



1-Consider the model of an ideal pendulum:

$$J\ddot{q} + mgl * \sin(q) = \tau$$

Assume that we apply the PD controller with compensation

$$\tau = k_p \tilde{q} + k_v \dot{\tilde{q}} + J[\ddot{q}_d + \lambda \dot{\tilde{q}}] + mgl * \sin(q)$$

Where $\lambda = k_p / k_v$ and k_p and k_v are positive numbers.

a) Obtain the closed-loop equation in terms of the state vector $[\tilde{q} \ \dot{\tilde{q}}]^T$. Verify that the origin is its unique equilibrium point.

b) Show that the origin $[\tilde{q} \ \dot{\tilde{q}}]^T = 0 \in R^2$ is globally asymptotically stable.

Hint : Use the Lyapunov function candidate:

$$V(\tilde{q}, \lambda \dot{\tilde{q}}) = \frac{1}{2} J (\dot{\tilde{q}} + \lambda \tilde{q})^2 + k_p \tilde{q}^2$$



2) Develop a controller for a one-dof mass-spring-damper system of the form :

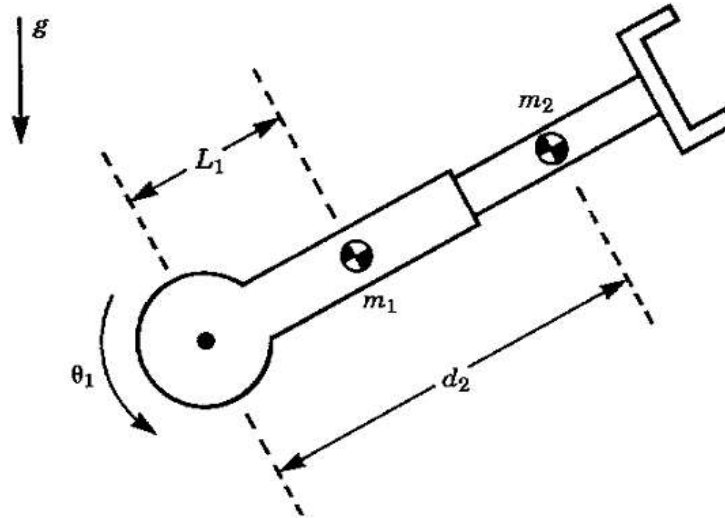
$$m\ddot{x} + b\dot{x} + kx = f$$

Where f is control force and $m = 4\text{ kg}$, $b = 2\text{ Ns/m}$, and $k = 0.1\text{ N/m}$.

- What is the damping ratio of the uncontrolled system? Is the uncontrolled system overdamped, underdamped, or critically damped? If it is underdamped, what is the damped natural frequency? What is the time constant of convergence to the origin?
- Choose a P controller $f = K_p x_e$, where $x_e = x_d - x$ is the position error and $x_d = 0$. What value of K_p yields critical damping?
- Choose a D controller $f = K_d \dot{x}_e$ where $\dot{x}_d = 0$. What value of K_d yields critical damping?
- Choose a PD controller that yields critical damping and a 2% settling time of 0.01 s.
- For the PD controller above if $x_d = 1$ and $\dot{x}_d = \ddot{x}_d = 0$ what is the steady state error $x_e(t)$ as t goes to infinity? What is the steady-state control force?
- Now insert a PID controller for f . Assume $x_d \neq 0$ and $\dot{x}_d = \ddot{x}_d = 0$. Write the error dynamics in terms of \ddot{x}_e , \dot{x}_e , x_e and $\int x_e(t) dt$ on the left-hand side and a constant forcing term on the right-hand side. (Hint: You can write kx as $-k(x_d - x) + kx_d$). Take the time derivative of this equation and give the conditions on K_p , K_i and K_d for stability. Show that zero steady-state error is possible with a PID controller.



3) consider the 2-link RP shown below:



Its equation of motion shown here:

$$\tau_1 = (m_1 L_1^2 + I_{zz1} + I_{zz2} + m_2 d_2^2) \ddot{\theta}_1 + 2m_2 d_2 \dot{\theta}_1 \dot{d}_2 + (m_1 L_1 + m_2 d_2) g \cos(\theta_1)$$
$$\tau_2 = m_2 \ddot{d}_2 - m_2 d_2 \dot{\theta}_1^2 + m_2 g \sin(\theta_1)$$

The manipulated parameter have the following numerical values: $L_1 = 0.2\text{m}$, $m_1 = 1\text{kg}$, $m_2 = 0.8\text{kg}$, $I_{zz1} = 0.1\text{kgm}^2$, $I_{zz2} = 0.07\text{kgm}^2$ and the range of d_2 is between 0.5m and 1.0m

- Using inverse dynamic method and design a controller for this robot .
- Find the value for the gains of PD compensator part such that the closed loop system for joint 1 is critically damped with natural frequency of 20 rad/sec , and the closed-loop system for joint 2 is critically damped with natural frequency of 25 rad/sec



4) An identification has been done to design a linear controller for a 3R robot. The following were resulted for the torque , angular velocity and acceleration of the third link

$$\ddot{\theta} = [1.3 \ 1.7 \ 1.9 \ 2.4 \ 2.8] \text{ (rad / s}^2\text{)}$$

$$\dot{\theta} = [.2 \ .5 \ .7 \ .9 \ 1.1] \text{ (rad / s)}$$

$$\tau = [5.08 \ 6.28 \ 6.92 \ 8.28 \ 9.4] \text{ (N.m)}$$

a) The dynamic model for this link is considered to be

$$\tau = J_e \ddot{\theta} + B_e \dot{\theta} + G_e$$

use the data from the identification test to determine the model parameters

- b) Draw a block diagram of the closed-loop system with G_e as a disturbance and design a PD controller such that the system has a steady state error to the disturbance G_e less than .1 and a damping ratio $\zeta = .707$
- c) How can we reduce the steady state error to zero?

Due Date:
May 29, 2020
(9 Khordad 99)

In the name of god

Advanced Robotics
Homework Assignment #6



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۵) ادامه تمرین حل شده در کلاس حل تمرین: اگر انعطاف (نرمی) مفاصل با یک فنر خطی به شکل

$$u = -k_p \tilde{\theta}_2 - k_v \dot{\tilde{\theta}}_2 \text{ مدل شود و کنترلر زیر استفاده شود}$$

(a) آیا سیستم حلقه بسته پایدار است؟

(b) آیا سیستم پایداری مجانبی است؟

استدلال کنید. توجه کنید که θ_2 زاویه موتور می باشد. و $\tilde{\theta}_2 = \theta_2 - \theta_d$ همچنین برای راحتی تمرین را در حالت اسکالر حل نمایید.

Good Luck!