

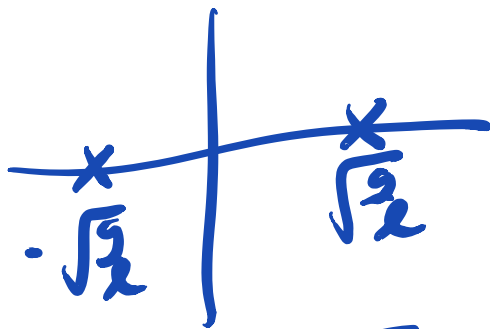
## Summary Lecture 16

- Bode plots for complex conjugate roots
- for asymptotic plots (sketches by hand), assume  $\gamma = 1$
- be aware that error between sketch & exact is unbounded as  $\gamma \rightarrow 0$

## Ch. 5

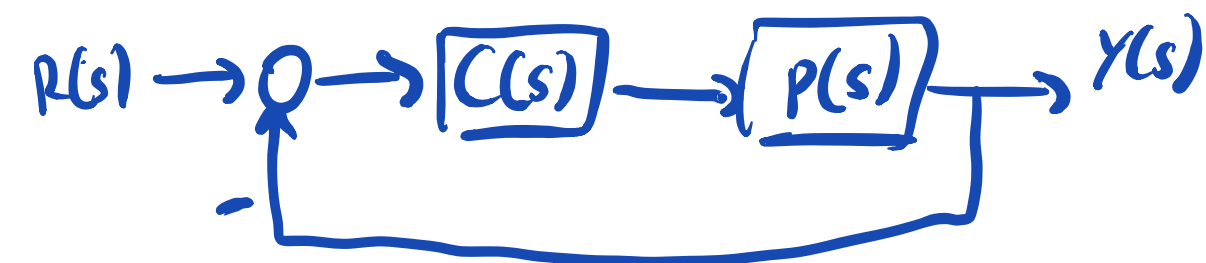
Transfer fun from  $U(s) = \mathcal{L}\{\delta u\}$  to  $Y(s) = \mathcal{L}\{\delta y\}$

$$\frac{Y(s)}{U(s)} = \frac{1}{M\Omega^2} \frac{1}{s^2 - \gamma\Omega^2} =: P(s) \text{ (plant)}$$



Poles of  $P$   $s = \pm \sqrt{\frac{\gamma}{2}}$

Assume  $\frac{\gamma}{2} = 1$ ,  $M\Omega^2 = 1$



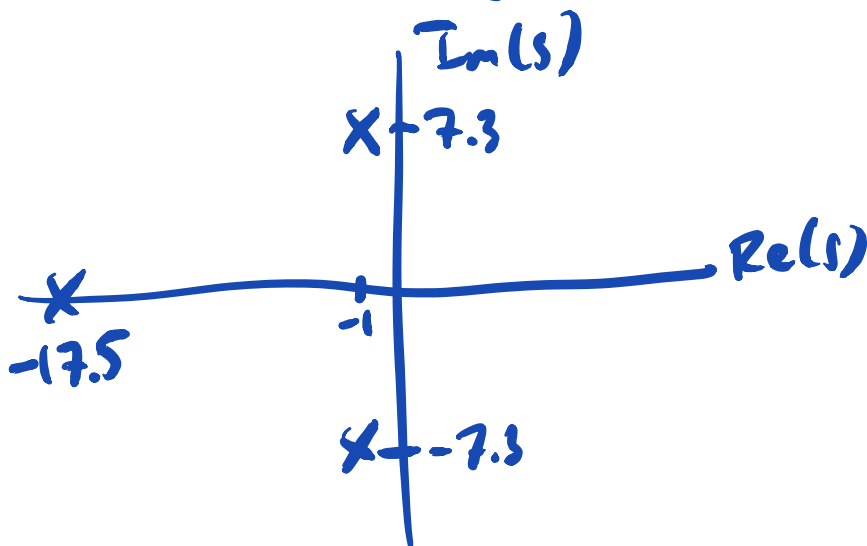
- One controller that stabilizes the upright position is:

$$C(s) = 100 \frac{s+10}{s+20} \quad (\text{"lead controller"})$$

- With this choice of  $C$ , the TF from  $R$  to  $Y$  is

$$\begin{aligned} \frac{Y(s)}{R(s)} &= \frac{C(s)P(s)}{1 + C(s)P(s)} \\ &= \frac{100s + 1000}{s^3 + 20s^2 + 99s + 180} \end{aligned}$$

Poles:  $\{-17.5, -1 \pm j7.3\}$

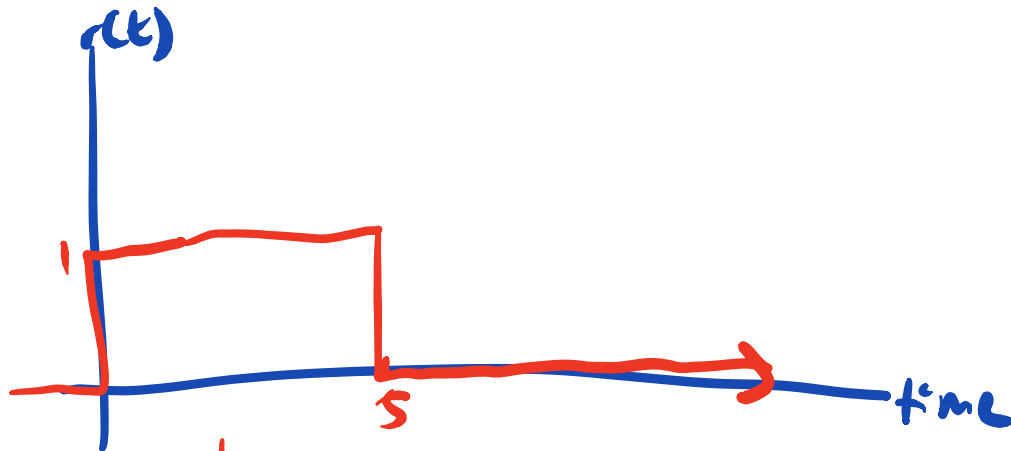


Observations:

- We have stabilized the upright position
- The dominant time-domain response comes from

the 2 poles closest to  $\text{Im}(s)$  axis.

- Based on the 2 dominant poles, we'd expect a relatively large settling time; a lot of overshoot; oscillations at around 7.3 rad/s



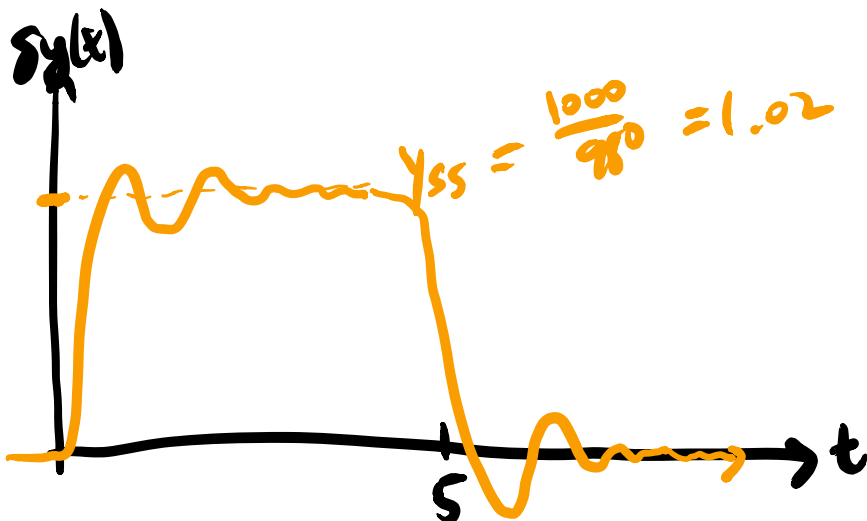
$\delta y = 1$

$0 \leq t \leq 5$

$t \geq 5$

$t \geq 5$

- move pendulum to  $\delta y = 1$  ( $57^\circ$ ) for  $0 \leq t \leq 5$  seconds then back to upright  $\delta y = 0$  for  $t \geq 5$



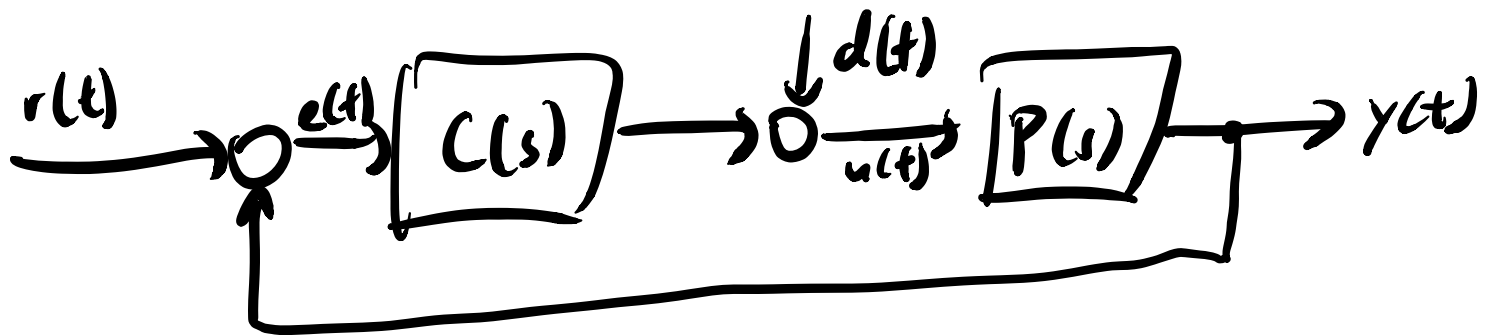
$$\frac{Y(s)}{R(s)} = \frac{100s + 1000}{s^3 + 20s^2 + 99s + 980}$$

To implement  $C(s)$  on a computer, we discretize the ODE relating  $e(t) = r(t) - y(t)$  to  $u(t)$

$$u[k] = \frac{1}{2+20T} (120T - 2)u[k-1] + 100(2+10T)e[k] + 100(10T-2)e[k-1]$$

(difference equation)

## 5.2 Stability of Feedback systems

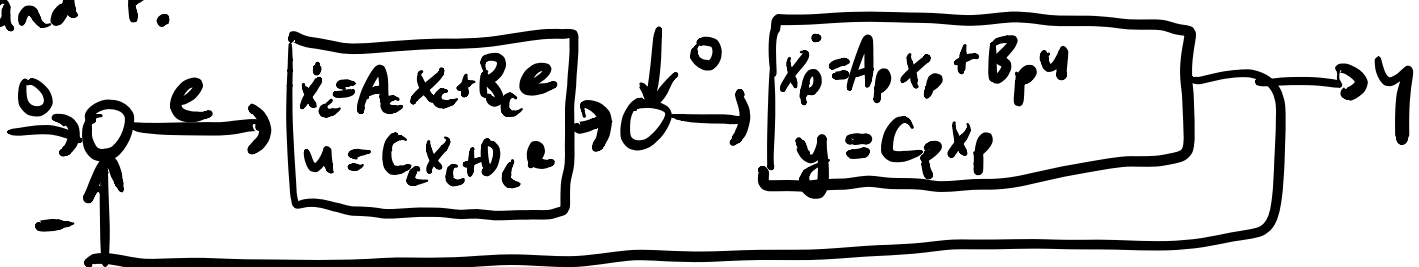


What does it mean for this system to be stable?

Assume:  $P$  and  $C$  are rational;  $P$  is strictly proper,  $C$  is proper.

### 5.2.1. Internal stability

Set  $r(t) = d(t) = 0$ . Bring in state models for  $C$  and  $P$ .



# Closed loop state model

$$\dot{x}_c = A_c x_c + B_c e$$

$$\dot{x}_p = A_p x_p + B_p u$$

$$u = C_c x_c + D_c e, \quad e = -C_p x_p$$

$$\begin{bmatrix} \dot{x}_p \\ \dot{x}_c \end{bmatrix} = \begin{bmatrix} A_p - B_p D_c C_p & B_p C_c \\ -B_c C_p & A_c \end{bmatrix} \begin{bmatrix} x_p \\ x_c \end{bmatrix} \quad \begin{matrix} \dot{x}_c = A_c x_c \\ y = C_c x_c \end{matrix}$$

## Definition 5.2.2

The closed-loop system is internally stable if

$\dot{x}_c = A_c x_c$  is asymptotically stable.

e.g.  $\dot{x}_c = -15x_c + e$        $\dot{x}_p = \begin{bmatrix} 0 & 1 \\ 0 & -5 \end{bmatrix} x_p + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$   
 $u = -1000x_c + 100e$        $y = \begin{bmatrix} 1 & 0 \end{bmatrix} x_p$

$$x_{cl} = \begin{bmatrix} x_p \\ x_c \end{bmatrix}, \quad A_{cl} = \left[ \begin{array}{c|c} \begin{bmatrix} 0 & 1 \\ 0 & -5 \end{bmatrix} - \begin{bmatrix} 0 \\ 1 \end{bmatrix} (100) \begin{bmatrix} 1 & 0 \end{bmatrix} & \begin{bmatrix} 0 \\ 1 \end{bmatrix} (-1000) \\ \hline -\begin{bmatrix} 1 & 0 \end{bmatrix} & -15 \end{array} \right]$$