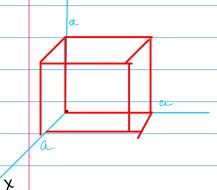
Example, uniform solid cube

$$f = \frac{M}{a^3}$$

We want the moment of inertia tensor around the fixed point (a/2, a/2, a/2)



$$I_{ij} = \int \int [r^{2} \xi_{ij} - \chi_{i} \chi_{i}] dV$$

$$\int I_{xx} = \int \int dx dy dz \int [(\chi^{2} + \chi^{2} + z^{2}) - \chi^{2}]$$

$$= ga \int \int dy dz (y^2 + z^2)$$

$$=25a^{2}+\frac{1}{3}a^{3}+\frac{1}{6}Ma^{2}$$

$$I_{yy} = \iiint_{-a_{12}} dx dy dz \left[ x^{2} + y^{2} + z^{2} - y^{2} \right] f = \iint_{-a_{12}} dx dz \left( x^{2} + z^{2} \right) f a$$

$$= \frac{1}{6}M\alpha^2 = \sum_{z=0}^{\infty}$$

off diagonal terms,

$$I = \frac{1}{6}Ma^2 \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix}$$

Moment of inertia around x-axis, through the center.

$$T_{S}^{X} = \hat{N} \cdot T \cdot \hat{N} = \frac{1}{6} Na^{2} (100) \left( \frac{100}{000} \right) \left( \frac{100}{000$$

Moment of inertia tensor around the fixed point (0,0,0)

$$\bar{J}_{xy} = -\beta \int \int dx dy dz (xy) = -\beta \alpha \frac{\alpha^4}{4} = -\frac{1}{2} M \alpha^2 = \bar{J}_{yx}$$

$$= \bar{J}_{ij} (i \neq j)$$

$$T = Ma^{2} \begin{pmatrix} 21_{3} & -1/4 & -1/4 \\ -1/4 & 21_{3} & -1/4 \end{pmatrix}$$

Moment of inertia around x-axis, through the corner

$$I_s^{\prime} = \hat{n} \cdot \hat{I} \cdot \hat{n} = Ma^2(100) \begin{pmatrix} b \\ b \\ 0 \end{pmatrix}$$

$$= Ma^{2} (100) \begin{pmatrix} 2/3 \\ -1/4 \\ -1/4 \end{pmatrix} = \frac{2}{3} Ma^{2}$$

$$\overline{L} = \overline{L} = \begin{pmatrix} 2/3 & -1/4 & -1/4 \\ -1/4 & 2/3 & -1/4 \end{pmatrix} = \begin{pmatrix} 2/3 \\ 0 \\ -1/4 & 2/3 \end{pmatrix} = Maw \begin{pmatrix} 2/3 \\ -1/4 \\ -1/4 \end{pmatrix}$$

ang mom does not point in the direction of omega

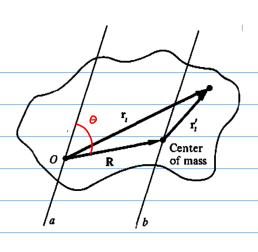
note: if we rotate on the diagonal (1,1,1), L will point in the direction of omega

## Parallel axis theorem

Lets relate moments of inertia between parallel axes of rotation, one of which is through the CM

$$T_s = m_i (r_i \times \hat{n})^2$$

$$r_i = R + r_i$$



$$\frac{1}{s} = m \cdot \left[ \left( \overline{R} + \overline{r} \cdot ' \right) \times \widehat{n} \right]^{\frac{1}{s}}$$

$$= \sum_{m} \cdot \left( \overline{R} \times \widehat{n} \right)^{2} + m \cdot \left( \overline{r} \cdot ' \times \widehat{n} \right) + 2 m \cdot \left( \overline{R} \times \widehat{n} \right) \cdot \left( \overline{r} \cdot ' \times \widehat{n} \right)$$

$$\frac{1}{s} = M \cdot \left( \overline{R} \times \widehat{n} \right)^{2} + \overline{1} \cdot S \qquad -2 \cdot \left( \overline{R} \times \widehat{n} \right) \cdot \left( \widehat{n} \times m \cdot \overline{r} \cdot ' \right)$$

$$\frac{1}{s} = M \cdot \left( \overline{R} \times \widehat{n} \right)^{2} + \overline{1} \cdot S \qquad -2 \cdot \left( \overline{R} \times \widehat{n} \right) \cdot \left( \overline{n} \times m \cdot \overline{r} \cdot ' \right)$$

$$\frac{1}{s} = M \cdot \left( \overline{R} \times \widehat{n} \right)^{2} + \overline{1} \cdot S \qquad -2 \cdot \left( \overline{R} \times \widehat{n} \right) \cdot \left( \overline{n} \times m \cdot \overline{r} \cdot ' \right)$$

$$|Rxn| = Rsin\theta$$
 perp distance between axes 'a' and 'b'

Moment of inertia about a given axis is the moment of inertia about a parallel axis through the CM plus the moment of inertia of the body, as if concentrated at the CM, with respect to the original axis.

## Principal Axes of Inertia

For the cases that L and omega are parallel we this axis a "principal axis".

$$\bar{a} = \lambda \bar{b}$$
 lambda is a real number

$$\frac{1}{1} = \lambda \omega$$
 lambdas are related to moments of inertia

If we find axes through which I is diagonal, these axes will be principal axes.

If we have principal axes, then the matrix of I will be diagonal.

There always exist 3 perpendicular axes through a point O for any rigid body at which I is diagonal, and L points along omega, when omega points in the direction of a principal axis.

The proof lies in the fact that we can always find a transformation of a real symmetric matrix into a diagonal.

$$\begin{array}{ccc}
\overline{L} = \overline{I} \overline{W} & L_1 = \overline{L}_1 W_1 \\
L_2 = \overline{I}_2 W_2 \\
L_3 = \overline{L}_1 W_3
\end{array}$$

Finding the Principal Axes

We want 
$$\overline{L} = \overline{L} \overline{\omega} = \lambda \overline{\omega} \implies (\overline{L} - \lambda) \overline{\omega} = 0$$

We just need to solve this eigenvalue equation.

There is a non-zero solution only if the determinant of the matrix on the lhs is zero.

characteristic equation 
$$\left| \begin{array}{cc} \mathbf{I} - \mathbf{\lambda} \end{array} \right| = 0$$

roots of this cubic equation will be the "principal moments". We can solve for the principal axes by using the eqn above

Example: cube wrt point (0,0,0). Find the principal moments and axes.

$$T = Ma^{2} \begin{pmatrix} 21_{3} & -1/4 & -1/4 \\ -1/4 & 21_{3} & -1/4 \end{pmatrix} = \frac{1}{12} Ma^{2} \begin{pmatrix} 8 - 3 - 3 \\ -3 & 8 - 3 \end{pmatrix}$$

$$\frac{1}{12} \begin{pmatrix} -1/4 & -1/4 & 21_{3} \\ -1/4 & -1/4 & 21_{3} \end{pmatrix} = \frac{1}{12} Ma^{2} \begin{pmatrix} 8 - 3 - 3 \\ -3 & 8 - 3 \end{pmatrix}$$

$$(8c-7) \left[ (8c-1)^{2} - 9c^{2} + 3c \left[ -3c(8c-1) - 9c^{2} \right] - 3c \left[ 9c^{2} + 3c(8c-1) \right] = 0$$

$$(2c-1) \left( 11c - 1 \right) = 0$$

$$e_2 = \frac{1}{\sqrt{6}} \begin{pmatrix} -1 \\ -1 \\ 2 \end{pmatrix}$$

$$e_3 = \frac{1}{\sqrt{2}} \left( \frac{1}{-1} \right)$$
  $e_i \cdot \hat{e}_j = \delta_{ij}$ 

The similarity transformation that will transform the coordinate system from the original to the final (in which the axes are principal axes) is given by:

$$T' = s^{-1}TS$$

$$S = \begin{pmatrix} 1 & -1 & 1 \\ 1 & -1 & -1 \\ 1 & 2 & 5 \end{pmatrix}$$

$$\frac{1}{1} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix}$$