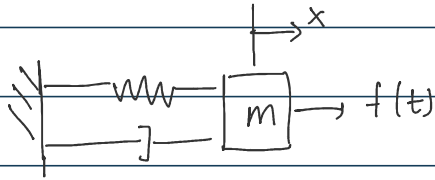


Vibrations

Monday, 8 April 2024 10:37 AM

⇒ Single degree of freedom vibrations



Equation of motion:

$$m\ddot{x} + c\dot{x} + kx = f(t) \quad \rightarrow 2^{\text{nd}} \text{ order linear ODE}$$

(I) Undamped free vibration:

$$m\ddot{x} + kx = 0$$

Ansatz: $x = ce^{st}$

$$ms^2 + k = 0 \Rightarrow s = \pm i \sqrt{\frac{k}{m}}$$

$$\therefore x(t) = c_1 \cos(\omega t) + c_2 \sin(\omega t) \quad \omega = \sqrt{\frac{k}{m}} : \text{Natural frequency}$$

↳ Simple harmonic motion.

c_1, c_2 can be obtained from the initial conditions $x(0), \dot{x}(0)$.

$$x(t) = x(0) \cos(\omega t) + \frac{\dot{x}(0)}{\omega} \sin(\omega t)$$

$$= X \cos(\omega t + \phi)$$

(II) Undamped vibrations under constant forcing

$$m\ddot{x} + kx = F$$

substitute $x - \frac{F}{k} = y$

$$m\ddot{y} + ky = 0$$

$$y = A \cos(\omega t) + B \sin(\omega t) \quad \omega_n = \sqrt{k/m}$$

$$x = \frac{F}{k} + A \cos(\omega_n t) + B \sin(\omega_n t)$$

i.e. Under a constant force, only the equilibrium position changes.

(III) Periodic forcing:

$$m\ddot{x} + kx = F_0 \cos(\omega t)$$

$$x_p = A \cos(\omega t) + B \sin(\omega t) \Rightarrow \text{particular solution.}$$

$$\dot{x}_p = -A\omega \sin(\omega t) + B\omega \cos(\omega t)$$

$$\ddot{x}_p = -A\omega^2 \cos(\omega t) - B\omega^2 \sin(\omega t)$$

$$\cos(\omega t) \cdot [-mA\omega^2 + kA] + \sin(\omega t) [-mB\omega^2 + kB] = F_0 \cos(\omega t)$$

$$\therefore B = 0 \quad A = \frac{F_0}{k - m\omega^2}$$

$$x_p = \frac{F_0}{k - m\omega^2} \cos(\omega t) = \frac{F_0/m}{\omega_n^2 - \omega^2} \cos(\omega t)$$

$$x(t) = \frac{F_0/k}{1 - (\omega/\omega_n)^2} \cos(\omega t) + A \cos(\omega_n t) + B \sin(\omega_n t)$$

If $\omega \rightarrow \omega_n$, amplitude blows up and we have resonance.

(IV) Damped free vibrations:

$$m\ddot{x} + c\dot{x} + kx = 0$$

$$\text{Ansatz: } x = c e^{st}$$

$$ms^2 + cs + k = 0$$

$$s = -\frac{c}{2m} \pm \sqrt{\left(\frac{c}{2m}\right)^2 - \frac{k}{m}}$$

$$\text{When } \left(\frac{c}{2m}\right)^2 - \frac{k}{m} = 0 \quad (\text{critically damped})$$

$$c_c = \sqrt{4mk}$$

$$\text{Define } \zeta = \frac{c}{c_c}$$

Define $\zeta = \frac{c}{\sqrt{4mk}}$

Rewriting equation of motion:

$$\ddot{x} + 2\zeta\omega_n \dot{x} + \omega_n^2 x = 0$$

$$s_{\pm} = -\zeta\omega_n \pm \omega_n \sqrt{\zeta^2 - 1}$$

Case 1: $\zeta < 1$. (Underdamped)

$$s = -\zeta\omega_n \pm i\omega_n \sqrt{1 - \zeta^2}$$

$$x(t) = e^{-\zeta\omega_n t} (A \cos(\omega_d t) + B \sin(\omega_d t))$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2} \Rightarrow \text{Damped Natural frequency}$$

Case 2: $\zeta > 1$. (Overdamped)

$$s = -\zeta\omega_n \pm \omega_n \sqrt{\zeta^2 - 1}$$

$$x(t) = e^{-\zeta\omega_n t} (A \cosh(\omega_n \sqrt{\zeta^2 - 1} t) + B \sinh(\omega_n \sqrt{\zeta^2 - 1} t))$$

↳ Exponential decay.

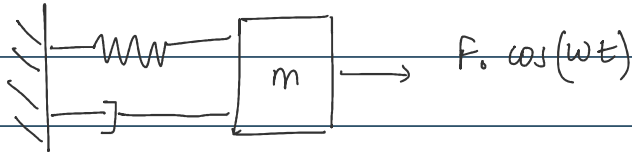
Case 3: $\zeta = 1$ (critically damped)

$$s = -\omega_n, -\omega_n$$

$$\therefore x(t) = (A + Bt) e^{-\zeta \omega_n t}$$

↳ Exponential decay

(v) Forced damped vibrations



$$m\ddot{x} + c\dot{x} + kx = F_0 \cos(\omega t)$$

Particular solution: $x_p = A \cos(\omega t) + B \sin(\omega t)$

$$-m\omega^2 (A \cos(\omega t) + B \sin(\omega t)) + c\omega (-A \sin(\omega t) + B \cos(\omega t))$$

$$+ k (A \cos(\omega t) + B \sin(\omega t)) = F_0 \cos(\omega t)$$

$$-m\omega^2 B - A\omega c + kB = 0$$

$$B(k - m\omega^2) = A\omega c \quad B = \frac{\omega c}{k - m\omega^2} A$$

$$-m\omega^2 A + B\omega c + kA = F_0$$

$$A \left(k - m\omega^2 + \frac{\omega c \cdot \omega c}{k - m\omega^2} \right) = F_0$$

$$A = \frac{F_0 (k - m\omega^2)}{(k - m\omega^2)^2 + (c\omega)^2}$$

$$x(t) = x_h(t) + x_p(t) \quad (s < 1)$$

$$= e^{-\xi \omega_n t} \cdot (A \cos(\omega_d t) + B \sin(\omega_d t)) +$$

$$\frac{F_0 (k - m\omega^2)}{(k - m\omega^2)^2 + (c\omega)^2} \cdot \left[\cos(\omega t) + \frac{(c\omega)}{(k - m\omega^2)} \sin(\omega t) \right]$$

$$x_p(t) = X_p \cos(\omega t - \psi)$$

$$\psi = \tan^{-1} \left(\frac{2\xi \omega_n}{1 - r^2} \right)$$

$$r = \omega / \omega_n$$

$$X_p = \frac{F_0/k}{[(1-r^2)^2 + (2\xi r)^2]^{1/2}}$$

$$\xi = c/c_c$$

$$\frac{X_p}{(F_0/k)} = \frac{1}{[(1-r^2)^2 + (2\xi r)^2]^{1/2}} = A$$

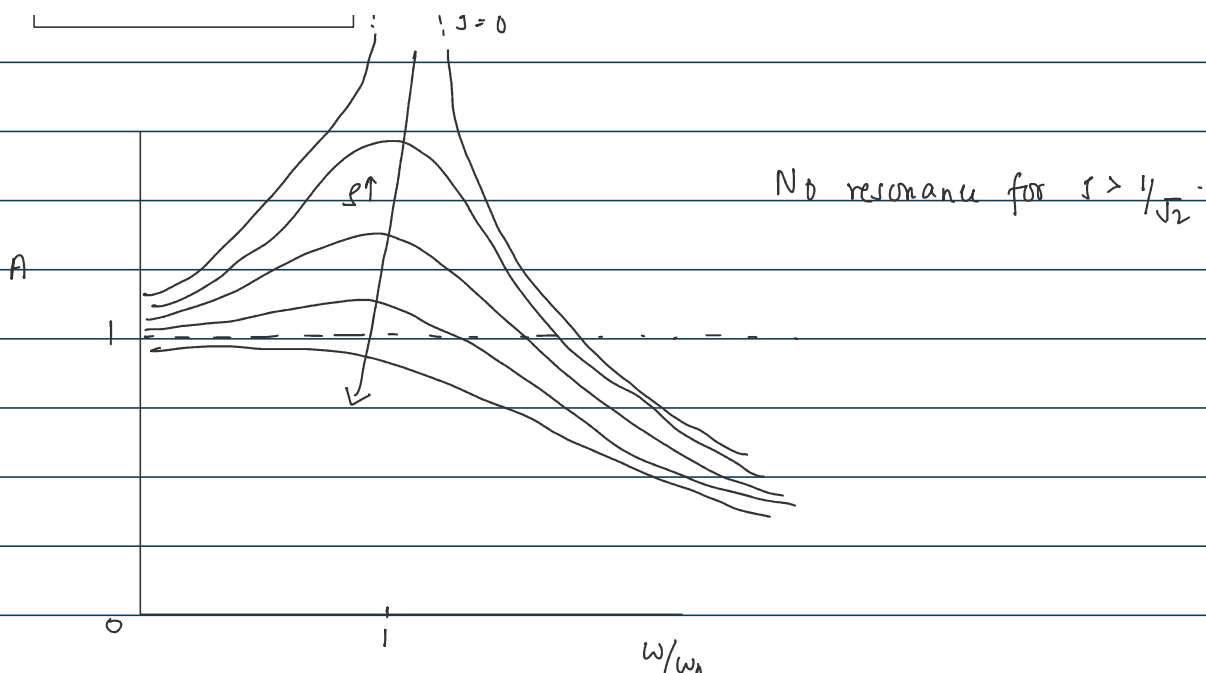
homogeneous part of solution decays with time.

At $t = \infty$, system oscillates with forcing frequency.

$$\text{Resonance: } \frac{d}{d\omega} \left(\left(\frac{1 - \omega^2}{\omega_n^2} \right)^2 + \left(\frac{2\xi \omega}{\omega_n} \right)^2 \right) = 0$$

$$\boxed{\omega_r = \omega_n \sqrt{1 - 2\xi^2}}$$

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s=0



(vi) Non-periodic forcing.

eg. Impulse. $F(t) = I_m \delta(t)$

$$m\ddot{x} + c\dot{x} + kx = I_m \delta(t)$$

Impulse gives initial condition. $v(0^+) = \dot{x}(0^+) = \frac{I_m}{m}$

$$x(t) = \frac{I_m}{m\omega_d} e^{-\zeta\omega_n t} \sin(\omega_d t)$$

For a general non-periodic forcing, response can be found by the convolution integral of the function with the impulse response.

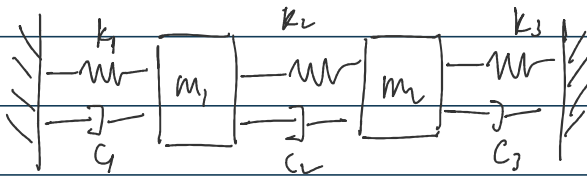
$$x(t) = \int_0^t g(t, \tau) F(\tau) d\tau$$

where $G(t, \tau)$ is the greens function solution to the ODE.

for the second order system,

$$G(t, \tau) = \frac{e^{-\zeta \omega_n (t-\tau)}}{m \omega_d} \sin(\omega_d (t-\tau))$$

\Rightarrow Two Degree of Freedom Systems:



EOM:

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{pmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{pmatrix} + \begin{bmatrix} c_1 + c_2 & -c_2 \\ -c_2 & c_2 + c_3 \end{bmatrix} \begin{pmatrix} \dot{x}_1 \\ \dot{x}_2 \end{pmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 + k_3 \end{bmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} = \begin{pmatrix} F_1 \\ F_2 \end{pmatrix}$$

or

$$M\ddot{X} + C\dot{X} + KX = 0$$

Example: Undamped response.

Homogeneous Solution: $X = V e^{\lambda t}$

$$(M\ddot{d} + K) V e^{i\omega t} = 0$$

$$\therefore \det(M\ddot{d} + K) = 0$$

$$\begin{bmatrix} \ddot{d}_1 m_1 + k_1 H_{k_1} & -k_2 \\ -k_2 & \ddot{d}_2 m_2 + k_2 H_{k_2} \end{bmatrix} = 0$$

$$\ddot{d}_1^4 m_1 m_2 + \ddot{d}_1^2 \left[\frac{k_2 H_{k_2}}{m_2} + \frac{k_1 H_{k_1}}{m_1} \right] + \frac{(k_1 H_{k_1})(k_2 H_{k_2}) - k_2^2}{m_1 m_2} = 0$$

$$\ddot{d}^4 + \ddot{d}^2 \left[\frac{k_2 H_{k_2}}{m_2} + \frac{k_1 H_{k_1}}{m_1} \right] + \frac{(k_1 H_{k_1} + k_2 H_{k_2})}{m_1 m_2} = 0$$

$$\ddot{d}^2 = \frac{- \left(\frac{k_2 H_{k_2}}{m_2} + \frac{k_1 H_{k_1}}{m_1} \right) \pm \sqrt{\left(\frac{k_2 H_{k_2}}{m_2} + \frac{k_1 H_{k_1}}{m_1} \right)^2 - 4 \frac{(k_1 H_{k_1} + k_2 H_{k_2})}{m_1 m_2}}}{2}$$

For the trivial case $(k_1 = k_2 = k), (m_1 = m_2)$

$$\ddot{d}^2 = \frac{- \frac{4k}{m} \pm \sqrt{\frac{16k^2}{m^2} - \frac{12k^2}{m^2}}}{2}$$

$$= - \frac{2k}{m} \pm \frac{k}{m}$$

$$\ddot{d}_1 = \pm i \sqrt{\frac{3k}{m}} \quad \ddot{d}_2 = \pm i \sqrt{\frac{k}{m}}$$

$$X(t) = V_1 \left(A \cos(\omega_1 t) + B \sin(\omega_1 t) \right) + V_2 \left(C \cos(\omega_2 t) + D \sin(\omega_2 t) \right)$$

$$X(t) = V_1 \left(A \cos(\omega_1 t) + B \sin(\omega_1 t) \right) + V_2 \left(C \cos(\omega_2 t) + D \sin(\omega_2 t) \right)$$

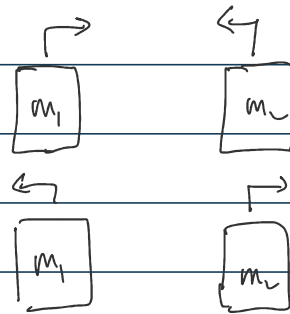
V_1, V_2 : found by solving eigenvalue equation with ω_1, ω_2 .

A, B, C, D found from initial conditions.

Modes (shapes) :

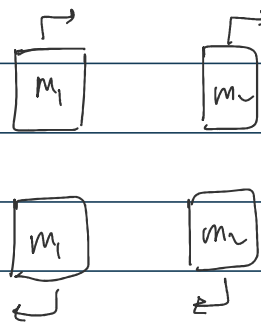
$$\omega = \omega_1 \Rightarrow \begin{bmatrix} -k & -k \\ -k & -k \end{bmatrix} \begin{bmatrix} V_1 \\ V_2 \end{bmatrix} = 0$$

$$V_1 = 1 \quad V_2 = -1 \Rightarrow$$



$$\omega = \omega_2 \Rightarrow \begin{bmatrix} k & -k \\ -k & k \end{bmatrix} \begin{bmatrix} V_1 \\ V_2 \end{bmatrix} = 0$$

$$V_1 = 1 \quad V_2 = 1$$



Forced undamped vibrations:

$$\text{Let } F = \begin{pmatrix} F_0 \cos(\omega t) \\ 0 \end{pmatrix}$$

$$M \ddot{x} + kx = F$$

$$M\ddot{x} + Kx = F$$

$$\text{Let } x_p = \begin{pmatrix} A \\ B \end{pmatrix} \omega(\alpha t) \quad \left(\text{No } \sin(\alpha t) \text{ because no first derivative on RHS} \right)$$

$$\therefore \begin{bmatrix} -m\alpha^2 + 2k & -k \\ -k & -m\alpha^2 + 2k \end{bmatrix} \begin{pmatrix} A \\ B \end{pmatrix} = \begin{pmatrix} F_0 \\ 0 \end{pmatrix}$$

$$\begin{pmatrix} -m\alpha^2 + 2k & -k \\ -k & -m\alpha^2 + 2k \end{pmatrix} \begin{pmatrix} A \\ B \end{pmatrix} = \begin{pmatrix} F_0 \\ 0 \end{pmatrix}$$

$$(-m\alpha^2 + 2k)A - kB = F_0$$

$$-kA + (-m\alpha^2 + 2k)B = 0$$

$$B = \frac{Ak}{2k - m\alpha^2}$$

$$(2k - m\alpha^2)A - \frac{Ak}{2k - m\alpha^2} = F_0$$

$$A \left(\frac{(2k - m\alpha^2)^2 - k}{2k - m\alpha^2} \right) = F_0$$

$$A = \frac{F_0 (2k - m\alpha^2)}{3k^2 - 4mk\alpha^2 + (m\alpha^2)^2} = \frac{F_0/3k \left(2 - \frac{m}{k}\alpha^2 \right)}{\left(1 - \frac{4m}{3k}\alpha^2 + \frac{m}{3k}\alpha^2 \frac{m\alpha^2}{k} \right)}$$

$$= \frac{F_0/3k}{\left(2 - \left(\frac{\alpha}{\omega_1} \right)^2 \right)}$$

$$= \frac{F_0/3k \left(2 - \left(\frac{\alpha}{\omega_1} \right)^2 \right)}{1 - \left(\frac{\alpha^2}{\omega_1^2} + \frac{\alpha^2}{\omega_2^2} \right) + \frac{\alpha^2}{\omega_1^2} \frac{\alpha^2}{\omega_2^2}}$$

$$A = \frac{F_0}{3k} \frac{\left(2 - \left(\frac{\alpha}{\omega_1} \right)^2 \right)}{\left(1 - \left(\frac{\alpha}{\omega_1} \right)^2 \right) \left(1 - \left(\frac{\alpha}{\omega_2} \right)^2 \right)}$$

$$\omega_1 = \sqrt{\frac{k}{m}}$$

$$\omega_2 = \sqrt{\frac{3k}{m}}$$

$$B = \frac{F_0/3k}{\left(1 - \left(\frac{\alpha}{\omega_1} \right)^2 \right) \left(1 - \left(\frac{\alpha}{\omega_2} \right)^2 \right)}$$

Vibration isolation - Make $A = 0$. (Choose m, k such that $\alpha = \sqrt{2}\omega$)