

Wave-Based Time-Domain Room Acoustics with Frequency-Dependent Impedance

The use of wave-based techniques for room acoustic simulations has spread in the last years due to the increase in computational performance as well as the development of new numerical methods. The challenge of including realistic impedance conditions at walls is traditionally solved in the frequency domain. Resent research has focused on the implementation of frequency-dependent impedance in the time-domain models using partial fraction representation of the frequency-dependent data (see Ref. 1, Ref. 2, Ref. 3).

This tutorial shows how to get partial fraction representation of frequency-dependent wall impedance data via the Partial Fraction Fit function and how to use the results to set up the built-in impedance boundary conditions for room acoustic response simulation in the time domain. The model uses the Pressure Acoustics, Time Explicit interface to simulate the propagation of sound. The physics interface is based on the discontinuous Galerkin (dG-FEM) method which uses a matrix free approach and a time explicit solver. The method is very memory efficient and well suited for distributed computing on a cluster architecture.

Model Definition

The model studies the response of a small 50.5 m³ room shown in Figure 1 on the left. The loudspeaker to the left of the TV emits an acoustic signal given as a sine wave modulated by a Gaussian envelope at the center frequency $f_0 = 700$ Hz. As the signal propagates from the speaker, the acoustic pressure is measured at four listening points located equidistantly on a line drawn from the speaker to the sofa, as shown in Figure 1 on the right.

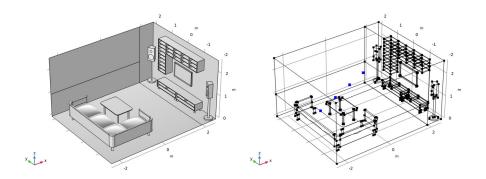


Figure 1: Room geometry (left) and location of listening points (right).

The sound absorption properties of the carpet, ceiling, sofa, and walls are modeled using the simplified local reacting approximation which states that the response at a certain point of the surface depends on the sound pressure at that point. This behavior is described by an impedance boundary condition imposed on the surface. When not purely resistive, as, for example, for porous materials, the boundary impedance will depend on the frequency, resulting in the following boundary condition in the frequency domain

$$u_n(\omega) = \frac{p(\omega)}{Z(\omega)} = Y(\omega)p(\omega),$$
 (1)

where Z is the frequency-dependent specific impedance and Y is the admittance. The time-domain equivalent of Equation 1 involves a convolution instead of the multiplication

$$u_n(t) = \int_{-\infty}^{t} Y(t - \tau) p(\tau) d\tau$$
 (2)

and thus requires the knowledge of Y(t) in the time domain and thus the computation of the inverse Fourier (or Laplace) transform, which, obviously, cannot be done analytically for a general $Y(\omega)$ obtained from various poroacoustic models or measurements. Moreover, it is not enough to know the values of the impedance (admittance) for a number of frequencies or for a frequency range to translate the boundary condition from the frequency to the time domain. Therefore, $Y(\omega)$ has to be extended to the whole complex plane.

An approximate analytical representation of Y(t) can be retrieved from a rational approximation of $Y(\omega)$ that is defined on the whole complex plane:

$$Y(\omega) \approx \frac{a_0 + a_1(i\omega) + \dots + a_N(i\omega)^N}{1 + b_1(i\omega) + \dots + b_N(i\omega)^N} = Y_{\infty} + \sum_{k=1}^{N} \frac{A_k}{i\omega - \alpha_k}.$$
 (3)

Indeed, each fraction term on the right-hand side of Equation 3 corresponds to an exponential decay in the time domain

$$L^{-1}\left(\frac{A_k}{s - \alpha_k}\right) = A_k e^{\alpha_k t} H(t), \qquad (4)$$

where L^{-1} is the inverse Laplace transform and H(t) is the Heaviside step function.

However, the approximation given by Equation 3 has to fulfill three conditions in order for the time-domain boundary condition to be physical:

• Causality, $Y(\omega)$ is analytic and nonzero in $\Im(\omega) > 0$;

- Reality, $\overline{Y}(\omega) = Y(-\omega)$, that is, $\Re(Y(\omega))$ is even and $\Im(Y(\omega))$ is odd;
- Passivity, $\Re(Y(\omega)) > 0$ for all real ω .

The causality and reality conditions are fulfilled if Y_{∞} is real; the residues, A_k , and poles, α_k , are either real or come in complex-conjugate pairs; and $\Re(\alpha_k) < 0$ (for the exponentials in Equation 4 to decay as the time increases). That is

$$Y(\omega) \approx Y_{\infty} + \sum_{k=1}^{N_R} \frac{R_k}{i\omega - \xi_k} + \frac{1}{2} \sum_{k=1}^{N_C} \left[\frac{Q_k}{i\omega - \zeta_k} + \frac{\overline{Q_k}}{i\omega - \overline{\zeta_k}} \right], \tag{5}$$

where N_R and N_C are the numbers of pure real poles and complex-conjugate pole pairs, respectively.

The form given in Equation 5 is used to reduce the evaluation of the integral in Equation 2 to system of auxiliary ordinary differential equations (ODEs) for memory variables. This approach is referred to as ADE method (see Ref. 1, Ref. 2, Ref. 3). The system of ODEs is automatically created and solved when you set up an Impedance boundary condition with a General local reacting (rational approximation) option.

Results and Discussion

The real and imaginary parts of the fitted frequency dependent admittance data are depicted in Figure 2. As seen, the sofa surface is more absorptive that the others, especially at the higher frequencies.

Figure 3 shows the acoustic pressure recorded at the listening points in blue, green, red, and cyan colors as the point moves away from the source. The values are normalized to the maximum pressure at the source. As seen from the plot, the pressure magnitude decreases with the time.

The history of the signal propagation through the room is shown in Figure 4. The signal emitted from the loudspeaker reaches listening point 1 at $t = 4T_0$ ($T_0 = 1/f_0$) and listening point 4 at $t = 11T_0$. Then multiple reflections occur while the signal magnitude goes down at $t = 18T_0$ and $t = 25T_0$.

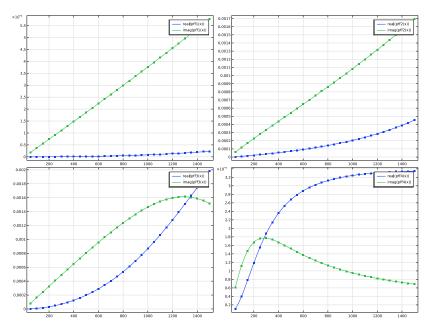


Figure 2: Original admittance data and partial fraction expansions for the carpet, ceiling, sofa, and wall (from the top, left to right).

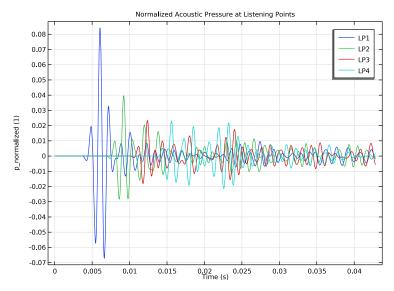


Figure 3: Acoustic pressure at listening points.

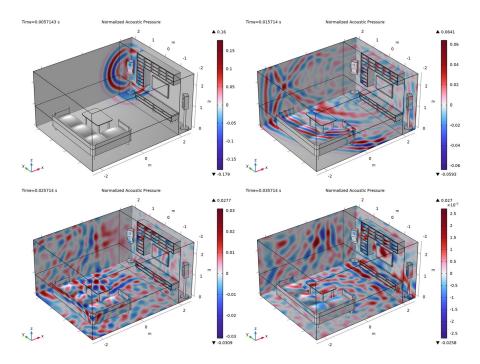


Figure 4: Normalized acoustic pressure at $t=4T_0,11T_0,18T_0$, and $25T_0$ (from the top, left to right).

Notes About the COMSOL Implementation

PARTIAL FRACTION FIT OF ADMITTANCE DATA

The partial fraction approximation (expansion) is obtained for the carpet, ceiling, sofa, and walls admittance through the built-in *Partial Fraction Fit* function within the frequency range from 50 Hz to 1.5 kHz. The form given by Equation 5 ensures that the reality condition is fulfilled. This is not always the case for the causality and passivity conditions. However, the *Partial Fraction Fit* function has the necessary tools that can be used to make the result fulfill the causality and reality conditions, thus making the time-domain impedance boundary condition physical.

The *Partial Fraction Fit* function fits the input data within a given tolerance that is found in the **Advanced** section (default 10^{-3}). A higher tolerance error results in a higher degree, N, of the polynomials in Equation 3 and therefore a larger number of terms in the expansion Equation 5. The number of poles/residues in the expansion in turns

corresponds to the number of ODEs solved for the memory variables. The default tolerance provides a balance between the approximation accuracy and the number of terms in the expansion, thus the computation costs when the expansion is used in an impedance boundary condition. On the other hand, increasing the tolerance may yield better results.

For example, the first *Partial Fraction Fit* function present in the model fits the admittance of the carpet. The default tolerance results in one real-valued pole and one complex-valued pole pair. From a distance the result looks good, but a closer look reveals that the passivity condition is violated at low frequencies: the real part of Y becomes negative between 50 and 90 Hz. A tighter tolerance of 10⁻⁵ yields two extra terms with real-valued poles and a partial fraction expansion that is passive within the given frequency range (see Figure 5).

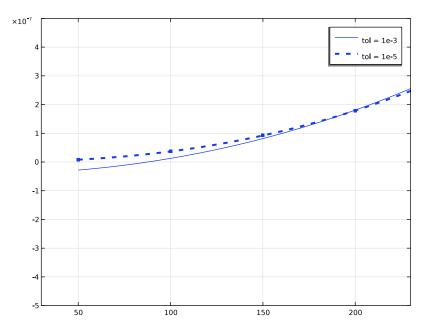


Figure 5: Real part of the carpet admittance fit obtained with tolerance of 10^{-3} and 10^{-5} .

The violation of the causality condition takes place when one or more poles in the partial fraction expansion have positive real part (that is, they are unstable), which can usually happen to the pure real poles. Click the Flip Poles button to flip the unstable poles to the left half plane and circumvent the issue. The residues will be recomputed and updated automatically. This procedure is carried out when fitting the ceiling admittance data with the second Partial Fraction Fit function.

A different situation appears with the third Partial Fraction Fit function that is used to build a partial fraction expansion for the sofa admittance. Fitting the data with the default tolerance yields an approximation that fulfills all three conditions. However, the absolute value of the real-valued pole is much larger than that of the complex-valued pole. This results in a stiff ODE, which affects the stability of the time-integration scheme. Indeed, the Pressure Acoustics, Time Explicit physics interface used in this tutorial relies on explicit time-integration schemes. The time step is deduced solely from the minimum mesh element size (see the section below) and the speed of sound, and it can be too large to achieve a stable solution of the ODE.

The influence of such poles is often localized and therefore they can be discarded with the following update of the residues by clicking the **Update Residues** button. For the default tolerance, the approximation becomes worse after discarding the real-valued pole (see Figure 6 on the left). Increasing the tolerance results in an extra complex-valued pole pair, and the result visually remains the same after the real-valued pole has been removed.

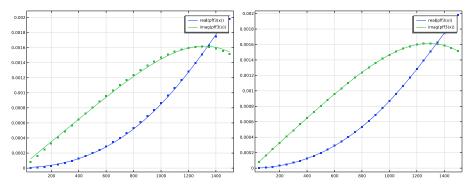


Figure 6: Sofa admittance approximation computed with the tolerance of 10⁻³ (left) and 10^{-5} (right) after discarding the real-valued pole.

The same procedure should be applied when the expansion contains spurious poles (also known as Froissart doublets). Those are the poles with residues close to zero, which is equivalent to very near pole and zero pairs in the rational approximation given by Equation 3.

MESH AND TIME EXPLICIT SOLVER

Solving wave propagation problems in the time domain has some requirements on both spatial and temporal resolution of the wave pattern. The mesh has to be fine enough to resolve the frequency content of the signal. For the quartic discretization used by default in the Nonlinear Pressure Acoustic, Time Explicit physics interface, the proper accuracy is achieved when the maximum mesh element size does not exceed 2/3 of the minimum wavelength. This tutorial studies the propagation of a broadband signal shown together with its frequency content in Figure 7. As seen, the frequency content is not limited by $f_0 = 700$ Hz. Therefore, the mesh should resolve smaller wavelengths to achieve accurate results; at least up to $\lambda_0/2 = c_0/(2f_0)$. This yields $h_{\text{max}} \le \lambda_0/3$.

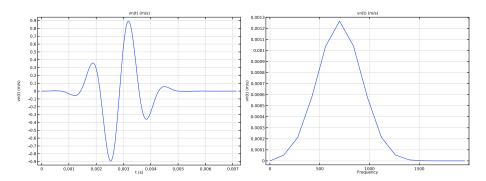


Figure 7: Source signal (left) and its frequency content (right).

The Pressure Acoustic, Time Explicit interface is based on dG-FEM and uses an explicit time integration schemes. The time step is supposed to obey the CFL condition to ensure the stability of the time integration method. That is, $\Delta t \leq h_{\min}/c_0$, where h_{\min} is the minimum mesh element size. The solver automatically deduces the time step from the mesh and the speed of sound.

References

- 1. H. Wang, M. Cosnefroy, and M. Hornikx, "An arbitrary high-order discontinuous Galerkin method with local time-stepping for linear acoustic wave propagation," J. Acoust. Soc. Am., vol. 149, p. 569, 2021; https://doi.org/10.1121/10.0003340.
- 2. F. Pind, A.P. Eising-Karup, C-H Jeong, J.S. Hesthaven, and J. Strømann-Andersen, "Time-domain room acoustic simulations with extended-reacting porous absorbers using the discontinuous Galerkin method," J. Acoust. Soc. Am., vol. 148, p. 2851, 2020; https://doi.org/10.1121/10.0002448.
- 3. H. Wang and M. Hornikx, "Time-domain impedance boundary condition modeling with the discontinuous Galerkin method forroom acoustics simulations," J. Acoust. Soc. Am., vol. 147, p. 2534, 2020; https://doi.org/10.1121/10.0001128.

Application Library path: Acoustics Module/Building and Room Acoustics/ wave_based_room

Modeling Instructions

From the File menu, choose New.

In the New window, click Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click **3D**.
- 2 In the Select Physics tree, select Acoustics>Pressure Acoustics, Time Explicit (pate).
- 3 Click Add.
- 4 Click Study.
- 5 In the Select Study tree, select General Studies>Time Dependent.
- 6 Click M Done.

GEOMETRY I

- I In the Model Builder window, under Component I (compl) click Geometry I.
- 2 In the Settings window for Geometry, locate the Advanced section.
- 3 From the Geometry representation list, choose CAD kernel.

Import I (impl)

- I In the **Home** toolbar, click **Import**.
- 2 In the Settings window for Import, locate the Import section.
- 3 Click **Browse**.
- 4 Browse to the model's Application Libraries folder and double-click the file wave_based_room.mphbin.
- 5 Click Import.
- 6 Click the Wireframe Rendering button in the Graphics toolbar.

Point I (btl)

- I In the Geometry toolbar, click \bigcirc More Primitives and choose Point.
- 2 In the Settings window for Point, locate the Point section.
- 3 In the x text field, type 1.2[m] 0.2[m] -0.8[m] -1.8[m].
- 4 In the y text field, type 0.75*1.75[m] 0.5*1.75[m] 0.25*1.75[m] 0[m].
- 5 In the z text field, type 1[m] 1[m] 1[m] 1[m].
- 6 Click Pauld Selected.

DEFINITIONS

Sofa

- I In the **Definitions** toolbar, click **\(\frac{1}{2} \) Explicit**.
- 2 In the Settings window for Explicit, type Sofa in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- 5 In the Paste Selection dialog box, type 10-20, 25-31, 51, 52, 61, 68-71 in the Selection text field.
- 6 Click OK.

Sofa Legs

- 2 In the Settings window for Explicit, type Sofa Legs in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- 5 In the Paste Selection dialog box, type 21-24, 32-35, 53-56 in the Selection text field.
- 6 Click OK.

Shelves

- I In the **Definitions** toolbar, click **\(\bigcap_{\text{a}} \) Explicit**.
- 2 In the Settings window for Explicit, type Shelves in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- **5** In the **Paste Selection** dialog box, type 116-206, 265-270, 272-274, 277-279, 281-289 in the **Selection** text field.
- 6 Click OK.

TV

- I In the **Definitions** toolbar, click 🔓 **Explicit**.
- 2 In the Settings window for Explicit, type TV in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- 5 In the Paste Selection dialog box, type 241-261 in the Selection text field.
- 6 Click OK.

TV Table

- I In the **Definitions** toolbar, click **\(\frac{1}{2} \) Explicit**.
- 2 In the Settings window for Explicit, type TV Table in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- 5 In the Paste Selection dialog box, type 79-115, 233-240, 263, 264, 271, 275, 276, 280 in the Selection text field.
- 6 Click OK.

Sofa Table

- I In the **Definitions** toolbar, click **\(\bigcap_{\text{a}} \) Explicit**.
- 2 In the Settings window for Explicit, type Sofa Table in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- 5 In the Paste Selection dialog box, type 36-50, 57-60, 62-67, 72 in the Selection text field.
- 6 Click OK.

Speaker Legs

- I In the **Definitions** toolbar, click 🔓 **Explicit**.
- 2 In the Settings window for Explicit, type Speaker Legs in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- 5 In the Paste Selection dialog box, type 207-212, 225-228 in the Selection text field.
- 6 Click OK.

Ceiling

I In the **Definitions** toolbar, click **\(\bigcap_{\bigcap} \) Explicit**.

- 2 In the Settings window for Explicit, type Ceiling in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- 5 In the Paste Selection dialog box, type 7, 77 in the Selection text field.
- 6 Click OK.

Carbet

- I In the **Definitions** toolbar, click **\(\frac{1}{2} \) Explicit**.
- 2 In the Settings window for Explicit, type Carpet in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- 5 In the Paste Selection dialog box, type 3, 75 in the Selection text field.
- 6 Click OK.

Wall

- I In the **Definitions** toolbar, click **\(\frac{1}{2} \) Explicit**.
- 2 In the Settings window for Explicit, type Wall in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Click Paste Selection.
- 5 In the Paste Selection dialog box, type 1, 2, 4, 5, 8, 9, 74, 78, 262 in the Selection text field.
- 6 Click OK.

All Surfaces

- 2 In the Settings window for Explicit, type All Surfaces in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Select the All boundaries check box.

Listening Points

- I In the **Definitions** toolbar, click **\(\frac{1}{2} \) Explicit**.
- 2 In the Settings window for Explicit, locate the Input Entities section.
- 3 From the Geometric entity level list, choose Point.
- 4 Select Points 35, 53, 121, and 122 only.
- 5 In the Label text field, type Listening Points.

Create four **Point Probes** for calculating the acoustic pressure on the way from the loudspeaker to the sofa.

Point Probe I (point I)

- I In the **Definitions** toolbar, click Probes and choose **Point Probe**.
- 2 In the Settings window for Point Probe, locate the Source Selection section.
- 3 Click Clear Selection.
- 4 Select Point 122 only.
- 5 Locate the Expression section. In the Expression text field, type pate.p t/(1[m/s]* pate.Z).
- 6 Select the **Description** check box. In the associated text field, type LP1.
- 7 Right-click Point Probe I (point I) and choose Duplicate.

Point Probe 2 (point2)

- I In the Model Builder window, click Point Probe 2 (point2).
- 2 In the Settings window for Point Probe, locate the Source Selection section.
- 3 Click Clear Selection.
- 4 Select Point 121 only.
- **5** Locate the **Expression** section. In the **Description** text field, type LP2.
- 6 Right-click Point Probe 2 (point2) and choose Duplicate.

Point Probe 3 (point3)

- I In the Model Builder window, click Point Probe 3 (point3).
- 2 In the Settings window for Point Probe, locate the Source Selection section.
- 3 Click Clear Selection.
- 4 Select Point 53 only.
- **5** Locate the **Expression** section. In the **Description** text field, type LP3.
- 6 Right-click Point Probe 3 (point3) and choose Duplicate.

Point Probe 4 (boint4)

- I In the Model Builder window, click Point Probe 4 (point4).
- 2 In the Settings window for Point Probe, locate the Source Selection section.
- 3 Click Clear Selection.
- **4** Select Point 35 only.
- **5** Locate the **Expression** section. In the **Description** text field, type LP4.

GLOBAL DEFINITIONS

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- **3** In the table, enter the following settings:

Name	Expression	Value	Description
f0	700[Hz]	700 Hz	Signal center frequency
ТО	1/f0	0.0014286 s	Signal period at center frequency
c0	343[m/s]	343 m/s	Speed of sound
lam0	c0/f0	0.49 m	Signal wavelength at center frequency

Analytic I (an I)

- I In the **Definitions** toolbar, click $\stackrel{f_{\infty}}{Q}$ **Analytic**.
- 2 In the Settings window for Analytic, type vn in the Function name text field.
- 3 Locate the **Definition** section. In the **Arguments** text field, type t.
- 4 In the Expression text field, type $\exp(-(t 2*T0)^2/(T0^2/2))*\sin(2*pi*f0*t)$.
- **5** Locate the **Units** section. In the **Function** text field, type m/s.
- **6** In the table, enter the following settings:

Argument	Unit
t	S

7 Locate the Plot Parameters section. In the table, enter the following settings:

Plot	Argument	Lower limit	Upper limit	Fixed value	Unit
$\sqrt{}$	t	0	5*T0	0	s

RESULTS

Source Signal Frequency Content

In the Settings window for ID Plot Group, type Source Signal Frequency Content in the Label text field.

Function I

- I In the Model Builder window, expand the Source Signal Frequency Content node, then click Function 1.
- 2 In the Settings window for Function, locate the Output section.
- 3 From the Display list, choose Discrete Fourier transform.
- 4 From the Show list, choose Frequency spectrum.
- 5 From the Scale list, choose Multiply by sampling period.
- 6 Select the Frequency range check box.
- 7 In the Maximum text field, type 3*f0.
- 8 In the Source Signal Frequency Content toolbar, click Plot.

GLOBAL DEFINITIONS

Partial Fraction Fit - Carpet

- I In the Home toolbar, click f(X) Functions and choose Global>Partial Fraction Fit.
- 2 In the Settings window for Partial Fraction Fit, type Partial Fraction Fit Carpet in the Label text field.
- 3 Locate the **Data** section. Click **Browse**.
- 4 Browse to the model's Application Libraries folder and double-click the file wave_based_room_admittance_carpet.txt.

Tighten the tolerance for the result to fulfill the passivity condition.

- 5 Click to expand the Advanced section. In the Tolerance text field, type 1e-5.
- 6 Click Fit Parameters.
- 7 Click Plot.

Partial Fraction Fit - Ceiling

- I In the Home toolbar, click f(X) Functions and choose Global>Partial Fraction Fit.
- 2 In the Settings window for Partial Fraction Fit, type Partial Fraction Fit Ceiling in the Label text field.
- 3 Locate the **Data** section. Click **Browse**.
- 4 Browse to the model's Application Libraries folder and double-click the file wave_based_room_admittance_ceiling.txt.
- 5 Click Fit Parameters.

One of the real-valued poles is unstable and should be flipped to the left half plane.

6 Click Flip Poles.

7 Click I Plot.

Partial Fraction Fit - Sofa

- I In the Home toolbar, click f(X) Functions and choose Global>Partial Fraction Fit.
- 2 In the Settings window for Partial Fraction Fit, type Partial Fraction Fit Sofa in the Label text field.
- 3 Locate the **Data** section. Click **Browse**.
- **4** Browse to the model's Application Libraries folder and double-click the file wave based room admittance sofa.txt.

Tune the tolerance and discard the real-valued pole as discussed earlier.

- 5 Locate the Advanced section. In the Tolerance text field, type 1e-5.
- 6 Click Fit Parameters.
- 7 Locate the Poles and Residues section. Find the Real residues and poles subsection. Click Clear Table or Delete the table row containing the real-valued residue and pole.
- 8 Click J Update Residues.
- 9 Click I Plot.

Partial Fraction Fit - Wall

- I In the Home toolbar, click f(X) Functions and choose Global>Partial Fraction Fit.
- 2 In the Settings window for Partial Fraction Fit, type Partial Fraction Fit Wallin the Label text field.
- 3 Locate the **Data** section. Click **Browse**.
- 4 Browse to the model's Application Libraries folder and double-click the file wave_based_room_admittance_wall.txt.
- 5 Click Fit Parameters.
- 6 Click om Plot.

ADD MATERIAL

- I In the Home toolbar, click **Add Material** to open the Add Material window.
- 2 Go to the Add Material window.
- 3 In the tree, select Built-in>Air.
- 4 Click Add to Component in the window toolbar.

MATERIALS

Air (mat1)

In the Home toolbar, click 4 Add Material to close the Add Material window.

PRESSURE ACOUSTICS, TIME EXPLICIT (PATE)

Normal Velocity I

- I In the Model Builder window, under Component I (compl) right-click Pressure Acoustics, Time Explicit (pate) and choose Normal Velocity.
- 2 Select Boundary 222 only.
- 3 In the Settings window for Normal Velocity, locate the Normal Velocity section.
- **4** In the v_n text field, type vn(t).

Impedance I - Carpet

- I In the Physics toolbar, click **Boundaries** and choose Impedance.
- 2 In the Settings window for Impedance, type Impedance 1 Carpet in the Label text field.
- 3 Locate the Boundary Selection section. From the Selection list, choose Carpet.
- 4 Locate the Impedance section. From the Impedance model list, choose General local reacting (rational approximation).
- 5 From the Partial fraction fit list, choose From function.
- 6 From the Reference list, choose Partial Fraction Fit Carpet (pff1).
- 7 Click Import.
- 8 Right-click Impedance I Carpet and choose Duplicate.

Impedance 2 - Ceiling

- I In the Model Builder window, under Component I (compl)>Pressure Acoustics, Time Explicit (pate) click Impedance I - Carpet I.
- 2 In the Settings window for Impedance, type Impedance 2 Ceiling in the Label text field.
- 3 Locate the Boundary Selection section. From the Selection list, choose Ceiling.
- 4 Locate the Impedance section. From the Reference list, choose Partial Fraction Fit -Ceiling (pff2).
- 5 Click Import.
- 6 Right-click Impedance 2 Ceiling and choose Duplicate.

Impedance 3 - Sofa

- I In the Model Builder window, under Component I (compl)>Pressure Acoustics, Time Explicit (pate) click Impedance 2 - Ceiling 1.
- 2 In the Settings window for Impedance, type Impedance 3 Sofa in the Label text field.
- 3 Locate the Boundary Selection section. From the Selection list, choose Sofa.
- 4 Locate the Impedance section. From the Reference list, choose Partial Fraction Fit -Sofa (pff3).
- 5 Click Import.
- 6 Right-click Impedance 3 Sofa and choose Duplicate.

Impedance 4 - Wall

- I In the Model Builder window, under Component I (compl)>Pressure Acoustics, Time Explicit (pate) click Impedance 3 - Sofa 1.
- 2 In the Settings window for Impedance, type Impedance 4 Wall in the Label text field.
- 3 Locate the Boundary Selection section. From the Selection list, choose Wall.
- 4 Locate the Impedance section. From the Reference list, choose Partial Fraction Fit -Wall (pff4).
- 5 Click Import.

MESH I

Swebt I

- I In the Mesh toolbar, click A Swept.
- 2 In the Settings window for Swept, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- 4 Select Domain 2 only.

Free Triangular 1

- I In the Mesh toolbar, click A More Generators and choose Free Triangular.
- 2 In the Settings window for Free Triangular, locate the Boundary Selection section.
- 3 From the Selection list, choose Sofa.

Free Tetrahedral I

- I In the Mesh toolbar, click A Free Tetrahedral.
- 2 In the Settings window for Free Tetrahedral, click to expand the **Element Quality Optimization** section.
- 3 From the Optimization level list, choose High.

4 Select the Avoid too small elements check box.

Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- **3** Click the **Custom** button.
- 4 Locate the Element Size Parameters section. In the Maximum element size text field, type lam0/3.
- 5 In the Minimum element size text field, type 0.04.
- 6 In the Curvature factor text field, type 0.3.
- 7 Click III Build All.

STUDY I

Step 1: Time Dependent

- I In the Model Builder window, expand the Study I node, then click Step 1: Time Dependent.
- 2 In the Settings window for Time Dependent, locate the Study Settings section.
- 3 In the Output times text field, type range (0, T0, 30*T0).
 - This setting specifies at which time steps the solution is saved and only influences the stored solution (and thus the file size). The internal time steps taken by the solver are automatically controlled by COMSOL to fulfill the appropriate CFL condition. Note that the acoustic pressure recorded at the probe points is computed with a higher resolution at the time steps taken by the solver.
- 4 Click to expand the **Store in Output** section. In the table, enter the following settings:

Interface	Output	Selection
Pressure Acoustics, Time Explicit (pate)	Selection	

- 5 Under Selections, click + Add.
- 6 In the Add dialog box, select All Surfaces in the Selections list.
- 7 Click OK.
- 8 In the Model Builder window, click Study I.
- 9 In the Settings window for Study, locate the Study Settings section.
- 10 Clear the Generate default plots check box.
- II In the Study toolbar, click $\underset{t=0}{\cup}$ Get Initial Value.

Increase the table size as there will be more than 10000 cells in the probe table.

RESULTS

In the Model Builder window, collapse the Results>Tables node.

Probe Table 1

- I In the Model Builder window, expand the Results>Tables node, then click Probe Table I.
- 2 In the Settings window for Table, locate the Storage section.
- 3 In the Maximum number of rows text field, type 20000.

STUDY I

Solution I (soll)

- I In the Model Builder window, expand the Solver Configurations node.
- 2 In the Model Builder window, expand the Solution I (soll) node.
- 3 In the Model Builder window, expand the Study I>Solver Configurations> Solution I (soll)>Time-Dependent Solver I node, then click Direct.
- 4 In the Settings window for Direct, locate the General section.
- 5 From the Solver list, choose PARDISO.
- 6 In the Study toolbar, click **Compute**.

All the plots are depicted in the previous sections of the documentation.

RESULTS

Normalized Acoustic Pressure at Listening Points

- I In the Model Builder window, expand the Results node, then click Probe Plot Group 2.
- 2 In the Settings window for ID Plot Group, type Normalized Acoustic Pressure at Listening Points in the Label text field.
- **3** Click to expand the **Title** section. From the **Title type** list, choose **Label**.
- 4 Locate the **Plot Settings** section.
- 5 Select the y-axis label check box. In the associated text field, type p normalized (1).

Probe Table Graph 1

- I In the Model Builder window, expand the Normalized Acoustic Pressure at Listening Points node, then click Probe Table Graph 1.
- 2 In the Settings window for Table Graph, click to expand the Legends section.
- 3 From the Legends list, choose Manual.

4 In the table, enter the following settings:

Legends	
LP1	
LP2	
LP3	
LP4	

5 In the Normalized Acoustic Pressure at Listening Points toolbar, click Plot.

Normalized Acoustic Pressure

- I In the Home toolbar, click **Add Plot Group** and choose **3D Plot Group**.
- 2 In the Settings window for 3D Plot Group, type Normalized Acoustic Pressure in the Label text field.
- 3 Locate the Data section. From the Time (s) list, choose 0.015714.
- 4 Click to expand the **Title** section. From the **Title type** list, choose **Label**.
- 5 Locate the Color Legend section. Select the Show maximum and minimum values check box.
- 6 Select the **Show units** check box.

Surface I

- I In the Normalized Acoustic Pressure toolbar, click Surface.
- 2 In the Settings window for Surface, locate the Expression section.
- 3 In the Expression text field, type pate.p_t/(1[m/s]*pate.Z).
- 4 Locate the Coloring and Style section. Click Change Color Table.
- 5 In the Color Table dialog box, select Wave>Wave in the tree.
- 6 Click OK.
- 7 In the Settings window for Surface, locate the Coloring and Style section.
- 8 From the Scale list, choose Linear symmetric.

Selection I

- I In the Normalized Acoustic Pressure toolbar, click \(\bigsigma_{\text{h}}\) Selection.
- 2 In the Settings window for Selection, locate the Selection section.
- 3 Click Paste Selection.
- 4 In the Paste Selection dialog box, type 3, 8-72, 75, 78-289 in the Selection text field.
- 5 Click OK.