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(1)

Design calculations

Tee Casting

- * casting weight - 2900 grams
- * overflow weight - 500 grams
- * Runner weight - 1800 grams
- * Shot weight - 5200 grams
- * Density - 2.6 gram/cm³
- * Plunger diameter - 100 mm
- * Active sleeve length - 680 mm

Tonnage calculation

$T = \text{Projected area of component} [\text{including runner and overflow}] \times \text{casting pressure} \times \text{Factor of safety}$

$$T = 1303.9245 \times \frac{700}{100} \times 1.2 \rightarrow \text{FoS}$$

$$T = 1095296.58 \text{ kgf} \Rightarrow \frac{1095296.58}{1000} = 1095.29 \text{ T}$$

$$\boxed{T = 1095.29 \text{ ton}}$$

Shot volume

$$= \frac{\text{Shot weight}}{\text{Density}} \Rightarrow \frac{5200}{2.6 \rightarrow \text{g/cm}^3} \Rightarrow 2000 \text{ cm}^3$$

$$\boxed{\text{Shot volume} = 2000 \text{ cm}^3}$$

3) Sleeve volume

$$= \frac{\pi}{4} \times D^2 \times L \xrightarrow{\text{Plunger dia}} \text{Active sleeve length}$$

$$\Rightarrow \frac{3.14}{4} \times 10^2 \times 68 \xrightarrow[cm]{cm} \Rightarrow 5338 \text{ cm}^3$$

Sleeve volume = 5338 cm³

4) Fill ratio

$$= \frac{\text{Shot volume}}{\text{Sleeve volume}} = \frac{2000}{5338} \xrightarrow[\text{cm}^3]{\text{cm}^3} = 0.3746$$

$$= 0.3746 \times 100 = 37.46\%$$

Fill ratio = 37.46%

5) Fill time

k = Conductivity [For H13 = 0.0346 Sec/mm]

t = casting wall thickness mm

T_i = Temp of metal entering in die

T_F = Solidification Point of metal 570°C approx

T_d = Die temperature before shot.

s = Solidification Factor [For HPD = 25% Const]

Z = Unit Conversion Factor [For $Z = 3.8^{\circ}\text{C}/\%$]

$$F_T = 12 \left[\frac{T_i - T_F + (st)}{T_F - T_d} \right] \times t$$

$$= 0.0346 \left[\frac{\frac{640 - 570}{^{\circ}F} + \frac{25 \times 38}{^{\circ}C}}{570 - 170} \right] \times 4 \text{ myh}$$

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$F_t = 0.057 \text{ sec}$

6) Discharge

$$Q = \frac{\text{weight after gate}}{\text{Density} \times \text{Fill time}}$$

$$= \frac{3400 \text{ g}}{2.6 \times 0.057 \text{ sec}} = 22941.97 \text{ cm}^3/\text{s}$$

$$\text{g/cm}^3$$

$Q = 22941.97 \text{ cm}^3/\text{s}$

7) Plunger area

$$= \frac{\pi}{4} \times D^2 = \frac{3.14}{4} \times 10^2 = 78.5 \text{ cm}^2$$

$\text{Plunger area} = 78.5 \text{ cm}^2$

8) s_1

$$s_1 = L - [s_2 + \text{Biscuit thickness}]$$

$$s_2 = \frac{\text{Weight after gate}/26}{\text{Plunger area}} = \frac{3400 / 2.6 \text{ g/cm}^2}{78.5 \text{ cm}^2}$$

$$s_2 = 16.6 \text{ cm} \Rightarrow 166 \text{ mm}$$

$s_2 = 166 \text{ mm}$

$$S_1 = L - [S_2 + \text{Biscuit Height}]$$

$$S_1 = 680 - [166 + 30]$$

$$\boxed{S_1 = 484 \text{ mm}}$$

i) Velocity (v_2)

$$v_2 = \frac{\text{Discharge}}{\text{Plunger surface area}} = \frac{22941.97}{78.5}$$

cwt/s
cwt

$$v_2 = 292.25 \text{ cm/s}$$

$$\boxed{v_2 = 2.92 \text{ m/s}},$$

(ii) Gate area :- Method 1

$$G_A = \frac{\text{Weight after gate}}{\text{Density} \times \text{Fill time} \times \text{gate velocity}}$$

→ g/cm³
g/cm³ sec cwt/s

Considering Gate velocity = 40 M/S \Rightarrow 4000 cm/s

$$G_A = \frac{3400}{2.6 \times 0.057 \times 4000} = \frac{3400}{592.8}$$

g/cm³ sec cwt/s

$$G_A = 5.73 \text{ cm}^2 \Rightarrow 5.73 \times 100 = 573 \text{ mm}^2$$

$$\boxed{G_A = 573 \text{ mm}^2} \Leftrightarrow \text{Method 1}$$

10(b)) Gate area [Method : 2]

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$$G_A \cdot G_v = P_A \cdot P_v$$

G_A = Gate area

G_v = Gate velocity

P_A = Plunger area

P_v = Plunger velocity

$$G_A = \frac{P_A \times P_v}{G_v} = \frac{78.5 \times 292}{4000 \text{ cm/s}} = 5.73 \text{ cm}^2$$

$$\boxed{G_A = 5.73 \text{ mm}^2}$$

ii) Velocity (v_i) [Method : 1]

$$v_i = \frac{\sqrt{\text{Plunger dia}} \text{ cm} \times \text{Coefficient Factor}}{\text{Fill ratio } \gamma} \rightarrow 0.7 \text{ constant}$$

$$v_i = \frac{\sqrt{100} \text{ mm} \times 0.7}{37 \gamma}$$

$$\boxed{v_i = 0.189 \text{ m/s}}$$

ii(b)) Velocity (v_i) [Method : 2]

$$v_i = C_c \left[\frac{100\% - \text{Fill ratio } \gamma}{100\%} \right] \times \sqrt{d +}$$

C_c = Curved constant $[0.579 \times 2]^{1/2}$

d = Plunger dia in m

$$v_1 = 0.579 \times 2 [0.3746] \times \sqrt{0.1}$$

$$\boxed{v_1 = 0.136 \text{ m/s}}$$

[Note:- For v_1 calculation there is a problem with unit cancellation and there is no constant formula]

12) Pressure hydraulic

$$P_s = \left[\frac{D_H}{D_p} \right]^2 \times P_h$$

P_s = Specific casting pressure

Machine

D_H = Hydraulic cylinder piston diameter on selected

D_p = Plunger diameter

P_h = Hydraulic pressure

Recommended pressure

Standard part = 400 bar

Technical part = 400 to 600 bar

Pressure tight part = 800 to 1000 bar

$$P_s = \left[\frac{D_H}{D_p} \right]^2 \times P_h$$

$$700 = \left[\frac{\frac{160}{100}}{\frac{100}{myte}} \right]^2 \times P_h$$

$$700 = 2.6 \times P_h$$

$$P_h = \frac{700}{2.6}$$

$$\boxed{P_h = 269.23 \text{ bar}}$$

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13) Vent area [Method : 1]

$$\text{Vent area} = \frac{\text{Discharge}}{\text{Air velocity}}$$

Air velocity = 200 to 300 m/s \Rightarrow 20000 cm/s

$$\text{Vent area} = \frac{22941.97}{20000 \text{ cm/s}} = 1.147 \text{ cm}^2$$

$$\boxed{\text{Vent area} = 114.7 \text{ mm}^2}$$

13 (B) Vent area, [Method : 2]

$$V_A V_r = P_A P_r$$

$$V_A = \frac{P_A \times P_r}{V_r} = \frac{78.5 \times 292}{20000 \text{ cm/s}} = 1.146 \text{ cm}^2$$

$$\boxed{V_A = 114.6 \text{ mm}^2}$$

$$\boxed{V_A = 114.6 \text{ mm}^2}$$

14) Gate and Runner design

As per calculation Gate area $i.e. = 573 \text{ mm}^2$

Runner area = $[1.2 \text{ to } 1.3] \times$ Gate area \rightarrow Assumption

Assuming Runner area = $1.2 \times$ Gate area $\left[\begin{array}{l} \text{Condition} \\ -1 \end{array} \right]$

$$\text{by dividing into 4 gates} = \frac{573}{4} = 143.25 \text{ mm}^2$$

$$\text{Gate: 1 [considering gate thickness } 3\text{m}] = (\overset{\text{width}}{4.8} \times \overset{\text{depth}}{3}) = 144 \text{ mm}^2$$

$$\text{Runner area} = \text{Runner width} \times \text{Runner depth} \quad [\text{Condition - 2}]$$

$$\text{Runner width} = 1.5 \text{ times} \times \text{Runner depth}$$

$$[\text{So according to Condition - 2}]$$

$$\text{Runner area} = 1.5 \text{ times of } \frac{\text{runner depth}}{\text{runner depth}} \times \text{runner depth}$$

$$\text{runner area} = 1.5 \times R_d \times P_d$$

$$\underline{\text{runner area} = 1.5 \times R_d^2} \rightarrow [\text{Condition - 2}]$$

$$\overset{\text{runner area}}{\overset{\text{Condition - 1}}{\text{runner area}}} = 1.2 \text{ times of gate area}$$

Substituting for Condition - 2 in Condition - 1

So

$$1.5 \times R_d^2 = 1.2 \times 575$$

$$1.5 \times R_d^2 = 690$$

$$R_d^2 = \frac{690}{1.5} = 460$$

$$R_d^2 = 460 \Rightarrow \sqrt{460}$$

$$R_d = \sqrt{460} = 21.5 \text{ mm}^2$$

$$\boxed{\text{Runner depth} = 21.5 \text{ mm}^2}$$

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Runner width = 1.5 times of \times Runner depth

$$= 1.5 \times 21.5$$

$$\boxed{\text{Runner width} = 32.25 \text{ mm}^2}$$

15) Cooling calculation

a) Heat input = $H \times G \times N$

H = Heat factor for H13 Steel 143 K cal/kg

N = Total no of shot produced per hour [hourly output of specific die in specific machine]

G = Shot weight in kg

$$\begin{matrix} \text{Heat input} & = 143 \times 5.13 \times 45 \\ & \downarrow \quad \downarrow \quad \downarrow \\ & \text{K cal/kg} \quad \text{kg} \quad \text{h} \end{matrix}$$

$$\boxed{\text{Heat input} = 33011.55 \text{ K cal/h}}$$

B) Heat accumulated

Assuming heat loss 50% by conduction and by spray cooling

$$\text{Heat accumulated} = \text{Heat input} \times \frac{50}{100}$$

$$\text{Heat accumulated} = 33011.55 \times \frac{50}{100}$$

$$\boxed{\text{Heat acc} = 16505.775 \text{ kcal/h}}$$

c) Amount of cooling
water req

$$A_w = \frac{\text{Heat accumulated}}{k \times [T_o - T_i]}$$

k = Thermal Conductivity of H13 steel

is $0.64 \text{ kcal/hr.m.}^{\circ}\text{C}$

T_o = outlet temperature of water in $^{\circ}\text{C}$

T_i = inlet temperature of water in $^{\circ}\text{C}$

$$A_w = \frac{16505.775}{0.6 \times (24 - 40)} = \frac{16505.775}{9.6}$$

$$\boxed{A_w = 1719.35 \text{ lits per hour}}$$

$$A_w = 1719.35 / 60$$

$$\boxed{A_w = 28 \text{ lits per minute}}$$