

Vibration Control in a Motor Drive Using an Active Magnetic Bearing

Introduction

Active magnetic bearings (AMBs) are used for controlling the vibration in mechanical systems by using a feedback control. In this model, we consider a motor driven assembly in which the speed of the system is gradually increased. As the speed of the system increases, it crosses through many critical speeds. Vibrations in the system due to existing imbalances are amplified when the speed of the rotor is near critical speed.

Two different time dependent studies are performed on the system with the objective of studying the influence of the active magnetic bearings on the overall vibration of the system. The first study excludes the active magnetic bearing to get the vibration levels in the system without any control. In the second study, an AMB is switched on when the vibration in the system is high due to resonance. Subsequently, the system responds to the additional control forces from the AMB. The response of the system including the AMB is compared with the response excluding the AMB.

The results include the vibration at different locations in the system as well as stress and control currents.

Model Definition

The model consists of a rotor driven by a motor. The rotor is supported by two journal bearings along its length. Two additional bearings are located inside the motor. A fan is also located inside the motor, outside the two bearings. The whole assembly is mounted on a foundation through a motor pedestal and bearing housings. In the model, the rotor speed is gradually increased to simulate the run-up of the system. During the run-up, the system crosses through critical speeds. As a result, high-amplitude vibrations are observed in the system. The system also has an electromagnetic bearing, placed between the two journal bearings on the rotor, to control the vibrations during resonance.

Figure 1 shows a schematic representation of the system, while Figure 2 shows an inside view of the motor to highlight the rotor components inside the motor.

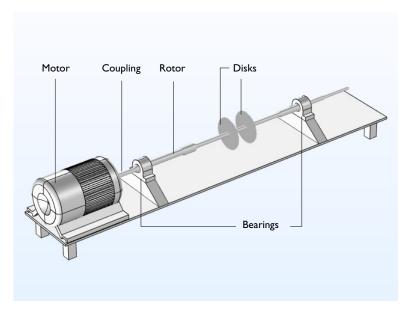


Figure 1: Motor-driven rotor assembly.

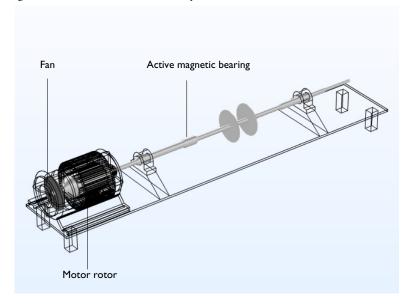


Figure 2: Inside view of the motor.

The motor and the foundation are modeled using a Solid Mechanics interface. The rotor (inside and outside of the motor) is modeled using a Beam Rotor interface. The rotor inside the motor is connected to the rotor outside the motor through a rigid coupling. Rayleigh damping is added to both rotors to dampen out vibrations. Hence, the coupling is not modeled, rather the whole rotor is considered as a single entity.

The two disks on the rotor and the fan inside the motor are modeled using the Disk feature in the Beam Rotor interface. Both bearings inside the motor and the two external journal bearings are modeled using a lumped approximation with equivalent stiffness and damping coefficients. The electromagnetic bearing is modeled using the Active Magnetic Bearing feature in the Beam Rotor interface. The rotors are connected to the static structure through Attachments on the static structure as a foundation in the bearing nodes. The electromagnetic interaction between the stator and the rotor in the motor is neglected in this study. The properties of the rotor are given in Table 1, and the length and diameter between different stations are given in Table 2. Lastly, the properties of the bearings are given in Table 3.

Two *Time Dependent* studies are performed on the system:

- I Excluding the active magnetic bearing to observe the vibration amplitudes without any control.
- 2 Including the active magnetic bearing and switching it on when the resonance is about to occur.

The results are then compared to analyze the effectiveness of the control mechanism.

TABLE I: ROTOR PROPERTIES.

Parameter	Value
Young's modulus of the rotor, \emph{E}	211 GPa
Poisson's ratio of the rotor, ν	0.3
Density of the rotor, ρ	7850 kg/m ³
Mass damping parameter of the rotor, $lpha_{\mathrm{d}M}$	7.0 1/s
Stiffness damping parameter, $eta_{ ext{d}K}$	3.2·10 ⁻⁵ s
ength of the rotor in motor, L_{m}	0.343 m
ength of the rotor outside motor, L _e	1.3 m
ocation of the first bearing in motor (Station 2)	0.0467 m
ocation of the second bearing in motor (Station 5)	0.2933 m
ocation of the coupling (Station 6)	0.343 m
ocation of the first external bearing (Station 8)	0.4683 m

TABLE I: ROTOR PROPERTIES.

Parameter	Value
Location of the second external bearing (Station 17)	1.3683 m
Location of the active magnetic bearing (Station 11)	0.7183 m
Location of the first disk (Station 13)	0.9433 m
Location of the second disk (Station 14)	1.0433 m
Mass of the fan, $m_{ m f}$	5.0 kg
Transverse moment of inertia of the fan, $I_{ m df}$	0.0079 kg·m ²
Polar moment of inertia of the fan, $I_{ m pf}$	0.0154 kg·m ²
Mass of the external disks, m	3.9 kg
Transverse moment of inertia of the external disks, $I_{ m d}$	0.0057 kg·m ²
Polar moment of inertia of the external disks, $I_{ m p}$	0.0113 kg·m ²
Eccentricity in the first disk in local y direction	73 μm
Eccentricity in the second disk in local z direction	222 μm

TABLE 2: GEOMETRIC PROPERTIES OF THE ROTOR

Rotor segment	Length (m)	Diameter (m)
Station 1-2	0.0467	0.02
Station 2-3	0.0467	0.02
Station 3-4	0.15	0.1
Station 4-5	0.05	0.02
Station 5-6	0.05	0.02
Station 6-7	0.1	0.02
Station 7-8	0.025	0.04
Station 8-9	0.025	0.04
Station 9-10	0.2	0.03
Station 10-11	0.025	0.04
Station 11-12	0.025	0.04
Station 12-13	0.2	0.02
Station 13-14	0.1	0.02
Station 14-15	0.2	0.02
Station 15-16	0.1	0.03
Station 16-17	0.025	0.04
Station 17-18	0.025	0.04

TABLE 2: GEOMETRIC PROPERTIES OF THE ROTOR

Rotor segment	Length (m)	Diameter (m)
Station 18-19	0.1	0.02
Station 19-20	0.15	0.015

TABLE 3: BEARING PROPERTIES.

Parameter	Value
Bearing stiffness, $k_{ m b}$	357 MN/m
Bearing damping coefficient, $c_{ m b}$	I 30 kN·s/m
AMB gap, h	2 mm
Bias current positive axis, $i_{ m bp}$	I A
Bias current negative axis, $i_{ m bn}$	I A
Maximum current, $i_{ m max}$	5 A
Proportional gain, $K_{ m p}$	$2(i_{bp}^2 + i_{bn}^2)/((i_{bp} + i_{bn})h)$
Integral gain, $K_{ m i}$	$K_{ m p}$ /0.001
Derivative gain, $K_{ m d}$	0.005 $K_{ m p}$
Force constant, $F_{ m c}$	20

Results and Discussion

The stress in the assembly without the AMB is shown in Figure 3. The maximum stress occurs in the central region of the rotor. The eccentricity of the disks cause the rotor to bend between the bearings, thus stressing the rotor. Moreover, the diameter of the rotor is smaller close to the disks and, as a result, the maximum stress is observed in this part of the rotor. The figure also shows that the foundation is primarily deformed in the first bending mode.

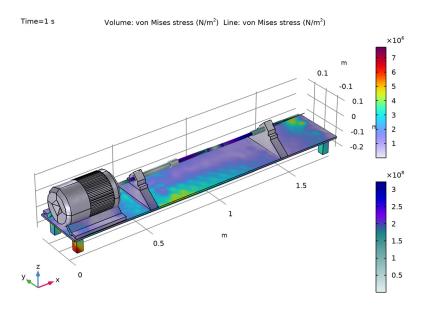


Figure 3: Stress in the system without the AMB at 1 s.

The stress profile in the assembly with the AMB active is shown in Figure 4. Again, we observe higher stresses in the part of the rotor near the disks. In this case, the deformation of the foundation is likewise dominated by the first bending mode. The rotor is found to deform in a higher bending mode.

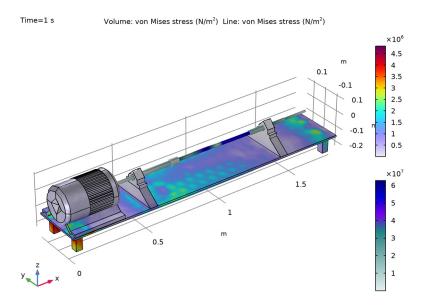


Figure 4: Stress in the system with the AMB at 1 s.

The vertical displacement of the rotor at the coupling location is compared in Figure 5. It is found that the system without the AMB undergoes multiple resonances during the runup. In the presence of the AMB in the system, the vibration levels during resonance have decreased significantly. To analyze it further, a comparison of the vibrations near the AMB itself is conducted. The associated plot is shown in Figure 6. In the figure, it is clear that any vibrations after the activation of the AMB are suppressed. Thus, the AMB helps in isolating any vibrations caused by imbalances in the rotor and prevents them from propagating to the motor.

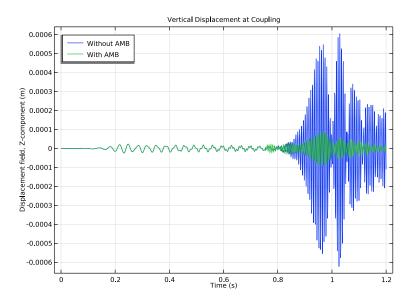


Figure 5: Vertical displacement at coupling.

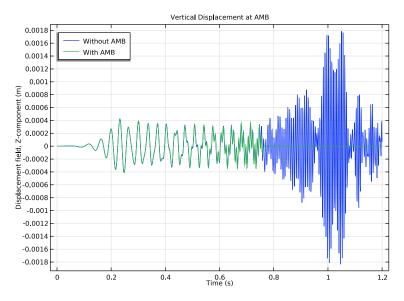


Figure 6: Vertical displacement at active magnetic bearing.

The variation in the control current in the AMB in both transverse directions is shown in Figure 7. Since a step function is used to switch the bearing forces on, the control current in the bearing still responds to the displacement variations before the AMB is switched on. After the bearing is switched on, it quickly suppresses the vibration level in the system. Consequently, the required control current also decreases.

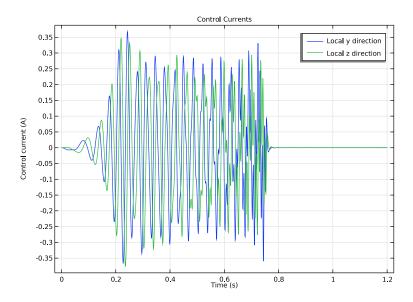


Figure 7: Variation of the control current.

The force in the AMB in the transverse directions is shown in Figure 8. Before the bearing is switched on, the force remains zero. In the later part, when the AMB is active, we can clearly see significant variation in the control forces, even though the variation in the control currents is small.

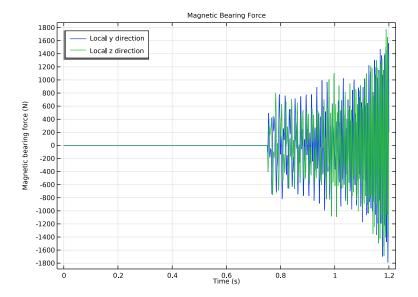


Figure 8: Force in the electromagnetic bearing.

Application Library path: Rotordynamics_Module/Tutorials/vibration_control_with_amb

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click 1 3D.
- 2 In the Select Physics tree, select Structural Mechanics>Solid Mechanics (solid) and Structural Mechanics>Rotordynamics>Beam Rotor (rotbm).
- 3 Click Add.
- 4 Click Study.

- 5 In the Select Study tree, select General Studies>Time Dependent.
- 6 Click M Done.

GEOMETRY I

Import the geometry of a rotor-stator assembly.

Import I (impl)

- I In the **Home** toolbar, click **Import**.
- 2 In the Settings window for Import, locate the Import section.
- 3 Click Browse.
- 4 Browse to the model's Application Libraries folder and double-click the file vibration control with amb.mphbin.
- 5 Click Import.

Import the rotor and bearing parameters.

GLOBAL DEFINITIONS

Parameters: Rotor

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file vibration_control_with_amb_rotor.txt.
- 5 In the Label text field, type Parameters: Rotor.

Parameters: Bearing

- I In the Home toolbar, click Pi Parameters and choose Add>Parameters.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click Load from File.
- 4 Browse to the model's Application Libraries folder and double-click the file vibration_control_with_amb_bearing.txt.
- 5 In the Label text field, type Parameters: Bearing.

Create some selections for later use.

DEFINITIONS

Motor Bearing I

- I In the **Definitions** toolbar, click **\(\frac{1}{2} \) Explicit**.
- 2 In the Settings window for Explicit, type Motor Bearing 1 in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Select the Group by continuous tangent check box.
- **5** Select Boundaries 85, 86, 89, and 91 only.
- 6 Right-click Motor Bearing I and choose Duplicate.

Motor Bearing 2

- I In the Model Builder window, under Component I (compl)>Definitions>Selections click Motor Bearing 1.1.
- 2 In the Settings window for Explicit, type Motor Bearing 2 in the Label text field.
- 3 Locate the Input Entities section. Click Clear Selection.
- **4** Select Boundaries 485, 486, 490, 493, 505, 506, 510, 513, 524, 525, 529, 532, 541, 542, 546, and 549 only.
- 5 Right-click Motor Bearing 2 and choose Duplicate.

External Bearing I

- I In the Model Builder window, click Motor Bearing 2.1.
- 2 In the Settings window for Explicit, locate the Input Entities section.
- 3 Click Clear Selection.
- 4 Select Boundaries 565–568 only.
- 5 In the Label text field, type External Bearing 1.
- 6 Right-click External Bearing I and choose Duplicate.

External Bearing 2

- I In the Model Builder window, under Component I (compl)>Definitions>Selections click External Bearing 1.1.
- 2 In the Settings window for Explicit, type External Bearing 2 in the Label text field.
- 3 Locate the Input Entities section. Click Clear Selection.
- 4 Select Boundaries 586–589 only.

Rotor

I In the **Definitions** toolbar, click **\(\) Explicit**.

- 2 In the Settings window for Explicit, locate the Input Entities section.
- 3 From the Geometric entity level list, choose Edge.
- **4** Select Edge 1341 only.
- 5 Select the Group by continuous tangent check box.
- 6 In the Label text field, type Rotor.

ADD MATERIAL

- I In the Home toolbar, click **Add Material** to open the **Add Material** window.
- 2 Go to the Add Material window.
- 3 In the tree, select Built-in>Structural steel.
- 4 Click Add to Global Materials in the window toolbar.
- 5 In the Home toolbar, click **‡** Add Material to close the Add Material window.

MATERIALS

Material Link I (matlnk I)

- I In the Model Builder window, under Component I (compl) right-click Materials and choose More Materials>Material Link.
- 2 Right-click Material Link I (matlnkI) and choose Duplicate.

Material Link 2 (matlnk2)

- I In the Model Builder window, click Material Link 2 (matlnk2).
- 2 In the Settings window for Material Link, locate the Geometric Entity Selection section.
- **3** From the **Geometric entity level** list, choose **Edge**.
- 4 From the Selection list, choose Rotor.

SOLID MECHANICS (SOLID)

Change the discretization to linear to reduce the model size and the computational time. If you want to capture the stresses accurately, you can continue with the default discretization.

- I In the Model Builder window, under Component I (compl) click Solid Mechanics (solid).
- 2 In the Settings window for Solid Mechanics, click to expand the Discretization section.
- 3 From the Displacement field list, choose Linear.

Fixed Constraint I

I In the Physics toolbar, click **Boundaries** and choose **Fixed Constraint**.

2 Select Boundaries 71, 76, 600, and 605 only.

Attachment: Motor Bearing 1

- I In the Physics toolbar, click **Boundaries** and choose **Attachment**.
- 2 In the Settings window for Attachment, locate the Boundary Selection section.
- 3 From the Selection list, choose Motor Bearing 1.
- 4 In the Label text field, type Attachment: Motor Bearing 1.
- 5 Right-click Attachment: Motor Bearing I and choose Duplicate.

Attachment: Motor Bearing 2

- I In the Model Builder window, click Attachment: Motor Bearing 1.1.
- 2 In the Settings window for Attachment, locate the Boundary Selection section.
- 3 From the Selection list, choose Motor Bearing 2.
- 4 In the Label text field, type Attachment: Motor Bearing 2.
- **5** Right-click **Attachment: Motor Bearing 2** and choose **Duplicate**.

Attachment: External Bearing 1

- I In the Model Builder window, under Component I (compl)>Solid Mechanics (solid) click Attachment: Motor Bearing 2.1.
- 2 In the Settings window for Attachment, type Attachment: External Bearing 1 in the Label text field.
- 3 Locate the Boundary Selection section. From the Selection list, choose External Bearing 1.
- 4 Right-click Attachment: External Bearing I and choose Duplicate.

Attachment: External Bearing 2

- I In the Model Builder window, under Component I (compl)>Solid Mechanics (solid) click Attachment: External Bearing 1.1.
- 2 In the Settings window for Attachment, type Attachment: External Bearing 2 in the Label text field
- 3 Locate the Boundary Selection section. From the Selection list, choose External Bearing 2.

BEAM ROTOR (ROTBM)

- I In the Model Builder window, under Component I (compl) click Beam Rotor (rotbm).
- 2 In the Settings window for Beam Rotor, locate the Edge Selection section.
- **3** From the **Selection** list, choose **Rotor**.
- 4 Locate the Rotor Speed section. In the text field, type 10000[rpm/s]*t.

Linear Elastic Material I

In the Model Builder window, under Component I (compl)>Beam Rotor (rotbm) click Linear Elastic Material I.

Dambing I

- I In the Physics toolbar, click 💂 Attributes and choose Damping.
- 2 In the Settings window for Damping, locate the Damping Settings section.
- **3** In the α_{dM} text field, type 7.
- **4** In the β_{dK} text field, type **3.2e-5**.

Rotor Cross Section I

- I In the Model Builder window, under Component I (compl)>Beam Rotor (rotbm) click Rotor Cross Section 1
- 2 In the Settings window for Rotor Cross Section, locate the Cross-Section Definition section.
- 3 In the d_0 text field, type ds1.
- 4 Right-click Component I (compl)>Beam Rotor (rotbm)>Rotor Cross Section I and choose **Duplicate**.

Rotor Cross Section 2

- I In the Model Builder window, click Rotor Cross Section 2.
- 2 In the Settings window for Rotor Cross Section, locate the Edge Selection section.
- 3 Click Clear Selection.
- 4 Select Edge 492 only.
- **5** Locate the Cross-Section Definition section. In the $d_{
 m o}$ text field, type dR.
- 6 Right-click Rotor Cross Section 2 and choose Duplicate.

Rotor Cross Section 3

- I In the Model Builder window, click Rotor Cross Section 3.
- 2 In the Settings window for Rotor Cross Section, locate the Edge Selection section.
- 3 Click Clear Selection.
- **4** Select Edges 1301, 1316, 1336, 1337, 1359, and 1374 only.
- **5** Locate the **Cross-Section Definition** section. In the d_0 text field, type ds2.
- 6 Right-click Rotor Cross Section 3 and choose Duplicate.

Rotor Cross Section 4

- I In the Model Builder window, click Rotor Cross Section 4.
- 2 In the Settings window for Rotor Cross Section, locate the Edge Selection section.

- 3 Click Clear Selection.
- 4 Select Edges 1328 and 1341 only.
- **5** Locate the **Cross-Section Definition** section. In the $d_{\mathbf{0}}$ text field, type ds3.
- 6 Right-click Rotor Cross Section 4 and choose Duplicate.

Rotor Cross Section 5

- I In the Model Builder window, click Rotor Cross Section 5.
- 2 In the Settings window for Rotor Cross Section, locate the Edge Selection section.
- 3 Click Clear Selection.
- 4 Select Edge 1394 only.
- **5** Locate the **Cross-Section Definition** section. In the d_0 text field, type ds11.

Disk: Motor Fan

- I In the Physics toolbar, click Points and choose Disk.
- 2 Select Point 63 only.
- 3 In the Settings window for Disk, type Disk: Motor Fan in the Label text field.
- **4** Locate the **Disk Properties** section. In the m text field, type mf.
- **5** In the I_p text field, type Ipf.
- **6** In the $I_{\rm d}$ text field, type Idf.

Disk 1: Rotor

- I In the Physics toolbar, click Points and choose Disk.
- 2 In the Settings window for Disk, type Disk 1: Rotor in the Label text field.
- **3** Select Point 785 only.
- 4 Locate the **Disk Properties** section. From the **Center of mass** list, choose **Offset from selected points**.
- 5 In the z_r text field, type e1.
- **6** In the *m* text field, type m.
- **7** In the I_p text field, type Ip.
- **8** In the $I_{\rm d}$ text field, type Id.
- 9 Right-click Disk I: Rotor and choose Duplicate.

Disk 2: Rotor

I In the Model Builder window, under Component I (compl)>Beam Rotor (rotbm) click Disk I: Rotor I.

- 2 In the Settings window for Disk, type Disk 2: Rotor in the Label text field.
- 3 Locate the Point Selection section. Click Clear Selection.
- **4** Select Point 786 only.
- **5** Locate the **Disk Properties** section. In the z_r text field, type **e2**.
- **6** In the ϕ text field, type pi/2.

Iournal Bearing I (Motor)

- I In the Physics toolbar, click Points and choose Journal Bearing.
- **2** Select Point 88 only.
- 3 In the Settings window for Journal Bearing, type Journal Bearing 1 (Motor) in the Label text field.
- 4 Locate the Bearing Properties section. From the Bearing model list, choose Total spring and damping constant.
- **5** In the \mathbf{k}_u table, enter the following settings:

kb	0
0	kb

6 In the \mathbf{c}_u table, enter the following settings:

cb	0
0	cb

- 7 Locate the Foundation Properties section. From the list, choose Attachment: Motor Bearing I (solid).
- 8 Right-click Journal Bearing I (Motor) and choose Duplicate.

Journal Bearing 2 (Motor)

- I In the Model Builder window, under Component I (compl)>Beam Rotor (rotbm) click Journal Bearing I (Motor) I.
- 2 In the Settings window for Journal Bearing, type Journal Bearing 2 (Motor) in the Label text field.
- 3 Locate the Point Selection section. Click Clear Selection.
- 4 Select Point 721 only.
- 5 Locate the Foundation Properties section. From the list, choose Attachment: Motor Bearing 2 (solid).
- 6 Right-click Journal Bearing 2 (Motor) and choose Duplicate.

Journal Bearing 3 (External)

- I In the Model Builder window, under Component I (compl)>Beam Rotor (rotbm) click Journal Bearing 2 (Motor) 1.
- 2 In the **Settings** window for **Journal Bearing**, type Journal Bearing 3 (External) in the **Label** text field.
- 3 Locate the Point Selection section. Click Clear Selection.
- 4 Select Point 765 only.
- 5 Locate the Foundation Properties section. From the list, choose Attachment: External Bearing I (solid).
- 6 Right-click Journal Bearing 3 (External) and choose Duplicate.

Journal Bearing 4 (External)

- I In the Model Builder window, under Component I (compl)>Beam Rotor (rotbm) click | Journal Bearing 3 (External) I.
- 2 In the Settings window for Journal Bearing, type Journal Bearing 4 (External) in the Label text field.
- 3 Locate the Point Selection section. Click Clear Selection.
- **4** Select Point 804 only.
- 5 Locate the Foundation Properties section. From the list, choose Attachment: External Bearing 2 (solid).

MESH I

Create a coarser mesh to reduce the problem size and the computational time. For better accuracy you can continue with the normal mesh.

- I In the Model Builder window, under Component I (compl) click Mesh I.
- 2 In the Settings window for Mesh, locate the Physics-Controlled Mesh section.
- 3 From the Element size list, choose Coarser.

STUDY I

Step 1: Time Dependent

- I In the Model Builder window, under Study I click Step I: Time Dependent.
- 2 In the Settings window for Time Dependent, locate the Study Settings section.
- 3 In the Output times text field, type range (0, 1e-3, 1.2).

Add a fully coupled solver and set the Newton solver to **Automatic Newton**.

Solution I (soll)

- I In the Study toolbar, click Show Default Solver.
- 2 In the Model Builder window, expand the Solution I (soll) node.
- 3 In the Model Builder window, under Study I>Solver Configurations>Solution I (soll) click Time-Dependent Solver I.
- 4 In the Settings window for Time-Dependent Solver, click to expand the Time Stepping section.
- 5 Right-click Study I>Solver Configurations>Solution I (soll)>Time-Dependent Solver I and choose Fully Coupled.
- 6 In the Settings window for Fully Coupled, click to expand the Method and Termination section.
- 7 From the Nonlinear method list, choose Automatic (Newton).
- 8 In the Model Builder window, click Study 1.
- 9 In the Settings window for Study, type Study: Without AMB in the Label text field.
- **10** In the **Study** toolbar, click **Compute**.

Stress in the solid rotor and beam rotor are the default plots. Follow the instructions below to combine both the plots together as shown in Figure 3.

RESULTS

Line 1

- I In the Model Builder window, expand the Results>Stress (rotbm) node.
- 2 Right-click Line I and choose Copy.

Stress (solid)

In the Model Builder window, under Results right-click Stress (solid) and choose Paste Line.

line l

- I In the Model Builder window, click Line I.
- 2 In the Settings window for Line, locate the Coloring and Style section.
- 3 Select the Radius scale factor check box. In the associated text field, type 1.
- 4 Click Change Color Table.
- 5 In the Color Table dialog box, select Aurora>AuroraAustralisDark in the tree.
- 6 Click OK.

Deformation

- I In the Model Builder window, expand the Line I node, then click Deformation.
- 2 In the Settings window for Deformation, locate the Scale section.
- 3 Select the Scale factor check box. In the associated text field, type 10.
- 4 In the Stress (solid) toolbar, click Plot.

Volume 1

The stator part is solved using linear elements. Thus, stresses within the elements are constant. Change the **Smoothing Threshold** to **None** to smoothen the stress.

- I In the Model Builder window, under Results>Stress (solid) click Volume 1.
- 2 In the Settings window for Volume, click to expand the Quality section.
- 3 From the Smoothing threshold list, choose None.

Deformation

- I In the Model Builder window, expand the Volume I node, then click Deformation.
- 2 In the Settings window for Deformation, locate the Scale section.
- **3** Select the **Scale factor** check box. In the associated text field, type 150.

Stress (solid)

- I In the Model Builder window, under Results click Stress (solid).
- 2 In the Settings window for 3D Plot Group, locate the Color Legend section.
- 3 From the Position list, choose Right double.
- 4 Locate the Data section. From the Time (s) list, choose 1.
- 5 Click the Go to Default View button in the Graphics toolbar.
- 6 Click the **Zoom Extents** button in the **Graphics** toolbar.
- 7 In the Stress (solid) toolbar, click Plot.

Plot the vertical displacement of the rotor at the coupling location using the instructions below.

Vertical Displacement at Coupling

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Vertical Displacement at Coupling in the Label text field.
- 3 Click to expand the **Title** section. From the **Title type** list, choose **Label**.
- 4 Locate the Legend section. From the Position list, choose Upper left.

Point Graph 1

- I Right-click Vertical Displacement at Coupling and choose Point Graph.
- **2** Select Point 748 only.
- 3 In the Settings window for Point Graph, locate the y-Axis Data section.
- 4 In the Expression text field, type w2.

Create a legend for the plot. This will later help in comparing this result with the result including active magnetic bearing.

- **5** Click to expand the **Legends** section. Select the **Show legends** check box.
- 6 From the Legends list, choose Manual.
- 7 In the table, enter the following settings:

Legends	
Without	AMB

Now add an active magnetic bearing in the model to control the resonant vibrations in the system. Start by creating a step function to activate the bearing just before the resonance.

DEFINITIONS

Steb | (steb|)

- I In the Home toolbar, click f(x) Functions and choose Local>Step.
- 2 In the Settings window for Step, locate the Parameters section.
- 3 In the Location text field, type 0.8[s].

BEAM ROTOR (ROTBM)

Active Magnetic Bearing I

- I In the Physics toolbar, click Points and choose Active Magnetic Bearing.
- **2** Select Point 783 only.
- 3 In the Settings window for Active Magnetic Bearing, locate the Air Gap section.
- **4** In the *h* table, enter the following settings:

gap gap **5** Locate the **Control Parameters** section. In the K_p table, enter the following settings:

Кр	0
0	Кр

6 In the K_i table, enter the following settings:

Ki	0
0	Κi

7 In the $K_{\rm d}$ table, enter the following settings:

Kd	0
0	Kd

8 In the $F_{\rm c}$ table, enter the following settings:

Fc*step1(t)	0
0	Fc*step1(t)

9 Locate the **Currents** section. In the $i_{\mathrm{b,p}}$ table, enter the following settings:

ibp	0
0	ibp

IO In the $i_{b,n}$ table, enter the following settings:

ibn	0
0	ibn

II In the i_{max} table, enter the following settings:

imax	0
0	imax

Add a new study to study the vibration response with active magnetic bearing. Use the same solver settings as in the first study.

ADD STUDY

- I In the Home toolbar, click Add Study to open the Add Study window.
- 2 Go to the Add Study window.

- 3 Find the Studies subsection. In the Select Study tree, select General Studies> Time Dependent.
- 4 Click Add Study in the window toolbar.
- 5 In the Home toolbar, click Add Study to close the Add Study window.

STUDY 2

Step 1: Time Dependent

- I In the Settings window for Time Dependent, locate the Study Settings section.
- 2 In the Output times text field, type range (0, 1e-3, 1.2).

Solution 2 (sol2)

- I In the Study toolbar, click Show Default Solver. Change the scaling of the control current variable in **Active Magnetic Bearing**.
- 2 In the Model Builder window, expand the Solution 2 (sol2) node.
- 3 In the Model Builder window, expand the Study 2>Solver Configurations> Solution 2 (sol2)>Dependent Variables I node, then click Control current (compl.rotbm.ambl.ic).
- 4 In the Settings window for Field, locate the Scaling section.
- 5 From the Method list, choose Manual.
- 6 In the Scale text field, type 10.
- 7 In the Model Builder window, expand the Study 2>Solver Configurations> Solution 2 (sol2)>Time-Dependent Solver I node.
- 8 Right-click Study 2>Solver Configurations>Solution 2 (sol2)>Time-Dependent Solver I and choose Fully Coupled.
- 9 In the Settings window for Fully Coupled, locate the Method and Termination section.
- 10 From the Nonlinear method list, choose Automatic (Newton).
- II In the Model Builder window, click Study 2.
- 12 In the Settings window for Study, type Study: With AMB in the Label text field.
- 13 In the Study toolbar, click **Compute**.

Follow the similar instructions again to combine the stress results from solid rotor and beam rotor for the second study. This plot is shown in Figure 4.

RESULTS

line l

- I In the Model Builder window, expand the Results>Stress (rotbm) I node.
- 2 Right-click Line I and choose Copy.

Stress (solid) 1

In the Model Builder window, under Results right-click Stress (solid) I and choose Paste Line.

line 1

- I In the Model Builder window, click Line I.
- 2 In the Settings window for Line, locate the Coloring and Style section.
- 3 Select the Radius scale factor check box. In the associated text field, type 1.
- 4 Click Change Color Table.
- 5 In the Color Table dialog box, select Aurora Aurora Australis Dark in the tree.
- 6 Click OK.

Deformation

- I In the Model Builder window, expand the Line I node, then click Deformation.
- 2 In the Settings window for Deformation, locate the Scale section.
- 3 Select the Scale factor check box. In the associated text field, type 10.

Volume 1

- I In the Model Builder window, under Results>Stress (solid) I click Volume I.
- 2 In the Settings window for Volume, click to expand the Quality section.
- 3 From the Smoothing threshold list, choose None.

Deformation

- I In the Model Builder window, expand the Volume I node, then click Deformation.
- 2 In the Settings window for Deformation, locate the Scale section.
- 3 Select the Scale factor check box. In the associated text field, type 150.

Stress (solid) I

- I In the Model Builder window, under Results click Stress (solid) I.
- 2 In the Settings window for 3D Plot Group, locate the Color Legend section.
- 3 From the Position list, choose Right double.
- 4 Locate the Data section. From the Time (s) list, choose 1.

- 5 Click the Go to Default View button in the Graphics toolbar.
- 6 Click the **Zoom Extents** button in the **Graphics** toolbar.
- 7 In the Stress (solid) I toolbar, click **Plot**.

Now you will compare the vibrations response with active magnetic bearing at the coupling location with the one without it. Start by duplicating the **Point Graph 1** in the Vertical Displacement at Coupling node and changing the solution and legend. The resulting plot is shown in Figure 5.

Point Graph 1

- I In the Model Builder window, expand the Results>Vertical Displacement at Coupling node.
- 2 Right-click Point Graph I and choose Duplicate.

Point Graph 2

- I In the Model Builder window, click Point Graph 2.
- 2 In the Settings window for Point Graph, locate the Data section.
- 3 From the Dataset list, choose Study: With AMB/Solution 2 (sol2).
- **4** Locate the **Legends** section. In the table, enter the following settings:

Legends With AMB

5 In the Vertical Displacement at Coupling toolbar, click **Plot**.

Duplicate the current plot and change the selection to compare the rotor displacement at the active magnetic bearing as shown in Figure 6.

Vertical Displacement at Coupling

In the Model Builder window, right-click Vertical Displacement at Coupling and choose Duplicate.

Vertical Displacement at AMB

- I In the Model Builder window, expand the Results>Vertical Displacement at Coupling I node, then click Vertical Displacement at Coupling 1.
- 2 In the Settings window for ID Plot Group, type Vertical Displacement at AMB in the Label text field.

Point Graph 1

- I In the Model Builder window, click Point Graph I.
- 2 In the Settings window for Point Graph, locate the Selection section.

- **3** Click to select the **Activate Selection** toggle button.
- 4 Click Clear Selection.
- **5** Select Point 783 only.

Point Graph 2

- I In the Model Builder window, click Point Graph 2.
- 2 In the Settings window for Point Graph, locate the Selection section.
- 3 Click Clear Selection.
- 4 Select Point 783 only.
- 5 In the Vertical Displacement at AMB toolbar, click Plot.

Follow the instructions below to plot the control current variation in the bearing as shown in Figure 7.

Control Currents

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, locate the Data section.
- 3 From the Dataset list, choose Study: With AMB/Solution 2 (sol2).
- 4 In the Label text field, type Control Currents.
- **5** Locate the **Title** section. From the **Title type** list, choose **Label**.
- 6 Locate the Plot Settings section.
- 7 Select the y-axis label check box. In the associated text field, type Control current (A).

Point Graph 1

- I Right-click Control Currents and choose Point Graph.
- **2** Select Point 783 only.
- 3 In the Settings window for Point Graph, locate the y-Axis Data section.
- 4 In the Expression text field, type rotbm.amb1.ic1.
- **5** Locate the **Legends** section. Select the **Show legends** check box.
- 6 From the Legends list, choose Manual.
- 7 In the table, enter the following settings:

Legends	
Local v	direction

8 Right-click Point Graph I and choose Duplicate.

Point Graph 2

- I In the Model Builder window, click Point Graph 2.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type rotbm.amb1.ic2.
- **4** Locate the **Legends** section. In the table, enter the following settings:

Legends Local z direction

5 In the Control Currents toolbar, click **Plot**.

Duplicate the **Control Currents** plot and follow the instructions below to plot the forces in the electromagnetic bearing as shown in Figure 8.

Control Currents

In the Model Builder window, right-click Control Currents and choose Duplicate.

Magnetic Bearing Force

- I In the Model Builder window, expand the Results>Control Currents I node, then click Control Currents 1.
- 2 In the Settings window for ID Plot Group, type Magnetic Bearing Force in the Label text field.
- 3 Locate the Plot Settings section. In the y-axis label text field, type Magnetic bearing force (N).
- 4 Locate the Legend section. From the Position list, choose Upper left.

Point Graph 1

- I In the Model Builder window, click Point Graph I.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type rotbm.amb1.F2.

Point Graph 2

- I In the Model Builder window, click Point Graph 2.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type rotbm.amb1.F3.
- 4 In the Magnetic Bearing Force toolbar, click Plot.

Finally, create an animation of the stress variation in the system for the second study using the following instructions.

Animation I

- I In the **Results** toolbar, click **Animation** and choose **Player**.
- 2 In the Settings window for Animation, locate the Scene section.
- 3 From the Subject list, choose Stress (solid) 1.
- 4 Locate the Animation Editing section. From the Time selection list, choose Manual.
- 5 In the Time indices (1-1201) text field, type range (1100, 1, 1201).
- 6 Locate the Frames section. In the Number of frames text field, type 50.
- 7 Click the Play button in the Graphics toolbar.

Disable the Active Magnetic Bearing in the first study to make sure that results in this study are not influenced by it even after rerunning the study.

STUDY: WITHOUT AMB

Step 1: Time Dependent

- I In the Model Builder window, expand the Study: Without AMB node, then click Step 1: Time Dependent.
- 2 In the Settings window for Time Dependent, locate the Physics and Variables Selection section.
- 3 Select the Modify model configuration for study step check box.
- 4 In the tree, select Component I (compl)>Beam Rotor (rotbm)>Active Magnetic Bearing I.
- **5** Right-click and choose **Disable**.