ESO204A: Fluid Mechanics and Rate Processes

Kinematics

We will try to describe the fluid motion without being concerned with the forces necessary to cause the motion

©Malay K. Das mkdas@iitk.ac.in Department of Mechanical Engineering Indian Institute of Technology Kanpur Kanpur UP 208016

Announcements

- 1. Tutorial: Thursday, 1100-1200, T203-T212
- 2. Please check the Section list
- 3. Try to solve the problems; bring a copy of the problems in the tutorial
- **4.** My office hours: Thursday, 1800-1900, SL210; you are welcome with any questions/doubts

Kinematics: how to describe fluid motion?

Lagrangian vs Eulerian descriptions

Material (total) vs partial (local) derivatives

Reference: Chapter 1 of F. M. White

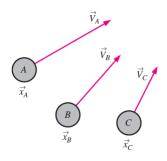
We will now start applying vector calculus concepts, take some time to revisit your Maths book

Lagrangian descriptions

oLagrangian: Attention is fixed on a particular mass of fluid as it flows

oTrack position and velocity vectors of each fluid particles (or any fixed mass system) as a function of time



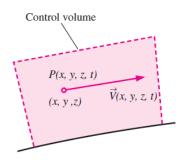


Independent variables: initial position, time

Eulerian description

oWe focus on an imaginary control volume (or a point), usually fixed in space; fluid flows in and out of the control volume

oWe do not track each particle, instead rely on 'field' variables (usually velocity and pressure fields)



oLagrangian description helps in deriving the governing Equations, Eulerian frame is used for the solution

Eulerian (Field) representation

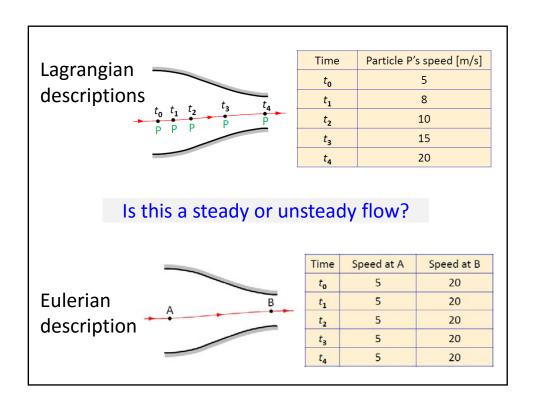
olmportant flow variables (velocity and pressure fields) are expressed as a function of space of time, such as

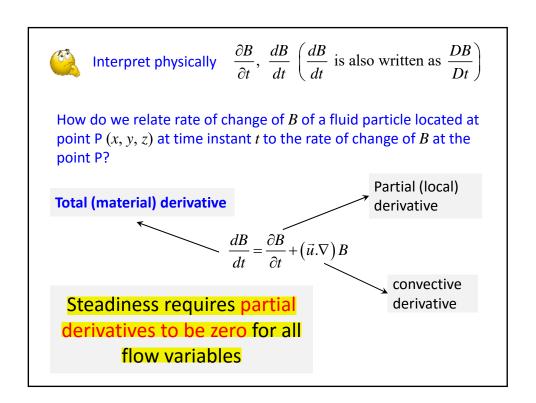
$$\vec{u} = \vec{u}(t, x, y, z), p = p(t, x, y, z)$$

oA fluid particle (contains large number of molecules), at a particular point, attains the values (of the field variables) assigned at that point

OSteady field: time derivative to be zero; unsteady otherwise; transient fields evolve into steady or periodic (unsteady) state

oDimensionality: dependence on spatial coordinate; maximum three (03)



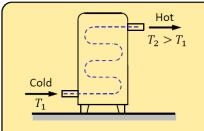


$$\frac{dB(t,x)}{dt} = \lim_{\Delta t \to 0} \left[\frac{B(t + \Delta t, x + \Delta x) - B(t,x)}{\Delta t} \right]
= \lim_{\Delta t \to 0} \left[\frac{B(t + \Delta t, x) - B(t,x)}{\Delta t} \right] + \lim_{\Delta t \to 0} \left[\frac{B(t + \Delta t, x + \Delta x) - B(t + \Delta t, x)}{\Delta t} \right]
= \frac{\partial B}{\partial t} + \lim_{\Delta t \to 0} \left[\frac{B(t + \Delta t, x + \Delta x) - B(t + \Delta t, x)}{\Delta x} \frac{\Delta x}{\Delta t} \right]
= \frac{\partial B}{\partial t} + u \frac{\partial B}{\partial x} \quad \text{(assuming } \Delta x \to 0\text{)}$$

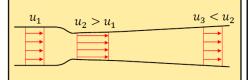
Can be extended for 3-D case; in general

$$\frac{dB}{dt} = \frac{\partial B}{\partial t} + (\vec{u}.\nabla)B \quad \text{where } \vec{u}.\nabla = u\frac{\partial}{\partial x} + v\frac{\partial}{\partial y} + w\frac{\partial}{\partial z}$$

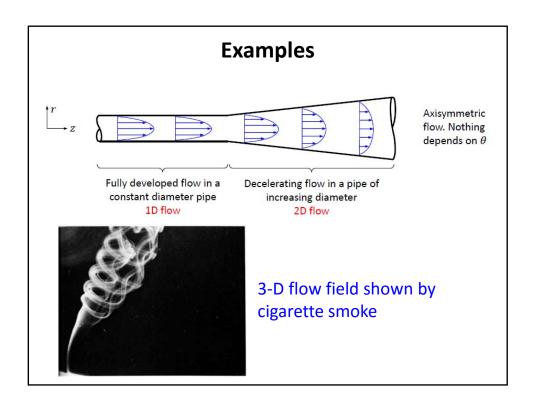
Examples



- Steady state operation of a water heater.
- Fluid heats up in the heater.
- $\partial T/\partial t$ of any fluid particle is zero, but dT/dt is not zero.
- ullet Convective derivative of T is not zero.



- Steady state uniform flow in a convergingdiverging nozzle.
- Fluid particles first accelerate and than decelerate
- $\partial u/\partial t$ of a fluid particle is zero, but du/dt is not zero
- Convective derivative of u is not zero.



Find the expression of
$$(\vec{u}.\nabla)\vec{u}$$
 where
$$\vec{u}(t,x,y,z) = u_1(t,x,y,z)\vec{i} - u_2(t,x,y,z)\vec{j} + u_3(t,x,y,z)\vec{k}$$

$$\vec{u}.\nabla = u_1\frac{\partial}{\partial x} + u_2\frac{\partial}{\partial y} + u_3\frac{\partial}{\partial z}$$

$$(\vec{u}.\nabla)\vec{u} = (\vec{u}.\nabla)u_1\vec{i} + (\vec{u}.\nabla)u_2\vec{j} + (\vec{u}.\nabla)u_3\vec{k}$$

$$(\vec{u}.\nabla)u_1 = \left(u_1\frac{\partial}{\partial x} + u_2\frac{\partial}{\partial y} + u_3\frac{\partial}{\partial z}\right)u_1 = u_1\frac{\partial u_1}{\partial x} + u_2\frac{\partial u_1}{\partial y} + u_3\frac{\partial u_1}{\partial z}$$

$$(\vec{u}.\nabla)u_2 = \left(u_1\frac{\partial}{\partial x} + u_2\frac{\partial}{\partial y} + u_3\frac{\partial}{\partial z}\right)u_2 = u_1\frac{\partial u_2}{\partial x} + u_2\frac{\partial u_2}{\partial y} + u_3\frac{\partial u_2}{\partial z}$$

$$(\vec{u}.\nabla)u_3 = \left(u_1\frac{\partial}{\partial x} + u_2\frac{\partial}{\partial y} + u_3\frac{\partial}{\partial z}\right)u_3 = u_1\frac{\partial u_3}{\partial x} + u_2\frac{\partial u_3}{\partial y} + u_3\frac{\partial u_3}{\partial z}$$

$$\begin{split} \left(\vec{u}.\nabla\right)\vec{u} &= \left(\vec{u}.\nabla\right)u_{1}\vec{i} + \left(\vec{u}.\nabla\right)u_{2}\vec{j} + \left(\vec{u}.\nabla\right)u_{3}\vec{k} \\ &= \left(u_{1}\frac{\partial u_{1}}{\partial x} + u_{2}\frac{\partial u_{1}}{\partial y} + u_{3}\frac{\partial u_{1}}{\partial z}\right)\vec{i} + \left(u_{1}\frac{\partial u_{2}}{\partial x} + u_{2}\frac{\partial u_{2}}{\partial y} + u_{3}\frac{\partial u_{2}}{\partial z}\right)\vec{j} \\ &+ \left(u_{1}\frac{\partial u_{3}}{\partial x} + u_{2}\frac{\partial u_{3}}{\partial y} + u_{3}\frac{\partial u_{3}}{\partial z}\right)\vec{k} \end{split}$$

Particular case $u_1 = 4tx, u_2 = -2t^2y, u_3 = 4xz$

$$(\vec{u}.\nabla)\vec{u} = (16t^2x)\vec{i} + (4t^4y)\vec{j} + (16txz + 16x^2z)\vec{k}$$

Find the acceleration of a particle at (-1,1,0) in the following velocity field

$$\vec{u} = u_1 \vec{i} + u_2 \vec{j} + u_3 \vec{k} \qquad u_1 = 4tx, u_2 = -2t^2 y, u_3 = 4xz$$
$$\vec{a} = \frac{d\vec{u}}{dt} = \frac{\partial \vec{u}}{\partial t} + (\vec{u}.\nabla)\vec{u}$$

$$\frac{\partial \vec{u}}{\partial t} = 4x\vec{i} - 4ty\vec{j} \qquad (\vec{u}.\nabla)\vec{u} = (16t^2x)\vec{i} + (4t^4y)\vec{j} + (16txz + 16x^2z)\vec{k}$$

Local acceleration

Convective acceleration

$$\vec{a} = (4x + 16t^2x)\vec{i} + (-4ty + 4t^4y)\vec{j} + (16txz + 16x^2z)\vec{k}$$

at point
$$(-1,1,0)$$
 $\vec{a} = -4(1+4t^2)\vec{i} - 4t(1-t^3)\vec{j}$