Homework Set 2

Due date: February 5, 2016 after class

Problem 1: Consider a plane wall with a thickness of $l_w = 0.5$ m and <u>a heat sink</u> that is a function of temperature given as: $\dot{q} = -0.8T \, \text{W/m}^3$, where T is in K. The temperature and heat flux at the inlet to the wall (x = 0) are 500 K and 0.4 kW/m², respectively, and the thermal conductivity is $k_w = 20 \, \text{W/m·K}$. At the exit of the wall, convective heat exchange occurs with the surrounding that are maintained at $T_\infty = 250 \, \text{K}$. Assuming steady, one-dimensional conduction in the wall, constant properties, and negligible radiative heat transfer to the environment, determine the following:

a) The temperature at the exit of the wall (x = L) in °C.

$$k\frac{d^{2}T}{dx} + \dot{q}_{gen} = 0$$

$$\frac{d^{2}T}{dx^{2}} - \frac{0.8 \text{ W} \cdot \text{m}^{-3}}{20 \text{ W/m} \cdot \text{K}}T = 0 = \frac{d^{2}T}{dx^{2}} - \frac{0.8 \text{ W} \cdot \text{m}^{-3}}{20 \text{ W/m} \cdot \text{K}}T = 0$$

$$\frac{d^{2}T}{dx^{2}} - 0.04T = 0$$

$$T = C_{1} \exp(-0.2x) + C_{2} \exp(0.2x)$$

Boundary conditions:

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$$\frac{dT}{dx}\Big|_{x=0} = -\frac{q_{w}^{''}}{k} = -20 \text{ K/m and T(0)=500 K}$$

$$500 = C_{1} + C_{2}$$

$$100 = C_{1} - C_{2}$$

$$C_{1} = 300 \text{K and } C_{2} = 200 \text{K}$$

$$T = 300 \exp(-0.2x) + 200 \exp(0.2x) K$$

$$T(0.5) = 300 \exp(-0.2 \cdot 0.5) + 200 \exp(0.2 \cdot 0.5) K = 492.5 \text{K or } 219.4 ^{\circ}\text{C}$$

OR due to repeating roots

$$T = C_1' \sinh(0.2x) + C_2' \cosh(0.2x)$$

$$T(x = 0) = C_1' \sinh(0) + C_2' \cosh(0) = 500 \text{ K} \Rightarrow C_2' = 500 \text{ K}$$

$$\frac{dT}{dx}\Big|_{x=0} = -\frac{400 \text{ W} \cdot \text{m}^{-2}}{20 \text{W} \cdot \text{m}^{-1} \cdot \text{K}^{-1}} = -20 \text{ K/m} = 0.2 C_1' \cosh(0) + 0.2 \sinh(0) \Rightarrow C_1' = -100$$

$$T(x) = -100 \sinh(0.2x) + 500 \cosh(0.2x)$$

$$T(x) = -100 \sinh(0.2 \times 0.5) + 500 \cosh(0.2 \times 0.5) = 492.5 \text{ K or } 219.4^{\circ}\text{C}$$

b) The convective heat transfer coefficient at the exit to the surroundings in W/m²·K.

$$q''|_{x=0.5\text{m}} = -k \frac{dT}{dx}|_{x=0.5\text{m}} = -20 \frac{\text{W}}{\text{m} \cdot \text{K}} \left[-0.2 \cdot 300 \exp(-0.2 \cdot 0.5) + 0.2 \cdot 200 \exp(0.2 \cdot 0.5) \right]$$

$$q''|_{x=0.5\text{m}} = 201.7 \text{ W} \cdot \text{m}^{-2}$$

$$h = \frac{q''_{\text{w}}}{T(0.5) - T_{\text{m}}} = 0.8316 \frac{\text{W}}{\text{m}^2 \cdot \text{K}}$$

Problem 2: Consider a cylindrical pipe with steam running through it at a temperature of $T_{\rm s}$ = 250 °C and a convective heat transfer coefficient of h = 100 W/m²·K. The inner and outer diameters of the pipe are 0.1m and 0.11 m, respectively. The pipe is covered with 0.05 m thick fiberglass pipe insulation to prevent heat loss, and there is a contact resistance between the insulation and the pipe of R'' = 0.002 m²·K/W. The temperature of the air on the outside of the insulation is T_{∞} = 30 °C and the combined heat transfer coefficient is h = 25 W/m²·K. The thermal conductivities of the pipe and insulation are $k_{\rm pipe}$ =60.5 W/m·K and $k_{\rm ins}$ = 0.038 W/m·K, respectively. Assuming steady, radial conduction, determine the following:

(a) The temperature drop between the pipe and insulation due to contact resistance.

$$\frac{UA}{L} = \left[\left(2\pi h_{i} r_{i} \right)^{-1} + \left(\frac{\ln \left(r_{o} / r_{i} \right)}{2\pi k_{\text{pipe}}} \right) + \frac{R_{c}^{"}}{2\pi r_{o}} + \left(\frac{\ln \left(\left(r_{o} + d \right) / r_{o} \right)}{2\pi k_{\text{ins}}} \right) + \left(2\pi h_{o} \left(r_{o} + d \right) \right)^{-1} \right]^{-1} = 0.360 \text{ W/m} \cdot \text{K}$$

$$q' = \frac{UA}{L} \left[T_{i,\infty} - T_{o,\infty} \right] = 79.19 \text{ W/m}$$

$$\Delta T = \frac{R_{c}^{"} q'}{2\pi r_{o}} = 0.4583 \text{ K}$$

(b) The surface temperature on the inner surface of the pipe in °C.

$$q' = 2\pi r_i h_i \left[T_{i,\infty} - T_1 \right] \Rightarrow T_1 = T_{i,\infty} - \frac{q'}{2\pi r_i h_i} = 247.5$$
°C