

# A study on the energy transfer of a square prism under fluid-elastic galloping

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## Abstract

Extracting useful energy from flow induced vibrations has become a developing area of research in recent years. In this paper, we analyse power transfer of an elastically mounted body under the influence of fluid-elastic galloping. The system and the power transfer is analysed by numerically integrating the quasi-steady state model equations. The power transfer is analysed for both high ( $Re = 22300$ ) and low ( $Re = 200$ ) Reynolds numbers cases.

A combined mass-damping coefficient,  $\Pi_2$ , that can be derived from the equation of motion, is shown to be the parameter that governs power output. The system is a balance between the power delivered to the system due to fluid-dynamic forcing and power removed through mechanical damping which are governed by the fluid-dynamic forcing characteristics (i.e. the lift force as a function of incident angle) and mechanical damping coefficient respectively. Comparing the DNS results with the QSS data uncovered that a good agreement of the data could be obtained even at low Reynolds numbers when the inertia of the system (mass ratio) is substantially high.

*Keywords:*

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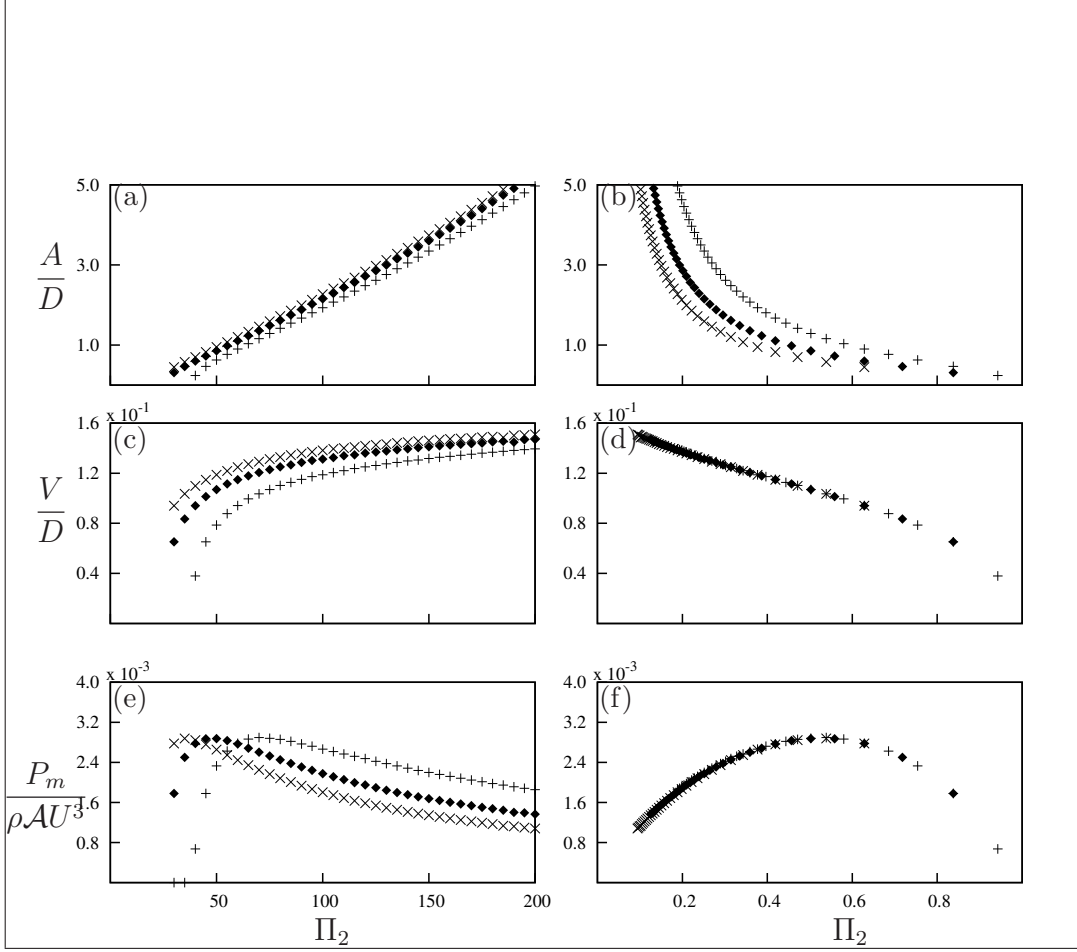


Figure 1: Comparison of the mean power data using different independent variables. (a) using classical VIV parameters  $U^*$  and  $\zeta$  at  $Re = 200$  and  $m^* = 20$  at three different damping ratios:  $\zeta = 0.075$  ( $\times$ ),  $\zeta = 0.1$  ( $\blacklozenge$ ) and  $\zeta = 0.15$  ( $+$ ) and (b) the same data collapsed using  $\Pi_2$  as the independent variable.

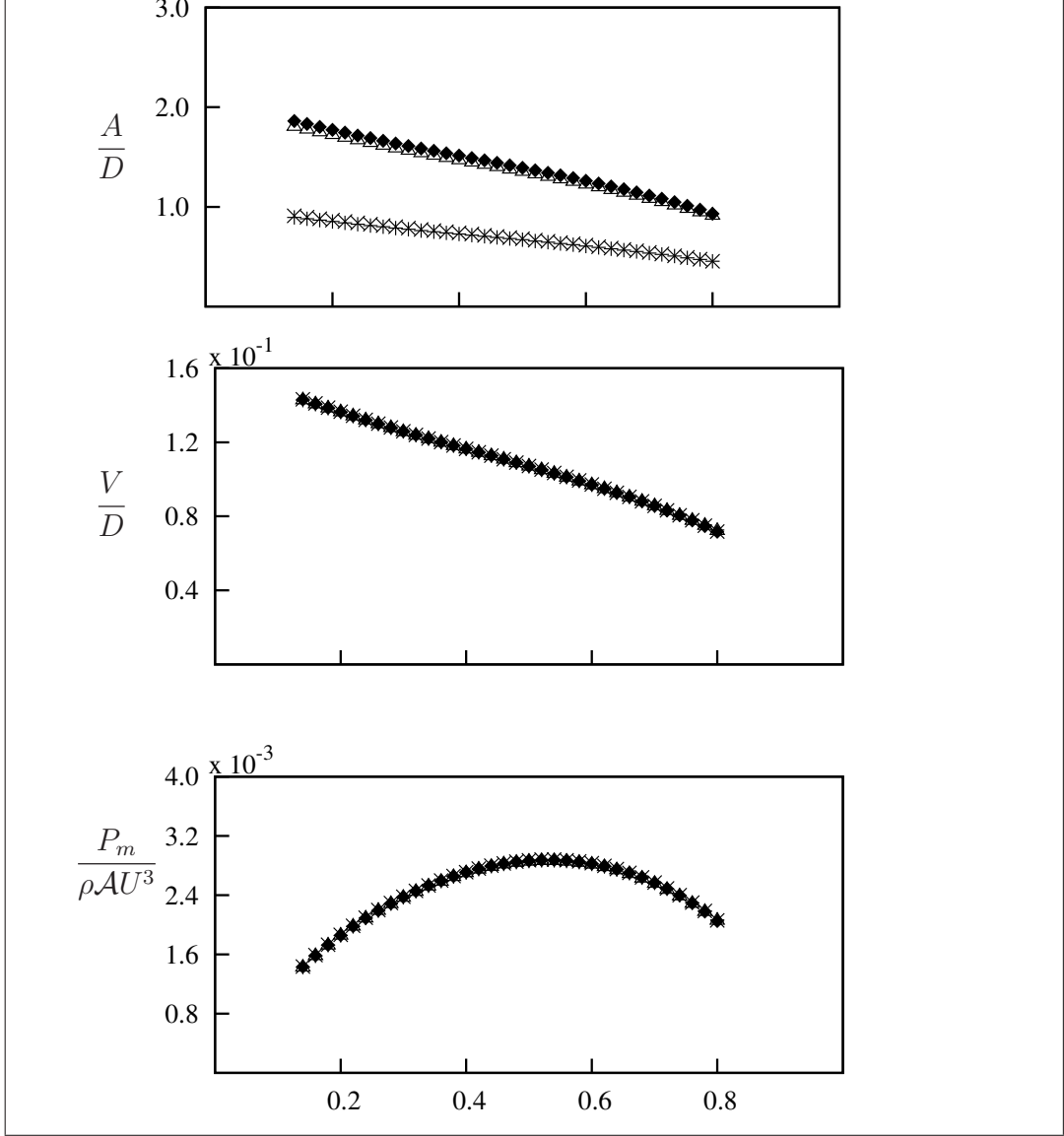


Figure 2: QSS data at high  $\Pi_1$  levels. (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of  $\Pi_2$ . Data presented at four different combined mass-stiffness levels.  $\Pi_1 = 10$  ( $m^* = 20$ ,  $U^* \approx 40$ ) (◆),  $\Pi_1 = 100$  ( $m^* = 130$ ,  $U^* \approx 80$ ) (+) and  $\Pi_1 = 1000$  ( $m^* = 400$ ,  $U^* \approx 40$ ) (△)

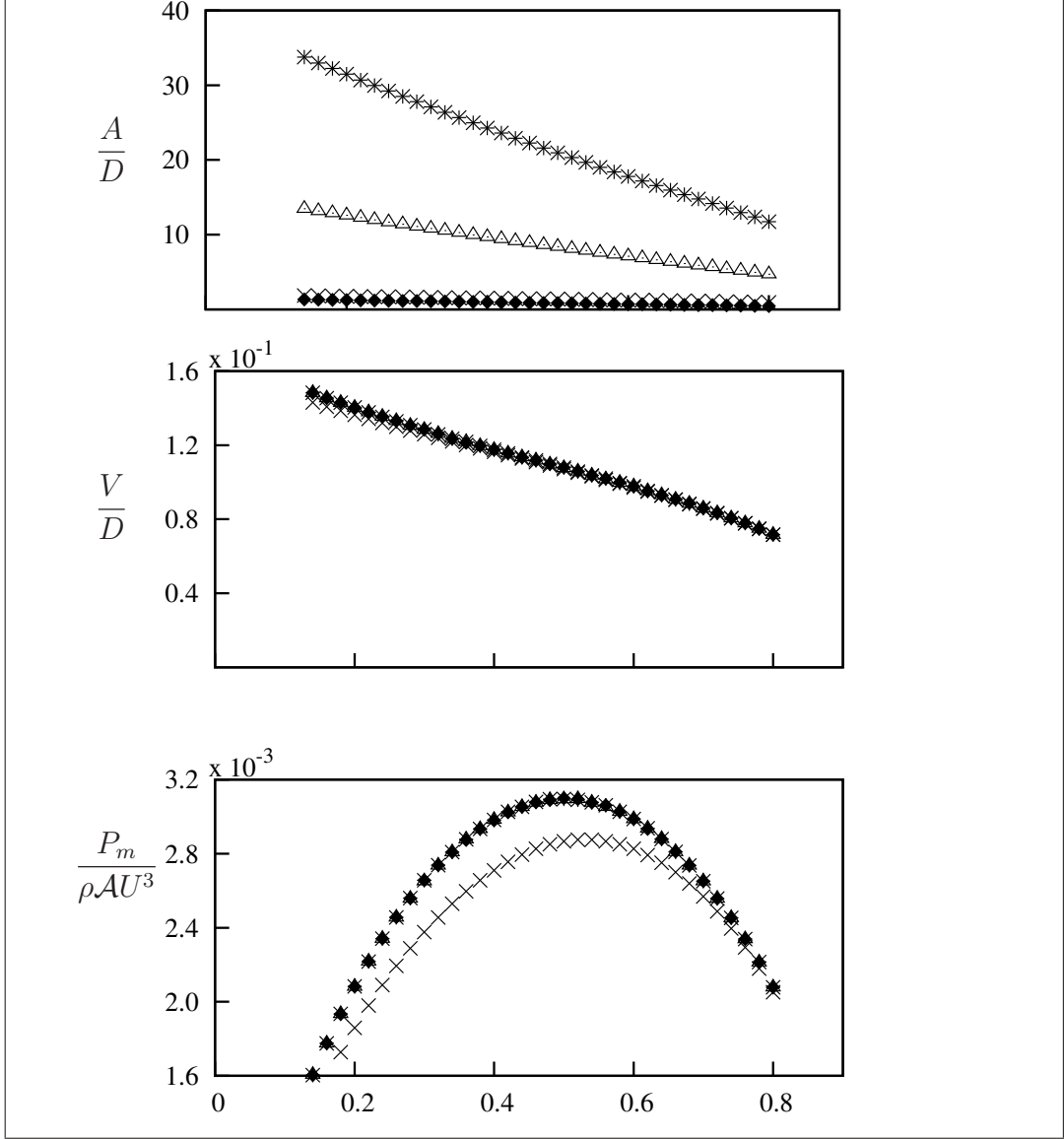


Figure 3: Comparison of QSS data at high and low  $\Pi_1$ . (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of  $\Pi_2$ . Data presented at  $\Pi_1 = 100$   $m^* = 130$ (+),  $\Pi_1 = 0.1$   $m^* = 2$  (◆),  $\Pi_1 = 0.1$   $m^* = 20$  (△) and  $\Pi_1 = 0.1$   $m^* = 50$  (\*)

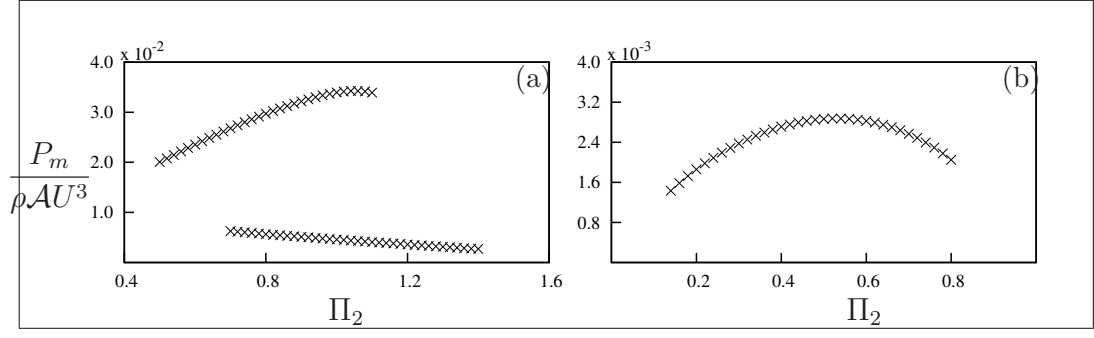


Figure 4:

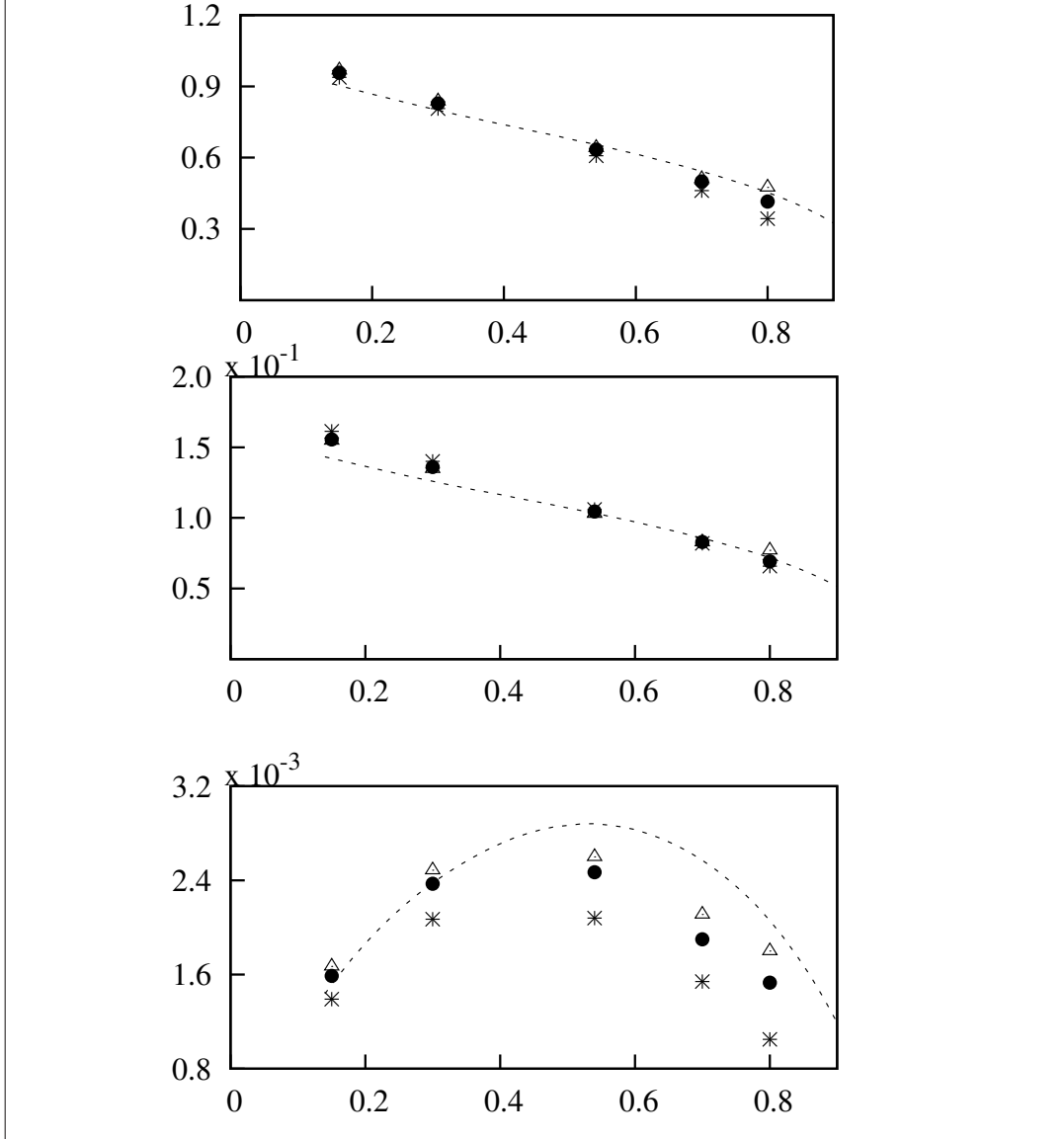


Figure 5: Comparison of data generated using the quasi-static theory and full DNS simulations . (a) Displacement amplitude, (b) velocity amplitude and (c) mean power as functions of  $\Pi_2$ . Data were obtained at  $Re = 200$  at three different combined values  $\Pi_2 = 10$  ( $m^* \approx 20$ ) (\*),  $\Pi_2 = 60$  ( $m^* \approx 50$ ) (●),  $\Pi_2 = 250$  ( $m^* \approx 100$ ) (△),  $\Pi_2 = 1000$  ( $m^* \approx 250$ ) and  $\Pi_2 = 6200$  ( $m^* \approx 500$ )

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