

Test Bench Car Interior

In this application, a point source generates a pressure wave in the Sound Brick, a test bench car interior (Ref. 1). The sound pressure level is measured at another point and at a range of frequencies high enough that a good mesh resolution is required to properly resolve the wave. To get an idea of the accuracy of the model, the sound pressure level response is studied using four different mesh resolutions.

Model Definition

The geometrical and material parameters in this model come from Ref. 2. The geometry of the Sound Brick is shown in Figure 1. The model solves for the acoustic pressure p in the frequency domain using the Pressure Acoustics, Frequency Domain interface.

A monopole point flow source of strength $Q_S = 10^{-5} \text{ m}^3/\text{s}$ located at the point $\mathbf{R}_0 = (0.21, 0, 1.28)$ drives the system. The walls of the Sound Brick are assumed to be perfectly reflecting and have the default sound hard boundary condition. The sound pressure level is measured at the point \mathbf{R}_1 = (1.34, 1.22, 0.8) at a range of frequencies from 743 Hz to 745 Hz.

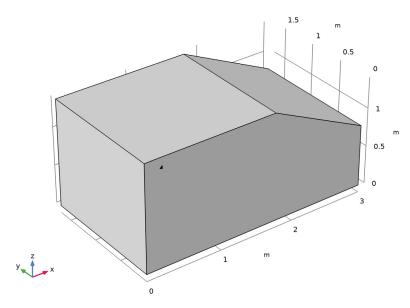


Figure 1: The geometry of the test bench car interior.

The dimensions of the test bench car are length \times height \times depth = 3.0 m \times 1.4 m \times 1.7 m. The windshield has its lower end 0.8 m above the floor and with an inclination such that the entire volume of the geometry is 6.5 m^3 .

Results and Discussion

Figure 2 shows the instantaneous pressure distribution in the car interior at 745 Hz.

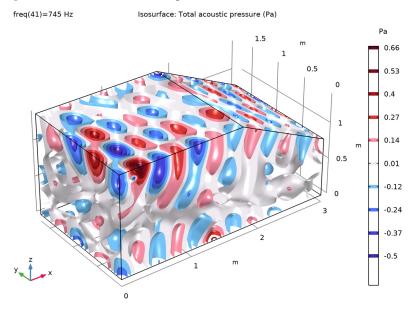


Figure 2: Isosurface plot of the pressure distribution at 745 Hz, for the coarsest one of the 4 meshes used in the model.

A direct comparison of the computational results with those published in Ref. 2 is difficult for two reasons:

- The sampling frequency is too small to resolve some of the resonances at the higher frequencies
- The amplitude for the point source is not given

Nevertheless, it is possible to examine the accuracy of the results by studying how they converge when using successively greater mesh resolutions.

To provide an idea of the accuracy of the various solutions, Figure 3 shows the measured sound pressure level between 743 Hz and 745 Hz, computed at 0.005 Hz intervals for

the listed mesh resolutions. The plot can be reproduced by changing the frequency resolution from 0.05 Hz to 0.005 Hz when following the Modeling Instructions. As you can see in the plot, all the curves follow each other except at the dips and peaks, where the curves predict slightly different frequencies for the resonances. This is an idealized setup where there is no sources of damping, so a perfect resolution in space and time would result in infinite sound levels in the peaks and absolutely no sound in the dips. In real-life applications, the extremes would be much less pronounced than in the plot because the sound sources are rarely point-like and the walls are usually not perfectly reflecting. In such cases, you can expect a similar solution accuracy at the peaks and the dips as in between them.

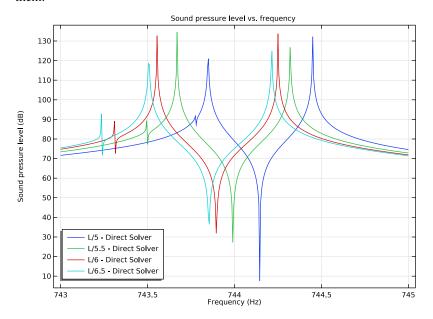


Figure 3: Sound pressure level (dB) at the point of measurement as a function of frequency (Hz) for a few different mesh densities. The legend shows the maximum mesh element size in fractions of the wavelength L.

Notes about Modeling in COMSOL Multiphysics

In general, 5 to 6 second-order elements per wavelength are needed to resolve the waves. In the case treated in the step-by-step instructions, you generate the initial mesh by specifying a maximum element size of $\lambda/5$, where $\lambda = c/f$ is the free-space wavelength of the sound waves at 745 Hz. After showing how you to set up, run, and analyze the model with this mesh, the instructions take you through the mesh convergence study. As it is seen

from this model, depending on the level of accuracy desired, 5 second-order elements per wavelength is not always fine enough to get accurate resonances. When a model has sharp resonances or it has small geometry details, the mesh should be finer. Therefore it is always recommended to do a mesh convergence study when performing simulations, as is done in this model. Moreover, as it is clear from this model, having a fine enough frequency resolution is also important as it is clear that the two solutions match well between the resonances.

Note: It is good simulation practice to perform a mesh convergence analysis to be sure that the obtained results have converged.

References

- 1. The Sound Brick is located in Acoustic Competence Centre, Graz, Austria http://portal.tugraz.at/portal/page/portal/TU_Graz
- 2. A. Hepberger, H. Priebsch, W. Desmet, B. Van Hal, B. Pluymers, and P. Sas, "Application of the Wave-Based Method for the Steady-state Acoustic Response Prediction of a Car Cavity in the Mid-frequency Range," Proceedings of the International Conference on Noise and Vibration Engineering, ISMA2002, Leuven, Belgium, pp. 877-884, 2002.

Application Library path: Acoustics Module/Automotive/ test bench car interior

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click Model Wizard.

MODEL WIZARD

I In the Model Wizard window, click **3D**.

- 2 In the Select Physics tree, select Acoustics>Pressure Acoustics>Pressure Acoustics, Frequency Domain (acpr).
- 3 Click Add.
- 4 Click Study.
- 5 In the Select Study tree, select General Studies>Frequency Domain.
- 6 Click M Done.

GLOBAL DEFINITIONS

Now, either enter the parameters used for the model manually or load them from the file test_bench_car_interior_parameters.txt.

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- **3** In the table, enter the following settings:

Name	Expression	Value	Description
rho0	1.2[kg/m^3]	1.2 kg/m³	Air density
c0	343.8[m/s]	343.8 m/s	Speed of sound in air
f0	745[Hz]	745 Hz	Highest frequency used in the model
L	c0/f0	0.46148 m	Shortest wavelength
S	1e-5[m^3/s]	IE-5 m³/s	Flow source strength

You will use L, the wavelength corresponding to the highest frequency, when defining the mesh size.

Import the geometry of the test bench car interior from a file.

GEOMETRY I

Import I (impl)

- I In the Home toolbar, click Import.
- 2 In the Settings window for Import, locate the Import section.
- 3 Click Browse.
- **4** Browse to the model's Application Libraries folder and double-click the file test bench car interior.mphbin.

5 Click Import.

Create two points, one for the source and one for the receiver.

Point I (btl)

- I In the Geometry toolbar, click \bigoplus More Primitives and choose Point.
- 2 In the Settings window for Point, locate the Point section.
- 3 In the x text field, type 0.21.
- 4 In the z text field, type 1.28.

Point 2 (bt2)

- I In the Geometry toolbar, click \bigoplus More Primitives and choose Point.
- 2 In the Settings window for Point, locate the Point section.
- 3 In the x text field, type 1.34.
- 4 In the y text field, type 1.22.
- 5 In the z text field, type 0.8.

Form Union (fin)

- I In the Model Builder window, click Form Union (fin).
- 2 In the Settings window for Form Union/Assembly, click | Build Selected.
- 3 Click the **Zoom Extents** button in the **Graphics** toolbar.

MATERIALS

Material I (mat I)

- I In the Model Builder window, under Component I (compl) right-click Materials and choose Blank Material.
- 2 Select Domain 1 only.
- 3 In the Settings window for Material, locate the Material Contents section.
- **4** In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Density	rho	rho0	kg/m³	Basic
Speed of sound	С	c0	m/s	Basic

PRESSURE ACOUSTICS, FREQUENCY DOMAIN (ACPR)

I In the Model Builder window, under Component I (compl) click Pressure Acoustics, Frequency Domain (acpr).

- 2 In the Settings window for Pressure Acoustics, Frequency Domain, click to expand the **Discretization** section.
- 3 From the Element order list, choose Quadratic serendipity.

Monopole Point Source I

- I In the Physics toolbar, click Points and choose Monopole Point Source.
- **2** Select Point 5 only.

Point 5 is the one on the geometry's front wall.

- 3 In the Settings window for Monopole Point Source, locate the Monopole Point Source section.
- **4** In the Q_S text field, type i*S.

Introducing a phase shift by multiplying the source strength with the imaginary unit gives nonzero pressure plots at the default zero value for the complex pressure variable's phase. You can also here simply enter S in the Q_S text field and then enter pi/2 for the phase \(\phi \).

MESH I - L/5

In general, 5 to 6 second-order elements per wavelength are needed to resolve the waves. For more details, see Meshing (Resolving the Waves) in the Acoustics Module User's Guide. In this model, start by using 5 elements per wavelength and set up the mesh manually to get a structured swept mesh. This will lead to fewer degrees of freedom to be solved for. Proceed by directly adding the desired mesh component.

- I In the Model Builder window, under Component I (compl) click Mesh I.
- 2 In the Settings window for Mesh, type Mesh 1 L/5 in the Label text field.

Swept I

In the Mesh toolbar, click A Swept.

Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- **3** Click the **Custom** button.
- 4 Locate the Element Size Parameters section. In the Maximum element size text field, type L/5.

Swept I

I In the Model Builder window, click Swept I.

- 2 In the Settings window for Swept, locate the Source Faces section.
- **3** Click to select the **Activate Selection** toggle button.
- 4 Select Boundary 2 only.
- 5 Click III Build All.

STUDY I

Steb 1: Frequency Domain

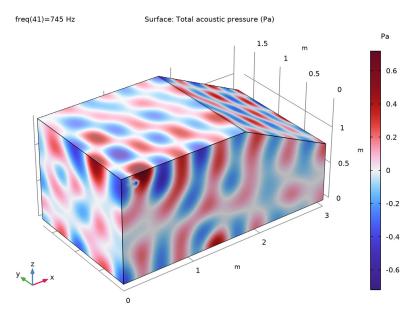
- I In the Model Builder window, under Study I click Step I: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Study Settings section.
- 3 In the Frequencies text field, type range (f0-2,0.05,f0).
- 4 In the Model Builder window, click Study 1.
- 5 In the Settings window for Study, type Study 1 L/5 Direct Solver in the Label text field.

Solution I (soll)

- 2 In the Model Builder window, expand the Solution I (soll) node. Enabling the **Block low rank factorization** will reduce both running time and memory consumption. This option is good for pressure acoustics models that do not include nonlocal couplings.
- 3 In the Model Builder window, expand the Study I L/5 Direct Solver> Solver Configurations>Solution I (soll)>Stationary Solver I node, then click Suggested Direct Solver (acpr).
- 4 In the Settings window for Direct, locate the General section.
- 5 Select the Block low rank factorization check box.
- 6 In the Study toolbar, click **Compute**.

RESULTS

Acoustic Pressure (acpr)



The first two of the default plots show the pressure and sound pressure level distributions on the walls of the car at 500 Hz. The third plot gives a closer look at the standing waves inside the car in the form of isosurfaces for the pressure (see Figure 2).

Acoustic Pressure, Isosurfaces (acpr)

Proceed to group the plots to facilitate the navigation through the results.

Acoustic Pressure (acpr), Acoustic Pressure, Isosurfaces (acpr), Sound Pressure Level (acpr)

- I In the Model Builder window, under Results, Ctrl-click to select Acoustic Pressure (acpr), Sound Pressure Level (acpr), and Acoustic Pressure, Isosurfaces (acpr).
- 2 Right-click and choose **Group**.

Results - L/5 - Direct Solver

In the Settings window for Group, type Results - L/5 - Direct Solver in the Label text field.

ID Plot Group 4

I In the Home toolbar, click **Add Plot Group** and choose ID Plot Group.

- 2 In the Settings window for ID Plot Group, click to expand the Title section.
- 3 From the Title type list, choose Manual.
- **4** In the **Title** text area, type Sound pressure level vs. frequency.

Point Graph 1

- I Right-click ID Plot Group 4 and choose Point Graph.
- 2 Click the Wireframe Rendering button in the Graphics toolbar.
- 3 Select Point 6 only.

Wireframe rendering makes it easier to select Point 6, which is in the interior of the car.

- 4 In the Settings window for Point Graph, click Replace Expression in the upper-right corner of the y-Axis Data section. From the menu, choose Component I (compl)> Pressure Acoustics, Frequency Domain>Pressure and sound pressure level>acpr.Lp_t -Total sound pressure level - dB.
- 5 Click Replace Expression in the upper-right corner of the x-Axis Data section. From the menu, choose Component I (compl)>Pressure Acoustics, Frequency Domain>Global> acpr.freg - Frequency - Hz.
- 6 Click to expand the **Legends** section. Select the **Show legends** check box.
- 7 From the Legends list, choose Manual.
- **8** In the table, enter the following settings:

Legends L/5 - Direct Solver

9 In the ID Plot Group 4 toolbar, click Plot.

This concludes the coarse mesh version of the model. The rest of the instructions show you how to perform a mesh convergence study, by repeating the solution procedure with three consecutively finer meshes and comparing the results. The approach in this model is useful if you want to keep all the results in the model and easily compare them afterward. However, if you systematically want to process solutions on many different meshes or geometries, a batch run would be more convenient, as it avoids the need to perform any of the settings more than once.

To prepare for your convergence study, start by creating the meshes. Each one should have a successively smaller maximum element size.

MESH 2 - L/5.5

I In the Mesh toolbar, click Add Mesh and choose Add Mesh.

2 In the Settings window for Mesh, type Mesh 2 - L/5.5 in the Label text field.

Swept I

In the Mesh toolbar, click Swept.

Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size Parameters section.
- 3 In the Maximum element size text field, type L/5.5.

Swebt 1

- I In the Model Builder window, click Swept I.
- 2 In the Settings window for Swept, locate the Source Faces section.
- **3** Click to select the **Activate Selection** toggle button.
- 4 Select Boundary 2 only.

MESH 3 - L/6

- I In the Mesh toolbar, click Add Mesh and choose Add Mesh.
- 2 In the Settings window for Mesh, type Mesh 3 L/6 in the Label text field.
- 3 In the Mesh toolbar, click A Swept.

Size

- I In the Model Builder window, under Component I (compl)>Meshes>Mesh 3 L/6 click Size.
- 2 In the Settings window for Size, locate the Element Size Parameters section.
- 3 In the Maximum element size text field, type L/6.

Swebt I

- I In the Model Builder window, click Swept I.
- 2 In the Settings window for Swept, locate the Source Faces section.
- **3** Click to select the **Activate Selection** toggle button.
- 4 Select Boundary 2 only.

MESH 4 - L/6.5

- I In the Mesh toolbar, click Add Mesh and choose Add Mesh.
- 2 In the Settings window for Mesh, type Mesh 4 L/6.5 in the Label text field.
- 3 In the Mesh toolbar, click A Swept.

Size

- I In the Model Builder window, under Component I (compl)>Meshes>Mesh 4 L/6.5 click Size.
- 2 In the Settings window for Size, locate the Element Size Parameters section.
- 3 In the Maximum element size text field, type L/6.5.

Swept I

- I In the Model Builder window, click Swept I.
- 2 In the Settings window for Swept, locate the Source Faces section.
- **3** Click to select the **Activate Selection** toggle button.
- 4 Select Boundary 2 only.

ADD STUDY

- I In the Home toolbar, click Add Study to open the Add Study window.
- 2 Go to the Add Study window.
- 3 Find the Studies subsection. In the Select Study tree, select General Studies> Frequency Domain.
- 4 Click Add Study in the window toolbar.
- 5 In the Select Study tree, select General Studies>Frequency Domain.
- 6 Click Add Study in the window toolbar.
- 7 In the Select Study tree, select General Studies>Frequency Domain.
- 8 Click Add Study in the window toolbar.
- 9 In the Home toolbar, click Add Study to close the Add Study window.

STUDY 2

Step 1: Frequency Domain

- I In the Model Builder window, under Study 2 click Step 1: Frequency Domain.
- 2 In the Settings window for Frequency Domain, click to expand the Mesh Selection section.
- **3** In the table, enter the following settings:

Component	Mesh
Component I	Mesh 2 - L/5.5

STUDY 3

I In the Model Builder window, under Study 3 click Step 1: Frequency Domain.

- 2 In the Settings window for Frequency Domain, locate the Mesh Selection section.
- **3** In the table, enter the following settings:

Component	Mesh
Component I	Mesh 3 - L/6

STUDY 4

- I In the Model Builder window, under Study 4 click Step 1: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Mesh Selection section.
- **3** In the table, enter the following settings:

Component	Mesh
Component I	Mesh 4 - L/6.5

STUDY 2

- I In the Model Builder window, under Study 2 click Step 1: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Study Settings section.
- 3 In the Frequencies text field, type range (f0-2,0.05,f0).

STUDY 3

- I In the Model Builder window, under Study 3 click Step 1: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Study Settings section.
- 3 In the Frequencies text field, type range (f0-2,0.05,f0).

STUDY 4

- I In the Model Builder window, under Study 4 click Step 1: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Study Settings section.
- 3 In the Frequencies text field, type range (f0-2,0.05,f0).

Having set up a Frequency Domain study step for each study, you are now ready to start the solution process. The first solution already exists, so start by solving the second one.

STUDY 2 - L/5.5 - DIRECT SOLVER

- I In the Model Builder window, click Study 2.
- 2 In the Settings window for Study, type Study 2 L/5.5 Direct Solver in the **Label** text field.

Solution 2 (sol2)

- I In the Study toolbar, click Show Default Solver.
- 2 In the Model Builder window, expand the Solution 2 (sol2) node.
- 3 In the Model Builder window, expand the Study 2 L/5.5 Direct Solver> Solver Configurations>Solution 2 (sol2)>Stationary Solver I node, then click Suggested Direct Solver (acpr).
- 4 In the Settings window for Direct, locate the General section.
- 5 Select the Block low rank factorization check box.
- 6 In the Study toolbar, click **Compute**.

RESULTS

Acoustic Pressure (acpr) I

When the solver is done, three more plots will appear, this time for the second one of your meshes. Group these plots as you did with the previous results to facilitate the navigation through the results. Then, proceed to solve for the next mesh.

Acoustic Pressure (acpr) I, Acoustic Pressure, Isosurfaces (acpr) I, Sound Pressure Level (acpr) I

- I In the Model Builder window, under Results, Ctrl-click to select Acoustic Pressure (acpr) I, Sound Pressure Level (acpr) I, and Acoustic Pressure, Isosurfaces (acpr) 1.
- 2 Right-click and choose Group.

Results - L/5.5 - Direct Solver

In the Settings window for Group, type Results - L/5.5 - Direct Solver in the Label text field.

STUDY 3 - L/6 - DIRECT SOLVER

- I In the Model Builder window, click Study 3.
- 2 In the Settings window for Study, type Study 3 L/6 Direct Solver in the Label text field.

Solution 3 (sol3)

- 2 In the Model Builder window, expand the Solution 3 (sol3) node.

- 3 In the Model Builder window, expand the Study 3 L/6 Direct Solver> Solver Configurations>Solution 3 (sol3)>Stationary Solver I node, then click Suggested Direct Solver (acpr).
- 4 In the Settings window for Direct, locate the General section.
- 5 Select the Block low rank factorization check box.
- 6 In the Study toolbar, click **Compute**.

RESULTS

Acoustic Pressure, Isosurfaces (acpr) 1 Group these results.

Acoustic Pressure (acpr) 2, Acoustic Pressure, Isosurfaces (acpr) 2, Sound Pressure Level (acbr) 2

- I In the Model Builder window, under Results, Ctrl-click to select Acoustic Pressure (acpr) 2, Sound Pressure Level (acpr) 2, and Acoustic Pressure, Isosurfaces (acpr) 2.
- 2 Right-click and choose **Group**.

Results - L/6 - Direct Solver

In the Settings window for Group, type Results - L/6 - Direct Solver in the Label text field.

Finally, solve for the finest mesh case.

STUDY 4 - L/6.5 - DIRECT SOLVER

- I In the Model Builder window, click Study 4.
- 2 In the Settings window for Study, type Study 4 L/6.5 Direct Solver in the Label text field.

Solution 4 (sol4)

- 2 In the Model Builder window, expand the Solution 4 (sol4) node.
- 3 In the Model Builder window, expand the Study 4 L/6.5 Direct Solver> Solver Configurations>Solution 4 (sol4)>Stationary Solver I node, then click Suggested Direct Solver (acpr).
- 4 In the Settings window for Direct, locate the General section.
- 5 Select the Block low rank factorization check box.
- 6 In the Study toolbar, click **Compute**.

Group these results.

RESULTS

Acoustic Pressure (acpr) 3, Acoustic Pressure, Isosurfaces (acpr) 3, Sound Pressure Level (acbr) 3

- I In the Model Builder window, under Results, Ctrl-click to select Acoustic Pressure (acpr) 3, Sound Pressure Level (acpr) 3, and Acoustic Pressure, Isosurfaces (acpr) 3.
- 2 Right-click and choose Group.

Results - L/6.5 - Direct Solver

In the Settings window for Group, type Results - L/6.5 - Direct Solver in the Label text field.

ID Plot Group 4

- I In the Model Builder window, click ID Plot Group 4.
- 2 Drag and drop below Results L/6.5 Direct Solver.

Point Graph 1

In the Model Builder window, right-click Point Graph I and choose Duplicate.

Point Graph 2

- I In the Model Builder window, click Point Graph 2.
- 2 In the Settings window for Point Graph, locate the Data section.
- 3 From the Dataset list, choose Study 2 L/5.5 Direct Solver/Solution 2 (sol2).
- **4** Locate the **Legends** section. In the table, enter the following settings:

Legends L/5.5 - Direct Solver

5 Right-click Point Graph 2 and choose Duplicate.

Point Graph 3

- I In the Model Builder window, click Point Graph 3.
- 2 In the Settings window for Point Graph, locate the Data section.
- 3 From the Dataset list, choose Study 3 L/6 Direct Solver/Solution 3 (sol3).

4 Locate the **Legends** section. In the table, enter the following settings:

Legends				
L/6	-	Direct	Solver	

5 Right-click **Point Graph 3** and choose **Duplicate**.

Point Graph 4

- I In the Model Builder window, click Point Graph 4.
- 2 In the Settings window for Point Graph, locate the Data section.
- 3 From the Dataset list, choose Study 4 L/6.5 Direct Solver/Solution 4 (sol4).
- **4** Locate the **Legends** section. In the table, enter the following settings:

Legends					
	L/6.5	-	Direct	Solver	

5 In the ID Plot Group 4 toolbar, click Plot.

SPL Response

- I In the Model Builder window, under Results click ID Plot Group 4.
- 2 In the Settings window for ID Plot Group, type SPL Response in the Label text field.
- 3 Locate the Plot Settings section. Select the x-axis label check box.
- 4 Select the y-axis label check box. In the associated text field, type Sound pressure level (dB).
- 5 Locate the Legend section. From the Position list, choose Lower left.

The plot should look like the figure below. A version of the plot with a higher frequency resolution of 0.005 Hz is depicted in Figure 3.

