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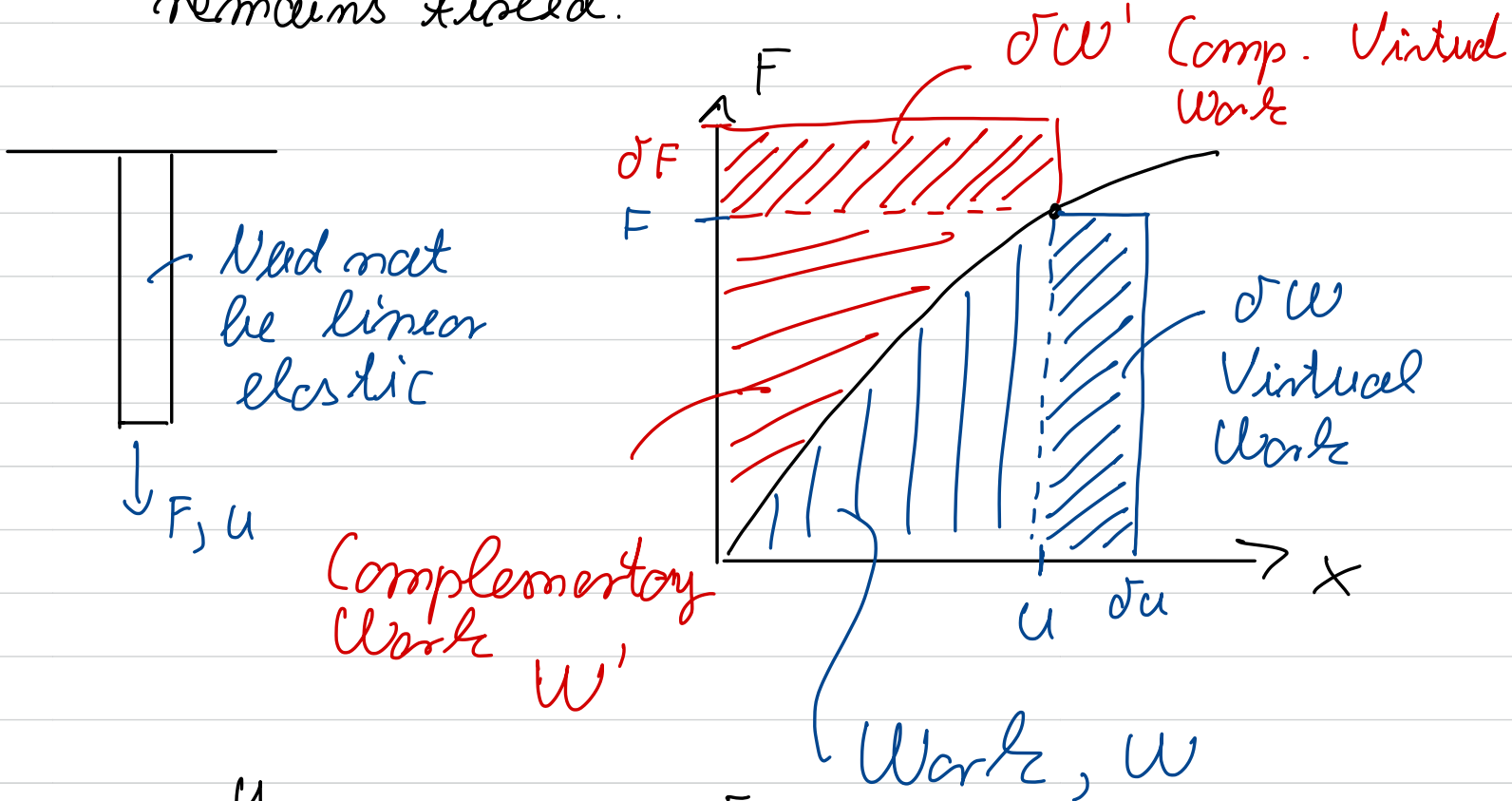
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# Complementary Virtual Work

\* Defined as the work done by virtual forces acting through real displacements.

\* The real quantity, i.e. the displacement remains fixed.



$$W = \int_0^u F du, \quad W' = \int_0^F u dF$$

\* For a non-linear elastic material,  $W$  and  $W'$  are not equal.

\* For both linear and non-linear materials

$$W + W' = F \cdot u$$

For a linear elastic material,  
 $F = k \cdot u$ , and  $W = W'$

$$W = \int_0^u F du = \int_0^u k u du = \frac{k u^2}{2} = \frac{F \cdot u}{2}$$

$$W' = \int_0^F u dF = \int_0^F \frac{F}{k} dF = \frac{F^2}{2k} = \frac{F \cdot u}{2}$$

Principle of Complementary Virtual Work (PCVW)

\* The deformation in a deformable system is compatible if and only if

$$\delta W' = \delta W_E' + \delta W_I' = 0$$

for all statically admissible virtual forces.

\* Statically admissible virtual forces satisfy equilibrium.

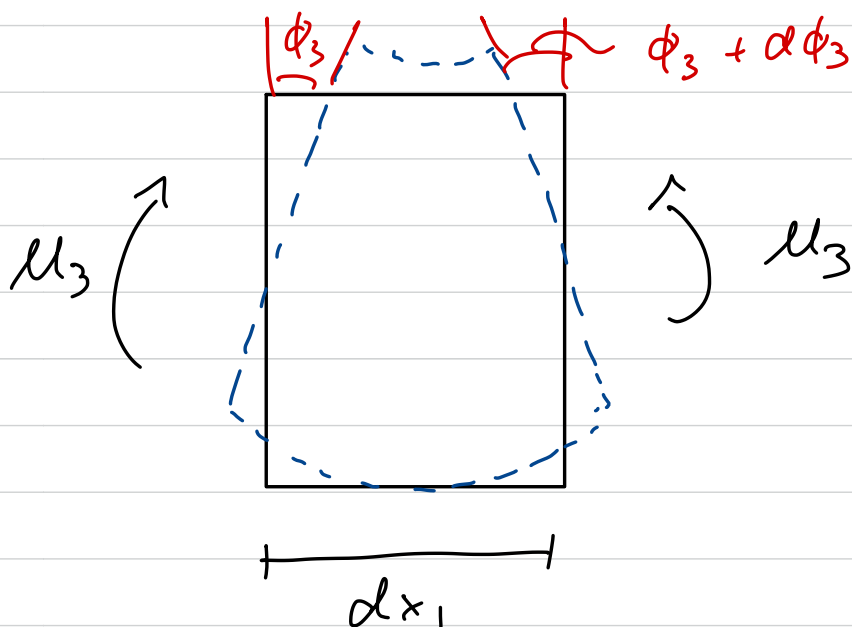
\* PCVW is equivalent to compatibility

\* PVW is equivalent to equilibrium.

For elastic bodies, the points within a body can move and hence the internal forces can do work

→ Need to compute  $\delta W_I$  and  $\delta W_I'$  for elastic beams.

## Beam in Bending



$$dW_I = -\mu_3 \phi_3 + \mu_3 (\phi_3 + d\phi_3)$$

$$dW_I = \mu_3 \left( \frac{d\phi_3}{dx_1} \right) dx_1$$

$$W_I = - \int_0^L \mu_3 \frac{d\phi_3}{dx_1} dx_1 = - \int_0^L \mu_3 \kappa_3 dx_1$$

Virtual Quantities.

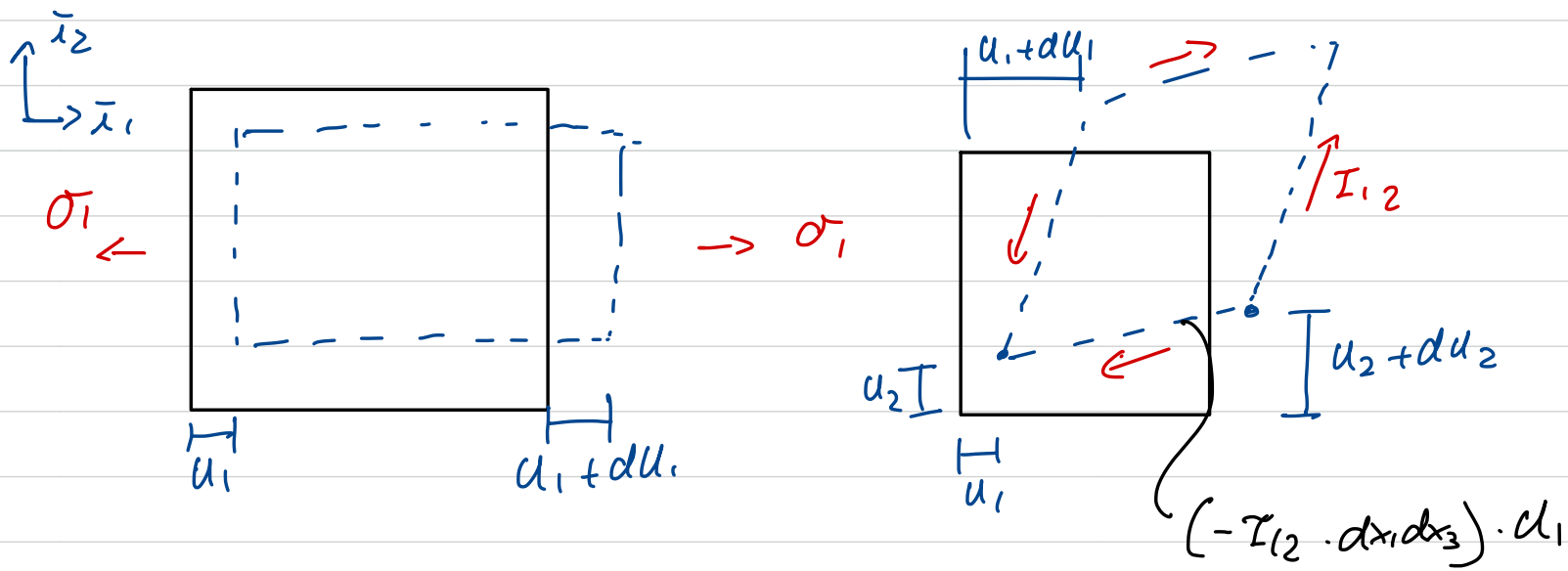
$$\delta W_I = - \int_0^L \mu_3 \cdot \delta \kappa_3 \cdot dx_1$$

Virtual Curvature

$$\delta W_I' = - \int_0^L \delta \mu_3 \cdot \kappa_3 dx_1$$

Virtual Moment

# 3D - Solid



## Axial

$$W_I = - \int_V \sigma_1 \cdot \left( \frac{\partial u_1}{\partial x_1} \right) dx_1 dx_2 dx_3$$

$$= - \int_V \sigma_1 \epsilon_1 dV$$

## Shear

$$dW_I = -(I_{12} dx_1 dx_3) u_1$$

$$(I_{12} dx_1 dx_3) (u_1 + du_1)$$

$$- (I_{12} dx_2 dx_3) u_2$$

$$(I_{12} dx_2 dx_3) (u_2 + du_2)$$

$$dW_I = -T_{12} \left( \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) dx_1 dx_2 dx_3$$

$$W_I = - \int_V T_{12} \left( \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) dV$$

$$= - \int_V T_{12} \gamma_{12} dV$$

Generally

$$W_I = - \int_V (\sigma_1 \epsilon_1 + \sigma_2 \epsilon_2 + \sigma_3 \epsilon_3 + T_{12} \gamma_{12} + T_{13} \gamma_{13} + T_{23} \gamma_{23}) dV$$

$$W_I = - \int_V \underline{\sigma}^T \underline{\epsilon} dV$$

Stress  
array

Strain  
array.

# Euler-Bernoulli Beams

$$U_I = - \int_V \sigma_1 \epsilon_1 dV$$

$$= - \int_V \sigma_1 (\bar{\epsilon}_1 + x_3 \kappa_2 - x_2 \kappa_3) dV$$

$$U_I = - \int_0^L \left[ \bar{\epsilon}_1 \underbrace{\left( \int_A \sigma_1 dA \right)}_{N_1} + \kappa_2 \underbrace{\left( \int_A \sigma_1 x_3 dA \right)}_{\mu_2} + \kappa_3 \underbrace{\left( - \int_A \sigma_1 x_2 dA \right)}_{\mu_3} \right] dx_1$$

$$U_I = - \int_0^L (N_1 \bar{\epsilon}_1 + \mu_2 \kappa_2 + \mu_3 \kappa_3) dx_1$$

## Virtual Work

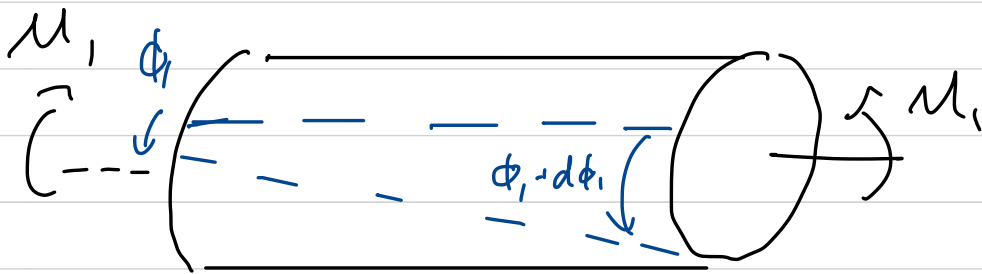
$$\delta W_I = - \int_0^L \left( \mu_1 \cdot \delta \bar{E}_1 + \mu_2 \delta K_2 + \mu_3 \delta K_3 \right) dx_1$$

## Complementary Virtual Work

$$\delta W_I^I = - \int_0^L \left( \bar{E}_1 \cdot \delta \mu_1 + K_2 \cdot \delta \mu_2 + K_3 \cdot \delta \mu_3 \right) dx_1$$

## Torsion

$$W_I = - \int_V \tau_s \cdot \gamma_s dV = - \int_0^L \mu_1 \cdot \kappa_1 dx_1$$



$$\delta W_I = - \int_0^L \mu_1 \cdot \delta \kappa_1 dx_1$$

$$\delta W_I^I = - \int_0^L \kappa_1 \cdot \delta \mu_1 \cdot dx_1$$