Georgia Institute of Technology School of Aerospace Engineering Atlanta, Georgia 30332

AE 6115 — Fundamentals of Aerospace Structural Analysis — Spring 2020

Quiz No.1

Problem 1. Double Cantilevered Beam with Concentrated Moment (100 Points)

Consider the multi-material beam cantilevered on both ends as shown in Fig. 1. The beam is subjected to a concentrated moment of magnitude Q at its mid-span. The cross-section of the beam (right of Fig. 1) is composed of a multi-layered material with the left layer having a Young's modulus E_A , and the right layer having a Young's modulus E_B . Assume that $E_A > E_B$.

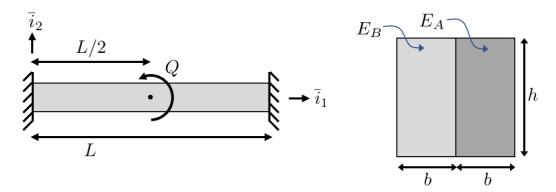
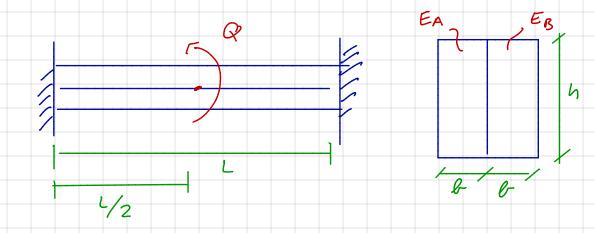


Figure 1: Multi-material beam cantilevered on both ends with a concentrated moment at its mid-span.

- a) Solve for the displacement components $u_1(x_1, x_2, x_3)$, $u_2(x_1, x_2, x_3)$, and $u_3(x_1, x_2, x_3)$. You may leave your answer in terms of any non-zero stiffnesses.
- b) Solve for the axial stress $\sigma_1(x_1, x_2, x_3)$.
- c) Sketch how you expect the beam to deform (i.e. sketch the u_2 transverse displacement). Sketch the internal moment distribution (i.e. sketch $M_3(x_1)$)
- d) Extra credit (attempt only after you've exhausted the other parts). Solve for the stiffnesses H_{22}^c , H_{33}^c , H_{23}^c and S as a function of the geometric properties $\{L, h, b\}$ and the material properties $\{E_A, E_B\}$



a) CONSIDER HOW THE BEAM WILL DEFORM

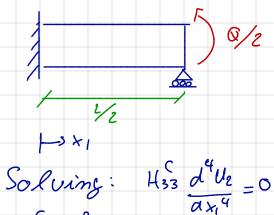


* By impedion His=0

* WE CAN SEE THE SOLUTION IS
ANTISYMMETRIC AND WE MAY SOLVE
ONLY FOR HALF OF THE BEA

* FURTHER, WE CAN NOTE THAT DUE TO SYMMETRY
THERE IS NO UZ DIS PEACMENT AT THE MID-SPAN

WE MAY THUS TREAT THE PROBLEM AS



B. C.'s
$$Q \times_{1} = 0$$

$$U_{2} = 0 \quad dV_{2}/dx_{1} = 0$$

$$Q \times_{1} = L$$

$$U_{2} = 0 \quad H_{33} = 0$$

$$Q \times_{1} = 0$$

Applying B.C.
$$H_{53}^{C} \frac{\alpha^{2} d_{2}}{\alpha x_{1}^{2}} = \frac{Q}{2}$$
 $\frac{Q}{2} = A + B \quad B = \frac{Q}{2} - A + \frac{1}{2}$
 $H_{53}^{C} \frac{\alpha^{2} d_{2}}{\alpha x_{1}^{2}} = A \left(x_{1} - \frac{1}{2} \right) + \frac{Q}{2}$
 $H_{53}^{C} \frac{\alpha^{2} d_{2}}{\alpha x_{1}} = A \left(\frac{x_{1}^{C}}{2} - \frac{1}{2} x_{1} \right) + \frac{Q}{2} x_{1} + \frac{Q}{2} x_{1}^{C} + \frac{Q}{2} x_{1}^{C} = 0$
 $H_{53}^{C} \quad U_{2} = A \left(\frac{x_{1}^{C}}{6} - \frac{1}{4} x_{1}^{C} \right) + \frac{Q}{2} x_{1}^{C} + \frac{Q}{2} x_{1}^{C} = 0$
 $V_{00} \quad U_{0} \quad U_{0}^{C} \quad U_{0$

Apprach #2

* ONE MAY ERCHLY SOLVE THE PROBLEM W/OUT MAKING USG OF SYMMETRY.

$$Q \times 1 = Q \qquad U_2 = 0 \qquad (3)$$

$$d V_2 / d \times 1 = Q \qquad (4)$$

$$@x_1=\frac{L}{2} & x_1=\frac{L}{2}$$

(5)
$$U_2^{(x_1=\frac{1}{2})} = U_2^{(x_1=\frac{1}{2})}$$

(6)
$$dU_2^{\prime}/dx_1 (x_1=4/2) = -dU_2^{\prime\prime}/dx_1 (x_1=4/2)$$

RIGHT HALF

XI C

$$(7) \quad H_{23} \quad d^{3}u_{2} / dx_{1}^{3}(x_{1} = L/z) = H_{33} \quad d^{3}u_{2} / d\bar{x}_{1}^{3} \quad (\bar{x}_{1} = L/z) \quad (\bar{y}_{1}^{2} \quad \bar{y}_{2}^{3}) \quad (\bar{y}_{1}^{2} \quad \bar{y}_{2}^{2}) \quad (\bar{y}_{2}^{2} \quad \bar{y}_{2}^{2}) \quad (\bar{y}_{1}^{2} \quad \bar{y}_{2}^{2}) \quad (\bar{y}_{2}^{2} \quad \bar{y}_{2}^{2}) \quad (\bar{y}_{1}^{2} \quad \bar{y}_{2}^{2}) \quad (\bar{y}_{1}^{2}$$

$$H_{23}^{C} \frac{dV_{2}}{dx_{1}^{4}} = 0$$

$$H_{33}^{C} \frac{d^{4} u_{2}}{d \bar{x}_{i}^{4}} = 0$$

$$H_{33}^{C} \frac{d^{3} u_{2}}{d x_{1}^{3}} = A_{1}$$

$$H_{33}^{C} \frac{d^{3}Uz}{d\bar{x}_{1}^{3}} = Az$$

$$H_{33}^{C} \frac{d^{3}U_{2}}{d\bar{x}_{i}^{3}} = -A_{1}$$

$$H_{33}^{C} \frac{d^{3}U_{2}}{dx_{1}^{3}} = A_{1}$$

$$H_{33}^{C} \frac{d^{3}U_{2}}{dx_{1}^{2}} = A_{1} \times_{1} + B_{1}$$

$$H_{33}^{C} \frac{d^{2}U_{2}}{dx_{1}^{2}} = A_{1} \times_{1} + B_{1}$$

$$H_{33}^{C} \frac{d^{2}U_{2}}{dx_{1}^{2}} = -A_{1} \times_{1} + B_{2} + Q$$

$$H_{33}^{C} \frac{d^{2}U_{2}}{dx_{1}^{2}} = -A_{1} \times_{1} + B_{2} + Q$$

$$H_{33}^{C} \frac{d^{2}U_{2}}{dx_{1}^{2}} = A_{1}(x_{1} - L) + B_{2} + Q$$

$$H_{33}^{C} \frac{d^{2}U_{2}}{dx_{1}^{2}} = A_{1}(x_{1} - L) + B_{2} + Q$$

$$H_{33}^{C} \frac{d^{2}U_{2}}{dx_{1}^{2}} = A_{1}(x_{1} - L) + B_{2} + Q$$

$$H_{33}^{C} \frac{d^{2}U_{2}}{dx_{1}^{2}} = -A_{1} \times_{1} \times_{1}$$

$$\beta_2 = A_1 L - \frac{Q}{Z}$$

$$3 d u_2 - A_1 \left(x_1^2 - L x_1 \right) + Q x_1 + \frac{Q}{Z}$$

$$H_{33}^{C} U_{2} = A_{1} \left(\frac{x_{1}^{3} - Lx_{1}^{2}}{6} \right) + \frac{QX_{1}^{3}}{4} + D_{1}^{70} U_{5} i N G (1)$$

Using (5):

$$H_{53} = \frac{3}{2} \frac{Q}{6} \left(\frac{x_1^3 - Lx_1^2}{4} \right) + \frac{Qx_1^2}{4}$$

$$H_{33}^{C} \frac{d^{3}U_{2}}{d\bar{x}_{i}^{3}} = -A_{1}$$

$$H_{33} = \frac{Q^2 U_2}{a \bar{x}_1^2} - A_1 \bar{x}_1 + B_2$$

$$H_{33} \frac{d^2 U_2}{d\bar{x_1}^2} = -A_1 \bar{x_1} + B_2$$

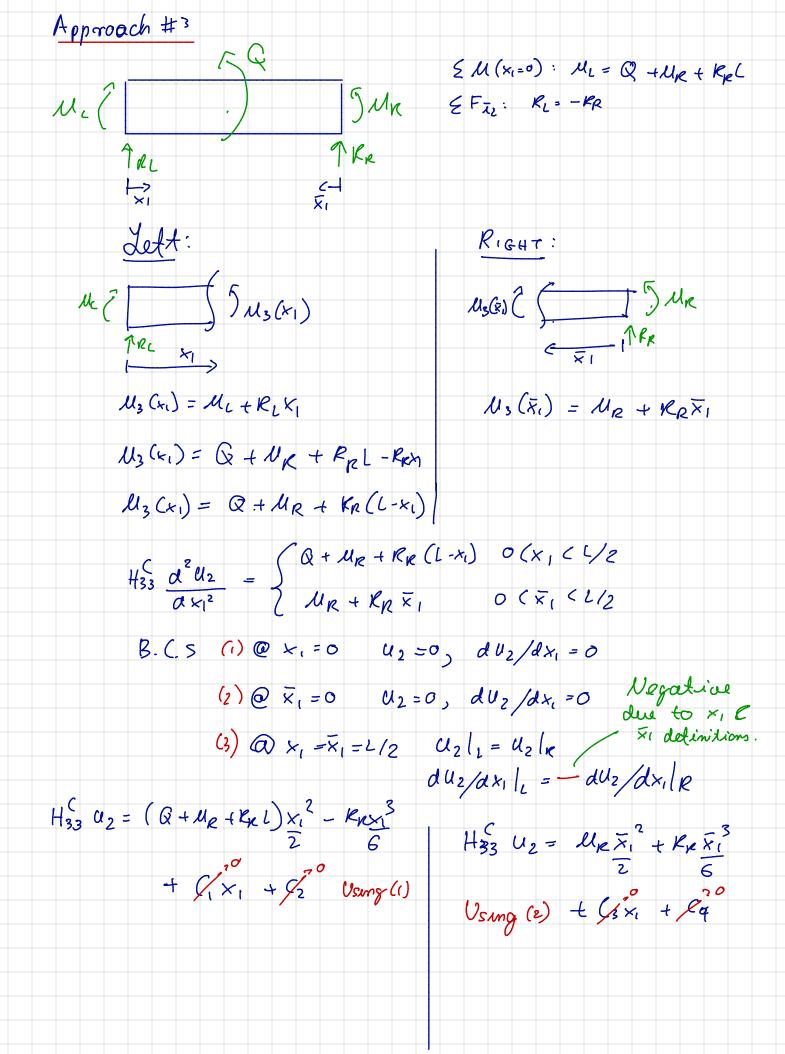
$$H_{33}^{C} \frac{dU_{2}}{d\bar{x_{1}}} = -A_{1} \frac{\bar{x_{1}}^{2}}{2} + B_{2} \bar{x_{1}}$$

$$+C_{2}^{70} \quad U_{sing} \quad (4)$$

$$H_{33} \quad U_{2} = A_{1} \left(-\frac{\bar{x}_{1}^{3}}{6} + \frac{L\bar{x}_{1}^{2}}{4} \right) - \bar{\alpha}\bar{x}_{1}^{2} + D_{2}^{70} \quad U_{SiNG}(2)$$

$$-\frac{A_{1}L^{3}}{24} + \frac{QL^{2}}{16} = \frac{A_{1}L^{3}}{24} - \frac{QL^{2}}{16} \longrightarrow A_{1} = \frac{24}{16} \frac{Q}{L} = \frac{3}{2} \frac{Q}{L}$$

* H₃₃
$$U_2 = -\frac{3}{2} \frac{Q}{L} \left(\frac{\overline{x_1}}{6} - \frac{L\overline{x_1}^2}{12} \right)$$



$$H_{33}^{C} u_{2} = (Q + M_{R} + K_{R} L) \times (2 - K_{R} L)^{3}$$
 $H_{33}^{C} u_{2} = M_{R} \times (2 + K_{R} L)^{3} \times (2 + K_{R} L)^{3} \times (3 \times 1)^{3} \times (3 \times 1)^{$

$$H_{33}^{C}$$
 $U_{2} = U_{R} \bar{x}_{1}^{2} + R_{R} \bar{x}_{1}^{2}$

$$(Q + M_R + R_R L) \frac{L^2}{8} - \frac{R_R L^3}{6.8} = M_R \frac{L^2}{8} + \frac{R_R L^3}{6.8}$$

$$Q L^2 = R_K L^2 \left(\frac{1}{6.8} + \frac{1}{6.8} - \frac{1}{8} \right) \longrightarrow R_R = -\frac{3}{2} \frac{G}{L}$$

$$H_{33}^{C}$$
 $U_{2} = M_{R} \times_{1}^{2} / 2$

$$-\frac{3}{2} \frac{Q}{Q} \left(\frac{L \times_{1}^{2}}{6} - \frac{\times_{1}^{3}}{6} \right)$$

Using (3.2)

$$M_{R} = \frac{L^{2}}{8} = \frac{3}{2} \frac{Q}{L} \left(\frac{L}{6} \frac{L^{2}}{4} - \frac{L^{3}}{6.8} \right) = -M_{R} \frac{L^{2}}{8} + \frac{3}{2} \frac{Q}{L} \frac{L^{3}}{6.8}$$

$$U_{R} \frac{L^{2}}{4} = \frac{3}{2} \frac{G}{2} \left(\frac{L^{3}}{6.8} + \frac{L^{3}}{6.4} - \frac{L^{3}}{6.8} \right)$$

$$U_{R} = \frac{3}{2} \frac{Q}{L} \cdot \frac{1}{6}$$

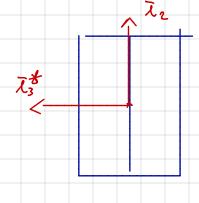
$$H_{33}^{C}U_{2} = -\frac{3}{2}C\left(\frac{2\times i^{3}}{12} - \frac{\times i^{3}}{6}\right)$$

PART B)

LETS DEFINE A COORDINATE

SYSTEM WHICH WE NOTE IS NOT

CENTROIDAL BUT WILL HELP DEFINE OF



$$O_1 = -E(\bar{x}_3) \cdot x_2 K_3$$

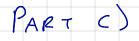
$$W_{1TH} = \begin{cases} E_A & \bar{x}_3 > 0 \\ E_B & \bar{x}_3 < 0 \end{cases}$$

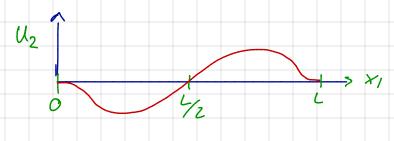
For
$$x_1 < \frac{L}{2}$$

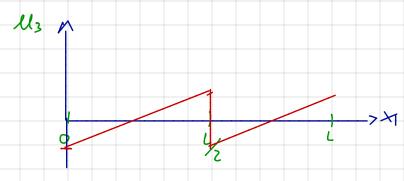
$$\sigma_1 = E(\bar{x}_3) \times_2 Q\left(\frac{3}{2}x_1 - \frac{L}{4}\right)$$

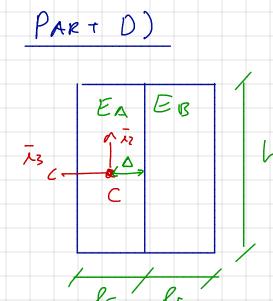
For
$$x_1 > \frac{L}{2}$$

$$\sigma_1 = -E(\bar{x}_3) \times_2 \frac{Q}{2} \left(\frac{3}{2}(L - x_1) - \frac{L}{4}\right)$$









$$S = (E_A + E_B)(hb)$$

$$H_{23}^{C} = 0$$

$$h \quad H_{23}^{C} = \frac{g - h^3}{12}(E_A + E_B)$$

$$H_{22}^{C} = E_A \left(\frac{hb^3 + (gh)(\Delta - g^2)}{(2^2)^2}\right)$$

$$+ E_B \left(\frac{hb^3 + (gh)(\Delta + g^2)}{(2^2)^2}\right)$$

$$\Delta = S_3 / S$$

$$\Delta = \left[\int_0^E E_A \times_3 + \int_{-e}^E E_B \times_3^2 \right] h$$

$$(E_A + E_B)(hb)$$

$$\Delta = \frac{g - (E_A - E_B)}{2(E_A + E_B)}$$