

# Vibrations of a Disk Backed by an Air-Filled Cylinder

The vibration modes of a thin or thick circular disc are well known, and it is possible to compute the corresponding eigenfrequencies with an arbitrary precision from a series solution. The same is true for the acoustic modes of an air-filled cylinder with perfectly rigid walls. A more interesting question to ask is: What happens if the cylinder is sealed in one end not by a rigid wall but by a thin disc? This is the question addressed in this tutorial.

**Note:** This application uses the Acoustic-Shell Interaction, Frequency Domain interface, which is available if you have both the Acoustics Module and the Structural Mechanics Module.

# Model Definition

The geometry is a rigid steel cylinder with a height of 255 mm and a radius of 38 mm. One end is welded to a heavy slab, while the other is sealed with a steel disc only 0.38 mm thick. The disc is modeled using shell elements with the outer edge of the disc fixed. The acoustics in the cylinder is described in terms of the acoustic (differential) pressure. The eigenvalue equation for the pressure is

$$-\Delta p = \frac{\omega^2}{c^2} p$$

where c is the speed of sound and  $\omega = 2\pi f$  defines the eigenfrequency, f.

A first step is to calculate the eigenfrequencies for the disc and the cylinder separately and compare them with theoretical values. This way you can verify the basic components of the model and assess the accuracy of the finite-element solution before modeling the coupled system. When computing the decoupled problem, the acoustic domain is completely surrounded by sound hard boundaries. In the coupled analysis, the boundary at the disc instead has the accelerations of the disc as boundary conditions. At the same time, the acoustic pressure supplies a load on the disc. Such a coupling is set up automatically by the Acoustic-Structure Boundary multiphysics coupling.

To be able to study the effects of the coupling, we first look at the solution of the uncoupled problem. Figure 1 through Figure 4 show the two first uncoupled structural and acoustic modes.

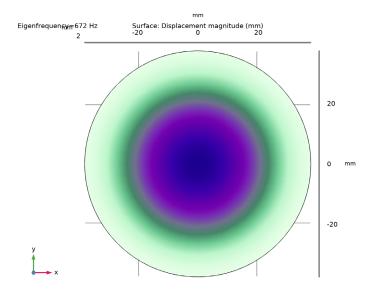


Figure 1: First structural mode displayed by displacement magnitude.

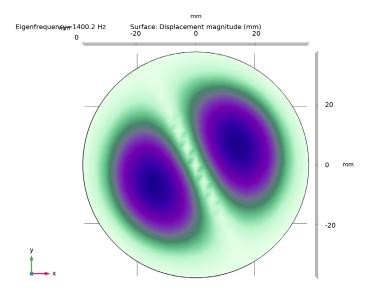


Figure 2: Second structural mode displayed by displacement magnitude..

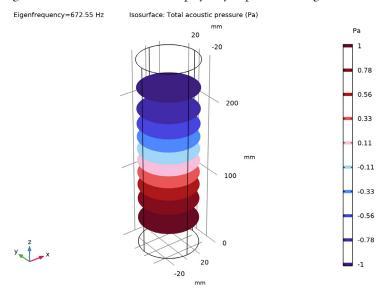


Figure 3: First acoustic mode displayed by pressure isosurfaces.

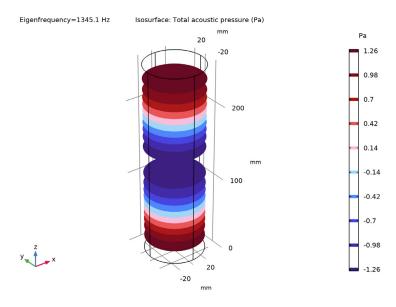


Figure 4: Second acoustic mode displayed by pressure isosurfaces.

In Ref. 1, D.G. Gorman and others have thoroughly investigated the coupled model at hand, and they have developed a semi-analytical solution verified by experiments. Their results for the coupled problem are presented in Table 1, together with the computed results from the COMSOL Multiphysics analysis.

TABLE I: RESULTS FROM SEMI-ANALYTICAL AND COMSOL MULTIPHYSICS ANALYSIS AND EXPERIMENTAL DATA.

Dominated by	Semi-analytical (Hz)	Computed (Hz)	Experimental (Hz)
str/ac	636.9	637.I	630
str/ac	707.7	707.6	685
ac	1347	1347	1348
str	1394	1396	1376
ac	2018	2019	2040
str	2289	2292/2304	2170
str/ac	2607	2623	2596
ac	2645	2646	_
str/ac	2697	2697	2689

TABLE I: RESULTS FROM SEMI-ANALYTICAL AND COMSOL MULTIPHYSICS ANALYSIS AND EXPERIMENTAL DATA.

Dominated by	Semi-analytical (Hz)	Computed (Hz)	Experimental (Hz)
ac	2730	2730	2756
ac	2968	2968	2971

As the table shows, the computed eigenfrequencies are in good agreement with both the theoretical predictions and the experimentally measured values. The table also states whether the modes are structurally dominated (str), acoustically dominated (ac), or tightly coupled (str/ac). The eigenfrequency precision is generally better for the acoustically dominated modes.

Most of the modes show rather weak coupling between the structural bending of the disc and the pressure field in the cylinder. It is, however, interesting to note that some of the uncoupled modes have been split into one covibrating and one contravibrating mode with distinct eigenfrequencies. This is, for example, the case for modes 1 and 2 in the FEM solution.

In Figure 5, the first coupled mode is shown in terms of disc displacements and air pressure. The coupling effect can be clearly displayed using a plot of pressure gradients, as in Figure 6.

Eigenfrequency=637.11 Hz Surface: Displacement magnitude (mm) Slice: Total acoustic pressure (Pa)

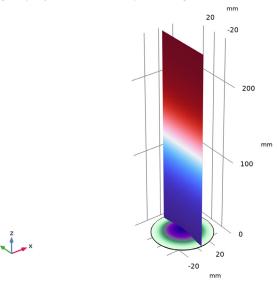


Figure 5: Disc deformation and pressure slice plot for the first coupled mode.

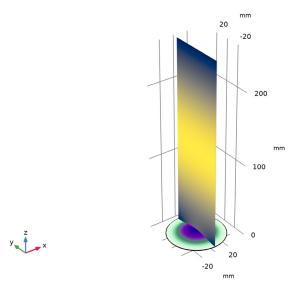


Figure 6: Disc deformation and pressure gradient slice plot for the first coupled mode.

# Reference

1. D.G. Gorman, J.M. Reese, J. Horacek, and D. Dedouch, "Vibration Analysis of a Circular Disk Backed by a Cylindrical Cavity," Proc. Instn. Mech. Engrs., Part C, vol. 215, no. 11, pp. 1303-1311. 2001.

# Notes About the COMSOL Implementation

Specify the part of the physics for which to compute the uncoupled eigenvalues by selecting the physics interface in the eigenfrequency study.

This model uses the Acoustic-Shell Interaction, Frequency Domain interface. If you decide to couple Pressure Acoustics, Frequency Domain to Shell manually, be careful when selecting the sign of the coupling terms so that they act in the intended direction. You should specify the acceleration in the inward normal direction for the pressure acoustics domain, which in this case is the positive z-acceleration of the disc. The acceleration is denoted wtt as it is the second time-derivative of the variable w. The pressure on the shell can be given using global directions, so that a positive pressure acts as a face load in the negative z direction. This is handled automatically by the Acoustic-Structure Boundary multiphysics coupling.

Application Library path: Acoustics Module/Verification Examples/ coupled vibrations

# Modeling Instructions

From the File menu, choose New.

#### NEW

In the New window, click Model Wizard.

# MODEL WIZARD

- I In the Model Wizard window, click **3D**.
- 2 In the Select Physics tree, select Acoustics>Acoustic-Structure Interaction>Acoustic-Shell Interaction, Frequency Domain.
- 3 Click Add.
- 4 Click 🔁 Study.
- 5 In the Select Study tree, select General Studies>Eigenfrequency.
- 6 Click M Done.

#### GEOMETRY I

- I In the Model Builder window, under Component I (compl) click Geometry I.
- 2 In the Settings window for Geometry, locate the Units section.
- 3 From the Length unit list, choose mm.

## Cylinder I (cyl1)

- I In the Geometry toolbar, click ( Cylinder.
- 2 In the Settings window for Cylinder, locate the Size and Shape section.
- 3 In the Radius text field, type 38.
- 4 In the Height text field, type 255.
- 5 Click | Build Selected.

## SHELL (SHELL)

- I In the Model Builder window, under Component I (compl) click Shell (shell).
- 2 In the Settings window for Shell, locate the Boundary Selection section.
- 3 Click Clear Selection.
- 4 Select Boundary 3 only.

# Thickness and Offset I

- I In the Model Builder window, under Component I (compl)>Shell (shell) click Thickness and Offset 1.
- 2 In the Settings window for Thickness and Offset, locate the Thickness and Offset section.
- **3** In the  $d_0$  text field, type 0.38[mm].

#### Fixed Constraint I

- I In the Physics toolbar, click **Edges** and choose **Fixed Constraint**.
- 2 Select Edges 2, 3, 7, and 10 only.

#### MATERIALS

#### Acoustic Material

- I In the Model Builder window, under Component I (comp I) right-click Materials and choose Blank Material.
- 2 In the Settings window for Material, type Acoustic Material in the Label text field.
- 3 Locate the Material Contents section. In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Density	rho	1.2	kg/m³	Basic
Speed of sound	С	343	m/s	Basic

#### Structural Material

- I Right-click Materials and choose Blank Material.
- 2 In the Settings window for Material, type Structural Material in the Label text field.
- 3 Locate the Geometric Entity Selection section. From the Geometric entity level list, choose **Boundary**.
- 4 Select Boundary 3 only.

**5** Locate the **Material Contents** section. In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Young's modulus	E	210[GPa]	Pa	Young's modulus and Poisson's ratio
Poisson's ratio	nu	0.3	I	Young's modulus and Poisson's ratio
Density	rho	7800	kg/m³	Basic

#### MESH I

In this model, the mesh is set up manually. Proceed by directly adding the desired mesh component.

# Free Quad I

- I In the Mesh toolbar, click A More Generators and choose Free Quad.
- 2 Select Boundary 3 only.

## Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- **3** Click the **Custom** button.
- 4 Locate the Element Size Parameters section. In the Maximum element size text field, type

In general, 5 to 6 second-order elements per wavelength are needed to resolve the waves. For more details, see Meshing (Resolving the Waves) in the Acoustics Module User's Guide. The mesh size in this model corresponds to resolving the wavelength with about 9 elements at 4000 Hz, the maximum eigenfrequency studied in the model. The wavelength in air at 4000 Hz is 8.6 cm.

5 Click Build Selected.

# Free Quad I

In the Model Builder window, right-click Free Quad I and choose Build Selected.

#### Swebt I

- I In the Mesh toolbar, click A Swept.
- 2 In the Settings window for Swept, click | Build Selected.

## STRUCTURAL ANALYSIS

In the first study, you solve only the structural problem.

- I In the Model Builder window, click Study I.
- 2 In the Settings window for Study, type Structural Analysis in the Label text field.

Step 1: Eigenfrequency

- I In the Model Builder window, under Structural Analysis click Step 1: Eigenfrequency.
- 2 In the Settings window for Eigenfrequency, locate the Study Settings section.
- 3 Select the Desired number of eigenfrequencies check box. In the associated text field, type 20.
- 4 In the Search for eigenfrequencies around shift text field, type 500.
- 5 Locate the Physics and Variables Selection section. In the table, clear the Solve for check box for Pressure Acoustics, Frequency Domain (acpr).
- 6 In the Home toolbar, click **Compute**.

#### RESULTS

Mode Shape, Structural Analysis

In the **Settings** window for **3D Plot Group**, type Mode Shape, Structural Analysis in the **Label** text field.

Surface I

- I Click the  $\int_{-\infty}^{\infty} xy$  Go to XY View button in the Graphics toolbar.
- 2 Click the **Scene Light** button in the **Graphics** toolbar.

Mode Shape, Structural Analysis

- I In the Model Builder window, expand the Mode Shape, Structural Analysis node, then click Results>Mode Shape, Structural Analysis.
- 2 In the Settings window for 3D Plot Group, locate the Data section.
- 3 From the Eigenfrequency (Hz) list, choose 1400.2.
- 4 In the Mode Shape, Structural Analysis toolbar, click Plot.

#### ROOT

Add a second study to solve the acoustics problem only.

#### ADD STUDY

- I In the Home toolbar, click Add Study to open the Add Study window.
- 2 Go to the Add Study window.
- 3 Find the Studies subsection. In the Select Study tree, select General Studies> Eigenfrequency.

- 4 Click Add Study in the window toolbar.
- 5 In the Home toolbar, click Add Study to close the Add Study window.

#### **ACOUSTICS ANALYSIS**

- I In the Model Builder window, click Study 2.
- 2 In the Settings window for Study, type Acoustics Analysis in the Label text field.

#### Step 1: Eigenfrequency

- I In the Model Builder window, under Acoustics Analysis click Step 1: Eigenfrequency.
- 2 In the Settings window for Eigenfrequency, locate the Study Settings section.
- 3 Select the **Desired number of eigenfrequencies** check box. In the associated text field, type 20.
- 4 In the Search for eigenfrequencies around shift text field, type 500.
- 5 From the Search method around shift list, choose Larger real part.
- 6 Locate the Physics and Variables Selection section. In the table, clear the Solve for check box for Shell (shell).
- 7 In the Home toolbar, click **Compute**.

#### RESULTS

Acoustic Pressure, Acoustics Analysis, Isosurfaces

- I In the Model Builder window, under Results click Acoustic Pressure, Isosurfaces (acpr).
- 2 In the Settings window for 3D Plot Group, type Acoustic Pressure, Acoustics Analysis, Isosurfaces in the Label text field.
- 3 Locate the Data section. From the Eigenfrequency (Hz) list, choose 672.55.
- 4 In the Acoustic Pressure, Acoustics Analysis, Isosurfaces toolbar, click Plot.
- 5 Click the Go to Default View button in the Graphics toolbar.
- 6 In the Acoustic Pressure, Acoustics Analysis, Isosurfaces toolbar, click Plot.
- 7 From the Eigenfrequency (Hz) list, choose 1345.1.
- 8 In the Acoustic Pressure, Acoustics Analysis, Isosurfaces toolbar, click Plot.

# ROOT

Finally, add a third study for the coupled problem.

#### ADD STUDY

I In the Home toolbar, click Add Study to open the Add Study window.

- **2** Go to the **Add Study** window.
- 3 Find the Studies subsection. In the Select Study tree, select General Studies> Eigenfrequency.
- 4 Click Add Study in the window toolbar.
- 5 In the Home toolbar, click Add Study to close the Add Study window.

#### COUPLED ANALYSIS

- I In the Model Builder window, click Study 3.
- 2 In the Settings window for Study, type Coupled Analysis in the Label text field.

# Step 1: Eigenfrequency

- I In the Model Builder window, under Coupled Analysis click Step 1: Eigenfrequency.
- 2 In the Settings window for Eigenfrequency, locate the Study Settings section.
- 3 Select the Desired number of eigenfrequencies check box. In the associated text field, type 20.
- 4 In the Search for eigenfrequencies around shift text field, type 500.
- 5 From the Search method around shift list, choose Larger real part.
- 6 In the Home toolbar, click **Compute**.

# RESULTS

Mode Shape, Coupled Analysis

- I In the Model Builder window, under Results click Mode Shape (shell).
- 2 In the Settings window for 3D Plot Group, type Mode Shape, Coupled Analysis in the Label text field.
- 3 Locate the Data section. From the Eigenfrequency (Hz) list, choose 637.11.
- 4 In the Mode Shape, Coupled Analysis toolbar, click  **Plot**.

#### Slice 1

- I In the Model Builder window, expand the Mode Shape, Coupled Analysis node.
- 2 Right-click Mode Shape, Coupled Analysis and choose Slice.
- 3 In the Settings window for Slice, locate the Data section.
- 4 From the Dataset list, choose Coupled Analysis/Solution 3 (sol3).
- 5 From the Solution parameters list, choose From parent.
- 6 Locate the Plane Data section. In the Planes text field, type 1.
- 7 In the Mode Shape, Coupled Analysis toolbar, click Plot.

- 8 Locate the Coloring and Style section. Click Change Color Table.
- 9 In the Color Table dialog box, select Wave>Wave in the tree.
- 10 Click OK.
- II In the Mode Shape, Coupled Analysis toolbar, click Plot.
- 12 Click the **Zoom Extents** button in the **Graphics** toolbar.

Plot the pressure gradient to display the connection to the disk shape.

Mode Shape, Coupled Analysis

In the Model Builder window, right-click Mode Shape, Coupled Analysis and choose Duplicate.

# Coupling Effect

- I In the Model Builder window, under Results click Mode Shape, Coupled Analysis I.
- 2 In the Settings window for 3D Plot Group, type Coupling Effect in the Label text field.

#### Slice 1

- I In the Model Builder window, expand the Coupling Effect node, then click Slice I.
- 2 In the Settings window for Slice, locate the Expression section.
- **3** In the **Expression** text field, type pz.
- 4 In the **Unit** field, type Pa/m.
- 5 Locate the Coloring and Style section. From the Scale list, choose Linear.
- 6 Click Change Color Table.
- 7 In the Color Table dialog box, select Linear>Cividis in the tree.
- 8 Click OK.
- I In the Model Builder window, click Slice I.
- 2 In the Coupling Effect toolbar, click Plot.

Notice the Eigenfrequency evaluation groups located last under the Results node. These are automatically created when an eigenfrequency analysis is carried out. The generated table shows the eigenfrequency, angular frequency, damping ratio, and Q factor.

# Eigenfrequencies (Coupled Analysis)

- I In the Model Builder window, under Results click Eigenfrequencies (Coupled Analysis).
- 2 In the Eigenfrequencies (Coupled Analysis) toolbar, click **=** Evaluate.

The final instruction steps are optional. Here you rename the remaining of the plot groups to reflect which study and physics they refer to. It is always good modeling practice to give plots and plot groups proper names. This simplifies debugging models and gives a better overview.

Acoustic Pressure, Acoustics Analysis

- I In the Model Builder window, under Results click Acoustic Pressure (acpr).
- 2 In the Settings window for 3D Plot Group, type Acoustic Pressure, Acoustics Analysis in the Label text field.

Sound Pressure Level, Acoustics Analysis

- I In the Model Builder window, under Results click Sound Pressure Level (acpr).
- 2 In the Settings window for 3D Plot Group, type Sound Pressure Level, Acoustics Analysis in the Label text field.

Acoustic Pressure, Coupled Analysis

- I In the Model Builder window, under Results click Acoustic Pressure (acpr) 1.
- 2 In the Settings window for 3D Plot Group, type Acoustic Pressure, Coupled Analysis in the Label text field.

Sound Pressure Level, Coupled Analysis

- I In the Model Builder window, under Results click Sound Pressure Level (acpr) 1.
- 2 In the Settings window for 3D Plot Group, type Sound Pressure Level, Coupled Analysis in the **Label** text field.

Acoustic Pressure, Coupled Analysis, Isosurfaces

- I In the Model Builder window, under Results click Acoustic Pressure, Isosurfaces (acpr).
- 2 In the Settings window for 3D Plot Group, type Acoustic Pressure, Coupled Analysis, Isosurfaces in the Label text field.