

Thermal Stress in a Rotor due to Bearing Heat Loss

The lubricant in hydrodynamic bearings gets sheared due to the relative velocity between the surfaces of the journal and the bushing, which causes viscous heat dissipation. This heat is conducted to the neighboring journal and bearing housing. Surfaces of the rotor and the housing finally convects this heat to the atmosphere. In a steady state situations, the temperature profile in the system is such that the heat generated in the bearings is equal to the heat convected to the atmosphere. A varying temperature profile in the system causes a thermal deformation as well as thermal stresses.

In this model, a rotor-bearing system is considered to study the thermal equilibrium due to heat dissipation in bearings by performing a stationary analysis. The resulting stresses and deformations are also studied.

Model Definition

The model consists of a rotor supported by two hydrodynamic bearings. The housing of each bearing is also modeled. The rotor is supported by thrust bearings at its ends, and another thrust bearing at the collar on the rotor between both the two journal bearings. The geometry of the system is shown in Figure 1.

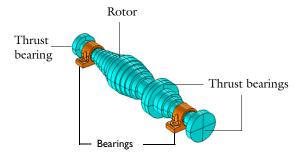


Figure 1: Rotor geometry.

Modeling of the problem requires coupling between the different physical phenomena and physics interfaces; these are listed in the following:

- Hydrodynamic Bearing: Models the pressure distribution in the lubrication film corresponding to the static weight on each bearing. Viscous heat dissipation due to shearing of the lubricant is also computed.
- Heat Transfer in Films: Models the heat transfer in the lubricant film. Heat is convected
 due to churning of the lubricant in the bearing. The convective velocity and heat
 dissipation are obtained from the Hydrodynamic Bearing interface. The top and
 bottom surfaces in the film are in contact with the journal and bushing, respectively. A
 continuity condition is used to connect the heat flow from film to the rotor and the
 housing.
- Heat Transfer in Solids: Models the heat transfer in the rotor and the bearing housings. Heat flowing in from the lubricant film is convected to the atmosphere from the surfaces of the rotor and the housings. The temperature profile in the steady state is obtained from the balance of heat flowing in and out of the solids.
- Solid Mechanics: Models the thermal expansion and stress in the rotor and the housings.
- Thermal Expansion Multiphysics Feature: Transfers the temperature from the Heat Transfer in Solids interface to Solid Mechanics to compute the thermal strain.

Note that the multiphysics interface **Thermal Stress, Solid** automatically adds the **Heat Transfer in Solids** and **Solid Mechanics** interfaces together with the **Thermal Expansion** multiphysics coupling. Therefore, the multiphysics interface can be used instead of adding them individually.

HEAT CONVECTION

The rotor and bearing housings both convect heat to the atmosphere. The housings are stationary and only natural convection takes place. A nominal value of $5~\mathrm{W/(m^2 \cdot K)}$ is used as a heat transfer coefficient. Heat convection from the rotor is more complicated due to the rotation of the rotor. The heat transfer coefficient for the external surfaces of the rotor is given by

$$h_{\rm shaft} = \frac{k_{\rm air} R_{e,\, \rm air}^{2/3} P_{\rm r,\, air}^{1/3}}{15 R}$$

where k_{air} is the thermal conductivity of the air, R is the radius of the rotor, $R_{e,air}$ is the Reynolds number of the rotor given by

$$R_{e, \, \text{air}} = \frac{\Omega D^2}{v_{\text{air}}}$$

where Ω is the angular speed of the rotor, D is the diameter of the rotor and v_{air} is the kinematic viscosity of the air. The Prandtl number, $P_{r,air}$, is given by

$$P_{r,\text{air}} = \frac{C_{p,\text{air}} \rho_{\text{air}} v_{\text{air}}}{k_{\text{air}}}$$

The properties of the bearing are given in Table 1, and the properties of air in Table 2.

TABLE I: BEARING PROPERTIES.

PARAMETER	VALUE
Journal diameter, $d_{ m J}$	0.05 m
Bushing length, $L_{ m b}$	0.04 m
Bearing clearance, ${\cal C}$	$0.001d_{ m J}$
Journal speed, N	3000 rpm
Static load, W	2000 N
Oil viscosity, $\mu_{\mbox{\scriptsize O}}$	0.0028 Pa.s
Oil density, $\rho_{\mbox{\scriptsize O}}$	866 kg/m ³
Oil heat conductivity, k_{O}	0.13 W/m.K
Oil heat capacity, $C_{p\mathrm{O}}$	2000 J/kg.K
Oil heat capacity ratio, $\gamma_{\mbox{\scriptsize O}}$	1.4

TABLE 2: AIR PROPERTIES.

PARAMETER	VALUE
Air density, ρ_{air}	1.225 kg/m ³
Air kinematic viscosity, v_{air}	17·10 ⁻⁶ m ² /s
Air thermal conductivity, $k_{ m air}$	0.028 W/m.K
Air heat capacity, $C_{p, { m air}}$	1.006 kJ/kg.K
Gas constant for air, $R_{ m d}$	287 J/kg.K

This model only considers a unidirectional coupling in which the viscous heat dissipation changes the temperature of the system, causing thermal deformation and stress. In general, viscous heat dissipation can depend on the temperature through the viscosity and density of the lubricant, and on thermal deformation changing the clearance of the bearing. These effects are ignored in this model. They can, however, easily be included by making the

lubricant properties functions of temperature and adjusting the bearing clearance to incorporate the change of the gap due to thermal expansion of the materials.

Results and Discussion

The pressure in the bearings is shown in Figure 2. The maximum pressure is located at the bottom of the bearing to support the static load.

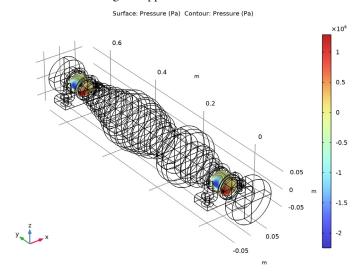


Figure 2: Pressure in the bearings.

The viscous heat dissipation in the bearing is shown in Figure 3. The maximum dissipation occurs where the pressure gradient is the largest.

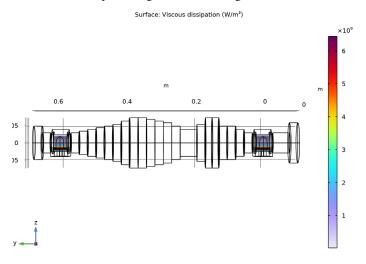


Figure 3: Viscous heat dissipation in the bearings.

The temperature profile in the bearing is shown in Figure 4. Significant heat dissipation occurs only at the location of the minimum film thickness. The temperature profile in the bearing is uniform due to the heat convection through the flow of the lubricant. Note that the temperature in both bearings is slightly different due to asymmetry in the heat convection from the rotor. The temperature in the rotor and the bearing housings is shown in Figure 5.



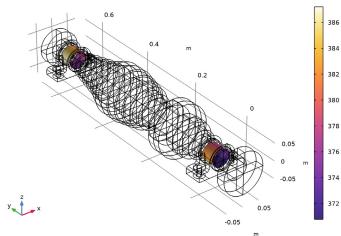


Figure 4: Temperature in the film.

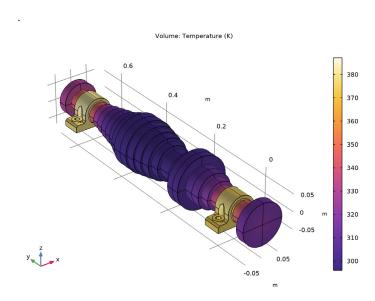


Figure 5: Temperature in the rotor and the bearing housings.

The stress in the rotor and the housings is shown in Figure 6. In the housings, the maximum stress occurs in the connection holes. As the connection holes are fixed, thermal expansion is resisted at these locations. In the rotor, the largest stresses appear near the thrust bearings. The rotor will expand radially as well as axially due the temperature increase, and the axial expansion is restricted by the thrust bearings giving rise to high stresses in these locations.

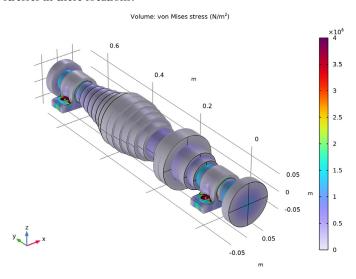


Figure 6: Stress in the rotor and the bearing housings.

Application Library path: Rotordynamics_Module/Tutorials/ rotor_thermal_stress

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click Model Wizard.

MODEL WIZARD

I In the Model Wizard window, click **3D**.

The problem involves the bearing modeling together with heat transfer in the film and heat transfer in the housing and shaft. Add corresponding interfaces to model all the physical phenomena.

- 2 In the Select Physics tree, select Heat Transfer>Thin Structures> Heat Transfer in Films (htlsh), Structural Mechanics>Rotordynamics> Hydrodynamic Bearing (hdb), and Structural Mechanics>Thermal-Structure Interaction> Thermal Stress, Solid.
- 3 Click Add.
- 4 Click 🔵 Study.
- 5 In the Select Study tree, select General Studies>Stationary.
- 6 Click M Done.

GEOMETRY I

Import I (impl)

- I In the Home toolbar, click Import.
- 2 In the Settings window for Import, locate the Import section.
- 3 Click Browse.
- 4 Browse to the model's Application Libraries folder and double-click the file rotor_thermal_stress.mphbin.
- 5 Click Import.

Form Union (fin)

- I In the Model Builder window, under Component I (compl)>Geometry I click Form Union (fin).
- 2 In the Settings window for Form Union/Assembly, locate the Form Union/Assembly section.
- 3 From the Action list, choose Form an assembly. Create an imprint on the shaft to use it as journal surface.
- 4 Select the Create imprints check box.
- 5 In the Home toolbar, click **Build All**.

GLOBAL DEFINITIONS

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click **Load from File**.
- **4** Browse to the model's Application Libraries folder and double-click the file rotor thermal stress.txt.

Create some selections for later use.

DEFINITIONS

Bearing Housing

- I In the **Definitions** toolbar, click **Explicit**.
- 2 Select Domains 28 and 29 only.
- 3 In the Settings window for Explicit, type Bearing Housing in the Label text field.

Rotor

- I In the **Definitions** toolbar, click **\int_{a} Complement**.
- 2 In the Settings window for Complement, type Rotor in the Label text field.
- 3 Locate the Input Entities section. Under Selections to invert, click + Add.
- 4 In the Add dialog box, select Bearing Housing in the Selections to invert list.
- 5 Click OK.

Identity Boundary Pair I (ap I)

- I In the Model Builder window, under Component I (compl)>Definitions click Identity Boundary Pair I (apl).
- 2 In the Settings window for Pair, locate the Source Boundaries section.
- 3 Click **Create Selection**.
- 4 In the Create Selection dialog box, type Journal 1 in the Selection name text field.
- 5 Click OK.
- 6 In the Settings window for Pair, locate the Destination Boundaries section.
- 7 Click **Greate Selection**.
- 8 In the Create Selection dialog box, type Bearing 1 in the Selection name text field.
- 9 Click OK.

Identity Boundary Pair 2 (ap2)

- I In the Model Builder window, click Identity Boundary Pair 2 (ap2).
- 2 In the Settings window for Pair, locate the Source Boundaries section.
- 3 Click **Greate Selection**.
- 4 In the Create Selection dialog box, type Journal 2 in the Selection name text field.
- 5 Click OK.
- 6 In the Settings window for Pair, locate the Destination Boundaries section.
- 7 Click **\mathbb{c}** Create Selection.
- 8 In the Create Selection dialog box, type Bearing 2 in the Selection name text field.
- 9 Click OK.

Housing foundation

- I In the **Definitions** toolbar, click **\(\bigcap_{\bigcap} \) Explicit**.
- 2 In the Settings window for Explicit, type Housing foundation in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- **4** Select Boundaries 367, 409, 430, and 472 only.
- 5 Select the Group by continuous tangent check box.

lournals

- I In the **Definitions** toolbar, click **I Union**.
- 2 In the Settings window for Union, type Journals in the Label text field.
- 3 Locate the Geometric Entity Level section. From the Level list, choose Boundary.
- 4 Locate the Input Entities section. Under Selections to add, click + Add.
- 5 In the Add dialog box, in the Selections to add list, choose Journal 1 and Journal 2.
- 6 Click OK.

Bearings

- I In the **Definitions** toolbar, click **Union**.
- 2 In the Settings window for Union, type Bearings in the Label text field.
- 3 Locate the Geometric Entity Level section. From the Level list, choose Boundary.
- 4 Locate the Input Entities section. Under Selections to add, click + Add.
- 5 In the Add dialog box, in the Selections to add list, choose Bearing 1 and Bearing 2.
- 6 Click OK.

Shaft Exterior

- I In the **Definitions** toolbar, click **\(\) Adjacent**.
- 2 In the Settings window for Adjacent, locate the Input Entities section.
- 3 Under Input selections, click + Add.
- 4 In the Add dialog box, select Rotor in the Input selections list.
- 5 Click OK.
- 6 In the Settings window for Adjacent, type Shaft Exterior in the Label text field.
- 7 Right-click Shaft Exterior and choose Duplicate.

Housing Exterior

- I In the Model Builder window, click Shaft Exterior I.
- 2 In the Settings window for Adjacent, locate the Input Entities section.
- 3 In the Input selections list, select Rotor.
- 4 Under Input selections, click Delete.
- 5 Under Input selections, click + Add.
- 6 In the Add dialog box, select Bearing Housing in the Input selections list.
- 7 Click OK.
- 8 In the Settings window for Adjacent, type Housing Exterior in the Label text field.

Convective boundaries (Shaft)

- I In the **Definitions** toolbar, click **Difference**.
- 2 In the Settings window for Difference, locate the Geometric Entity Level section.
- 3 From the Level list, choose Boundary.
- 4 Locate the Input Entities section. Under Selections to add, click + Add.
- 5 In the Add dialog box, select Shaft Exterior in the Selections to add list.
- 6 Click OK.
- 7 In the Settings window for Difference, locate the Input Entities section.
- 8 Under Selections to subtract, click + Add.
- 9 In the Add dialog box, select Journals in the Selections to subtract list.
- IO Click OK.
- II In the Settings window for Difference, type Convective boundaries (Shaft) in the Label text field.
- 12 Right-click Convective boundaries (Shaft) and choose Duplicate.

Convective boundaries (Housing)

- I In the Model Builder window, under Component I (comp1)>Definitions>Selections click
 Convective boundaries (Shaft) I.
- 2 In the Settings window for Difference, type Convective boundaries (Housing) in the Label text field.
- 3 Locate the Input Entities section. In the Selections to add list, select Shaft Exterior.
- 4 Under Selections to add, click Delete.
- 5 Under Selections to add, click + Add.
- 6 In the Add dialog box, select Housing Exterior in the Selections to add list.
- 7 Click OK.
- 8 In the Settings window for Difference, locate the Input Entities section.
- 9 In the Selections to subtract list, select Journals.
- 10 Under Selections to subtract, click **Delete**.
- II Under Selections to subtract, click + Add.
- 12 In the Add dialog box, select Bearings in the Selections to subtract list.
- I3 Click OK.

Exterior Edges (Journal)

- I In the **Definitions** toolbar, click **\bigcip_a Adjacent**.
- 2 In the Settings window for Adjacent, locate the Input Entities section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 Under Input selections, click Add.
- 5 In the Add dialog box, select Journals in the Input selections list.
- 6 Click OK.
- 7 In the Settings window for Adjacent, locate the Output Entities section.
- 8 From the Geometric entity level list, choose Adjacent edges.
- 9 In the Label text field, type Exterior Edges (Journal).
- **10** Right-click **Exterior Edges (Journal)** and choose **Duplicate**.

Interior Edges (Journal)

- I In the Model Builder window, under Component I (comp1)>Definitions>Selections click Exterior Edges (Journal) 1.
- 2 In the Settings window for Adjacent, type Interior Edges (Journal) in the Label text field.

- 3 Locate the Output Entities section. Clear the Exterior edges check box.
- 4 Select the Interior edges check box.

Create variable for convection coefficient on the shaft surface.

Variables 1

- I In the Model Builder window, right-click Definitions and choose Variables.
- 2 In the Settings window for Variables, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Convective boundaries (Shaft).
- **5** Locate the **Variables** section. In the table, enter the following settings:

Name	Expression	Unit	Description
d_rot	2*sqrt(X^2+Z^2)	m	Diameter of the shaft
Re_air	2*pi*fr*d_rot^2/ nu_air		Reynolds number for the flow
Pr_air	Cp_air*rho_air* nu_air/k_air		Prandtl number for air
h_shaft	<pre>k_air*Re_air^(2/3)* Pr_air^(1/3)/(15* d_rot/2)</pre>	W/(m²·K)	Heat transfer coefficient of the shaft

Add a **General Extrusion** operator to pick the temperature of the housing as a boundary temperature in lubricant film.

General Extrusion I (genext1)

- I In the Definitions toolbar, click / Nonlocal Couplings and choose General Extrusion.
- 2 In the Settings window for General Extrusion, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Bearings.
- **5** Locate the **Destination Map** section. In the **x-expression** text field, type X.
- 6 In the y-expression text field, type Y.
- 7 In the **z-expression** text field, type Z.
- 8 Locate the Source section. From the Source frame list, choose Material (X, Y, Z).

ADD MATERIAL

- I In the Home toolbar, click **Add Material** to open the **Add Material** window.
- 2 Go to the Add Material window.

- 3 In the tree, select Built-in>Structural steel.
- 4 Click Add to Component in the window toolbar.
- 5 In the Home toolbar, click 4 Add Material to close the Add Material window.

MATERIALS

Material 2 (mat2)

- I In the Model Builder window, under Component I (compl) right-click Materials and choose Blank Material.
- 2 In the Settings window for Material, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Journals.

HEAT TRANSFER IN FILMS (HTLSH)

- I In the Model Builder window, under Component I (compl) click Heat Transfer in Films (htlsh).
- 2 In the Settings window for Heat Transfer in Films, locate the Boundary Selection section.
- 3 From the Selection list, choose Journals.

HYDRODYNAMIC BEARING (HDB)

- I In the Model Builder window, under Component I (compl) click Hydrodynamic Bearing (hdb).
- 2 In the Settings window for Hydrodynamic Bearing, locate the Boundary Selection section.
- 3 From the Selection list, choose Journals.

MATERIALS

- I In the Model Builder window, under Component I (compl)>Materials click Material 2 (mat2).
- 2 In the Settings window for Material, locate the Material Contents section.
- **3** In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Heat capacity at constant pressure	Ср	СрО	J/(kg·K)	Basic
Density	rho	rho0	kg/m³	Basic

Property	Variable	Value	Unit	Property group
Thermal conductivity	k_iso ; kii = k_iso, kij = 0	k0	W/(m·K)	Basic
Thickness	lth	С	m	Shell
Rotation	lrot	0.0	deg	Shell
Mesh elements	Ine	2	I	Shell
Dynamic viscosity	mu	muO	Pa·s	Basic

4 In the Label text field, type Material: Oil.

HEAT TRANSFER IN FILMS (HTLSH)

Fluid 1

- I In the Model Builder window, under Component I (compl)>Heat Transfer in Films (htlsh) click Fluid 1.
- 2 In the Settings window for Fluid, locate the Layer Model section.
- 3 From the Layer type list, choose General.

Use the mean flow velocity of the film as a convective velocity for the heat transfer.

4 Locate the **Heat Convection** section. From the **u** list, choose Mean fluid velocity (material frame) (hdb/hjbl).

Heat Source 1

- I In the Physics toolbar, click **Boundaries** and choose **Heat Source**.
- 2 In the Settings window for Heat Source, locate the Boundary Selection section.
- 3 From the Selection list, choose Journals.

Use the viscous heat dissipation from bearing as a heat source in the film.

4 Locate the Heat Source section. From the Q_0 list, choose Viscous dissipation (hdb/hjb1).

Temperature is continuous between the lubricant film and the surrounding boundaries in contact with the film. Assign the temperature from the shaft as temperature at the top interface of the film. Similarly, assign the temperature of the housing as the temperature at the bottom interface of the film.

Temperature, Interface 1

- I In the Physics toolbar, click **Boundaries** and choose **Temperature**, **Interface**.
- 2 In the Settings window for Temperature, Interface, locate the Interface Selection section.
- 3 From the Apply to list, choose Top interface.

- **4** Locate the **Temperature** section. In the T_0 text field, type T2.
- 5 Locate the Boundary Selection section. From the Selection list, choose Journals.
- **6** Right-click **Temperature**, **Interface I** and choose **Duplicate**.

Temperature, Interface 2

- I In the Model Builder window, click Temperature, Interface 2.
- 2 In the Settings window for Temperature, Interface, locate the Interface Selection section.
- 3 From the Apply to list, choose Bottom interface.
- **4** Locate the **Temperature** section. In the T_0 text field, type genext1(T2). Since, the film is modeled on the journal boundary, a General Extrusion operator is used to pick the temperature from the housing boundary.

HYDRODYNAMIC BEARING (HDB)

- I Click the Show More Options button in the Model Builder toolbar.
- 2 In the Show More Options dialog box, in the tree, select the check box for the node Physics>Advanced Physics Options.
- 3 Click OK.
- 4 In the Model Builder window, under Component I (compl) click Hydrodynamic Bearing (hdb).
- 5 In the Settings window for Hydrodynamic Bearing, locate the Physical Model section.
- 6 From the Fluid type list, choose Liquid with cavitation.

Hydrodynamic Journal Bearing 1

- I In the Model Builder window, under Component I (compl)>Hydrodynamic Bearing (hdb) click Hydrodynamic Journal Bearing I.
- 2 In the Settings window for Hydrodynamic Journal Bearing, locate the Bearing Properties section.
- **3** In the *C* text field, type C.
- **4** From the \mathbf{X}_c list, choose From geometry. Bearing pressure supports the static load on the shaft. Assign half of the static load in each bearing.
- 5 Locate the Journal Properties section. From the Specify list, choose Load.

6 Specify the W_i vector as

0	х
0	у
-W/2	z

7 In the Ω text field, type 2*pi[rad]*fr.

Duplicate the current feature and change the selection to model the second bearing.

8 Right-click Component I (compl)>Hydrodynamic Bearing (hdb)> Hydrodynamic Journal Bearing I and choose Duplicate.

Hydrodynamic Journal Bearing 2

- I In the Model Builder window, click Hydrodynamic Journal Bearing 2.
- 2 In the Settings window for Hydrodynamic Journal Bearing, locate the Boundary Selection section.
- 3 From the Selection list, choose Journal 2.

Bearing Orientation I

- I In the Model Builder window, click Bearing Orientation 1.
- 2 In the Settings window for Bearing Orientation, locate the Bearing Orientation section.
- 3 From the Axis list, choose y-axis.
- **4** Specify the *V* vector as

1	x
0	у
0	z

SOLID MECHANICS (SOLID)

In the Model Builder window, under Component I (compl) click Solid Mechanics (solid).

Fixed Constraint I

- I In the Physics toolbar, click **Boundaries** and choose **Fixed Constraint**.
- 2 In the Settings window for Fixed Constraint, locate the Boundary Selection section.
- 3 From the Selection list, choose Housing foundation.

Add a **Prescribed Displacement** to model the thrust bearings. Two of the thrust bearings are at rotor ends and one is located in between the journal bearings at collars.

Prescribed Displacement 1

- I In the Physics toolbar, click **Boundaries** and choose **Prescribed Displacement**.
- **2** Select Boundaries 9, 10, 41, 42, 89, 90, 165, 168, 226, 231, 354, and 355 only.
- 3 In the Settings window for Prescribed Displacement, locate the Prescribed Displacement section.
- 4 From the Displacement in y direction list, choose Prescribed.

Add another **Prescribed Displacement** node at the axial edges of the rotor in the journal bearings. This is to restrict the motion of the rotor in the lateral directions at the bearing locations.

Prescribed Displacement 2

- I In the Physics toolbar, click Edges and choose Prescribed Displacement.
- 2 Select Edges 219 and 479 only.
- 3 In the Settings window for Prescribed Displacement, locate the Prescribed Displacement section.
- 4 From the Displacement in x direction list, choose Prescribed.
- 5 From the Displacement in z direction list, choose Prescribed.

HEAT TRANSFER IN SOLIDS (HT)

In the Model Builder window, under Component I (compl) click Heat Transfer in Solids (ht).

Heat Flux: Shaft

- I In the Physics toolbar, click **Boundaries** and choose **Heat Flux**.
- 2 In the Settings window for Heat Flux, type Heat Flux: Shaft in the Label text field.
- 3 Locate the Boundary Selection section. From the Selection list, choose Convective boundaries (Shaft).
- 4 Locate the Heat Flux section. From the Flux type list, choose Convective heat flux.
- **5** In the h text field, type h shaft.
- 6 Right-click Heat Flux: Shaft and choose Duplicate.

Heat Flux: Housing

- I In the Model Builder window, under Component I (compl)>Heat Transfer in Solids (ht) click Heat Flux: Shaft I.
- 2 In the Settings window for Heat Flux, type Heat Flux: Housing in the Label text field.
- 3 Locate the **Boundary Selection** section. From the **Selection** list, choose **Convective boundaries (Housing)**.

4 Locate the **Heat Flux** section. In the *h* text field, type 5.

MESH I

Swebt 1

- I In the Mesh toolbar, click Swept.
- 2 In the Settings window for Swept, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- 4 From the Selection list, choose Rotor.

Distribution I

- I Right-click Swept I and choose Distribution.
- 2 In the Settings window for Distribution, locate the Distribution section.
- 3 In the Number of elements text field, type 40.

Size 1

- I In the Model Builder window, right-click Swept I and choose Size.
- 2 In the Settings window for Size, locate the Geometric Entity Selection section.
- 3 Click Clear Selection.
- 4 Select Domains 26 and 27 only.
- 5 Locate the Element Size section. From the Predefined list, choose Finer.

Coby Face I

- I In the Mesh toolbar, click Copy and choose Copy Face.
- 2 In the Settings window for Copy Face, locate the Source Boundaries section.
- 3 From the Selection list, choose Journals.
- 4 Locate the Destination Boundaries section. From the Selection list, choose Bearings.

Free Triangular I

- I In the Mesh toolbar, click A More Generators and choose Free Triangular.
- **2** Select Boundaries 362, 404, 425, and 467 only.

Size 1

- I Right-click Free Triangular I and choose Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 From the Predefined list, choose Fine.
- 4 Click **Build Selected**.

5 Click III Build All.

Free Tetrahedral I

- I In the Mesh toolbar, click A Free Tetrahedral.
- 2 In the Settings window for Free Tetrahedral, click | Build All.

STUDY I

Add a fully coupled solver to solve for all the physics simultaneously.

Solution I (soll)

- I In the Study toolbar, click Show Default Solver.
- 2 In the Model Builder window, expand the Solution I (soll) node.
- 3 In the Model Builder window, expand the Study I>Solver Configurations> Solution I (soll)>Stationary Solver I node.
- 4 Right-click Study I>Solver Configurations>Solution I (sol1)>Stationary Solver I and choose Fully Coupled.
- 5 In the Study toolbar, click **Compute**.

RESULTS

Temperature, Shell (htlsh)

The temperature in the lubricant film is a default plot from the **Heat Transfer in Films** interface. This plot is shown in Figure 4.

- I Click the **Go to Default View** button in the **Graphics** toolbar.
- 2 In the Temperature, Shell (htlsh) toolbar, click on Plot.

The pressure in the lubricant film is a default plot from the **Hydrodynamic Bearing** interface as shown in Figure 2.

Fluid Pressure (hdb)

- I In the Model Builder window, click Fluid Pressure (hdb).
- 2 In the Fluid Pressure (hdb) toolbar, click Plot.

Adjust the color range in the default stress plot to highlight the stresses as shown in Figure 6.

Volume 1

- I In the Model Builder window, expand the Stress (solid) node, then click Volume I.
- 2 In the Settings window for Volume, click to expand the Range section.

- 3 Select the Manual color range check box.
- 4 Set the Maximum value to 400000000.

Deformation

- I In the Model Builder window, expand the Volume I node, then click Deformation.
- 2 In the Settings window for Deformation, locate the Scale section.
- 3 Select the Scale factor check box. In the associated text field, type 50.
- 4 In the Stress (solid) toolbar, click Plot.

The default temperature plot in the bearing housing and the rotor is shown in Figure 5.

Temperature (ht)

- I In the Model Builder window, under Results click Temperature (ht).
- 2 In the Temperature (ht) toolbar, click Plot.

Follow the instructions below to plot the viscous heat dissipation in the lubricant film as shown in Figure 3.

Viscous Heat

- I In the Home toolbar, click In Add Plot Group and choose 3D Plot Group.
- 2 In the Settings window for 3D Plot Group, type Viscous Heat in the Label text field.
- 3 Locate the Plot Settings section. From the View list, choose New view.

Surface I

- I Right-click Viscous Heat and choose Surface.
- 2 In the Settings window for Surface, click Replace Expression in the upper-right corner of the Expression section. From the menu, choose Component I (compl)> Hydrodynamic Bearing>Journal and bearing properties>Viscous loss>hdb.Qvd -Viscous dissipation - W/m3.
- 3 Locate the Coloring and Style section. Click Change Color Table.
- 4 In the Color Table dialog box, select Rainbow>Prism in the tree.
- 5 Click OK.
- 6 In the Viscous Heat toolbar, click Plot.
- 7 Click the YZ Go to YZ View button in the Graphics toolbar.
- 8 Click the YZ Go to YZ View button in the Graphics toolbar.
- 9 Click the YZ Go to YZ View button in the Graphics toolbar.
- **10** Click the **Zoom Extents** button in the **Graphics** toolbar.

Stress (solid)

Click the **Go to Default View** button in the **Graphics** toolbar.