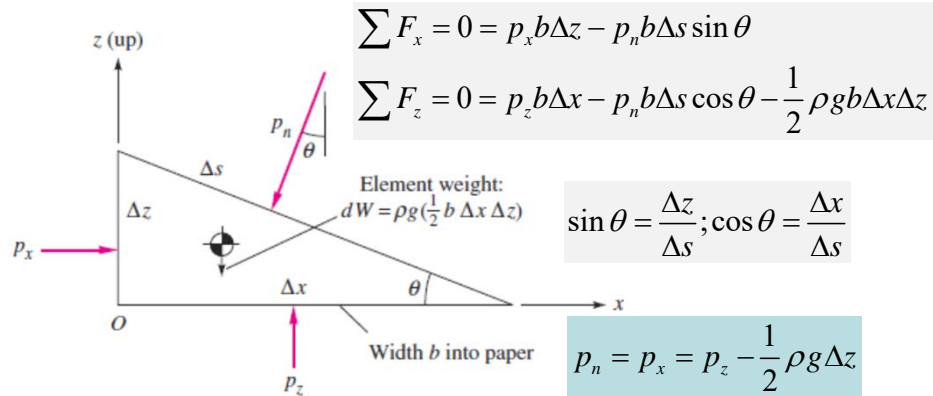
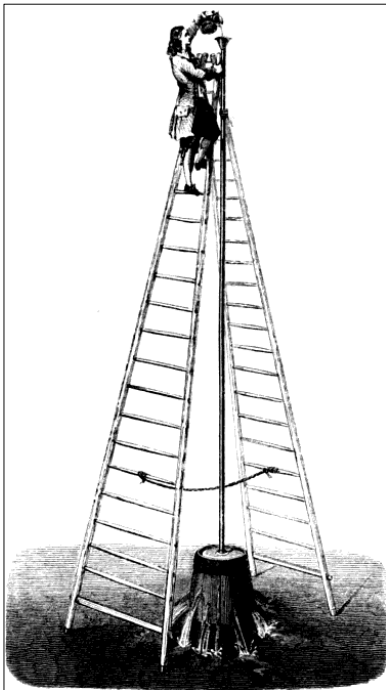


### Static equilibrium of a fluid element



For an infinitesimal fluid element  $\Delta z \rightarrow 0 \Rightarrow p_n = p_x = p_z$

**Pressure at a point is a scalar  
(direction-independent)**



Blaise Pascal (1623-1662)

Pascal's 'Barrel-Buster'  
experiment (~1646)

**Static equilibrium of a fluid element**

Element weight:  $dW = \rho g \left( \frac{1}{2} b \Delta x \Delta z \right)$

Width  $b$  into paper

**Surface force**

**Body force**

$$p_x(b\Delta z) - p_n(b\Delta s \sin \theta) = 0 = p_z(b\Delta x) - p_n(b\Delta s \cos \theta) - \rho g \left( \frac{1}{2} b \Delta x \Delta z \right)$$

**Static equilibrium:  $\Sigma(\text{surface forces} + \text{body forces}) = 0$**

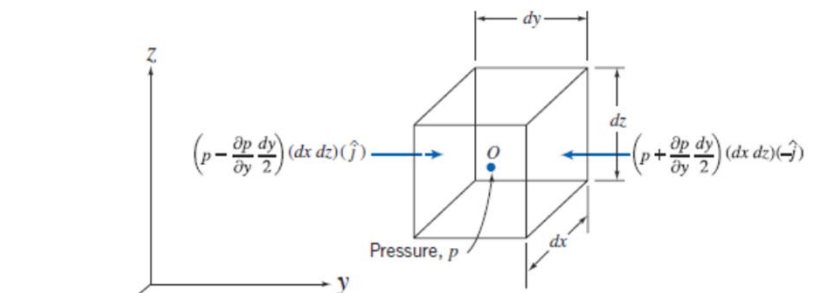
**As the size of the fluid element reduces, body force drops faster than surface force**

**Governing Equation of fluid statics**

Pressure,  $p$

**Pressure at faces may be found from Taylor series expansion (neglecting higher order terms)**

$$p_{y \pm dy/2} = p \pm \frac{\partial p}{\partial y} \frac{dy}{2} + \dots \text{higher order terms}$$



Net **surface force** in y-direction

$$d\vec{F}_{sy} = \left( p - \frac{\partial p}{\partial y} \frac{dy}{2} \right) dx dz (\hat{j}) + \left( p + \frac{\partial p}{\partial y} \frac{dy}{2} \right) dx dz (-\hat{j}) = -\frac{\partial p}{\partial y} \hat{j} dV$$

Total surface force, considering all three directions:

$$d\vec{F}_s = -\left( \frac{\partial p}{\partial x} \hat{i} + \frac{\partial p}{\partial y} \hat{j} + \frac{\partial p}{\partial z} \hat{k} \right) dV = -\nabla p dV$$

**Body force** (weight) of the fluid element  $d\vec{F}_B = \rho \vec{g} dV$

Total force on the fluid element

$$d\vec{F} = d\vec{F}_s + d\vec{F}_B = -\nabla p dV + \rho \vec{g} dV$$

For a static fluid element  $d\vec{F} = 0 \Rightarrow -\nabla p + \rho \vec{g} = 0$

**Governing Equation of fluid statics**  $\nabla p = \rho \vec{g}$

**Note:** The above concept can be extended for rigid body motion as well, but we will skip that discussion here (interested students may see the texts)

## Fluid Statics

# Manometry

Principles of fluid statics used in pressure measurement

M. K. Das, mkdas@iitk.ac.in

## Governing Equation of fluid statics: $\nabla p = \rho \vec{g}$

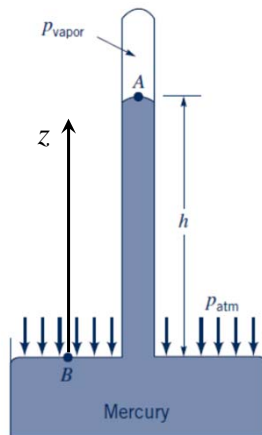
$$x\text{-direction: } \frac{\partial p}{\partial x} = \rho g_x \quad y\text{-direction: } \frac{\partial p}{\partial y} = \rho g_y \quad z\text{-direction: } \frac{\partial p}{\partial z} = \rho g_z$$

Sometimes we align one axis (let's say  $z$ ) vertically upward (against gravity), such that  $g_x = g_y = 0; g_z = -g$

$$\Rightarrow \frac{\partial p}{\partial x} = \frac{\partial p}{\partial y} = 0 \Rightarrow p = p(z) \quad \frac{\partial p}{\partial z} = -\rho g = -\gamma$$

**Integrating**  $p(z=h) = p(z=0) - \gamma h$  for constant  $\rho$

Above Eq. shows that measurement of length can help us to know the pressure difference; briefly, this is the **principle of manometry**



## Barometer

$$p_{\text{vap}} = p_{\text{atm}} - \rho gh$$

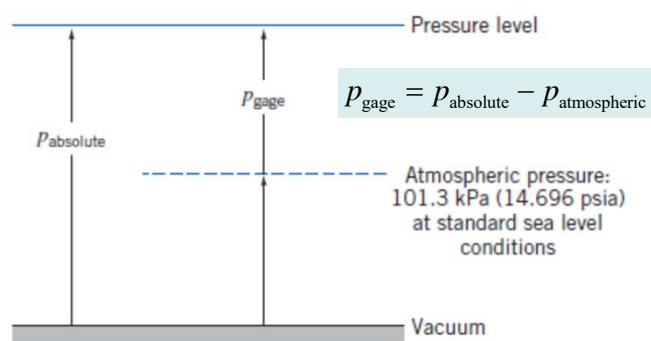
$$p_{\text{vap}} \ll p_{\text{atm}} \Rightarrow p_{\text{atm}} = \rho gh$$

Barometer is the primitive form of manometer, more involved designs are there

$$p(z=h) = p(z=0) - \rho gh$$

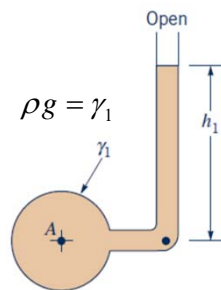
Instead of mercury vapor pressure (or vacuum), most gages use ambient pressure as base pressure

## Absolute and gage pressures



Pressure gages usually show gage pressure

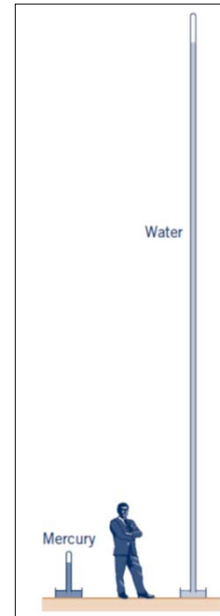
## Simplest manometer (piezometer)



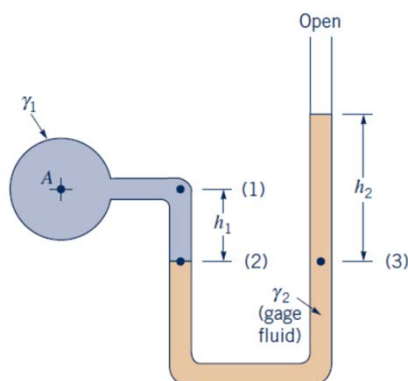
$$p_{A, \text{absolute}} = p_{\text{atm}} + \rho g h_1$$

$$p_{A, \text{gage}} = \rho g h_1$$

Piezometer contains the same fluid as the container A, this could be a serious problem for practical applications



## U-tube manometer



Analysis key: start from one end !!

$$p_{A, \text{absolute}} + \gamma_1 h_1 - \gamma_2 h_2 = p_{\text{atm}}$$

$$p_{A, \text{gage}} = \gamma_2 h_2 - \gamma_1 h_1$$

To obtain gage pressure directly, put atmospheric pressure to be zero

Several other manometer types will be discussed in tutorials/discussions