

# Optimization of a Tesla Microvalve

A Tesla valve inhibits backward flow on a fixed geometry by utilizing friction forces instead of moving parts, Ref. 1. This means fluid can flow freely in one direction but not in the reverse direction. Typically the Reynolds number of the flow in microfluidics is up to 100.

# Model Definition

The model solves two instances of the Navier-Stokes equations, one for the forward flow and one for the reverse. The Reynolds number is 100 in this example. A measure of the effectiveness of the design is the ratio of the pressure drop between the inlet and outlet for the forward and reverse flow. The pressure drop is defined as:

$$\Delta p_{\text{forward}} = \int_{\text{inlet}} pdS - \int_{\text{outlet}} pdS$$

For the reverse flow the same expression is used, except the inlet and outlet correspond to different boundaries in the model. The ratio of the pressure drop between the reverse and forward flow is then:

$$D_i = \frac{\Delta p_{\text{backward}}}{\Delta p_{\text{forward}}}$$

An alternative expression based on the dissipation is used for the optimization problem, so that the effect of the in- and outlet boxes can be disregarded. The energy dissipation is:

$$\phi = \int_{\Omega} (\tau : \mathbf{S} + \alpha (\mathbf{u} \cdot \mathbf{u})) dV$$

where  $\tau$  is the viscous stress and **S** is the strain rate tensor:

$$\mathbf{S} = \frac{1}{2} (\nabla \mathbf{u} + (\nabla \mathbf{u})^T)$$

The fluid flow is described by the Navier-Stokes equations:

$$\rho(\mathbf{u} \cdot \nabla)\mathbf{u} = -\nabla p + \nabla \cdot \eta(\nabla \mathbf{u} + (\nabla \mathbf{u})^T) - \alpha(\theta)\mathbf{u}$$

$$\nabla \cdot \mathbf{u} = 0$$

where the coefficient  $\alpha(\theta)$  depends on the distribution of material which impedes the flow within the device. In this example,  $\alpha(\theta)$  is given by:

$$\begin{split} \alpha(\theta_{\rm p}) &= \alpha_0 \theta_{\rm p}, \quad \theta_{\rm p} = \frac{\alpha_0 (q + \theta)}{q + \theta} \\ \theta &= \frac{(\tanh(\beta(\theta_{\rm f} - \theta_{\beta})) + \tanh(\beta\theta_{\beta}))}{(\tanh(\beta(1 - \theta_{\beta})) + \tanh(\beta\theta_{\beta}))} \\ \theta_{\rm f} &= R_{\rm min}^2 \nabla^2 \theta_{\rm f} + \theta_{\rm c} \end{split}$$

where  $\theta_c$  and  $\theta_f$  are the control- and filtered material volume factors. To avoid the effect of grayscale, the filtered field is projected to construct the material volume factor,  $\theta$ . Finally, the damping term,  $\alpha(\theta_p)$ , is expressed in terms of the maximum damping,  $\alpha_0$ , and the penalized material volume factor,  $\theta_p$ . This model involves fluid flow, so a convex function is used for the interpolation; see Ref. 3. The scheme avoids grayscale and spurious holes; see Figure 1.

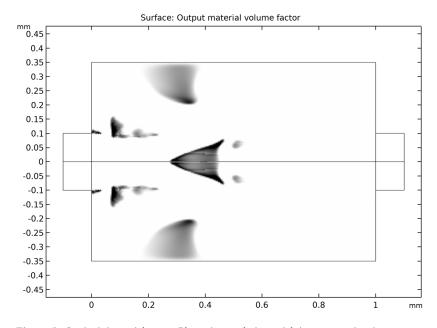


Figure 1: Optimizing without a filter gives a design with interpretation issues.

# Results and Discussion

As expected, the forward and reverse flow are identical but in opposite directions for the initial topology; see Figure 2 and Figure 3. The forward flow in the optimized design after 200 iterations can be seen in Figure 4. Material has been added close to the outlet in a

triangular shape that makes the forward flow bend smoothly around it. This smooth diversion of the flow from the point of impingement results in an overall low pressure drop between the inlet and the outlet. The reverse flow, shown in Figure 5, is far more interesting. The triangular shaped obstacle has a flat edge normal to the incident fluid, which means the velocity is redirected upward and downward toward the exterior walls. The redirected flow is then directed toward additional obstacles that further impede the flow path.

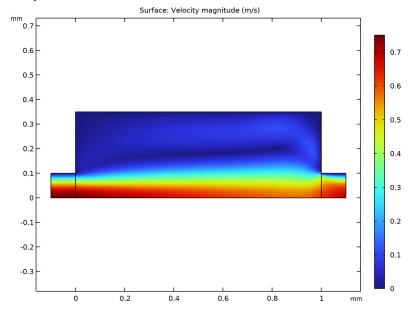
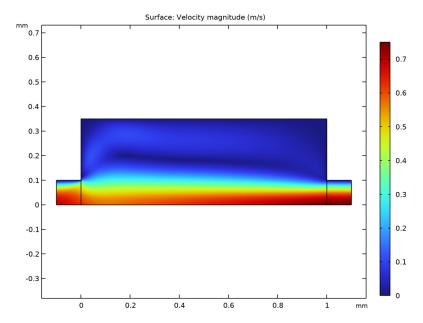


Figure 2: Forward flow velocity field, initial geometry.



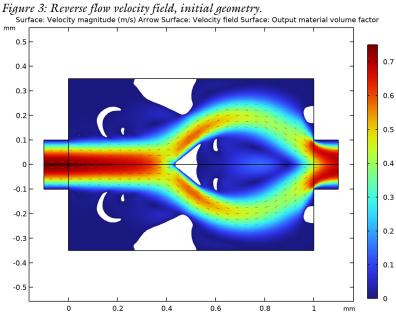


Figure 4: Forward flow, optimized topology.

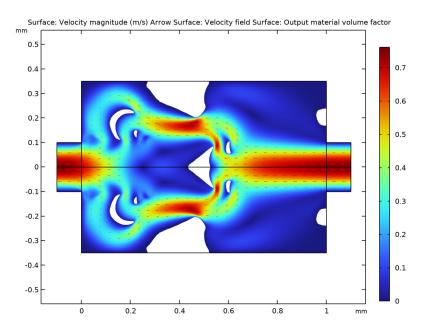


Figure 5: Reverse flow, optimized topology.

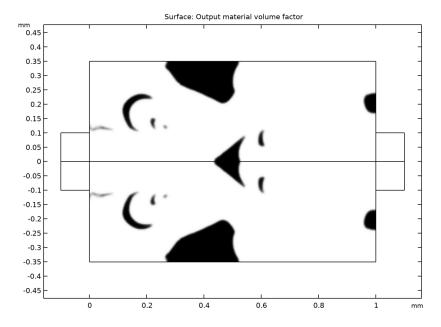


Figure 6: Control variable after optimization.

Finally, the optimized results is transferred to a new component for verification. This results in a larger diodicity.

# References

- 1. S. Lin, "Topology Optimization of Micro Tesla Valve in low and moderate Reynolds number," Chinese Academy of Sciences, China, September 27, 2011.
- 2. L. Højgaard Olesen, F. Okkels, and H. Bruus, "A high-level programming-language implementation of topology optimization applied to steady-state Navier-Stokes flow," Int. J. Numer. Methods Eng., vol. 65, pp. 975-1001, 2006.
- 3. T. Borrvall and J. Petersson, "Topology optimization of fluids in Stokes flow," Int. J. Numer. Meth. Fluid, vol. 41, pp. 77-107, 2003.

# Notes About the COMSOL Implementation

The model is set up using two Laminar Flow interfaces, one for the forward flow and one for the reverse. The original MMA optimization solver (which is not globally convergent) is used.

Application Library path: Microfluidics Module/Fluid Flow/ tesla microvalve optimization

# Modeling Instructions

From the File menu, choose New.

In the New window, click Model Wizard.

#### MODEL WIZARD

- I In the Model Wizard window, click **2** 2D.
- 2 In the Select Physics tree, select Fluid Flow>Single-Phase Flow>Laminar Flow (spf). Add two Laminar Flow interfaces for the forward and backward flow, respectively.
- 3 Click Add.
- 4 In the Select Physics tree, select Fluid Flow>Single-Phase Flow>Laminar Flow (spf).
- 5 Click Add.
- 6 Click Study.
- 7 In the Select Study tree, select General Studies>Stationary.
- 8 Click **Done**.

#### **GEOMETRY I**

The geometry is on the order of millimeters, so change the geometry unit.

- I In the Model Builder window, under Component I (compl) click Geometry I.
- 2 In the Settings window for Geometry, locate the Units section.
- 3 From the Length unit list, choose mm.

Add some parameters for computing the average inlet velocity for the flow, based on a chosen Reynolds number, in this case 100.

#### **GLOBAL DEFINITIONS**

#### Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- **3** In the table, enter the following settings:

Name	Expression	Value	Description
Re	100	100	Reynolds number
D	0.2[mm]	2E-4 m	Characteristic dimension
L	5*D	0.001 m	Length of channel
Н	1.75*D	3.5E-4 m	Width of channel
mu0	1E-3[Pa*s]	0.001 Pa·s	Dynamic viscosity
rho0	1E3[kg/m^3]	1000 kg/m³	Density
Uin	Re*muO/(rhoO*D)	0.5 m/s	Average inlet velocity
meshsz	0.005*L	5E-6 m	Mesh size

#### GEOMETRY I

Rectang	le 1	(rl)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type L.
- 4 In the **Height** text field, type H.

# Rectangle 2 (r2)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Width** text field, type D/2.
- 4 In the Height text field, type D/2.
- **5** Locate the **Position** section. In the **x** text field, type -D/2.
- 6 Click Pauld Selected.

# Rectangle 3 (r3)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.

- 3 In the Width text field, type D/2.
- 4 In the Height text field, type D/2.
- **5** Locate the **Position** section. In the **x** text field, type L.

# Symmetry

- I In the Geometry toolbar, click \( \frac{1}{2} \) Selections and choose Box Selection.
- 2 In the Settings window for Box Selection, type Symmetry in the Label text field.
- 3 Locate the Geometric Entity Level section. From the Level list, choose Boundary.
- 4 Locate the Box Limits section. In the y maximum text field, type 1e3\*eps.
- 5 Locate the Output Entities section. From the Include entity if list, choose Entity inside box.

#### Left

- I In the Geometry toolbar, click \( \frac{1}{2} \) Selections and choose Box Selection.
- 2 In the Settings window for Box Selection, type Left in the Label text field.
- 3 Locate the Geometric Entity Level section. From the Level list, choose Boundary.
- 4 Locate the Box Limits section. In the x maximum text field, type -D/2+1e3\*eps.
- 5 Locate the Output Entities section. From the Include entity if list, choose Entity inside box.

#### Right

- I In the Geometry toolbar, click Selections and choose Box Selection.
- 2 In the Settings window for Box Selection, type Right in the Label text field.
- 3 Locate the Geometric Entity Level section. From the Level list, choose Boundary.
- 4 Locate the Box Limits section. In the x minimum text field, type L+D/2-1e3\*eps.
- 5 Locate the Output Entities section. From the Include entity if list, choose Entity inside box.
- 6 Click **Build All Objects**.
- 7 Click the **Zoom Extents** button in the **Graphics** toolbar.

#### **GLOBAL DEFINITIONS**

Add a blank material for rho0 and mu0.

#### Material I (mat I)

I In the Model Builder window, under Global Definitions right-click Materials and choose Blank Material.

- 2 In the Materials toolbar, click User-Defined Property Group.
- 3 In the Settings window for Basic, locate the Output Properties section.
- 4 Click + Select Quantity.
- 5 In the Physical Quantity dialog box, type density in the text field.
- 6 Click **Filter**.
- 7 In the tree, select General>Density (kg/m^3).
- 8 Click OK.
- 9 In the Settings window for Basic, locate the Output Properties section.
- **10** In the table, enter the following settings:

Property	Variable	Expression	Unit	Size
Density	rho	rho0	kg/m³	lxl

II Click + Select Quantity.

12 In the Physical Quantity dialog box, type viscosity in the text field.

I3 Click **Filter**.

14 In the tree, select Transport>Dynamic viscosity (Pa\*s).

I5 Click OK.

16 In the Settings window for Basic, locate the Output Properties section.

17 In the table, enter the following settings:

Property	Variable	Expression	Unit	Size
Dynamic viscosity	mu	muO	Pa·s	lxl

### MATERIALS

Material Link I (matlnk I)

In the Model Builder window, under Component I (compl) right-click Materials and choose More Materials>Material Link.

# LAMINAR FLOW (SPF)

Inlet I

- I In the Model Builder window, under Component I (compl) right-click Laminar Flow (spf) and choose Inlet.
- 2 In the Settings window for Inlet, locate the Boundary Selection section.

- 3 From the Selection list, choose Left.
- 4 Locate the Boundary Condition section. From the list, choose Fully developed flow.
- **5** Locate the **Fully Developed Flow** section. In the  $U_{\mathrm{av}}$  text field, type Uin.

#### Outlet I

- I In the Physics toolbar, click Boundaries and choose Outlet.
- 2 In the Settings window for Outlet, locate the Boundary Selection section.
- 3 From the Selection list, choose Right.

#### Wall 2

- I In the Physics toolbar, click Boundaries and choose Wall.
- 2 In the Settings window for Wall, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Symmetry**.
- 4 Locate the Boundary Condition section. From the Wall condition list, choose Slip.

# LAMINAR FLOW 2 (SPF2)

In the Model Builder window, under Component I (compl) click Laminar Flow 2 (spf2).

#### Inlet 1

- I In the **Physics** toolbar, click **Boundaries** and choose **Inlet**.
- 2 In the Settings window for Inlet, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Right**.
- 4 Locate the Boundary Condition section. From the list, choose Fully developed flow.
- 5 Locate the Fully Developed Flow section. In the  $U_{
  m av}$  text field, type Uin.

#### Outlet 1

- I In the Physics toolbar, click Boundaries and choose Outlet.
- 2 In the Settings window for Outlet, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Left**.

#### Wall 2

- I In the Physics toolbar, click Boundaries and choose Wall.
- 2 In the Settings window for Wall, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Symmetry**.
- 4 Locate the Boundary Condition section. From the Wall condition list, choose Slip.

#### MESH I

- I In the Model Builder window, under Component I (compl) click Mesh I.
- 2 In the Settings window for Mesh, locate the Physics-Controlled Mesh section.
- 3 From the Element size list, choose Extremely fine.

#### Free Triangular I

In the Mesh toolbar, click Free Triangular.

#### Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, click to expand the Element Size Parameters section.
- 3 Locate the **Element Size** section. Click the **Custom** button.
- 4 Locate the Element Size Parameters section. In the Maximum element size text field, type meshsz.
- 5 Click III Build All.

#### STUDY I

In the **Home** toolbar, click **Compute**.

#### RESULTS

#### Mirror 2D I

- I In the Results toolbar, click More Datasets and choose Mirror 2D.
- 2 In the Settings window for Mirror 2D, locate the Axis Data section.
- 3 In row Point 2, set x to 1 and y to 0.

#### Velocity (sbf)

- I In the Model Builder window, under Results click Velocity (spf).
- 2 In the Velocity (spf) toolbar, click Plot.
- 3 Click the **Zoom Extents** button in the **Graphics** toolbar.

# Velocity (spf2)

- I In the Model Builder window, click Velocity (spf2).
- 2 In the Velocity (spf2) toolbar, click  **Plot**.

Now define two average operators on the inlet and outlet of the modeling domain. These will be used to compute the pressure ratio between the inlet and outlet for the forward and reverse flow.

#### DEFINITIONS

Average I (aveop I)

- I In the Definitions toolbar, click Monlocal Couplings and choose Average.
- 2 In the Settings window for Average, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Left.

Average 2 (aveop2)

- I In the **Definitions** toolbar, click Nonlocal Couplings and choose Average.
- 2 In the Settings window for Average, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Right.

Integration | (intob|)

- I In the Definitions toolbar, click // Nonlocal Couplings and choose Integration.
- 2 Select Domain 2 only.

Define a variable corresponding to the ratio of the pressure difference between the forward and backward flow.

#### Variables 1

- I In the **Definitions** toolbar, click **a= Local Variables**.
- 2 In the Settings window for Variables, locate the Variables section.
- 3 In the table, enter the following settings:

Name	Expression	Unit	Description
dP_forward	aveop1(p)-aveop2(p)	Pa	Pressure difference, forward direction
dP_backward	aveop2(p2)-aveop1(p2)	Pa	Pressure difference, backward direction
Di	dP_backward/dP_forward		Ratio of pressure differences

In order to evaluate the ratio of the pressure differences, the model needs updating. Note that the model does not need to be solved again.

#### STUDY I

In the Study toolbar, click C Update Solution.

#### RESULTS

#### Diodicity

- I In the Results toolbar, click Evaluation Group.
- 2 In the Settings window for Evaluation Group, type Diodicity in the Label text field.

#### Global Evaluation 1

- I Right-click Diodicity and choose Global Evaluation.
- 2 In the Settings window for Global Evaluation, click Add Expression in the upper-right corner of the Expressions section. From the menu, choose Component I (compl)> Definitions>Variables>Di - Ratio of pressure differences - I.
- 3 In the Diodicity toolbar, click **Evaluate**.

#### DIODICITY

I Go to the Diodicity window.

Observe that the ratio of the pressure differences is very close to 1. This concludes solving of the forward problem; now set up the optimization problem.

Add a density topology feature, which can be used to distinguish between free flow and solid regions. This variable will be coupled back to the Laminar Flow interfaces later. The filter radius should not be smaller than the mesh element size. The default value will work, but a fixed value has to be chosen to make the result of the optimization mesh independent.

#### COMPONENT I (COMPI)

Density Model I (dtopol)

- I In the Physics toolbar, click of Optimization and choose Topology Optimization. Only the center part of the channel geometry is needed in the optimization, so you only have to define the feature there.
- 2 Select Domain 2 only.
- 3 In the Settings window for Density Model, locate the Filtering section.
- **4** From the  $R_{\min}$  list, choose **User defined**.
- 5 In the text field, type 1.5\*meshsz.
  - Use a filter to prevent a checkerboard instability that may otherwise occur for the control variable theta.
- **6** Locate the **Interpolation** section. From the **Interpolation type** list, choose **Darcy**.

- 7 Locate the Projection section. From the Projection type list, choose Hyperbolic tangent projection.
- **8** Locate the **Interpolation** section. In the  $q_{\mathrm{Darcy}}$  text field, type 1.
- **9** Locate the **Control Variable Initial Value** section. In the  $\theta_0$  text field, type 1.
- 10 Locate the Control Variable Discretization section. From the Element order list, choose Constant.

Now the design variable used in the optimization is defined. The initial value 1 corresponds to a channel free from porous material.

Now define the friction force to be used in the Laminar Flow interfaces in the form of viscous and friction dissipation that can be integrated over the domain to obtain a suitable objective function.

#### DEFINITIONS

Variables 1

- I In the Model Builder window, under Component I (compl)>Definitions click Variables I.
- 2 In the Settings window for Variables, locate the Variables section.
- **3** In the table, enter the following settings:

Name	Expression	Unit	Description
phi_forward	spf.Qvd+alpha*(u^2+v^2)	W/m³	Dissipation density, forward flow
phi_backward	spf2.Qvd+alpha*(u2^2+ v2^2)	W/m³	Dissipation density, backward flow
phi_total	phi_backward+phi_forward	W/m³	Total dissipation
E_forward	<pre>intop1(phi_forward)</pre>	W/m	Energy dissipation, forward flow
E_backward	<pre>intop1(phi_backward)</pre>	W/m	Energy dissipation, backward flow
obj	E_backward/E_forward		Objective function
alpha	16.*mu0*dtopo1.theta_p/meshsz^2	Pa·s/m²	Friction force

#### LAMINAR FLOW (SPF)

Add the friction force to the Laminar Flow interfaces so that the fluid flows around regions where theta is 0 and through regions where theta is 1.

I In the Model Builder window, under Component I (compl) click Laminar Flow (spf).

#### Volume Force 1

- I In the Physics toolbar, click **Domains** and choose **Volume Force**.
- 2 Select Domain 2 only.
- 3 In the Settings window for Volume Force, locate the Volume Force section.
- **4** Specify the **F** vector as

-alpha*u	x
-alpha*v	у

#### LAMINAR FLOW 2 (SPF2)

- I In the Model Builder window, under Component I (compl) click Laminar Flow 2 (spf2).
- 2 In the Physics toolbar, click **Domains** and choose **Volume Force**.
- I Select Domain 2 only.
- 2 In the Settings window for Volume Force, locate the Volume Force section.
- **3** Specify the  $\mathbf{F}$  vector as

-alpha*u2	x
-alpha*v2	у

#### **OPTIMIZATION**

- I In the Model Builder window, click Study I.
- 2 In the Settings window for Study, type Optimization in the Label text field.

#### Topology Optimization

- I In the Study toolbar, click optimization and choose Topology Optimization.
- 2 In the Settings window for Topology Optimization, locate the Optimization Solver section.
- 3 In the Maximum number of iterations text field, type 200.
- 4 Select the Move limits check box. In the associated text field, type 0.2.
- 5 Click Add Expression in the upper-right corner of the Objective Function section. From the menu, choose Component I (compl)>Definitions>Variables>compl.obj -Objective function - I.

- 6 Locate the Objective Function section. From the Type list, choose Maximization.
- 7 In the Study toolbar, click  $t_{=0}^{\cup}$  Get Initial Value.

#### Solver Configurations

In the Model Builder window, expand the Optimization>Solver Configurations node.

# Solution I (soll)

The old version of MMA (1987) is less prone to premature convergence.

- I In the Model Builder window, expand the Optimization>Solver Configurations> Solution I (soll) node, then click Optimization Solver I.
- 2 In the Settings window for Optimization Solver, locate the Optimization Solver section.
- 3 Clear the Globally Convergent MMA check box.

#### RESULTS

#### Mirror 2D I

Use the mirrored dataset to plot the value of theta during the optimization.

# Topology

- I In the Home toolbar, click Add Plot Group and choose 2D Plot Group.
- 2 In the Settings window for 2D Plot Group, type Topology in the Label text field.
- 3 Locate the Data section. From the Dataset list, choose Mirror 2D 1.

#### Surface I

- I Right-click **Topology** and choose **Surface**.
- 2 In the Settings window for Surface, locate the Expression section.
- 3 In the Expression text field, type dtopo1.theta.
- 4 Locate the Coloring and Style section. Click | Change Color Table.
- 5 In the Color Table dialog box, select Linear>GrayScale in the tree.
- 6 Click OK.
- 7 In the Settings window for Surface, locate the Coloring and Style section.
- 8 Clear the Color legend check box.
- **9** Click to expand the Range section. Select the Manual color range check box.
- 10 In the Maximum text field, type 1.
- II In the **Topology** toolbar, click  **Plot**.

#### **OPTIMIZATION**

Topology Optimization

- I In the Model Builder window, under Optimization click Topology Optimization.
- 2 In the Settings window for Topology Optimization, locate the Output While Solving section.
- **3** Select the **Plot** check box.
- 4 From the Plot group list, choose Topology.
- 5 In the Home toolbar, click **Compute**.

#### RESULTS

Forward Flow

The forward and backward flow patterns computed using the optimization solver can now be visualized.

- I In the Settings window for 2D Plot Group, locate the Data section.
- 2 From the Dataset list, choose Mirror 2D 1.
- 3 In the Velocity (spf) toolbar, click Plot.
- 4 In the Label text field, type Forward Flow.

Arrow Surface 1

- I Right-click Forward Flow and choose Arrow Surface.
- 2 In the Settings window for Arrow Surface, locate the Arrow Positioning section.
- 3 Find the x grid points subsection. In the Points text field, type 30.
- 4 Find the y grid points subsection. In the Points text field, type 30.
- 5 In the Forward Flow toolbar, click Plot.

#### Surface 2

- I Right-click Forward Flow and choose Surface.
- 2 In the Settings window for Surface, locate the Expression section.
- **3** In the **Expression** text field, type dtopo1.theta.
- 4 Locate the Coloring and Style section. From the Coloring list, choose Uniform.
- 5 From the Color list, choose White.
- 6 Locate the Range section. Select the Manual data range check box.
- 7 In the Maximum text field, type 0.5.
- 8 In the Forward Flow toolbar, click Plot.

**9** Click the **Zoom Extents** button in the **Graphics** toolbar.

Now follow a similar procedure for the backward flow.

Arrow Surface 1, Surface 2

- I In the Model Builder window, under Results>Forward Flow, Ctrl-click to select Arrow Surface I and Surface 2.
- 2 Right-click and choose Copy.

#### Backward Flow

- I In the Model Builder window, under Results click Velocity (spf2).
- 2 In the Settings window for 2D Plot Group, type Backward Flow in the Label text field.
- 3 Right-click Results>Backward Flow and choose Paste Multiple Items.

#### Arrow Surface I

- I In the Model Builder window, click Arrow Surface I.
- 2 In the Settings window for Arrow Surface, click Replace Expression in the upper-right corner of the Expression section. From the menu, choose Component I (compl)> Laminar Flow 2>Velocity and pressure>u2,v2 - Velocity field.
- 3 In the Backward Flow toolbar, click Plot.

#### Backward Flow

- I In the Model Builder window, click Backward Flow.
- 2 In the Settings window for 2D Plot Group, locate the Data section.
- 3 From the Dataset list, choose Mirror 2D 1.
- 4 In the Backward Flow toolbar, click Plot.
- 5 Click the **Zoom Extents** button in the **Graphics** toolbar.
- 6 In the Backward Flow toolbar, click Plot.

# Topology

- I In the Model Builder window, click Topology.
- 2 In the **Topology** toolbar, click **Plot**.
- 3 Click the **Zoom Extents** button in the **Graphics** toolbar.

Backward Flow, Forward Flow, Pressure (spf), Pressure (spf2), Topology

- I In the Model Builder window, under Results, Ctrl-click to select Forward Flow, Pressure (spf), Backward Flow, Pressure (spf2), and Topology.
- 2 Right-click and choose **Group**.

#### Optimized Design

In the Settings window for Group, type Optimized Design in the Label text field.

Next, evaluate the damping with respect to the relative amount of dissipation in the solid region to see if the optimization result can be trusted.

### Dissipation in Solid Regions

- I In the Results toolbar, click Evaluation Group.
- 2 In the Settings window for Evaluation Group, type Dissipation in Solid Regions in the Label text field.

#### Global Evaluation 1

- I Right-click Dissipation in Solid Regions and choose Global Evaluation.
- 2 In the Settings window for Global Evaluation, locate the Expressions section.
- **3** In the table, enter the following settings:

Expression	Unit	Description	
<pre>intop1(phi_total*(1- dtopo1.theta))/ intop1(phi_total)</pre>	1	Relative dissipation in solid	

4 In the Dissipation in Solid Regions toolbar, click **= Evaluate**.

There is significant power loss in the solid regions, which is unphysical. Therefore it is a good idea to set up a verification analysis in a new component.

The **Density Model** generates a **Filter Dataset** when the default plots are created. However, since **Density Model** was added later, a **Filter Dataset** must be added manually.

#### Filter I

- I In the **Results** toolbar, click **More Datasets** and choose **Filter**.
- 2 In the Settings window for Filter, locate the Expression section.
- 3 In the Expression text field, type dtopo1.theta.
- 4 Locate the Filter section. In the Lower bound text field, type 0.5.
- **5** Locate the **Evaluation** section. Clear the **Use derivatives** check box. Create a new component from the filter dataset.
- 6 Right-click Filter I and choose Create Mesh in New Component.

#### MESH 2

#### Import I

- I In the Settings window for Import, locate the Import section.
- 2 Click Import.

#### Adabt I

- I In the Mesh toolbar, click **Modify** and choose Adapt.
- 2 In the Settings window for Adapt, locate the Adaptation section.
- **3** From the **Solution** list, choose **None**.
- 4 From the Type of expression list, choose Absolute size.
- 5 In the Size expression text field, type meshsz.
- 6 In the Maximum number of refinements text field, type 0.
- 7 In the Maximum coarsening factor text field, type Inf.
- 8 Click Pauld Selected.

#### MATERIALS

Material Link 2 (matlnk2)

In the Model Builder window, under Component 2 (comp2) right-click Materials and choose More Materials>Material Link.

# LAMINAR FLOW (SPF), LAMINAR FLOW 2 (SPF2)

I In the Model Builder window, under Component I (compl), Ctrl-click to select Laminar Flow (spf) and Laminar Flow 2 (spf2).

Copy and paste the physics from the first component and fix the selections.

2 Right-click and choose Copy.

#### COMPONENT 2 (COMP2)

In the Model Builder window, right-click Component 2 (comp2) and choose Paste Multiple Items.

#### LAMINAR FLOW (SPF3), LAMINAR FLOW 2 (SPF4)

- I In the Model Builder window, under Component 2 (comp2), Ctrl-click to select Laminar Flow (spf3) and Laminar Flow 2 (spf4).
- 2 In the Messages from Paste dialog box, click OK.

#### Inlet I

- I In the Model Builder window, expand the Component 2 (comp2)>Laminar Flow (spf3) node, then click Inlet 1.
- 2 In the Settings window for Inlet, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Left**.

# Outlet I

- I In the Model Builder window, click Outlet 1.
- 2 In the Settings window for Outlet, locate the Boundary Selection section.
- 3 From the Selection list, choose Right.

#### Wall 2

- I In the Model Builder window, click Wall 2.
- 2 In the Settings window for Wall, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Symmetry**.

#### Volume Force 1

In the Model Builder window, right-click Volume Force I and choose Delete.

#### LAMINAR FLOW 2 (SPF4)

#### Inlet 1

- I In the Model Builder window, expand the Component 2 (comp2)>Laminar Flow 2 (spf4) node, then click Inlet 1.
- 2 In the Settings window for Inlet, locate the Boundary Selection section.
- 3 From the Selection list, choose Right.

#### Outlet I

- I In the Model Builder window, click Outlet 1.
- 2 In the Settings window for Outlet, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Left**.

#### Wall 2

- I In the Model Builder window, click Wall 2.
- 2 In the Settings window for Wall, locate the Boundary Selection section.
- 3 From the Selection list, choose Symmetry.

#### Volume Force 1

In the Model Builder window, right-click Volume Force I and choose Delete.

#### **DEFINITIONS (COMP2)**

Set up a new average operator so that a new diodicity variable can be defined for the new component.

# Average 3 (aveop3)

- I In the Definitions toolbar, click // Nonlocal Couplings and choose Average.
- 2 In the Settings window for Average, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- **4** From the **Selection** list, choose **Left**.
- 5 Right-click Average 3 (aveop3) and choose Duplicate.

# Average 4 (aveop4)

- I In the Model Builder window, click Average 4 (aveop4).
- 2 In the Settings window for Average, locate the Source Selection section.
- 3 From the Selection list, choose Right.

#### Variables 2

- I In the Model Builder window, right-click Definitions and choose Variables.
- 2 In the Settings window for Variables, locate the Variables section.
- **3** In the table, enter the following settings:

Name	Expression	Unit	Description
Di	(aveop4(p2)-aveop3(p2))/ (aveop3(p)-aveop4(p))		Ratio of pressure differences

#### ADD STUDY

- I In the Home toolbar, click Add Study to open the Add Study window.
- 2 Go to the Add Study window.
- 3 Find the Studies subsection. In the Select Study tree, select General Studies>Stationary.
- 4 Find the Physics interfaces in study subsection. In the table, clear the Solve check boxes for Laminar Flow (spf) and Laminar Flow 2 (spf2).
- 5 Click Add Study in the window toolbar.
- 6 In the Model Builder window, click the root node.
- 7 In the Home toolbar, click Add Study to close the Add Study window.

#### **OPTIMIZATION**

Step 1: Stationary

- I In the Settings window for Stationary, locate the Physics and Variables Selection section.
- 2 In the table, clear the Solve for check boxes for Laminar Flow (spf3) and Laminar Flow 2 (spf4).

#### **VERIFICATION**

- I In the Model Builder window, under Study 2 click Step 1: Stationary.
- 2 In the Settings window for Stationary, locate the Physics and Variables Selection section.
- 3 In the table, clear the Solve for check box for Topology Optimization (Component 1).
- 4 In the Model Builder window, click Study 2.
- 5 In the Settings window for Study, type Verification in the Label text field.
- 6 In the Home toolbar, click **Compute**.

#### RESULTS

Pressure (spf3), Pressure (spf4), Velocity (spf3), Velocity (spf4)

- I In the Model Builder window, under Results, Ctrl-click to select Velocity (spf3), Pressure (spf3), Velocity (spf4), and Pressure (spf4).
- 2 Right-click and choose Group.

Verification

In the **Settings** window for **Group**, type Verification in the **Label** text field.

Topology Optimization

In the Model Builder window, under Results right-click Topology Optimization and choose Delete.

Diodicity

Compute the diodicity for the component (without power loss in solid domains).

Global Evaluation 2

- I In the Model Builder window, right-click Diodicity and choose Global Evaluation.
- 2 In the Settings window for Global Evaluation, locate the Data section.
- 3 From the Dataset list, choose Verification/Solution 2 (3) (sol2).
- 4 Click Add Expression in the upper-right corner of the Expressions section. From the menu, choose Component 2 (comp2)>Definitions>Variables>Di -

Ratio of pressure differences - I.

5 In the Diodicity toolbar, click **= Evaluate**.

The diodicity is actually higher for the verification simulation, so the optimization has found a good design topology.