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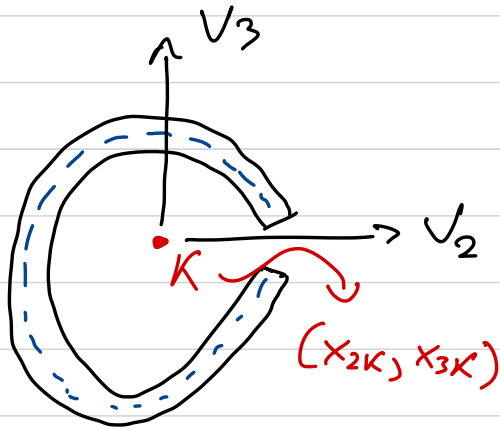
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# Shear Center



\* Assume the beam is subjected to shear forces  $V_2$  and  $V_3$  only ( $M_1 = 0$ )

\* The resulting shear flow should generate no torque.

\* The problem above is not well defined, the lines of action for the shear forces are not given.

\* The torque generated by the shear flow must vanish when computed w.r.t. the shear center

$$M_{1K} = \int_{C_1} \tau \cdot r_K ds = 0$$

\* A beam bends without twisting, if and only if, the transverse loads are applied about the shear center.

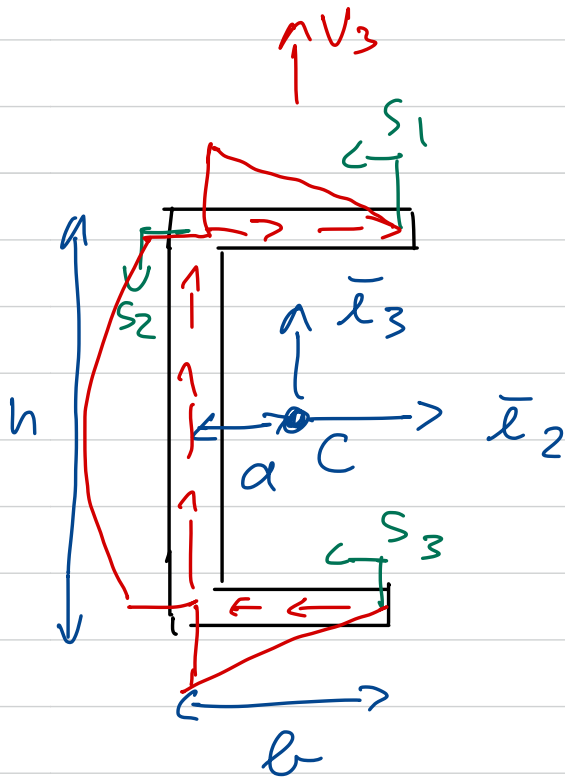
1) Find the location at K  
by solving

$$U_K = \int_{C_1} k r_k = 0$$

$$\begin{aligned} \Sigma U: U_K &= U_A - (x_{2K} - x_{2A}) \cdot V_3 \\ &\quad + (x_{3K} - x_{3A}) V_2 = 0 \end{aligned}$$

$$U_A = (x_{2K} - x_{2A}) V_3 - (x_{3K} - x_{3A}) V_2$$

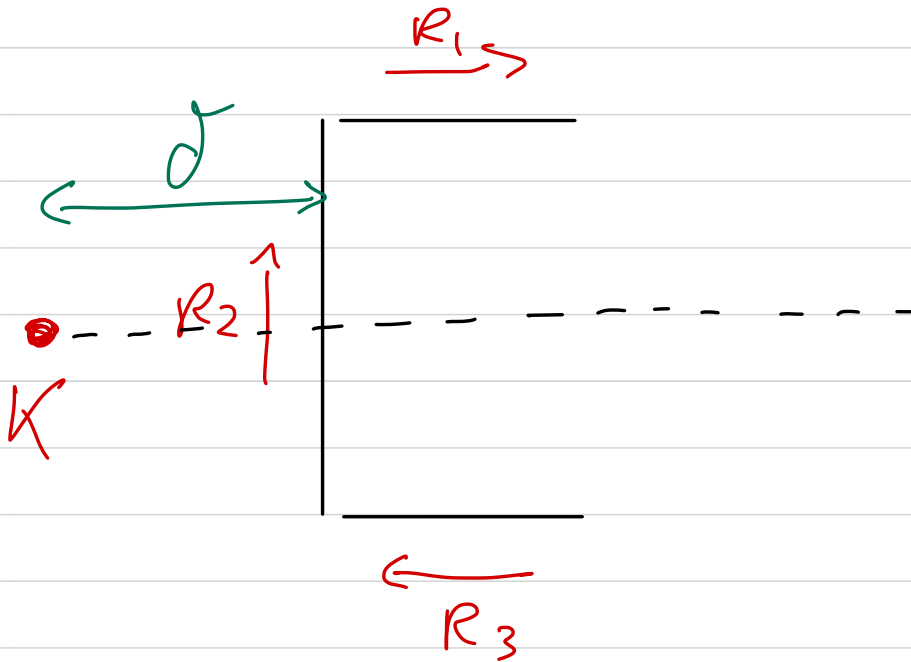
# Shear Center in a "I"-channel



$$f(s_1) = - \frac{E t h s_1}{2} \frac{V_3}{H_{22}^C}$$

$$f(s_2) = E t \left( - \frac{b h}{2} - \frac{s_2 h}{2} + \frac{s_2^2}{2} \right) \frac{V_3}{H_{22}^C}$$

$$f(s_3) = \frac{E t h s_3}{2} \frac{V_3}{H_{22}^C}$$



## Procedure

- 1) If there is a symmetry plane the shear center must be on it.
- 2) Apply a load  $V_3$  only ( $V_2=0$ ) solve for  $x_{2K}$
- 3) Apply a load  $V_2$  only ( $V_3=0$ ) solve for  $x_{3K}$ .

$$\begin{aligned} M_{1K} = & \int_0^b t(s_1) \cdot \frac{h}{2} ds \\ & - \int_0^h t(s_2) \cdot \sigma ds \\ & - \int_0^b t(s_3) \frac{h}{2} ds = 0 \end{aligned}$$

$$M_{1K} = -R_1 \frac{h}{2} + R_2 \sigma - R_3 \frac{h}{2} = 0$$

$$\sigma = \frac{\frac{h}{2} (R_1 + R_3)}{R_2}$$

$$R_1 = - \int_0^b - \frac{E \epsilon h S_1}{2} \frac{V_3}{H_{22}^c} ds$$

$$R_1 = \frac{E \epsilon h b^2}{4} \frac{V_3}{H_{22}^c}$$

$$R_2 = - \int_0^h E \epsilon \left( -\frac{b h}{2} - \frac{S_2 h}{2} + \frac{S_2^2}{2} \right) \frac{V_3}{H_{22}^c} ds$$

$$R_2 = E \epsilon \left( \frac{b h^2}{2} + \frac{h^3}{12} \right) \frac{V_3}{H_{22}^c}$$

$$H_{22}^c = E \epsilon \left( \frac{h^3}{12} + \frac{b h^2}{2} \right) \quad \curvearrowright$$

$$R_2 = V_3$$

$$R_3 = \int_0^b \frac{E \epsilon h S_3}{2} \frac{V_3}{H_{22}^c} ds$$

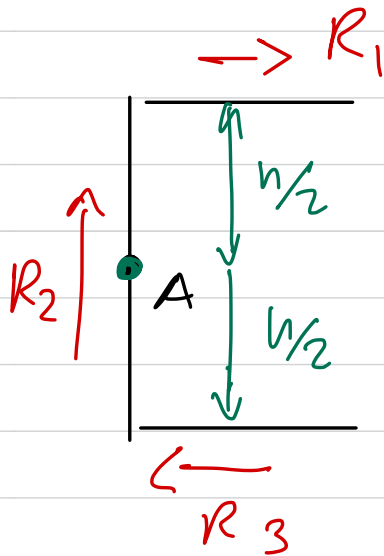
$$= \frac{E \epsilon h b^2}{4} \frac{V_3}{H_{22}^c}$$

$$\varepsilon F \tilde{x}_2 : R_1 - R_3 = 0 \quad R_1 = R_3$$

$$\begin{aligned}
 \sigma &= \frac{h}{2} \frac{(R_1 + R_3)}{R_2} \\
 &= \frac{h}{2} \frac{1}{2} E t h l^2 \frac{V^3}{H_{22}^c} \cdot \frac{1}{V_3} \\
 &= \frac{1}{4} \frac{E t h^2 l^2}{H_{22}^c}
 \end{aligned}$$

$$\sigma = \frac{1}{4} \frac{h^2 l^2}{\left( \frac{h^3}{12} + \frac{e h^2}{2} \right)} = \frac{3 b}{h/b + 6}$$

Approach #2:



$$U_{1A} = -\frac{h}{2} R_1 - \frac{h}{2} R_3$$

$$-\frac{h}{2} (R_1 + R_3) = (\underbrace{x_{2R} - x_{2A}}_{\sigma}) V_3$$

$$\sigma = -\frac{h}{2} \frac{(R_1 + R_3)}{V_3}$$