

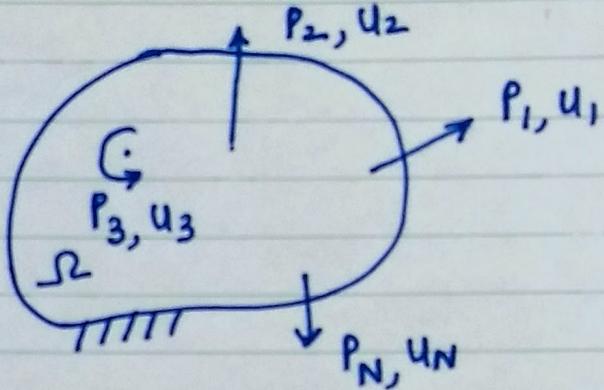
ME 202 Strength of Materials

CT2 Castigliano Theorem 2

03 April 2023

Castigliano's Theorem II CT2 (w/o derivation)

Let Ω be a linear elastic solid in static equilibrium

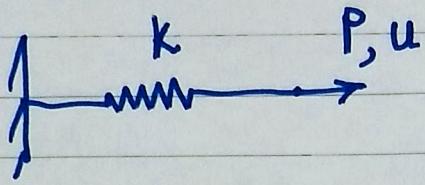


with applied forces/moment P_1, P_2, \dots, P_N
and displacements u_1, u_2, \dots, u_N
where in the direction of forces and at the point
of application of the respective forces.

Let $U(P_1, P_2, \dots, P_N)$ be the stored elastic energy/strain energy of the solid.

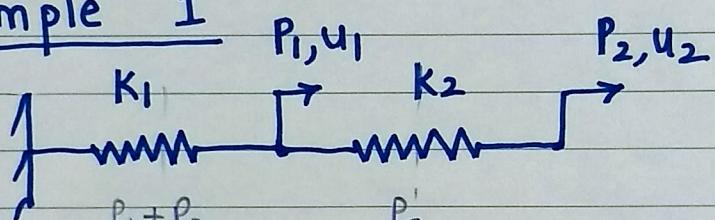
$$\text{Then } u_i = \frac{\partial U}{\partial P_i}$$

- P_i generalized force / moment
 u_i generalized displacement / angle i.e.
angular displacement
- exact solution. not an approximation.
- u_i in the direction of P_i and
at the point of application of P_i
- needs equilibrium / FBD analysis (see ahead)

FEW Example 0

$$U = \frac{1}{2} \frac{P^2}{k}$$

$$u = \frac{\partial U}{\partial P} = \frac{P}{k}$$

Example 1

$$U = \frac{1}{2} \frac{(P_1 + P_2)^2}{k_1} + \frac{1}{2} \frac{P_2^2}{k_2}$$

$$u_1 = \frac{\partial U}{\partial P_1} = \frac{P_1 + P_2}{k_1}, \quad u_2 = \frac{\partial U}{\partial P_2} = \frac{P_1 + P_2}{k_1} + \frac{P_2}{k_2}$$

compare with PMPE method,

$$\Pi = \frac{1}{2} k_1 (u_1)^2 + \frac{1}{2} k_2 (u_2 - u_1)^2 - P_1 u_1 - P_2 u_2$$

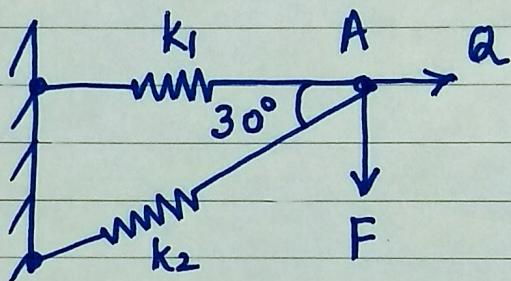
$$\begin{aligned} \frac{\partial \Pi}{\partial u_1} &= 0 \Rightarrow k_1 u_1 - k_2 (u_2 - u_1) = P_1 \\ \frac{\partial \Pi}{\partial u_2} &= 0 \Rightarrow k_2 (u_2 - u_1) = P_2 \end{aligned} \quad \left. \begin{aligned} &\left[\begin{array}{cc} k_1 + k_2 & -k_2 \\ -k_2 & k_2 \end{array} \right] \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} \\ &= \begin{bmatrix} P_1 \\ P_2 \end{bmatrix} \end{aligned} \right\}$$

same ans as above

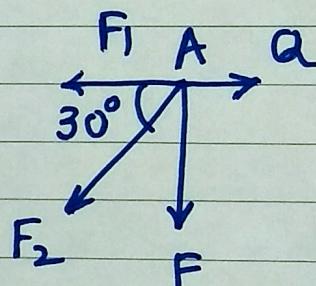
Note: we had to use equilibrium to get forces in each component. FBDS

If asked to calculate disp at a point where no force is applied or in a direction where $\underline{\underline{Q}}$, apply a dummy load $\underline{\underline{Q}}$ at the appr. point in the appr. direction, use CT2 Then set $\underline{\underline{Q}} = 0$.

Example 3



$\underline{\underline{Q}}$ not part of given problem



Get horiz. disp. of A.

F_1, F_2 tensile forces in springs
 k_1, k_2 resp'y.

$$\Rightarrow F_1 = Q + F\sqrt{3}, \quad F_2 = -2F$$

$$U(F, Q) = \frac{1}{2} \frac{F_1^2}{k_1} + \frac{1}{2} \frac{F_2^2}{k_2}$$

$$= \frac{1}{2} \frac{(Q + F\sqrt{3})^2}{k_1} + \frac{1}{2} \frac{(-2F)^2}{k_2}$$

$$\text{Horiz. disp } @ A = \frac{\partial U}{\partial Q} = \frac{Q + F\sqrt{3}}{k_1}$$

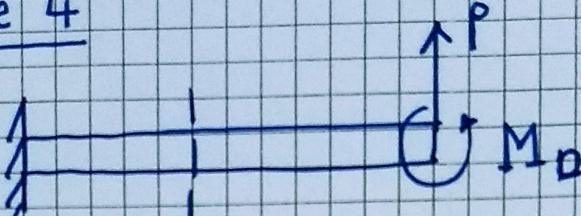
Now set $Q=0$,

$$\text{horiz. disp @ A} = \frac{\sqrt{3} F}{K_1}$$

$$\text{vert. disp @ A} = \frac{\partial U}{\partial F} = \frac{(Q+F\sqrt{3})}{K_1} \sqrt{3} + \frac{4F}{K_2}$$

$$= \left(\frac{3}{K_1} + \frac{4}{K_2} \right) F \quad \text{after setting } Q=0.$$

Same problem using PMPE for comparison.

Example 4

$$M(z) = M_D + P(L-z)$$

strain energy density = $\frac{1}{2} \sigma \epsilon$
(J/m³)

$$U = \int_{\Omega} \frac{1}{2} \sigma \epsilon dV = \int_0^L dz \int_A \frac{1}{2} \sigma \epsilon da$$

$$\sigma = -\frac{Mx}{I}, \quad \epsilon = -\frac{Mx}{EI}$$

$$U = \int_0^L dz \frac{M^2}{2EI^2} \int_A x^2 da$$

$$U = \int_0^L \frac{M^2 dz}{2EI} \quad \text{in general for bending}$$

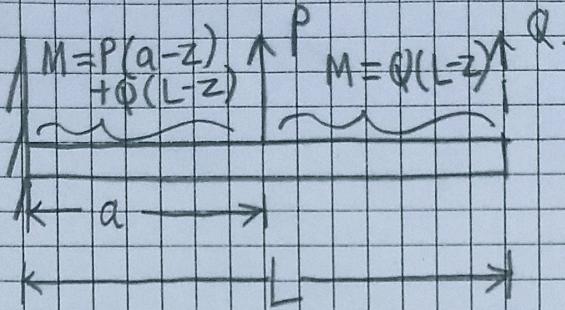
in this case

$$U = \int_0^L \frac{1}{2EI} (M_D + P(L-z))^2 dz$$

$$= \frac{(P^2 L^3 + 3L^2 M_D P + 3M_D^2 L)}{6EI}$$

$$\frac{\partial U}{\partial P} = \frac{PL^3}{3EI} + \frac{M_D L^2}{2EI} \quad \text{deflection at } P, \text{ along } P$$

$$\frac{\partial U}{\partial M_D} = \frac{PL^2}{2EI} + \frac{M_D L}{EI} \quad \text{deflection (angle) at } M_D, \text{ along } M_D$$

Castiglione ThmExample 5.1

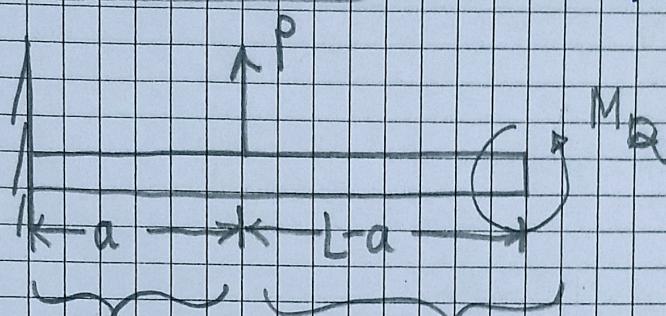
Cantilever. Load P applied at a , want deflection at L .

Apply dummy load Q at L .

$$\begin{aligned}
 U &= \int_0^L \frac{M^2}{2EI} dz \\
 &= \int_0^a \frac{(P(a-z) + Q(L-z))^2}{2EI} dz + \int_a^L \frac{(Q(L-z))^2}{2EI} dz \\
 &= \frac{(P+Q) \frac{a^3}{6EI}}{2EI} + \frac{a(Pa+LQ) - a^2(P+Q)}{2EI} - \frac{(Pa+LQ)}{2EI} \\
 \frac{\partial U}{\partial Q} &= \frac{a}{6EI} (6L^2Q - Pa^2 + 2Qa^2 + 3La^2 - 6LQa)
 \end{aligned}$$

$$S_L = \left. \frac{\partial U}{\partial Q} \right|_{Q=0} = \frac{Pa^2}{6EI} (3L - a)$$

want slope at L



$$M_1 = P(a-z) + M_Q \quad M_2 = M_Q.$$

$$\begin{aligned} U &= \int_0^a \frac{M_1^2}{2EI} dz + \int_a^L \frac{M_2^2}{2EI} dz \\ &= \frac{a}{6EI} (3M_Q^2 + 3M_Q Pa + Pa^2) + \frac{M_Q^2 (L-a)}{2EI} \end{aligned}$$

$$\frac{\partial U}{\partial M_Q} = \frac{Pa^2 + 2LM_Q}{2EI}$$

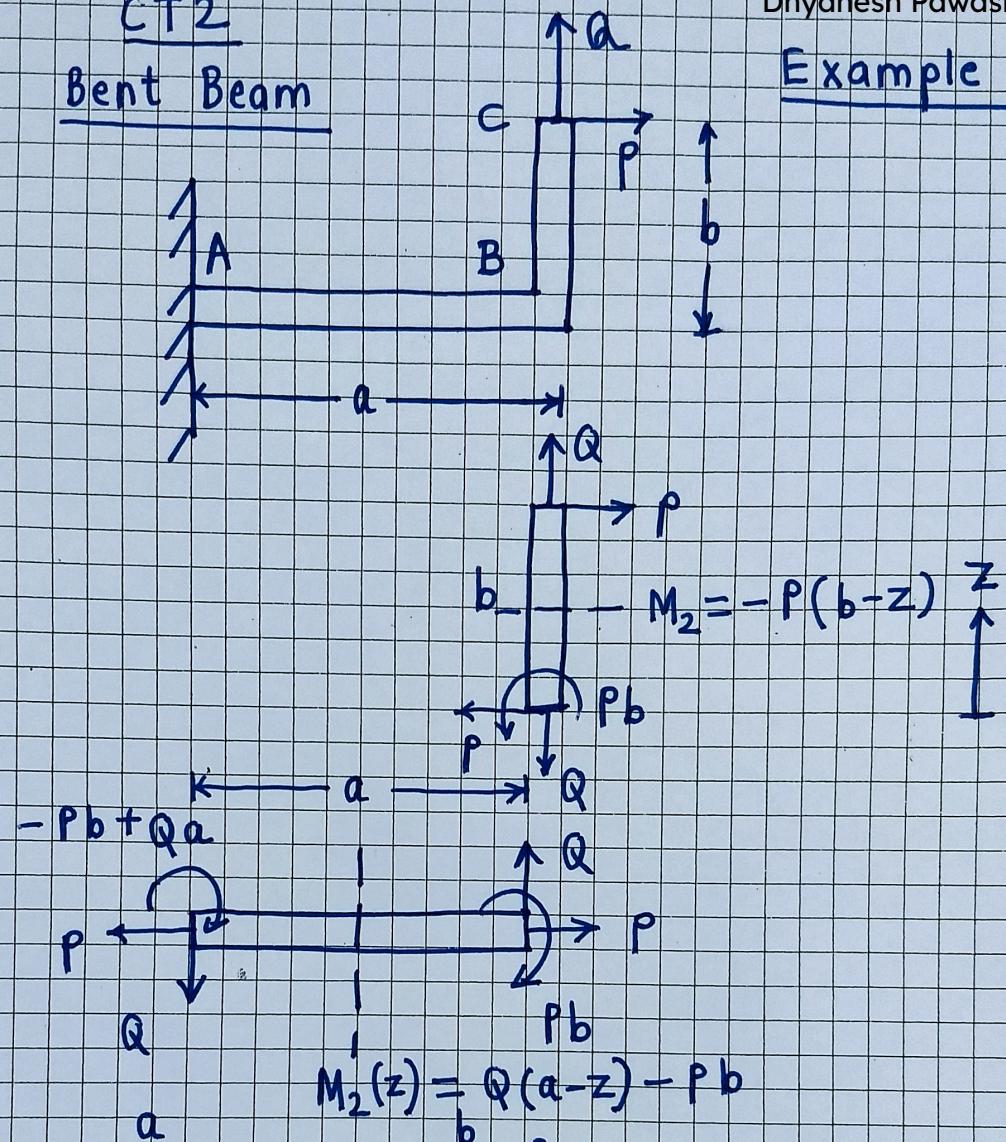
$$\theta_L = \left. \frac{\partial U}{\partial M_Q} \right|_{M_Q=0} = \frac{Pa^2}{2EI}$$

$$M_Q = 0$$

Example 5.2
virtual
dummy moment

CT 2
Bent Beam

Example 6



$$U = \int_0^a \frac{M_1^2}{2EI} dz + \int_0^b \frac{M_2^2}{2EI} dz$$

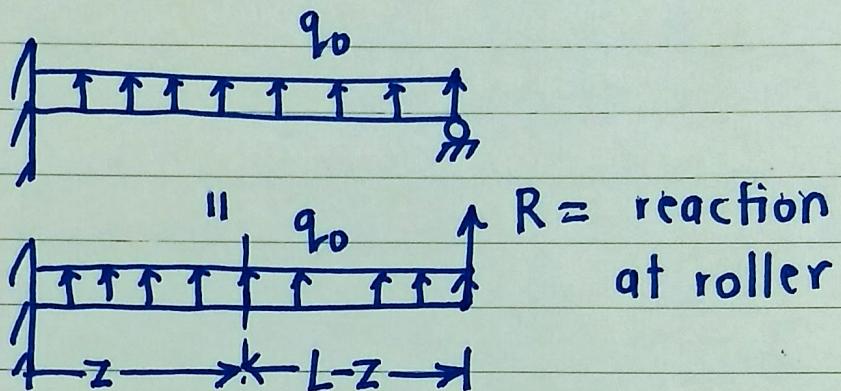
$$= \frac{P^2 ab^2}{2EI} - \frac{PQ a^2 b}{2EI} + \frac{Q a^3}{6EI} + \frac{P^2 b^3}{6EI}$$

$$\text{Disp in dir of } Q \text{ at } a = \frac{\partial U}{\partial Q} = \frac{Q a^3}{3EI} - \frac{Pba^2}{2EI}$$

$$\text{Disp in dir of } P \text{ at } P = \frac{\partial U}{\partial P} = \frac{Pb^3}{3EI} + \frac{Pab^2}{2EI} - \frac{Qa^2b}{2EI}$$

Example 47

Obtain reaction of a statically indeterminate beam.



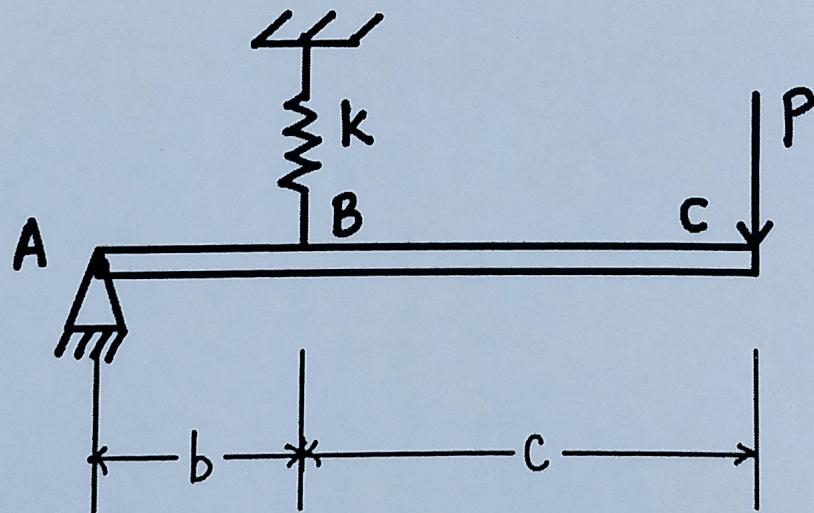
$$M(z) = R(L-z) + q_0(L-z) \frac{(L-z)}{2}$$

$$\begin{aligned} U &= \int_{0}^{L} \frac{1}{2EI} M^2(z) dz \\ &= \frac{L^3}{120EI} (3L^2q_0^2 + 15LRq_0 + 20R^2) \end{aligned}$$

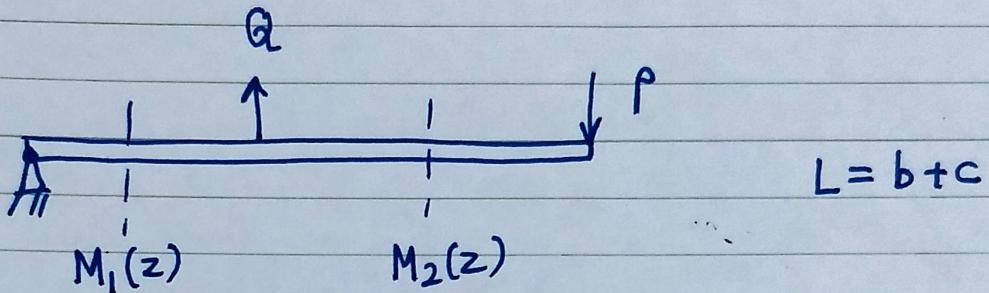
$$\frac{\partial U}{\partial R} = \frac{q_0 L^4}{8EI} + \frac{RL^3}{3EI} = 0 \quad \text{as roller at } z=L$$

$$\Rightarrow R = -\frac{3Lq_0}{8}$$

3. (8 points) A beam ABC with flexural rigidity EI is simply supported at A and held by a linear spring with stiffness k at B. A load P acts at the free end C. Find the deflection at C due to the applied load.



Solution by Castigliano Thm Example 8



$$M_1(z) = -P(L-z) + Q(L-z-c)$$

$$M_2(z) = -P(L-z)$$

$$Qb = P(b+c)$$

$$\begin{aligned} U(P) &= \int_{0}^b \frac{M_1^2}{2EI} dz + \int_{b}^{b+c} \frac{M_2^2}{2EI} dz + \frac{1}{2} \frac{Q^2}{K} \\ &= \frac{P^2 c^3}{6EI} + \frac{P^2 c^2 b}{6EI} + \frac{P^2 (b+c)^2}{2b^2 K} \end{aligned}$$

$$\delta_c = \frac{\partial U}{\partial P}$$

$$= \frac{Pc^3}{3EI} + \frac{Pbc^2}{3EI} + \frac{P(b+c)^2}{b^2 K}$$

$$= P(b+c) \left[\frac{c^2}{3EI} + \frac{b+c}{b^2 K} \right] \quad \checkmark$$