Shallow water

Partial differential equations

Model formulation

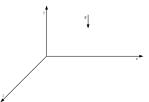
Case of smooth solutions

Linearization

Traveling wave solutions

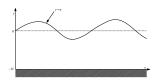
Spatial domain

We consider the motion of a body of water that is infinite in the z direction, with or without boundary in the x direction, and the vertical direction of gravity taken as the y direction.



From now on, suppose z direction uniform (the same for all z), so ignore z except for the sake of argument.

- Water depth at rest, H, small compared to distance L₀ over which significant changes can occur in the x direction.
- Undisturbed water surface, y = 0.
 - ▶ Moving upper free surface $y = \eta$, measured from y = 0.
 - ▶ Sea floor y = -H.



- u velocity in the x direction. Assume independent of depth y.
- ρ mass density of water.
- p(x,y,t) pressure in fluid at point (x,y) at time t. In water, magnitude at any (x,y) is same in all directions.

Fluid motion independent of z, so

- ightharpoonup u = u(x, t)
- $\ \, \eta = \eta(x,t).$

Model formulation

Weight of water column:

$$\iint \int_{-1}^{32} (-\rho g) \, dy dx dz$$

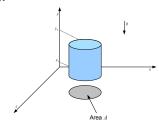
Pressure differential:

$$\iint (p(x, y_2, t) - p(x, y_1, t)) \ dxdz$$

So we must have

$$\iint \int\limits_{-}^{y_2} (-\rho g) \ dy dx dz = \iint \left(\rho(x,y_2,t) - \rho(x,y_1,t) \right) \ dx dz$$

Take a cylindrical water column, with base area A, between y_1 and $y_2 > y_1$.



Force equilibrium in the y direction in this cylinder requires balance of weight of water column and pressure differential between bottom face $y=y_1$ and top face $y=y_2$.

 $\iint\limits_{A} \int\limits_{y_1}^{y_2} (-\rho g) \ dy dx dz = \iint\limits_{A} \left(\rho(x, y_2, t) - \rho(x, y_1, t) \right) \ dx dz$

is equivalent to

$$\iint \int_{a}^{y_2} \left(\frac{\partial p}{\partial y} + \rho g \right) dy dx dz = 0$$

This must be true for any water column, i.e., any A, y_1, y_2 . Therefore

$$\frac{\partial p}{\partial u} + \rho g = 0$$

(otherwise, we would be able to find a water column where the integrand is positive, leading to a positive value of the integral on that column).

Water is incompressible

If you force a body of water to deform, the volume of that body of water remains constant, i.e., water is an *incompressible fluid*.

 $\Rightarrow \rho,$ the density, is a constant, and from

$$\frac{\partial p}{\partial v} + \rho g = 0$$

we get

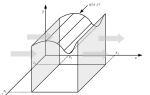
$$p = -\rho gy + C$$

so if p is measured relative to the pressure above the free upper surface $y=\eta$,

$$p = \rho g(\eta - y)$$

Model formulation

Water enters V through x_1 face and leaves V through x_2 face.



Rate of water accumulation in V is

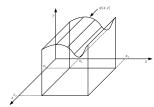
$$\frac{d}{dt} \int_{z_1}^{z_2} \int_{y_1}^{x_2} \int_{-H}^{\eta} \rho \ dy dx dz = \Delta z \frac{d}{dt} \int_{y_1}^{x_2} \rho h \ dx,$$

with $\Delta z = z_2 - z_1$, and $h(x,t) = \eta + H$ the height of water at time t at spatial location x.

Water accumulation

Consider a fixed volume V,

$$V=\{z_1\leq z\leq z_2, x_1\leq x\leq x_2, -H\leq y\leq \eta\}$$



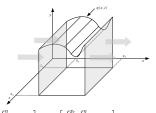
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Model formulation

/ater flux



Net flux of water entering V through its faces $x=x_1$ and $x=x_2$ is



 $\left[\int_{z_1}^{z_2} \int_{-H}^{\eta} u \ dy dz\right]_{x=x_1} - \left[\int_{z_1}^{z_2} \int_{-H}^{\eta} u \ dy dz\right]_{x=x_2} = -\Delta z \left[\rho u h\right]_{x_1}^{x_2}$ There is no flux through y=-H and $y=\eta$, and no net flux

through $z = z_1$ and $z = z_2$.

Conservation of mass

Of course, the mass must conserve in V, so the two expressions must be equal, i.e.,

$$\frac{d}{dt}\int_{x_1}^{x_2} \rho h \ dx + [\rho u h]_{x_1}^{x_2} = 0$$

Newton's second law for deformable media (Euler): rate of increase of horizontal momentum (in the x direction) in V must equal the sum of the net influx of momentum into the volume and the net horizontal force acting on the column.

(Momentum: product of mass and velocity of an object).

Rate of increase of momentum

$$\frac{d}{dt}\int\limits_{z_{1}}^{z_{2}}\int\limits_{x_{1}}^{\infty}\int\limits_{-H}^{\eta}\rho u\ dydxdz=\Delta z\frac{d}{dt}\int\limits_{x_{1}}^{\infty}\rho uhdx$$

Model formulation

Momentum flux

Net influx of momentum through faces $x=x_1$ and $x=x_2$ is

$$\begin{bmatrix} \displaystyle \int\limits_{z_1}^{z_2} \int\limits_{-H}^{\eta} (\rho u) u \ dy dz \end{bmatrix}_{x=x_1} - \begin{bmatrix} \displaystyle \int\limits_{z_1}^{z_2} \int\limits_{-H}^{\eta} (\rho u) u \ dy dz \end{bmatrix}_{x=x_2} \\ = -\Delta z \left[\rho u^2 h \right]_{x_2}^{x_2}$$

There is no flux through y=-H and $y=\eta$, and no net flux through $z=z_1$ and $z=z_2$.

Model formulation

Forces acting on V

Ignore friction at y=-H. Then only contributions to horizontal forces come from pressure at $x=x_1$ and $x=x_2$, so net horizontal forces acting on V is

$$\begin{split} \left[\int\limits_{z_1}^{z_2} \int\limits_{-H}^{\eta} \rho \ dy dz \right]_{x_1}^{x_2} &= - \left[\Delta z \int\limits_{-H}^{\eta} \rho g(\eta - y) \ dy \right]_{x_1}^{x_2} \\ &= \left[-\Delta z \rho g(\eta y - \frac{1}{2} y^2) \Big|_{-H}^{\eta} \right]_{x_1}^{x_2} \\ &= \left[-\frac{1}{2} \Delta z \rho g h^2 \right]_{x_1}^{x_2} \end{split}$$

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Conclusion from Newton's second law

Pressure magnitude:

 $p = \rho g(n - v)$ (1)

Horizontal velocity:

The general model

 $\frac{d}{dt}\int_{-\infty}^{\infty} \rho h \ dx + [\rho u h]_{x_1}^{x_2} = 0$ (2)

Free surface height:

$$\frac{d}{dt} \int_{-\infty}^{\infty} \rho u h \ dx + \left[\rho u^2 h + \frac{1}{2} \rho g h^2 \right]_{x_1}^{x_2} = 0$$
 (3)

Model formulation

Model formulation

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Suppose u and h are smooth (with continuous first order partial derivatives), then (2) and (3) take a much simpler form,

$$\int_{-\infty}^{x_2} \left(\frac{\partial h}{\partial t} + \frac{\partial}{\partial x} (uh) \right) dx = 0$$

and

 $\int_{0}^{x_{2}} \left(\frac{\partial}{\partial t} (uh) + \frac{\partial}{\partial x} (u^{2}h + \frac{1}{2}gh^{2}) \right) dx = 0$

Since the intervals of integration $[x_1, x_2]$ are arbitrary, and that the integrands are continuous, we have

$$\frac{\partial h}{\partial t} + \frac{\partial}{\partial x}(uh) = 0$$

and

$$\frac{\partial}{\partial t}(uh) + \frac{\partial}{\partial x}(u^2h + \frac{1}{2}gh^2) = 0$$

$$\frac{d}{dt} \int_{x_1}^{\infty} \rho u h \ dx + \left[\rho u^2 h + \frac{1}{2} \rho g h^2 \right]_{x_1}^{x_2} = 0$$

Case of smooth solutions

We write

$$\frac{\partial h}{\partial t} + \frac{\partial}{\partial x}(uh) = 0$$

and

$$\frac{\partial}{\partial t}(uh) + \frac{\partial}{\partial x}(u^2h + \frac{1}{2}gh^2) = 0$$

as

$$h_t + (uh)_x = 0 (4)$$

and

$$(uh)_t + (u^2h + \frac{1}{2}gh^2)_x = 0 (5)$$

From (4),

$$h_t = -(uh)_x = -(u_x h + uh_x)$$

Equation (5) can be rewritten as

$$\begin{split} (5) &\Leftrightarrow u_t h + u h_t + \left(u^2 h + \frac{1}{2} g h^2\right)_x = 0 \\ &\Leftrightarrow u_t h - u \left(u_x h + u h_x\right) + 2 u u_x h + u^2 h_x + g h h_x = 0 \\ &\Leftrightarrow u_t h - u u_x h - u^2 h_x + 2 u u_x h + u^2 h_x + g h h_x = 0 \\ &\Leftrightarrow u_t h + u u_x h + g h h_x = 0 \end{split}$$

Therefore, provided $h \neq 0$, we get

$$h_t + (uh)_x = 0 (6a)$$

$$u_t + uu_x + gh_x = 0 (6b)$$

which describes the evolution of u and h.

Case of smooth solutions

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The model for smooth solutions

$$h_t + (uh)_x = 0 (6a)$$

$$u_t + uu_x + gh_x = 0 (6b)$$

If $-\infty < x < \infty$, then all we need is an initial condition, i.e., functions describing the initial state of u and h:

$$u(x,0) = u_0(x), \quad h(x,0) = h_0(x), \quad -\infty < x < \infty,$$

If x has a boundary, then we need boundary conditions.

Model formulation

Case of smooth solutions

Linearization

Traveling wave solutions

Case of smooth solutions p. 22

Suppose the bottom is flat (H is constant), and that the deviation from the undisturbed depth H is small compared to H itself, then

$$h = (H + \zeta) = H(1 + \frac{\zeta}{H}) \simeq H, \qquad h_t = \zeta_t, \qquad h_x = \zeta_x.$$

If |u| is also small, then uu_x can be neglected. Then we can linearize

$$h_t + (uh)_x = 0 (6a)$$

 $u_t + uu_x + gh_x = 0, (6b)$

getting

Linearization

$$\zeta_t + Hu_x = 0 \tag{7a}$$

$$u_t + g\zeta_x = 0 (7b)$$

Differentiate (7b) with respect to x:

$$u_{tx} + g\zeta_{xx} = 0$$

and therefore,

$$u_{\mathsf{tx}} = -g\zeta_{\mathsf{xx}} \tag{8}$$

Differentiate (7a) with respect to t:

$$\zeta_{tt} + Hu_{xt} = 0 \tag{9}$$

Clairaut's theorem, $u_{tx}=u_{xt}$. Therefore, substituting (8) into (9), $\zeta_{tt}-HG\zeta_{xx}=0$

If u has continuous second-order partial derivatives, then from

$$\zeta_{tt} = c^2 \zeta_{xx}, \qquad c^2 = Hg$$

that is

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The one-dimensional wave equation (1)

The partial differential equation

$$\zeta_{tt} = c^2 \zeta_{xx} \tag{10}$$

with $c^2={\it Hg}$, is the one-dimensional wave equation. Initial conditions are given by

$$\zeta(x,0) = h_0(x) - H \equiv \zeta_0(x)$$

 $\zeta_t(x,0) = -Hu_x(x,0) = -H[u_0(x)]_x \equiv \nu_0(x)$

Linearization

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The one-dimensional wave equation (2)

Things can also be expressed in terms of u. Using the same type of

$$u_{tt} = c^2 u_{xx} \tag{11}$$

with $c^2={\it Hg}\,.$ Initial conditions are given by

simplification used before for ζ , we get

$$u(x,0) = u_0(x) u_t(x,0) = -g\zeta_x(x,0) = -g[h_0(x)]_x \equiv v_0(x)$$

Model formulation

Case of smooth solutions

Linearization

Traveling wave solutions

Derivation of the solution

Introduce the new variables

$$a = x - ct$$
 and $b = x + ct$

We have

have
$$\frac{\partial u}{\partial x} = \frac{\partial u}{\partial a} + \frac{\partial u}{\partial b} \qquad \frac{\partial u}{\partial t} = -c\frac{\partial u}{\partial a} + c\frac{\partial u}{\partial b}$$

$$\frac{\partial^2}{\partial x^2} u = \left(\frac{\partial}{\partial a} + \frac{\partial}{\partial b}\right)^2 u = \frac{\partial^2 u}{\partial a^2} + 2\frac{\partial^2 u}{\partial a \partial b} + \frac{\partial^2 u}{\partial b^2}$$

$$\frac{\partial^2}{\partial t^2} u = \left(-c\frac{\partial}{\partial a} + c\frac{\partial}{\partial b}\right)^2 u = c^2 \left(\frac{\partial^2 u}{\partial a^2} - 2\frac{\partial^2 u}{\partial a \partial b} + \frac{\partial^2 u}{\partial b^2}\right)$$

Traveling wave solutions

This was obtained by d'Alembert. Consider

$$u_{tt} = c^2 u_{xx} \tag{11}$$

Note that this can be written as

$$\left(\frac{\partial}{\partial t} - c\frac{\partial}{\partial x}\right) \left(\frac{\partial}{\partial t} + c\frac{\partial}{\partial x}\right) u = 0$$

This implies that for any F, G, the sum

$$u(x,t) = F(x-ct) + G(x+ct)$$

satisfies (11).

Traveling wave solutions

So the equation

$$u_{tt} = c^2 u_{xx} (11$$

is written

$$4\frac{\partial^2 u}{\partial a \partial b} = 0$$

Integrate with respect to b:

$$\frac{\partial u}{\partial a} = \xi(a)$$

and thus

$$u(x,t) = u(a,b) = \int \xi(a)da + G(b)$$
$$= F(a) + G(b)$$
$$= F(x - ct) + G(x + ct)$$

Traveling wave solutions

Set

$$u(x,0) = f(x)$$
 $u_t(x,0) = g(x)$

Then d'Alembert's formula gives

$$u(x,t) = \frac{f(x-ct) + f(x+ct)}{2} + \frac{1}{2c} \int_{x-ct}^{x+ct} g(s) ds$$

Traveling wave solutions

For simplicity, take c=1. This gives

$$u(x,t) = \frac{1}{2} \{ H(x+t) - H(x-t) \},\,$$

Case of a Dirac delta initial condition

Suppose $u_0(x)=0$ and $v_0(x)=\delta(x)$, for $-\infty < x < \infty$, with δ the Dirac delta.

$$\delta(x) = \begin{cases} \infty & \text{if } x = 0 \\ 0 & \text{otherwise.} \end{cases}$$

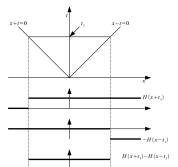
Therefore,

$$u(x,t) = \frac{1}{2c} \int_{x-ct}^{x+ct} \delta(z) dz = \frac{1}{2c} \left\{ H(x+ct) - H(x-ct) \right\},$$

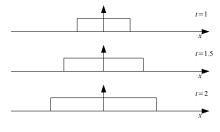
with \boldsymbol{H} the Heaviside function,

$$H(x) = \begin{cases} 0 & \text{if } x < 0 \\ 1 & \text{if } x > 0. \end{cases}$$

Traveling wave solutions



As t increases, we move further up in the top graph in (x,t)-space, resulting in a wider and wider square pulse.



Traveling wave solutions