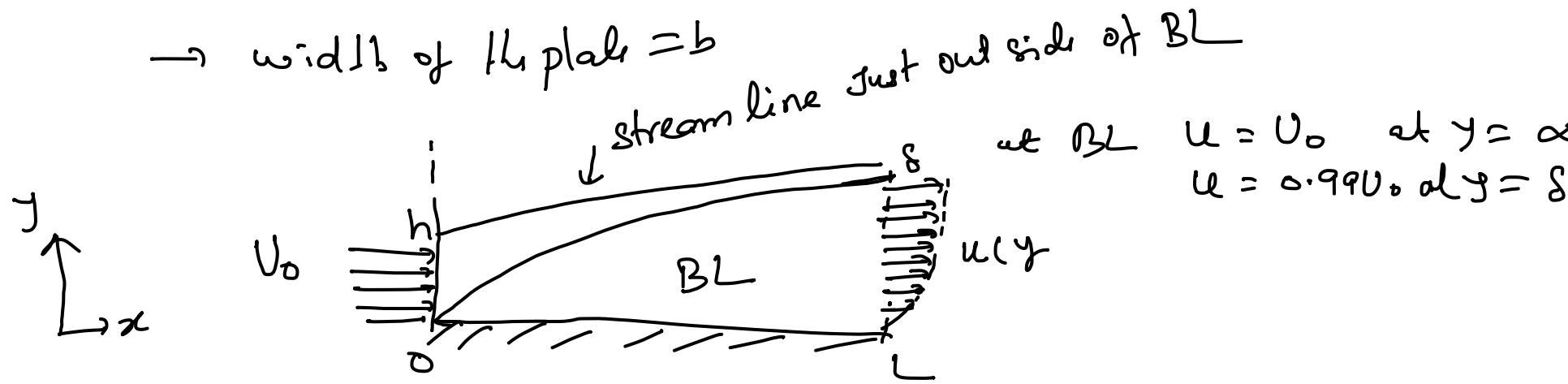


Drag on a flat plate

Boundary layer theory.... continued

- Newton's law of motion is valid for any kind of flow (laminar or turbulent)

→ width of the plate = b



$$\text{at BL } u = U_0 \quad \text{at } y = \infty$$

$$u = 0.99U_0 \quad \frac{dy}{dx} = \delta$$

control volume $0 \times h \times \delta \times b \rightarrow$ Apply newton's law of motion

Drag force on the surface = $D \hat{i}$

x -force on OL surface = $-D \hat{i}$

The four sides of CV

(i) $0h$ side \Rightarrow inlet $\vec{V} \cdot \hat{n} = -U_0$

(ii) $h\delta$ side \Rightarrow inlet/outlet $\vec{V} \cdot \hat{n} = 0$ $\vec{V} \parallel dS$

(iii) δL side \Rightarrow outlet $\vec{V} \cdot \hat{n} = u(y)$

(iv) $L0$ side \Rightarrow surface $\vec{V} \cdot \hat{n} = 0$

x -momentum balance at steady state

$$-D = \cancel{\frac{d}{dt}(\rho)} + \int u P(v \cdot n) dA + \int u P(v \cdot n) dA$$

(i) (ii) (iii)

$$-D = U_0 P(-U_0) b h + \int u(y) P u(y) b dy \quad |_{x=L} \quad (i)$$

\rightarrow we want to replace h in terms of δ .

\rightarrow mass balance on CV

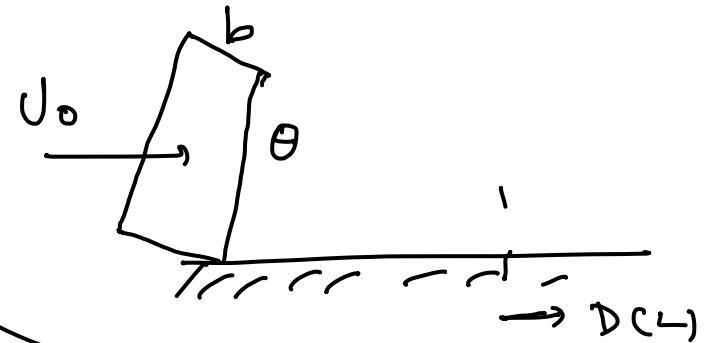
$$0 = \cancel{\frac{d}{dt}(\rho)} + \int_0^\delta \rho b u(y) dy \Big|_{x=L} - \rho b U_0 h \quad (ii)$$

$$U_0 h = \int_0^\delta u dy \Big|_{x=L}$$

$$D = P b U_0 \int_0^\delta u dy \Big|_{x=L} - \rho b \int_0^\delta u^2 dy \Big|_{x=L}$$

$$D = \rho b \int_0^L u (U_0 - \bar{u}) dy \Big|_{x=L}$$

Momentum thickness θ



$$\text{momentum in} = \rho U_0 b \theta \quad U_0 = \rho U_0^2 b \theta$$

$$\Rightarrow \theta = \int_0^L \frac{u}{U_0} \left(1 - \frac{u}{U_0}\right) dy$$

$$D(x) = b \int_0^x c_w dx \Rightarrow \frac{dD(x)}{dx} = b c_w$$

$$\frac{dD(x)}{dx} = b c_w = \rho U_0^2 b \frac{d\theta}{dx}$$

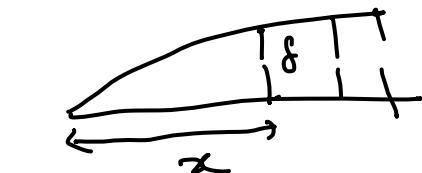
$$c_w = \rho U_0^2 \frac{d\theta}{dx}$$

- momentum integral eqn for flat plate
- valid laminar / turbulent

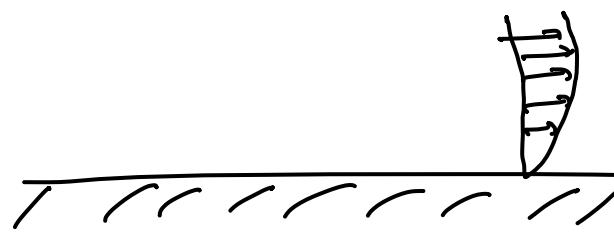
\rightarrow goal \rightarrow find $\delta(x)$

\rightarrow Drag force \rightarrow friction coefficient c_f \leftarrow [similar to friction factor for pressure drop]

$$D(x) = \rho b \int_0^x u (U_0 - u) dy$$



Laminar flow



Karman → empirical
 $\underline{u(y)}$
 $\underline{\underline{Re}} \rightarrow \text{High}$

Assumption
 $u(y) = \text{Parabolic f^n of } y$
 → Approximate sol^n
 → Exact sol^n by solving
 Navier-Stokes for BL

Approximate sol^n

$$u(x,y) = u_0 \left(\frac{2y}{\delta} - \frac{y^2}{\delta^2} \right)^n \quad 0 \leq y \leq \delta(x)$$

$$\theta = \int_0^\delta \frac{u_0}{U_0} \left(1 - \frac{y}{U_0} \right) dy$$

$$= \frac{2}{15} \delta$$

$$\tau_w = \mu \frac{\partial u}{\partial y} \Big|_{y=0} = \frac{2 U_0 \mu}{\delta}$$

$$\tau_w = \rho U_0^2 \frac{d\theta}{dx} = \frac{2 U_0 \mu}{\delta}$$

$$\rho U_0^2 \left(\frac{2}{15} \right) \frac{d\delta}{dx} = \frac{2 U_0 \mu}{\delta}$$

$$\frac{1}{2} \delta^2 = \frac{15}{U_0} \frac{\partial}{\partial x}$$

$$\frac{\delta}{x} \approx \frac{5.5}{U_0 x} \left(\frac{x}{U_0} \right)^{1/2} = \frac{5.5}{Rex^{1/2}}$$

Approximate

Exact

$$\frac{\delta}{x} = \frac{5.0}{Rex^{1/2}}$$

Drag force
 Skin-friction coefficient C_f

$$C_f = \frac{\tau_w}{\frac{1}{2} \rho U_0^2} = \frac{2 U_0 \mu}{\delta \left(\frac{1}{2} \rho U_0^2 \right)} = \frac{4 \mu}{\delta \rho U_0}$$

$$\delta = \left(\frac{4}{15 \times 2} \right)^{1/2} \left(\frac{x}{U_0} \right)^{1/2} = \frac{0.73}{Rex^{1/2}} \quad (\text{Approximate})$$

$$C_f = \frac{0.664}{Rex^{1/2}} \quad (\text{Exact})$$

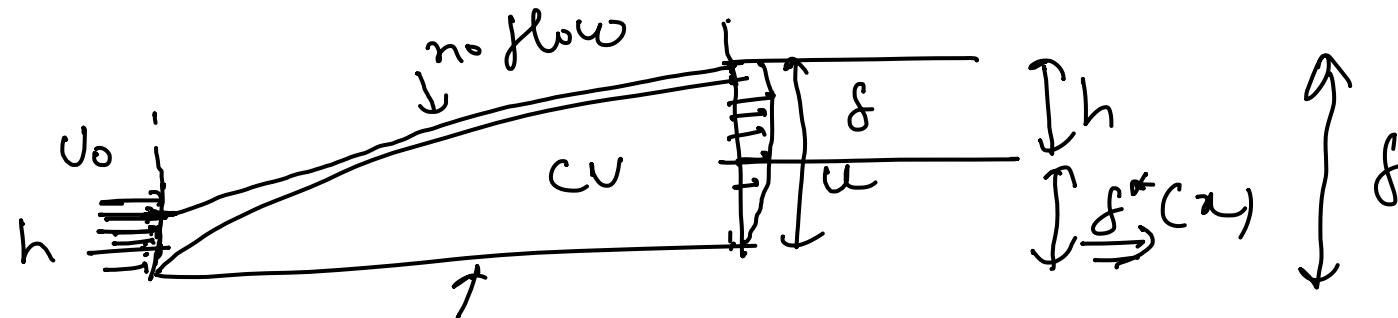
$Re \rightarrow \text{High} \Rightarrow \delta \ll L \Rightarrow$ Boundary layer theory
 thin

$$\frac{\delta}{x} \leq 0.1 \Rightarrow 0.1 = \frac{5}{(Rex)^{1/2}} \Rightarrow Rex > 2500 \quad \text{thin}$$

- for thick BL, boundary layer theory fails because thick layer has significant effect on the outer - inviscid flow $Re \rightarrow$ High but laminar

Displacement thickness ($\delta^*(x)$)

- stream lines outside of BL are deflected by a distance $\delta^*(x)$ to satisfy mass conservation between the inlet and outlet



$$\delta = \delta^* + h$$

mass balance on CV

$$0 = \frac{d(P)}{dx} + \int \rho u \frac{\partial y}{\partial x} dy - \rho U_0 V h$$

$$U_0 h = \int_0^\delta u dy = \int_0^\delta (U_0 + u - U_0) dy = \\ = U_0 \delta + \int (u - U_0) dy$$

$$U_0 h = U_0 (\delta^* + h) + \int (u - U_0) dy$$

$$- U_0 \delta^* = \int (u - U_0) dy \Rightarrow \delta^* = \int_0^\delta \left(1 - \frac{u}{U_0}\right) dy$$

Approximate

$$u = U_0 \left(\frac{2y}{\delta} - \frac{y^2}{\delta^2}\right)$$

$$\delta^* = \int_0^\delta \left(1 - \frac{2y}{\delta} + \frac{y^2}{\delta^2}\right) dy = \frac{1}{3} \delta$$

$$\left[\frac{\delta^*}{x} = \frac{1}{3} \quad \frac{\delta}{x} = \frac{1}{3} \frac{5.5}{Re^{1/2}} = \frac{1.83}{Re^{1/2}} \right]$$

Exact

$$\delta^* = 0.344 \delta$$

$$\frac{\delta}{x} = \frac{5.0}{Re^{1/2}}$$

$$\left[\frac{\delta^*}{x} = \frac{1.721}{Re^{1/2}} \right]$$