



# Cooling Tower Experiments

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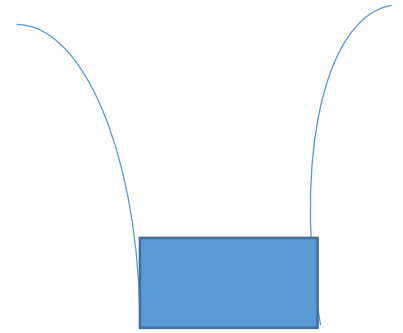


# Cooling Tower

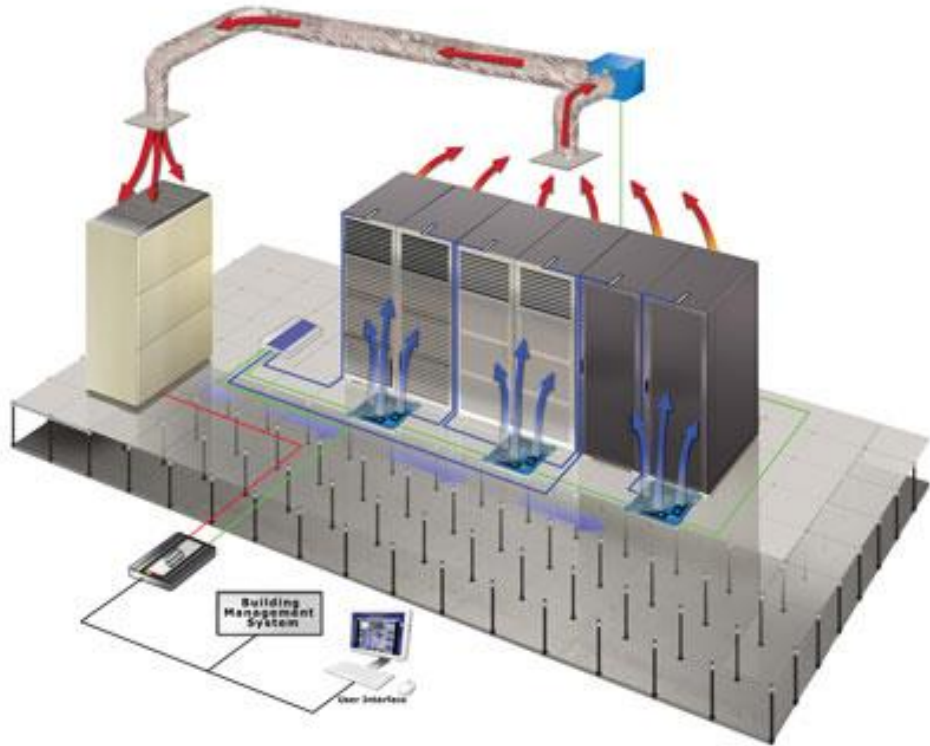


# Introduction

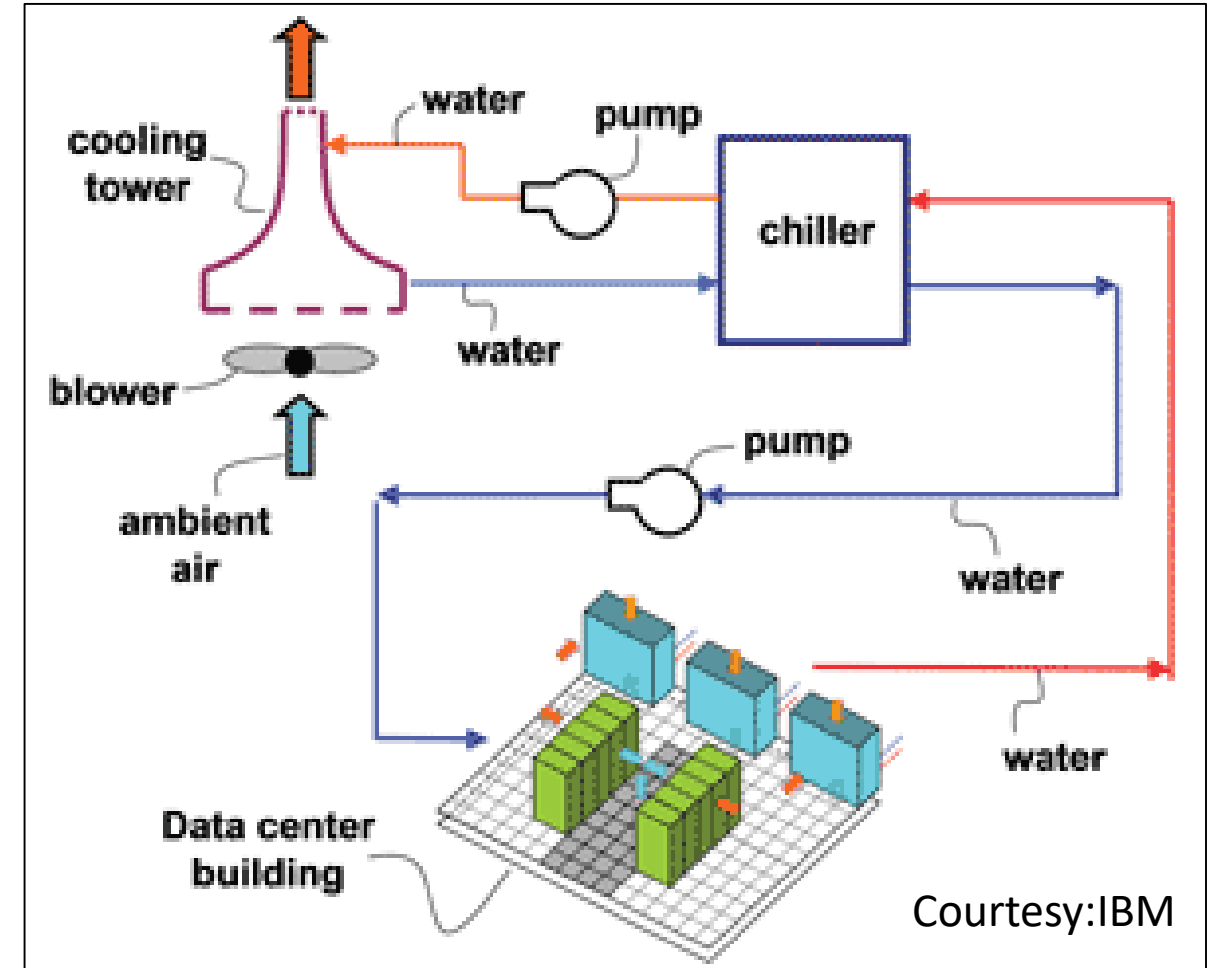
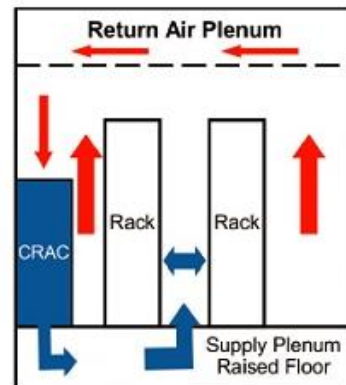
- What is it?
- Applications: powerplants, refrigeration cycles, process industries...
- Types
  - Atmospheric (aspirated flows, hyperbolic towers)
  - Mechanical draft (forced, induced)
  - Hybrid
- Counterflow, crossflow, coflow...
- Shapes



# (1) Datacenter Cooling



Courtesy: eecnet.com



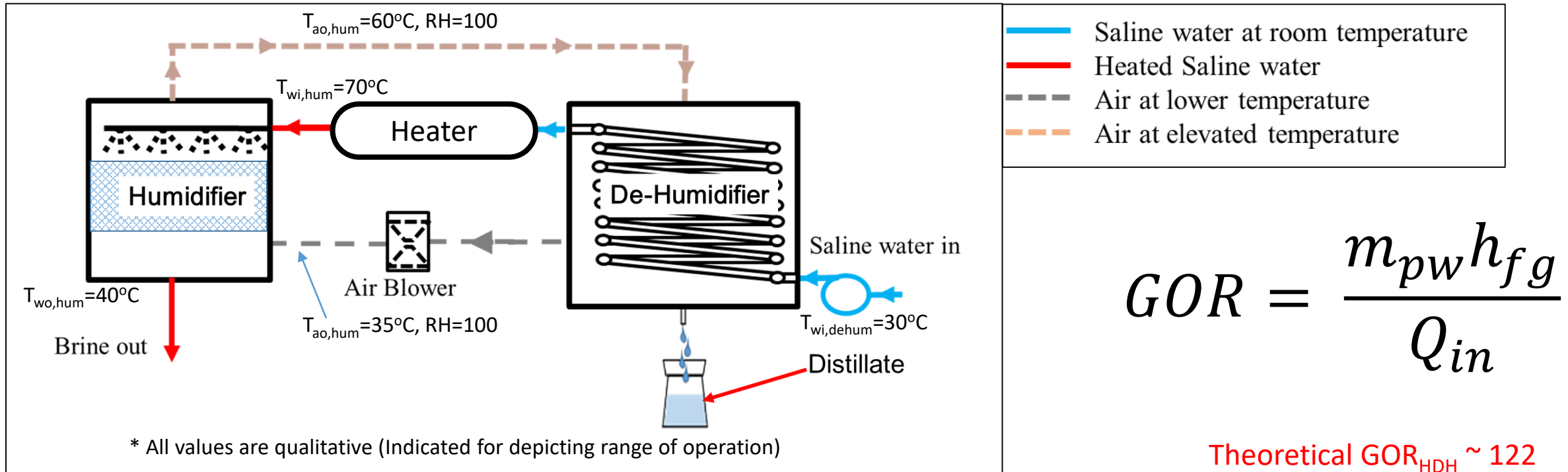
Courtesy: IBM

An Example Air-Cooled Datacenter



# (2) Concept of HDH Desalination

## Decentralized drinking water technology

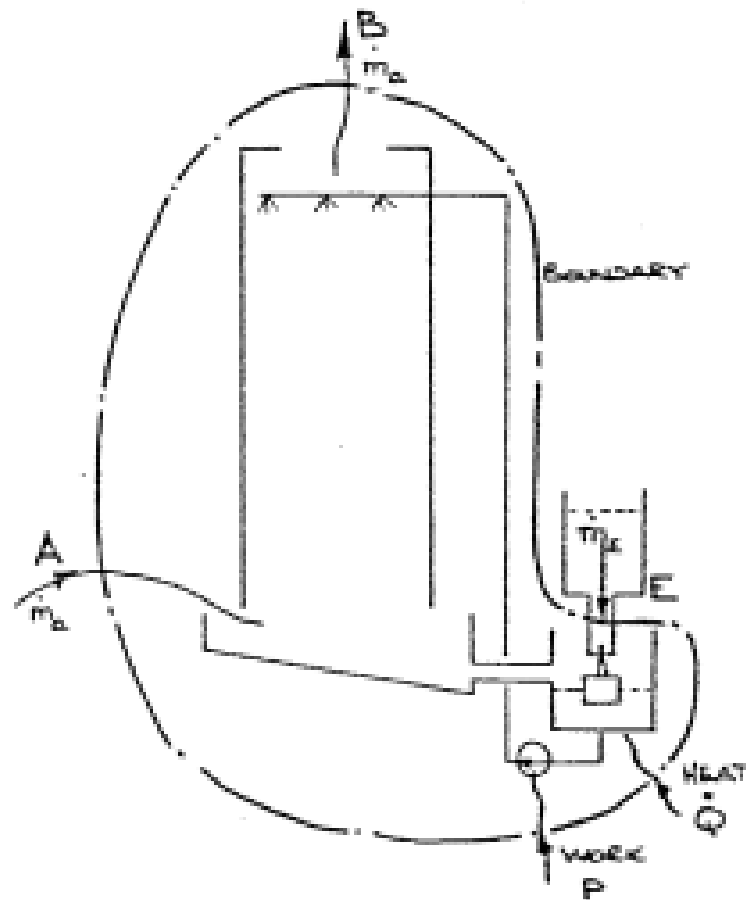


### Advantageous features of HDH desalination

- Wide range of low grade thermal waste heat and solar energy can be used to run HDH system
- Simple and low-cost construction
- Non/low skilled operation requirement
- Membrane free (low maintenance construction)
- Robust under high TDS
- Sub-boiling temperature operation (approx.  $60^{\circ}\text{C}$  -  $80^{\circ}\text{C}$ )



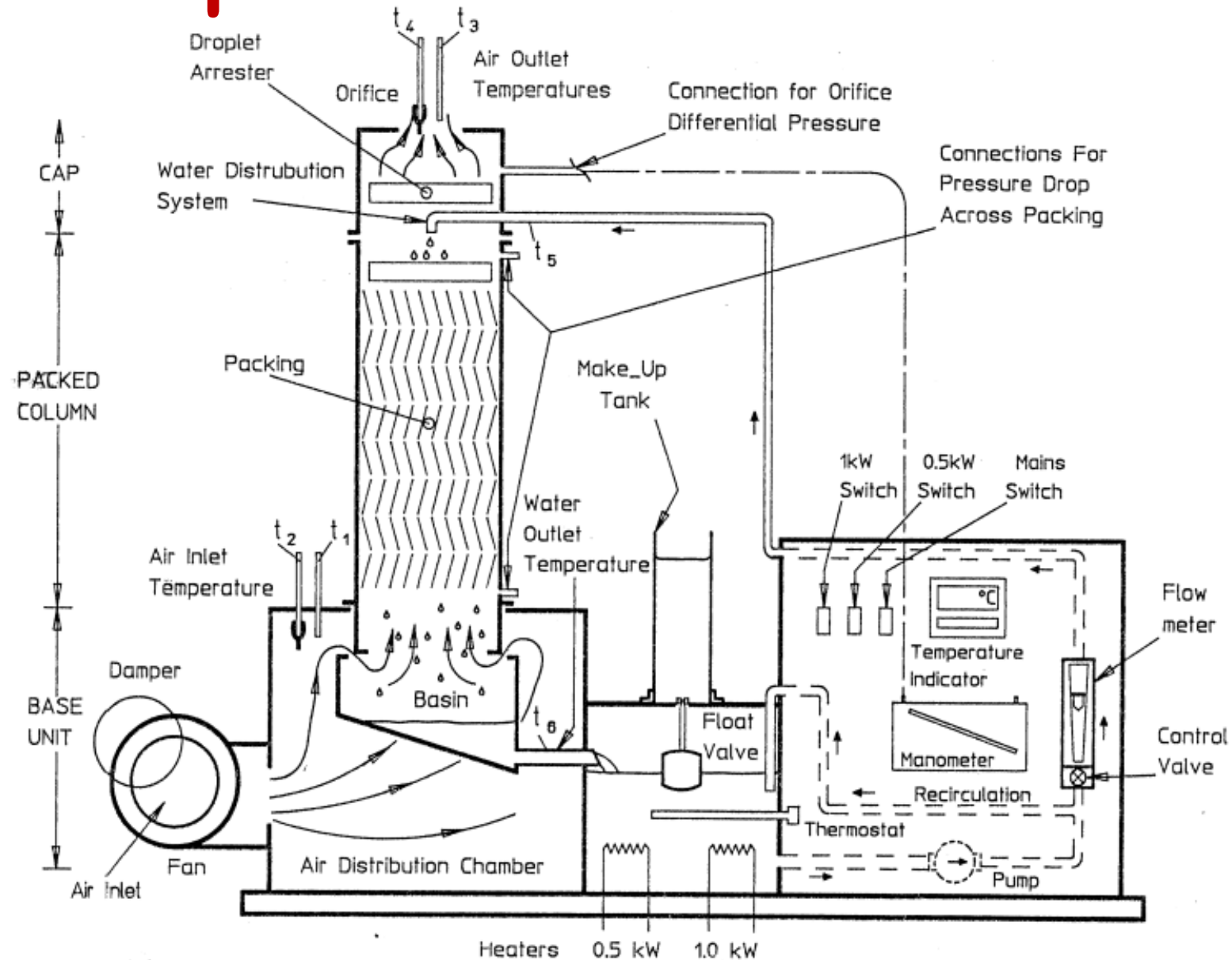
# Experimental Setup



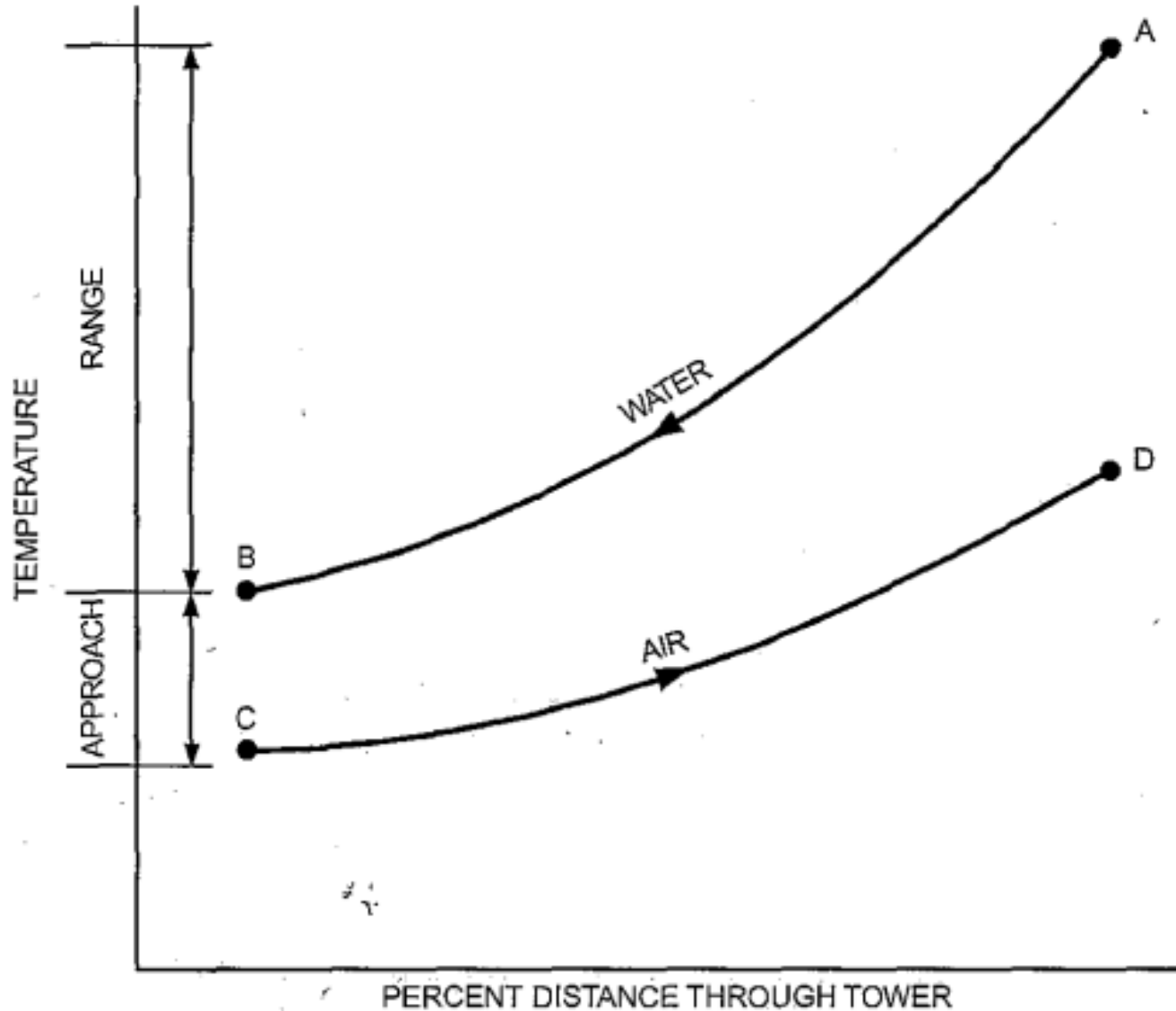
$$\dot{Q} - P = \dot{H}_{Exit} - \dot{H}_{Entry}$$

$$\dot{Q} - P = (\dot{m}_B h_{da} + \dot{m}_B h_s)_B - (\dot{m}_A h_{da} + \dot{m}_B h_s)_A - \dot{m}_E h_E$$

$$\dot{Q} - P = \dot{m}_B (h_B - h_A) - \dot{m}_E h_E$$



# Cooling Tower Basics



Specific humidity: mass of water vapor to total mass of moist air

Relative humidity: mole fraction of water vapor in moist air to that of saturated air

Humidity ratio: ratio of mass of water vapor to dry air





# Parameters of Cooling Towers

A number of parameters describe the performance of a cooling tower.

- **Range** is the temperature difference between the hot water entering the cooling tower and the cold water leaving.
  - Note that the range is not determined by performance of the tower, but is determined by the heat loading.
- **Approach** is the difference between the temperature of the water leaving the tower and the wet bulb temperature of the entering air.
  - The approach is affected by the cooling tower capability.
  - For a given heat loading, water flow rate, and entering air conditions, a larger tower will produce a smaller approach; i.e., the water leaving the tower will be colder.
- **Water/Air Ratio ( $m_w/m_a$ )** is the mass ratio of water (Liquid) flowing through the tower to the air (Gas) flow.
  - Each tower will have a designed water/air ratio.
  - An increase in this ratio will result in an increase of the approach, that is, warmer water will be leaving the tower.





# Thermodynamics of Air Water Systems

Humidity Ratio  $\omega = \frac{\text{Mass Flow of Water Vapour}}{\text{Mass Flow of Dry Air}} = \frac{\dot{m}_v}{\dot{m}_a}$

$$p_a \dot{V} = \dot{m}_a R_a T = \frac{\dot{m}_a R_u T}{M_a} \qquad p_v \dot{V} = \dot{m}_v R_v T = \frac{\dot{m}_v R_u T}{M_v}$$

$$\omega = \frac{\frac{M_v p_v \dot{V}}{R_u T}}{\frac{M_a p_a \dot{V}}{R_u T}} = \left( \frac{M_v}{M_a} \right) \times \left( \frac{p_v}{p_a} \right) = 0.622 \left( \frac{p_v}{p_a} \right)$$



# Model for Cooling Tower

Conservation of Mass for dry air:

$$\dot{m}_{air,in} = \dot{m}_{air,out} = \dot{m}_{air}$$

Conservation of Mass for water:

$$\dot{m}_{water,in} + \omega_{air,in} \dot{m}_{air,in} = \dot{m}_{water,out} + \omega_{air,out} \dot{m}_{air,out}$$

First Law Analysis:

$$\text{Specific enthalpy, } h = h_{air} + \omega h_{vapor}$$

$$\dot{m}_{CW,i} C_W T_{wi} - \dot{m}_{CW,e} C_W T_{we} =$$

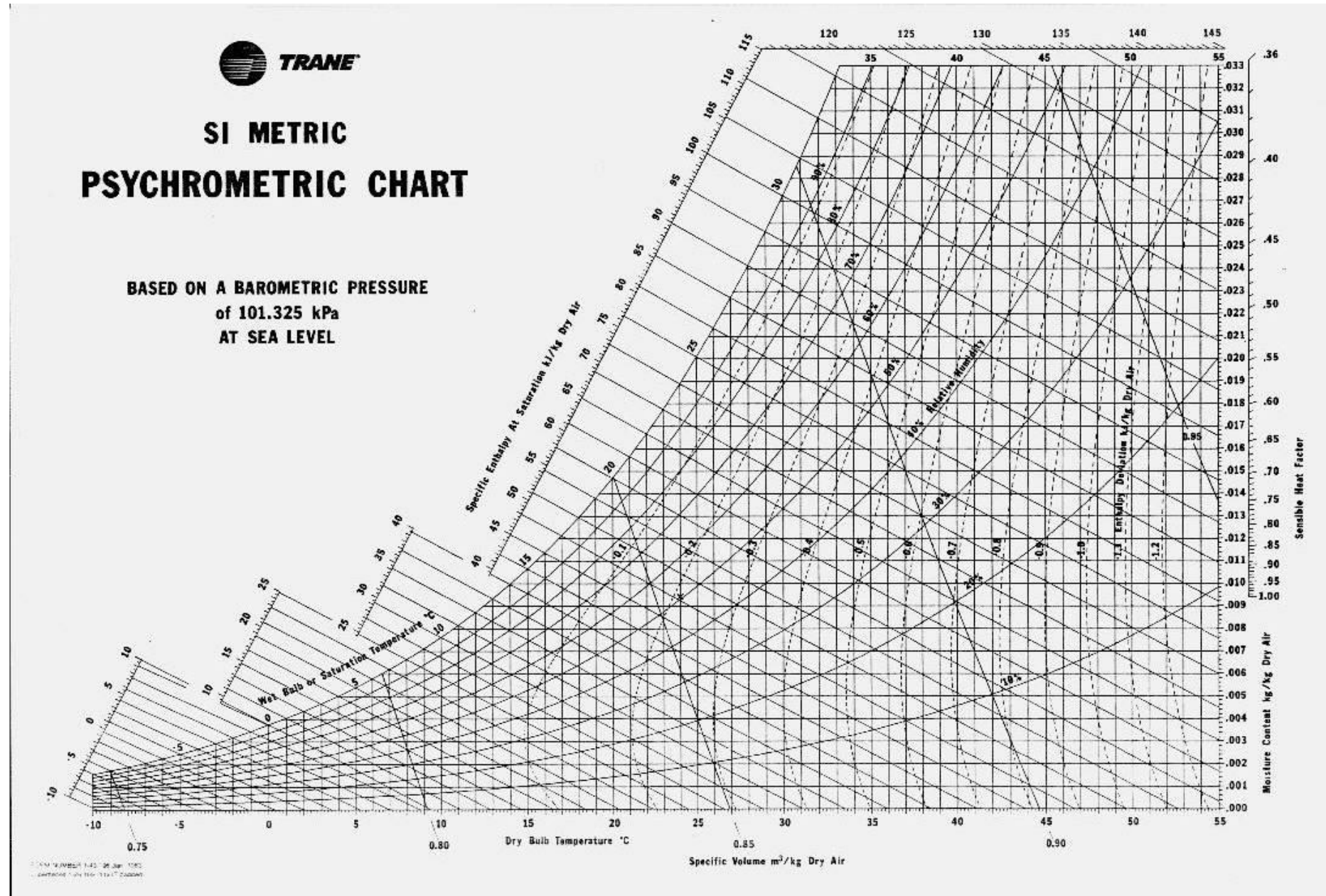
$$\dot{m}_{air} \left\{ \left( C_{p,air} + \omega_e C_{p,steam} \right) \times T_{air,e} - \left( C_{p,air} + \omega_i C_{p,steam} \right) \times T_{air,i} + (\omega_e - \omega_i) (h_{fg}) \right\}$$

**Sensible Heat**

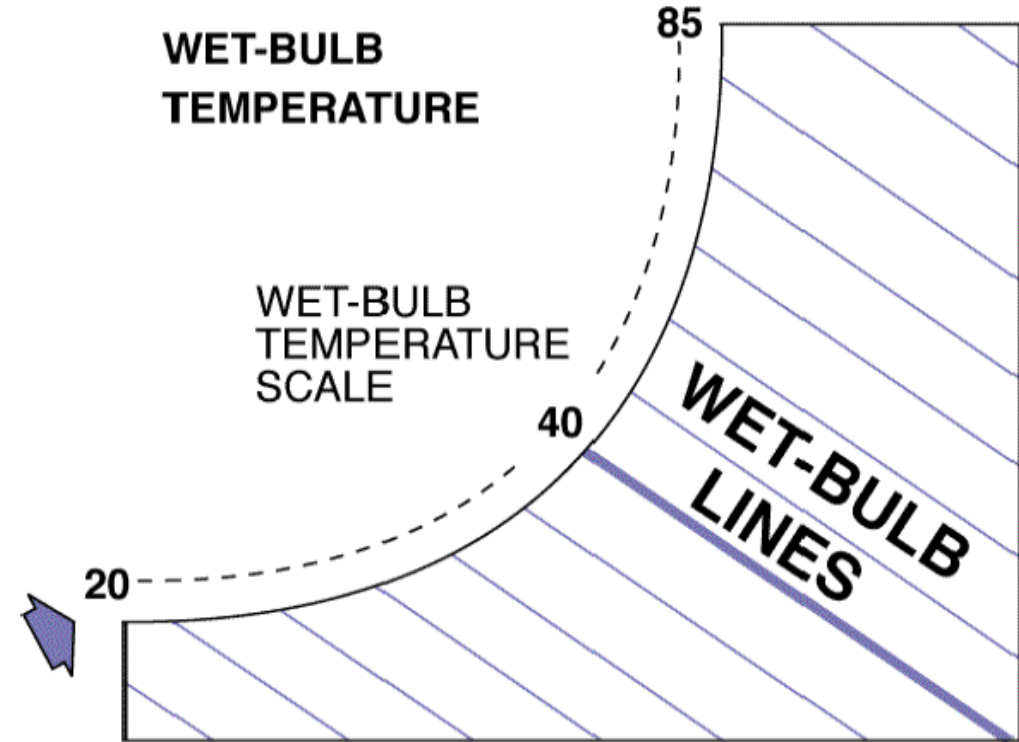
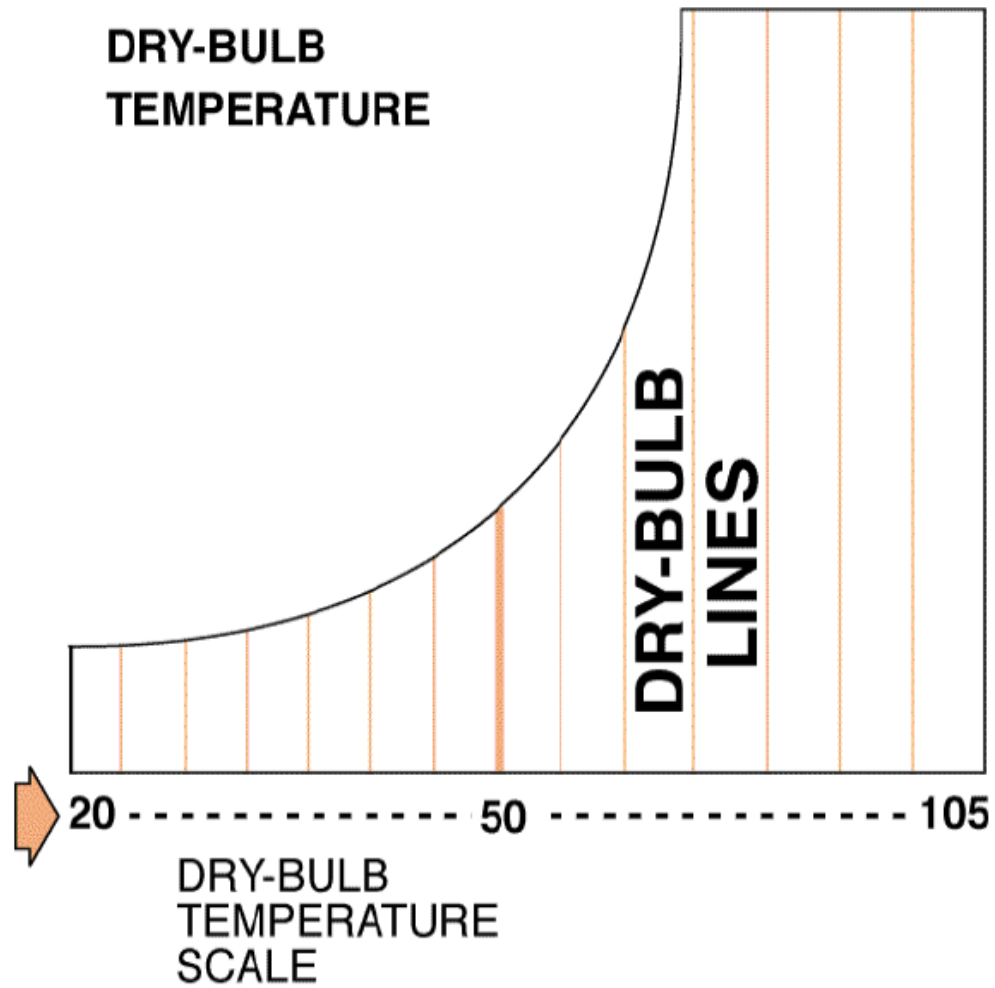
**Latent Heat**



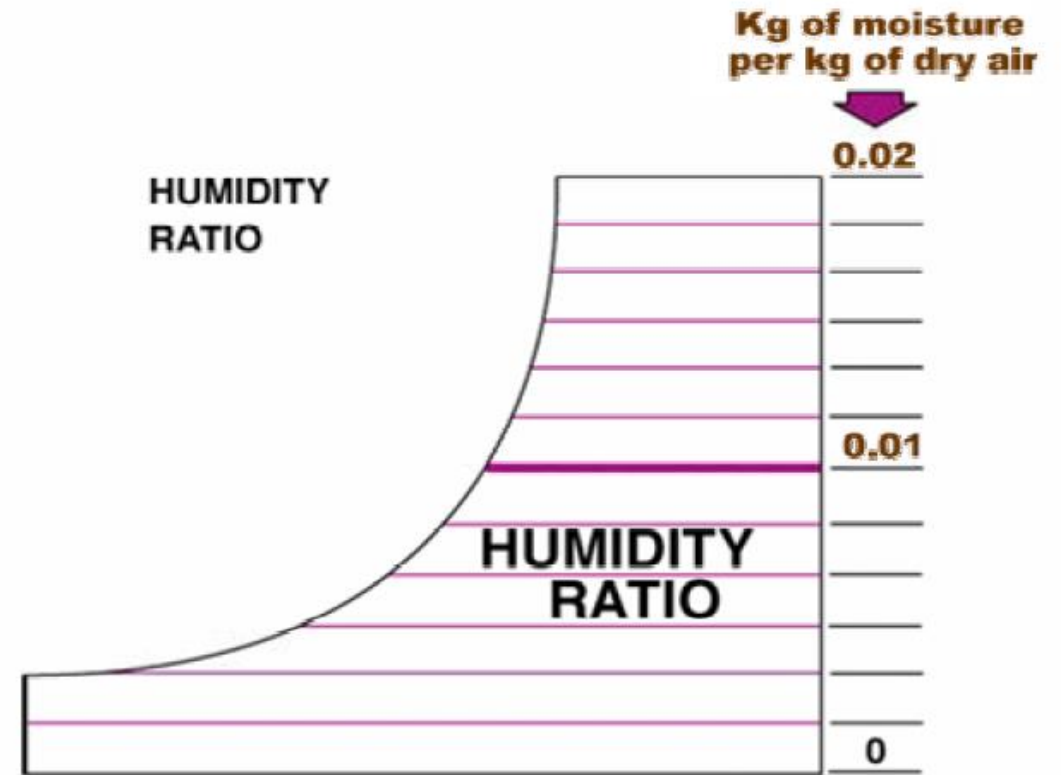
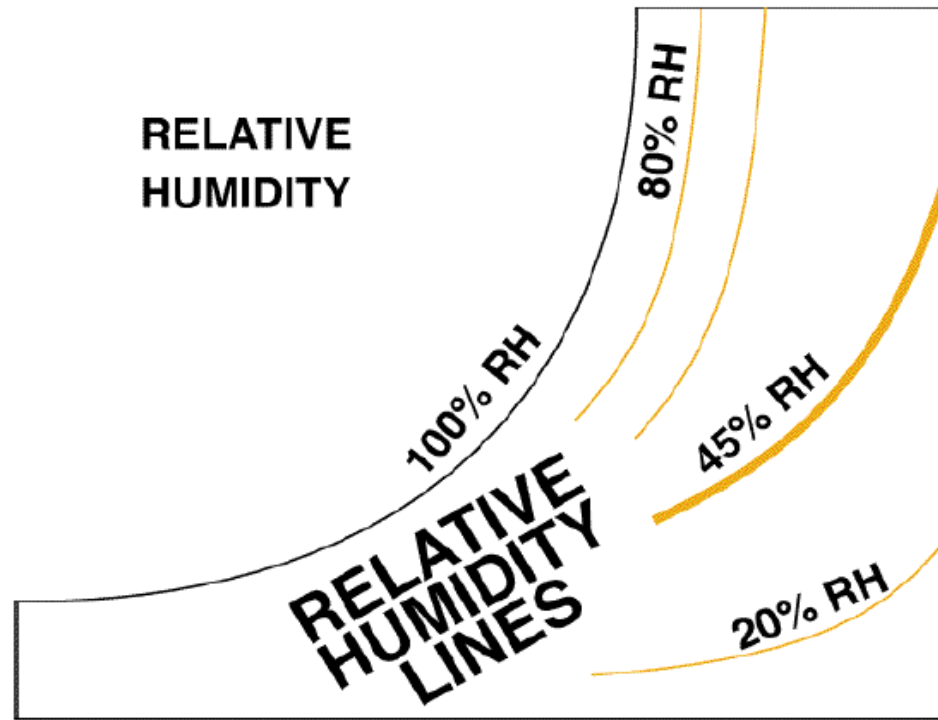
# Psychrometric Chart



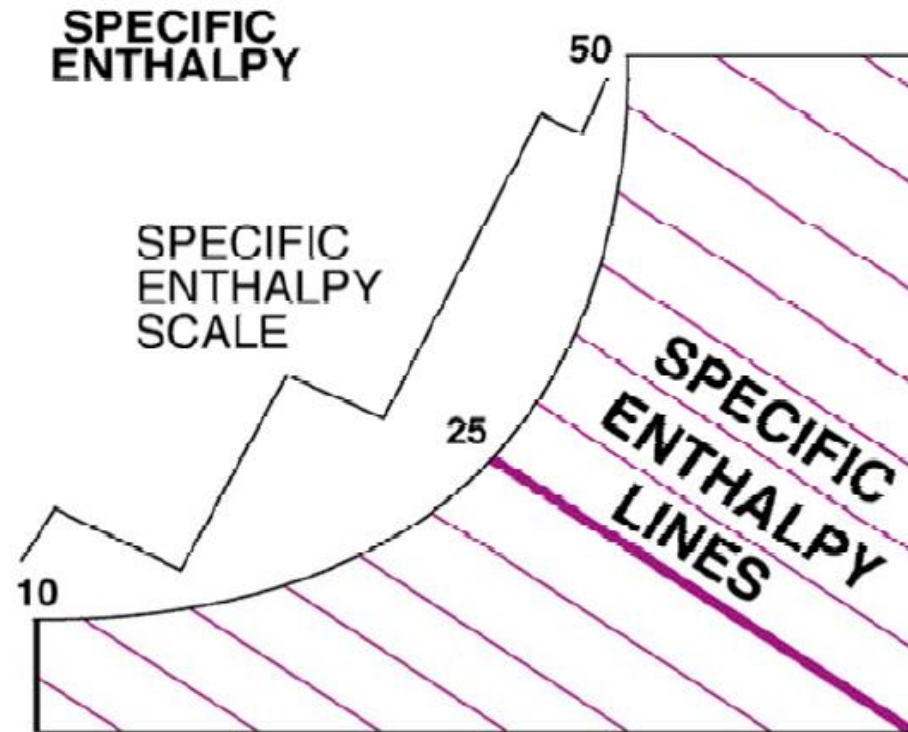
# Dry Bulb vs. Wet Bulb



# RH vs. Humidity Ratio

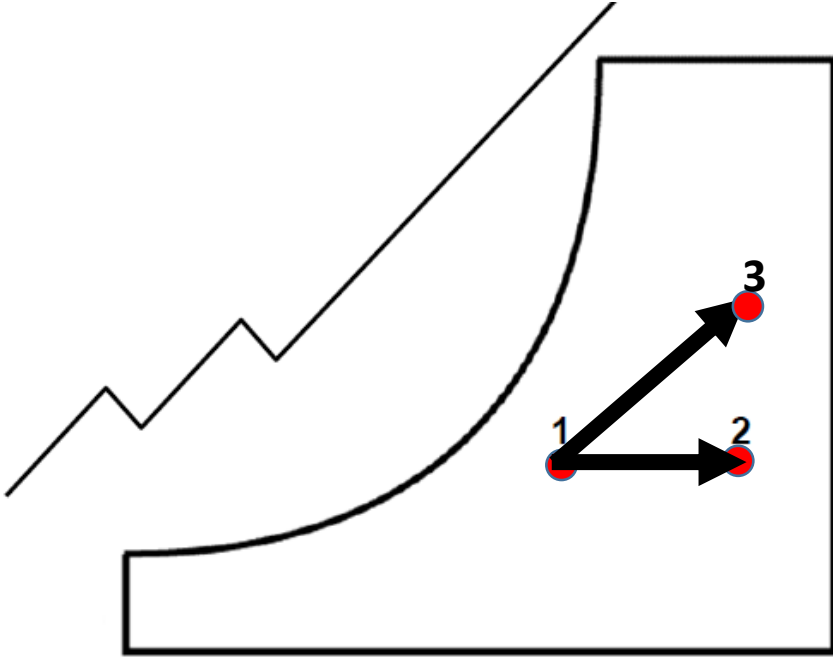


# Specific Enthalpy

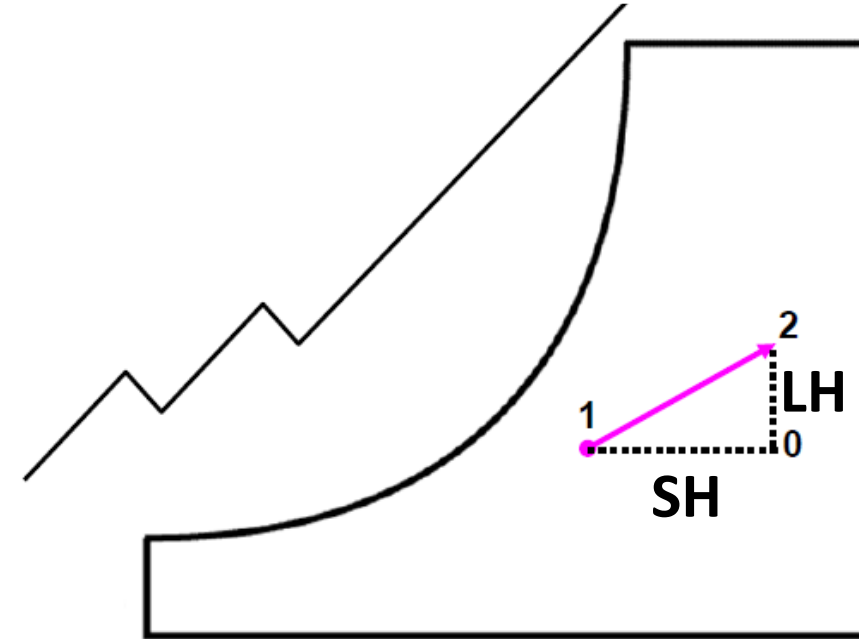




# Sensible Heating/Cooling vs. Humidification



Sensible Heating

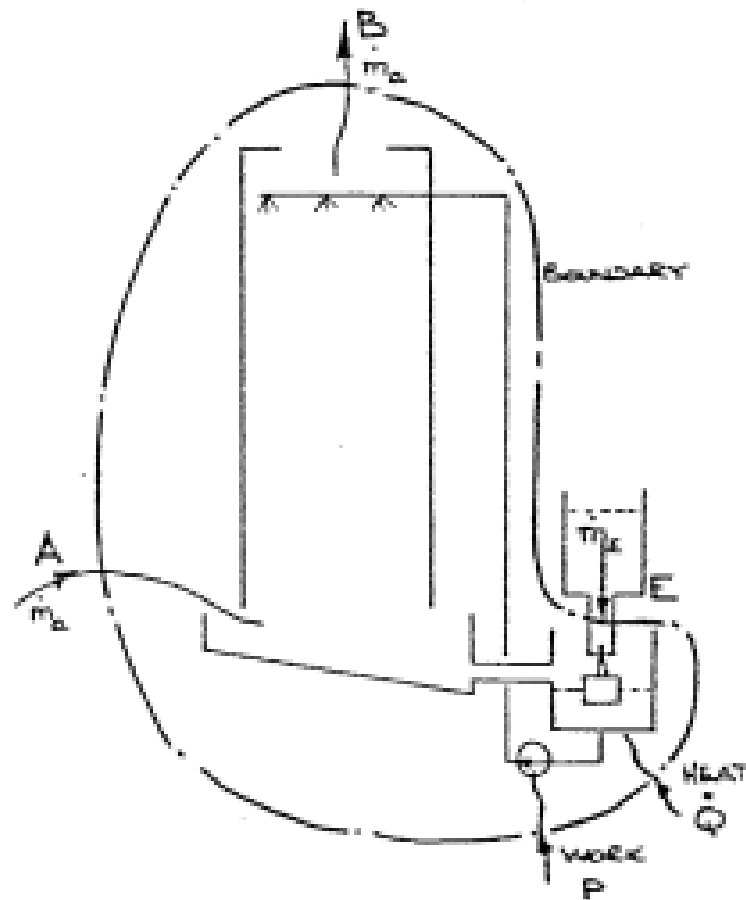


Heating and  
Humidification





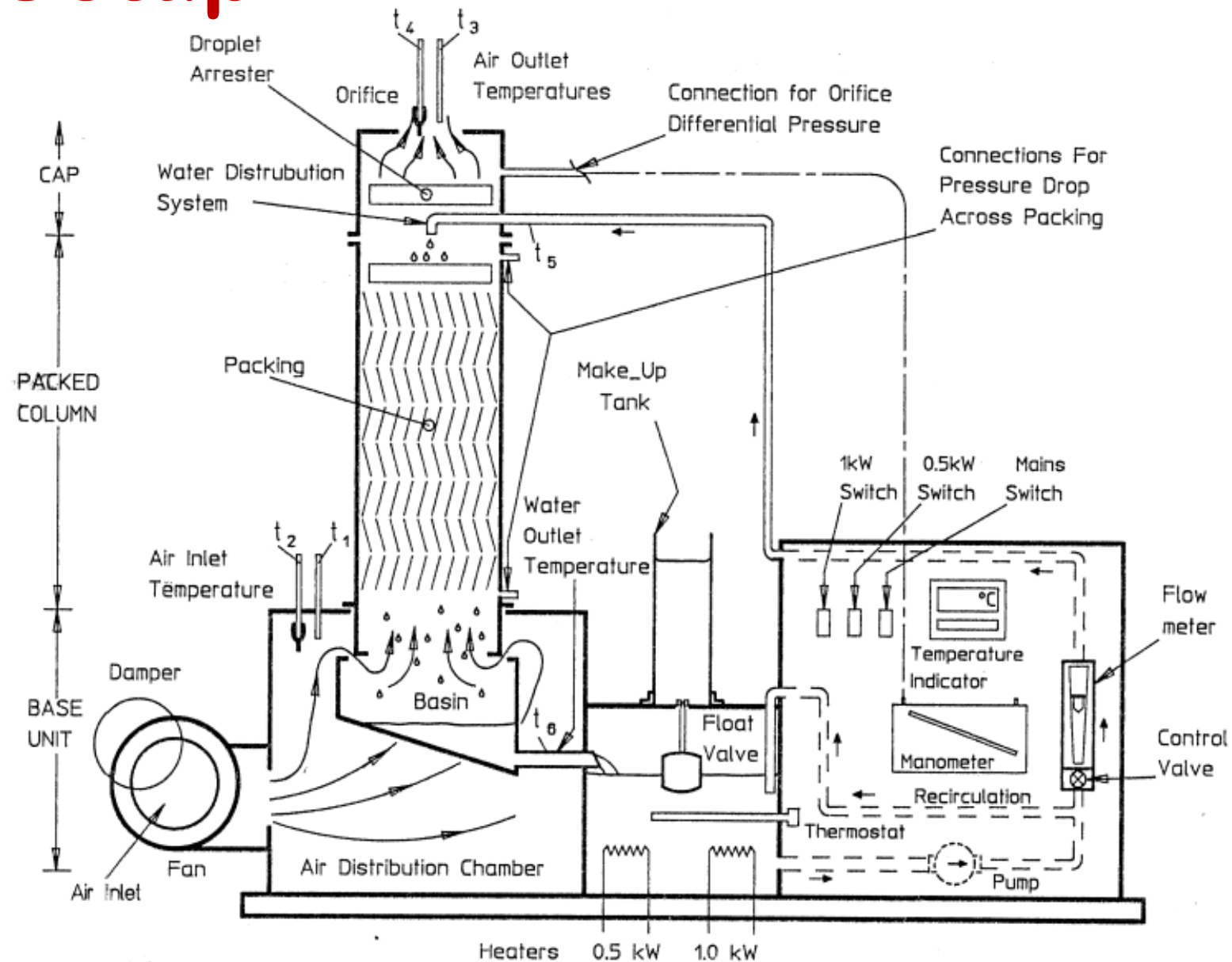
# Experimental Setup



$$\dot{Q} - P = \dot{H}_{Exit} - \dot{H}_{Entry}$$

$$\dot{Q} - P = (\dot{m}_B h_{da} + \dot{m}_S h_S)_B - (\dot{m}_A h_{da} + \dot{m}_S h_S)_A - \dot{m}_E h_E$$

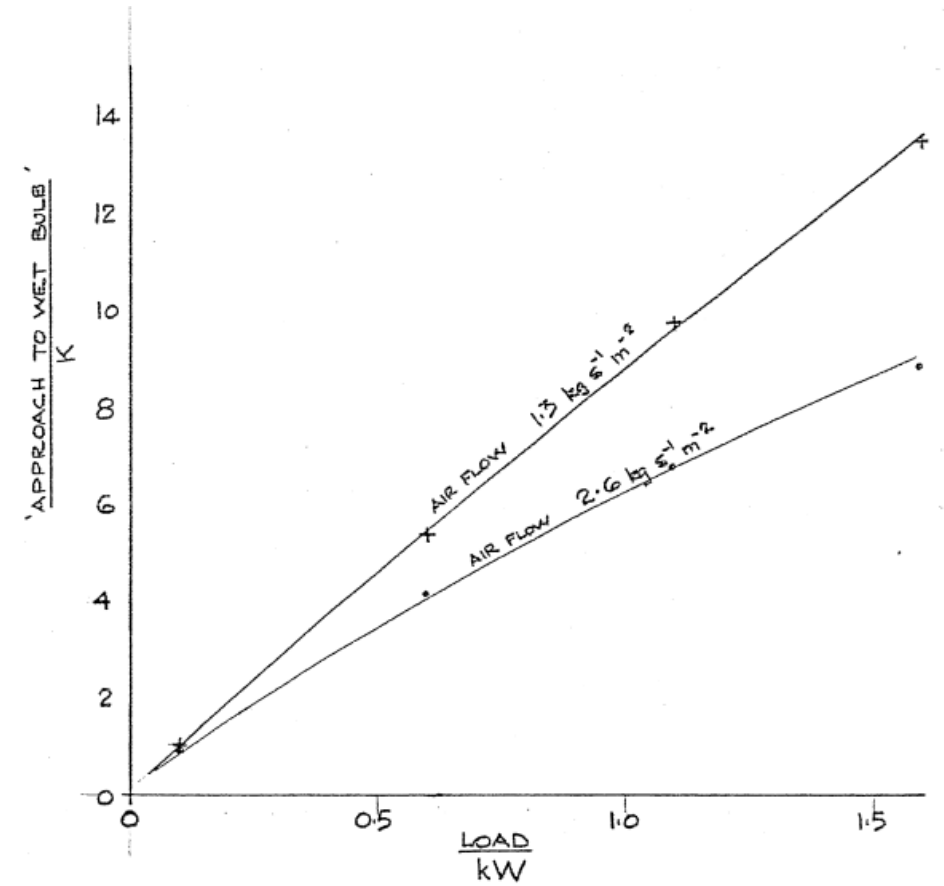
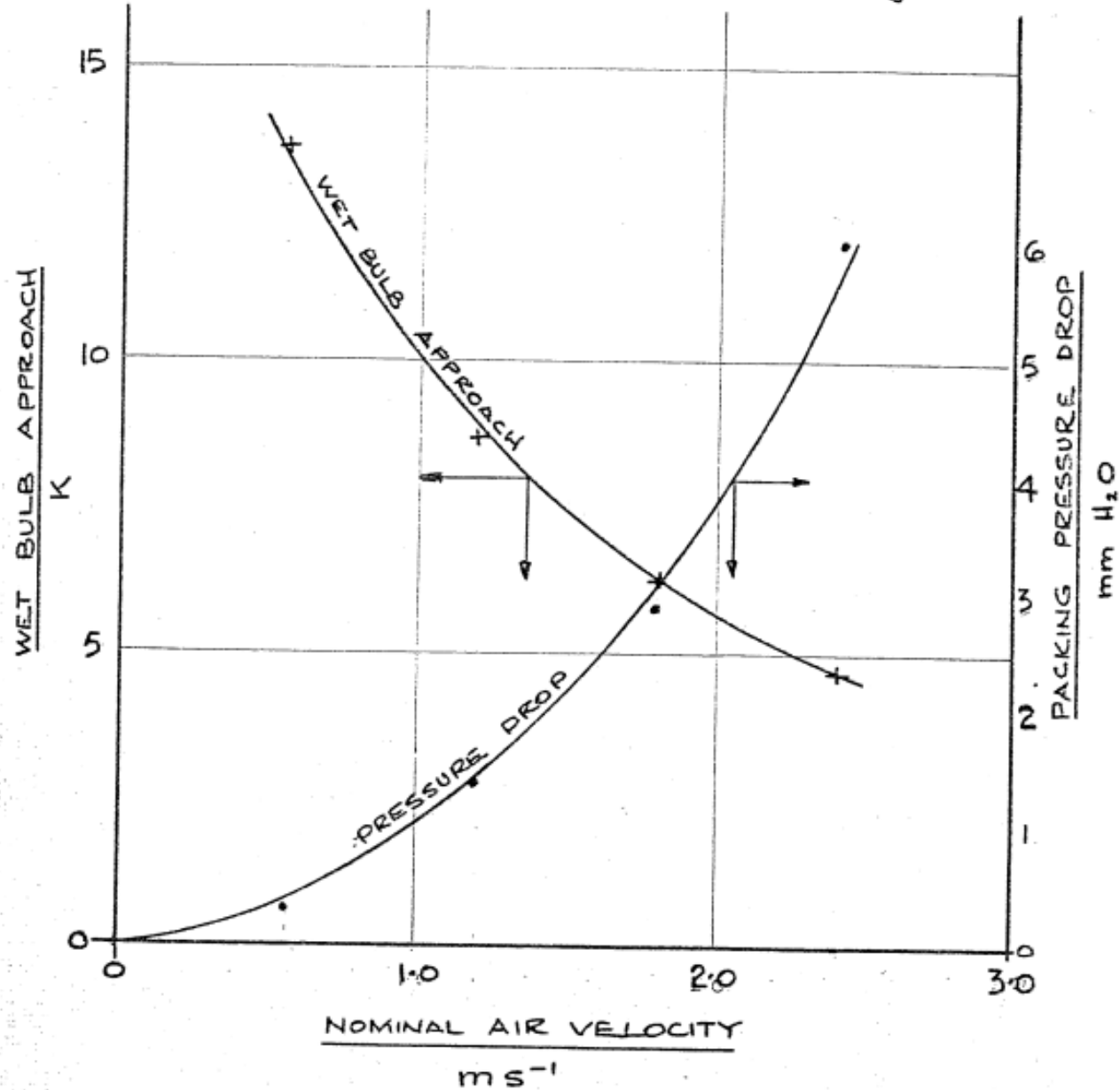
$$\dot{Q} - P = \dot{m}_B(h_B - h_A) - \dot{m}_E h_E$$





DATA:- INLET WET BULB TEMPERATURE  
COOLING LOAD  
PACKING INSTALLED  
WATER FLOW RATE

18°C  
1.1 kW  
C (200m<sup>-1</sup>)  
40 g m s<sup>-1</sup>



DATA:-  
 AIR FLOW RATE  $\approx 0.058 \text{ kg s}^{-1}$   
 PACKING B  
 WATER FLOW RATE  $40 \text{ gm s}^{-1}$

