

A study on the energy transfer of a square prism under fluid-elastic galloping

H.G.K.G. Jayatunga, B.T. Tan, J. S. Leontini

Abstract

Extracting useful energy from flow induced vibrations has become a developing area of research in recent years. In this paper, we analyse power transfer of an elastically mounted body under the influence of fluid-elastic galloping. The system and the power transfer is analysed by numerically integrating the quasi-steady state model equations and direct numerical simulations. The power transfer is analysed for both high ($Re = 22300$) and low ($Re = 200$) Reynolds numbers cases.

The linear analysis of the model equation shows that the system is governed by two parameters, namely the combined mass-stiffness parameter Π_1 and the combined mass damping parameter Π_2 . A combined mass-damping coefficient, Π_2 , that can be derived from the equation of motion, is shown to be the parameter that governs power output. The system is a balance between the power delivered to the system due to fluid-dynamic forcing and power removed through mechanical damping which are governed by the fluid-dynamic forcing characteristics (i.e. the lift force as a function of incident angle) and mechanical damping coefficient as represented by Π_2 respectively. Comparing the DNS results with the QSS data uncovered that a good agreement of the data could be obtained even at low Reynolds numbers when the mass-stiffness, Π_1 , is high representing a high mass to stiffness ratio. At low Π_1 , the system shows a significant response to forces associated with shedding and this is shown to suppress the galloping response.

Keywords:

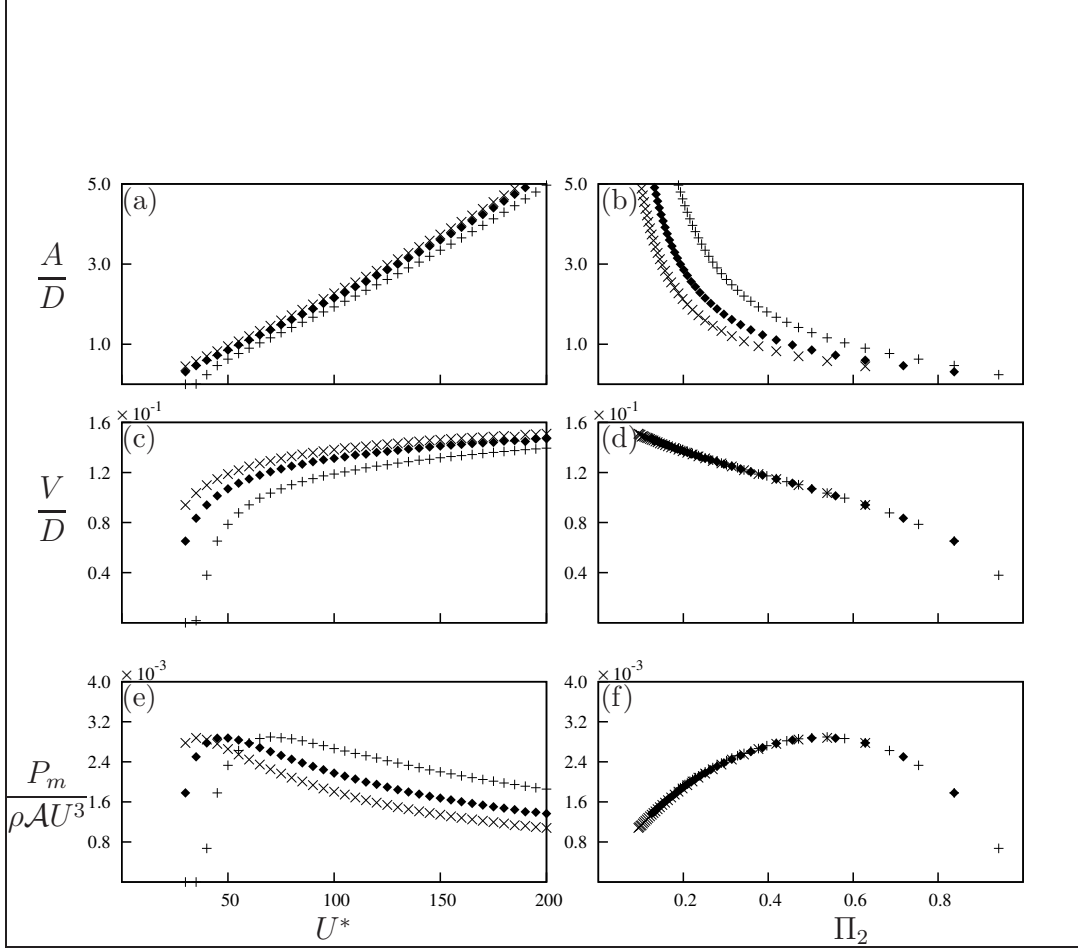


Figure 1: Displacement amplitude, velocity amplitude and mean power data as functions of two different independent variables. Data presented in (a), (c) and (e) using the classical VIV parameter U^* , obtained at $Re = 200$ and $m^* = 20$ at three different damping ratios: $\zeta = 0.075$ (x), $\zeta = 0.1$ (◆) and $\zeta = 0.15$ (+). (b) (d) and (f) are the same data presented using the combined mass-damping parameter (Π_2) as the independent variable.

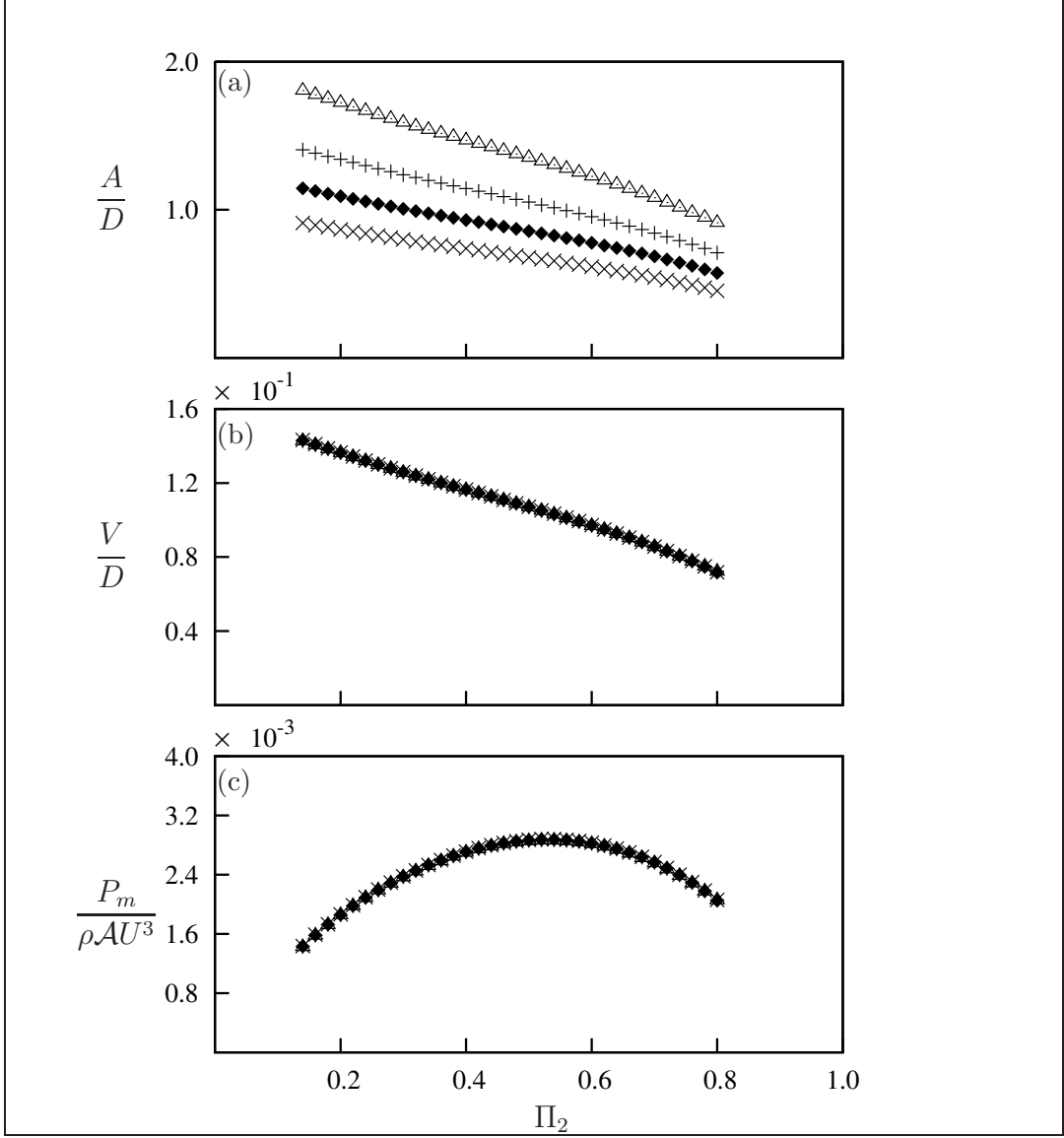


Figure 2: QSS data at high Π_1 levels. (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of Π_2 . Data presented at four different combined mass-stiffness levels. $\Pi_1 = 10$ ($m^* = 20$, $U^* = 40$) (\times), $\Pi_1 = 100$ ($m^* = 80$, $U^* = 50$) ($+$), $\Pi_1 = 500$ ($m^* = 220$, $U^* = 60$) (\blacklozenge) and $\Pi_1 = 1000$ ($m^* = 400$, $U^* = 40$) (\triangle).

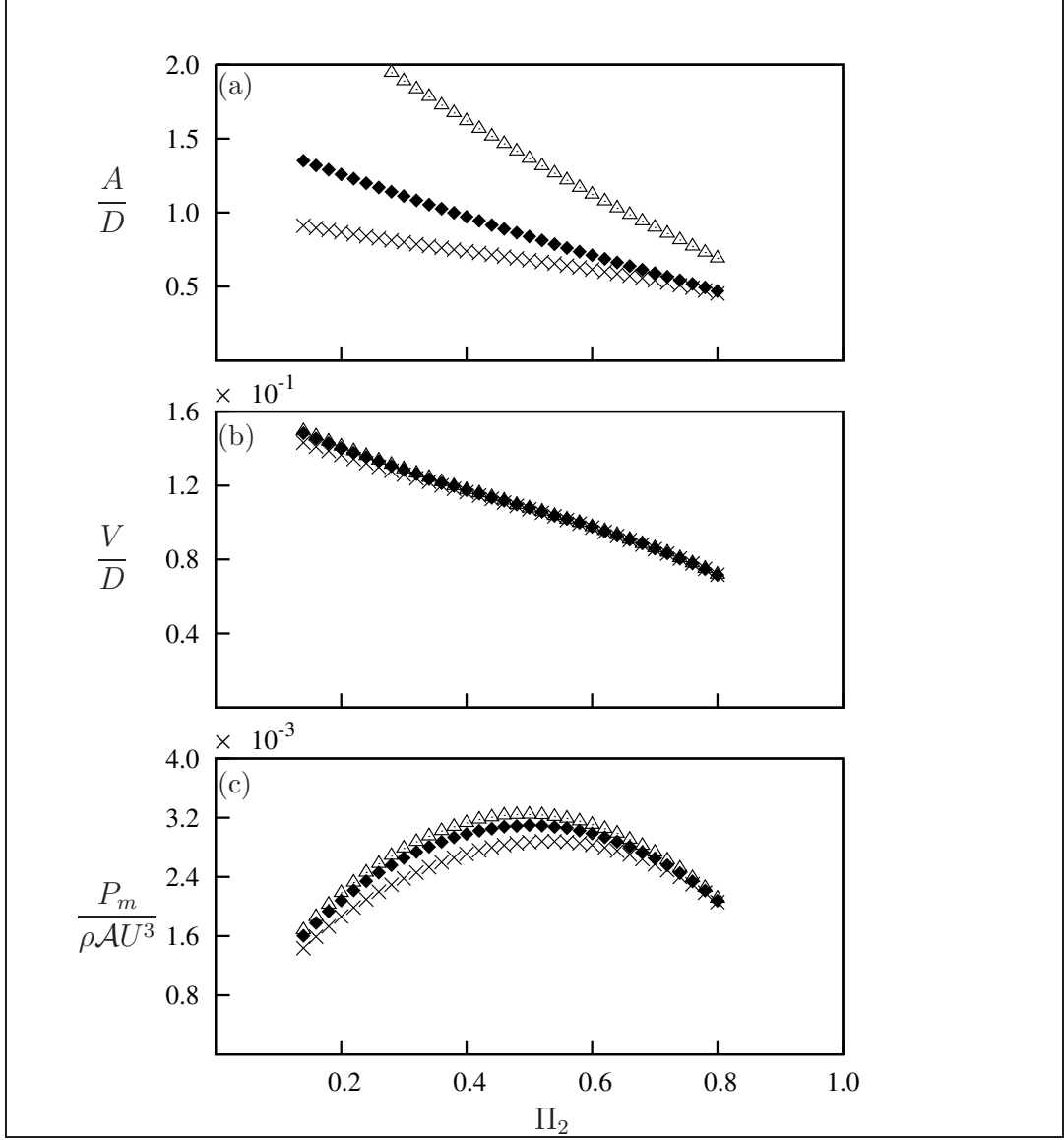


Figure 3: QSS data at high and low Π_1 . (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of Π_2 . Data presented at $\Pi_1 = 10$ (\times), $\Pi_1 = 0.1$ (\blacklozenge), and $\Pi_1 = 0.05$ (\triangle).

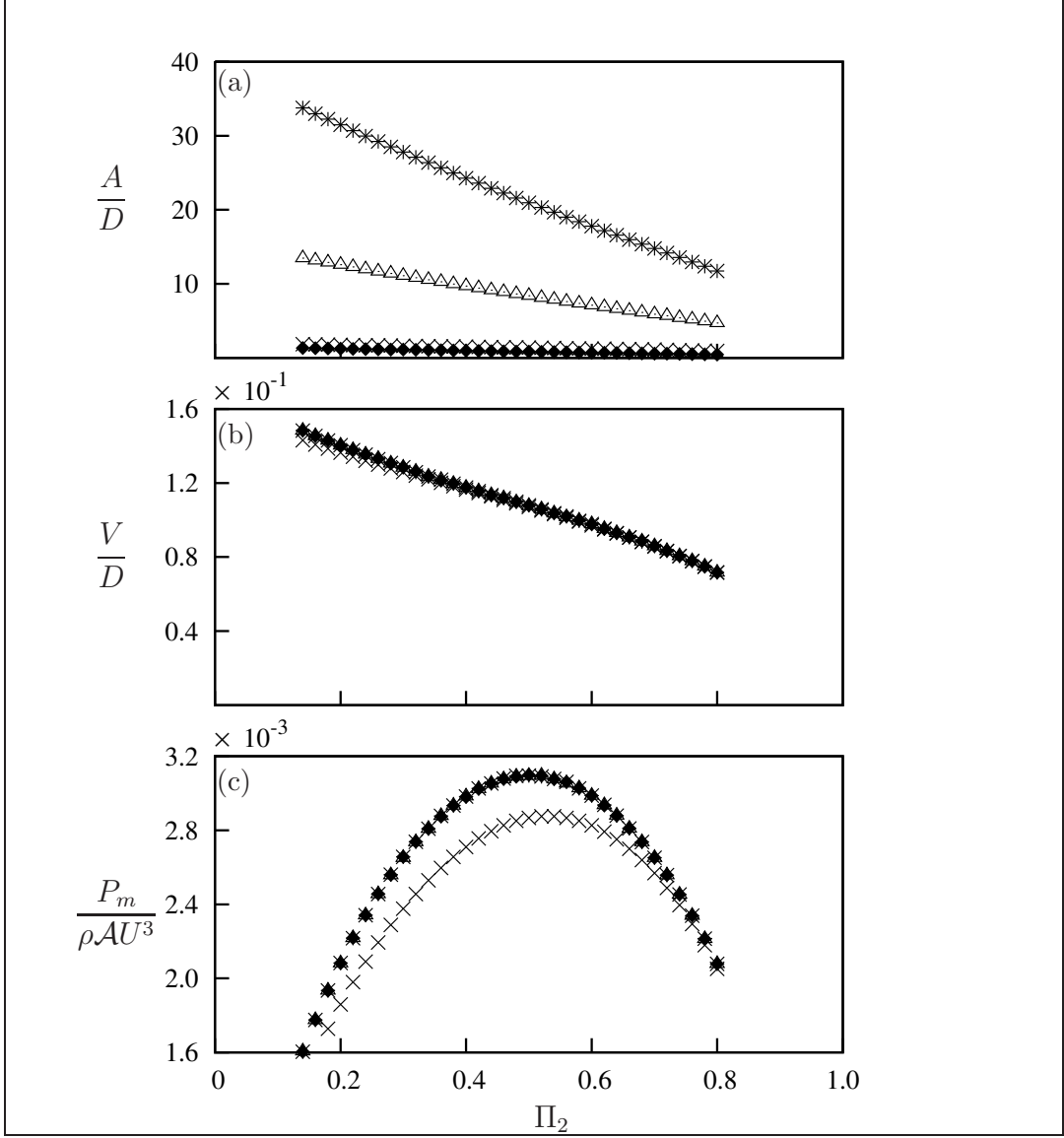


Figure 4: Comparison of QSS data at high and low Π_1 . (a) displacement amplitude, (b) velocity amplitude and (c) mean power as a function of Π_2 . Data presented at $\Pi_1 = 100$ (×) $m^* = 130$ (+), $\Pi_1 = 0.1$ $m^* = 2$ (◆), $\Pi_1 = 0.1$ $m^* = 20$ (△) and $\Pi_1 = 0.1$ $m^* = 50$ (*). The mass ratio does not have an effect on Π_1 even at low Π_1

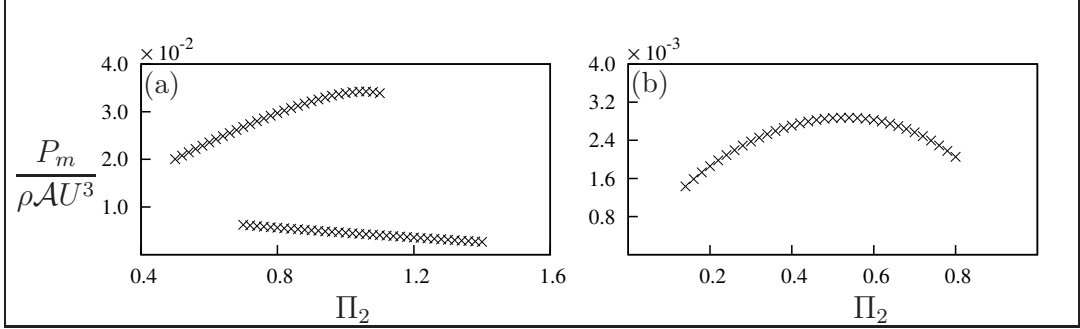


Figure 5: Mean power as a function of Π_2 . Data presented at (a) $Re = 22300$, $\Pi_2 = 20000$ and (b) $Re = 200$, $\Pi_2 = 100$. Hysteresis could be observed at high Re

Π_1	% error
10	30.19%
60	10.71%
250	5.48%
1000	1.16%

Table 1: Error values of power between QSS and DNS data calculated using equation ?? at different Π_1 and Π_2 . data obtained at $U^* = 40$ and $Re = 200$

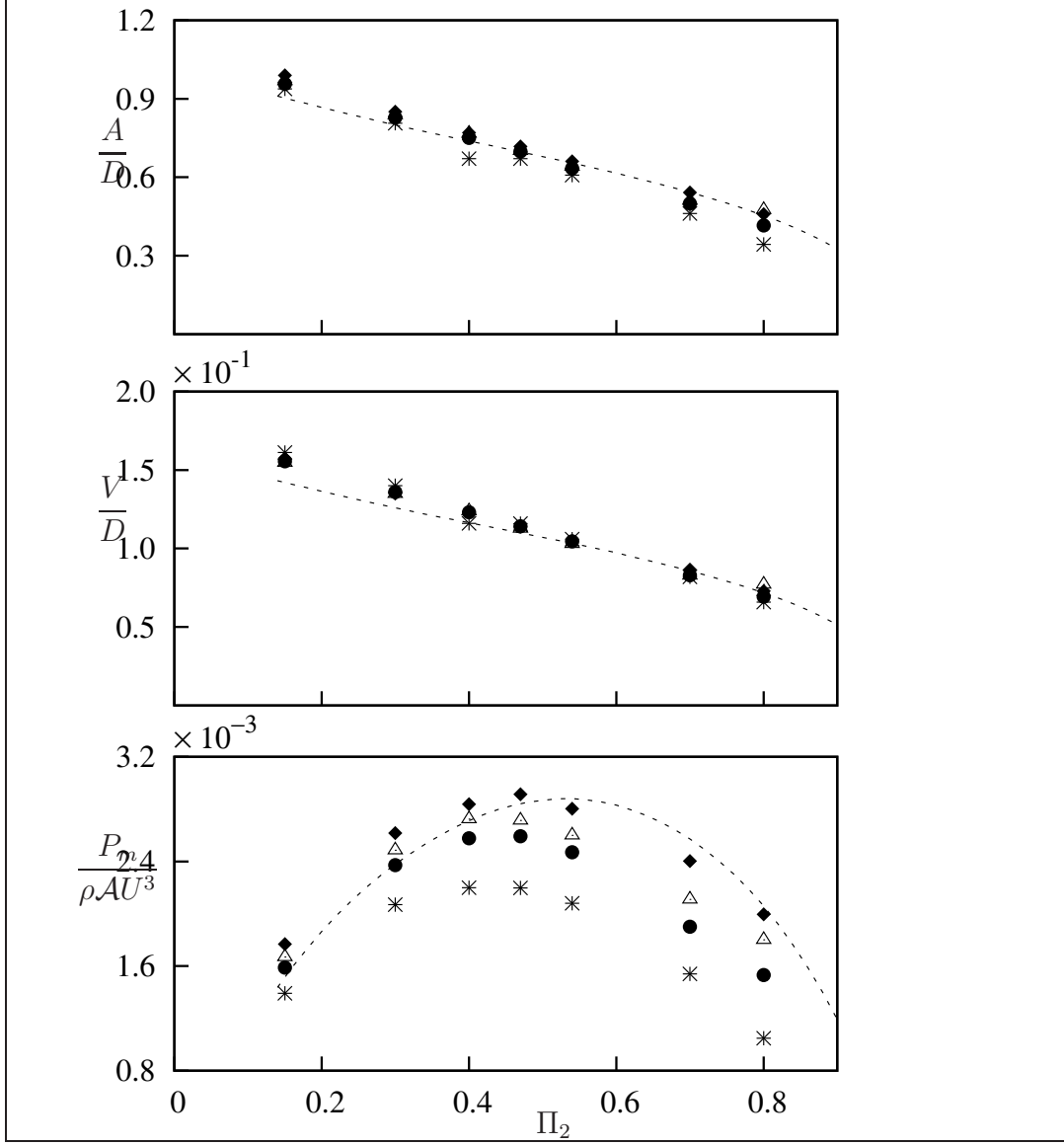


Figure 6: Comparison of data generated using the quasi-static theory and full DNS simulations. (a) Displacement amplitude, (b) velocity amplitude and (c) mean power as functions of Π_2 . Data were obtained at $Re = 200$ at three different combined values $\Pi_1 = 10$ ($m^* \approx 20$) (*), $\Pi_1 = 60$ ($m^* \approx 50$) (●), $\Pi_1 = 250$ ($m^* \approx 100$) (△), $\Pi_1 = 1000$ ($m^* \approx 250$) and $\Pi_1 = 6200$ ($m^* \approx 500$). The QSS data at $\Pi_1 = 10$ are represented by (---)

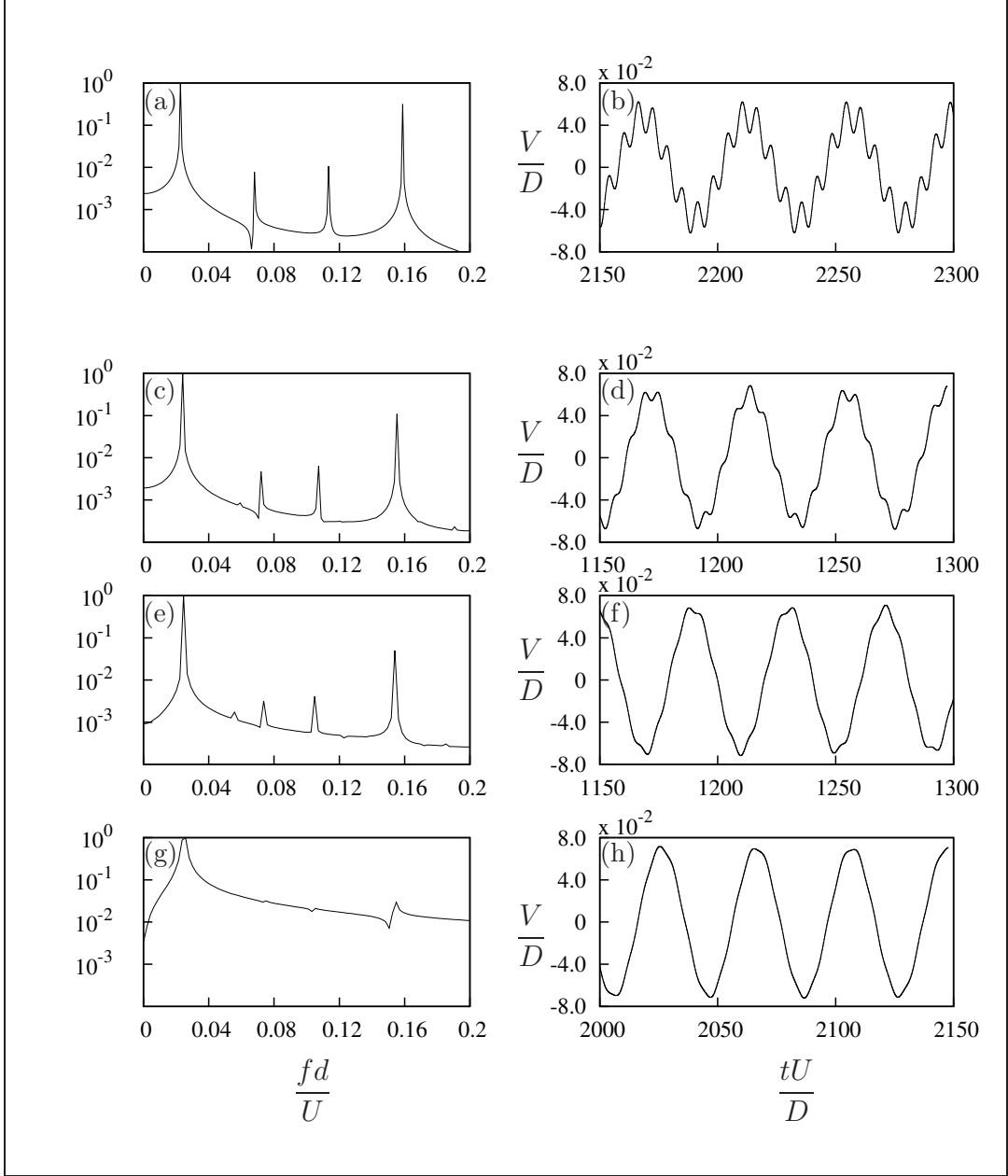


Figure 7: Velocity signal (right) and the corresponding power spectrum (left) of the DNS data at 3 different Π_1 at $\Pi_2 = 0.8$. (a) and (b) $\Pi_1 = 10$, (c) and (d) $\Pi_1 = 60$, (e) and (f) $\Pi_1 = 250$, (g) and (h) $\Pi_1 = 1000$. U^* is kept at 40 therefore the mass ratio increases as Π_1 increases. It is evident that the influence of vortex shedding reduces as the inertia of the system increases.

References

- Barrero-Gil, A., Alonso, G., Sanz-Andres, A., Jul. 2010. Energy harvesting from transverse galloping. *Journal of Sound and Vibration* 329 (14), 2873–2883.
- Barrero-Gil, A., Sanz-Andrés, A., Roura, M., Oct. 2009. Transverse galloping at low Reynolds numbers. *Journal of Fluids and Structures* 25 (7), 1236–1242.
- Den Hartog, J. P., 1956. *Mechanical Vibrations*. Dover Books on Engineering. Dover Publications.
- Glauert, H., 1919. The rotation of an aerofoil about a fixed axis. Tech. rep., Advisory Committee on Aeronautics R and M 595. HMSO, London.
- Griffith, M. D., Leontini, J. S., Thompson, M. C., Hourigan, K., 2011. Vortex shedding and three-dimensional behaviour of flow past a cylinder confined in a channel. *Journal of Fluids and Structures* 27 (5-6), 855–860.
- Joly, A., Etienne, S., Pelletier, D., Jan. 2012. Galloping of square cylinders in cross-flow at low Reynolds numbers. *Journal of Fluids and Structures* 28, 232–243.
- Leontini, J. S., Lo Jacono, D., Thompson, M. C., Nov. 2011. A numerical study of an inline oscillating cylinder in a free stream. *Journal of Fluid Mechanics* 688, 551–568.
- Leontini, J. S., Thompson, M. C., 2013. Vortex-induced vibrations of a diamond cross-section: Sensitivity to corner sharpness. *Journal of Fluids and Structures* 39, 371–390.
- Leontini, J. S., Thompson, M. C., Hourigan, K., Apr. 2007. Three-dimensional transition in the wake of a transversely oscillating cylinder. *Journal of Fluid Mechanics* 577, 79.
- Luo, S., Chew, Y., Ng, Y., Aug. 2003. Hysteresis phenomenon in the galloping oscillation of a square cylinder. *Journal of Fluids and Structures* 18 (1), 103–118.

- Ng, Y., Luo, S., Chew, Y., Jan. 2005. On using high-order polynomial curve fits in the quasi-steady theory for square-cylinder galloping. *Journal of Fluids and Structures* 20 (1), 141–146.
- Païdoussis, M., Price, S., de Langre, E., 2010. *Fluid-Structure Interactions : Cross-Flow-Induced Instabilities*. Cambridge University Press.
- Parkinson, G. V., Smith, J. D., 1964. The square prism as an aeroelastic non-linear oscillator. *The Quarterly Journal of Mechanics and Applied Mathematics* 17 (2), 225–239.
- Thompson, M., Hourigan, K., Sheridan, J., Feb. 1996. Three-dimensional instabilities in the wake of a circular cylinder. *Experimental Thermal and Fluid Science* 12 (2), 190–196.
- Thompson, M. C., Hourigan, K., Cheung, A., Leweke, T., Nov. 2006. Hydrodynamics of a particle impact on a wall. *Applied Mathematical Modelling* 30 (11), 1356–1369.
- Vio, G., Dimitriadis, G., Cooper, J., Oct. 2007. Bifurcation analysis and limit cycle oscillation amplitude prediction methods applied to the aeroelastic galloping problem. *Journal of Fluids and Structures* 23 (7), 983–1011.