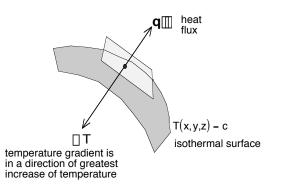
## Fourier's law

### heat conduction in continuous medium





coefficient of thermal conductivity

coefficient of convective heat transfer

thermal diffusivity

- - □W [ m³

heat generation per unit volume

- $q_g = \dot{q}V$

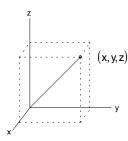
rate of heat generation

- surface emissivity

## **Heat Equation**

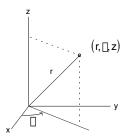
#### Cartesian coordinates

$$\mathbf{q} = \left\langle \left[ k \frac{\partial T}{\partial x}, \left[ k \frac{\partial T}{\partial y}, \left[ k \frac{\partial T}{\partial z} \right] \right] \right\rangle$$



$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} + \frac{\dot{q}}{k} = \frac{1}{\Box} \frac{\partial T}{\partial t}$$

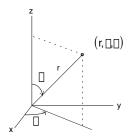
## cylindrical coordinates



$$\frac{1}{r}\frac{\partial}{\partial r} \left[ \frac{\partial T}{\partial r} \right] + \frac{1}{r^2}\frac{\partial^2 T}{\partial \square^2} + \frac{\partial^2 T}{\partial x^2} + \frac{q}{k} = \frac{1}{\square}\frac{\partial T}{\partial t}$$

## spherical coordinates

$$\textbf{q} \text{ } = \left\langle \text{ } \text{ } \text{ } \text{ } \frac{\partial T}{\partial r}, \text{ } \text{ } \text{ } \frac{k}{r sin \text{ } \text{ } |} \frac{\partial T}{\partial \text{ } \text{ } |}, \text{ } \text{ } \text{ } \frac{k}{r} \frac{\partial T}{\partial \text{ } \text{ } |} \right\rangle$$



$$\begin{split} \frac{1}{r^2} \frac{\partial}{\partial r} \left[ \frac{1}{r^2} \frac{\partial T}{\partial r} \right] + \frac{1}{r^2 \sin^2 \frac{\partial}{\partial r}} \frac{\partial^2 T}{\partial \frac{\partial}{\partial r}} \\ + \frac{1}{r^2 \sin \frac{\partial}{\partial r}} \sin \frac{\partial T}{\partial \frac{\partial}{\partial r}} + \frac{\dot{q}}{k} = \frac{1}{r} \frac{\partial T}{\partial t} \end{split}$$

# **Boundary Conditions**

non-linear boundary condtions:

at 
$$v = 0$$

$$\left. \left[ k \frac{\partial T}{\partial x} \right|_{t=0} = h_i \left[ T_{i, \cdot} \ \Box T(0) \right] + \left[ \Box \left[ T_{i, \text{sur}}^4 \ \Box T^4(0) \right] \right]$$

1-d plane wall T(x)<sup>2</sup> specified temperature conduction

at x = L

$$k \frac{\partial T}{\partial x}\Big|_{x=L} = h_2 \Big[ T_2, \Box T(L) \Big] + \Box \Box \Big[ T_{2,sur}^4 \Box T^4(L) \Big]$$

 $= 5.67e \ B$   $= \frac{W}{m^2K^4}$  Stefan - Boltzmann constant

classification of linearized boundary condtions:

- **Dirichlet**
- $T|_{x,0} = T_1$
- constant surface temperature

$$T|_{x=L} = T_2$$

- **Neumann**
- $\left| k \frac{\partial T}{\partial x} \right| = q$
- constant heat flux at the wall

$$k \frac{\partial T}{\partial x} = q \mathbb{Q}$$

 $\left.k\frac{\partial T}{\partial x}\right|_{x=L} = q \text{ adiabatic surface:} \\ \left.\frac{\partial T}{\partial x}\right|_{x=0} = 0 \\ \begin{array}{c} \text{adiabatic surface:} \\ \text{perfectly insulated surface} \\ \text{(no flux thru the wall)} \end{array}$ 

- Ш Robin
- $\prod_{i=1}^{n} k \frac{\partial T}{\partial x} + h_1 T = f_1$
- convective boundary condition

$$\begin{bmatrix} \mathbf{k} \frac{\partial \mathbf{T}}{\partial \mathbf{x}} + \mathbf{h}_2 \mathbf{T} \end{bmatrix}_{\mathbf{k} = \mathbf{k}} = \mathbf{f}_2$$