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ME 202

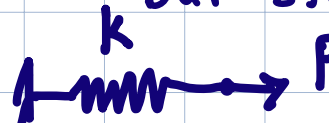
## Energy Methods

- Minimum Potential Energy for Beams
  - Approximate solutions
  - Automated algos which improve the solution
  - "There are no solutions. Only tradeoffs"
  - "Don't let the perfect be the enemy of the good"
  - Basis of FEM (Finite Element Method)
- Minimum Complementary Potential Energy

## Principle of Minimum Potential Energy

Recall,

Bar structure

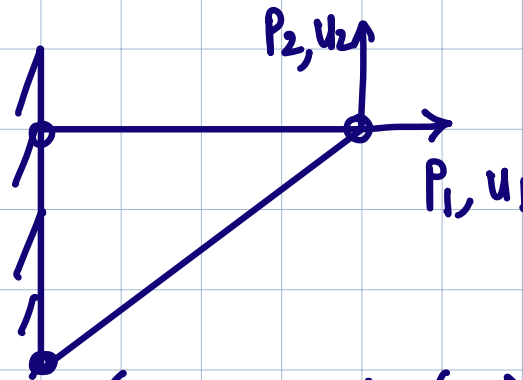


$$\Pi = \frac{1}{2} k u^2 + (-P u)$$

$$\underline{K} \underline{u} = \underline{P}$$

$\Pi$

$$\Pi(u_1, u_2)$$

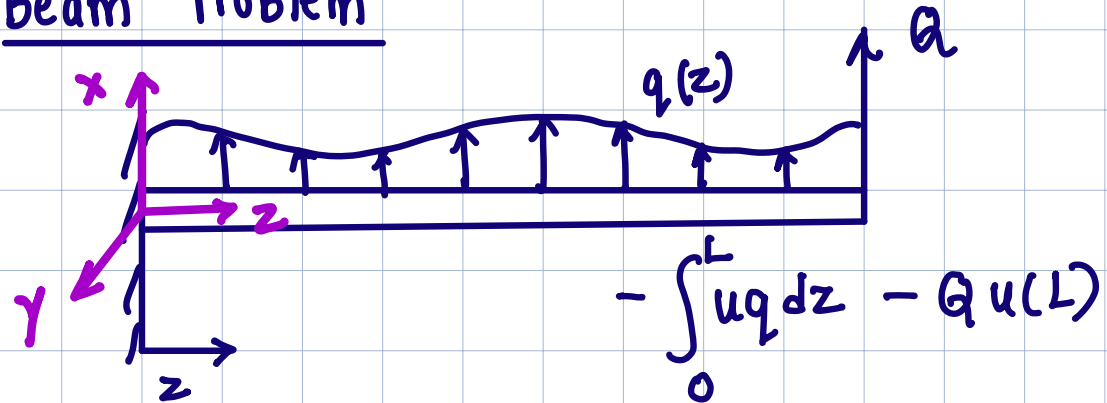


$$\begin{pmatrix} K_{11} & K_{12} \\ K_{21} & K_{22} \end{pmatrix} \begin{pmatrix} u_1 \\ u_2 \end{pmatrix} = \begin{pmatrix} P_1 \\ P_2 \end{pmatrix}$$

Linear algebraic system

$$\Pi = \text{Stored Elastic Energy} + \text{Pot. of Ext. Forces}$$

## Beam Problem



strain  
energy  
density

$$\begin{aligned}
 SED &= \frac{1}{2} \sigma_{zz} \epsilon_{zz} \quad \text{J/m}^3 & \epsilon_{zz} &= -x u'' \\
 &= \frac{1}{2} E \epsilon_{zz}^2 = \frac{1}{2} E x^2 u''^2 & \sigma_{zz} &= E \epsilon_{zz}
 \end{aligned}$$

$$\text{Stored Elastic Energy} = \int_0^L dz \int_{\Omega} \frac{1}{2} E x^2 u''^2 da \quad \uparrow \quad dx dy$$

$$= \int_0^L dz \frac{1}{2} E u''^2 I$$

$$= \int_0^L \frac{1}{2} EI \left( \frac{d^2 u}{dz^2} \right)^2 dz$$

$$\Pi = \underbrace{\int_0^L \frac{1}{2} EI u''^2 dz}_{\text{stored elastic energy}} + \underbrace{\left( - \int_0^L q(z) u dz - Qu(L) \right)}_{\text{pot. energy of applied forces}}$$

Note:  $\Pi$  is a function of  $u(z)$ .

Acc to PMPE,  $u(z)$  which minimizes  $\Pi$

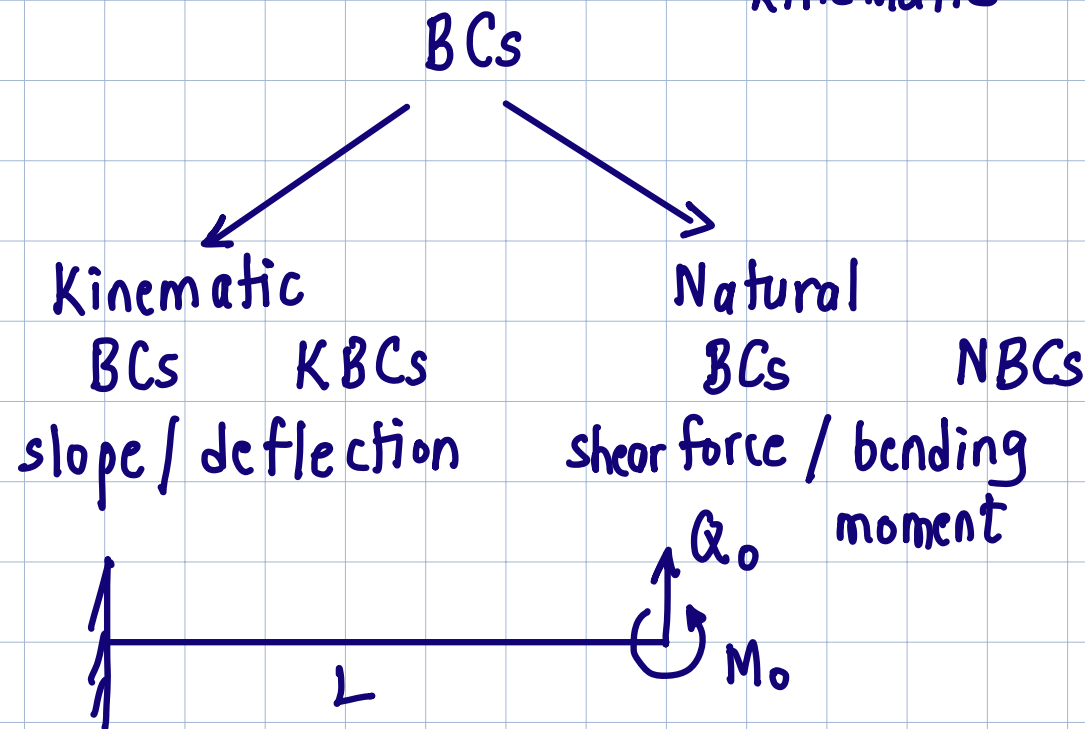
is the one that keeps the beam in equilibrium.

↓  
satisfies the 2<sup>nd</sup>/4<sup>th</sup> order beam equation.

### Approximations

Come up with a linear algebraic system.

- ① choose a  $u(z)$  with unknown coefficients but which will obey the <sup>kinematic</sup> boundary conditions.



$$\text{KBCs: } u(0)=0, \quad u'(0)=0$$

$$\begin{aligned} \text{NBCs: } -EI u'''(L) &= Q_0 \\ EI u''(L) &= M_0 \end{aligned}$$