

Hyperelastic Seal

In this example you study the force-deflection relation of a car door seal made from a soft rubber material. The model uses a hyperelastic material together with formulations that can account for the large deformations and contact conditions. It is of special interest to investigate the effect of air confined within the seal.

See the Nonlinear Structural Materials Module User's Guide for theory about hyperelastic material.

Model Definition

The seal is compressed between a stationary plane surface and an indenting cylinder. There is also a vertical rigid wall at a distance of 1 mm from the initial position of the seal. Figure 1 shows the undeformed geometry of the seal and the contacting surfaces.

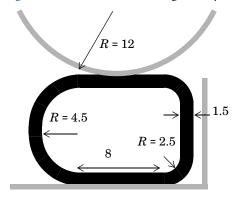


Figure 1: Model geometry.

The seal is modeled in 2D assuming plane strain conditions. The (arbitrary) thickness in the out-of-plane direction is 50 mm. The contacting surfaces are rigid when compared to the seal.

MATERIAL PROPERTIES

- The rubber is hyperelastic and modeled as a Mooney-Rivlin material with C_{10} = 0.37 MPa and C_{01} = 0.11 MPa. The material is almost incompressible, so the bulk modulus is set to 10^4 MPa. A mixed formulation is automatically used for this material model.
- The compression of the confined air is assumed to be adiabatic, giving the pressuredensity relation

$$\frac{p}{p_0} = \left(\frac{\rho}{\rho_0}\right)^{\gamma} = \left(\frac{A_0}{A}\right)^{\gamma}$$

Here, the undeformed and deformed cross-section areas are denoted by A_0 and A, respectively. The ratio of specific heat, γ , has the value 1.4 and $p_0 = 1$ atm is the standard air pressure. The load acting on the interior of the seal is then

$$\Delta p = p - p_0 = p_0 \left[\left(\frac{A_0}{A} \right)^{\gamma} - 1 \right]$$

CONSTRAINTS AND LOADS

- One contact pair is used between the cylinder and the seal.
- One contact pair is used between the stationary plates and the seal.
- The lower straight part of the seal is glued to the car body. This is modeled with an adhesion condition.
- The rigid cylinder is lowered using the parameter of the parametric continuation solver as the negative y displacement. It starts with a gap of 0 mm and is lowered 4 mm.

Figure 2 shows the deformed shape at the lowest cylinder position — corresponding to an indentation of 4 mm — without internal pressure. The deformation scale is 1:1, that is, a true shape. The plot shows a detachment region of significant size.

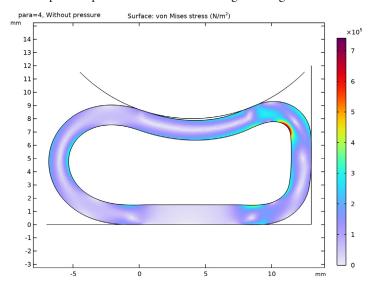


Figure 2: Seal deformation at 4 mm indentation without internal pressure.

Figure 3 shows the corresponding contact pressure plot. The detachment region appears first at an indentation just over 2.5 mm and grows as the indentation increases further. The actual contact areas are reduced to two spots at the sides.

Such a significant change in the contact pressure distribution indicates that the computations must be performed using a fine mesh together with sufficiently small steps in the parametric analysis with respect to the indentation value.

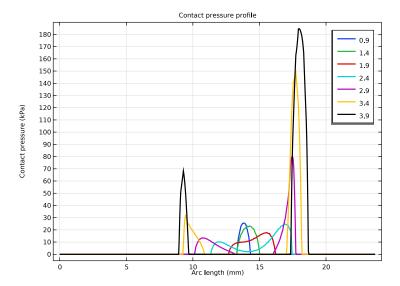


Figure 3: The contact pressure distribution over the area between the seal and cylinder for different indentations without internal pressure.

Figure 4 shows the result of the computations with the internal pressure taken into account. The seal profile appears inflated. The contact pressure plot in Figure 5 confirms that there is no detachment region even though the contact pressure has a pronounced minimum in the middle part.

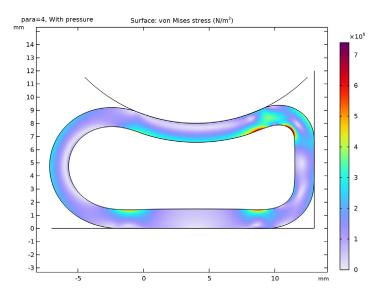
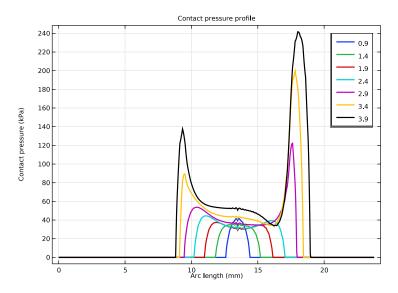


Figure 4: Seal deformation at 4 mm indentation with the internal pressure.



 $\label{thm:contact} \emph{Figure 5: The contact pressure distribution for different indentations with the internal pressure taken into account.}$

Figure 6 shows a plot of the force per unit length versus the indentation of the rigid cylinder, with and without the internal pressure taken into account. The distinct change in slope of the curves is attributed to the rightmost part of the seal coming into contact with the vertical wall, so that the seal can no longer deform in that direction.

Notice that the forces needed to compress the seal can be almost one order of magnitude larger when the effect of the confined air is taken into account.

In reality, a car door seal contains small holes through which the air can escape as long as the compression is not too fast. Thus, the computed values are the limits corresponding to very slow and very fast compression, respectively.

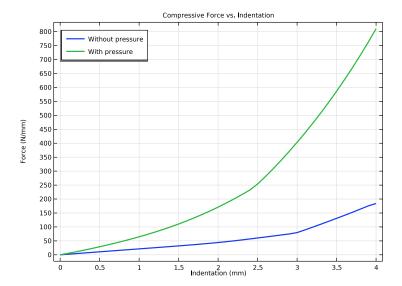


Figure 6: Compressive force per unit length versus indentation with and without internal

Notes About the COMSOL Implementation

The confined air inside the seal is modeled using the **Enclosed Cavity** feature, which automatically computes the undeformed and deformed volume (or cross-section area) enclosed by the inner seal boundaries with the divergence theorem. The Fluid subnode adds the pressure load resulting from the volume change, in this case assuming the adiabatic compression of air.

Application Library path: Nonlinear_Structural_Materials_Module/ Hyperelasticity/hyperelastic_seal

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click **2** 2D.
- 2 In the Select Physics tree, select Structural Mechanics>Solid Mechanics (solid).
- 3 Click Add.
- 4 Click Study.
- 5 In the Select Study tree, select General Studies>Stationary.
- 6 Click **Done**.

GEOMETRY I

If you do not want to build all the geometry, you can load the geometry sequence from the stored model. In the Model Builder window, under Component I (compl) right-click Geometry I and choose Insert Sequence. Browse to the model's Application Libraries folder and double-click the file hyperelastic seal.mph. You can then continue to the **Global Definitions** section below.

To build the geometry from scratch, continue from here.

- I In the Model Builder window, under Component I (compl) click Geometry I.
- 2 In the Settings window for Geometry, locate the Units section.
- 3 From the Length unit list, choose mm.

Rectangle I (rI)

- I In the **Geometry** toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Object Type section.
- **3** From the **Type** list, choose **Curve**.

- 4 Locate the Size and Shape section. In the Width text field, type 18.
- 5 In the Height text field, type 12.
- 6 Locate the Position section. In the x text field, type -6.
- 7 Click **Build Selected**.

Fillet | (fill)

- I In the **Geometry** toolbar, click **Fillet**.
- 2 On the object r1, select Points 1 and 4 only.
- 3 In the Settings window for Fillet, locate the Radius section.
- 4 In the Radius text field, type 6.
- 5 Click | Build Selected.

Fillet 2 (fil2)

- I In the **Geometry** toolbar, click **Fillet**.
- 2 On the object fill, select Points 4 and 5 only.
- 3 In the Settings window for Fillet, locate the Radius section.
- 4 In the Radius text field, type 4.
- 5 Click | Build Selected.

Thicken I (thil)

- I In the Geometry toolbar, click Conversions and choose Thicken.
- 2 Select the object fil2 only.
- 3 In the Settings window for Thicken, locate the Options section.
- 4 From the Offset list, choose Asymmetric.
- 5 In the **Upside thickness** text field, type 1.5.
- 6 Click **Build Selected**.

Create the indenter.

Indenter

- I In the Geometry toolbar, click Circle.
- 2 In the Settings window for Circle, type Indenter in the Label text field.
- 3 Locate the Size and Shape section. In the Radius text field, type 12.
- 4 In the Sector angle text field, type 90.
- **5** Locate the **Position** section. In the **x** text field, type 4.
- 6 In the y text field, type 24.

- 7 Locate the Rotation Angle section. In the Rotation text field, type -135.
- 8 Locate the Selections of Resulting Entities section. Select the Resulting objects selection check box.
- **9** From the Show in physics list, choose Boundary selection.

Create the support.

Rectangle 2 (r2)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type 20.
- 4 Locate the Position section. In the x text field, type -7.
- 5 In the y text field, type -1.
- 6 Locate the Selections of Resulting Entities section. Select the Resulting objects selection check box.
- 7 From the Show in physics list, choose Boundary selection.
- 8 Find the Cumulative selection subsection. Click New.
- 9 In the New Cumulative Selection dialog box, type Rigid base in the Name text field.
- IO Click OK.

Rectangle 3 (r3)

- I In the **Geometry** toolbar, click **Rectangle**.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Height** text field, type 12.
- 4 Locate the **Position** section. In the **x** text field, type 13.
- 5 Locate the Selections of Resulting Entities section. Find the Cumulative selection subsection. From the Contribute to list, choose Rigid base.

Convert to Curve I (ccurl)

- I In the Geometry toolbar, click Conversions and choose Convert to Curve.
- 2 Select the objects c1, r2, and r3 only.
- 3 In the Settings window for Convert to Curve, click | Build Selected.

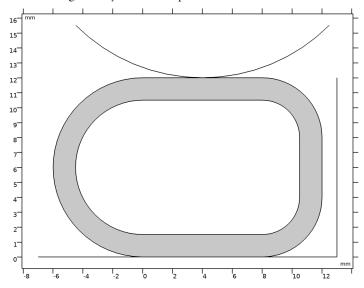
Delete Entities I (dell)

- I In the Model Builder window, right-click Geometry I and choose Delete Entities.
- 2 On the object ccurl, select Boundaries 1, 2, 4–6, and 8–10 only.

Form Union (fin)

- I In the Model Builder window, under Component I (compl)>Geometry I click Form Union (fin).
- 2 In the Settings window for Form Union/Assembly, locate the Form Union/Assembly section.
- 3 From the Action list, choose Form an assembly.
- 4 Clear the Create pairs check box.
- 5 Click | Build Selected.
- 6 Click the Zoom Extents button in the Graphics toolbar.

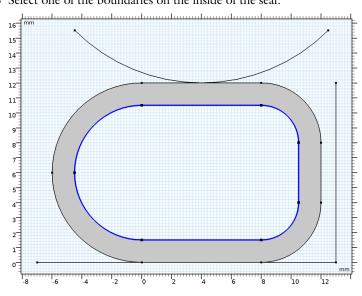
The model geometry is now complete.



Inner Seal Boundary

- I In the Geometry toolbar, click \(\frac{1}{2} \) Selections and choose Explicit Selection.
- 2 In the Settings window for Explicit Selection, type Inner Seal Boundary in the Label text field.
- 3 Locate the Entities to Select section. From the Geometric entity level list, choose Boundary.
- 4 Select the Group by continuous tangent check box.

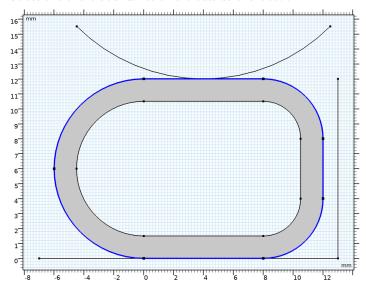
5 Select one of the boundaries on the inside of the seal.



Outer Seal Boundary

- I In the Geometry toolbar, click \(\frac{1}{2} \) Selections and choose Explicit Selection.
- 2 In the Settings window for Explicit Selection, type Outer Seal Boundary in the Label text field.
- 3 Locate the Entities to Select section. From the Geometric entity level list, choose Boundary.
- 4 Select the Group by continuous tangent check box.

5 Select one of the boundaries on the outside of the seal.



Glued Seal Boundary

- I In the Geometry toolbar, click \(\frac{1}{2} \) Selections and choose Explicit Selection.
- 2 In the Settings window for Explicit Selection, type Glued Seal Boundary in the Label text field.
- 3 Locate the Entities to Select section. From the Geometric entity level list, choose Boundary.
- 4 On the object fin, select Boundary 4 only.

Fillet I (fill), Fillet 2 (fil2), Rectangle I (rI), Thicken I (thil)

- I In the Model Builder window, under Component I (compl)>Geometry I, Ctrl-click to select Rectangle I (rI), Fillet I (fill), Fillet 2 (fil2), and Thicken I (thil).
- 2 Right-click and choose Group.

Seal

In the Settings window for Group, type Seal in the Label text field.

GLOBAL DEFINITIONS

Add a parameter that you can use to gradually increase the vertical displacement.

Parameters 1

I In the Model Builder window, under Global Definitions click Parameters I.

- 2 In the Settings window for Parameters, locate the Parameters section.
- **3** In the table, enter the following settings:

Name	Expression	Value	Description
para	0	0	Vertical displacement parameter
d	50[mm]	0.05 m	Out-of-plane thickness

DEFINITIONS

Contact Pair I (b1)

- I In the **Definitions** toolbar, click **Pairs** and choose **Contact Pair**.
- 2 In the Settings window for Pair, type upper in the Pair name text field.
- 3 Locate the Source Boundaries section. From the Selection list, choose Indenter.
- 4 Locate the Destination Boundaries section. From the Selection list, choose Outer Seal Boundary.

Contact Pair 2 (b2)

- I In the **Definitions** toolbar, click **Pairs** and choose **Contact Pair**.
- 2 In the Settings window for Pair, type lower in the Pair name text field.
- 3 Locate the Source Boundaries section. From the Selection list, choose Rigid base.
- 4 Locate the Destination Boundaries section. From the Selection list, choose Outer Seal Boundary.

The boundaries in the contact pairs are unnecessarily large because it was convenient to reuse existing selections. In large 3D models, you should however keep down the size of the contact boundaries for performance reasons.

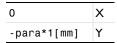
Since the indenter is only modeled as a rigid boundary, prescribe its deformation using a moving mesh. Alternatively, the indenter could be modeled as a rigid or elastic domain in the Solid Mechanics interface, in which case its deformation is prescribed in the interface.

COMPONENT I (COMPI)

Prescribed Deformation I

- I In the Physics toolbar, click Moving Mesh and choose Prescribed Deformation.
- 2 In the Settings window for Prescribed Deformation, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Boundary.

- 4 From the Selection list, choose Indenter.
- **5** Locate the **Prescribed Deformation** section. Specify the dx vector as



SOLID MECHANICS (SOLID)

- I In the Model Builder window, under Component I (compl) click Solid Mechanics (solid).
- 2 In the Settings window for Solid Mechanics, locate the Thickness section.
- **3** In the d text field, type d. In the plane strain approximation, this setting only affects total force computations.

Hyperelastic Material I

- I In the Physics toolbar, click **Domains** and choose Hyperelastic Material.
- 2 In the Settings window for Hyperelastic Material, locate the Domain Selection section.
- 3 From the Selection list, choose All domains.
- 4 Locate the Hyperelastic Material section. From the Material model list, choose Mooney— Rivlin, two parameters.
- **5** In the κ text field, type 1e4[MPa].

Contact I

In the Model Builder window, click Contact 1.

Friction I

- I In the Physics toolbar, click ___ Attributes and choose Friction.
- 2 In the Settings window for Friction, locate the Friction Parameters section.
- 3 In the μ text field, type 0.3.

Add an adhesion condition to model the glue layer at the bottom of the seal.

Contact I

In the Model Builder window, click Contact 1.

Adhesion I

- I In the Physics toolbar, click _ Attributes and choose Adhesion.
- 2 In the Settings window for Adhesion, locate the Adhesive Activation section.
- 3 From the Activation criterion list, choose User defined.
- 4 In the text field, type dom==4.

- 5 Locate the Adhesive Stiffness section. From the Adhesive stiffness list, choose User defined.
- **6** Specify the **k** vector as

1e10[N/m^3]	tl
2e10[N/m^3]	n

Add an **Enclosed Cavity** node to model the effect of air being compressed inside the seal.

Enclosed Cavity I

- I In the Physics toolbar, click Boundaries and choose Enclosed Cavity.
- 2 In the Settings window for Enclosed Cavity, locate the Boundary Selection section.
- 3 From the Selection list, choose Inner Seal Boundary.

Fluid 1

Inspect the Fluid subnode, which defines the properties of the gas being compressed. To study the effect of the confined air, add the Fluid node to a load group.

- I In the Model Builder window, click Fluid I.
- 2 In the Physics toolbar, click Load Group and choose New Load Group.

MATERIALS

Material I (mat I)

- I In the Model Builder window, under Component I (compl) right-click Materials and choose Blank Material.
- 2 In the Settings window for Material, locate the Material Contents section.
- **3** In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Model parameters	C10	0.37[MPa]	Pa	Mooney-Rivlin
Model parameters	C0I	0.11[MPa]	Pa	Mooney-Rivlin
Density	rho	1100[kg/m^3]	kg/m³	Basic

MESH I

Edge 1

- I In the Mesh toolbar, click \textstyle More Generators and choose Edge.
- 2 Select Boundaries 1–3 only.

Distribution I

- I Right-click **Edge I** and choose **Distribution**.
- 2 In the Settings window for Distribution, locate the Distribution section.
- 3 In the Number of elements text field, type 1.
- 4 Locate the Boundary Selection section. From the Selection list, choose Rigid base.

Distribution 2

- I In the Model Builder window, right-click Edge I and choose Distribution.
- 2 In the Settings window for Distribution, locate the Distribution section.
- 3 In the Number of elements text field, type 50.
- 4 Locate the Boundary Selection section. From the Selection list, choose Indenter.

Free Quad I

In the Mesh toolbar, click Free Quad.

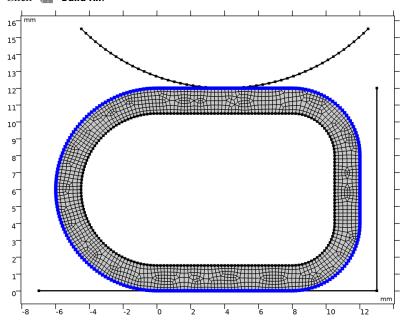
Size 1

- I Right-click Free Quad I and choose Size.
- 2 In the Settings window for Size, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Inner Seal Boundary.
- **5** Locate the **Element Size** section. Click the **Custom** button.
- **6** Locate the **Element Size Parameters** section.
- 7 Select the Maximum element size check box. In the associated text field, type 0.2.
- **8** Right-click **Size I** and choose **Duplicate**.

Size 2

- I In the Model Builder window, click Size 2.
- 2 In the Settings window for Size, locate the Geometric Entity Selection section.
- 3 From the Selection list, choose Outer Seal Boundary.

4 Click Build All.



STUDY I

Step 1: Stationary

Set up an auxiliary continuation sweep for the parameter para. Start at a nonzero value to avoid ill-conditioning during initiation of the contact.

- I In the Model Builder window, under Study I click Step I: Stationary.
- 2 In the Settings window for Stationary, click to expand the Study Extensions section.
- 3 Select the Auxiliary sweep check box.
- 4 Click + Add.
- **5** In the table, enter the following settings:

Parameter name	Parameter value list	Parameter unit
para (Vertical displacement parameter)	1e-3 range(0.1,0.1,4)	

Define two load cases to study the effect of the confined air.

- 6 Select the Define load cases check box.
- 7 Click + Add.

- 8 Click + Add.
- **9** In the table, enter the following settings:

Load case	lgl	Weight
Without pressure		1.0
With pressure	√	1.0

10 In the Study toolbar, click t=0 Get Initial Value.

Solver Configurations

In the Model Builder window, expand the Study I>Solver Configurations node.

Solution I (soll)

I In the Model Builder window, expand the Study I>Solver Configurations>Solution I (soll) node.

The default scale for the displacement variables is calculated from the entire geometry size. For models with prescribed displacements as domain or boundary constraints, the maximum prescribed displacement usually gives a better estimate of the scale.

- 2 In the Model Builder window, expand the Study I>Solver Configurations> Solution I (soll)>Dependent Variables I node, then click Displacement field (compl.u).
- 3 In the Settings window for Field, locate the Scaling section.
- 4 In the Scale text field, type 1e-3.

Change the scale for the auxiliary pressure to account for the material properties of the seal made of soft rubber.

- 5 In the Model Builder window, click Auxiliary pressure (compl.solid.hmml.pw).
- **6** In the **Settings** window for **Field**, locate the **Scaling** section.
- 7 In the Scale text field, type 1e5.
- 8 In the Model Builder window, expand the Study I>Solver Configurations> Solution I (soll)>Stationary Solver I node, then click Direct.
- **9** In the **Settings** window for **Direct**, locate the **General** section.
- 10 From the Solver list, choose PARDISO.

Step 1: Stationary

- I In the Model Builder window, under Study I click Step I: Stationary.
- 2 In the Settings window for Stationary, click to expand the Results While Solving section.
- **3** Select the **Plot** check box.
- 4 In the Study toolbar, click **Compute**.

5 Click the **Zoom Extents** button in the **Graphics** toolbar.

RESULTS

Stress (solid)

- choose Load case.
- 2 Click ► Plot First.
- 3 Click the **Zoom Extents** button in the **Graphics** toolbar.

The default plot shows the von Mises stress distribution in the seal. The case without confined air is shown in Figure 2.

Next, visualize the stress distribution with confined air.

4 Click → Plot Last.

You can see that the detachment region has disappeared as a result of the seal pressurization, compare with Figure 4.

Add a predefined plot to visualize the normal and tangential contact forces on the seal.

ADD PREDEFINED PLOT

- I In the Home toolbar, click Add Predefined Plot to open the Add Predefined Plot window.
- 2 Go to the Add Predefined Plot window.
- 3 In the tree, select Study I/Solution I (soll)>Solid Mechanics>Contact Forces (solid).
- 4 Click Add Plot in the window toolbar.
- 5 In the Home toolbar, click Add Predefined Plot to close the Add Predefined Plot window.

The following steps show how to display the contact pressure at the bottom of the seal.

RESULTS

Contact Pressure

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Contact Pressure in the Label text field.
- 3 Locate the Data section. From the Parameter selection (Load case) list, choose First.
- 4 From the Parameter selection (para) list, choose Manual.

- 5 In the Parameter indices (1-41) text field, type range (10,5,40).
- 6 Click to expand the **Title** section. From the **Title type** list, choose **Manual**.
- 7 In the **Title** text area, type Contact pressure profile.

Line Graph 1

- I Right-click Contact Pressure and choose Line Graph.
- **2** Select Boundaries 7, 11, and 17 only.
- 3 In the Settings window for Line Graph, click Replace Expression in the upper-right corner of the y-Axis Data section. From the menu, choose Component I (compl)> Solid Mechanics>Contact>solid.Tn - Contact pressure - N/m2.
- 4 Locate the y-Axis Data section. From the Unit list, choose kPa.
- **5** Click to expand the **Legends** section. Select the **Show legends** check box.
- **6** Find the **Include** subsection. Clear the **Solution** check box.
- 7 Find the Prefix and suffix subsection. In the Prefix text field, type eval (para).
- 8 Click to expand the Coloring and Style section. From the Width list, choose 2.
- **9** In the Contact Pressure toolbar, click **Plot**. The plot in the **Graphics** window should now look like that in Figure 3.

Contact Pressure

Next, plot the pressure profile when the internal air pressure is included.

- I In the Model Builder window, click Contact Pressure.
- 2 In the Settings window for ID Plot Group, locate the Data section.
- 3 From the Parameter selection (Load case) list, choose Last.
- 4 In the Contact Pressure toolbar, click **Plot**.

Finally, compute the force needed for the compression as the sum of all vertical reaction forces on the indenter.

Compressive Force vs. Indentation

- I In the Home toolbar, click **Add Plot Group** and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Compressive Force vs. Indentation in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Label.

Global I

I In the Compressive Force vs. Indentation toolbar, click (Global.

- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
-solid.dcnt1.T_toty_upper/d	N/m	

4 Click to expand the Coloring and Style section. From the Width list, choose 2.

Compressive Force vs. Indentation

- I In the Model Builder window, click Compressive Force vs. Indentation.
- 2 In the Settings window for ID Plot Group, locate the Plot Settings section.
- 3 Select the x-axis label check box.
- 4 Select the y-axis label check box.
- 5 In the x-axis label text field, type Indentation (mm).
- 6 In the y-axis label text field, type Force (N/mm).
- 7 Locate the Legend section. From the Position list, choose Upper left. Compare with the plot shown in Figure 6.