

# Connecting Shells and Solids

The Shell interface can be used to model thin structures in 3D. The thickness of a shell is taken into account mathematically, but is not explicitly modeled in the geometry. This makes the use of the Shell interface efficient because only boundaries have to be meshed. The Shell interface provides good results as long as the structures are thin. In practice, it sometimes happens that this requirement is not met in the entire geometry, so that local regions require a full solid model. In such a case, the Shell interface can be used for thinner parts, and then be coupled to a Solid Mechanics interface for other parts.

This example demonstrates how you model a transition from a shell to a solid. The assumptions and accuracy of such a transition is also discussed.

# Model Definition

The model geometry (Figure 1) consists of a cylindrical steel pipe with three circular cutouts along its length on one side. The wall thickness of 0.1 m is small compared to the radius of 1 m. The entire structure could thus be modeled with the Shell interface. To demonstrate the connection between shells and a solid domain, the geometry is instead separated in three sections. The middle section is modeled as a solid domain that is connected to a shell on either side.

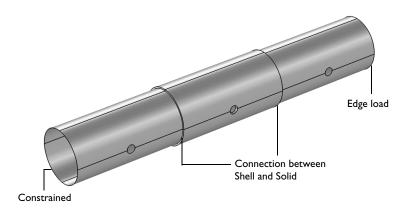


Figure 1: The geometry divided into regions modeled using shells and solids.

The model is set up with a Shell interface for the outer parts, and a Solid Mechanics interface for the solid domain in the middle. Where the edges of the shell midsurfaces meet the solid boundaries, special connection boundary conditions are added through the Solid-**Thin Structure Connection** multiphysics coupling. Different types of conditions are used for the connection on the left and right sides, in order to highlight their properties.

Three different load cases are analyzed, tension, torsion and internal pressure. One end of the geometry is constrained, while the other is subject to different edge loads corresponding to a tension, and torsion. In third load case the pipe is subject to an internal pressure.

# Results and Discussion

The von Mises stress distribution for the pressure load case is shown in Figure 2. The overall stress state in the shells is consistent with the results obtained in the solid domain. There is however a slight difference, since the solid captures the variation of the stress through the thickness correctly.

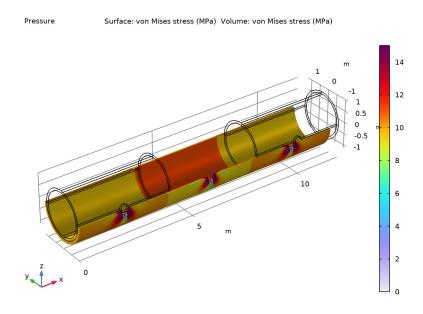


Figure 2: The von Mises stress distribution of the pressure load case.

The stress distribution along the top of the pipe for all three load cases is shown in Figure 3, Figure 4, and Figure 5. A closer look reveals some differences in the stress results, especially near the connected region. To interpret the graphs, note the following:

- The direction of the graphs runs from the constrained end of the pipe toward the loaded end.
- The graphs follow the perimeter of the holes.
- The left end of the Shell, Rigid graph is at the constrained end. The constraint may cause local disturbances there.
- The right end of the Shell, Rigid graph is connected to the left end of the Solid graph at the rigid connection.
- The right end of the Solid graph is connected to the left end of the Shell, Flexible graph at the flexible connection.
- The right end of the Shell, Flexible graph is at the loaded end, where some local disturbances can appear.

From the graphs, it can be seen that:

### Tension case

- The correspondence between the solid and shell results are in general very good. This can be expected since the axial tension should give a homogeneous stress state in the undisturbed regions. At the hole edge, where stress gradients are steep, and threedimensional effects important, there is a certain difference.
- There is a disturbance in the results at both transitions between shell and solid, but it is more pronounced where the rigid formulation is used. The transition effect is seen only in the solid.
- The details of the stresses in the transition can be seen in Figure 6 and Figure 7. In the latter figure it is clear that the flexible condition can describe the fact that the transverse stress should be zero much better than the rigid counterpart. The small bending stress seen also for the flexible connection depends on the fact that there is actually somewhat less material inside of the midsurface than it is outside of it. For a flat transition it would disappear almost completely.

### Torsion case

• There is a general difference between the stress level in the shell and in the solid. The shell solution overestimates the through-thickness gradient of the shear stress somewhat. This is not related to the mixture of element types.

- No transition effects can be seen. This is expected, since the stress state is pure shear in the plane of the shell. There are no transverse strains, so the same strain state should be possible to represent by both element types without extra assumptions.
- The stress at the transitions is shown in Figure 8. The nominal stress level is the theoretical value. The linear through-thickness distribution is predicted by the torsion theory, and the equivalent stress shows no influence of other stress components than the shear stress.

### Pressure case

- Also in this case, there is a difference in level between the shell solution and the solid solution. In the shell, only the midsurface stress is represented, whereas the solid can resolve the gradient in the thickness direction.
- The transition effects are rather small, but approximately of the same size at both transitions. The stress state is predominantly a direct stress in the circumferential direction.
- The hoop stress at the transitions is shown in Figure 9. The nominal value represents the theoretical value. It can be seen that the predictions using the flexible connection are significantly better than when using the rigid connection.

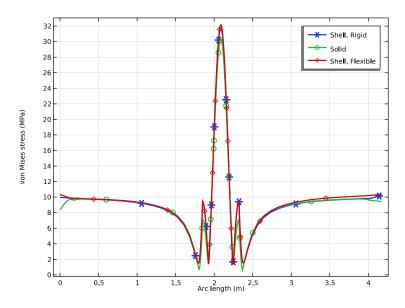


Figure 3: The von Mises stress plotted along the top of the pipe for the tension load case.

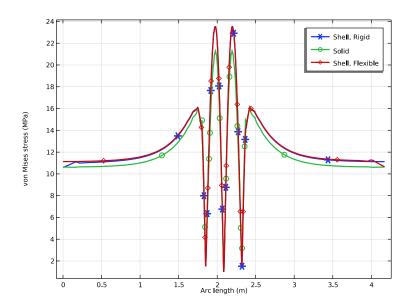


Figure 4: The von Mises stress plotted along the top of the pipe for the torsion load case.

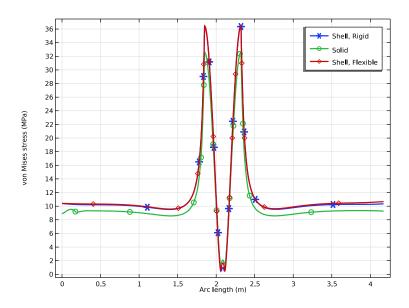


Figure 5: The von Mises stress plotted along the top of the pipe for the pressure load case.

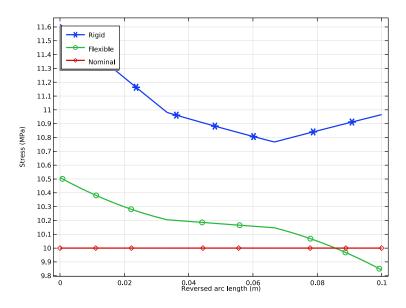


Figure 6: Axial stress through the thickness at the transitions (tension).

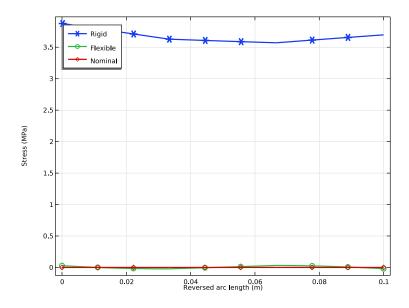


Figure 7: Transverse stress through the thickness at the transitions (tension).

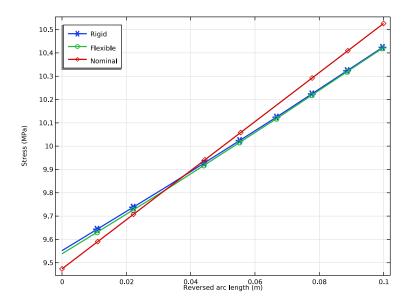


Figure 8: Equivalent stress through the thickness at the transitions (torsion).

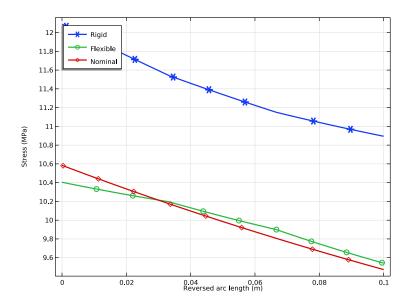


Figure 9: Hoop stress through the thickness at the transitions (pressure).

The built-in **Solid-Thin Structure Connection** multiphysics coupling is used to connect the Solid Mechanics and Shell interfaces. The shell deformation is described with 6 DOFs, three displacement components and three components for the rotation of the shell normal. In the Solid Mechanics domain the latter DOFs are not present, as the 3D state of deformation is resolved explicitly with a 3D mesh. The 6 DOFs per node of the shell edge thus have to be connected to the 3 DOFs per node on the boundary of the connection. There are two options for this connection, **Rigid** and **Flexible**. The first type violates the shell assumptions somewhat and leads to unphysical stress states locally at the connection. This aberration disappears within a few elements from the connection and the results can be acceptable in many situations. The flexible connection introduces new solution variables, the softening functions. The benefit is more accurate stress results at the connection, but it comes with an extra computational cost. The flexible connection cannot be used if the mesh in the thickness direction of the solid is too coarse. The required number of elements needed depends on the shape function order, but with standard quadratic shape function, at least three elements are needed in the thickness direction.

Additional details about the connection types can be found in the documentation for the Shell interface. In this model, both connection types are used, one on either side of the solid domain.

Application Library path: Structural Mechanics Module/Beams and Shells/ shell solid connection

# Modeling Instructions

From the File menu, choose New.

### NEW

In the New window, click Model Wizard.

### MODEL WIZARD

- I In the Model Wizard window, click **3D**.
- 2 In the Select Physics tree, select Structural Mechanics>Shell (shell).
- 3 Click Add.
- 4 In the Select Physics tree, select Structural Mechanics>Solid Mechanics (solid).

- 5 Click Add.
- 6 Click  $\bigcirc$  Study.
- 7 In the Select Study tree, select General Studies>Stationary.
- 8 Click **Done**.

### **GLOBAL DEFINITIONS**

Define Parameters and Load Groups for use in the model setup.

### Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click **Load from File**.
- 4 Browse to the model's Application Libraries folder and double-click the file shell solid connection parameters.txt.

# Load Group, Tension

- I In the Model Builder window, right-click Global Definitions and choose Load and Constraint Groups>Load Group.
- 2 In the Settings window for Load Group, type Load Group, Tension in the Label text field.
- 3 In the Parameter name text field, type 1gFx.

### Load Group, Torsion

- I In the Model Builder window, right-click Load and Constraint Groups and choose Load Group.
- 2 In the Settings window for Load Group, type Load Group, Torsion in the Label text field.
- 3 In the Parameter name text field, type 1gMx.

### Load Group, Pressure

- I Right-click Load and Constraint Groups and choose Load Group.
- 2 In the Settings window for Load Group, type Load Group, Pressure in the Label text field.
- 3 In the Parameter name text field, type 1gP.

### GEOMETRY I

Build the geometry by creating an array of three solid cylindrical parts with a cutout.

Cylinder I (cyl1)

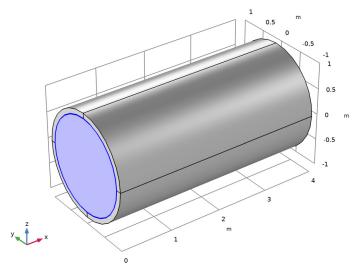
- I In the **Geometry** toolbar, click **Cylinder**.
- 2 In the Settings window for Cylinder, locate the Size and Shape section.
- 3 In the Radius text field, type r\_outer.
- 4 In the Height text field, type length.
- 5 Locate the Axis section. From the Axis type list, choose x-axis.
- **6** Click to expand the **Layers** section. In the table, enter the following settings:

Layer name	Thickness (m)
Layer 1	thickness

7 Click Pauld Selected.

Delete Entities I (del I)

- I In the Geometry toolbar, click **Delete**.
- 2 In the Settings window for Delete Entities, locate the Entities or Objects to Delete section.
- 3 From the Geometric entity level list, choose Domain.
- 4 On the object cyll, select Domain 3 only.



5 Click **Build Selected.** 

Cylinder 2 (cyl2)

I In the Geometry toolbar, click ( Cylinder.

- 2 In the Settings window for Cylinder, locate the Size and Shape section.
- 3 In the Radius text field, type r hole.
- 4 In the Height text field, type 1.5\*r outer.
- 5 Locate the **Position** section. In the x text field, type length/2.
- 6 In the y text field, type -1.5\*r\_outer.
- 7 Locate the Axis section. From the Axis type list, choose y-axis.
- 8 Click | Build Selected.

Difference I (dif1)

- I In the Geometry toolbar, click Booleans and Partitions and choose Difference.
- 2 Select the object dell only.
- 3 In the Settings window for Difference, locate the Difference section.
- 4 Click to select the Activate Selection toggle button for Objects to subtract.
- **5** Select the object **cyl2** only.
- 6 Click | Build Selected.

Array I (arrI)

- I In the Geometry toolbar, click \( \sum\_{\text{transforms}} \) Transforms and choose Array.
- 2 Select the object difl only.
- 3 In the Settings window for Array, locate the Size section.
- 4 In the x size text field, type 3.
- **5** Locate the **Displacement** section. In the **x** text field, type length.
- 6 Click | Build Selected.
- 7 Click the Go to Default View button in the Graphics toolbar.

### DEFINITIONS

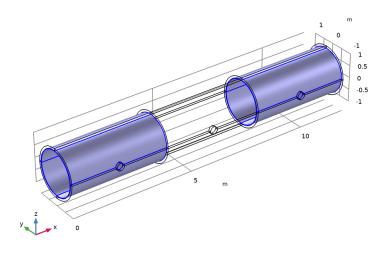
Create Explicit Selections for various boundary and edge groups. It is easier selecting geometric entities using the wireframe rendering.

I Click the Wireframe Rendering button in the Graphics toolbar.

Shell

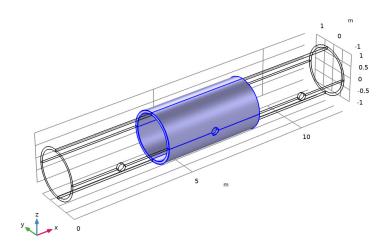
- I In the **Definitions** toolbar, click **\( \frac{1}{2} \) Explicit**.
- 2 In the Settings window for Explicit, type Shell in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.

**4** Select Boundaries 6, 7, 11, 14, 48, 49, 53, and 56 only.



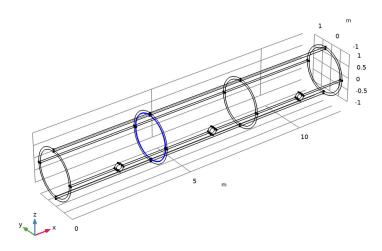
# Solid

- I In the Definitions toolbar, click 堶 Explicit.
- 2 In the Settings window for Explicit, type Solid in the Label text field.
- **3** Select Domains 5–8 only.



# Edges for Rigid Connection

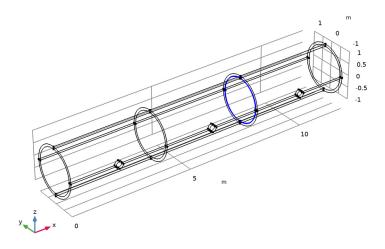
- I In the **Definitions** toolbar, click 🔓 **Explicit**.
  - The **Group by continuous tangent** option enables the selection of contiguous boundaries and edges with just one click.
- 2 In the Settings window for Explicit, type Edges for Rigid Connection in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Edge.
- 4 Select the Group by continuous tangent check box.
- **5** Select Edges 39, 40, 45, and 48 only.



# Edges for Flexible Connection

- I In the **Definitions** toolbar, click 堶 **Explicit**.
- 2 In the Settings window for Explicit, type Edges for Flexible Connection in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Edge.
- 4 Select the Group by continuous tangent check box.

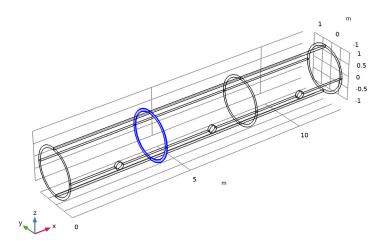
**5** Select Edges 73, 74, 79, and 82 only.



# Boundaries for Rigid Connection

- I In the **Definitions** toolbar, click **\( \bigcap\_{\text{a}} \) Explicit**.
- 2 In the Settings window for Explicit, type Boundaries for Rigid Connection in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Select the Group by continuous tangent check box.

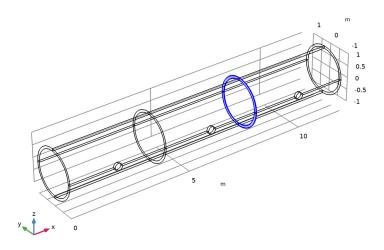
**5** Select Boundaries 22, 23, 29, and 33 only.



# Boundaries for Flexible Connection

- I In the **Definitions** toolbar, click **\( \bigcap\_{\text{a}} \) Explicit**.
- ${f 2}$  In the  ${f Settings}$  window for  ${f Explicit}$ , type Boundaries for Flexible Connection in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Select the Group by continuous tangent check box.

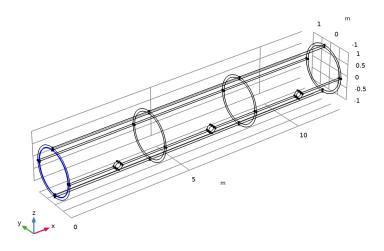
**5** Select Boundaries 43, 44, 50, and 54 only.



# Constrained Edges

- I In the Definitions toolbar, click 堶 Explicit.
- $\textbf{2} \ \ \text{In the Settings} \ \text{window} \ \text{for Explicit}, type \ \text{Constrained} \ \ \text{Edges} \ \text{in the Label} \ \text{text} \ \text{field}.$
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Edge.
- 4 Click the Go to Default View button in the Graphics toolbar.
- **5** Select the **Group by continuous tangent** check box.

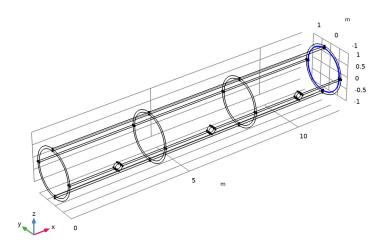
**6** Select Edges 5, 6, 11, and 14 only.



# Loaded Edges

- I In the **Definitions** toolbar, click 堶 **Explicit**.
- 2 In the Settings window for Explicit, type Loaded Edges in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Edge.
- 4 Select the Group by continuous tangent check box.

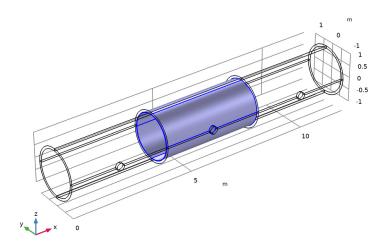
**5** Select Edges 106, 107, 110, and 112 only.



# Solid Inside

- I In the **Definitions** toolbar, click 🔓 **Explicit**.
- 2 In the Settings window for Explicit, type Solid Inside in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.

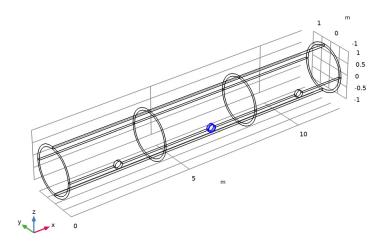
4 Select Boundaries 27, 28, 32, and 35 only.



# Solid Hole

- I In the **Definitions** toolbar, click 🔓 **Explicit**.
- 2 In the Settings window for Explicit, type Solid Hole in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Select the Group by continuous tangent check box.

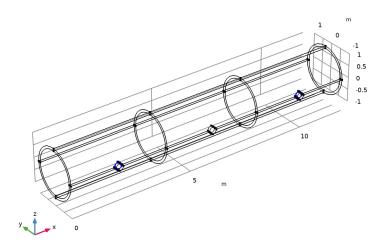
# **5** Select Boundaries 38–41 only.



# Shell Hole Edges

- I In the **Definitions** toolbar, click 堶 **Explicit**.
- 2 In the Settings window for Explicit, type Shell Hole Edges in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Edge.
- 4 Select the Group by continuous tangent check box.

**5** Select Edges 24, 25, 30, 31, 92, 93, 98, and 99 only.



6 Click the Wireframe Rendering button in the Graphics toolbar.

Cylindrical System 2 (sys2)

- I In the Definitions toolbar, click  $\sqrt[2]{x}$  Coordinate Systems and choose Cylindrical System. You will use this cylindrical coordinate system for the definition of constraint and the torsional load.
- 2 In the Settings window for Cylindrical System, locate the Settings section.
- 3 Find the Longitudinal axis subsection. In the table, enter the following settings:

х	у	z
1	0	0

**4** Find the **Direction of axis**  $\phi$ **=0** subsection. In the table, enter the following settings:

x	у	z
0	1	0

Since both the Shell and Solid Mechanics parts use the same material data, start by adding Structural Steel as a global material and then use material links at domain and boundary levels.

### ADD MATERIAL

In the Home toolbar, click **Add Material** to open the Add Material window.

### ADD MATERIAL

- I Go to the Add Material window.
- 2 In the tree, select Built-in>Structural steel.
- 3 Click the right end of the Add to Component split button in the window toolbar.
- 4 From the menu, choose Add to Global Materials.
- 5 In the Home toolbar, click **‡** Add Material to close the Add Material window.

### MATERIALS

Material Link I (matlnk I)

In the Model Builder window, under Component I (compl) right-click Materials and choose More Materials>Material Link.

Material Link 2 (matlnk2)

- I Right-click Materials and choose More Materials>Material Link.
- 2 In the Settings window for Material Link, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Shell.

### SHELL (SHELL)

- I In the Model Builder window, under Component I (compl) click Shell (shell).
- 2 In the Settings window for Shell, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Shell**.

Thickness and Offset I

- I In the Model Builder window, under Component I (compl)>Shell (shell) click Thickness and Offset 1.
- 2 In the Settings window for Thickness and Offset, locate the Thickness and Offset section.
- **3** In the  $d_0$  text field, type thickness. Since the shell is modeled on the inside of the pipe, describe the actual midsurface by an offset.
- 4 From the Position list, choose Top surface on boundary.

# Prescribed Displacement/Rotation I

- I In the Physics toolbar, click Figure Edges and choose Prescribed Displacement/Rotation.
- 2 In the Settings window for Prescribed Displacement/Rotation, locate the Edge Selection section.
- 3 From the Selection list, choose Constrained Edges.
- 4 Locate the Coordinate System Selection section. From the Coordinate system list, choose Cylindrical System 2 (sys2).
- 5 Locate the Prescribed Displacement section. From the Displacement in phi direction list, choose Prescribed.
- 6 From the Displacement in a direction list, choose Prescribed.

# Edge load, Tension

- I In the Physics toolbar, click Edges and choose Edge Load.
- 2 In the Settings window for Edge Load, type Edge load, Tension in the Label text field.
- 3 Locate the Edge Selection section. From the Selection list, choose Loaded Edges.
- 4 Locate the Force section. From the Load type list, choose Force per unit area.
- **5** Specify the  $\mathbf{F}_{\mathsf{A}}$  vector as

nominal_stress	х
0	у
0	z

Note that even though the shell interface is active on the inner boundary of the pipe, the load is considered to be applied on the true midsurface as given by the offset.

6 In the Physics toolbar, click Load Group and choose Load Group Mx.

### Edge load, Torsion

- I In the Physics toolbar, click Edges and choose Edge Load.
- 2 In the Settings window for Edge Load, type Edge load, Torsion in the Label text field.
- 3 Locate the Edge Selection section. From the Selection list, choose Loaded Edges.
- 4 Locate the Coordinate System Selection section. From the Coordinate system list, choose Cylindrical System 2 (sys2).
- 5 Locate the Force section. From the Load type list, choose Force per unit area.

# **6** Specify the $\mathbf{F}_A$ vector as

0	r
nominal_stress/sqrt(3)	phi
0	a

7 In the Physics toolbar, click Load Group and choose Load Group My.

Face load, Pressure

- I In the Physics toolbar, click **Boundaries** and choose Face Load.
- 2 In the Settings window for Face Load, type Face load, Pressure in the Label text field.
- 3 Locate the Boundary Selection section. From the Selection list, choose Shell.
- 4 Locate the Through-Thickness Location section. From the list, choose Top surface.
- 5 Locate the Force section. From the Load type list, choose Pressure.
- **6** In the p text field, type p int.

The pressure on the shell is applied on the inner boundary, that is top surface of the shell as the shell normal points inward which must always be checked when applying pressure loads.

7 In the Physics toolbar, click Load Group and choose Load Group Mz.

Edge load, External Load from Pressure

- I In the Physics toolbar, click Figure Edges and choose Edge Load.
- 2 In the Settings window for Edge Load, type Edge load, External Load from Pressure in the Label text field.
- 3 Locate the Edge Selection section. From the Selection list, choose Shell Hole Edges.
- 4 Locate the Force section. From the Load type list, choose Total force.
- **5** Specify the  $\mathbf{F}_{tot}$  vector as

0	x
2*f_extp	у
0	z

6 In the Physics toolbar, click Load Group and choose Load Group Mz.

# SOLID MECHANICS (SOLID)

- I In the Model Builder window, under Component I (compl) click Solid Mechanics (solid).
- 2 In the Settings window for Solid Mechanics, locate the Domain Selection section.

**3** From the **Selection** list, choose **Solid**.

Face load. Pressure

- I In the Physics toolbar, click **Boundaries** and choose **Boundary Load**.
- 2 In the Settings window for Boundary Load, type Face load, Pressure in the Label text field.
- 3 Locate the Boundary Selection section. From the Selection list, choose Solid Inside.
- 4 Locate the Force section. From the Load type list, choose Pressure.
- **5** In the *p* text field, type p int.
- 6 In the Physics toolbar, click Load Group and choose Load Group Mz.

Face load, External Load from Pressure

- I In the Physics toolbar, click **Boundaries** and choose **Boundary Load**.
- 2 In the Settings window for Boundary Load, type Face load, External Load from Pressure in the Label text field.
- 3 Locate the Boundary Selection section. From the Selection list, choose Solid Hole.
- 4 Locate the Force section. From the Load type list, choose Total force.
- **5** Specify the  $\mathbf{F}_{\text{tot}}$  vector as

0	x
f_extp	у
0	z

6 In the Physics toolbar, click Load Group and choose Load Group Mz.

### MULTIPHYSICS

Solid-Thin Structure Connection 1 (sshc1)

- I In the Physics toolbar, click Multiphysics Couplings and choose Global>Solid-Thin Structure Connection.
- 2 In the Settings window for Solid-Thin Structure Connection, locate the **Connection Settings** section.
- 3 Select the Manual control of selections check box.
- 4 Locate the Boundary Selection, Solid section. From the Selection list, choose **Boundaries for Rigid Connection.**
- 5 Locate the Edge Selection, Shell section. From the Selection list, choose **Edges for Rigid Connection.**

Solid-Thin Structure Connection 2 (sshc2)

- I In the Physics toolbar, click Multiphysics Couplings and choose Global>Solid-Thin Structure Connection.
- 2 In the Settings window for Solid-Thin Structure Connection, locate the Connection Settings section.
- 3 Select the Manual control of selections check box.
- 4 From the Method list, choose Flexible.
- 5 Locate the Boundary Selection, Solid section. From the Selection list, choose **Boundaries for Flexible Connection.**
- 6 Locate the Edge Selection, Shell section. From the Selection list, choose **Edges for Flexible Connection.**

### MESH I

# Mapped I

In the Mesh toolbar, click More Generators and choose Mapped.

### Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- **3** Click the **Custom** button.
- 4 Locate the Element Size Parameters section. In the Maximum element size text field, type 0.2.
- 5 In the Minimum element size text field, type 0.01.

### Mapped I

- I In the Model Builder window, click Mapped I.
- **2** Select Boundaries 7, 11, 14, 32, 35, 53, and 56 only.
- 3 In the Settings window for Mapped, click Build Selected.

### Distribution I

- I Right-click Mapped I and choose Distribution.
- 2 Select Edge 34 only.
- 3 In the Settings window for Distribution, locate the Distribution section.
- 4 From the Distribution type list, choose Predefined.
- 5 In the Number of elements text field, type 16.
- 6 In the Element ratio text field, type 4.

7 Right-click Distribution I and choose Duplicate.

### Distribution 2

- I In the Model Builder window, click Distribution 2.
- 2 In the Settings window for Distribution, locate the Edge Selection section.
- 3 Click Clear Selection.
- **4** Select Edge 7 only.
- **5** Locate the **Distribution** section. Select the **Reverse direction** check box.

In the Model Builder window, right-click Mapped I and choose Build Selected.

### Coby Face 1

- I In the Mesh toolbar, click Copy and choose Copy Face.
- **2** Select Boundary 7 only.
- 3 In the Settings window for Copy Face, locate the Destination Boundaries section.
- **4** Click to select the **Activate Selection** toggle button.
- **5** Select Boundaries 6, 27, 28, 48, and 49 only.
- 6 Click Pauld Selected.

### Swebt I

- I In the Mesh toolbar, click A Swept.
- 2 In the Settings window for Swept, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- 4 From the Selection list, choose Solid.

### Distribution I

- I Right-click Swept I and choose Distribution.
- 2 In the Settings window for Distribution, locate the Distribution section.
- 3 In the Number of elements text field, type 3.
- 4 Click Build Selected.

### STUDY I

### Steb 1: Stationary

- I In the Model Builder window, under Study I click Step I: Stationary.
- 2 In the Settings window for Stationary, click to expand the Study Extensions section.

- 3 Select the **Define load cases** check box.
- 4 Click + Add.
- **5** In the table, enter the following settings:

Load case	IgFx	Weight	lgMx	Weight	lgP	Weight
Tension	<b>V</b>	1.0		1.0		1.0

- 6 Click + Add.
- 7 In the table, enter the following settings:

Load case	lgFx	Weight	lgMx	Weight	lgP	Weight
Torsion		1.0	V	1.0		1.0

- 8 Click + Add.
- **9** In the table, enter the following settings:

Load case	IgFx	Weight	lg <b>M</b> x	Weight	IgP	Weight
Pressure		1.0		1.0	<b>V</b>	1.0

Solution I (soll)

- I In the Study toolbar, click Show Default Solver.
- 2 In the Model Builder window, expand the Solution I (soll) node.
- 3 Right-click Solution I (soll) and choose Compute.

### RESULTS

Stress, Solid and Shell

I In the Settings window for 3D Plot Group, type Stress, Solid and Shell in the Label text field.

Place the shell and solid results in the same plot, and examine the three load cases.

### Filter I

- I In the Model Builder window, expand the Stress, Solid and Shell node.
- 2 Right-click Surface I and choose Filter.
- 3 In the Settings window for Filter, locate the Element Selection section.
- 4 In the Logical expression for inclusion text field, type Z<-10\*eps.

### Surface I

I In the Model Builder window, click Surface I.

- 2 In the Settings window for Surface, locate the Expression section.
- 3 From the Unit list, choose MPa.
- 4 Click to expand the Range section. Select the Manual color range check box.
- 5 In the Maximum text field, type 15.

### Volume 1

- I In the Model Builder window, right-click Stress, Solid and Shell and choose Volume.
- 2 In the Settings window for Volume, locate the Data section.
- 3 From the Dataset list, choose Study I/Solution I (soll).
- 4 From the Solution parameters list, choose From parent.
- **5** Locate the **Expression** section. In the **Expression** text field, type solid.mises.
- 6 From the Unit list, choose MPa.

# Deformation I

Right-click Volume I and choose Deformation.

### Selection 1

- I In the Model Builder window, right-click Volume I and choose Selection.
- 2 Select Domains 5 and 7 only.

### Volume 1

- I In the Model Builder window, click Volume I.
- 2 In the Settings window for Volume, click to expand the Inherit Style section.
- 3 From the Plot list, choose Surface 1.
- 4 In the Stress, Solid and Shell toolbar, click Plot.

### Stress, Solid and Shell

- I In the Model Builder window, click Stress, Solid and Shell.
- 2 In the Settings window for 3D Plot Group, locate the Data section.
- 3 From the Load case list, choose Torsion.
- 4 In the Stress, Solid and Shell toolbar, click Plot.
- 5 From the Load case list, choose Tension.
- 6 In the Stress, Solid and Shell toolbar, click  **Plot**.

### Stress along Top, Tension

I In the Home toolbar, click **Add Plot Group** and choose ID Plot Group.

- 2 In the Settings window for ID Plot Group, type Stress along Top, Tension in the **Label** text field.
- 3 Locate the Data section. From the Parameter selection (Load case) list, choose From list.
- 4 In the Load cases list, select Tension.
- **5** Click to expand the **Title** section. From the **Title type** list, choose **None**.

### Shell, Rigid

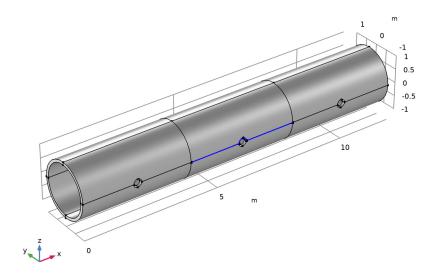
- I Right-click Stress along Top, Tension and choose Line Graph.
- 2 In the Settings window for Line Graph, type Shell, Rigid in the Label text field.
- **3** Select Edges 7, 25, 31, and 34 only.
- 4 Locate the y-Axis Data section. In the Expression text field, type shell.mises.
- 5 From the Unit list, choose MPa.
- **6** Find the **Parameters** subsection. In the table, enter the following settings:

Name	Value	Unit	Description
shell.z	- 1		Local z-coordinate [-I,I] for thickness-dependent results

- 7 Click to expand the Coloring and Style section. From the Width list, choose 2.
- 8 Find the Line markers subsection. From the Marker list, choose Cycle.
- **9** From the **Positioning** list, choose **Interpolated**.
- 10 In the Number text field, type 12.
- II Click to expand the Legends section. Select the Show legends check box.
- 12 Find the Include subsection. Clear the Solution check box.
- **I3** Select the **Label** check box.
- **14** Right-click **Shell, Rigid** and choose **Duplicate**.

- I In the Model Builder window, under Results>Stress along Top, Tension click Shell, Rigid I.
- 2 In the Settings window for Line Graph, type Solid in the Label text field.
- 3 Locate the Selection section. Click Clear Selection.

**4** Select Edges 38, 57, 63, and 67 only.



- 5 Locate the y-Axis Data section. In the Expression text field, type solid.mises.
- 6 Locate the Coloring and Style section. Find the Line markers subsection. In the Number text field, type 13.

# Shell, Rigid

In the Model Builder window, right-click Shell, Rigid and choose Duplicate.

### Shell, Flexible

- I In the Model Builder window, under Results>Stress along Top, Tension click Shell, Rigid I.
- 2 In the Settings window for Line Graph, type Shell, Flexible in the Label text field.
- 3 Locate the Selection section. Click Clear Selection.
- 4 Select Edges 75, 93, 99, and 102 only.
- 5 Locate the Coloring and Style section. Find the Line markers subsection. In the Number text field, type 14.
- 6 In the Stress along Top, Tension toolbar, click Plot.

# Stress along Top, Tension

In the Model Builder window, right-click Stress along Top, Tension and choose Duplicate.

Stress along Top, Torsion

- I In the Model Builder window, under Results click Stress along Top, Tension I.
- 2 In the Settings window for ID Plot Group, type Stress along Top, Torsion in the **Label** text field.
- 3 Locate the Data section. In the Load cases list, select Torsion.
- 4 In the Stress along Top, Torsion toolbar, click  **Plot**.
- 5 Right-click Stress along Top, Torsion and choose Duplicate.

### Stress along Top, Pressure

- I In the Model Builder window, under Results click Stress along Top, Torsion I.
- 2 In the Settings window for ID Plot Group, type Stress along Top, Pressure in the Label text field.
- 3 Locate the Data section. In the Load cases list, select Pressure.
- 4 In the Stress along Top, Pressure toolbar, click Plot.

## Axial Stress through Thickness, Tension

- I In the Home toolbar, click **Add Plot Group** and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Axial Stress through Thickness, Tension in the Label text field.
- 3 Locate the Data section. From the Parameter selection (Load case) list, choose From list.
- 4 In the Load cases list, select Tension.
- **5** Locate the **Title** section. From the **Title type** list, choose **None**.
- 6 Locate the Plot Settings section.
- 7 Select the y-axis label check box. In the associated text field, type Stress (MPa).
- 8 Locate the Legend section. From the Position list, choose Upper left.

### Rigid

- I Right-click Axial Stress through Thickness, Tension and choose Line Graph.
- **2** Select Edge 47 only.
- 3 In the Settings window for Line Graph, type Rigid in the Label text field.
- 4 Locate the y-Axis Data section. In the Expression text field, type solid.sx.
- 5 From the Unit list, choose MPa.
- 6 Locate the x-Axis Data section. From the Parameter list, choose Reversed arc length.
- 7 Locate the Coloring and Style section. From the Width list, choose 2.
- 8 Find the Line markers subsection. From the Marker list, choose Cycle.

- **9** From the **Positioning** list, choose **Interpolated**.
- 10 Locate the Legends section. Select the Show legends check box.
- II Find the Include subsection. Clear the Solution check box.
- 12 Select the Label check box.
- 13 Right-click Rigid and choose Duplicate.

### Flexible

- I In the Model Builder window, under Results>Axial Stress through Thickness, Tension click Rigid I.
- 2 In the Settings window for Line Graph, type Flexible in the Label text field.
- 3 Locate the Selection section. Click Clear Selection.
- **4** Select Edge 81 only.
- **5** Right-click **Flexible** and choose **Duplicate**.

### Nominal

- I In the Model Builder window, under Results>Axial Stress through Thickness, Tension click Flexible 1.
- 2 In the Settings window for Line Graph, type Nominal in the Label text field.
- 3 Locate the y-Axis Data section. In the Expression text field, type 10.

### Axial Stress through Thickness, Tension

In the Model Builder window, right-click Axial Stress through Thickness, Tension and choose Duplicate.

Transverse Stress through Thickness, Tension

- I In the Model Builder window, under Results click Axial Stress through Thickness, Tension I.
- 2 In the Settings window for ID Plot Group, type Transverse Stress through Thickness, Tension in the Label text field.

### Rigid

- I In the Model Builder window, expand the Transverse Stress through Thickness, Tension node, then click Rigid.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type solid.sz.

### Flexible

- I In the Model Builder window, click Flexible.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type solid.sz.

### Nominal

- I In the Model Builder window, click Nominal.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type 0.
- 4 In the Transverse Stress through Thickness, Tension toolbar, click Plot.

### Axial Stress through Thickness, Tension

In the Model Builder window, under Results right-click Axial Stress through Thickness, **Tension** and choose **Duplicate**.

### Equivalent Stress through Thickness, Torsion

- I In the Model Builder window, under Results click Axial Stress through Thickness, Tension I.
- 2 In the Settings window for ID Plot Group, type Equivalent Stress through Thickness, Torsion in the Label text field.
- 3 Locate the Data section. In the Load cases list, select Torsion.

### Rigid

- I In the Model Builder window, expand the Equivalent Stress through Thickness, Torsion node, then click Rigid.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type solid.mises.

### Flexible

- I In the Model Builder window, click Flexible.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type solid.mises.

### Nominal

- I In the Model Builder window, click Nominal.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type nominal\_stress\*sys2.r/r\_mid.
- 4 From the Unit list, choose MPa.

5 In the Equivalent Stress through Thickness, Torsion toolbar, click Plot.

Axial Stress through Thickness, Tension

In the Model Builder window, under Results right-click Axial Stress through Thickness, **Tension** and choose **Duplicate**.

Hoop Stress through Thickness, Pressure

- I In the Model Builder window, under Results click Axial Stress through Thickness, Tension I.
- 2 In the Settings window for ID Plot Group, type Hoop Stress through Thickness, Pressure in the Label text field.
- 3 Locate the Data section. In the Load cases list, select Pressure.

### Rigid

- I In the Model Builder window, expand the Hoop Stress through Thickness, Pressure node, then click Rigid.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type solid.sy.

### Flexible

- I In the Model Builder window, click Flexible.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type solid.sy.

### Nominal

- I In the Model Builder window, click Nominal.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type p int\*r inner^2/(r outer^2-r inner^2)\*(1+ r\_outer^2/sys2.r^2).
- 4 From the Unit list, choose MPa.
- 5 In the Hoop Stress through Thickness, Pressure toolbar, click Plot.