



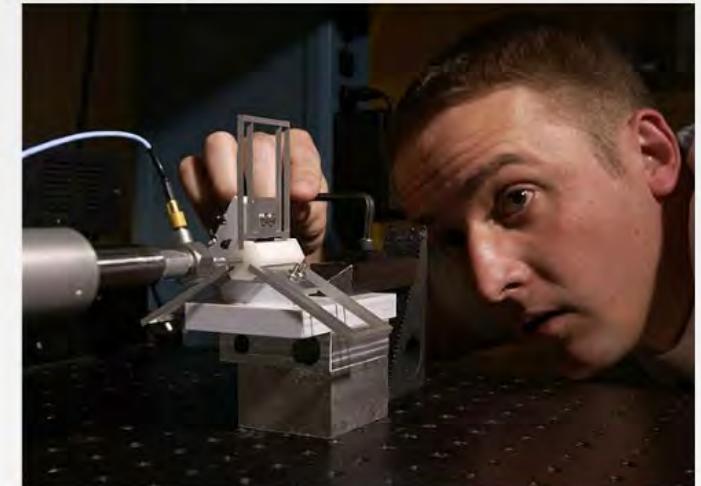
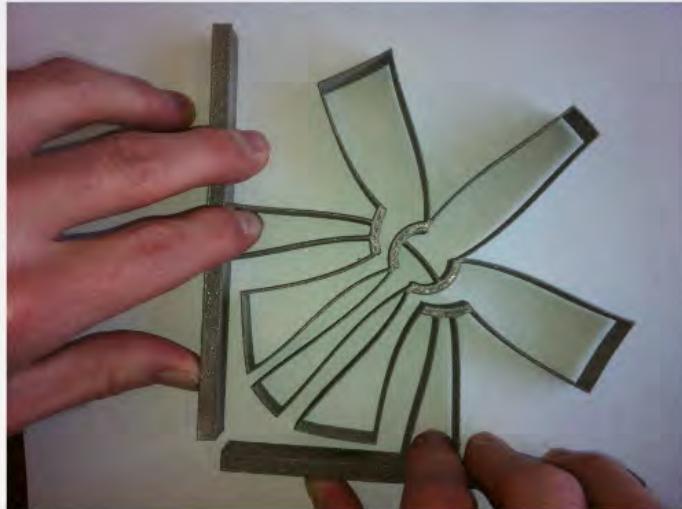
Compliant Mechanisms

Larry L. Howell
Brigham Young University



BRIGHAM YOUNG UNIVERSITY
COMPLIANT MECHANISMS RESEARCH

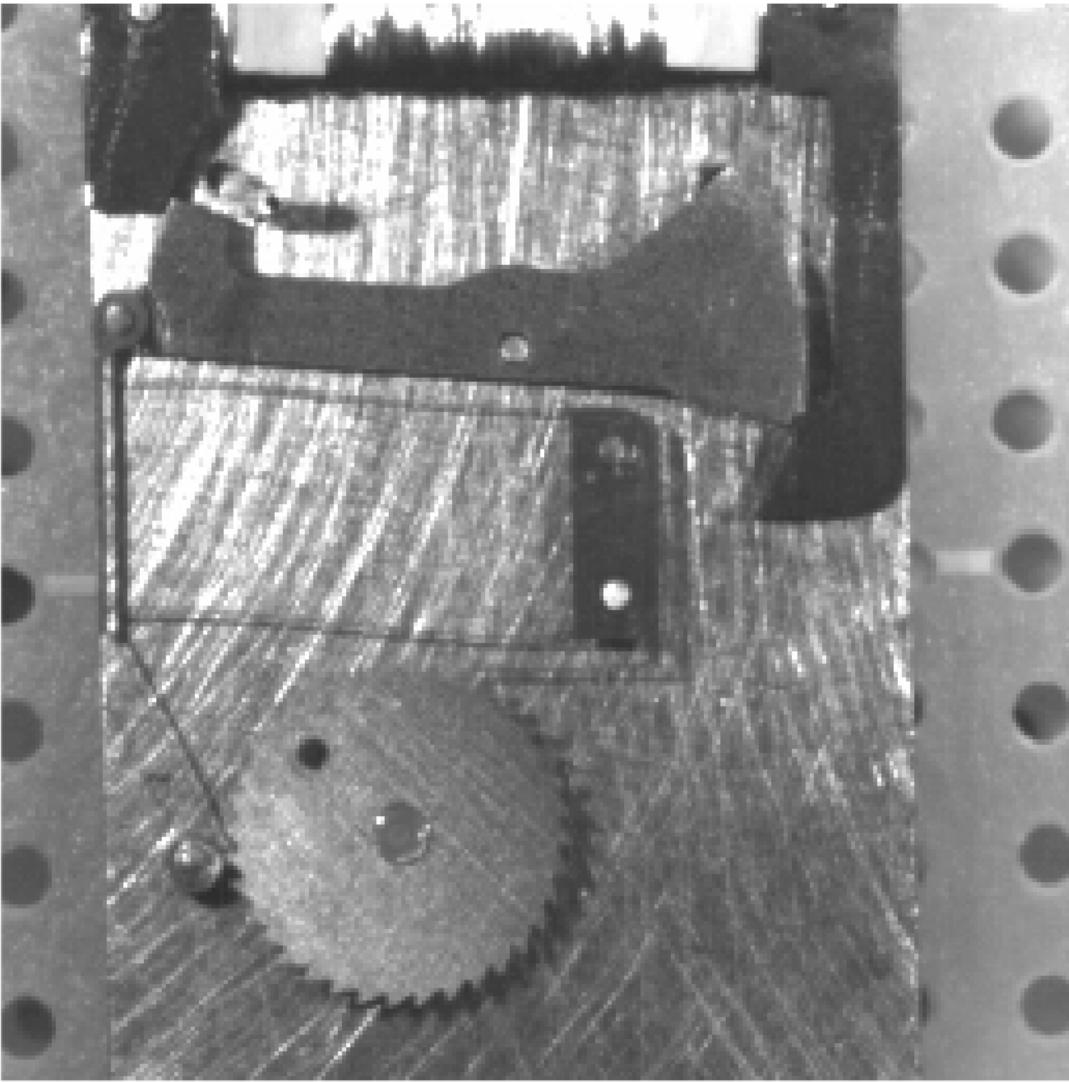
Compliant Mechanisms



A compliant mechanism gains some or all of its motion from the deflection of flexible members



Low cost
Minimal assembly



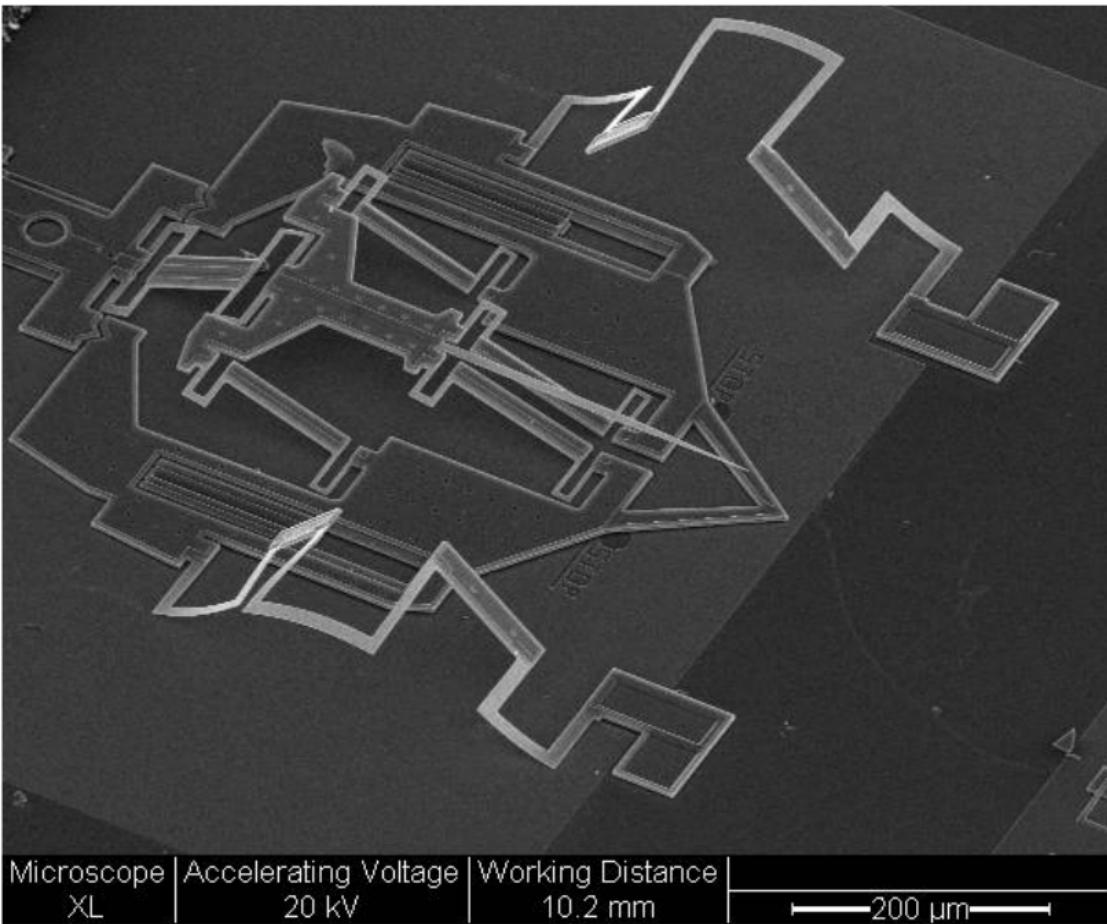
Compact
High precision



Reduced wear
Harsh environments



Light weight
Tailored force response



Microscope | Accelerating Voltage | Working Distance | — 200 μ m —
XL | 20 kV | 10.2 mm

Easily minaturized

One key thing to remember

Stiffness and
strength are

NOT

the same thing

It is possible to make
something both

Flexible

and

Strong

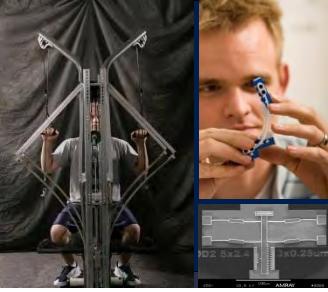
Pseudo-Rigid-Body Model



Larry L. Howell
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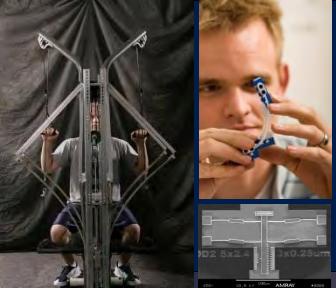
BYU CMR

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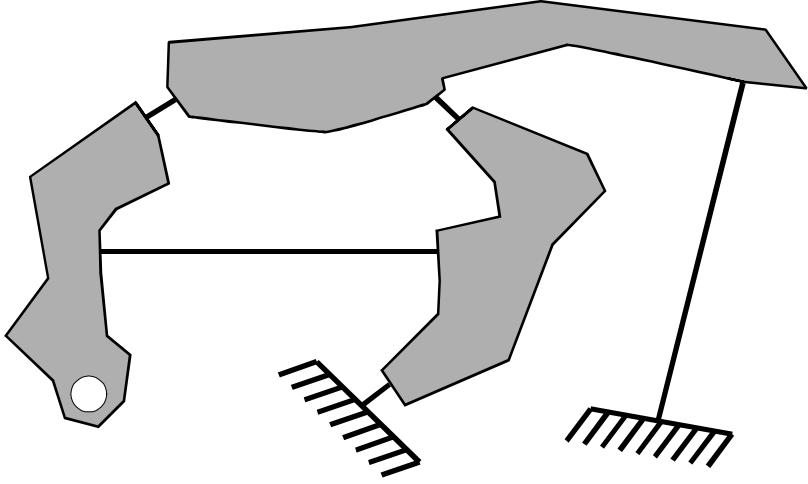
Pseudo-Rigid-Body Model

- Models compliant mechanisms as rigid-body mechanisms
- Allows use of decades of research in mechanical systems
- Unifies compliant mechanism and rigid-body mechanism theories



21ST CENTURY
KIN3
MATC5

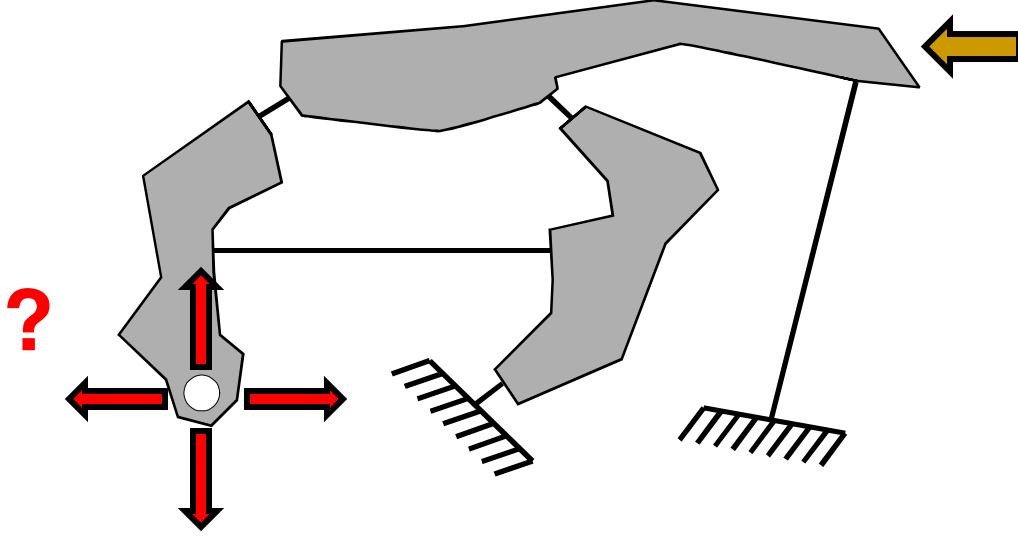
Example





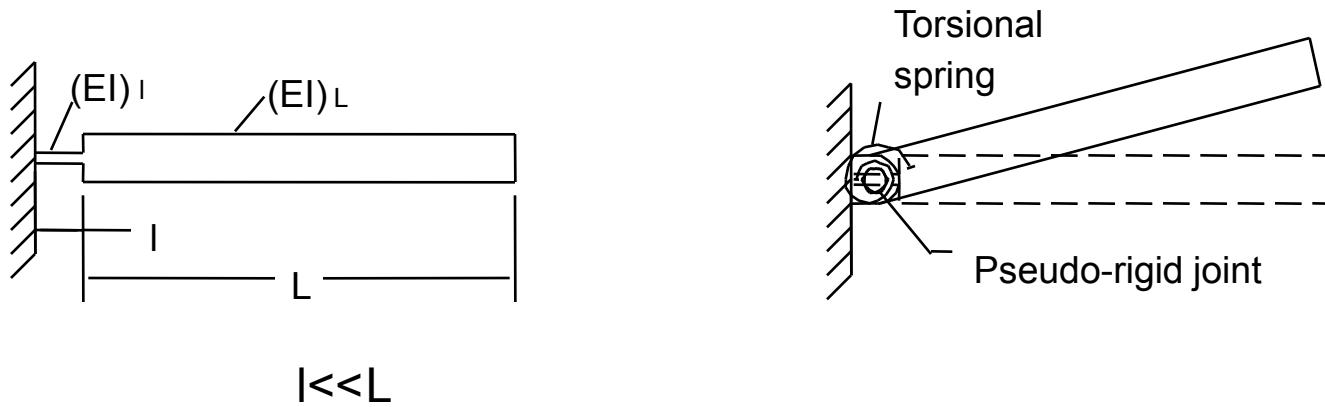
21ST CENTURY
KIN3
MATC5

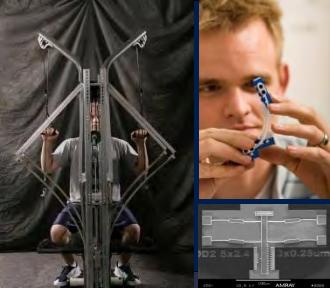
Example





PRBM: Small-Length Flexural Pivot





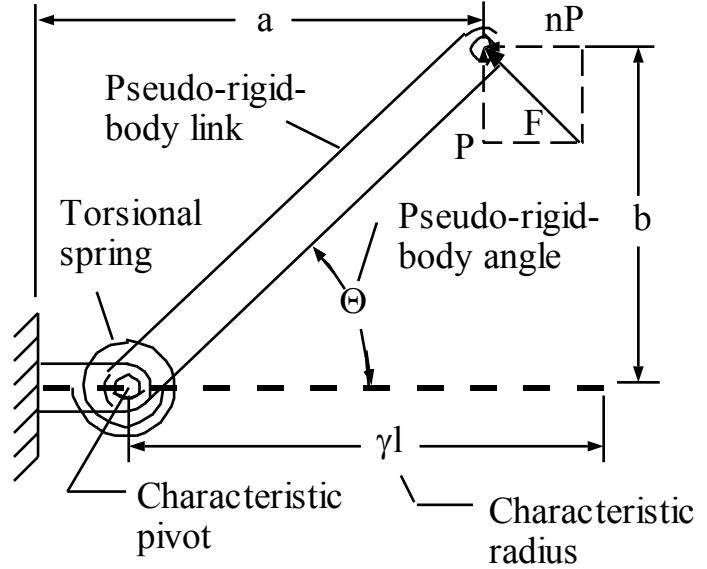
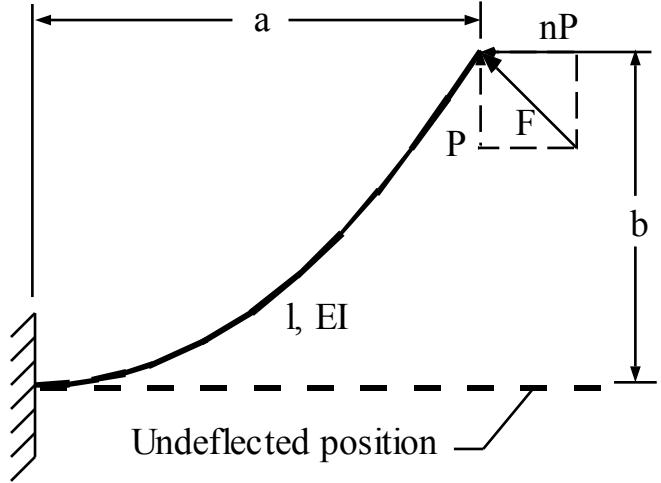
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KING
MAGIC5

Living Hinges

- Living hinge: extremely short and thin small-length flexural pivots
- PRBM is a pin joint at the center of the flexible segment.
- If other compliant elements are present, then can ignore spring for living hinge

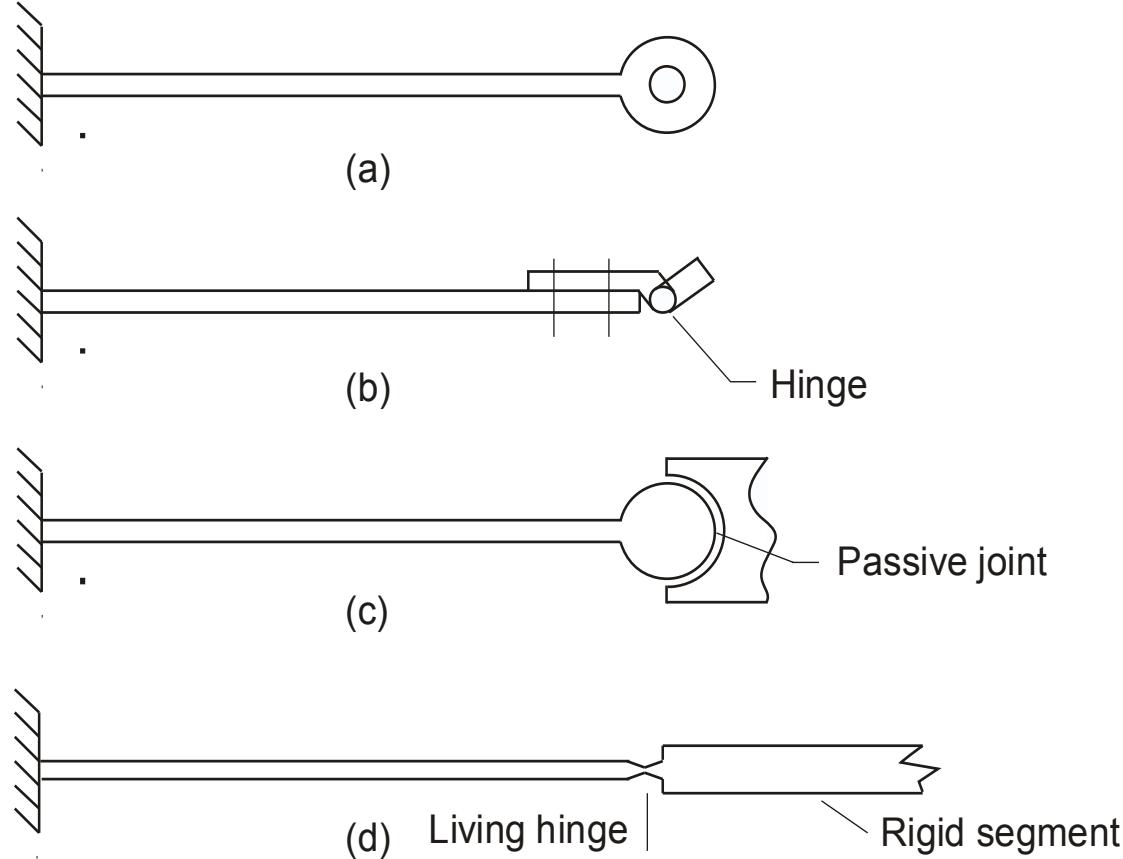


PRBM: Fixed-Pinned



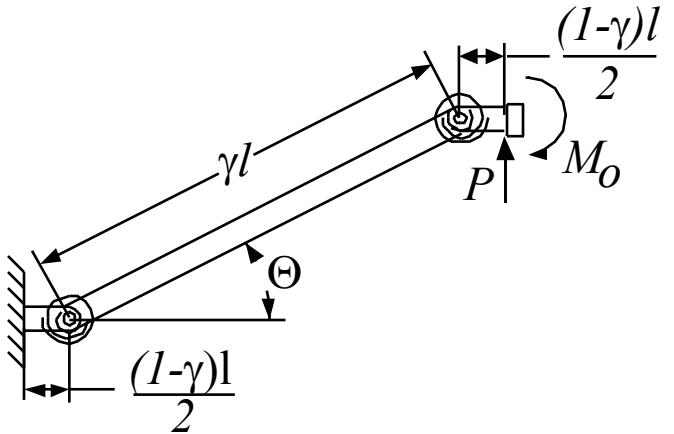
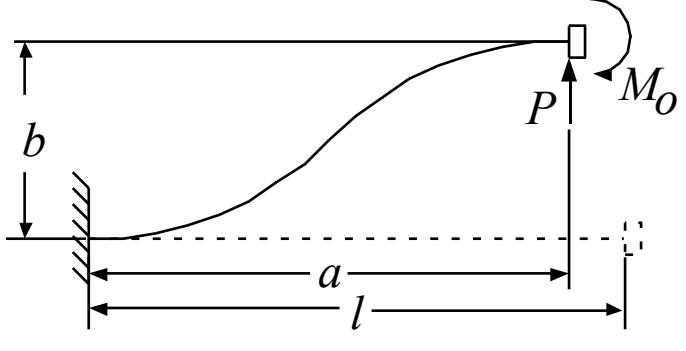


Practical Implementation



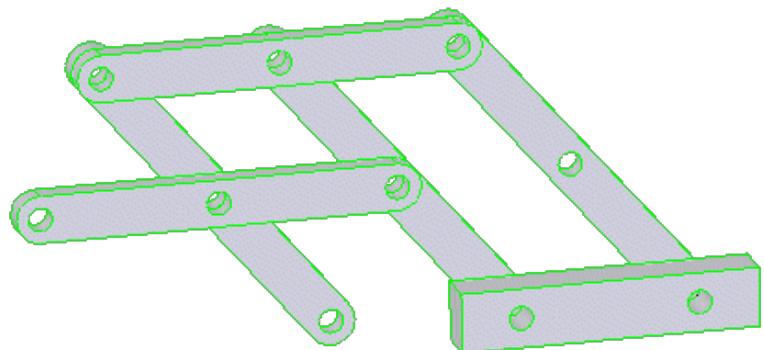
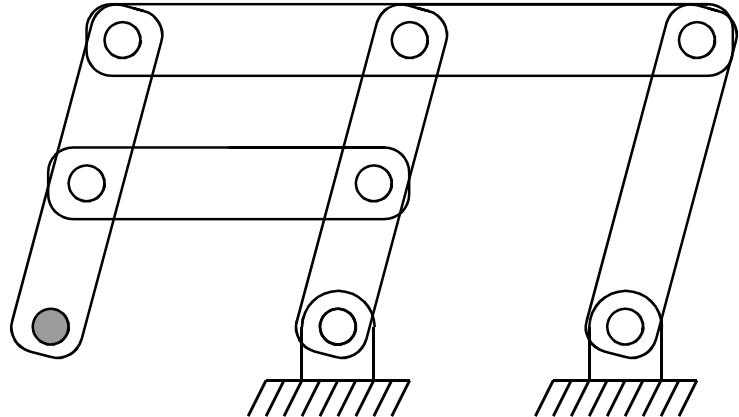
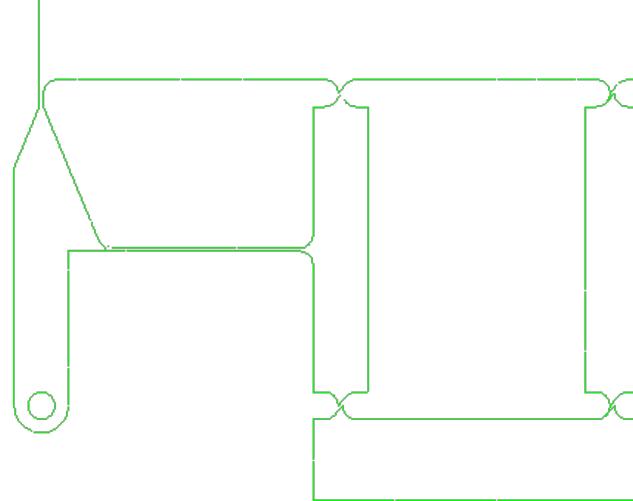
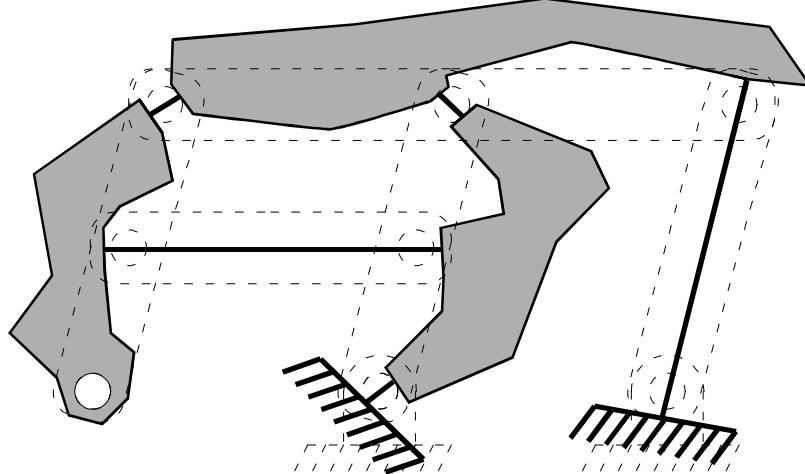


PRBM: Fixed-Guided





Example





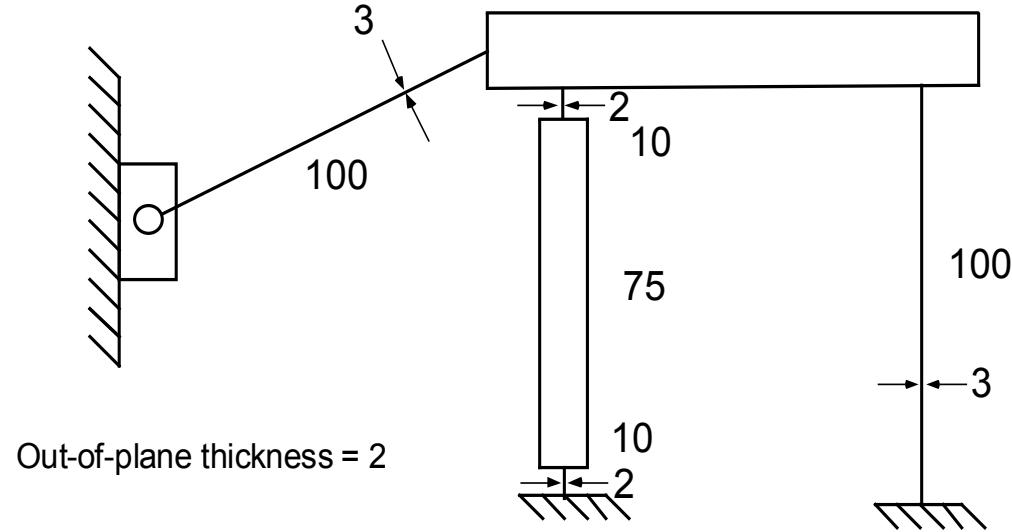
Exercise

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- (a) Sketch the PRBM
 - (b) Calculate the lengths of the links
 - (c) Write equations for spring constants symbolically
 - (d) Calculate numerical values of spring constants

Assume dimensions in mm and material is Aluminum ($E=72$ GPa)

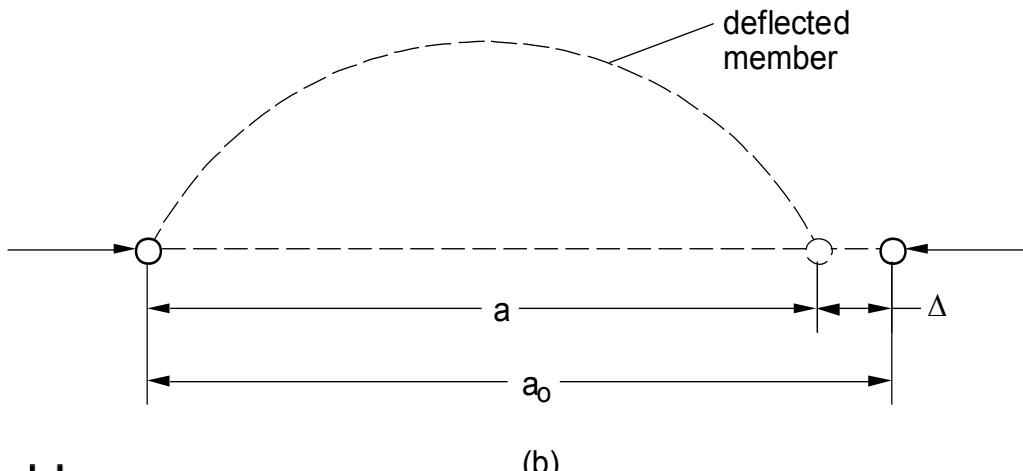
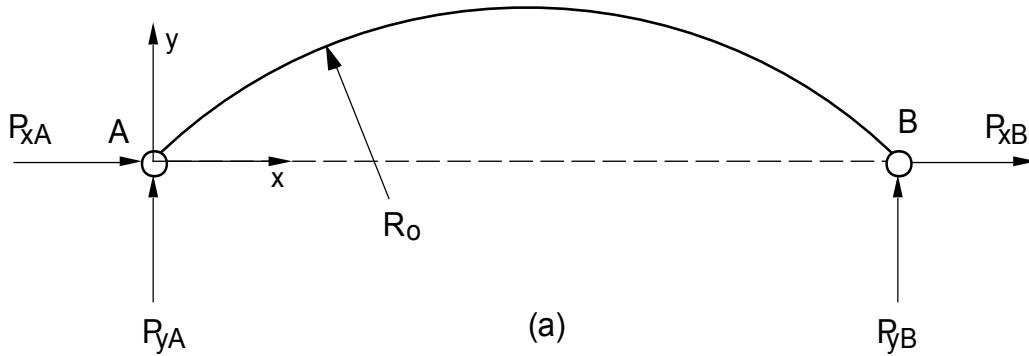
$$I = bh^3/12$$





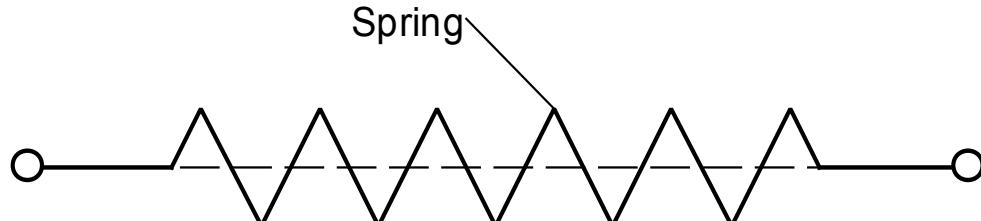
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PRBM: Pinned-pinned

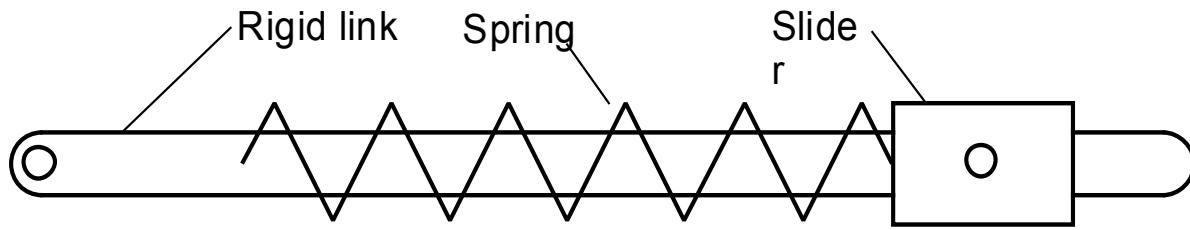




PRBM: Pinned-pinned



(a)



(b)



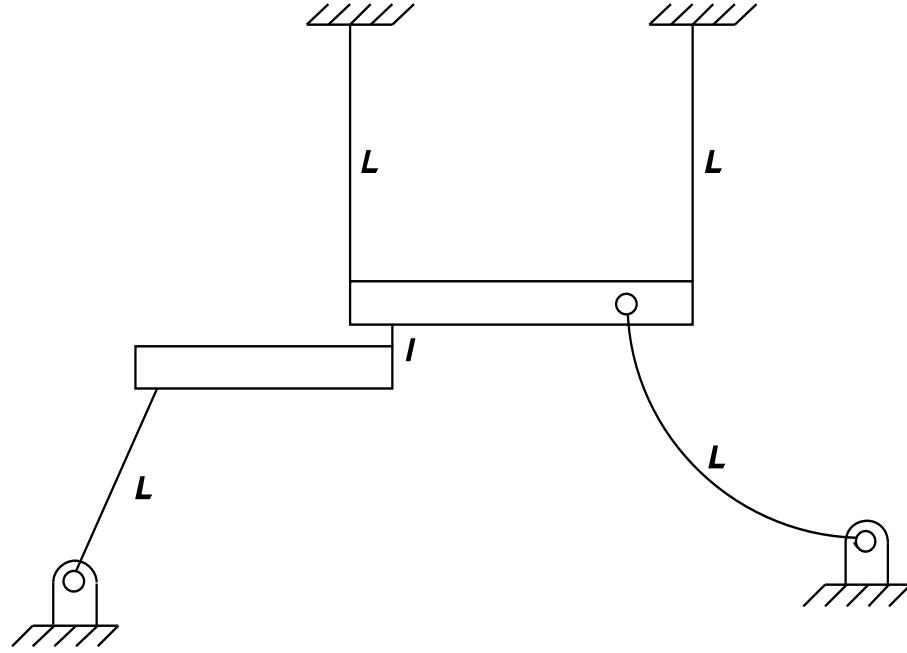
Other Pseudo-rigid-body Models



- Pure moment load
- Initially curved beam
- Other

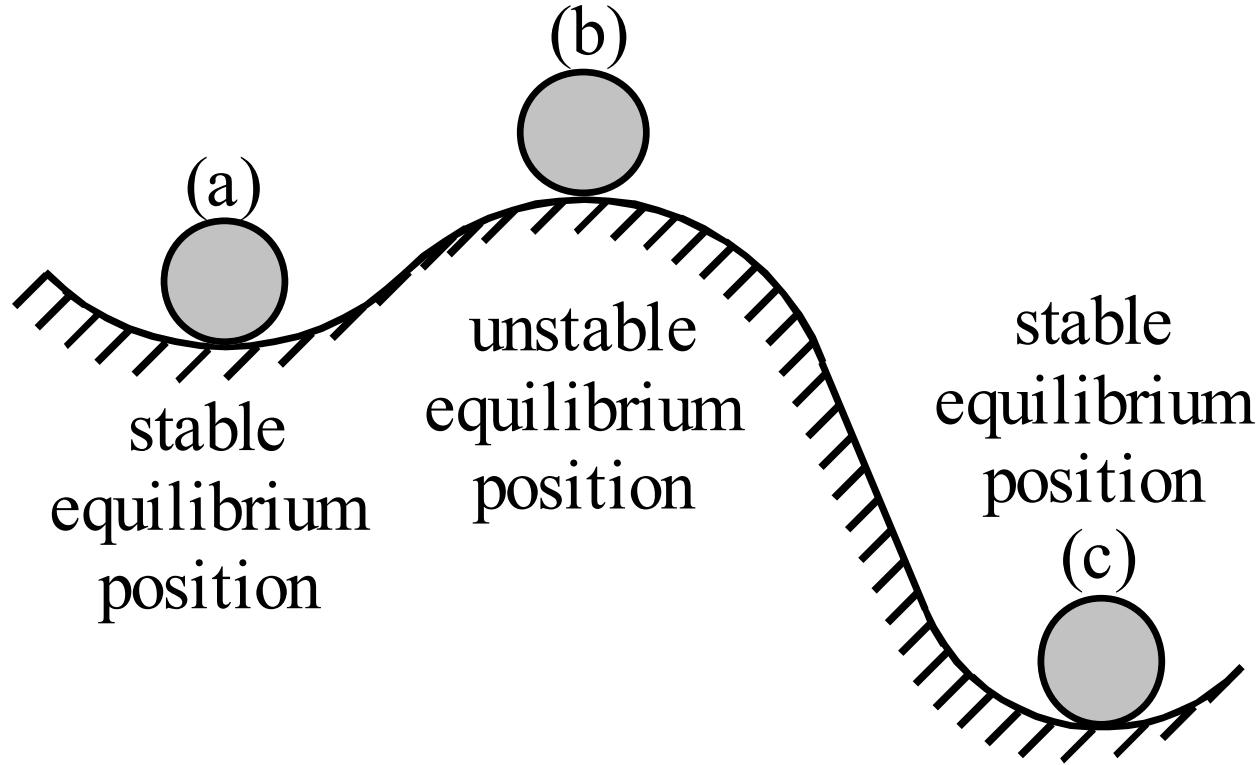


Exercise



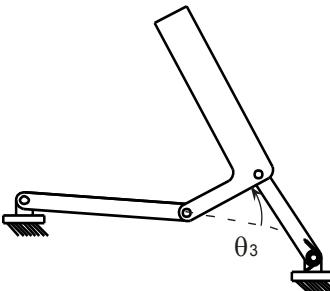
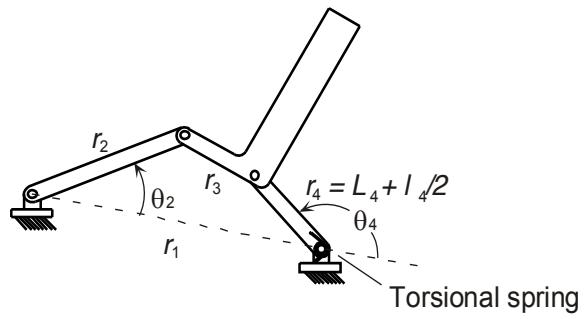
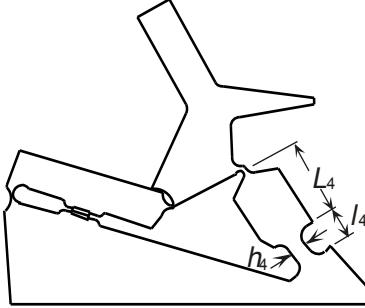
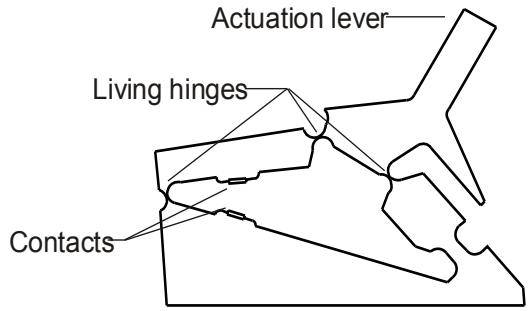


Example: Bistable Switch





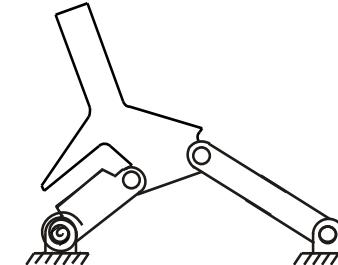
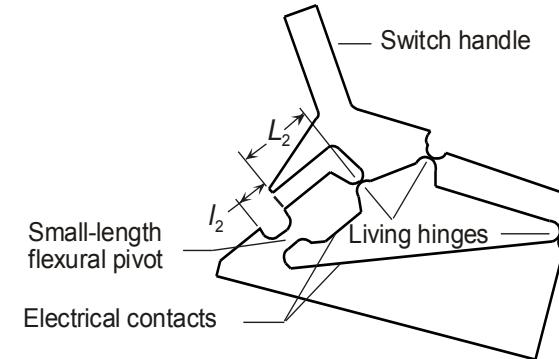
Example





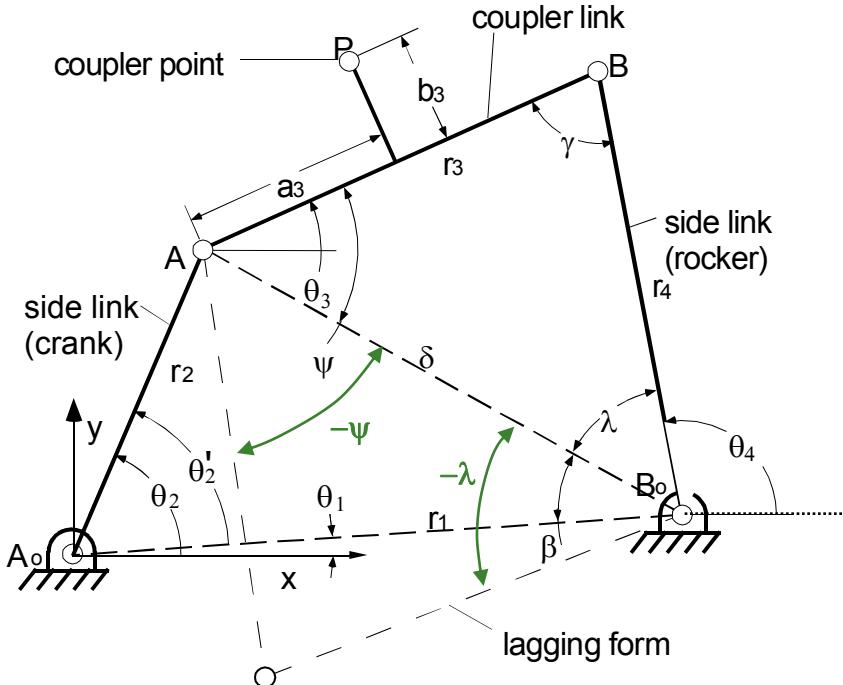
Example

- The pseudo-rigid-body model is a four-bar mechanism
- The potential energy is a function of the deflection of the torsional spring
- PRBM provides simple model that allows the design of needed position and force control





Position Analysis: 4-bar



The lagging (crossed) form is determined by $-\psi$ and $-\lambda$, using the second solutions from the $\cos^{-1}()$ equations.

$$\delta = \sqrt{r_1^2 + r_2^2 - 2r_1r_2 \cos \theta_2}; \quad \beta = \cos^{-1} \frac{r_1^2 + \delta^2 - r_2^2}{2r_1\delta}$$

$$\psi = \cos^{-1} \frac{r_3^2 + \delta^2 - r_4^2}{2r_3\delta}; \quad \lambda = \cos^{-1} \frac{r_4^2 + \delta^2 - r_3^2}{2r_4\delta}$$

For $0 \leq \theta_2 \leq \pi$

$$\theta_3 = \psi - (\beta - \theta_1); \quad \theta_4 = \pi - \lambda - (\beta - \theta_1)$$

For $\pi \leq \theta_2 \leq 2\pi$

$$\theta_3 = \psi + (\beta + \theta_1); \quad \theta_4 = \pi - \lambda + (\beta + \theta_1)$$

also

$$\gamma = \pm \cos^{-1} \frac{r_3^2 + r_4^2 - \delta^2}{2r_3r_4}$$

$$x_p = r_2 \cos \theta_2 + a_3 \cos \theta_3 - b_3 \sin \theta_3$$

$$y_p = r_2 \sin \theta_2 + a_3 \sin \theta_3 + b_3 \cos \theta_3$$



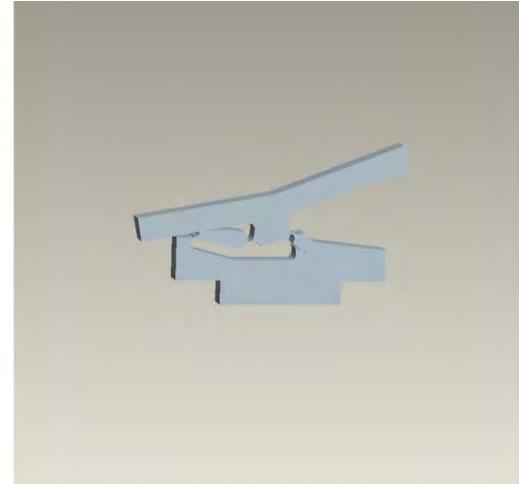
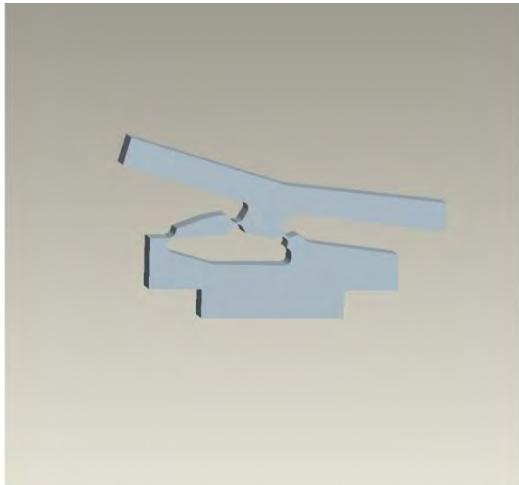
Parametric Models

- Powerful design tool
- Analyze many different designs quickly
- Integration with optimization tools
- Convert between different configurations



Example: Rocker Switch

- Same parametric models apply to move from a toggle switch to a rocker switch





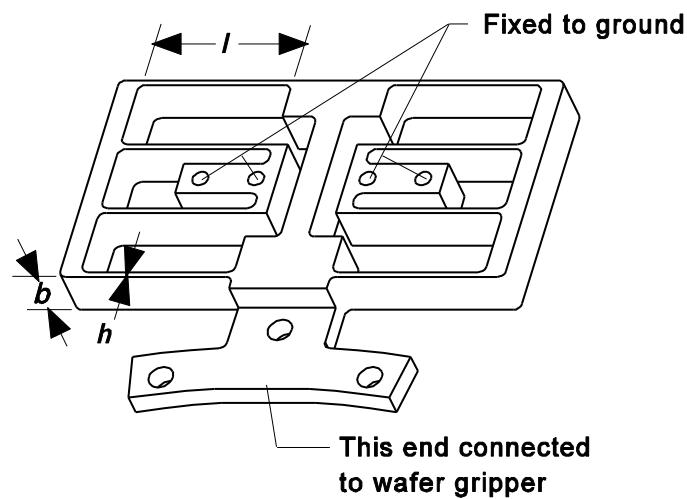
PRBM with CAE tools

- Spreadsheets, Matlab, etc
 - Switch example

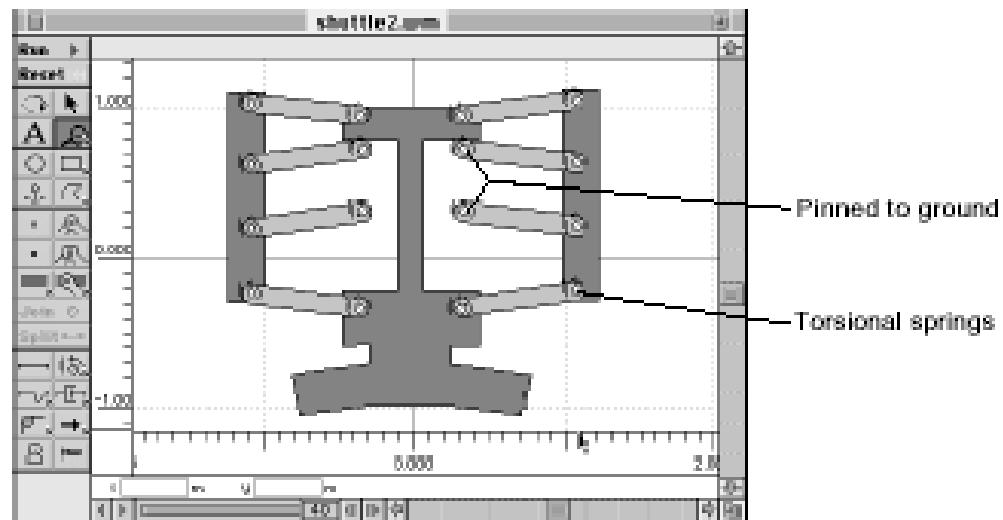


PRBM with CAE tools

- Multi-body dynamics tools (ADAMS, etc.)
 - Examples
 - folded-beam suspension
 - switch



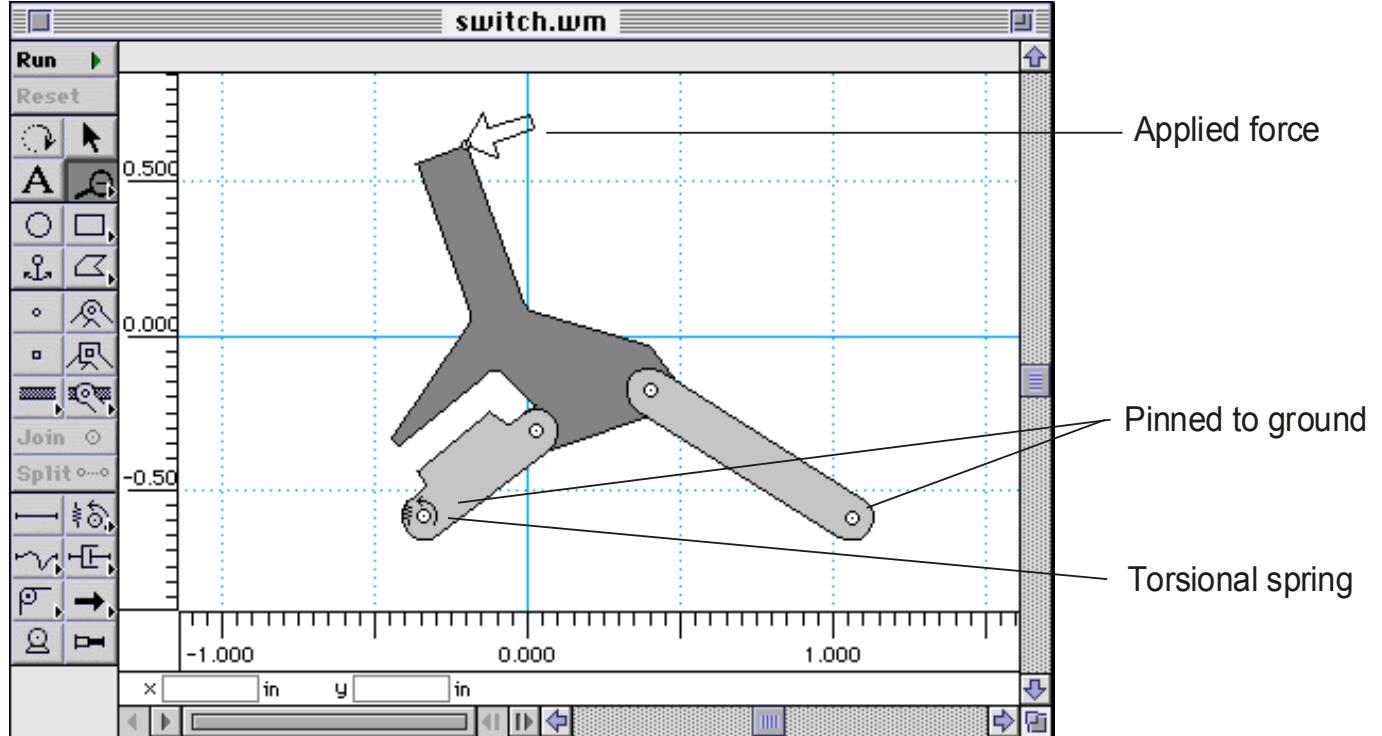
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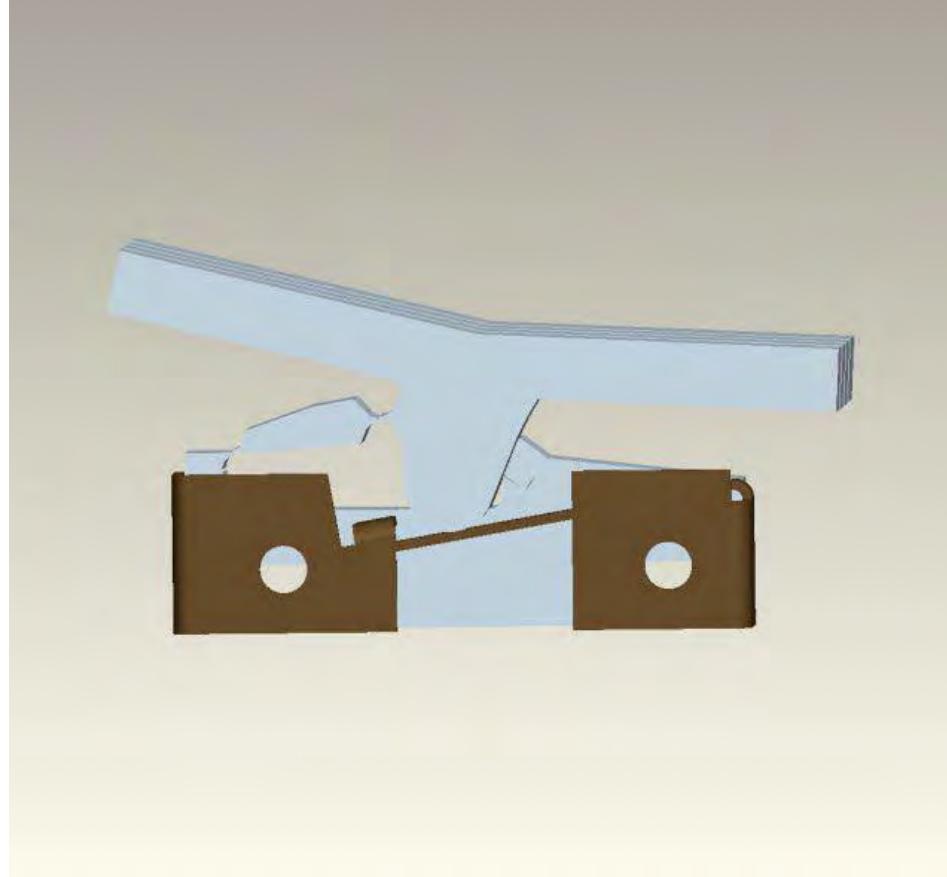
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Example: Switch





Rocker Switch



So why compliant
mechanisms now?

Computational capabilities

Materials and processes

Design methods

and . . .

New needs

High performance

- weight
- friction and wear
- precision

Size domains

- meso
- micro
- nano

Critical applications

- biomedical
- space
- economic

Cost

- part count
- assembly
- manufacturing

New motions

- morphing
- lamina emergent
- adaptive

High performance

- weight
- friction and wear
- precision

Size domains

- meso
- micro
- nano

Critical applications

- biomedical
- space
- economic

Co

-
-
-

Cost

- part count
- assembly
- manufacturing

New motions

- morphing
- lamina emergent
- adaptive



21st Century Compliant Mechanisms

Biomedical Implants



Microelectromechanical
Systems (MEMS)

Biomedical Implants



FlexBAC



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Biomedical Implants



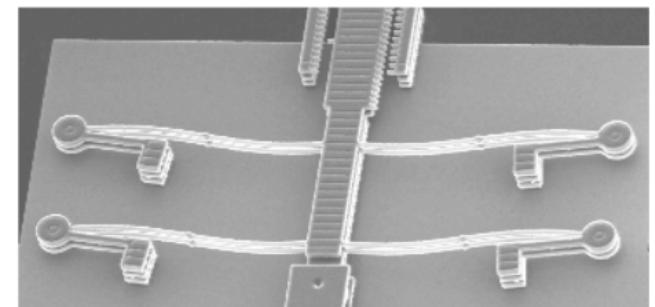
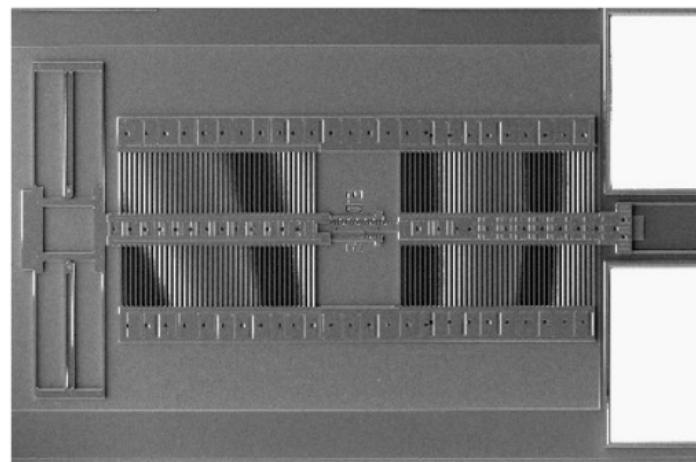
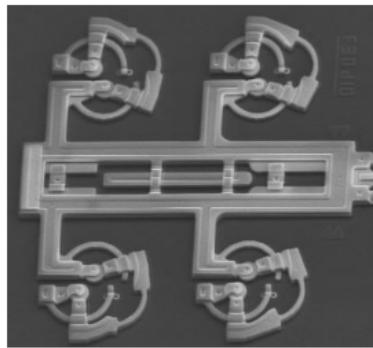
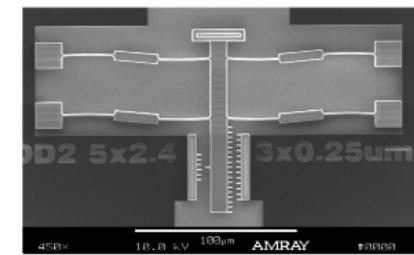
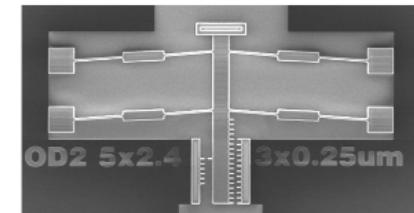
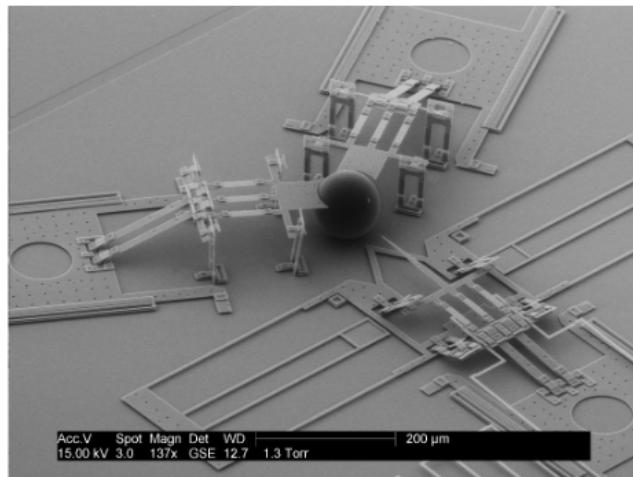
FlexBAC

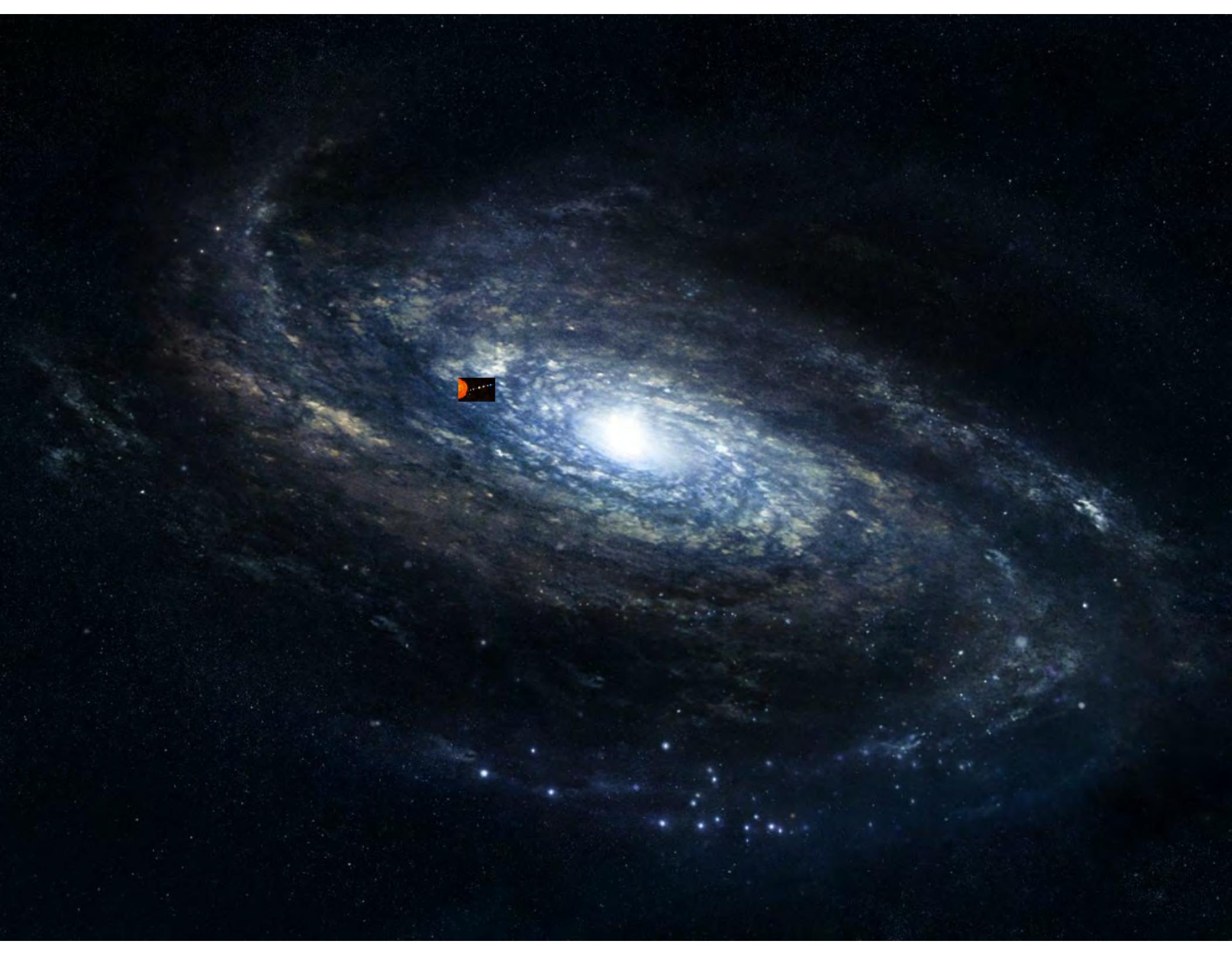


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Microelectromechanical Systems (MEMS)

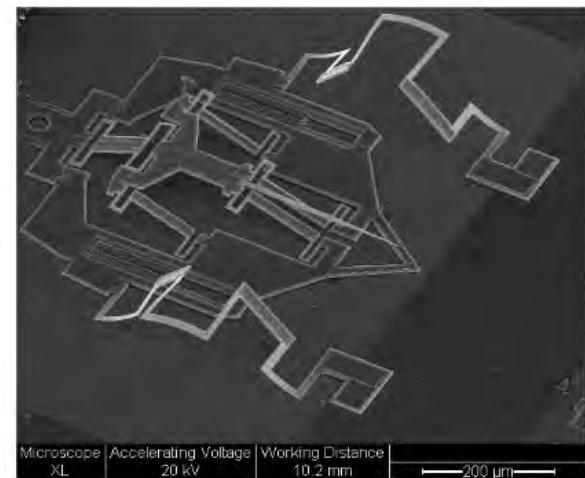




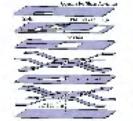
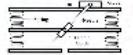
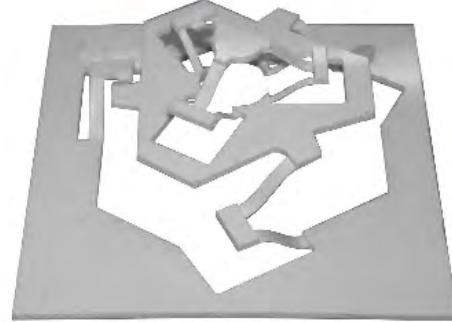
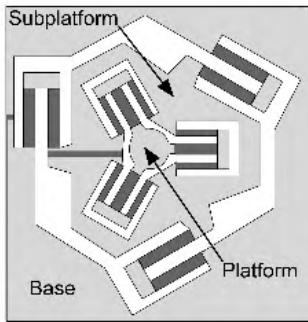
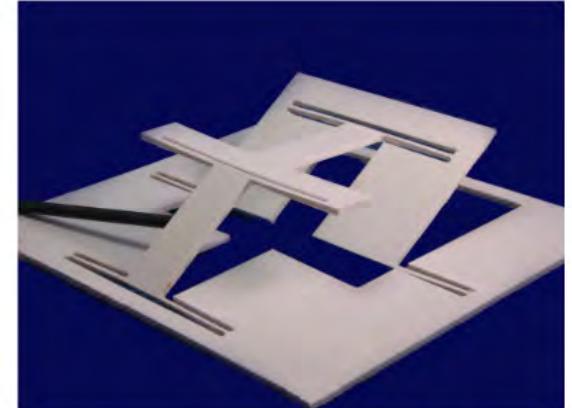
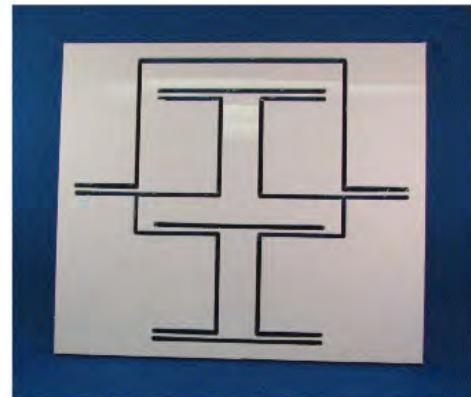
Engineering Tools of Scientific Discovery



The Grand Challenges of Engineering



Lamina Emergent Mechanisms



What do you think is next?

Hypercompact Mechanisms

Adaptive Morphing Systems

Disruptive Innovations

More Human-like Implants

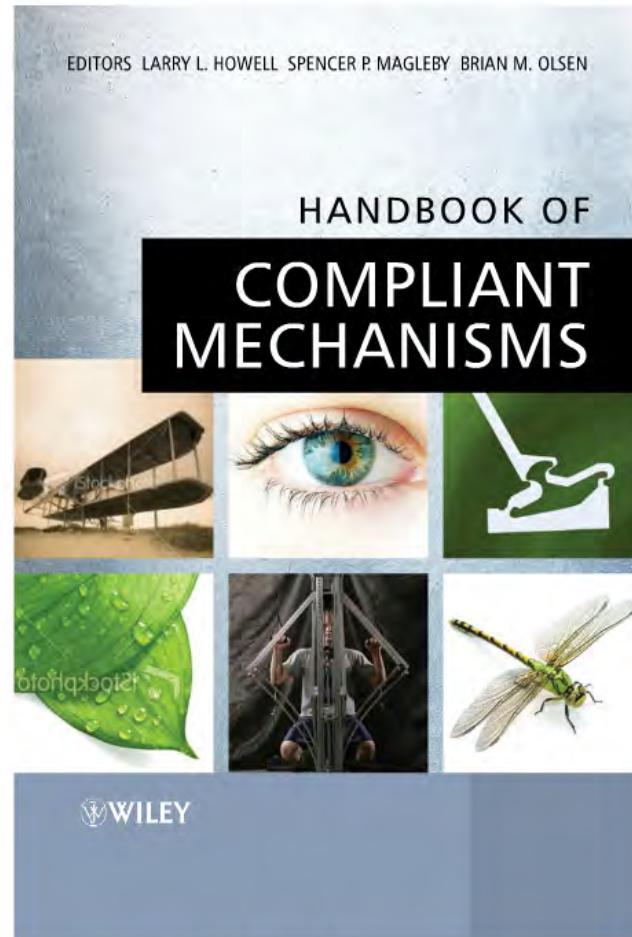
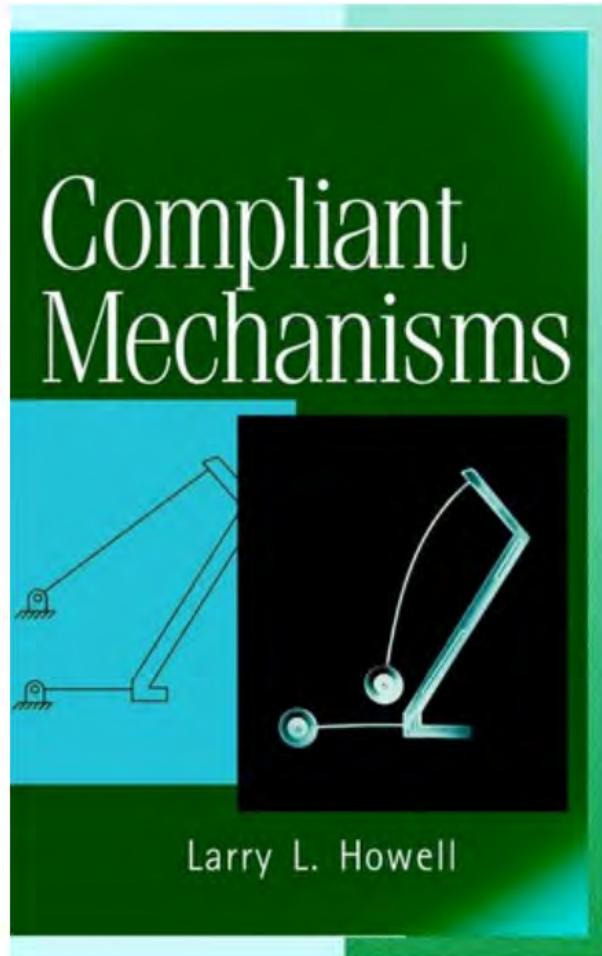
Nanomachines

Advanced Materials

Human-Robot Interactions

Products using Local Materials

Resources



Available soon

Acknowledgements

National Science Foundation

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NanoInjection Technologies, LLC

Compliant Mechanisms Research Group (CMR)

BYU Applied Biomechanics Engineering Laboratory (BABEL)

BYU Department of Microbiology & Molecular Biology

Collaborators:

- Prof. Spencer Magleby
- Prof. Brian Jensen
- Prof. Anton Bowden
- Prof. Sandra Burnett



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BYU
Microbiology &
Molecular Biology

Thank you!

