

# Mode separation in a mixed-mode I/II problem by the J-integral and higher-order theories

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Motivation

Semi-layerwise  
beam  
model

$J$ -Integral

Built-in  
config.

Mode-  
mixture

## ① Motivation

## ② Semi-layerwise beam model

## ③ $J$ -Integral

## ④ Built-in config.

## ⑤ Mode-mixture

# Composite materials

## Failure mechanism of composite materials

Motivation

Semi-layerwise beam model

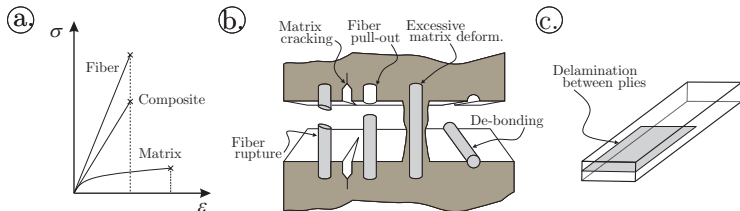
J-Integral

Built-in config.

Mode-mixity

Fiber reinforced thermoset polymer composites:

- multicomponent and multiphase materials
- high strength, high modulus reinforcement phase
- tough but stiff matrix phase



**Figure 1:** Tensile test of polymer composite and its components (a). Possible failure mechanism of composite materials (b-c).

Definition of the energy release rate:

$$G = -\frac{d\Pi}{dA}, \quad (1)$$

$$\Pi = U - W. \quad (2)$$

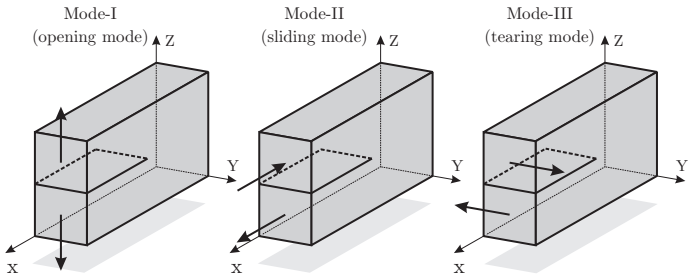


Figure 2: Basic fracture modes in linear elastic fracture mechanics.

In the case of beam-type specimens:

$$G_T = G_I + G_{II}. \quad (3)$$

# Semi-layerwise beam model

Displacement field of the undelaminated region

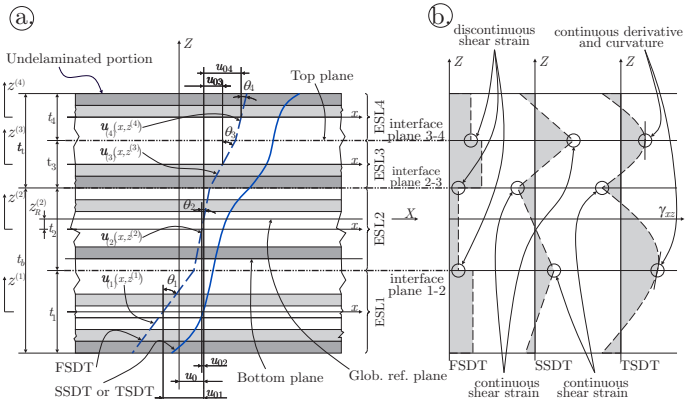


Figure 3: Cross section and the assumed deformation of the undelaminated portion in the  $X-Z$  plane (a) and distributions of the transverse shear strain by different theories (b) using 4ESLs.

$$u_{(i)}(x, z^{(i)}) = u_0(x) + u_{0i}(x) + \theta_i(x)z^{(i)} + \phi_i(x)[z^{(i)}]^2 + \lambda_i(x)[z^{(i)}]^3 \quad i = 1..4. \quad (4)$$

# Definition of the stress resultants

Undelaminated region

Invariant form of the displacement fields:

$$u_{(i)} = u_0 + \left( K_{ij}^{(0)} + K_{ij}^{(1)} z^{(i)} + K_{ij}^{(2)} [z^{(i)}]^2 + K_{ij}^{(3)} [z^{(i)}]^3 \right) \psi_j, \quad i = 1..4, \quad (5)$$

$$w_{(i)} = w(x), \quad i = 1..4.$$

Strain field:

$$\varepsilon_{x(i)} = \frac{\partial u_{(i)}}{\partial x}, \quad \gamma_{xz(i)} = \frac{\partial u_{(i)}}{\partial z^{(i)}} + \frac{\partial w_{(i)}}{\partial x}. \quad (6)$$

Constitutive equations:

$$\begin{pmatrix} \sigma_x \\ \tau_{xz} \end{pmatrix}_{(i)} = \bar{\mathbf{C}}_{(i)}^{(m)} \begin{pmatrix} \varepsilon_x \\ \gamma_{xz} \end{pmatrix}_{(i)} = \begin{bmatrix} E_{11}/(1 - \nu_{21}\nu_{12}) & 0 \\ 0 & G_{13} \end{bmatrix}_{(i)}^{(m)} \begin{pmatrix} \varepsilon_x \\ \gamma_{xz} \end{pmatrix}_{(i)}. \quad (7)$$

Stress resultants:

$$\begin{pmatrix} N_x \\ M_x \\ L_x \\ P_x \end{pmatrix}_{(i)} = \int_0^b \int_{-t_i/2}^{t_i/2} \sigma_x \begin{pmatrix} 1 \\ z \\ z^2 \\ z^3 \end{pmatrix} dz^{(i)} dy, \quad i = 1..4, \quad (8)$$

$$\begin{pmatrix} Q_{xz} \\ R_{xz} \\ S_{xz} \end{pmatrix}_{(i)} = \int_0^b \int_{-t_i/2}^{t_i/2} \tau_{xz} \begin{pmatrix} 1 \\ z \\ z^2 \end{pmatrix} dz^{(i)} dy, \quad i = 1..4. \quad (9)$$

# Equilibrium equations

Undelaminated region

Virtual work principle:

$$\int_{T_0}^{T_1} (\delta U - \delta W_F) dt = 0, \quad \delta U = \sum_i \delta U_{(i)}, \quad \delta W_F = \sum_i \delta W_{F(i)}. \quad (10)$$

Equilibrium equations of the undelaminated region:

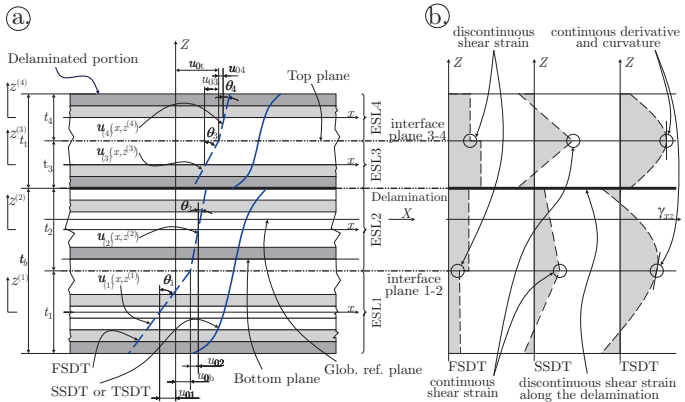
$$\delta u_0 : \sum_{i=1}^4 \left( \frac{\partial N_{x(i)}}{\partial x} \right) = 0, \quad (11)$$

$$\delta \psi_j : \sum_{i=1}^4 \left( K_{ij}^{(0)} \frac{\partial N_{x(i)}}{\partial x} + K_{ij}^{(1)} \frac{\partial M_{x(i)}}{\partial x} + K_{ij}^{(2)} \frac{\partial L_{x(i)}}{\partial x} + K_{ij}^{(3)} \frac{\partial P_{x(i)}}{\partial x} - \right. \\ \left. K_{ij}^{(1)} Q_{x(i)} - 2K_{ij}^{(2)} R_{x(i)} - 3K_{ij}^{(3)} S_{x(i)} \right) = 0, \quad j = 1..p, \quad (12)$$

$$\delta w : \sum_{i=1}^4 \left( \frac{\partial Q_{x(i)}}{\partial x} \right) + q = 0. \quad (13)$$

# Semi-layerwise beam model

Delaminated region



**Figure 4:** Cross section and the assumed deformation of the delaminated portion in the  $X - Z$  plane (a) and distributions of the transverse shear strain by different theories (b) using separately 2ESLs.

$$u_{(i)}(x, z^{(i)}) = u_{0b}(x) + u_{0i}(x) + \theta_i(x)z^{(i)} + \phi_i(x)[z^{(i)}]^2 + \lambda_i(x)[z^{(i)}]^3 \quad i = 1..2, \quad (14)$$

$$u_{(i)}(x, z^{(i)}) = u_{0f}(x) + u_{0i}(x) + \theta_i(x)z^{(i)} + \phi_i(x)[z^{(i)}]^2 + \lambda_i(x)[z^{(i)}]^3 \quad i = 3..4. \quad (15)$$



# Equilibrium equations

Delaminated portion

Invariant form of the displacement fields:

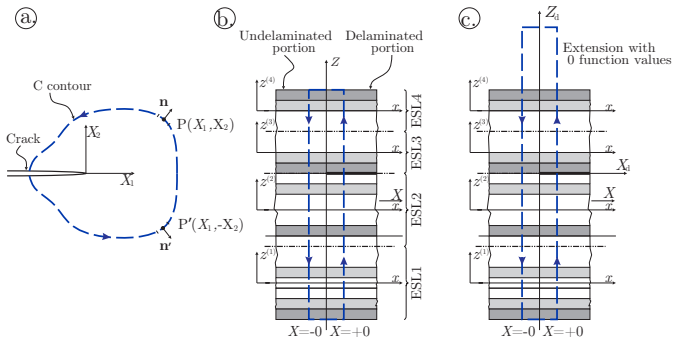
$$\begin{aligned} u_{(i)} &= u_{0b} + \left( K_{ij}^{(0)} + K_{ij}^{(1)} z^{(i)} + K_{ij}^{(2)} [z^{(i)}]^2 + K_{ij}^{(3)} [z^{(i)}]^3 \right) \psi_j, & i = 1..2, \\ u_{(i)} &= u_{0t} + \left( K_{ij}^{(0)} + K_{ij}^{(1)} z^{(i)} + K_{ij}^{(2)} [z^{(i)}]^2 + K_{ij}^{(3)} [z^{(i)}]^3 \right) \psi_j, & i = 3..4, \\ w_{(i)} &= w_b(x), & i = 1..2, \\ w_{(i)} &= w_t(x), & i = 3..4. \end{aligned} \quad (16)$$

Equilibrium equations of the delaminated portions:

$$\delta u_{0b} : \sum_{i=1}^2 \left( \frac{\partial N_{x(i)}}{\partial x} \right) = 0, \quad \delta u_{0t} : \sum_{i=3}^4 \left( \frac{\partial N_{x(i)}}{\partial x} \right) = 0, \quad (17)$$

$$\begin{aligned} \delta \psi_j : \sum_{i=1}^4 \left( K_{ij}^{(0)} \frac{\partial N_{x(i)}}{\partial x} + K_{ij}^{(1)} \frac{\partial M_{x(i)}}{\partial x} + K_{ij}^{(2)} \frac{\partial L_{x(i)}}{\partial x} + K_{ij}^{(3)} \frac{\partial P_{x(i)}}{\partial x} - \right. \\ \left. K_{ij}^{(1)} Q_{x(i)} - 2K_{ij}^{(2)} R_{x(i)} - 3K_{ij}^{(3)} S_{x(i)} \right) = 0, \quad j = 1..p, \end{aligned} \quad (18)$$

$$\delta v_b : \sum_{i=1}^2 \left( \frac{\partial Q_{x(i)}}{\partial x} \right) + q_b = 0, \quad \delta v_t : \sum_{i=3}^4 \left( \frac{\partial Q_{x(i)}}{\partial x} \right) + q_t = 0. \quad (19)$$



**Figure 5:** The general definition of the  $J$ -integral for in-plane problem (a). The application of the  $J$ -integral for semi-layerwise model using zero-area path (b) with the inevitable extension (c).

Definition of the  $J$ -integral:

$$J = G_T = \int_C \left\{ U n_1 - \sigma_{ij} n_j \frac{\partial u_i}{\partial x_1} \right\} ds. \quad (20)$$

Beam-type fracture sepecimens:

$$G_T = G_I + G_{II}. \quad (21)$$

Mode-I energy release rate:

$$G_I = \int_{-t}^{+t} \left\{ \left( -\frac{1}{2} \sigma_{x(sym)} \varepsilon_{x(sym)} + \tau_{xz(ant)} \left( \frac{1}{2} \gamma_{xz(ant)} - \frac{\partial w_{(ant)}}{\partial x} \right) \right) \right|_{x=-0}^{(undel)} + \left( \frac{1}{2} \sigma_{x(sym)} \varepsilon_{x(sym)} - \tau_{xz(ant)} \left( \frac{1}{2} \gamma_{xz(ant)} - \frac{\partial w_{(ant)}}{\partial x} \right) \right) \right|_{x=+0}^{(del)} \right\} dz_d. \quad (22)$$

Mode-II energy release rate:

$$G_{II} = \int_{-t}^{+t} \left\{ \left( -\frac{1}{2} \sigma_{x(ant)} \varepsilon_{x(ant)} + \tau_{xz(sym)} \left( \frac{1}{2} \gamma_{xz(sym)} - \frac{\partial w_{(sym)}}{\partial x} \right) \right) \right|_{x=-0}^{(undel)} + \left( \frac{1}{2} \sigma_{x(ant)} \varepsilon_{x(ant)} - \tau_{xz(sym)} \left( \frac{1}{2} \gamma_{xz(sym)} - \frac{\partial w_{(sym)}}{\partial x} \right) \right) \right|_{x=+0}^{(del)} \right\} dz_d. \quad (23)$$

Motivation

Semi-layerwise beam model

J-Integral

Built-in config.

Mode-mixity

## Built-in config. Transversely isotropic material

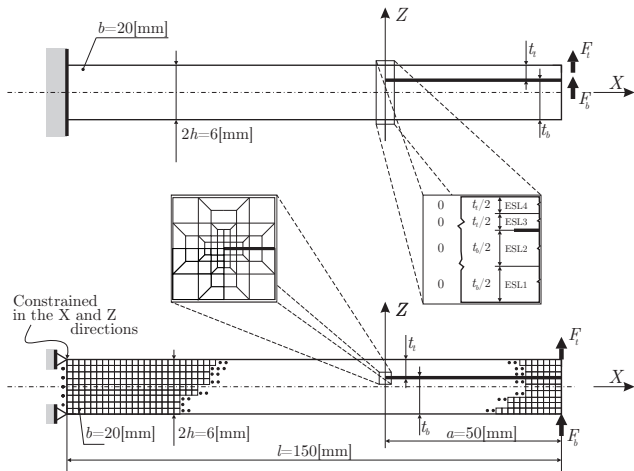
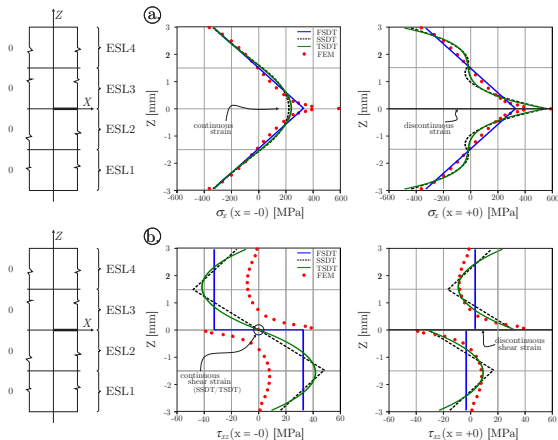


Figure 6: Built-in configuration of delaminated beam with transversely isotropic plies.

# Transversely isotropic material

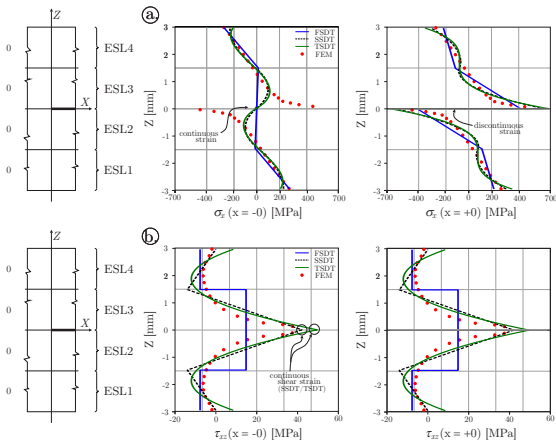
Symmetric delamination:  $F_t = 200$  N and  $F_b = -200$  N.



**Figure 7:** Distribution of the  $\sigma_x$  normal stresses (a) and the  $\tau_{xz}$  shear stresses at the delamination tip, transversely isotropic case,  $F_t = 200$  N and  $F_b = -200$  N.

# Transversely isotropic material

Symmetric delamination:  $F_t=200$  N and  $F_b=200$  N



**Figure 8:** Distribution of the  $\sigma_x$  normal stresses (a) and the  $\tau_{xz}$  shear stresses at the delamination tip, transversely isotropic case,  $F_t = 200$  N and  $F_b = 200$  N.

# Energy release rates

## Mode-mixity of transversely isotropic beams

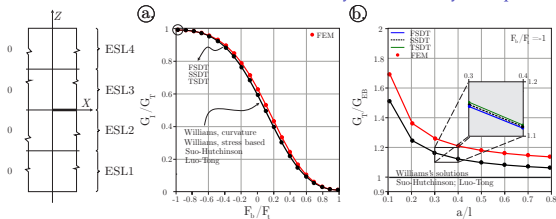


Figure 9: Mode mixity of symmetrically delaminated beam in different loading scenarios (a),  $a/l = 1/3, 2h = 6 \text{ mm}$ . The ratio of the  $G_T$  total and  $G_{EB}$  classical energy release rates (b),  $F_b/F_t = -1$ .

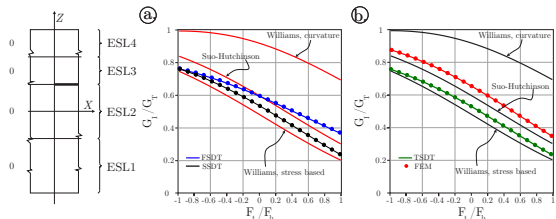


Figure 10: Mode mixity of asymmetrically delaminated beam in different loading scenarios using analytical (a) and numerical (b) solutions,  $t_t/t_b = 0.5, a/l = 1/3, 2h = 6 \text{ mm}$ .

## Built-in config.

Geometry and different delamination scenarios

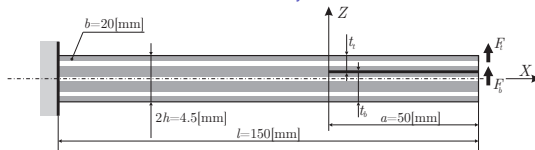


Figure 11: Built-in configuration of delaminated beam with orthotropic plies.

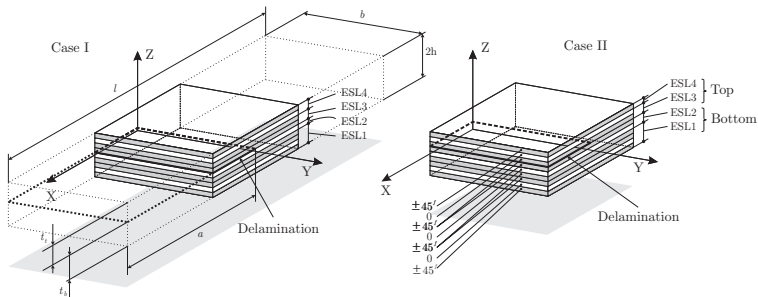


Figure 12: Different delamination scenarios of a beam element.



## Results - Displacement and stress

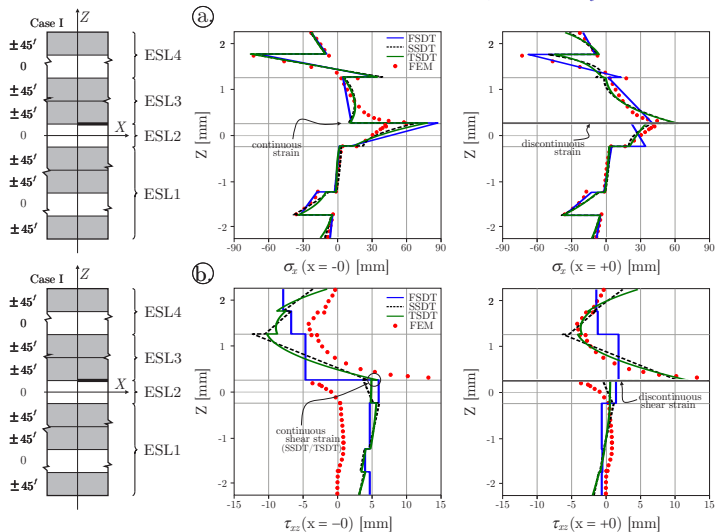
"Case I",  $F_t = 10$  N and  $F_b = -10$  N

Figure 13: Distribution of the  $\sigma_x$  normal stresses (a) and the  $\tau_{xz}$  shear stresses at the delamination tip, "Case I",  $F_t = 10$  N and  $F_b = -10$  N.

# Energy release rates

Mode mixity

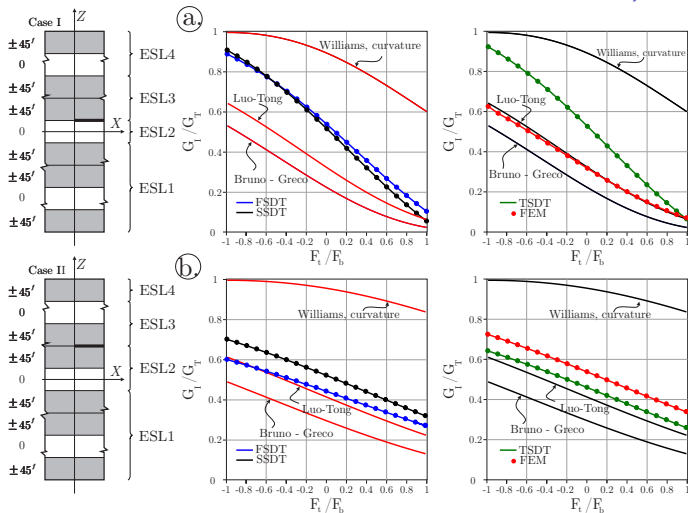


Figure 14: Mode mixity of different loading scenarios with different evaluation techniques. The delamination is located between 0 -  $\pm 45^\circ$  layers (a) and between  $\pm 45^\circ$  -  $\pm 45^\circ$  layers (b),  $a/l = 1/3$ ,  $b = 20$  mm,  $2h = 4.5$  mm.

# Summary

Motivation

Semi-layerwise beam model

$J$ -Integral

Built-in config.

Mode-mixity

- Semi-layerwise beam model
- Application of the  $J$ -integral
- More exact mode partitioning

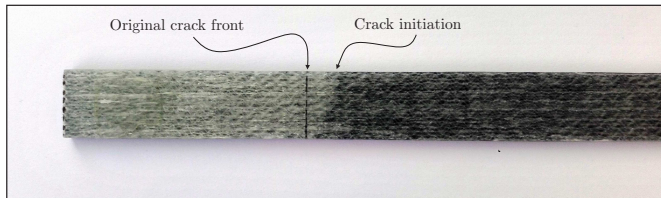


Figure 15: Straight crack front and the occurred crack initiation in the of bi-material specimen.

Motivation

Semi-  
layerwise  
beam  
model

$J$ -Integral

Built-in  
config.

Mode-  
mixity

# Thank you for your attention!

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