## A study on the energy transfer of a square prism under fluid-elastic galloping

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## Abstract

Extracting useful energy from flow induced vibrations has become a developing area of research in recent years. In this paper, we analyse power transfer of an elastically mounted body under the influence of fluid-elastic galloping. The system and the power transfer is analysed by numerically integrating the quasi-steady state model equations and direct numerical simulations. The power transfer is analysed for both high (Re = 22300) and low (Re = 200) Reynolds numbers cases.

The linear analysis of the model equation shows that the system is governed by two parameters, namely the combined mass-stiffness parameter  $\Pi_1$  and the combined mass damping parameter  $\Pi_2$ . A combined mass-damping coefficient,  $\Pi_2$ , that can be derived from the equation of motion, is shown to be the parameter that governs power output. The system is a balance between the power delivered to the system due to fluid-dynamic forcing and power removed through mechanical damping which are governed by the fluid-dynamic forcing characteristics (i.e. the lift force as a function of incident angle) and mechanical damping coefficient as represented by  $\Pi_2$  respectively. Comparing the DNS results with the QSS data uncovered that a good agreement of the data could be obtained even at low Reynolds numbers when the mass-stiffness, $\Pi_1$ , is high representing a high mass to stiffness ratio. At low  $\Pi_1$ , the system shows a significant response to forces associated with shedding and this is shown to suppress the galloping response.

Keywords:

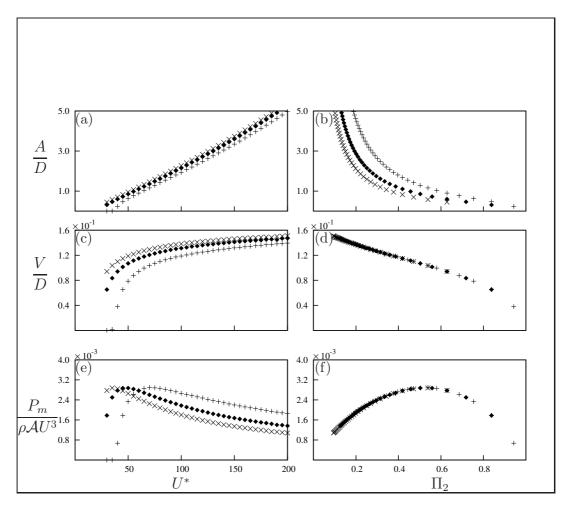


Figure 1: Displacement amplitude, velocity amplitude and mean power data as functions of two different independent varibles. Data presented in (a), (c) and (e) using the classical VIV parameter  $U^*$ , obtained at Re=200 and  $m^*=20$  at three different damping ratios:  $\zeta=0.075~(\times),~\zeta=0.1~(\spadesuit)$  and  $\zeta=0.15~(+)$ . (b) (d) and (f) are the same data presented using the combined mass-damping parameter  $(\Pi_2)$  as the independent variable.

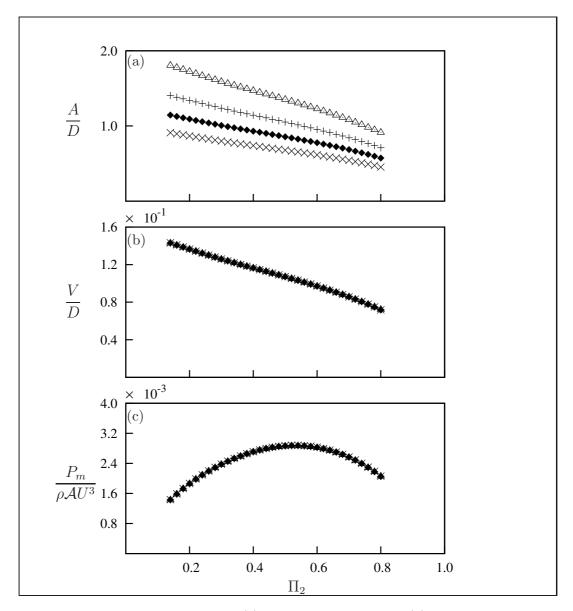


Figure 2: QSS data at high  $\Pi_1$  levels. (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of  $\Pi_2$ . Data presented at four different combined mass-stiffness levels.  $\Pi_1=10~(m^*=20,~U^*=40)~(\times),~\Pi_1=100~(m^*=80,~U^*=50)~(+),~\Pi_1=500~(m^*=220,~U^*=60)~(•)$  and  $\Pi_1=1000~(m^*=400,~U^*=40)~(\triangle)$ .

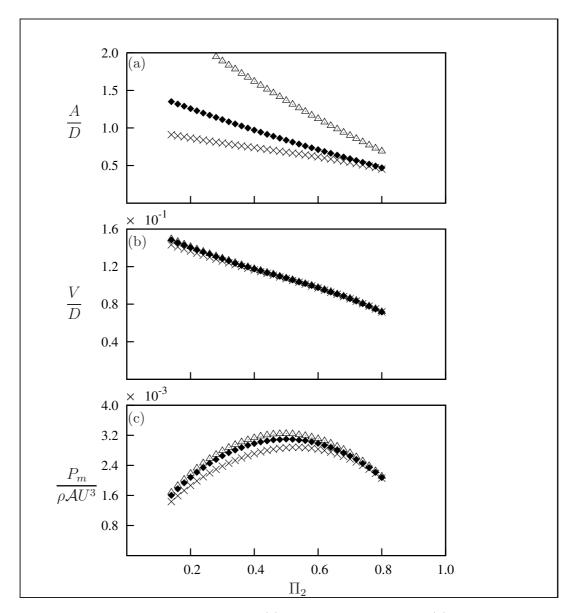


Figure 3: QSS data at high and low  $\Pi_1$ . (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of  $\Pi_2$ . Data presented at  $\Pi_1 = 10$  (×),  $\Pi_1 = 0.1$  ( $\spadesuit$ ), and  $\Pi_1 = 0.05$  ( $\triangle$ ).

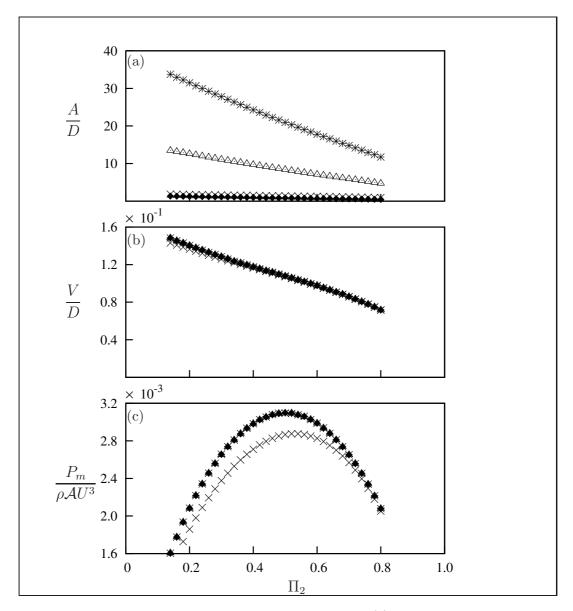


Figure 4: Comparison of QSS data at high and low  $\Pi_1$ . (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of  $\Pi_2$ . Data presented at  $\Pi_1 = 100 \ (\times) \ m^* = 130(+), \ \Pi_1 = 0.1 \ m^* = 2 \ (\spadesuit), \ \Pi_1 = 0.1 \ m^* = 20 \ (\triangle)$  and  $\Pi_1 = 0.1 \ m^* = 50 \ (*)$ . The mass ratio does not have an effect on  $\Pi_1$  even at low  $\Pi_1$ 

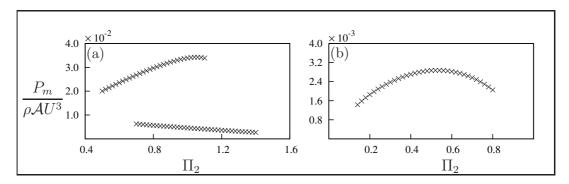


Figure 5: Mean power as a function of  $\Pi_2$ . Data presented at (a)  $Re=22300, \Pi_2=20000$  and (b)  $Re=200, \Pi_2=100$ . Hysteresis could be observed at high Re

$\Pi_1$	% error
10	30.19%
60	10.71%
250 1000	5.48% $1.16%$

Table 1: Error values of power between QSS and DNS data calculated using equation ?? at different  $\Pi_1$  and  $\Pi_2$ . data obtained at  $U^* = 40$  and Re = 200

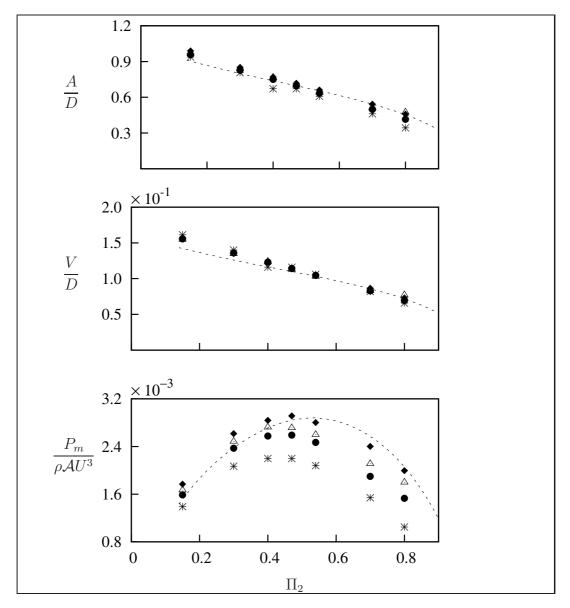


Figure 6: Comparison of data generated using the quasi-static theory and full DNS simulations . (a) Displacement amplitude, (b) velocity amplitude and (c) mean power as functions of  $\Pi_2$ . Data were obtained at Re = 200 at three different combined values  $\Pi_1 = 10 \ (m^* \approx 20) \ (*), \ \Pi_1 = 60 \ (m^* \approx 50) \ (\bullet), \ \Pi_2 = 250 \ (m^* \approx 100) \ (\triangle), \ \Pi_1 = 1000 \ (m^* \approx 250) \ \text{and} \ \Pi_1 = 6200 \ (m^* \approx 500).$  The QSS data at  $\Pi_1 = 10$  are represented by (---)

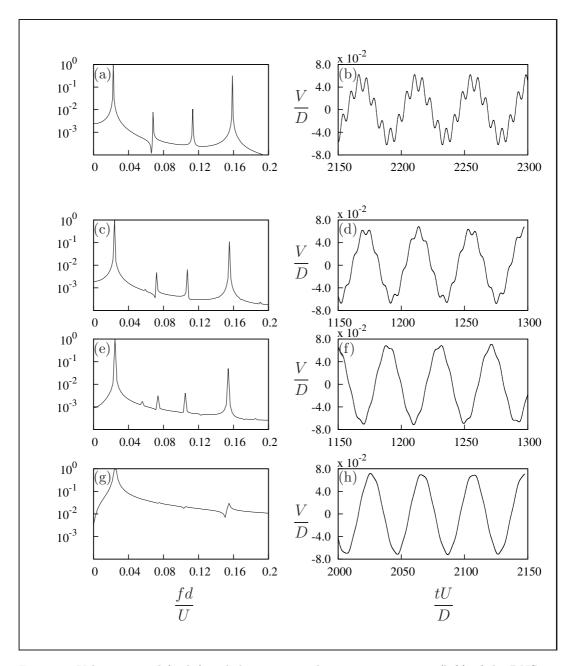


Figure 7: Velocity signal (right) and the corresponding power spectrum (left) of the DNS data at 3 different  $\Pi_1$  at  $\Pi_2 = 0.8$ . (a) and (b)  $\Pi_1 = 10$ , (c) and (d)  $\Pi_1 = 60$ , (e) and (f)  $\Pi_1 = 250$ , (g) and (h)  $\Pi_1 = 1000$ .  $U^*$  is kept at 40 therefore the mass ratio increases ans  $\Pi_1$  increases. It is evident that the influence of vortex shedding reduces as the inertia of the system increases.

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