## **Hotrop and Mennen's Method**

The total resistance of a ship can be subdivided into:

## $R_T=R_F(1+k_1)+R_{APP}+R_W+R_B+R_{TR}+R_A$ Where,

$R_F$	Frictional resistance according to
	the ITTC 1957 friction formula
	$= 0.5. \rho V^2 SC_F$ .

$$C_F$$
 = 0.075/(Log<sub>10</sub>Re-2)<sup>2</sup>  
Re Reynold's No.=  $\rho$ VL / $\mu$ 

 $1+k_1$  Form factor describing the viscous resistance of the hull form in relation to  $R_{\rm F}$ 

R<sub>APP</sub> Appendage resistance

R<sub>W</sub> Wave-making and wave-breaking

resistance

R<sub>B</sub> Additional pressure resistance due to bulbous bow near the water surface

R<sub>TR</sub> Additional pressure resistance of immersed transom stern

R<sub>A</sub> Model-ship correlation resistance

The form factor of the hull can be predicted by:

$$1 + k_1 = c_{13} \{ 0.93 + c_{12} (B / L_R)^{0.92497}$$

$$(0.95 - C_P)^{-0.521448} (1 - C_P + 0.0225 \, lcb)^{0.6906} \}$$

In this formula,  $C_P$  is the prismatic coefficient based on the waterline length, L and lcb is the longitudinal centre of buoyancy forward of 0.5L as a percentage of L. Here,  $L_R$  is a parameter reflecting the length of the run according to:

$$L_R/L=1-C_P+0.06C_Plcb(4C_P-1)$$

$$C_{12}$$
= $(T/L)^{0.2228446}$  if  $T/L > 0.05$   
= $48.20(T/L-0.02)^{2.078}$ + $0.479948$  if  $0.02$ < $T/L$ < $0.05$   
= $0.479948$  if  $T/L$ < $0.02$ 

Where *T* is the average moulded draught.

 $C_{13}$ =1+0.003 $C_{stern}$  $C_{stern}$  will be -10, 0 and +10 if the afterbody form is of V-shaped, Normal and U shaped sections respectively. The wetted area of the hull can be approximated by:

$$S = L(2T + B)\sqrt{C_M} (0.453 + 0.4425 C_B - 0.2862 C_M - 0.003467 B/T + 0.3696 C_{WP}) + 2.38 A_{BT}/C_B$$

where  $A_{BT}$  is the transverse sectional area of the bulb at the position where the still-water surface intersects the stem.

The appendage resistance can be determined from  $R_{APP}=0.5 \rho V^2 S_{APP}(1+k_2)_{eq} C_F$  Where  $S_{APP}$  the wetted area of the appendages,  $1+k_2$  the appendage resistance factor

Approximate 1+k2 values

Approximate 7 · k <sub>2</sub> values		
Rudder behind skeg	1.5~2.0	
Rudder behind stern	1.3~1.5	
Twin-screw balance rudders	2.8	
Shaft brackets	3.0	
Skeg	1.5~2.0	
Strut bossings	3.0	
Hull bossings	2.0	
Shafts	2.0~4.0	
Stabilizer fins	2.8	
Dome	2.7	
Bilge keels	1.4	

The equivalent  $1+k_2$  value for a combination of appendages is determined from:

$$(1+k_2)_{eq} = \frac{\sum (1+k_2)S_{APP}}{\sum S_{APP}}$$

The wave resistance is determined from:

$$R_{W} = c_{1}c_{2}c_{5}\nabla\rho g \exp\{m_{1}F_{n}^{d} + m_{2}\cos(\lambda F_{n}^{-2})\}$$
 with

$$\begin{split} c_1 &= 2223105 c_7^{3.78613} (T/B)^{1.07961} (90-i_E)^{-1.37565} \\ c_7 &= 0.229577 (B/L)^{0.33333} \quad \text{if B/L} < 0.11 \\ &= B/L \qquad \text{if 0.11} < B/L < 0.25 \\ &= 0.5 - 0.0625 \quad L/B \qquad \text{if B/L} > 0.25 \\ c_2 &= \exp(-1.89\sqrt{c_3} \end{split}$$

if L/B>12

$$\lambda = 1.446C_P - 0.03 \text{ L/B}$$
 if L/B < 12

 $c_5 = 1 - 0.48 A_T / (BTC_M)$ 

 $= 1.446C_P - 0.36$ 

$$\begin{split} m_1 &= 0.0140407L/T - 1.75254\nabla^{1/3}/L \\ &- 4.79323B/L - c_{16} \\ c_{16} &= 8.07981C_P - 13.8673C_P^2 + 6.984388C_P^3 \\ &= if C_P < 0.8 \\ &= 1.73014 - 0.7067C_P \quad \text{if } C_P > 0.8 \\ m_2 &= c_{15}C_P^2 \exp(-0.1F_n^{-2}) \\ c_{15} &= -1.69385 \text{ for } L^3/\Psi < 512 \\ &= 0 \quad \text{for } L^3/\Psi > 1727 \\ &= -1.69385 + (L/\Psi^{1/3} - 8.0)/2.36 \\ &= if 512 < L^3/\Psi < 1727 \\ d &= -0.9 \\ i_E &= 1 + 89 \exp\{-(L/B)^{0.80856}(1 - C_{WP})^{0.30484} \\ &= (1 - C_P - 0.0225lcb)^{0.6367}(L_R/B)^{0.34574} \\ &= (100\nabla/L^3)^{0.16302}\} \\ c_3 &= 0.56A_{BT}^{1.5}/\{BT(0.31\sqrt{A_{BT}} + T_F - h_B)\} \end{split}$$

where  $h_B$  is the position of the centre of the transverse area  $A_{BT}$  above the keel line and  $T_F$  is the forward draught of the ship.

$$\begin{split} R_B &= 0.11 \exp(-3P_B^{-2}) F_{ni}^3 A_{BT}^{1.5} \rho g \, / (1 + F_{ni}^2) \\ P_B &= 0.56 \sqrt{A_{BT}} \, / (T_F - 1.5 h_B) \\ F_{ni} &= V \, / \sqrt{g (T_F - h_B - 0.25 \sqrt{A_{BT}}) + 0.15 V^2} \\ R_{TR} &= 0.5 \rho V^2 A_T c_6 \\ c_6 &= 0.2 (1 - 0.2 F_{nT}) \quad \text{if } F_{nT} < 5 \\ &= 0 \qquad \qquad \text{if } F_{nT} \ge 5 \\ F_{nT} &= V \, / \sqrt{2 g A_T \, / (B + B C_{WP})} \\ R_A &= \frac{1}{2} \rho V^2 S C_A \end{split}$$

$$\begin{split} C_A &= 0.006(L+100)^{-0.16} - 0.00205 \\ &+ 0.003\sqrt{L/7.5}C_B^4c_2(0.04-c_4) \\ c_4 &= T_F/L \quad \text{when } T_F/L \leq 0.04 \\ c_4 &= 0.04 \quad \text{when } T_F/L > 0.04 \end{split}$$

<u>Problem:</u> The characteristics of a ship is as follows:

L.O.W L=205.00 m L.B.P.  $L_{PP}$  = 200.00 m Breadth moulded B = 32.00 mDraught moulded on F.P,  $T_F$ =10.00 m Draught moulded on A. P.  $T_A$ =10.00 m Displacement volume moulded, ▼=37500 m<sup>3</sup> Longitudinal centre of buoyancy 2.02% aft of  $1/2L_{PP}$ Transverse bulb area  $A_{BT}$  = 20.0 m<sup>2</sup> Centre of bulb area above keel line  $h_B = 4.0 \text{ m}$ Midship section coefficient  $C_M = 0.98$ Waterplane area coefficient  $C_{WP} = 0.75$ Transom area  $A_T = 16.0 \text{ m}^2$  $S_{APP} = 50.0 \text{ m}^2$ Wetted area appendages Stern shape parameter.  $C_{stern} = 10.0$ Propeller diameter, D = 8.0 mNumber of propeller blades Z = 4Clearance of propeller with keel line 0.20 m V=25.0 knos Ship speed Density,  $\rho = 1025.87$ Kinematic Viscosity,  $\nu$  = 1.18831e-006

Find R<sub>F</sub>, R<sub>APP</sub>, R<sub>W</sub>, R<sub>B</sub>, R<sub>TR</sub>, R<sub>A</sub>, R<sub>total</sub>.

**Reference:** J. Holtrop and G.G. J. Mennen, 1982: **An Approximate Power Prediction Method**, International Shipbuilding Progress, Vol. 29, No. 335.