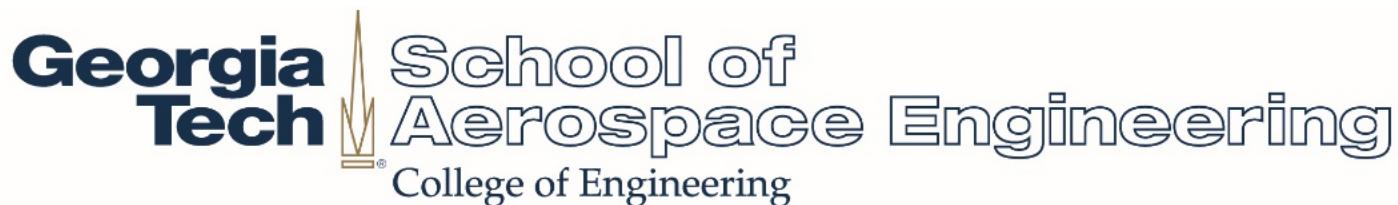
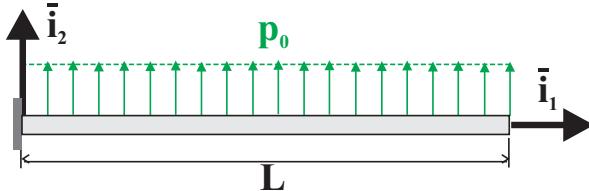


Rayleigh-Ritz Results and FEM overview

Claudio V. Di Leo
AE 3140 – Structural Analysis



Transverse Displacement



- 2nd-order Polynomial

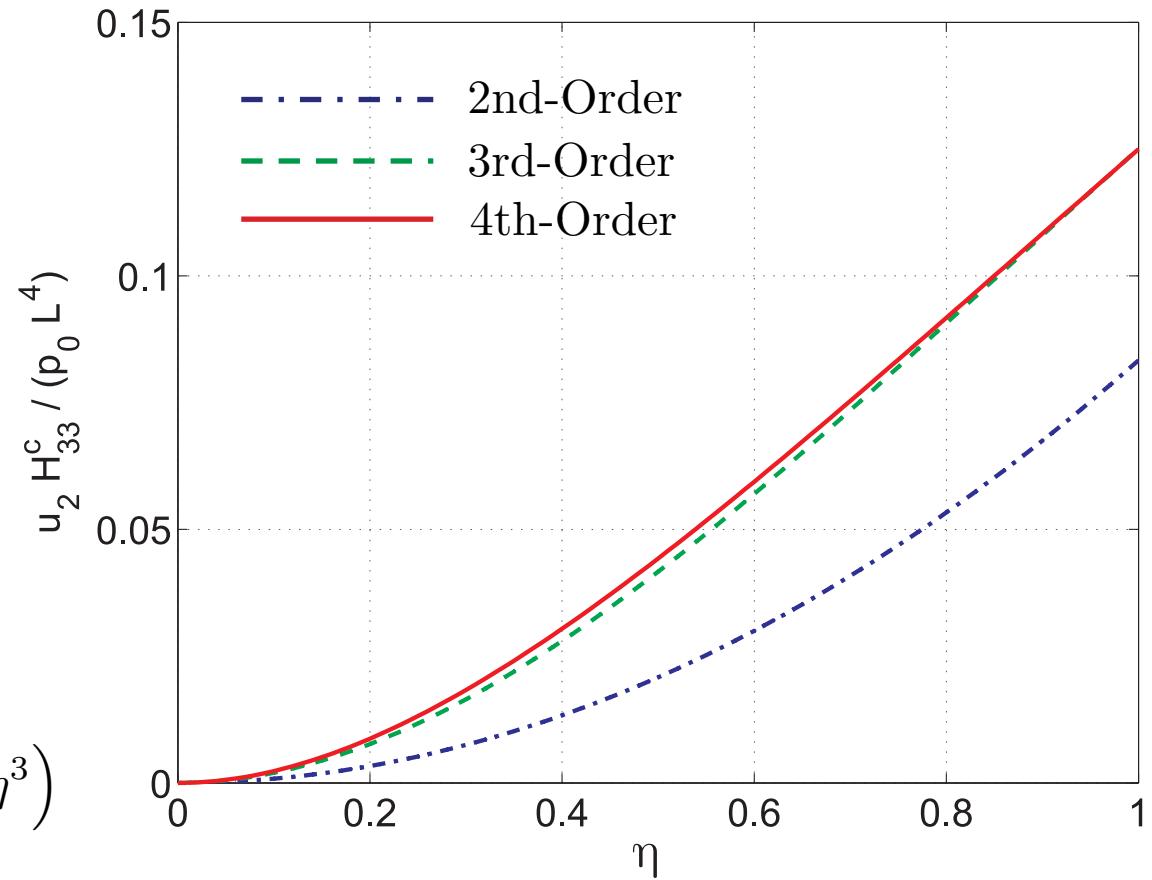
$$\bar{u}_2(\eta) = \frac{p_0 L^4}{H_{33}^c} \frac{1}{24} (2\eta^2)$$

- 3rd-order Polynomial

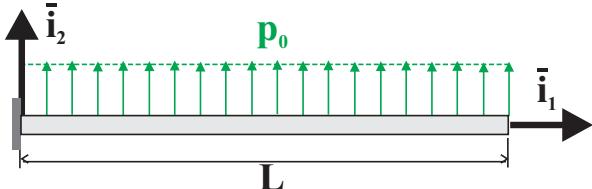
$$\bar{u}_2(x_1) = \frac{p_0 L^4}{H_{33}^c} \frac{1}{24} (5\eta^2 - 2\eta^3)$$

- 4th-order Polynomial (exact)

$$\bar{u}_2(x_1) = \frac{p_0 L^4}{H_{33}^c} \frac{1}{24} (6\eta^2 - 4\eta^3 + \eta^4)$$



Bending Moment and Shear?



- 2nd-order Polynomial

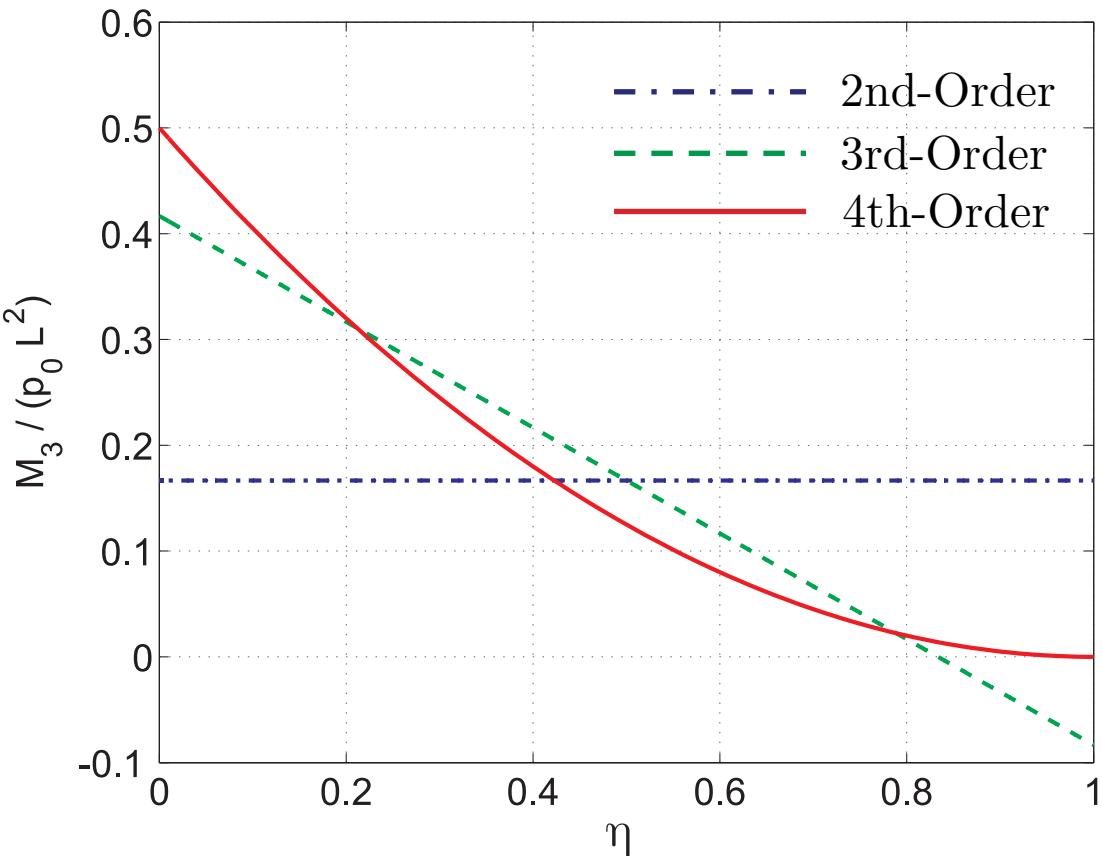
$$M_3 = H_{33}^c \frac{d^2 \bar{u}_2}{dx_1^2} = p_0 L^4 \frac{1}{24} (4)$$

- 3rd-order Polynomial

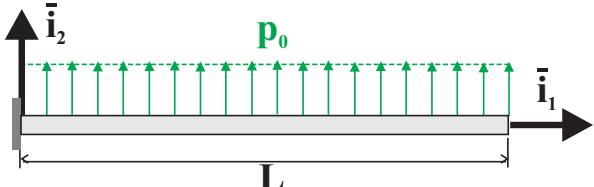
$$M_3 = p_0 L^4 \frac{1}{24} (10 - 12\eta)$$

- 4th-order Polynomial (exact)

$$M_3 = p_0 L^4 \frac{1}{24} (12 - 24\eta + 12\eta^2)$$



Bending Moment and Shear?



- 2nd-order Polynomial

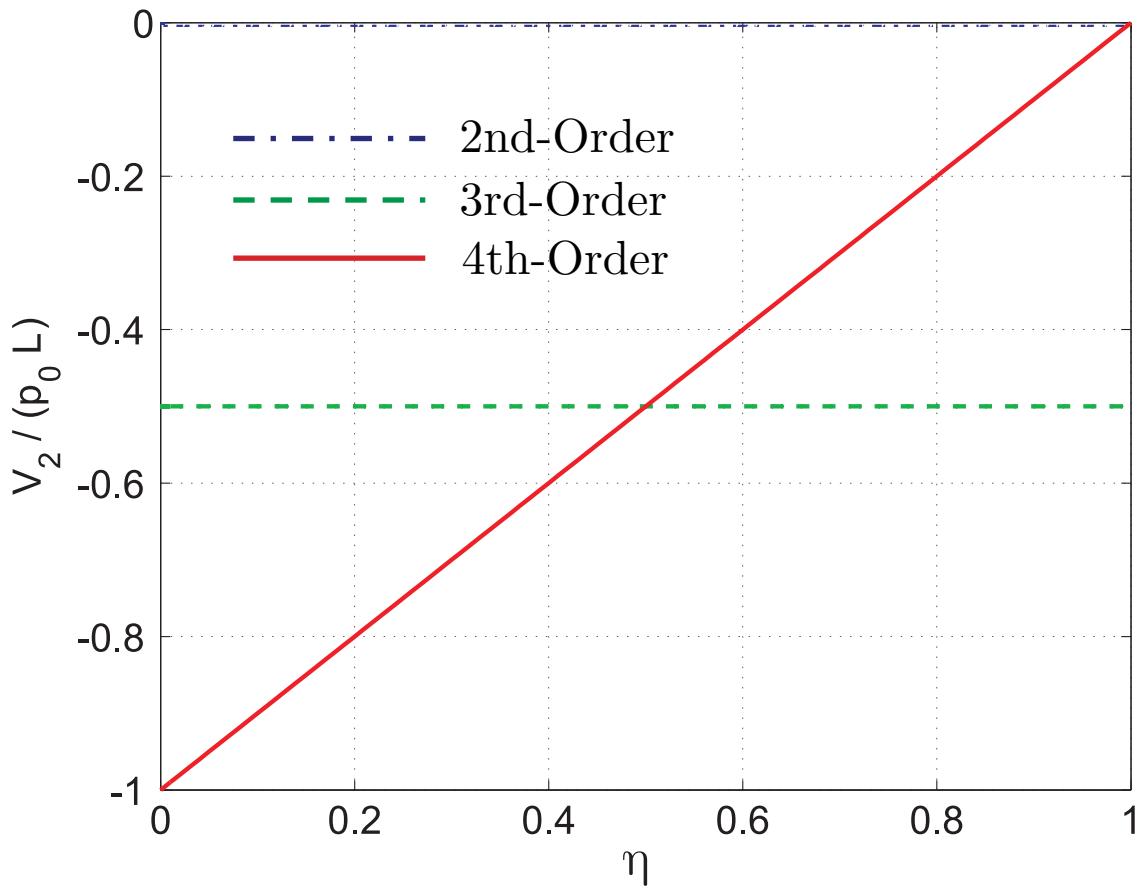
$$V_2 = -\frac{dM_3}{dx_1} = 0$$

- 3rd-order Polynomial

$$V_2 = p_0 L^4 \frac{1}{24} (-12)$$

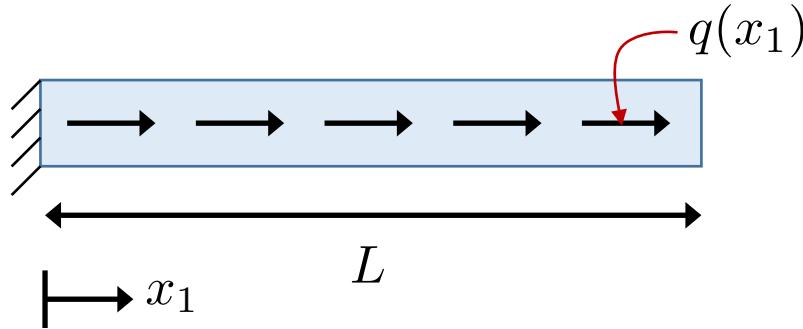
- 4th-order Polynomial (exact)

$$V_2 = p_0 L^4 \frac{1}{24} (-24 + 24\eta)$$



Finite Element Method – Example

- Beam in tension w/ distributed load.



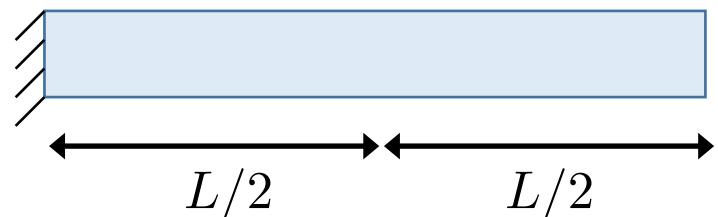
Exact Solution:

$$\bar{u}_1(x_1) = -\frac{q}{S} \left(\frac{x_1^2}{2} - Lx_1 \right)$$

- Potential

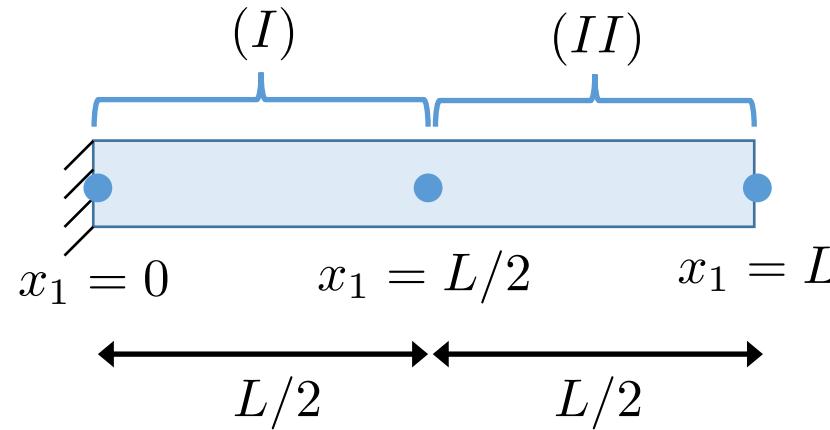
$$\Pi = \int_0^L \frac{1}{2} S \left(\frac{d\bar{u}_1}{dx_1} \right)^2 dx - \int_0^L q\bar{u}_1 dx$$

- Solve using Rayleigh-Ritz w/ Piecewise Linear Functions
- Express approximate \bar{u}_1 in terms of approximate displacements \hat{u}_i at points x_i
 - We call these points "nodes"
 - We call the regions between these points "elements"



Finite Element Method – Example

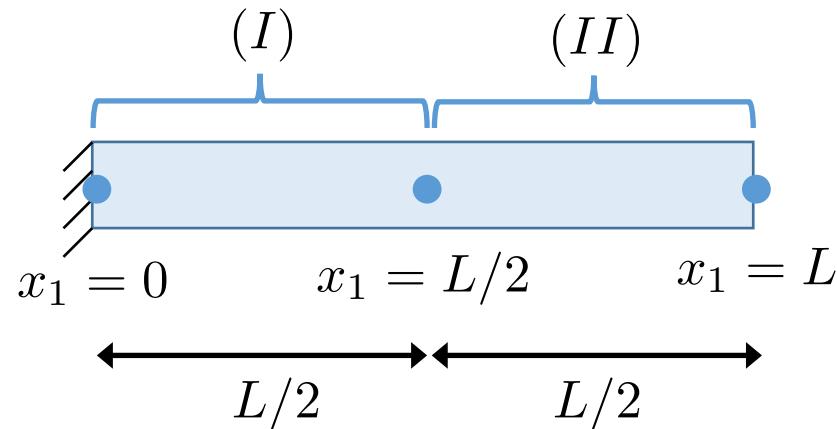
- Solve using Rayleigh-Ritz w/ Piecewise Linear Functions
- Express approximate \bar{u}_1 in terms of approximate displacements \hat{u}_i at points x_i
 - We call these points "nodes"
 - We call the regions between these points "elements"



$$\hat{u}_1 = \begin{cases} u_1 + \frac{u_2 - u_1}{L/2}x = \hat{u}^{(I)}(x) & \text{if } 0 \leq x < L/2 \\ u_2 + \frac{u_3 - u_2}{L/2}(x - \frac{L}{2}) = \hat{u}^{(II)}(x) & \text{if } L/2 \leq x < L \end{cases}$$

Finite Element Method – Example

- Solve using Rayleigh-Ritz w/ Piecewise Linear Functions
- Integrate and apply BCs ($u_1 = 0$)



$$\begin{aligned}\Pi = & \int_0^{L/2} \frac{1}{2} S \left(\frac{d\hat{u}^{(I)}}{dx} \right)^2 dx - \int_0^{L/2} q \hat{u}^{(I)} dx \\ & + \int_{L/2}^L \frac{1}{2} S \left(\frac{d\hat{u}^{(II)}}{dx} \right)^2 dx - \int_{L/2}^L q \hat{u}^{(II)} dx\end{aligned}$$

$$\Pi = \frac{1}{4L} (8Su_2^2 - 8Su_2u_3 + 4Su_3^2 - 2qu_2L^2 - qu_3L^2)$$

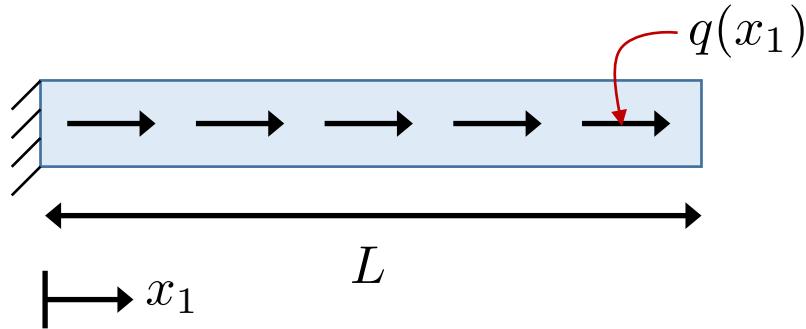
$$\frac{\partial \Pi}{\partial u_2} = \frac{1}{4L} (16Su_2 - 8Su_3 - 2qL^2) = 0$$

$$\frac{\partial \Pi}{\partial u_3} = \frac{S}{4L} (-8Su_2 + 8Su_3 - qL^2) = 0$$

$$S \begin{bmatrix} 16 & -8 \\ -8 & 8 \end{bmatrix} \begin{bmatrix} u_2 \\ u_3 \end{bmatrix} = \begin{bmatrix} 2qL^2 \\ qL^2 \end{bmatrix}$$

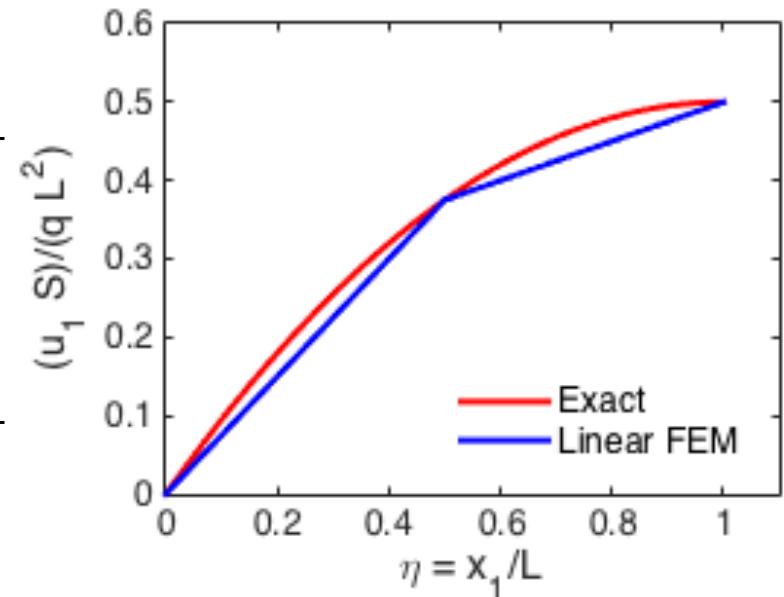
K u f

Finite Element Method – Example

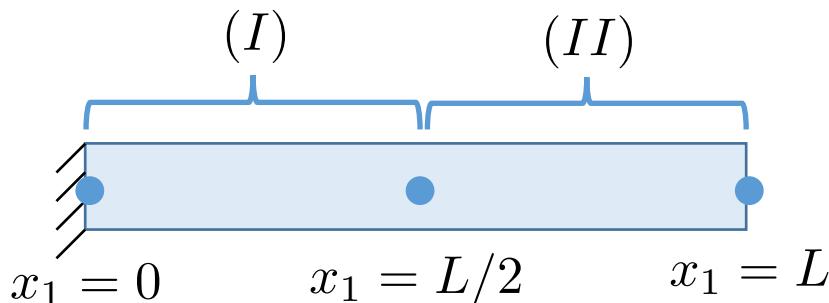


Exact Solution:

$$\bar{u}_1(x_1) = -\frac{q}{S} \left(\frac{x_1^2}{2} - Lx_1 \right)$$



- Solved using Rayleigh-Ritz with piecewise-linear functions over two "elements"
- This solution is a basic finite element solution
- For linear elasticity, FEM is Rayleigh-Ritz!
- It can be shown that as we refine the discretization $\hat{\Pi} \rightarrow \Pi$ and $\hat{u} \rightarrow u$

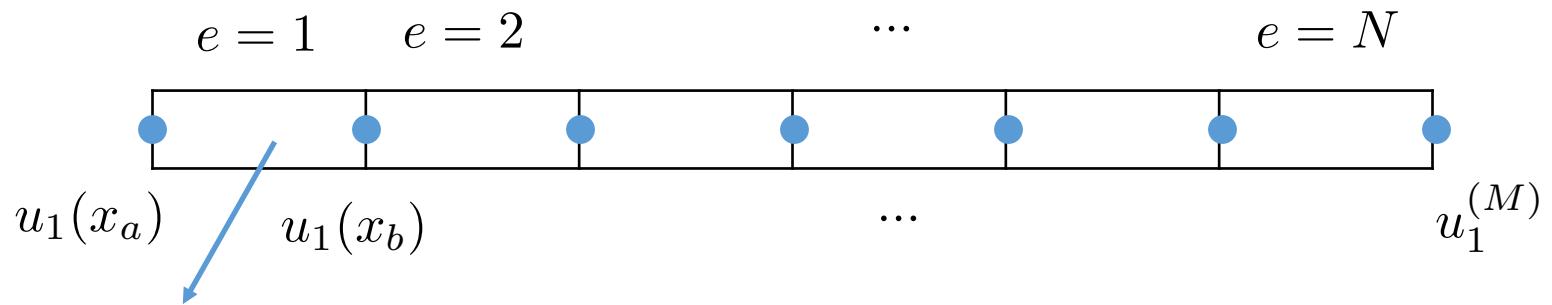


$$\hat{u}_1 = \begin{cases} \frac{3}{4} \frac{qL}{S} x & \text{if } 0 \leq x < L/2 \\ \frac{1}{4} \frac{qL^2}{S} + \frac{1}{4} \frac{qL}{S} x & \text{if } L/2 \leq x < L \end{cases}$$

Finite Element Method – High Level Overview

- Consider the total energy of the system as given by a sum of the energies over multiple smaller "elements"

$$\Pi = \sum_{e=1}^N \Pi^e \quad \Pi^e = \int_{\Omega^e} \left[\frac{1}{2} S(x) \left(\frac{du^e}{dx} \right)^2 - p(x) u^e(x) \right] dx - u^e(x_a) F_A - u^e(x_b) F_B$$



Each element is approximated with
simple polynomial shape functions

$$u_1^e = \sum_{j=1}^2 h_j u_1^{(j)}$$



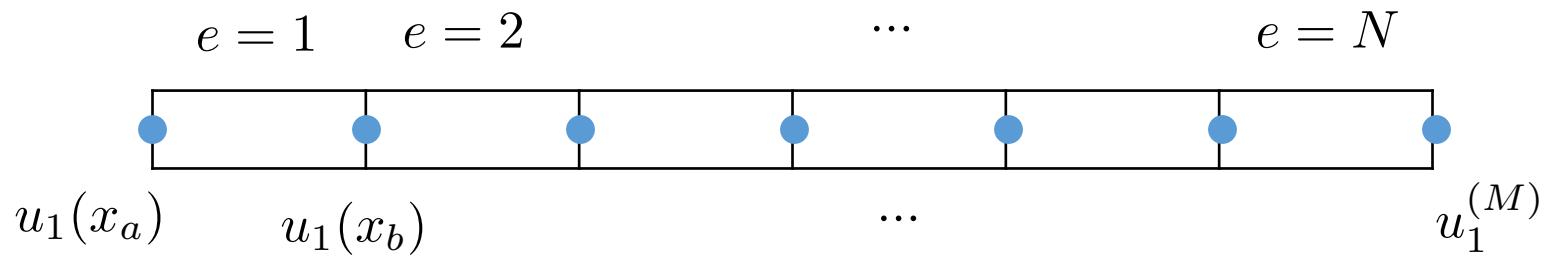
$$u_1^e(x) = \mathbf{H}(x)\mathbf{u}^e$$

$$\frac{du_1^e(x)}{dx} = \mathbf{B}(x)\mathbf{u}^e$$

Finite Element Method – High Level Overview

- Consider the total energy of the system as given by a sum of the energies over multiple smaller "elements"

$$\Pi = \sum_{e=1}^N \Pi^e \quad \Pi^e = \int_{\Omega^e} \left[\frac{1}{2} S(x) \left(\frac{du^e}{dx} \right)^2 - p(x) u^e(x) \right] dx - u^e(x_a) F_A - u^e(x_b) F_B$$



$$\Pi^e = \frac{1}{2} \mathbf{u}^{e,T} \mathbf{K}^e \mathbf{u}^e - \mathbf{u}^{e,T} \mathbf{f}^e$$

$$\delta\Pi = \sum_e \delta\Pi^e = 0$$

$$\mathbf{K}^e = \int_{\Omega^e} S(x) \mathbf{B}(x)^T \mathbf{B}(x) dx$$

$$\mathbf{f}^e = \int_{\Omega^e} \mathbf{H}^T(x) q(x) dx + \sum_i \mathbf{H}^T(x_i) \mathbf{F}_i$$

$$\boxed{\mathbf{K}\mathbf{u} = \mathbf{f}}$$

Solve system of linear equations with M unknowns (minus boundary conditions)!