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External forces can be of two kinds:

\* Body forces

↳ from fields: gravity, electromagnetic, etc.

\* Surface forces:

↳ short range interaction forces across  
 $\partial B$  resulting from interactions between  
 $B$  and its surrounding.

Mathematically:

Body Forces:  $\int_B \underline{b} dm = \int_B \underline{\underline{f}} \rho dv$

$\underline{b}$  = body force per unit mass (ex:  $\frac{N}{kg}$ )

Surface Forces:  $\int_{\partial B} \underline{f}^{\text{surf}} = \int_{\partial B} \underline{\underline{t}} dA$

qvals

Note:  $\underline{\underline{t}} = \lim_{\Delta A \rightarrow 0} \frac{\Delta f^{\text{surf}}}{\Delta A}$

it is a fundamental assumption of continuum mechanics that this limit exists.

Thus, we have:

$$\underline{F}^{\text{ext}}(B) = \int_B \underline{\underline{\rho}} \underline{\underline{f}} dV + \int_{\partial B} \underline{\underline{\epsilon}} \underline{\underline{f}} dA$$

Substituting into (1) (from  $a/2z$ )

$$\boxed{\frac{D}{Dt} \int_B \underline{\underline{\rho}} \underline{\underline{x}} dV = \int_B \underline{\underline{\rho}} \underline{\underline{f}} dV + \int_{\partial B} \underline{\underline{\epsilon}} \underline{\underline{f}} dA} \quad (a)$$

↑ over volume      ↑ over surface

Note:

$$\frac{d}{dt} \int_a^b f(x,t) dx = \int_a^b \frac{d}{dt} f(x,t) dx$$

but can't do this if  $a, b$  functions of time.

i.e.:  $\frac{d}{dt} \int_{a(t)}^{b(t)} f(x,t) dx$

\* Note: The integration domain in (2) changes w/ time.

↳ because the body is deforming.

\* Thus, the time derivative cannot

be just passed inside the integral.

To solve this issue, we make use of the conservation of mass principle:

$$\frac{D}{Dt} \int_B \rho \dot{x} dv = \frac{D}{Dt} \int_S \frac{\rho_0}{J} \dot{x} dv = \frac{D}{Dt} \int_{B_0} \frac{\rho_0}{J} \dot{x} J dv$$

(\*) cons. of mass

$$(*) \quad dv = J dV$$

Note:  $J = 0$  means  $dV = 0 dV$  (collapse the body)

↳ not possible, physically

$$= \frac{D}{Dt} \int_{B_0} \rho_0 \dot{x} dV = \int_{B_0} \frac{D}{Dt} (\rho_0 \dot{x}) dV$$

$$= \int_{B_0} \rho_0 \ddot{x} dV \xrightarrow{(*)} \int_{B_0} J \rho_0 \ddot{x} dV \xrightarrow{(\text{const.})} \int_B \rho \ddot{x} dv$$

which allows us to express the spatial form of the global balance of linear momentum as :

$$\int_B \rho \ddot{x}^i dV = \int_B \rho \ddot{v}^i dV + \int_{\partial B} \ddot{t}^i dA$$

(global expression)

boundary  
of  $B$ .

### Cauchy's stress principle.

- In order to obtain a local expression for the balance of linear momentum, it is necessary to write the previous expression for an arbitrary sub-body  $E \subset B$
- This is a problem for the traction term since  $\ddot{t}$  is defined in terms of forces acting in  $B$  across its outer surfaces.
- This issue was addressed by Cauchy's stress principle.

## Cauchy's Stress principle.

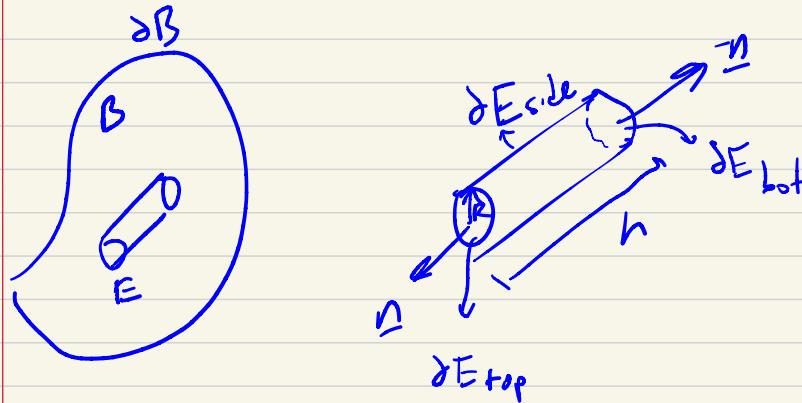
Material interactions across an internal surface in a body can be described as a distribution of tractions in the same way that the effect of external forces on physical surfaces of the body are described.

Thus, for an arbitrary subbody  $E \subset B$  we can write the following:

$$\int_E \rho \dot{\underline{\underline{\sigma}}} dv = \int_E \rho \dot{\underline{\underline{\sigma}}} dv + \int_{\partial E} \underline{\underline{\tau}} dA$$

$\left( \text{w/o free hat} \right)$

Let's consider an arbitrary  
"pillbox shaped" sub-body



$$\Rightarrow \int_E f (x - \frac{b}{2}) dV = \int_{\delta E_{side}} f dA + \int_{\delta E_{top}} f dA + \int_{\delta E_{bottom}} f dA$$

Taking limit for  $h \rightarrow 0$ :

$$\int_{\delta E_{top}} f dA + \int_{\delta E_{bottom}} f dA = 0$$

Finally, applying the mean value theorem and taking limit  $R \rightarrow \infty$ :  $\int_{\delta E_{top}} f(x) dA = - \int_{\delta E_{bottom}} f(x) dA$