

```
In [1]: import numpy as np
```

```
In [2]: np.zeros(3)
```

```
Out[2]: array([0., 0., 0.])
```

```
In [3]: # the above code was just used to test to see if the notebook was working
```

```
In [4]: # this entry is testing how to use latex within jupyter notebooks
$$$ Q1: Construct the Products  $A \cdot B$  and  $B \cdot A$ , with  $A$  and  $B$  being: $$$
```

```
A = np.array([[1,2,3],[4,5,6],[7,8,9]])
B = np.array([[10,11,12],[13,14,15],[16,17,18]])
```

```
In [5]: %%latex
Q1: Construct the Products  $A \cdot B$  and  $B \cdot A$ , with  $A$  and  $B$  being
$$$
A = \begin{bmatrix}
1 & 2 & 3 \\
4 & 5 & 6 \\
7 & 8 & 9
\end{bmatrix},
B = \begin{bmatrix}
10 & 11 & 12 \\
13 & 14 & 15 \\
16 & 17 & 18
\end{bmatrix}
$$$
```

Q1: Construct the Products $A \cdot B$ and $B \cdot A$, with A and B being $A = \begin{bmatrix} 1 & 2 & 3 \\ 4 & 5 & 6 \\ 7 & 8 & 9 \end{bmatrix}$, $B = \begin{bmatrix} 10 & 11 & 12 \\ 13 & 14 & 15 \\ 16 & 17 & 18 \end{bmatrix}$

```
In [6]: # using sympy in this section. not sure how to print these nicely.
from sympy import *
A = Matrix([[1,2,3],[4,5,6],[7,8,9]])
B = Matrix([[10,11,12],[13,14,15],[16,17,18]])
```

```
In [7]: print("A*B = ")
A*B
```

A*B =

```
Out[7]: 
$$\begin{bmatrix} 84 & 90 & 96 \\ 201 & 216 & 231 \\ 318 & 342 & 366 \end{bmatrix}$$

```

```
In [8]: print("B*A = ")
B*A
```

B*A =

Out[8]:
$$\left[\begin{matrix} 138 & 171 & 204 \\ 174 & 216 & 258 \\ 210 & 261 & 312 \end{matrix}\right]$$

In [9]: `%%latex`
 Q2: Using the matrix M defined below, determine M^T , the symmetric part of M , and the skew-symmetric part of M .

$$M = \begin{bmatrix} 4 & 5 & -5 \\ -1 & 3 & -1 \\ 7 & 1 & 1 \end{bmatrix}$$

Q2: Using the matrix M defined below, determine M^T , the symmetric part of M , and the skew-symmetric part of M .

$$M = \begin{bmatrix} 4 & 5 & -5 \\ -1 & 3 & -1 \\ 7 & 1 & 1 \end{bmatrix}$$

In [10]: `M = Matrix([[4,5,-5],[-1,3,-1],[7,1,1]])`
`M_transpose = M.T`

In [11]: `%%latex`
 The symmetric part of M , M_S , is defined as

$$M_S = \frac{1}{2}(M + M_T)$$

 And the skew-symmetric part of M , M_K , is defined as

$$M_K = \frac{1}{2}(M - M_T)$$

The symmetric part of M , M_S , is defined as $M_S = \frac{1}{2}(M + M_T)$ And the skew-symmetric part of M , M_K , is defined as $M_K = \frac{1}{2}(M - M_T)$

In [12]: `M_S = (1/2)*(M + M_transpose)`
`M_K = (1/2)*(M - M_transpose)`

In [13]: `print("Symmetric part of M = ")`
`M_S`

Symmetric part of M =

Out[13]:
$$\left[\begin{matrix} 4.0 & 2.0 & 1.0 \\ 2.0 & 3.0 & 0 \\ 1.0 & 0 & 1.0 \end{matrix}\right]$$

In [14]: `print("Skew-symmetric part of M = ")`
`M_K`

Skew-symmetric part of M =

Out[14]:
$$\left[\begin{matrix} 0 & 3.0 & -6.0 \\ -3.0 & 0 & -1.0 \\ 6.0 & 1.0 & 0 \end{matrix}\right]$$

In [15]: `%%latex`
 Q3: Show that the trace of the product $A \cdot B = 0$ if A is symmetric and B is skew-symmetric.

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symmetric

```
In [16]: # use different variables to show that it is ANY symmetric and skew-symmetric matrix
a1,b1,c1,d1,e1,f1,a2,b2,c2,d2,e2,f2 = symbols('a1 b1 c1 d1 e1 f1 a2 b2 c2 d2 e2 f2')
A = Matrix([[a1,b1,c1],[b1,d1,e1],[c1,e1,f1]])
B = Matrix([[0,b2,c2],[-1*b2,0,e2],[-1*c2,-1*e2,0]])
print("Generic symmetric matrix A = ")
print(pretty(A))
print("Generic skew-symmetric matrix B = ")
print(pretty(B))
```

Generic symmetric matrix A =

$$\begin{bmatrix} a_1 & b_1 & c_1 \\ b_1 & d_1 & e_1 \\ c_1 & e_1 & f_1 \end{bmatrix}$$

Generic skew-symmetric matrix B =

$$\begin{bmatrix} 0 & b_2 & c_2 \\ -b_2 & 0 & e_2 \\ -c_2 & -e_2 & 0 \end{bmatrix}$$

```
In [17]: %%latex
Note that for a matrix to be skew-symmetric, its main diagonal must be equal to 0.
 $A^T = -A$  is the definition of a skew-symmetric matrix, and  $x = -x$  iff  $x = 0$ .
We will now compute the trace of  $A \cdot B$ , which is show to be 0 for generic symmetric and skew-symmetric matrices.
```

Note that for a matrix to be skew-symmetric, its main diagonal must be equal to zero because $A^T = -A$ is the definition of a skew-symmetric matrix, and $x = -x$ iff $x = 0$ for all $x \in \mathbb{R}$. We will now compute the trace of $A \cdot B$, which is show to be 0 for generic symmetric and skew-symmetric matrices.

```
In [18]: # showing that the  $tr(A*B) = 0$ 
ab = A*B
#Trace(ab) does not work; sympy does not like computing the trace of variables in a matrix
#trace of a matrix = sum of the main diagonals
trace = ab[0,0] + ab[1,1] + ab[2,2]
print("The trace is equal to: " + str(trace))
```

The trace is equal to: 0

```
In [19]: %%latex
Q4: Find the derivative of  $A(t)$ , where

$$A(t) = \begin{bmatrix} t & t^2 & \sin(\omega t) \\ \cosh(t) & \ln(t) & 17t \\ 1/t & 1/(t^2) & \ln(t^2) \end{bmatrix}$$

```

Q4: Find the derivative of $A(t)$, where $A(t) = \begin{bmatrix} t & t^2 & \sin(\omega t) \\ \cosh(t) & \ln(t) & 17t \\ 1/t & 1/(t^2) & \ln(t^2) \end{bmatrix}$

$$\ln(t) \& 17t \begin{bmatrix} 1/t & 1/(t^2) & \ln(t^2) \end{bmatrix}$$

```
In [20]: t,w = symbols('t,w')
A = Matrix([[t,t**2,sin(w*t)],[cosh(t),ln(t),17*t],[1/t,1/(t**2),ln(t**2)]])
A_dx = A.diff(t)
print("The derivative of A(t) is:")
A_dx
```

The derivative of A(t) is:

```
Out[20]: $\\displaystyle \\left[\\begin{matrix}1 & 2 t & w \\cos\\left(t w \\right)\\\\\\sinh\\left(t \\right) & \\frac{1}{t} & 17\\\\- \\frac{1}{t^{2}} & - \\frac{2}{t^{3}} & \\frac{2}{t}\\end{matrix}\\right]$
```

```
In [21]: %%latex
Q5: Find the eigenvalues and eigenvectors of the matrix $G$;
$$
G = \\begin{bmatrix}
10 & 50 & -50 \\\\
50 & 100 & 10\\\\
-50 & 10 & 100 \\\\
\\end{bmatrix}
$$
```

Q5: Find the eigenvalues and eigenvectors of the matrix G ; $G = \begin{bmatrix} 10 & 50 & -50 \\ 50 & 100 & 10 \\ -50 & 10 & 100 \end{bmatrix}$

```
In [22]: G = Matrix([[10,50,-50],[50,100,10],[-50,10,100]])
#G_np = np.array([[1,2,3],[2,4,5],[3,5,6]])
eigen = list(G.eigenvals().keys())
eigen_vec = G.eigenvecs()
#eig_np,x = np.linalg.eig(G_np)
```

```
In [23]: #eig_np
print("Eigenvalues of G:")
print(float(eigen[0]))
print(float(eigen[1]))
print(float(eigen[2]))
print("Corresponding Eigenvectors of G:")
print(pretty(eigen_vec[0][2][0]))
print(pretty(eigen_vec[1][2][0]))
print(pretty(eigen_vec[2][2][0]))
```

Eigenvalues of G:

110.0

-31.240384046359605

131.2403840463596

Corresponding Eigenvectors of G:

$$\begin{bmatrix} 0 \\ 1 \\ 1 \\ 4 \\ 5 \end{bmatrix} + \frac{\sqrt{66}}{5}$$

$$\begin{bmatrix} -1 & \\ 4 & \frac{1}{\sqrt{66}} \\ 5 & \frac{1}{5} \\ -1 & \\ 1 & \end{bmatrix}$$

In [24]: `%%latex`
Q6: Determine G^{-1} then show that the eigenvalues of G^{-1} are the inverses of the eigenvalues of G .

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In [25]: `#G_np_inv = np.linalg.inv(G_np)
#G_np_inv_eig,z = np.linalg.eig(G_np_inv)
G_inv = G.inv()
G_inv`

Out[25]:
$$\left(\begin{bmatrix} \frac{9}{410} & \frac{1}{82} & -\frac{1}{82} & \frac{3}{902} & \frac{13}{2255} \\ -\frac{1}{82} & \frac{13}{2255} & \frac{3}{902} & \end{bmatrix}\right)$$

In [26]: `G_inv_eig = list(G_inv.eigenvals().keys())
for i in range(0,3):
 print("Inverse of eigenvalue " + str(i+1) + " of G and the corresponding eigenvalue of G_inv:
 print(float(1/eigen[i]))
 print(float(G_inv_eig[i]))
print("Clearly, the above values are equal")`

Inverse of eigenvalue 1 of G and the corresponding eigenvalue of G_inv:
0.00909090909090909
0.00909090909090909
Inverse of eigenvalue 2 of G and the corresponding eigenvalue of G_inv:
-0.03200984976740478
-0.03200984976740478
Inverse of eigenvalue 3 of G and the corresponding eigenvalue of G_inv:
0.007619605864965757
0.007619605864965757
Clearly, the above values are equal

In [27]: `#^ need to figure out how to index these; definitely are inverses of each other.`

In [28]: `%%latex
Q7 For the matrix T with the following Cartesian components
$$
T = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \frac{11}{4} & -\frac{\sqrt{3}}{4} \\ 0 & -\frac{\sqrt{3}}{4} & \frac{9}{4} \end{bmatrix}
$$`

Q7 For the matrix T with the following Cartesian components $T = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \frac{11}{4} & -\frac{\sqrt{3}}{4} \\ 0 & -\frac{\sqrt{3}}{4} & \frac{9}{4} \end{bmatrix}$

In [29]:

```
# a) Find the eigenvalues and eigenvectors
# b) Find the angle between the first eigenvector and the x1 axis
# c) That the eigenvectors form an orthonormal basis

T = np.array([[1,0,0],[0,11/4,-1*np.sqrt(3)/4],[0,-1*np.sqrt(3)/4,9/4]])
# eigenvalues and eigenvectors
T_eig,T_eigVec = np.linalg.eig(T)
# need to order these
print("Eigenvalues:")
print(np.sort(T_eig)[::-1])
# order these by size
print("First eigenvector:")
print(T_eigVec[1])
print("Second eigenvector:")
print(T_eigVec[2])
print("Third Eigenvector:")
print(T_eigVec[0])
# angle between first eigenvalue and x1 axis
theta_x1 = np.arccos(T_eigVec[1][0]/(np.sqrt(T_eigVec[1][0]**2 + T_eigVec[1][1]**2)))
print("Angle between first eigenvector and the x1 axis = " + str(theta_x1))
# Show that the eigenvectors form an orthonormal basis
print("Showing that all the eigenvectors are unit vectors:")
for j in range(0,3):
    print("norm of eigenvector " + str(T_eigVec[j]) + ": " + str(np.linalg.norm(T_eigVec[j])))
# show that they are all orthogonal
print("the inner product of all combinations of eigenvectors is 0")
print(np.dot(T_eigVec[0],T_eigVec[1]))
print(np.dot(T_eigVec[0],T_eigVec[2]))
print(np.dot(T_eigVec[1],T_eigVec[2]))
print("---hence, the eigenvectors form an orthonormal basis---
```

```
Eigenvalues:
[3. 2. 1.]
First eigenvector:
[-0.5      -0.8660254  0.          ]
Second eigenvector:
[-0.8660254  0.5        0.          ]
Third Eigenvector:
[0.  0.  1.]
Angle between first eigenvector and the x1 axis = 2.0943951023931957
Showing that all the eigenvectors are unit vectors:
norm of eigenvector [0. 0. 1.]: 1.0
norm of eigenvector [-0.5      -0.8660254  0.          ]: 1.0
norm of eigenvector [-0.8660254  0.5        0.          ]: 1.0
the inner product of all combinations of eigenvectors is 0
0.0
0.0
0.0
---hence, the eigenvectors form an orthonormal basis---
```

In [30]:

```
%%latex
2. The Stress tensor  $T$ 
Q8: The state of stress at a certain point in a body is given by
 $T = \begin{bmatrix} 1 & 2 & 3 \\ 2 & 4 & 5 \end{bmatrix}$ 
```

```

3 & 5 & 0 \\
\end{bmatrix} \text{MPa}

```

\$\$

On each of the coordinate planes (with normals e_1, e_2, e_3), determine the normal stress and the magnitude of the total shear stress.

\$\$

Answer: the normal stress on each coordinate plane e_i is equivalent to T_{ii} and the shear stress is equivalent to the magnitude of T_{ij} , where $i \neq j$. Hence, we have as follows:

2. The Stress tensor $Q8$: The state of stress at a certain point in a body is given by $T = \begin{bmatrix} 1 & 2 & 3 \\ 2 & 4 & 5 \\ 3 & 5 & 0 \end{bmatrix} \text{MPa}$ On each of the coordinate planes (with normals e_1, e_2, e_3), determine the normal stress and the magnitude of the total shear stress. Answer: the normal stress on each coordinate plane e_i is equivalent to T_{ii} , and the shear stress is equivalent to the magnitude of T_{ij} , where $i \neq j$. Hence, we have as follows:

In [31]:

```

T = Matrix([[1,2,3],[2,4,5],[3,5,0]])
print("Answer to Question 8:")
for i in range(0,3):
    print("On the plane e" + str(i+1) + ":")
    print("Normal stress = " + str(T[i,i]))
    print("Magnitude of Total Shear Stress = " + str(sqrt(T[i,i-1]**2. + T[i,i-2]**2.)))

```

Answer to Question 8:

On the plane e1:

Normal stress = 1

Magnitude of Total Shear Stress = 3.60555127546399

On the plane e2:

Normal stress = 4

Magnitude of Total Shear Stress = 5.38516480713450

On the plane e3:

Normal stress = 0

Magnitude of Total Shear Stress = 5.83095189484530

In [32]:

```

%%latex
Q9: The state of stress at a certain point in a body is given by
$$
T = \begin{bmatrix} 2 & -1 & 3 \\ -1 & 4 & 0 \\ 3 & 0 & -1 \end{bmatrix} \text{MPa}
$$
For the plane defined by its normal  $n = 2e_1 + 2e_2 + e_3$ , find the traction (or stress) vector, and the magnitude of the normal and shear stress.

```

Q9: The state of stress at a certain point in a body is given by $T = \begin{bmatrix} 2 & -1 & 3 \\ -1 & 4 & 0 \\ 3 & 0 & -1 \end{bmatrix} \text{MPa}$ For the plane defined by its normal $n = 2e_1 + 2e_2 + e_3$, find the traction (or stress) vector, and the magnitude of the normal and shear stress.

In []:

```
In [33]: # Q9 Answer:
print("Q9: Answer")
n = Matrix([2,2,1])
T = Matrix([[2,-1,3],[-1,4,0],[3,0,-1]])
# t = T*n
t = T*n
print("Traction vector t = ")
print(pretty(t))
# magnitude of normal stress on this plane = t dot n
normal_stress = t.dot(n)
# magnitude of shear stress on this plane = magnitude(t - normal_stress*n)
shear_stress = t - normal_stress*n
shear_stress_norm = shear_stress.norm()
print("Normal Stress = " + str(normal_stress))
print("Shear Stress = ")
print(pretty(shear_stress))
```

Q9: Answer

Traction vector $t =$

$$\begin{bmatrix} 5 \\ 6 \\ 5 \end{bmatrix}$$

Normal Stress = 27

Shear Stress =

$$\begin{bmatrix} -49 \\ -48 \\ -22 \end{bmatrix}$$

```
In [34]: %%latex
Q10: Given  $T_{11} = 1$  MPa and  $T_{22} = -1$  MPa, with all other  $T_{ij} = 0$  at a point in a continuum, show that the only plane in which the traction vector is zero is the plane whose normal is  $e_3$ .
```

Q10: Given $T_{11} = 1$ MPa and $T_{22} = -1$ MPa, with all other $T_{ij} = 0$ at a point in a continuum, show that the only plane in which the traction vector is zero is the plane whose normal is e_3 .

```
In [35]: T = Matrix([[1,0,0],[0,1,0],[0,0,0]])
# define arbitrary constants that give the components of an arbitrary normal vector
x, y, z = symbols('x y z')
n = Matrix([x,y,z])
t = T*n
print("Q10 Answer:")
print(pretty(t))
print("Regardless of the choice of plane, the only plane in which the traction vector is zero is the plane e3 (the third entry in the vector.)")
```

Q10 Answer:

$$\begin{bmatrix} x \\ y \\ 0 \end{bmatrix}$$

Regardless of the choice of plane, the only plane in which the traction vector is zero is the plane e_3 (the third entry in the vector.)

In [36]:

```
%%latex
3. The Strain tensor  $\epsilon$ 
Q11: The strain tensor at a point is given by

$$\epsilon = \begin{bmatrix} 1 & -3 & \sqrt{2} \\ -3 & 1 & -\sqrt{2} \\ \sqrt{2} & -\sqrt{2} & 4 \end{bmatrix}$$

Determine the extensional (i.e. normal) and shear strains in the direction
 $n = (1/2)e_1 - (1/2)e_2 + (1/\sqrt{2})e_3$ 
```

3. The Strain tensor ϵ Q11: The strain tensor at a point is given by $\epsilon = \begin{bmatrix} 1 & -3 & \sqrt{2} \\ -3 & 1 & -\sqrt{2} \\ \sqrt{2} & -\sqrt{2} & 4 \end{bmatrix}$ Determine the extensional (i.e. normal) and shear strains in the direction $n = (1/2)e_1 - (1/2)e_2 + (1/\sqrt{2})e_3$

In [37]:

```
# are these supposed to magnitudes, or vectors?
e = Matrix([[1,-3,sqrt(2)],[-3,1,-1*sqrt(2)],[sqrt(2),-1*sqrt(2),4]])
n = Matrix([1/2,-1/2,1/sqrt(2)])
e_n = e*n
e_n_normal = e_n.dot(n)
#test = Transpose(n)*e*n
print("Normal Strain in the given direction n = " + str(e_n_normal))
shear_strain = e_n - e_n_normal*n
shear_strain_norm = shear_strain.norm()
print("Shear Strain along the given direction n = " + str(pretty(shear_strain_norm)))
```

Normal Strain in the given direction n = 6.000000000000000
Shear Strain along the given direction n = $\sqrt{5189}$

In [38]:

```
%%latex
Q12: A displacement field is given by  $u_1 = 3x_1x_2^2$ ,  $u_2 = 2x_3x_1$ , and  $u_3 = x_1^2x_2 - x_1x_2^2$ . Determine the strain tensor  $\epsilon$ .
```

Q12: A displacement field is given by $u_1 = 3x_1x_2^2$, $u_2 = 2x_3x_1$, and $u_3 = x_1^2x_2 - x_1x_2^2$. Determine the strain tensor ϵ .

In [39]:

```
x1,x2,x3 = symbols('x1 x2 x3')
u = Matrix([3*x1*x2**2,2*x3*x1,x3**2 - x1*x2])
# normal strains
exx = diff(u[0],x1)
eyy = diff(u[1],x2)
ezz = diff(u[2],x3)
# shear strains
exy = (1/2)*(diff(u[0],x2) + diff(u[1],x1))
exz = (1/2)*(diff(u[0],x3) + diff(u[2],x1))
eyz = (1/2)*(diff(u[1],x3) + diff(u[2],x2))
# assembly of the strain tensor
e = Matrix([[exx,exy,exz],[exy,eyy,eyz],[exz,eyz,ezz]])
print("Strain Tensor e:")
e
```

Strain Tensor e:

Out[39]:
$$\begin{pmatrix} 3x_2^2 & 3.0x_1x_2 + 1.0x_3 & -0.5x_2 \\ 3.0x_1x_2 + 1.0x_3 & 0 & 0.5x_1 \\ -0.5x_2 & 0.5x_1 & 2x_3 \end{pmatrix}$$

In [40]: `%%latex`
 Q13: A displacement field is given by $x_1 = X_1 + AX_3$, $x_2 = X_2$, and $x_3 = X_3 - AX_1$. Calculate the volume change ΔV and show that $\Delta V \approx 0$ if A is a very small constant.

Q13: A displacement field is given by $x_1 = X_1 + AX_3$, $x_2 = X_2$, and $x_3 = X_3 - AX_1$. Calculate the volume change ΔV and show that $\Delta V \approx 0$ if A is a very small constant.

In [41]: `%%latex`
 Note: as given in class, one way to look at volume change is the following:

$$\Delta V = \frac{V_{\text{def}} - V_{\text{init}}}{V_{\text{init}}} = \epsilon_{11} + \epsilon_{22} + \epsilon_{33}$$

Note: as given in class, one way to look at volume change is the following: $\Delta V = \frac{V_{\text{def}} - V_{\text{init}}}{V_{\text{init}}} = \epsilon_{11} + \epsilon_{22} + \epsilon_{33}$

In [42]: `%%latex`
 We will instead use the deformation gradient F to calculate the change in volume

$$F_{ij} = \frac{\partial x_i}{\partial X_j}$$

 Following this formula, we find that in this case,

$$F = \begin{pmatrix} 1 & 0 & A \\ 0 & 1 & 0 \\ -A & 0 & 1 \end{pmatrix}$$

We will instead use the deformation gradient F to calculate the change in volume; $F_{ij} = \frac{\partial x_i}{\partial X_j}$ Following this formula, we find that in this case, $F = \begin{pmatrix} 1 & 0 & A \\ 0 & 1 & 0 \\ -A & 0 & 1 \end{pmatrix}$

In [43]: `A = symbols('A')`
`F = Matrix([[1,0,A],[0,1,0],[-A,0,1]])`
the determinant of F, the deformation gradient, is equal to the scale factor of volume change
`J = det(F)`
`print("General formula for the change in volume of the given displacement field: J")`

General formula for the change in volume of the given displacement field:

Out[43]:
$$A^2 + 1$$

In [44]: `%%latex`
 If A is a very small constant, $A \ll 1$, thus we can state

$$A^2 + 1 \approx 1$$

 Thus, as the scale factor of the volume change is 1, we can state that $\Delta V \approx 0$

If A is a very small constant, $A \ll 1$, thus we can state $A^2 + 1 \approx 1$. Thus, as the scale factor of the volume change is 1, we can state that $\Delta V \approx 0$.

In []:

In []: