

## Lecture 20 Summary

- Routh-Hurwitz examples
- Steady-state specs



$$y(s) = \frac{PC}{1+PC} R(s) + \frac{P}{1+PC} D(s)$$

We want, as  $t \rightarrow \infty$ ,  $y(t) \rightarrow r(t)$  despite  $d(t)$

More generally, if  $C(s)$  provides I.O. stability, and  $R(s)$  is a step, then the FVT applies.

$$\begin{aligned} e_{ss} &= \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} sE(s) \\ &= \lim_{s \rightarrow 0} \frac{s}{1+CP} \frac{r_0}{s} \\ &= \lim_{s \rightarrow 0} \frac{r_0}{1+C(s)P(s)} \end{aligned}$$

$$\text{So } e_{ss} = 0 \iff \lim_{s \rightarrow 0} (C(s)P(s)) = \infty \quad (\text{CP has pole at origin})$$

Conclusion: Integral control is fundamental to perfect step tracking.

If  $P(s)$  already has a pole at  $s=0$ , then

$C(s)$  just has to stabilize the loop to get perfect step tracking.

Otherwise, it is common to pick  $C(s) = \frac{1}{s} C_1(s)$  and design  $C_1(s)$  to get I.O. stability.

### Internal model principle

#### Theorem 5.4.3

Assume  $P$  is strictly proper,  $C(s)$  proper, and the feedback loop is I.O. stable. If  $C(s)P(s)$  contains an internal model of the unstable part of  $R(s)$ , then perfect asymptotic tracking occurs.

Say  $\{r(t)\} = R(s) = \frac{N_r(s)}{D_r(s)} = \frac{N_r(s)}{D_r^-(s) D_r^+(s)}$

roots of  $D_r^-$  have  $\text{Re}(s) < 0$ . Roots of  $D_r^+(s)$  have  $\text{Re}(s) \geq 0$ .

The IMP says that if

$$C(s)P(s) = \frac{N(s)}{D(s) D_r^+(s)} \quad \text{and we have}$$

I.O. stability, then  $\text{ess} = 0$ .

To apply FVT,  $sE(s)$  can't have poles in  $\overline{\mathbb{C}^+}$ .

$$sE(s) = s \frac{1}{1+PC} R(s) = \frac{sD(s) D_r^+(s) N_r(s)}{\underbrace{\pi(s) D_r^-(s) D_r^+(s)}_{\text{poles in } \mathfrak{C}^-}}$$

So FVT applies and  $\text{ess} = 0$ , as expected.

e.g. 5.4.3.  $P(s) = \frac{1}{s+1}$   $r(t) = r_0 \sin(t)$ ,  $r_0 \in \mathbb{R}$   
constant

$$\mathcal{Z}\{r(t)\} = \frac{r_0}{s^2 + 1} \Rightarrow D_r^+(s) = s^2 + 1 \quad D_r^-(s) = 1$$

This suggests that the controller  $C(s) = \frac{1}{D_r^+(s)} C_1(s)$

$$= \frac{1}{s^2 + 1} C_1(s).$$

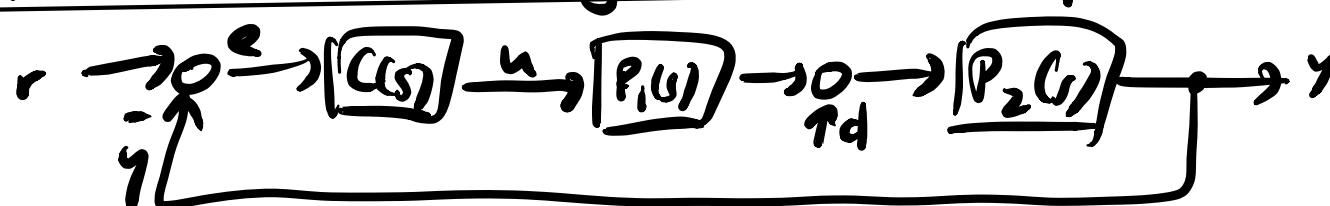
$C_1$  must be chosen to ensure I.O. stability.

Let's keep it simple, try  $C_1(s) = a_2 s^2 + a_1 s + a_0$ .

$$\begin{aligned} \pi(s) &= (s+1)(s^2+1) + a_2 s^2 + a_1 s + a_0 \\ &= s^3 + (a_2+1)s^2 + (a_1+1)s + (a_0+1) \end{aligned}$$

You can check that  $a_2 = a_0 = 0$ ,  $a_1 = 1$  works.  $\Delta$

### 5.4.2. Disturbance Rejection (Steady State)



Generalized Disturbance model

$P_1(s) = 1 \Rightarrow$  input disturbance

$P_2(s) = 1 \Rightarrow$  output disturbance

$$Y(s) = P_2(s)(D(s) + P_1(s)U(s)) \quad \text{and}$$

$$\frac{Y(s)}{D(s)} = \frac{P_2(s)}{1 + C(s)P_1(s)P_2(s)} \quad (\text{verify!})$$

Best case scenario:  $\lim_{t \rightarrow \infty} y(t) = 0$

(disturbance has no lasting effects on the output)

$$\text{Write } \mathcal{L}\{d(t)\} = D(s) = \frac{N_d(s)}{D_d^-(s) D_d^+(s)}$$

Assume  $P_1, P_2$  strictly proper,  $C$  proper, and I.O. stability.

Let's see when FVT will apply:

$$y_{ss} = \lim_{t \rightarrow \infty} y(t) \stackrel{?}{=} \lim_{s \rightarrow 0} s Y(s)$$

$$= \lim_{s \rightarrow 0} s \frac{P_2}{1 + CP_1P_2} \frac{N_d}{D_d^- D_d^+}$$

$$= \lim_{s \rightarrow 0} \underbrace{\frac{s D_c D_{P_1} N_{P_2}}{D_c D_p, D_p + N_c N_{P_1} N_{P_2}}}_{\text{Hurwitz by assumption}} \cdot \frac{N_d}{D_d^+ D_d^t}$$

Hurwitz by assumption

This suggests, to reject disturbances reliably, the controller should contain an internal model  $D_d^+(s)$ .

$$C(s) = \frac{1}{D_d^+(s)} C_i(s)$$

$\hookrightarrow$  chosen for  
internal model      feedback stability