A study on the energy transfer of a square prism under fluid-elastic galloping

H.G.K.G. Jayatunga, B.T. Tan, J. S. Leontini

Abstract

Extracting useful energy from flow induced vibrations has become a developing area of research in recent years. In this paper, we analyse power transfer of an elastically mounted body under the influence of fluid-elastic galloping. The system and the power transfer is analysed by numerically integrating the quasi-steady state model equations and direct numerical simulations. The power transfer is analysed for both high (Re = 22300) and low (Re = 200) Reynolds numbers cases.

The linear analysis of the model equation shows that the system is governed by two parameters, namely the combined mass-stiffness parameter Π_1 and the combined mass damping parameter Π_2 . A combined mass-damping coefficient, Π_2 , that can be derived from the equation of motion, is shown to be the parameter that governs power output. The system is a balance between the power delivered to the system due to fluid-dynamic forcing and power removed through mechanical damping which are governed by the fluid-dynamic forcing characteristics (i.e. the lift force as a function of incident angle) and mechanical damping coefficient as represented by Π_2 respectively. Comparing the DNS results with the QSS data uncovered that a good agreement of the data could be obtained even at low Reynolds numbers when the mass-stiffness, Π_1 , is high representing a high mass to stiffness ratio. At low Π_1 , the system shows a significant response to forces associated with shedding and this is shown to suppress the galloping response.

Keywords:

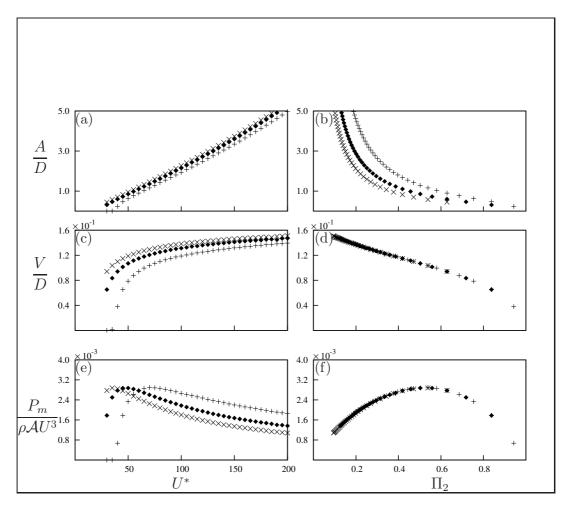


Figure 1: Displacement amplitude, velocity amplitude and mean power data as functions of two different independent varibles. Data presented in (a), (c) and (e) using the classical VIV parameter U^* , obtained at Re=200 and $m^*=20$ at three different damping ratios: $\zeta=0.075~(\times),~\zeta=0.1~(\spadesuit)$ and $\zeta=0.15~(+)$. (b) (d) and (f) are the same data presented using the combined mass-damping parameter (Π_2) as the independent variable.

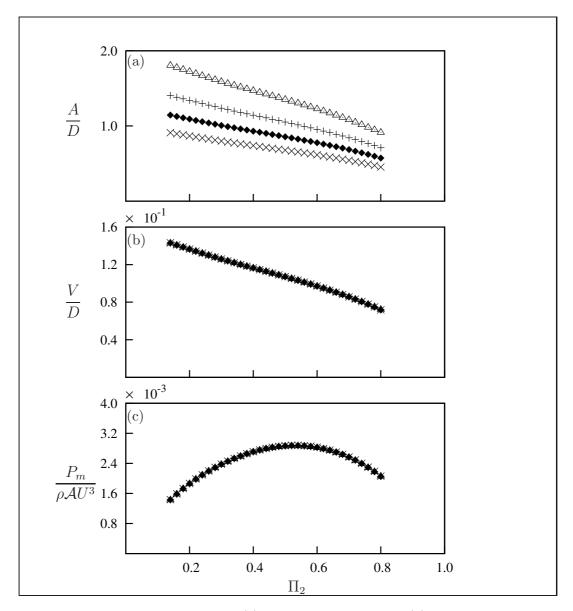


Figure 2: QSS data at high Π_1 levels. (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of Π_2 . Data presented at four different combined mass-stiffness levels. $\Pi_1=10~(m^*=20,~U^*=40)~(\times),~\Pi_1=100~(m^*=80,~U^*=50)~(+),~\Pi_1=500~(m^*=220,~U^*=60)~(•)$ and $\Pi_1=1000~(m^*=400,~U^*=40)~(\triangle)$.

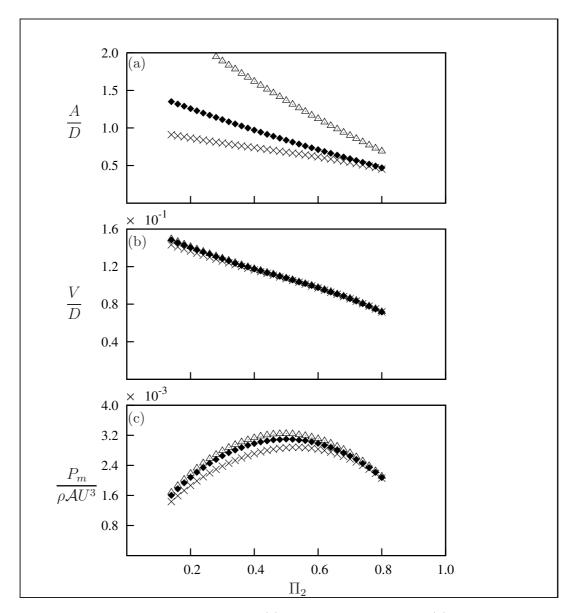


Figure 3: QSS data at high and low Π_1 . (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of Π_2 . Data presented at $\Pi_1 = 10$ (×), $\Pi_1 = 0.1$ (\spadesuit), and $\Pi_1 = 0.05$ (\triangle).

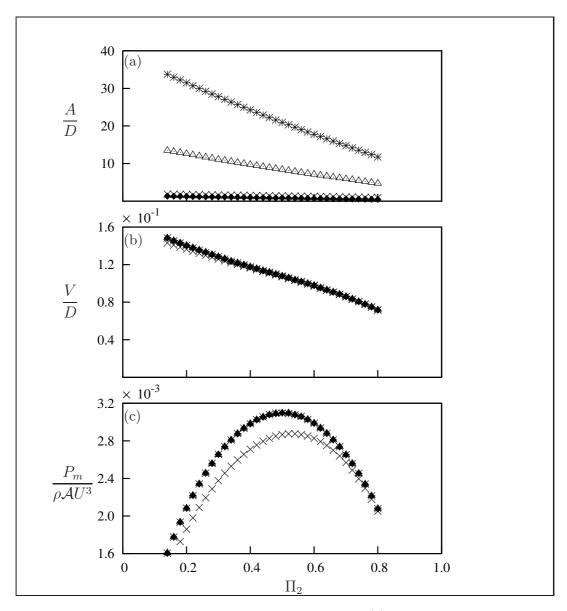


Figure 4: Comparison of QSS data at high and low Π_1 . (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of Π_2 . Data presented at $\Pi_1 = 100 \ m^* = 130(+)$, $\Pi_1 = 0.1 \ m^* = 2 \ (\spadesuit)$, $\Pi_1 = 0.1 \ m^* = 20 \ (\triangle)$ and $\Pi_1 = 0.1 \ m^* = 50 \ (*)$.

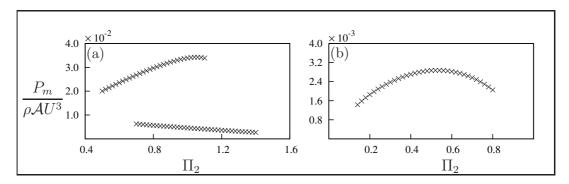


Figure 5: Mean power as a function of Π_2 . Data presented at (a) $Re=22300, \Pi_2=20000$ and (b) $Re=200, \Pi_2=100$. Hysteresis could be observed at high Re

Π_1	% error
10	30.19%
60	10.71%
250 1000	5.48% $1.16%$

Table 1: Error values of power between QSS and DNS data calculated using equation ?? at different Π_1 and Π_2 . data obtained at $U^* = 40$ and Re = 200

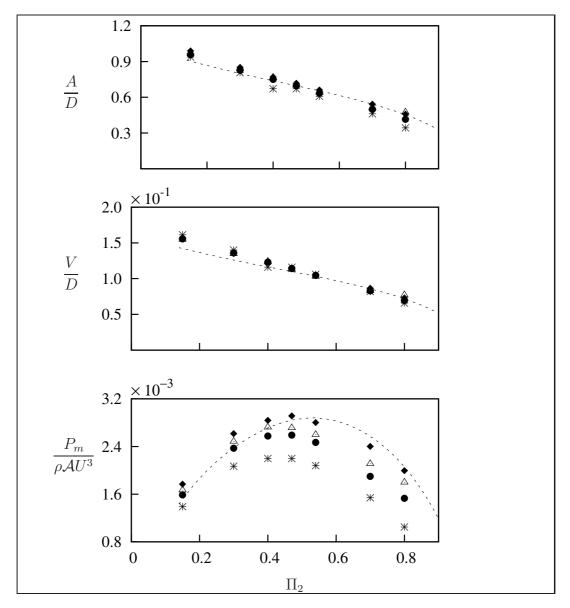


Figure 6: Comparison of data generated using the quasi-static theory and full DNS simulations . (a) Displacement amplitude, (b) velocity amplitude and (c) mean power as functions of Π_2 . Data were obtained at Re = 200 at three different combined values $\Pi_1 = 10 \ (m^* \approx 20) \ (*), \ \Pi_1 = 60 \ (m^* \approx 50) \ (\bullet), \ \Pi_2 = 250 \ (m^* \approx 100) \ (\triangle), \ \Pi_1 = 1000 \ (m^* \approx 250) \ \text{and} \ \Pi_1 = 6200 \ (m^* \approx 500).$ The QSS data at $\Pi_1 = 10$ are represented by (---)

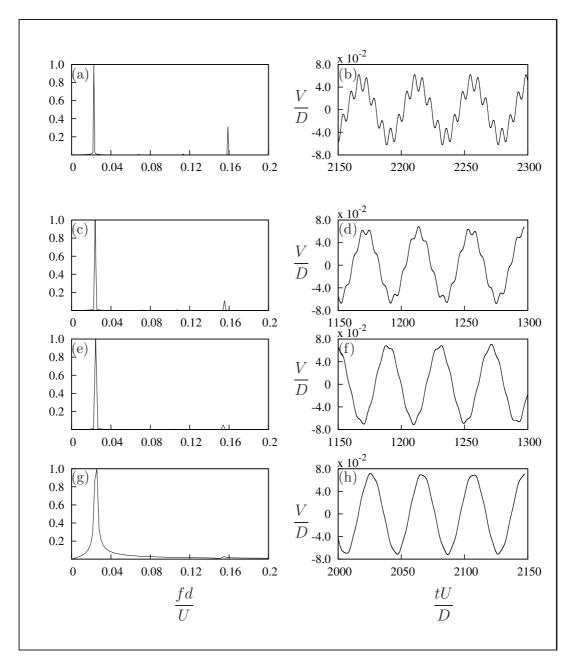


Figure 7: Velocity signal (right) and the corresponding power spectrum (left) of the DNS data at 3 different Π_1 at $\Pi_2 = 0.8$. (a) and (b) $\Pi_1 = 10$, (c) and (d) $\Pi_1 = 60$, (e) and (f) $\Pi_1 = 250$, (g) and (h) $\Pi_1 = 1000$. U^* is kept at 40 therefore the mass ratio increases ans Π_1 increases. It is evident that the influence of vortex shedding reduces as the inertia of the system increases.

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