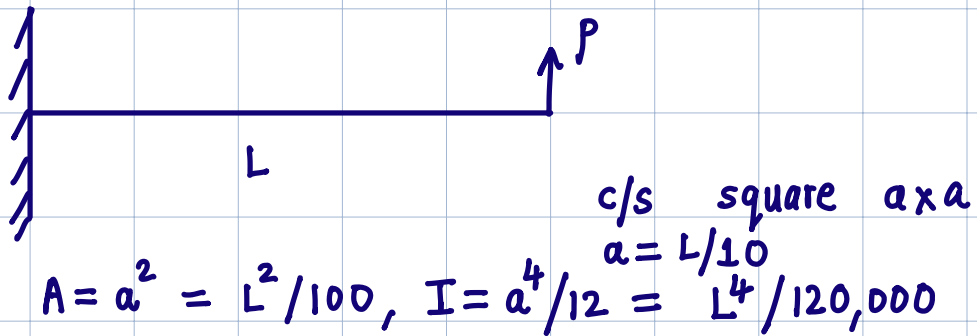


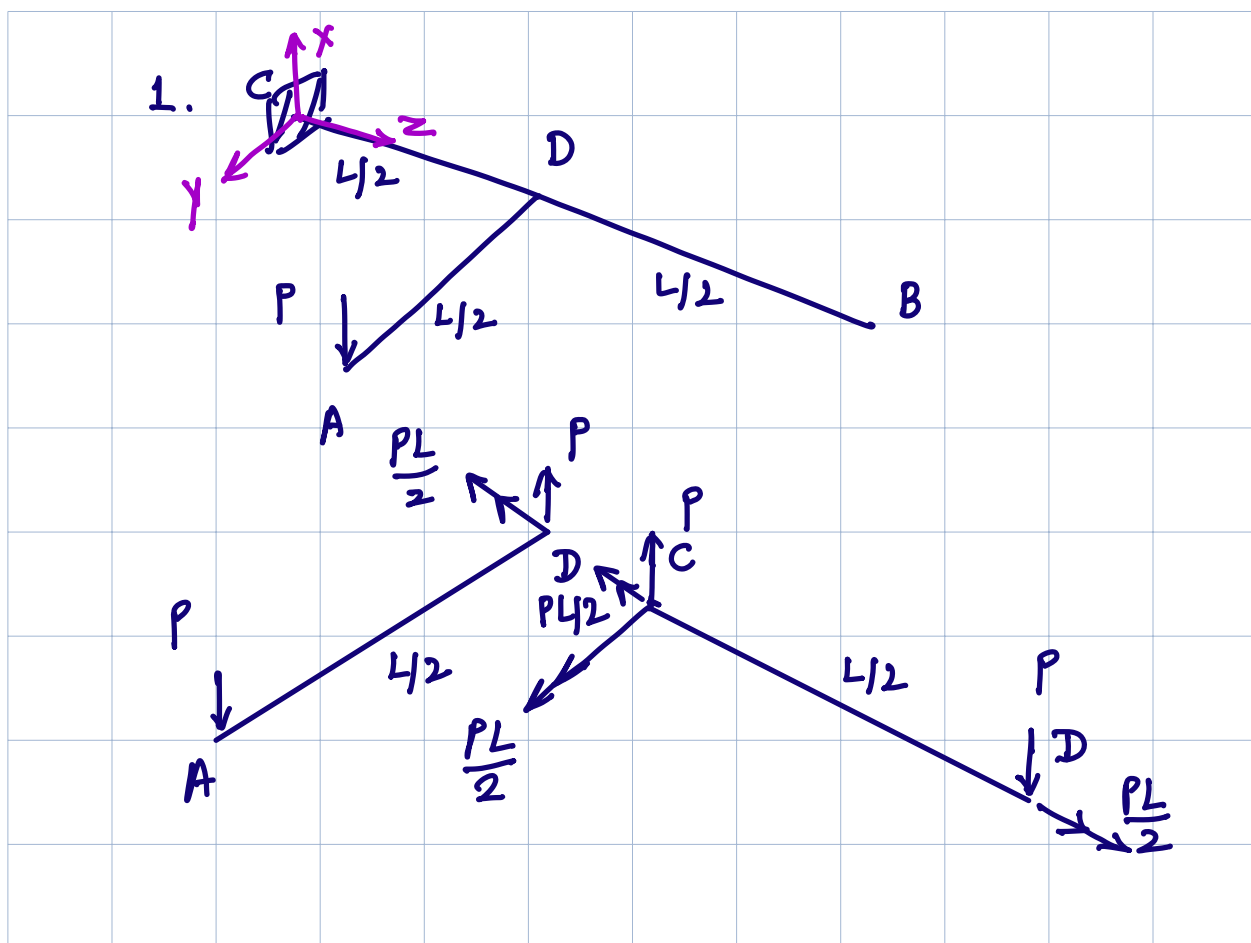
Axial vs Bending Deflections



$$A = a^2 = L^2/100, \quad I = a^4/12 = L^4/120,000$$

$$\delta_{\text{axial}} = \frac{PL}{AE} = 100 P/EL$$

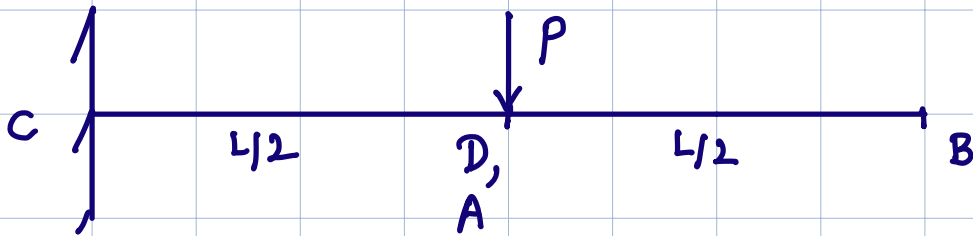
$$\delta_{\text{bending}} = \frac{PL^3}{3EI} = 40,000 P/EL \quad \begin{array}{l} \text{400 times} \\ \text{more} \end{array}$$



$$\delta_A = \frac{P(L/2)^3}{3EI} + \frac{P(L/2)^3}{3EI} + \underbrace{\frac{(PL/2)(L/2)(L/2)}{GJ}}_{\text{twist of CD}}$$

bending of AD bending of CD

$$= PL^3 \left(\frac{1}{12EI} + \frac{1}{8GJ} \right)$$



$$\delta_B = \frac{P}{EI} \left(\frac{a^2 z}{2} - \frac{a^3}{6} \right), \quad \begin{matrix} z=L \\ a=\frac{L}{2} \end{matrix} \quad \begin{matrix} \text{see} \\ \text{notes} \end{matrix}$$

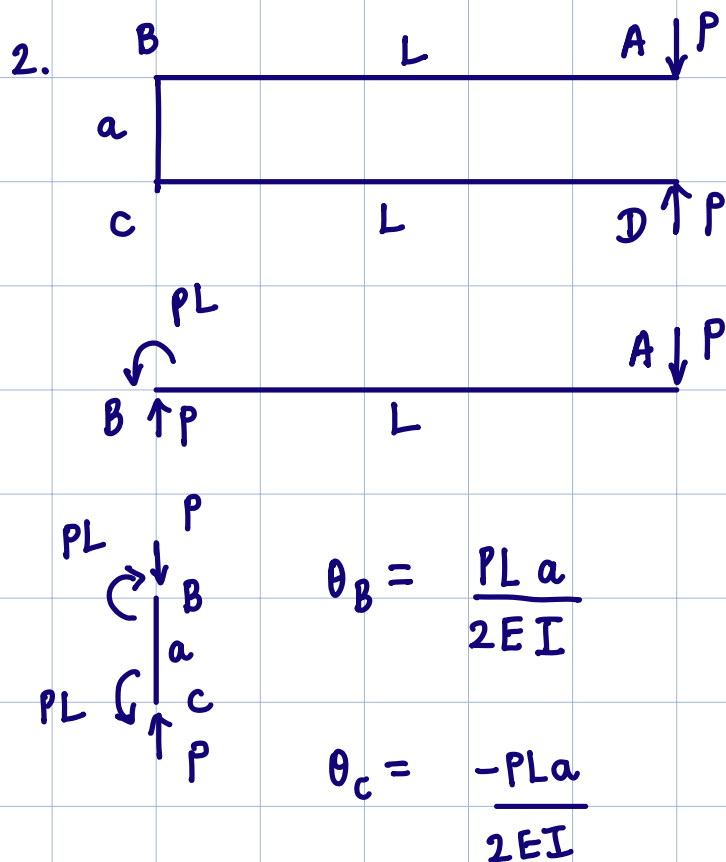
$$= \frac{P}{EI} L^3 \frac{5}{48}$$

$$\text{In CD, } T_{\max} = \frac{PL}{2}, \quad M_{\max} = \frac{PL}{2}$$

$$\sigma_{zz} = \frac{32M}{\pi D^3} = \frac{16PL}{\pi D^3}, \quad \sigma_{zx} = \frac{16T}{\pi D^3} = \frac{8PL}{\pi D^3}$$

$$\tau_{max} = \left(\left(\frac{\sigma_{zz} - \sigma_{xx}}{2} \right)^2 + \sigma_{zx}^2 \right)^{1/2}$$

$$= 8\sqrt{2} \frac{PL}{\pi D^3}$$



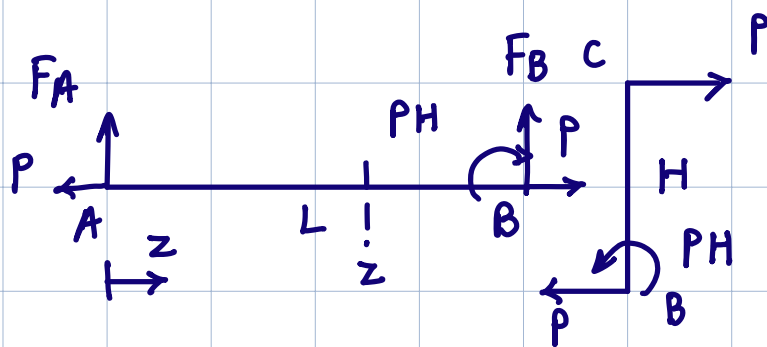
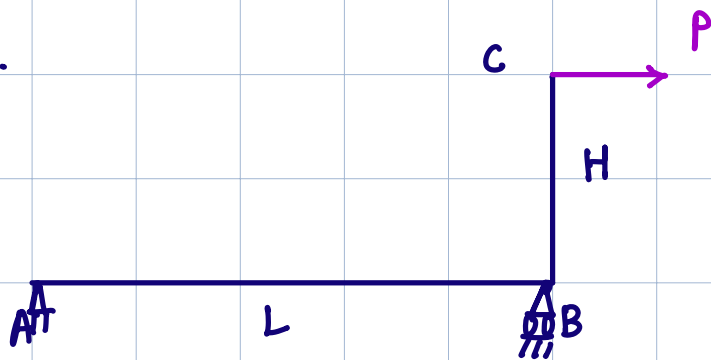
$$\delta_A = \underbrace{\frac{PL^3}{3EI}}_{\text{bending of AB}} + \underbrace{\theta_B L}_{\text{rotation @ B due to bending of BC}}$$

$$= \frac{PL^2}{6EI} (2L + 3a)$$

$$\Delta_D' = \Delta_D - 2\delta_A = a - \frac{PL^2}{3EI} (2L + 3a)$$

$$M_{\max} = PL$$

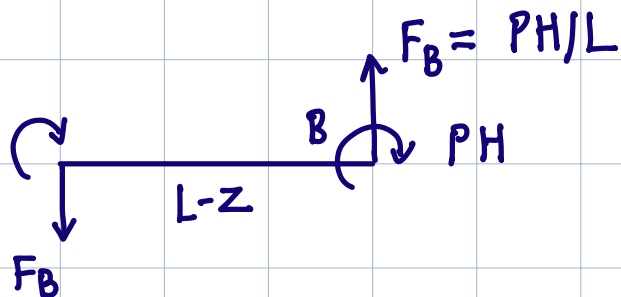
3.



$$F_A + F_B = 0$$

$$F_B L - PH = 0 \Rightarrow F_B = \frac{PH}{L}$$

$$M(z) = F_B(L-z) - PH$$



$$M(z) = -\frac{PHz}{L} = EI u''$$

check $M(0) = 0 \checkmark$

$$u'' = -\frac{PH}{EI} z$$

$$u' = \frac{-PH}{LEI} \left(\frac{z^2}{2} + C_1 \right)$$

$$u = \frac{-PH}{LEI} \left(\frac{z^3}{6} + C_1 z + C_2 \right)$$

$$\text{BCs } u(0) = 0, u(L) = 0$$

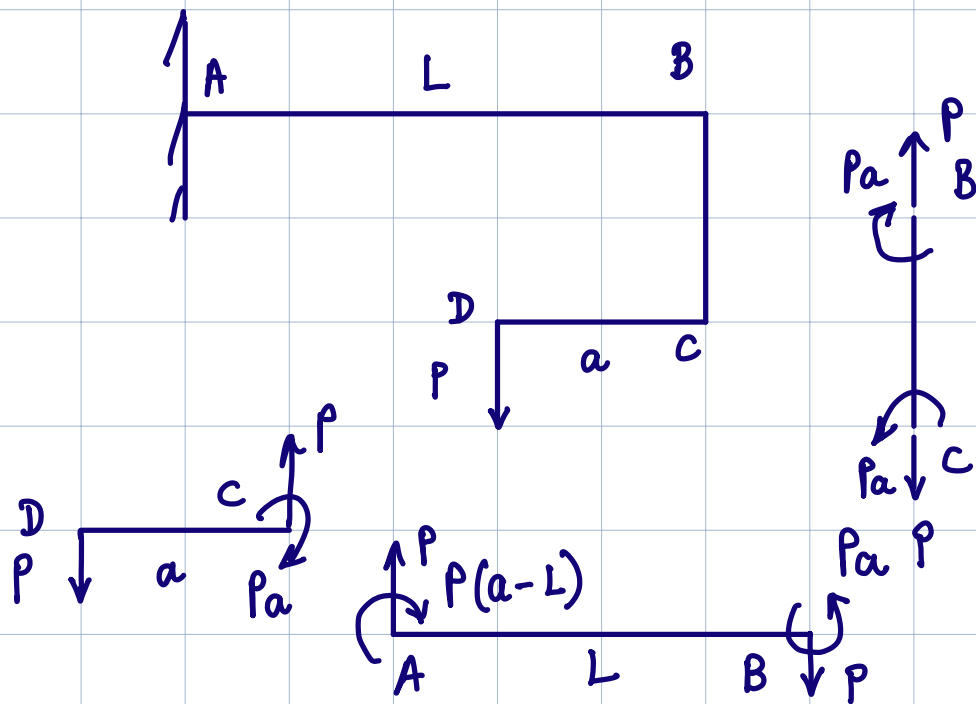
$$\Rightarrow C_2 = 0, C_1 = -L^2/6$$

$$\begin{aligned} \text{want angle at B, } u'(L) &= \frac{-PH}{LEI} \left(\frac{L^2}{2} - \frac{L^2}{6} \right) \\ &= \frac{-PHL}{3EI} \end{aligned}$$

$$\delta_c = \underbrace{\frac{PH^3}{3EI}}_{\text{bending of BC}} + \underbrace{\frac{PHL.H}{3EI}}_{\text{rotation at B from bending of AB}}$$

$$= 30.95 \text{ mm}$$

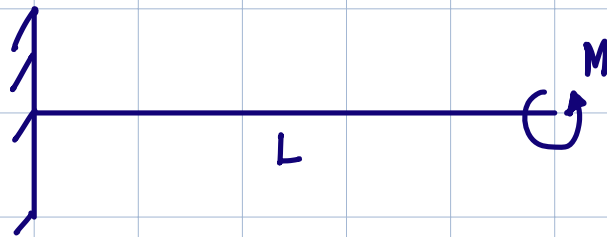
4.



$$\delta_B = \frac{-PL^3}{3EI} + \frac{PaL^2}{2EI} = 0 \text{ given}$$

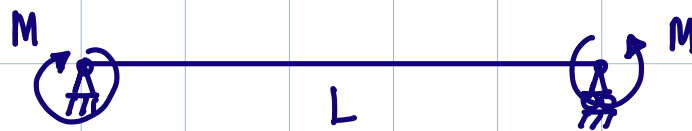
$$\Rightarrow a = \frac{2L}{3}$$

Useful Result for Angle Calculations in Bent Beam Problems



$$u = \frac{Mz^2}{2EI}, \quad u' = \theta = \frac{Mz}{EI}$$

$$\theta_L - \theta_0 = \frac{ML}{EI} - 0 = \frac{ML}{EI}$$



$$u = \frac{-Mz(L-z)}{2EI} = \frac{M(z^2 - Lz)}{2EI}$$

$$u' = \frac{M}{2EI} (2z - L) = \theta$$

$$\theta_L - \theta_0 = \frac{ML}{2EI} - \left(\frac{-ML}{2EI} \right) = \frac{ML}{EI}$$

Both are consistent as both beams have
same applied moment at ends.

This has been used in Prob #2 of
this tutorial.