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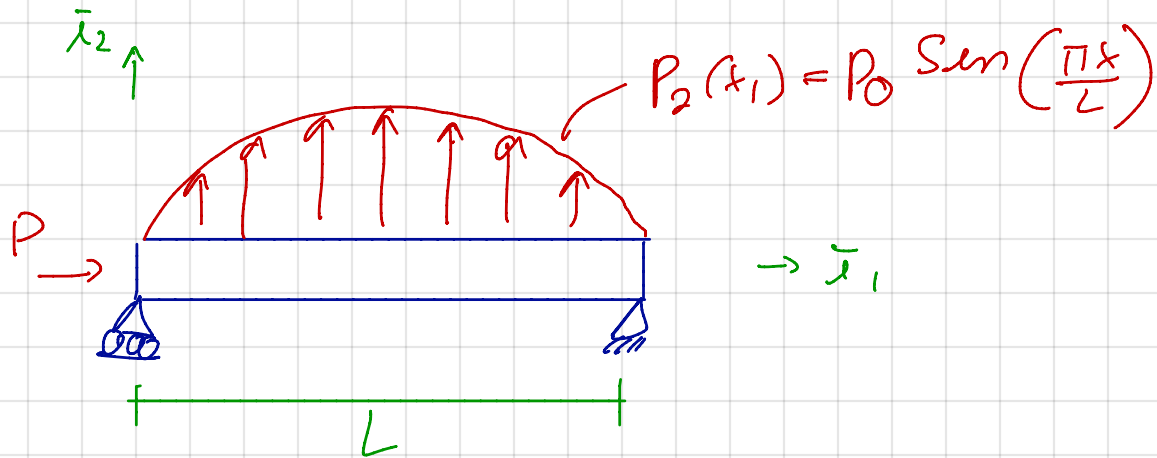
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Example:

$$H_{23}^C = 0$$



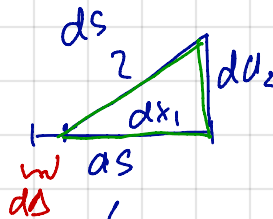
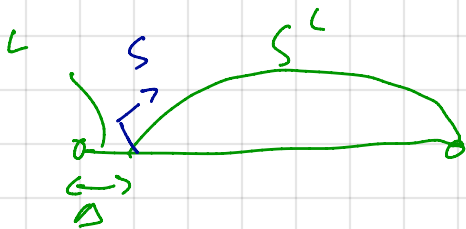
$$A = \frac{1}{2} \int_0^L \left[ S \left( \frac{d\bar{u}_1}{dx_1} \right)^2 + H_{33}^C \left( \frac{d^2\bar{u}_2}{dx_1^2} \right)^2 \right] dx_1$$

\* Interested only in  $u_2(x_1)$ . Neglect the axial strain energy

$$A = \frac{1}{2} \int_0^L H_{33}^C \left( \frac{d^2\bar{u}_2}{dx_1^2} \right)^2 dx_1$$

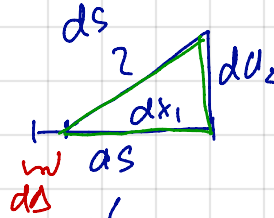
$$\Phi = - \int_0^L P_2(x_1) \cdot u_2(x_1) dx_1 - P \cdot \Delta$$

$\Delta$  - Horizontal displacement at the beam due to changes in curvature.



$$\Delta = \int_0^L ds - \int_0^L dx_1 = \int_0^L (ds - dx_1)$$

$$ds^2 = dx_1^2 + du_2^2$$



$$\Delta = \int_0^L ds - \int_0^L dx_1 = \int_0^L (ds - dx_1)$$

$$ds^2 = dx_1^2 + du_2^2$$

$$ds = \left( 1 + \left( \frac{du_2}{dx_1} \right)^2 \right)^{\frac{1}{2}} dx_1$$

$$ds = \left( 1 + \frac{1}{2} \left( \frac{du_2}{dx_1} \right)^2 + \dots \right) dx_1$$

$$\Delta \approx \int_0^L \frac{1}{2} \left( \frac{du_2}{dx_1} \right)^2 dx_1$$

$$A = \frac{1}{2} \int_0^L H_{33}^C \left( \frac{d^2 u_2}{dx_1^2} \right)^2 dx_1$$

$$\bar{\Phi} = - \int_0^L P_2(x_1) \cdot u_2(x_1) dx_1 - P \cdot \Delta$$

$$\bar{\Phi} = - \int_0^L \left[ P_2(x_1) u_2(x_1) + \frac{P}{2} \left( \frac{du_2}{dx_1} \right)^2 \right] dx_1$$

$$\Pi = A + \bar{\Phi}$$

$$\Pi = \int_0^L \left[ \frac{H_{33}^C}{2} \left( \frac{d^2 u_2}{dx_1^2} \right)^2 - \frac{P}{2} \left( \frac{du_2}{dx_1} \right)^2 - P_2(x_1) u_2(x_1) \right] dx_1$$

$$\Pi = \int_0^L \left[ \frac{H_{33}^C}{2} \left( \frac{d^2 u_2}{dx_1^2} \right)^2 - \frac{P}{2} \left( \frac{du_2}{dx_1} \right)^2 - P_2(x_1) u_2(x_1) \right] dx_1$$

\* The solution is of the form

$$u_2 = C \cdot \sin\left(\frac{\pi x}{L}\right) \quad - \text{! Unknown, } C.$$

$$\Pi = \int_0^L \left[ \frac{H_{33}^C}{2} \left( C \cdot \overset{L/2}{\sin\left(\frac{\pi x}{L}\right)} \frac{\pi^2}{L^2} \right)^2 - \frac{P}{2} \left( C \cdot \cos\left(\frac{\pi x}{L}\right) \frac{\pi}{L} \right)^2 - P_0 \cdot C \cdot \sin\left(\frac{\pi x}{L}\right)^2 \right] dx_1$$

→ Satisfies kinematic B.C.'s!

$$\Pi = \left[ \frac{H_{33}^C}{2} \left( C^2 \frac{L}{2} \frac{\pi^4}{L^4} \right) - \frac{P}{2} C^2 \frac{L}{2} \frac{\pi^2}{L^2} - P_0 C \frac{L}{2} \right]$$

$$\Pi = \frac{L}{4} \left( \frac{C^2 \pi^2}{L^2} \left( H_{33}^C \frac{\pi^2}{L^2} - P \right) - 2 P_0 C \right)$$

FOR THE system to be in equilibrium

$$d\Pi = \frac{\partial \Pi}{\partial C} = 0$$

$$\frac{\partial \Pi}{\partial C} = \frac{L}{4} \left( \underbrace{\frac{2 C \pi^2}{L^2} \left( H_{33}^C \frac{\pi^2}{L^2} - P \right) - 2 P_0}_{=0} \right) = 0$$

$$C = \frac{P_0 L^4}{\pi^2 (H_{33}^C \pi^2 - P L^2)}$$

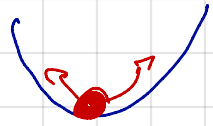
$$\Rightarrow u_2 = \frac{P_0 L^4}{\pi^2 (H_{33}^C \pi^2 - P L^2)} \cdot \sin\left(\frac{\pi x}{L}\right)$$

$$\frac{\partial \Pi}{\partial C} = \frac{L}{4} \left( \frac{2 C \Pi^2}{L^2} \left( H_{33}^C \frac{\Pi^2}{L^2} - P \right) - 2 P_0 \right) = 0$$

Is the equilibrium stable?

$$\frac{d^2 \Pi}{d C^2} = \frac{L}{4} \cdot \frac{2 \Pi^2}{L^2} \left( H_{33}^C \frac{\Pi^2}{L^2} - P \right) > 0$$

$\underbrace{\hspace{10em}}_{> 0}$



$$H_{33}^C \frac{\Pi^2}{L^2} - P > 0$$

$$P < H_{33}^C \frac{\Pi^2}{L^2}$$

$$P_{cr} = H_{33}^C \frac{\Pi^2}{L^2} \quad \text{Unstable!}$$

→ Euler Buckling Load!