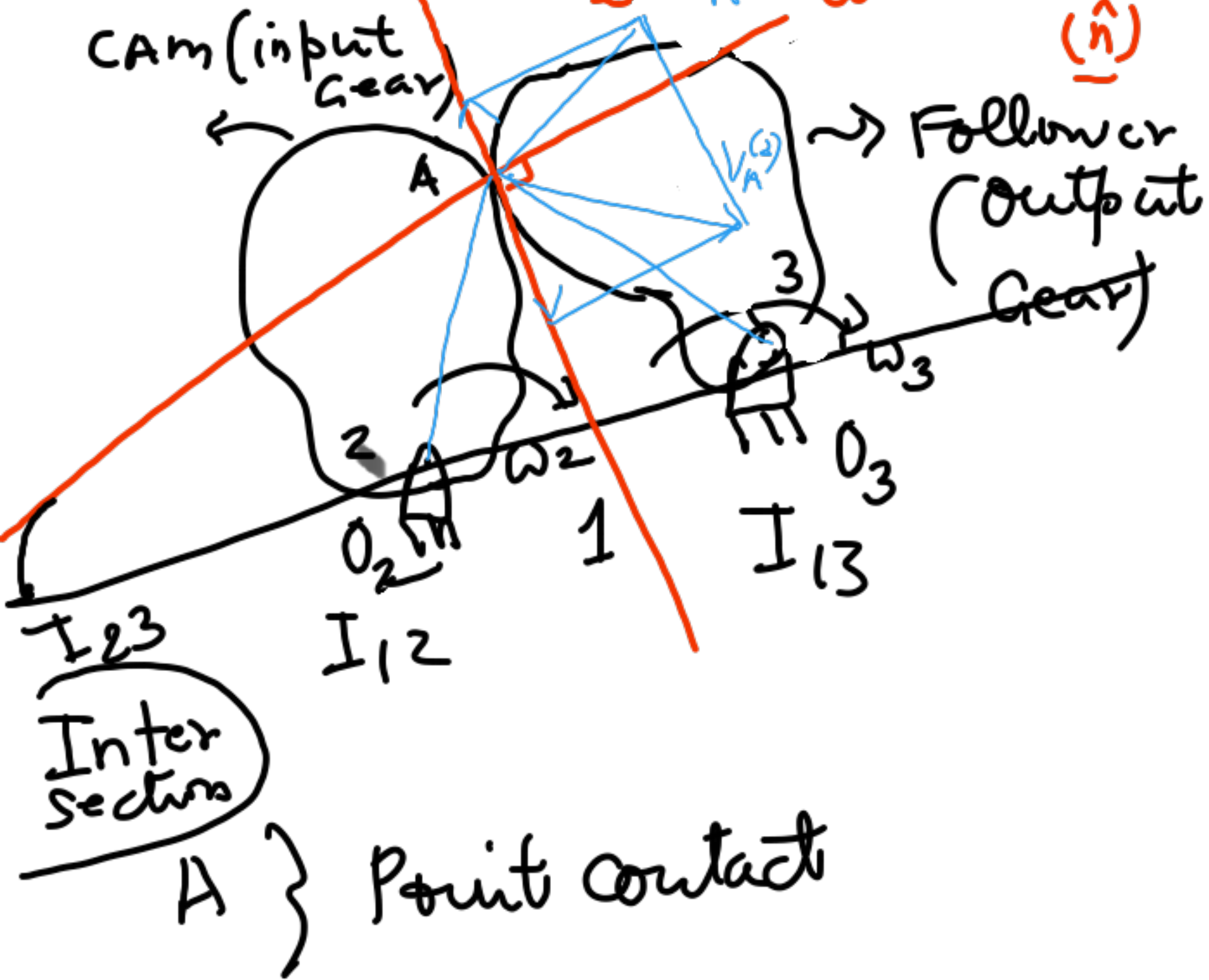


Mechanisms with higher pairs

① CAM and follower

② Gears. (\hat{t}) $\hat{v}_A^{(3)}$
Common tangent
Common normal (\hat{n})



Degree of freedom

$$F = 3(n-1) - 2j - h$$

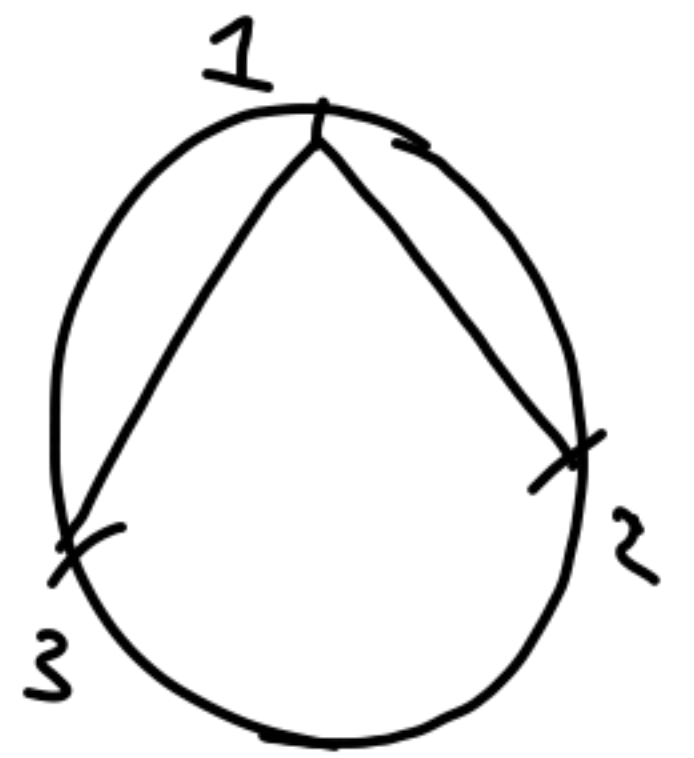
$$= 3(3-1) - 2 \times 2 - 1$$

$$= 6 - 4 - 1$$

$$F = 1$$

At point A, sliding occurs between bodies 2 and 3.

Calculation of instantaneous Centre of velocities (ICV's)



As per Aronhold Kennedy theorem,

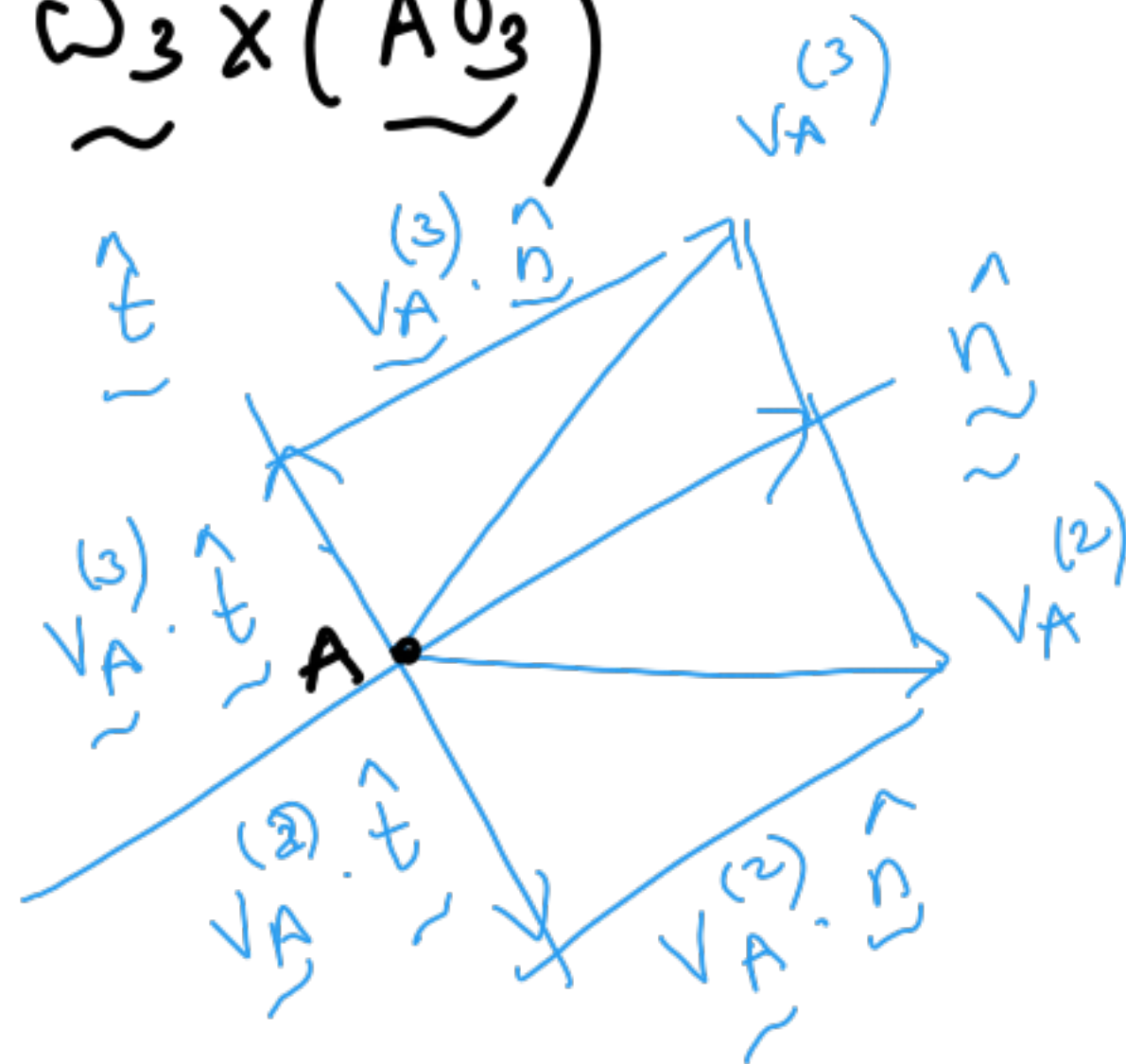
I_{23} has to lie on line joining O_2 and O_3 .

Velocity of pt. A :

$$\underline{\tilde{V}}_A^{(2)} = \underline{\tilde{\omega}}_2 \times (\underline{AO}_2)$$

↓
(Velocity
w.r.t
Body 2)

$$\underline{\tilde{V}}_A^{(3)} = \underline{\tilde{\omega}}_3 \times (\underline{AO}_3)$$



$$\underline{\tilde{V}}_A^{(2)} \cdot \underline{\hat{n}} = \underline{\tilde{V}}_A^{(3)} \cdot \underline{\hat{n}} \quad (*)$$

Necessary
to maintain
contact and
avoid

- (a) gap
 - (b) overlap
- between bodies
② and ③.

(*) can be used
to deduce relationship
between ω_2 and ω_3

$$\underline{\tilde{V}}_A^{(2)} - \underline{\tilde{V}}_A^{(3)}$$

$$= \left[\left(\underline{\tilde{V}}_A^{(2)} - \underline{\tilde{V}}_A^{(3)} \right) \cdot \underline{\hat{t}} \right] \underline{\hat{t}}$$

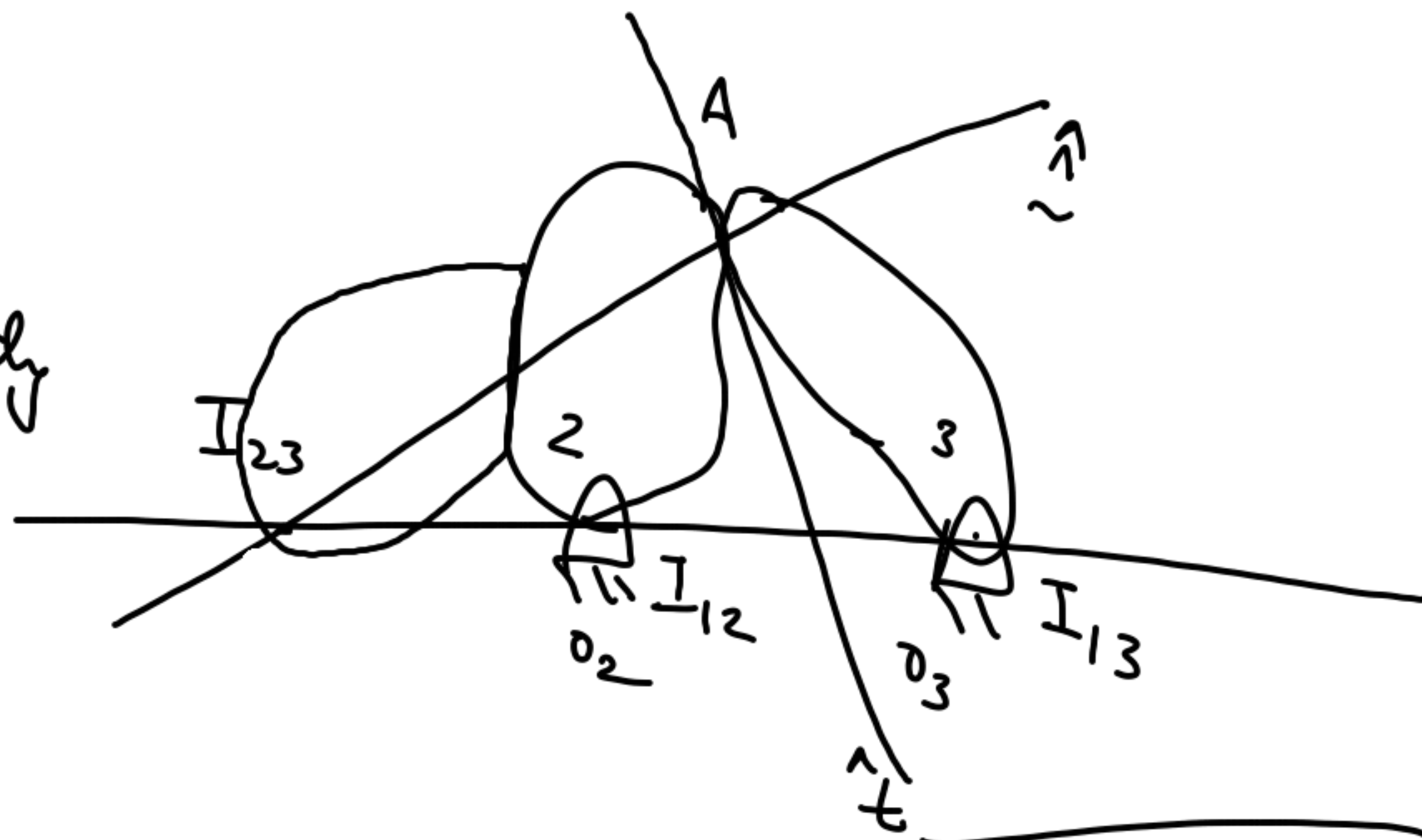
So the relative
velocity between
bodies 2 and
3 at point A
is along the
Common tangent.

W.r.t I₂₃, point
A as part of ?

will rotate.

So relative velocity
should be
perpendicular
to $(I_{23}A)$

So this implies that
 $I_{23}A$ should
coincide with
the common
normal.



Velocity ratio ; $\frac{\omega_3}{\omega_2} = \frac{(I_{23}O_2)}{(I_{23}O_3)}$

$V_{I_{23}}^{(2)} = \omega_2 (I_{23}O_2)$
 $V_{I_{23}}^{(3)} = \omega_3 (I_{23}O_3)$

For CAMs, location of I_{23} and
consequently, the velocity ratio
will keep on changing.

For gears,

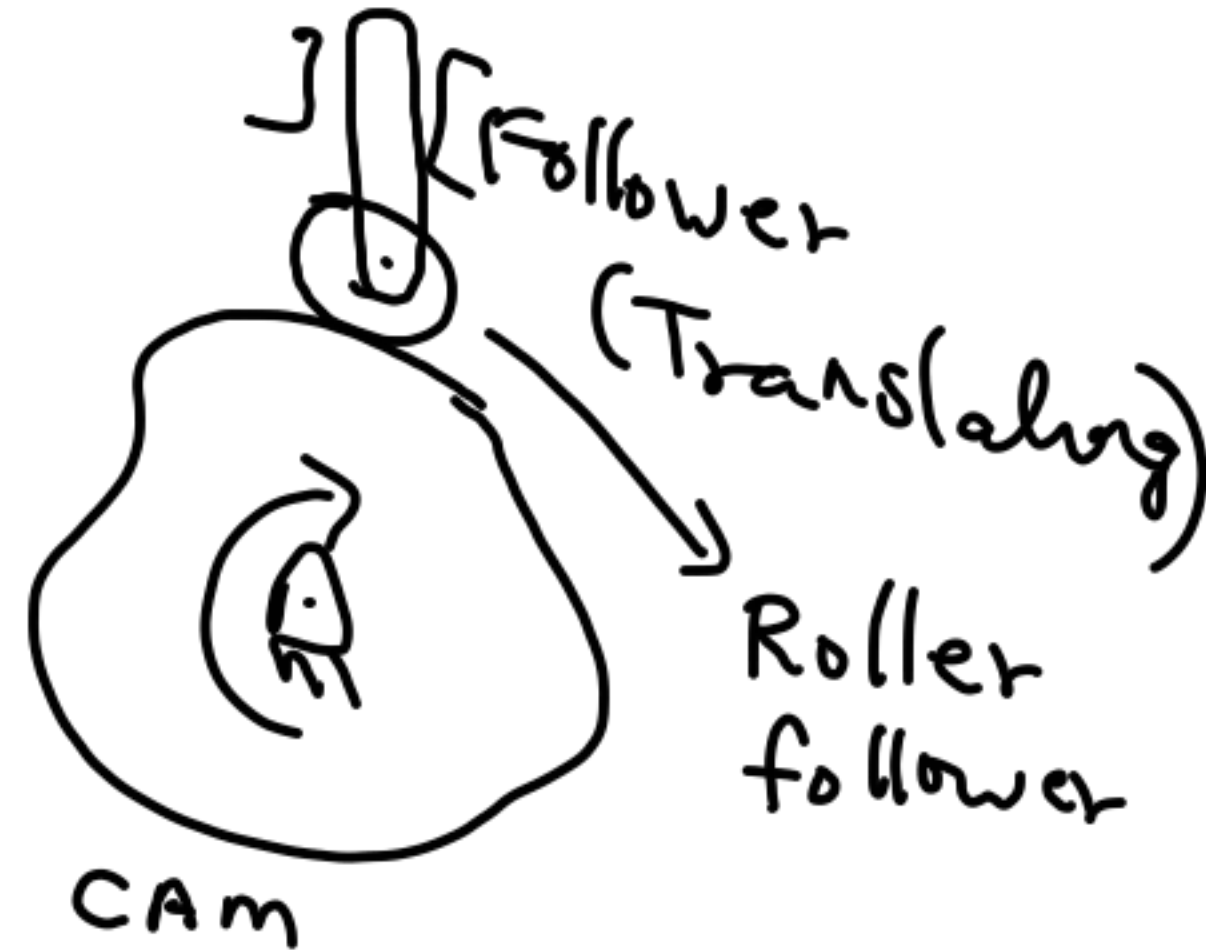
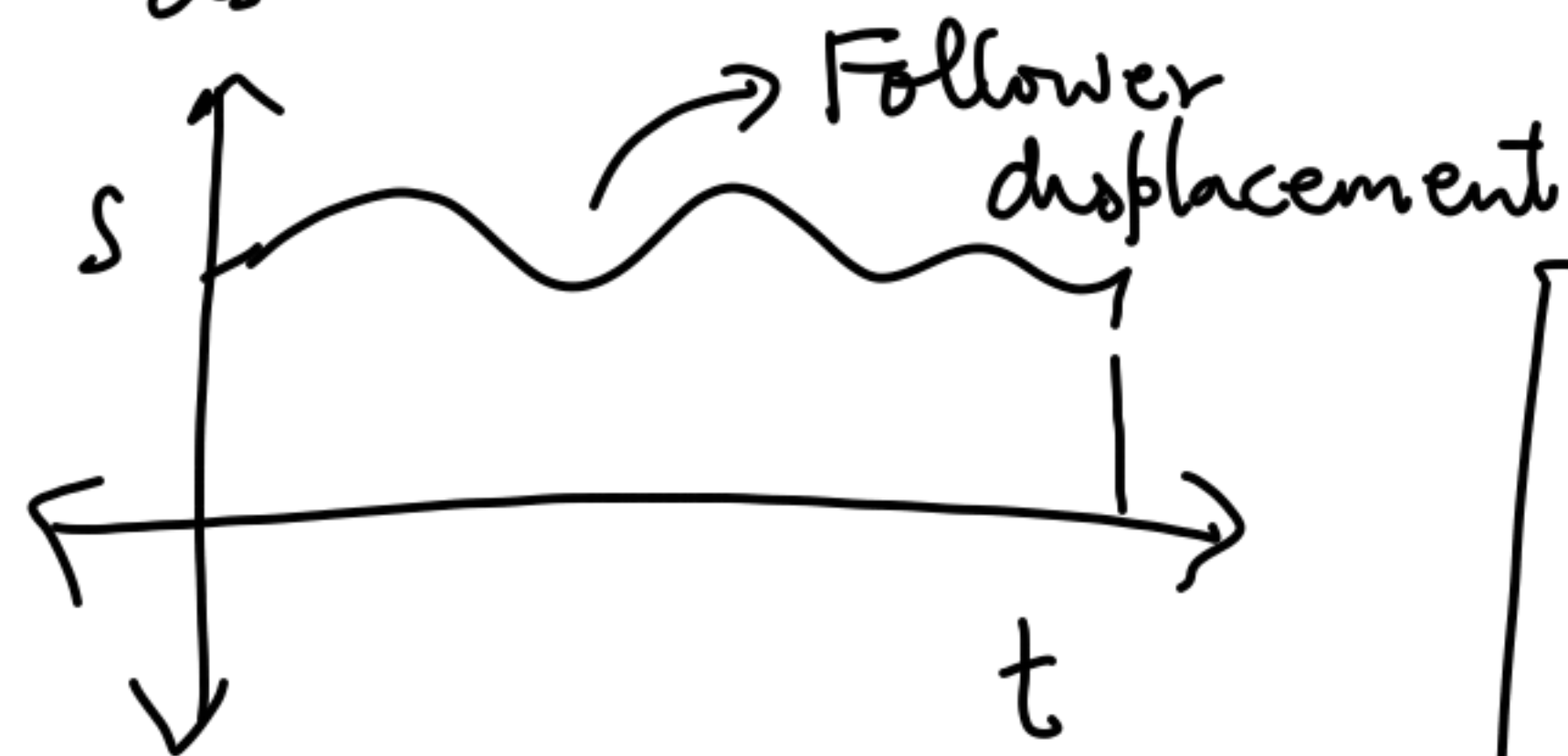
the surface
profile is
chosen such
that I_{23} is
also fixed.

So we are able
to achieve constant
speed ratio.

CAM and follower

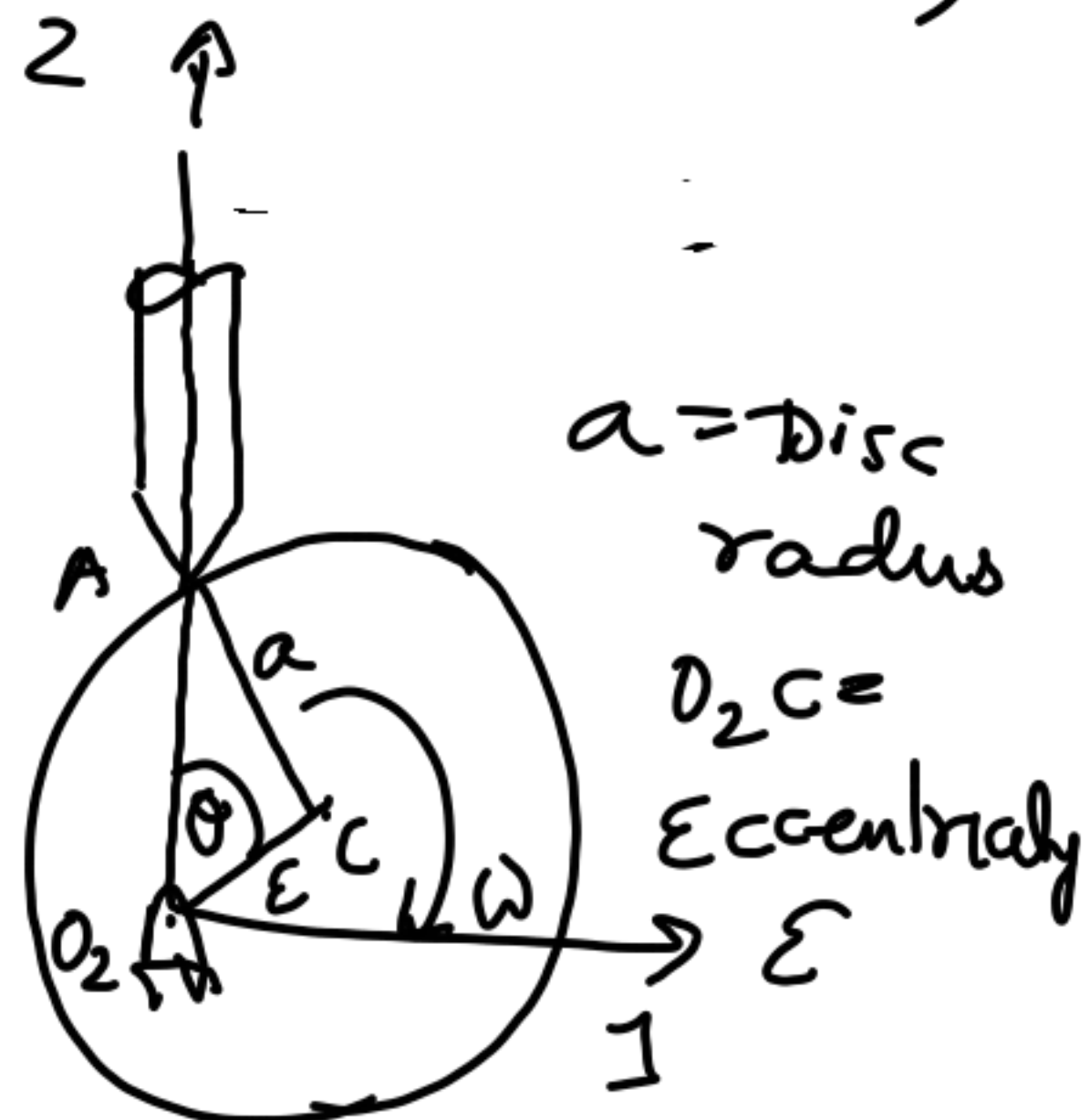
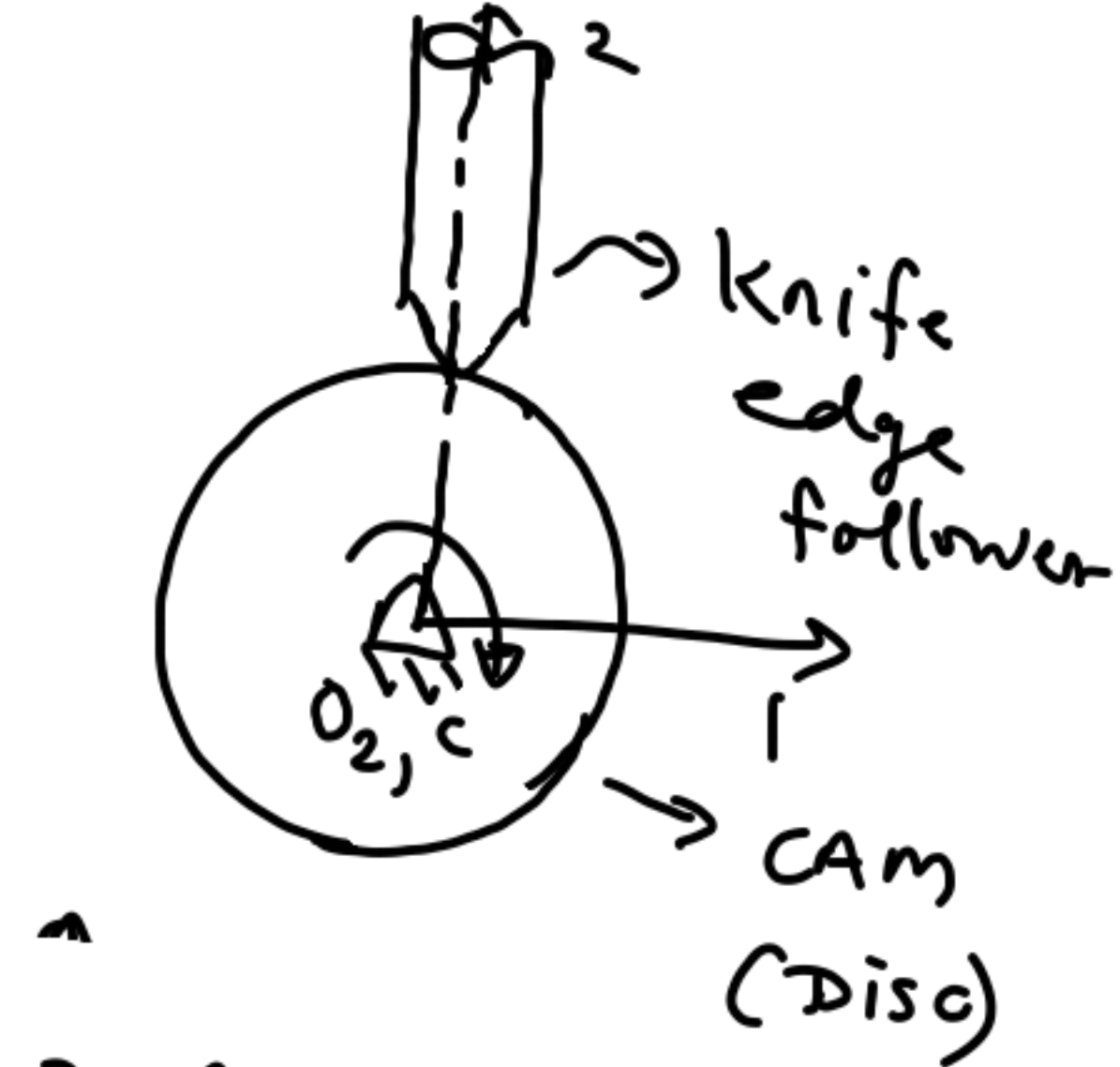
CAM is the input link, rotating at constant speed.

Follower usually follows the displacement as:



Follower is attached with a spring such that contact is not lost.

Circular disk and a knife edge follower:



Disc is rotating at constant speed ω

Follower
displacement

$$\in [a - \varepsilon, a + \varepsilon]$$

$$\cos \theta = \frac{(O_2 A)^2 + \varepsilon^2 - a^2}{2 (O_2 A) \varepsilon}$$

$$O_2 A = x_2$$

$$\therefore x_2^2 - 2(x_2 \varepsilon) \cos \theta + \varepsilon^2 - a^2 = 0$$

$$x_2 = \frac{2\varepsilon \cos \theta \pm \sqrt{4\varepsilon^2 \cos^2 \theta - 4(\varepsilon^2 - a^2)}}{2}$$

$$x_2 = \varepsilon \cos \theta \pm \sqrt{a^2 - \varepsilon^2 \sin^2 \theta}$$

For point C to lie
above O_2 at the start,
we choose "+" root

$$x_2 = \varepsilon \left[\cos \theta + \sqrt{\left(\frac{a}{\varepsilon}\right)^2 - \sin^2 \theta} \right]$$

$$\text{For } \underline{\theta = 0}: x_2 = (\varepsilon + a)$$

