

# [ 재 료 역 학 ]

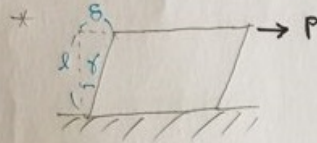
\* 변형률

$$\epsilon = \frac{l-l'}{l} = \frac{\Delta l}{l} \quad \text{세로, 축방향 변형률}$$

$$\epsilon' = \frac{d-d'}{d} = \frac{\Delta d}{d} \quad \text{가로, 횡방향 변형률}$$

\* 푸아송의 비

$$\mu = \frac{1}{m} = \frac{\epsilon'}{\epsilon} \quad m: \text{푸아송의 수}$$



$$\tan \gamma = \frac{\delta}{l}$$

\* (수직응력)  $\sigma = E\epsilon$

(전단응력)  $\tau = G\gamma$

\* (수직 변형률)  $\epsilon_A = \frac{\Delta A}{A} = \mu \epsilon$

(체적 변형률)  $\epsilon_V = \frac{\Delta V}{V} = \epsilon(1-\mu)$

$$= \epsilon_x + \epsilon_y + \epsilon_z$$

\*  $k = \frac{E}{3(1-\mu)}$

$$G = \frac{E}{2(1+\mu)}$$

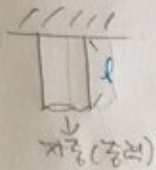
$$= \frac{mE}{2(m+1)}$$

\*  $\sigma_{av} = \frac{P}{A} \quad \sigma_{max} = \alpha \times \sigma_{av} \quad \alpha: \text{응력 집중 계수}$

(평균 수직응력)

(최대 응력 수직응력)

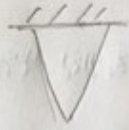
\* 자중을 고려한 응력, 처짐량



$$\sigma = \gamma l$$

$$\delta = \frac{\gamma l^3}{2E}$$

원뿔체  
x 1/2



$$\sigma = \gamma l \times \frac{1}{2}$$

$$\delta = \frac{\gamma l^3}{2E} \times \frac{1}{2}$$

\* 열응력

$$\epsilon = \alpha \Delta T \quad \text{열변형률}$$

$$\delta = l \alpha \Delta T \quad \text{열처짐량}$$

$$\sigma = E \alpha \Delta T = \frac{P}{A} \quad \text{열응력}$$

\* 탄성에너지 (U)

1. 1. 잡아당길때  $U = \frac{P\delta}{2} = \frac{\sigma^2 A l}{2E}$

2. 비틀릴때  $U = \frac{T\theta}{2}$

단위 체적당 탄성에너지 (u)

1. 1. 인장, 압축  $u = \frac{\sigma^2}{2E}$

2. 전단  $u = \frac{\tau^2}{2G}$

비틀림  $u = \frac{\tau^2}{4G}$

\* 레일리만 계수

$$= \frac{\text{응력} \cdot \text{탄성한도 비례}}{\text{탄성 계수} \cdot \text{단위체}}$$

\* 중력 계산할 때

$$\sigma = \sigma_0 \left( 1 + \sqrt{1 + \frac{\rho h}{\sigma_0}} \right)$$

$\sigma_0, \sigma$  : 정하중시 변형률과 응력

$$\sigma = \sigma_0 \left( 1 + \sqrt{1 + \frac{\rho h}{\sigma_0}} \right)$$

\* 나침반의 원리

\* 나침반의 원리

$$1. \text{ 종방향 } G_x = \frac{PD\delta}{4t}$$

$t$ : 두께,  $\delta$ : 안전율,  $\eta$ : 효율

$$t = \frac{PD}{4\sigma\eta}$$

$$2. \text{ 원주방향 } G_y = \frac{PD\delta}{2t}$$

$$t = \frac{PD}{2\sigma\eta}$$

$$\text{최대전단응력 } T_{max} = \frac{G_x - G_y}{2}$$

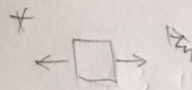
\* 압력 용기 원리

$$(\text{원주응력}) V = \frac{\pi D N}{60}$$

$$\delta = \frac{\gamma}{g} V^2$$

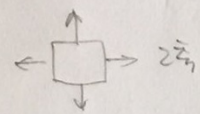
$$\gamma = 9800 \cdot \delta \quad \delta: \text{비중}$$

$\theta$ (°)	관계
0°	$G_{max} = G_x$
90°	$G_{min} = 0$
45°	$T_{max} = \frac{G_x}{2}$
135°	$T_{min} = -\frac{G_x}{2}$



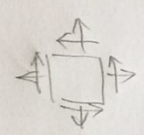
$$\sigma_n = \sigma_x \cos^2 \theta$$

$$\tau_n = \frac{\sigma_x}{2} \sin 2\theta$$



$$\sigma_n = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta$$

$$\tau_n = \frac{\sigma_x - \sigma_y}{2} \sin 2\theta$$



평행 (주응력)

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau^2}$$

↓  
 $\tau_{max}$

\* I (직사각형, 삼각형), Z (반원)

<p>I</p> $\frac{bh^3}{12}$	$\frac{bh^3}{36}$	$\frac{\pi d^4}{64}$
<p>Z</p> $\frac{bh^3}{6}$	$\frac{bh^3}{36}$	$\frac{\pi d^3}{32}$

(삼각형은 잘 안나옴)

\* 평행축 정리

$$I_{parallel} = I_{centroid} + A y_0^2$$

A: 면적

$y_0$ : 도심에서 떨어진 거리



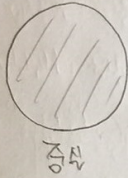
\* 극 2차 관성 모멘트

$$J_p = J_x + J_y$$

극 단면 제곱

$$Z_p = \frac{J_p}{y}$$

\*

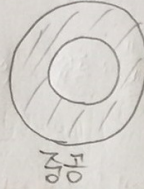


중심

$$J_p = \frac{\pi d^4}{32}$$

$$Z_p = \frac{\pi d^3}{16}$$

\*



중심

$$J_p = \frac{\pi d_o^4}{32} (1 - \alpha^4)$$

$$Z_p = \frac{\pi d_o^3}{16} (1 - \alpha^3)$$

\* 세장비 (λ)

$$\lambda = \frac{l}{k}$$

$$k = \sqrt{\frac{EI}{A}}$$

원의  $k = \frac{d}{4}$

사각형의  $k = \frac{\sqrt{I/A}}{2\sqrt{3}}$

k : 회전반경

\*

\* 굽힘, 비틀림 모멘트

$$M = \sigma \cdot Z = P \cdot l$$

$$T = \tau \cdot Z_p = P \cdot r = 116.2 \frac{PS}{N} \cdot 9.8 \quad (N \cdot m)$$

$$= 974 \frac{HW}{N} \cdot 9.8$$

$$\theta = \frac{TL}{GJ_p} \quad \left( \times \frac{180}{\pi} \text{ 라디안 단위} \right)$$

\* [rad]

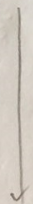
$$EI y = \delta$$

$$EI y' = \theta$$

$$EI y'' = -M$$

$$EI y''' = -V$$

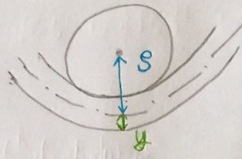
$$EI y^{IV} = -P$$



0점

절반수용 참고

\*



원통에 전단이 가해질 때

$$\frac{M}{I} \neq \frac{E}{S} \neq \frac{6}{y}$$

$\frac{1}{S}$ : 곡률  $S$ : 곡률반경

꼭해지는 것 기억! (꼭반비)

ex)  $M, S, E, y, I, 6$  반비례!

한번씩 나옴

$M, E, S, y, I, 6$  비례!

\* 보의 응력

$$\sigma = \frac{M}{S}$$

(굽힘응력)

$$\tau = \frac{V \cdot Q}{I \cdot b}$$

(전단응력)

$$Q = A \cdot \bar{y}$$

$b$ : 위하고 싶은 곳 자른 길이

$V$ : 전단력

\* 도형의  $\tau = \frac{3V}{2A}$

□ 도형의  $\tau = \frac{3V}{2A}$

○ 도형의  $\tau = \frac{4V}{3A}$

\* 보의 처짐

$$\delta = \theta \cdot \bar{x}$$

$A_m$ : BMD 면적

$$\theta = \frac{1}{EI} \cdot A_m$$

$\bar{x}$ : BMD 끝단에서 도심까지 길이

BMD				
$A_m$	$bh$	$\frac{bh}{2}$	$\frac{bh}{3}$	$\frac{bh}{4}$
$\bar{x}$	$\frac{1}{2}b$	$\frac{2}{3}b$	$\frac{3}{4}b$	$\frac{4}{5}b$



\* 단면

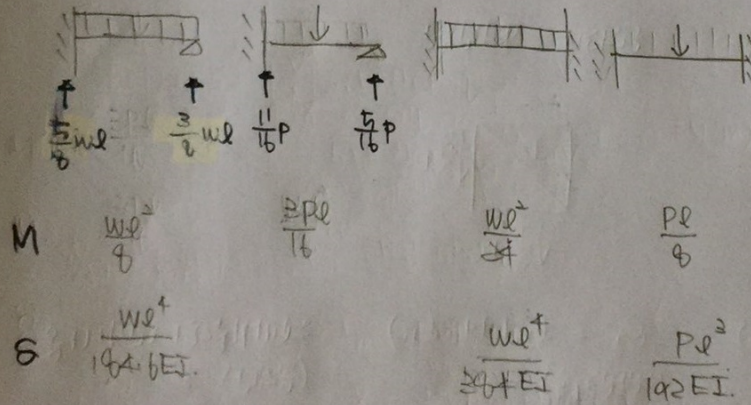
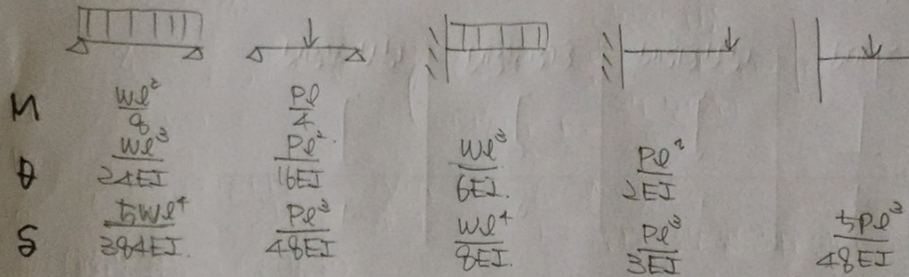
$$S = \frac{8nPD^3}{64d^4}$$

n: 겹개수  
d: 스프링 지름  
D: 원통 지름

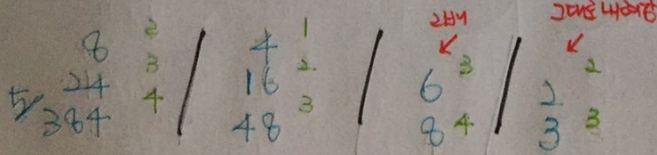
$$\theta = \frac{\delta}{dR}$$

$$\tau = \frac{8WD}{\pi d^3}$$

\* 단면도, 이차곡선, 부정정문



이차곡선 방정식 자판기



\* 도심위치

$$\bar{y} = \frac{A_1 x y_1 + A_2 x y_2}{A_1 + A_2}$$

\* 압축하중, 전곡하중

응력

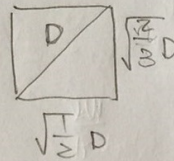
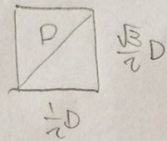
$$P_{cr} = \frac{n \pi^2 EI}{l^2} \rightarrow \sigma = \frac{n \pi^2 E}{\lambda^2}$$

n : 단면계수

종류	자유단	양단 회전단	일단 고정단	양단 고정단
n	1	1	2	4

\* 처짐최소 사각단면

응력최소 사각단면



\*  $\sigma_T = \sigma_n (1 + \epsilon_n)$   
(전응력)

$\epsilon_T = \epsilon_n (1 + \epsilon_n)$   
(전변형률)

±0.171, 0.281 한계나온  
(문41)