually orthogonal,
 normalized normalized

such that

$$F = \sum_{a=1}^3 \lambda_a \vec{n}_a \otimes \vec{N}_a$$

principal stretches

principal spatial directions/axes

principal referential directions/axes

$$F^{-1} = \sum_{a=1}^3 \lambda_a^{-1} \vec{N}_a \otimes \vec{n}_a$$

$$F \vec{N}_a = \sum_{b=1}^3 \lambda_b \vec{n}_b (\vec{N}_b \cdot \vec{N}_a) = \lambda_a \vec{n}_a$$

$$F^{-1} \vec{n}_a = \sum_{b=1}^3 \lambda_b^{-1} \vec{N}_b \vec{n}_b \cdot \vec{n}_a = \lambda_a^{-1} \vec{N}_a$$

$$C = \sum_{a=1}^3 \lambda_a^2 \vec{N}_a \otimes \vec{N}_a$$

$$b = \sum_{a=1}^3 \lambda_a^2 \vec{n}_a \otimes \vec{n}_a$$

$$\Rightarrow E = \frac{1}{2} \sum_{a=1}^3 (\lambda_a^2 - 1) \vec{N}_a \otimes \vec{N}_a$$

$$e = \frac{1}{2} \sum_{a=1}^3 (1 - \lambda_a^{-2}) \vec{n}_a \otimes \vec{n}_a$$

the notion of strain can be generalized:

$$E^{(n)} = \frac{1}{n} \sum_{a=1}^3 (\lambda_a^n - 1) \vec{N}_a \otimes \vec{N}_a$$

$$e^{(n)} = \frac{1}{n} \sum_{a=1}^3 (1 - \lambda_a^{-n}) \vec{n}_a \otimes \vec{n}_a$$

and the logarithmic strain

$$E^{(0)} = \sum_{a=1}^3 \ln \lambda_a \vec{N}_a \otimes \vec{N}_a$$

$$e^{(0)} = \sum_{a=1}^3 \ln \lambda_a \vec{n}_a \otimes \vec{n}_a$$

Interesting features.

- Incompressibility : $\text{tr } E^{(0)} = 0$

- Volumetric and ~~additive~~ isochoric parts are additively split.

Thus, finite strain theory becomes similar to small strain theory.

In practice, we often form $C = F^T F$ and obtain $\{\vec{N}_a\}$ & $\{\lambda_a\}$ by performing eigen-decomposition. Then $\vec{n}_a = F \vec{N}_a / \lambda_a$.

Algorithm: W.M. Scherzinger & C.R. Dohrmann

CMAME 197 (2008) 4007-4015.

To be open-sourced.

MC lab owns a C++ implementation. (65

Volumetric - Distortional decomposition

$$F = (\underbrace{J^{-1/3} I}_{\text{Volumetric part}}) \bar{F} \quad \leftarrow \text{volume-preserving or distortional part.}$$

$$\text{obviously } \bar{F} = J^{1/3} F.$$

$$\bar{F} = \sum_{a=1}^3 \bar{\lambda}_a \vec{n}_a \otimes \vec{N}_a \quad \bar{\lambda}_a = J^{-1/3} \lambda_a$$

also, modified
deformation
gradient.

modified principal stretches

Ref: J.C. Simo & R.L. Taylor, CMAME 85 (1991) 273-310

$$\bar{C} = \bar{F}^T \bar{F} = J^{-2/3} C$$

$$\bar{b} = \bar{F} \bar{F}^T = J^{-2/3} b.$$