

## Kinematics and Algebraic Geometry

Manfred L. Hustý,  
Hans-Peter  
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# Kinematics and Algebraic Geometry

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Workshop on 21<sup>st</sup> Century Kinematics, Chicago 2012



# Outline of Lecture

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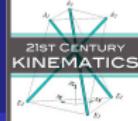
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**Computational Kinematics** is that branch of kinematics which involves intensive computations not only of numerical type but also of symbolic nature (Angeles 1993).

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- Within CK one tries to answer fundamental questions arising in the ***analysis and synthesis of kinematic chains.***

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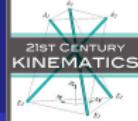
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- The fundamental questions, going far beyond the classical kinematics involve the number of solutions, complex or real to, for example, ***forward or inverse kinematics***, the description of ***singular solutions***, the mathematical solution of ***workspace or synthesis*** questions.



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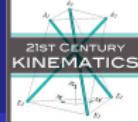
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- Such problems are often described by ***systems of multivariate algebraic or functional equations*** and it turns out that even relatively simple kinematic problems involving multi-parameter systems lead to complicated nonlinear equations.



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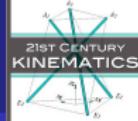
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- Such problems are often described by ***systems of multivariate algebraic or functional equations*** and it turns out that even relatively simple kinematic problems involving multi-parameter systems lead to complicated nonlinear equations.
- Geometric insight and geometric preprocessing are often key to the solution



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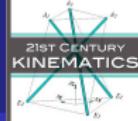
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Analytic description of kinematic chains:



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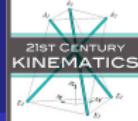
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Analytic description of kinematic chains:

- Parametric and implicit representations



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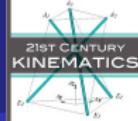
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Analytic description of kinematic chains:

- Parametric and implicit representations
- Different parametrizations of the displacement group  $SE(3)$  (Euler angles, Rodrigues parameters, Euler parameters, Study parameters, quaternions, dual quaternions)



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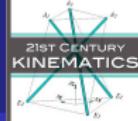
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- Most the time vector loop equations are used to describe the chains
- Very often only a single numerical solution is obtained



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- Complete analysis and synthesis needs all solutions



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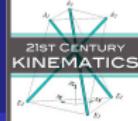
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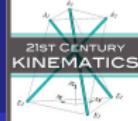
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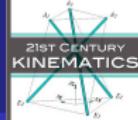
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- Geometric and algebraic preprocessing is needed before elimination, Gröbner base computation or numerical solution process starts
- Algebraic constraint equations yield answers to the overall behavior of a kinematic chain → **Global Kinematics**



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D. Walter



M. Pfurner



F. Pernkopf



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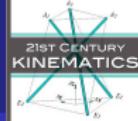
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Mehdi Tale Masouleh, Clément Gosselin (Laval University, Quebec City)

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P. Zsombor-Murray, M. J. D. Hayes (McGill, Montreal)

A. Karger (Charles University Prag, Czech Republic)



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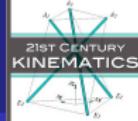
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- Some algebraic basics of kinematics



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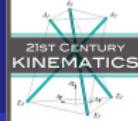
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- Some algebraic basics of kinematics
- How algebraic constraint equations can be obtained from parametric equations involving sines and cosines



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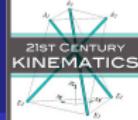
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- How freedom of mechanisms can be formulated within this frame



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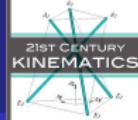
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- How freedom of mechanisms can be formulated within this frame
- How the same equations can be used for analysis and synthesis



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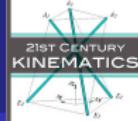
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- How freedom of mechanisms can be formulated within this frame
- How the same equations can be used for analysis and synthesis
- How singularities can be obtained within the algebraic formulation
- How this framework can be used for the analysis of lower dof parallel manipulators



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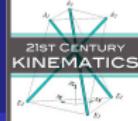
Euclidean displacement:

$$\gamma: \mathbb{R}^3 \rightarrow \mathbb{R}^3, \quad \mathbf{x} \mapsto \mathbf{Ax} + \mathbf{a} \quad (1)$$

$\mathbf{A}$  proper orthogonal  $3 \times 3$  matrix,  $\mathbf{a} \in \mathbb{R}^3$  ... vector

group of Euclidean displacements: SE(3)

$$\begin{bmatrix} 1 \\ \mathbf{x} \end{bmatrix} \mapsto \begin{bmatrix} 1 & \mathbf{o}^T \\ \mathbf{a} & \mathbf{A} \end{bmatrix} \cdot \begin{bmatrix} 1 \\ \mathbf{x} \end{bmatrix}. \quad (2)$$



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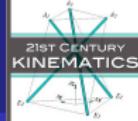
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Study's kinematic mapping  $\varkappa$ :

$$\varkappa: \alpha \in \text{SE}(3) \mapsto \mathbf{x} \in \mathbb{P}^7$$



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pre-image of  $\mathbf{x}$  is the displacement  $\alpha$

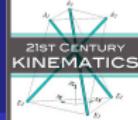
$$\frac{\Delta}{\Delta} \begin{bmatrix} \Delta & 0 & 0 \\ p & x_0^2 + x_1^2 - x_2^2 - x_3^2 & 2(x_1x_2 - x_0x_3) & 2(x_1x_3 + x_0x_2) \\ q & 2(x_1x_2 + x_0x_3) & x_0^2 - x_1^2 + x_2^2 - x_3^2 & 2(x_2x_3 - x_0x_1) \\ r & 2(x_1x_3 - x_0x_2) & 2(x_2x_3 + x_0x_1) & x_0^2 - x_1^2 - x_2^2 + x_3^2 \end{bmatrix} \quad (3)$$

$$p = 2(-x_0y_1 + x_1y_0 - x_2y_3 + x_3y_2),$$

$$q = 2(-x_0y_2 + x_1y_3 + x_2y_0 - x_3y_1), \quad (4)$$

$$r = 2(-x_0y_3 - x_1y_2 + x_2y_1 + x_3y_0),$$

$$\Delta = x_0^2 + x_1^2 + x_2^2 + x_3^2.$$



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Study's kinematic mapping  $\varkappa$ :

$$\varkappa: \alpha \in \text{SE}(3) \mapsto \mathbf{x} \in \mathbb{P}^7$$

pre-image of  $\mathbf{x}$  is the displacement  $\alpha$

$$\Delta \begin{bmatrix} \Delta \\ p \\ q \\ r \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 \\ x_0^2 + x_1^2 - x_2^2 - x_3^2 & 2(x_1x_2 - x_0x_3) & 2(x_1x_3 + x_0x_2) \\ 2(x_1x_2 + x_0x_3) & x_0^2 - x_1^2 + x_2^2 - x_3^2 & 2(x_2x_3 - x_0x_1) \\ 2(x_1x_3 - x_0x_2) & 2(x_2x_3 + x_0x_1) & x_0^2 - x_1^2 - x_2^2 + x_3^2 \end{bmatrix} \quad (3)$$

$$p = 2(-x_0y_1 + x_1y_0 - x_2y_3 + x_3y_2),$$

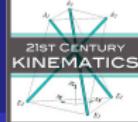
$$q = 2(-x_0y_2 + x_1y_3 + x_2y_0 - x_3y_1), \quad (4)$$

$$r = 2(-x_0y_3 - x_1y_2 + x_2y_1 + x_3y_0),$$

$$\Delta = x_0^2 + x_1^2 + x_2^2 + x_3^2.$$

$$S_6^2 : \quad x_0y_0 + x_1y_1 + x_2y_2 + x_3y_3 = 0, \quad x_i \text{ not all } 0 \quad (5)$$

$[x_0 : \dots : y_3]^T$  Study parameters = parametrization of  $\text{SE}(3)$  with dual  
quaternions



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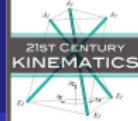
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How do we get the Study parameters when a proper orthogonal matrix  $\mathbf{A} = [a_{ij}]$  and the translation vector  $\mathbf{a} = [a_k]^T$  are given?



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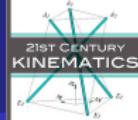
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Cayley map, not singularity free ( $180^\circ$ )

Rotation part:

$$\begin{aligned} X_0 : X_1 : X_2 : X_3 &= 1 + a_{11} + a_{22} + a_{33} : a_{32} - a_{23} : a_{13} - a_{31} : a_{21} - a_{12} \\ &= a_{32} - a_{23} : 1 + a_{11} - a_{22} - a_{33} : a_{12} + a_{21} : a_{31} + a_{13} \\ &= a_{13} - a_{31} : a_{12} + a_{21} : 1 - a_{11} + a_{22} - a_{33} : a_{23} + a_{32} \\ &= a_{21} - a_{12} : a_{31} + a_{13} : a_{23} - a_{32} : 1 - a_{11} - a_{22} + a_{33} \end{aligned} \quad (6)$$

Translation part:

$$\begin{aligned} 2y_0 &= a_1x_1 + a_2x_2 + a_3x_3, & 2y_1 &= -a_1x_0 + a_3x_2 - a_2x_3, \\ 2y_2 &= -a_2x_0 - a_3x_1 + a_1x_3, & 2y_3 &= -a_3x_0 + a_2x_1 - a_1x_2. \end{aligned} \quad (7)$$



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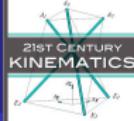
Remark: some people have been working on this topic like

E. Study, W. Blaschke, E.A. Weiss, ....

A. Yang, B. Roth, B. Ravani (and his students), A. Karger, W. Ströher, H. Stachel,....

sometimes using different names like Clifford Algebra:

M. McCarthy...



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Example:

A rotation about the z-axis through the angle  $\varphi$  is described by the matrix

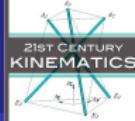
$$\begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & \cos \varphi & -\sin \varphi & 0 \\ 0 & \sin \varphi & \cos \varphi & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}. \quad (8)$$

Its kinematic image, computed via (6) and (7) is

$$\mathbf{r} = [1 + \cos \varphi : 0 : 0 : \sin \varphi : 0 : 0 : 0 : 0]^T. \quad (9)$$

As  $\varphi$  varies in  $[0, 2\pi]$ ,  $\mathbf{r}$  describes a straight line on the Study quadric which reads after algebraization

$$\mathbf{r} = [1 : 0 : 0 : u : 0 : 0 : 0 : 0]^T. \quad (10)$$



A special one parameter motion is defined by the matrix

$$\begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & \cos t & -\sin t & 0 \\ 0 & \sin t & \cos t & 0 \\ \sin \frac{t}{2} & 0 & 0 & 1 \end{bmatrix}. \quad (11)$$

Its kinematic image, computed via (6) and (7) reads

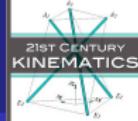
$$\mathbf{r} = \left[ 2 + 2 \cos t : 0 : 0 : 2 \sin t : \sin \frac{t}{2} \sin t : 0 : 0 : -\frac{1}{2} \sin \frac{t}{2} (2 + 2 \cos t) \right] \quad (12)$$

After algebraization and some manipulation we obtain

$$\mathbf{r} = [-1 + u^4 : 0 : 0 : -2u(1 + u^2) : 2u^2 : 0 : 0 : u(1 - u^2)], \quad (13)$$

represents a rational curve of degree four on the Study quadric.

The motion corresponding to this curve is a special case of the well known Bricard motions where all point-paths are spherical curves.



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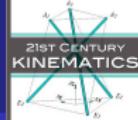
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$S_6^2$  is called Study quadric



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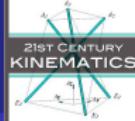
$S_6^2$  is called Study quadric

the map between  $S_6^2$  and  $SE(3)$  is not one to one,

$$F : x_0 = x_1 = x_2 = x_3 = 0, \quad E : y_0^2 + y_1^2 + y_2^2 + y_3^2 = 0. \quad (14)$$

Exceptional generator  $F$ , exceptional quadric  $E$

(these things come from the circle points in Euclidean geometry!)

Planar displacements:  $x_2 = x_3 = 0, y_0 = y_1 = 0$ 

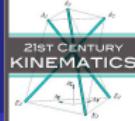
$$\frac{1}{x_0^2 + x_3^2} \begin{bmatrix} x_0^2 + x_3^2 & 0 & 0 \\ -2(x_0y_1 - x_3y_2) & x_0^2 - x_3^2 & -2x_0x_3 \\ -2(x_0y_2 + x_3y_1) & 2x_0x_3 & x_0^2 - x_3^2 \end{bmatrix}$$

SE(2) (we omit the last row and the last column)

Spherical displacements:  $y_i = 0$ 

$$\frac{1}{\Delta} \begin{bmatrix} x_0^2 + x_1^2 - x_2^2 - x_3^2 & 2(x_1x_2 - x_0x_3) & 2(x_1x_3 + x_0x_2) \\ 2(x_1x_2 + x_0x_3) & x_0^2 - x_1^2 + x_2^2 - x_3^2 & 2(x_2x_3 - x_0x_1) \\ 2(x_1x_3 - x_0x_2) & 2(x_2x_3 + x_0x_1) & x_0^2 - x_1^2 - x_2^2 + x_3^2 \end{bmatrix} \quad (15)$$

where  $\Delta = x_0^2 + x_1^2 + x_2^2 + x_3^2$ .  $\rightarrow SO^+(3)$ generate 3-spaces on  $S_6^2$



Planar displacements:  $x_2 = x_3 = 0, y_0 = y_1 = 0$

$$\frac{1}{x_0^2 + x_3^2} \begin{bmatrix} x_0^2 + x_3^2 & 0 & 0 \\ -2(x_0y_1 - x_3y_2) & x_0^2 - x_3^2 & -2x_0x_3 \\ -2(x_0y_2 + x_3y_1) & 2x_0x_3 & x_0^2 - x_3^2 \end{bmatrix}$$

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Spherical displacements:  $y_i = 0$

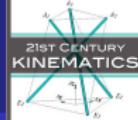
$$\frac{1}{\Delta} \begin{bmatrix} x_0^2 + x_1^2 - x_2^2 - x_3^2 & 2(x_1x_2 - x_0x_3) & 2(x_1x_3 + x_0x_2) \\ 2(x_1x_2 + x_0x_3) & x_0^2 - x_1^2 + x_2^2 - x_3^2 & 2(x_2x_3 - x_0x_1) \\ 2(x_1x_3 - x_0x_2) & 2(x_2x_3 + x_0x_1) & x_0^2 - x_1^2 - x_2^2 + x_3^2 \end{bmatrix} \quad (15)$$

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generate 3-spaces on  $S_6^2$   
more properties:

J. Selig, Geometric Fundamentals of Robotics, 2nd. ed. Springer 2005

H., Pfurner, Schröcker, Brunnthaler. Algebraic methods in mechanism analysis and synthesis. Robotica, 25(6):661-675, 2007.



# Quaternions

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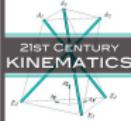
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The set of quaternions  $\mathbb{H}$  is the vector space  $\mathbb{R}^4$  together with the quaternion multiplication

$$(a_0, a_1, a_2, a_3) \star (b_0, b_1, b_2, b_3) = (a_0 b_0 - a_1 b_1 - a_2 b_2 - a_3 b_3, \\ a_0 b_1 + a_1 b_0 + a_2 b_3 - a_3 b_2, \\ a_0 b_2 - a_1 b_3 + a_2 b_0 - a_3 b_1, \\ a_0 b_3 - a_1 b_2 - a_2 b_1 + a_3 b_0). \quad (16)$$

The triple  $(\mathbb{H}, +, \star)$  (with component wise addition) forms a skew field. The real numbers can be embedded into this field via  $x \mapsto (x, 0, 0, 0)$ , and vectors  $\mathbf{x} \in \mathbb{R}^3$  are identified with quaternions of the shape  $(0, \mathbf{x})$ .



Every quaternion is a unique linear combination of the four basis quaternions  $\mathbf{1} = (1, 0, 0, 0)$ ,  $\mathbf{i} = (0, 1, 0, 0)$ ,  $\mathbf{j} = (0, 0, 1, 0)$ , and  $\mathbf{k} = (0, 0, 0, 1)$ .

The multiplication table is

*	1	i	j	k
1	1	i	j	k
i	i	-1	k	-j
j	j	-k	-1	i
k	k	j	-i	-1

*Conjugate quaternion* and *norm* are defined as

$$\bar{A} = (a_0, -a_1, -a_2, -a_3), \quad \|A\| = \sqrt{A * \bar{A}} = \sqrt{a_0^2 + a_1^2 + a_2^2 + a_3^2}. \quad (17)$$



Quaternions are closely related to spherical kinematic mapping.

Consider a vector  $\mathbf{a} = [a_1, a_2, a_3]^T$  and a matrix  $\mathbf{X}$  of the shape (15).

The product  $\mathbf{b} = \mathbf{X} \cdot \mathbf{a}$  can also be written as

$$B = X \star A \star \bar{X} \quad (18)$$

where  $X = (x_0, x_1, x_2, x_3)$ ,  $\|X\| = 1$  and  $A = (0, \mathbf{a})$ ,  $B = (0, \mathbf{b})$ .

From this follows:



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From this follows:

*Spherical displacements* can also be described by *unit quaternions* and *spherical kinematic mapping* maps a spherical displacement to the corresponding *unit quaternion*.



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To describe general Euclidean displacements extend the concept of quaternions.

A *dual quaternion*  $Q$  is a quaternion over the ring of dual numbers

$$Q = Q_0 + \varepsilon Q_1, \quad (19)$$

where  $\varepsilon^2 = 0$ .



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The algebra of dual quaternions has eight basis elements  $\mathbf{1}$ ,  $\mathbf{i}$ ,  $\mathbf{j}$ ,  $\mathbf{k}$ ,  $\varepsilon$ ,  $\varepsilon\mathbf{i}$ ,  $\varepsilon\mathbf{j}$ , and  $\varepsilon\mathbf{k}$  and the multiplication table

$\star$	$\mathbf{1}$	$\mathbf{i}$	$\mathbf{j}$	$\mathbf{k}$	$\varepsilon$	$\varepsilon\mathbf{i}$	$\varepsilon\mathbf{j}$	$\varepsilon\mathbf{k}$
$\mathbf{1}$	$\mathbf{1}$	$\mathbf{i}$	$\mathbf{j}$	$\mathbf{k}$	$\varepsilon$	$\varepsilon\mathbf{i}$	$\varepsilon\mathbf{j}$	$\varepsilon\mathbf{k}$
$\mathbf{i}$	$\mathbf{i}$	$-1$	$\mathbf{k}$	$-\mathbf{j}$	$\varepsilon\mathbf{i}$	$-\varepsilon\mathbf{1}$	$\varepsilon\mathbf{k}$	$-\varepsilon\mathbf{j}$
$\mathbf{j}$	$\mathbf{j}$	$-\mathbf{k}$	$-1$	$\mathbf{i}$	$\varepsilon\mathbf{j}$	$-\varepsilon\mathbf{k}$	$-\varepsilon\mathbf{1}$	$\varepsilon\mathbf{i}$
$\mathbf{k}$	$\mathbf{k}$	$\mathbf{j}$	$-\mathbf{i}$	$-1$	$\varepsilon\mathbf{k}$	$\varepsilon\mathbf{j}$	$-\varepsilon\mathbf{i}$	$-\varepsilon\mathbf{1}$
$\varepsilon\mathbf{1}$	$\varepsilon$	$\varepsilon\mathbf{i}$	$\varepsilon\mathbf{j}$	$\varepsilon\mathbf{k}$	$0$	$0$	$0$	$0$
$\varepsilon\mathbf{i}$	$\varepsilon\mathbf{i}$	$-\varepsilon\mathbf{1}$	$\varepsilon\mathbf{k}$	$-\varepsilon\mathbf{j}$	$0$	$0$	$0$	$0$
$\varepsilon\mathbf{j}$	$\varepsilon\mathbf{j}$	$-\varepsilon\mathbf{k}$	$-\varepsilon\mathbf{1}$	$\varepsilon\mathbf{i}$	$0$	$0$	$0$	$0$
$\varepsilon\mathbf{k}$	$\varepsilon\mathbf{k}$	$\varepsilon\mathbf{j}$	$-\varepsilon\mathbf{i}$	$-\varepsilon\mathbf{1}$	$0$	$0$	$0$	$0$



Dual quaternions know two types of conjugation.

The *conjugate quaternion* and the *conjugate dual quaternion* of a dual quaternion  $Q = x_0 + \varepsilon y_0 + \mathbf{x} + \varepsilon \mathbf{y}$  are defined as

$$\overline{Q} = x_0 + \varepsilon y_0 - \mathbf{x} - \varepsilon \mathbf{y} \quad \text{and} \quad Q_e = x_0 - \varepsilon y_0 + \mathbf{x} - \varepsilon \mathbf{y}, \quad (20)$$

respectively. The norm of a dual quaternion is

$$\|Q\| = \sqrt{Q\overline{Q}}. \quad (21)$$



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With these definitions, the equation  $\mathbf{b} = \mathbf{X} \cdot \mathbf{a}$  where  $\mathbf{X}$  is a matrix of the shape (3) can be written as

$$B = X_e * A * \bar{X} \quad (22)$$

where  $X = \mathbf{x} + \varepsilon \mathbf{y}$ ,  $\|X\| = 1$ ,  $\mathbf{x} = (x_0, \dots, x_3)^T$ ,  $\mathbf{y} = (y_0, \dots, y_3)^T$ , and  $\mathbf{x} \cdot \mathbf{y} = 0$ .

The last condition is precisely the Study condition (5).

$A$  and  $B$  are dual quaternions of the type:  $A = 1 + \varepsilon \mathbf{a}$ ,  $B = 1 + \varepsilon \mathbf{b}$

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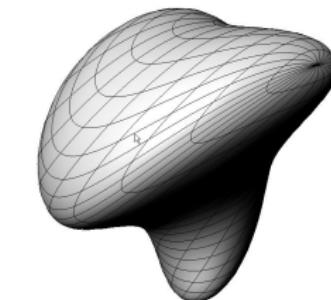
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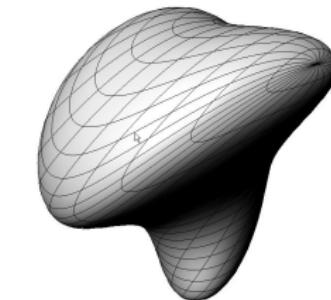
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- a constraint that removes one degree of freedom maps to a hyper-surface in  $\mathbb{P}^7$



# Constraint varieties

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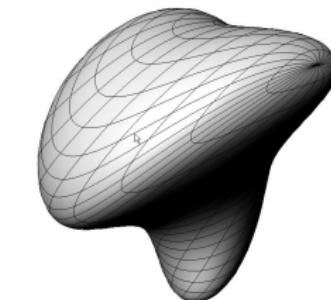
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- a constraint that removes one degree of freedom maps to a hyper-surface in  $\mathbb{P}^7$
- a set of constraints corresponds to a set of polynomial equations

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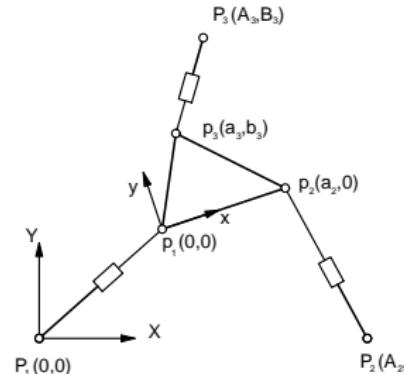


Figure: Planar 3-RPR parallel mechanism.

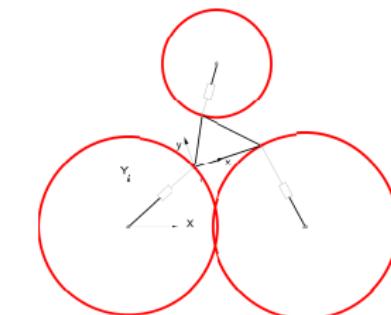


Figure: Geometric interpretations

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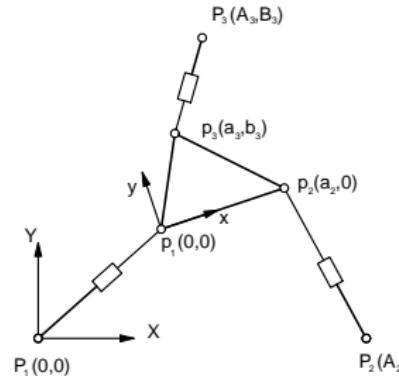


Figure: Planar 3-RPR parallel mechanism.

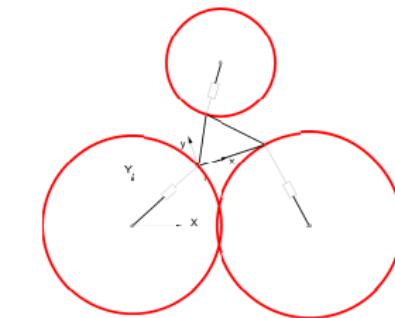
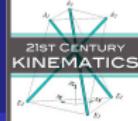


Figure: Geometric interpretations

Condition that one point of the moving system is bound to move on a circle

$$(x_2 - \frac{1}{2}(c_2 + C_2 - x_1(C_1 - c_1)))^2 + (x_3 - \frac{1}{2}(x_1(c_2 - C_2) - C_1 - c_1))^2 - \frac{1}{4}R^2(x_1^2 + 1) = 0, \quad (23)$$



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Three constraint equations:

$$h_1 : 4x_2^2 + 4x_3^2 + R_1 = 0$$

$$h_2 : 4x_2^2 - 4A_2 x_3 x_0 + 4x_3 x_0 a_2 + 4x_3^2 - 4x_1 x_2 a_2 - 4x_1 A_2 x_2 + 4x_1^2 A_2 a_2 - 2A_2 a_2 + R_2 = 0$$

$$h_3 : 4x_2^2 + 4B_3 x_0 x_2 - 4A_3 x_3 x_0 - 4x_2 x_0 b_3 + 4x_3 x_0 a_3 + 4x_3^2 - 4x_1 B_3 x_0 a_3 + 4x_1 A_3 x_0 b_3 - 4x_1 x_2 a_3 - 4x_1 B_3 x_3 - 4x_1 A_3 x_2 - 4x_1 x_3 b_3 - 4x_1^2 A_3 a_3 + 4x_1^2 B_3 b_3 - 2B_3 b_3 - 2A_3 a_3 + R_3 = 0. \quad (24)$$

# A simple example

Three constraint equations:

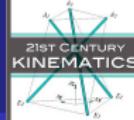
$$h_1 : 4x_2^2 + 4x_3^2 + R_1 = 0$$

$$h_2 : 4x_2^2 - 4A_2 x_3 x_0 + 4x_3 x_0 a_2 + 4x_3^2 - 4x_1 x_2 a_2 - 4x_1 A_2 x_2 + 4x_1^2 A_2 a_2 - 2A_2 a_2 + R_2 = 0$$

$$h_3 : 4x_2^2 + 4B_3 x_0 x_2 - 4A_3 x_3 x_0 - 4x_2 x_0 b_3 + 4x_3 x_0 a_3 + 4x_3^2 - 4x_1 B_3 x_0 a_3 + 4x_1 A_3 x_0 b_3 - 4x_1 x_2 a_3 - 4x_1 B_3 x_3 - 4x_1 A_3 x_2 - 4x_1 x_3 b_3 + 4x_1^2 A_3 a_3 + 4x_1^2 B_3 b_3 - 2B_3 b_3 - 2A_3 a_3 + R_3 = 0. \quad (24)$$



Figure: Geometric interpretation in kinematic image space



# Image space transformations

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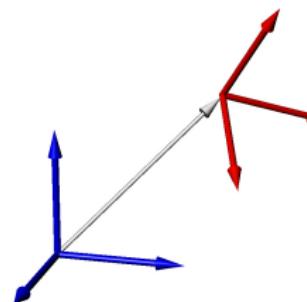


Figure: Fixed and moving coordinate systems

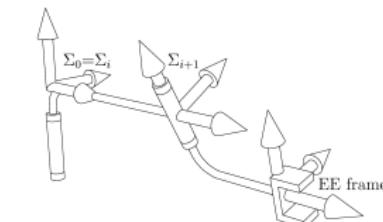


Figure: Robot coordinate systems

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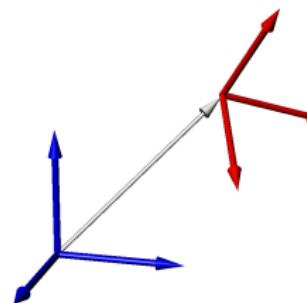
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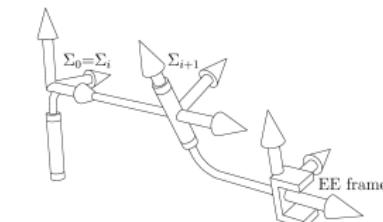
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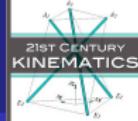


**Figure:** Fixed and moving coordinate systems



**Figure:** Robot coordinate systems

- The relative displacement  $\alpha$  depends on the choice of fixed and moving frame
- Coordinate systems are usually attached to the base and the end-effector of a mechanism
- Changes of fixed and moving frame induce transformations on  $S^2_6$ , impose a geometric structure on  $S^2_6$ .



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$$\mathbf{y} = \mathbf{T}_f \mathbf{T}_m \mathbf{x}, \quad \mathbf{T}_m = \begin{bmatrix} \mathbf{A} & \mathbf{O} \\ \mathbf{B} & \mathbf{A} \end{bmatrix}, \quad \mathbf{T}_f = \begin{bmatrix} \mathbf{C} & \mathbf{O} \\ \mathbf{D} & \mathbf{C} \end{bmatrix}, \quad (25)$$



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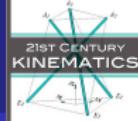
$$\mathbf{y} = \mathbf{T}_f \mathbf{T}_m \mathbf{x}, \quad \mathbf{T}_m = \begin{bmatrix} \mathbf{A} & \mathbf{O} \\ \mathbf{B} & \mathbf{A} \end{bmatrix}, \quad \mathbf{T}_f = \begin{bmatrix} \mathbf{C} & \mathbf{O} \\ \mathbf{D} & \mathbf{C} \end{bmatrix}, \quad (25)$$

$$\mathbf{A} = \begin{bmatrix} m_0 & -m_1 & -m_2 & -m_3 \\ m_1 & m_0 & m_3 & -m_2 \\ m_2 & -m_3 & m_0 & m_1 \\ m_3 & m_2 & -m_1 & m_0 \end{bmatrix}, \quad \mathbf{B} = \begin{bmatrix} m_4 & -m_5 & -m_6 & -m_7 \\ m_5 & m_4 & m_7 & -m_6 \\ m_6 & -m_7 & m_4 & m_5 \\ m_7 & m_6 & -m_5 & m_4 \end{bmatrix} \quad (26)$$

$$\mathbf{C} = \begin{bmatrix} f_0 & -f_1 & -f_2 & -f_3 \\ f_1 & f_0 & -f_3 & f_2 \\ f_2 & f_3 & f_0 & -f_1 \\ f_3 & -f_2 & f_1 & f_0 \end{bmatrix}, \quad \mathbf{D} = \begin{bmatrix} f_4 & -f_5 & -f_6 & -f_7 \\ f_5 & f_4 & -f_7 & f_6 \\ f_6 & f_7 & f_4 & -f_5 \\ f_7 & -f_6 & f_5 & f_4 \end{bmatrix} \quad (27)$$

and  $\mathbf{O}$  is the four by four zero matrix.

- $\mathbf{T}_m$  and  $\mathbf{T}_f$  commute
- $\mathbf{T}_m$  and  $\mathbf{T}_f$  induce transformations of  $P^7$  that fix  $S_6^2$ , the exceptional generator  $F$ , and the *exceptional quadric*  $E \subset F$



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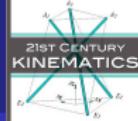
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- A set of constraints is described by a set of polynomials



# Affine (Projective) Varieties - Ideals

- A set of constraints is described by a set of polynomials
- The set of polynomials forms a ring which is denoted by  $k[x_0, \dots, x_n]$ .

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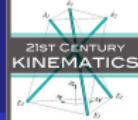
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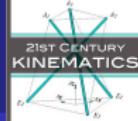
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- A set of constraints is described by a set of polynomials
- The set of polynomials forms a ring which is denoted by  $k[x_0, \dots, x_n]$ .
- If  $k$  is a field and  $f_1, \dots, f_s$  are polynomials in  $k[x_0, \dots, x_n]$ , and if

$$\mathbf{V}(f_1, \dots, f_s) = \{(a_1, \dots, a_n) \in k^n : f_i(a_1, \dots, a_n) = 0, \text{ for all } 1 \leq i \leq s\}$$

then  $\mathbf{V}(f_1, \dots, f_s)$  is called an affine variety defined by the polynomials  $f_i$ .



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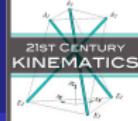
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then  $\mathbf{V}(f_1, \dots, f_s)$  is called an affine variety defined by the polynomials  $f_i$ .

- The definition says essentially that the affine variety is the zero set of the defining polynomials.



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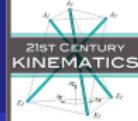
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then  $\mathbf{V}(f_1, \dots, f_s)$  is called an affine variety defined by the polynomials  $f_i$ .

- The definition says essentially that the affine variety is the zero set of the defining polynomials.
- In case of homogeneous polynomials the variety is called a projective variety.



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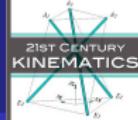
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- The definition says essentially that the affine variety is the zero set of the defining polynomials.
- In case of homogeneous polynomials the variety is called a projective variety.
- An ideal  $I$  is a subset of  $k[x_0, \dots, x_n]$  that satisfies the following properties:

$$(i) 0 \in I.$$

$$(ii) \text{ If } f, g \in I, \text{ then } f + g \in I.$$

$$(iii) \text{ If } f \in I, g \in k \text{ then } fg \in I.$$



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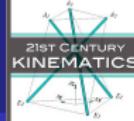
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D. A. Cox, J. B. Little, and D. O'Shea, *Ideals, Varieties and Algorithms*, Springer, third ed., 2007.



# Example: Stewart-Gough platform

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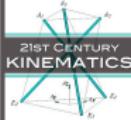


Figure: Stewart-Gough platform

Sphere constraint:

1 in canonical form

$$4y_0^2 + 4y_3^2 + 4y_2^2 + 4y_1^2 - (x_1^2 + x_2^2 + x_0^2 + x_3^2)r = 0$$



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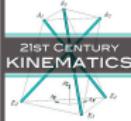
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Sphere constraint:

2 in general form

$$\begin{aligned} h: R(x_0^2 + x_1^2 + x_2^2 + x_3^2) + 4(y_0^2 + y_1^2 + y_2^2 + y_3^2) - 2x_0^2(Aa + Bb + Cc) \\ + 2x_1^2(-Aa + Bb + Cc) + 2x_2^2(Aa - Bb - Cc) + 2x_3^2(Aa + Bb + Cc) \\ + 2x_3^2(Aa + Bb - Cc) + 4[x_0x_1(Bc - Cb) + x_0x_2(Ca - Ac) \\ + x_0x_3(Ab - Ba) - x_1x_2(Ab + Ba) - x_1x_3(Ac + Ca) \\ - x_2x_3(Bc + Cb) + (x_0y_1 - y_0x_1)(A - a) + (x_0y_2 - y_0x_2)(B - b) \\ + (x_0y_3 - y_0x_3)(C - c) + (x_1y_2 - y_1x_2)(C + c) - (x_1y_3 - y_1x_3)(B + b) \\ + (x_2y_3 - y_2x_3)(A + a)] = 0, \end{aligned} \quad (28)$$

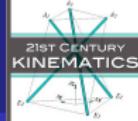


Sphere constraint:

2 in general form

$$\begin{aligned}
 h: R(x_0^2 + x_1^2 + x_2^2 + x_3^2) + 4(y_0^2 + y_1^2 + y_2^2 + y_3^2) - 2x_0^2(Aa + Bb + Cc) \\
 + 2x_1^2(-Aa + Bb + Cc) + 2x_2^2(Aa - Bb - Cc) + 2x_3^2(Aa + Bb + Cc) \\
 + 2x_3^2(Aa + Bb - Cc) + 4[x_0x_1(Bc - Cb) + x_0x_2(Ca - Ac) \\
 + x_0x_3(Ab - Ba) - x_1x_2(AB + Ba) - x_1x_3(AC + Ca) \\
 - x_2x_3(Bc + Cb) + (x_0y_1 - y_0x_1)(A - a) + (x_0y_2 - y_0x_2)(B - b) \\
 + (x_0y_3 - y_0x_3)(C - c) + (x_1y_2 - y_1x_2)(C + c) - (x_1y_3 - y_1x_3)(B + b) \\
 + (x_2y_3 - y_2x_3)(A + a)] = 0,
 \end{aligned} \tag{28}$$

$$\begin{aligned}
 F := & [177x_2y_3 - 177x_3y_2 - 20x_1y_0 + 20x_0y_1 - 34059x_0x_3 + 12236x_2x_1 - x_0^2s_1 - x_1^2s_1 - x_3^2s_1 - x_2^2s_1, \\
 & 156x_2y_3 - 156x_3y_2 - 101x_1y_0 + 101x_0y_1 + 68081x_0x_3 - 101796x_2x_1 - x_0^2s_2 - x_1^2s_2 - x_3^2s_2 - x_2^2s_2, \\
 & -x_0^2s_3 - x_1^2s_3 - x_3^2s_3 - x_2^2s_3 - 198x_2y_3 + 198x_3y_2 - 61x_1y_0 + 61x_0y_1 - 68203x_0x_3 - 126565x_2x_1, \\
 & 438313x_2^2 + x_0^2s_4 + x_1^2s_4 + x_3^2s_4 + x_2^2s_4 + 792x_2y_3 - 792x_3y_2 + 244x_1y_0 - 244x_0y_1 - 1370x_3y_1 + \\
 & 422x_0y_2 - 422x_2y_0 + 1370y_3x_1 - 544796x_0x_3 + 505072x_2x_1 - 437869x_1^2 - 11x_0^2 + 455x_3^2, \\
 & -438313x_2^2 - x_0^2s_5 - x_1^2s_5 - x_3^2s_5 - x_2^2s_5 + 792x_2y_3 - 792x_3y_2 + 244x_1y_0 - 244x_0y_1 + 1370x_3y_1, \\
 & -422x_0y_2 + 422x_2y_0 - 1370y_3x_1 - 544796x_0x_3 + 505072x_2x_1 + 437869x_1^2 + 11x_0^2 - 455x_3^2, \\
 & x_0y_0 + x_1y_1 + x_2y_2 + x_3y_3, \\
 & -x_0^2W_1 - x_1^2W_1 - x_3^2W_1 - x_2^2W_1 - 204402x_0x_3 - 297x_2x_1]
 \end{aligned}$$



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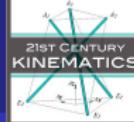
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Is there a method to generate constraint equations without (deep) insight in the geometric structure of a kinematic chain??



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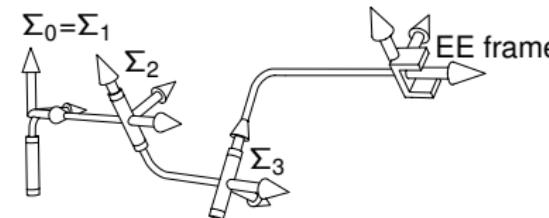


Figure: Canonical 3R-chain

the relative position of two rotation axes is described by the usual Denavit-Hartenberg parameters ( $\alpha_i, a_i, d_i$ )

$$\mathbf{G}_i = \begin{pmatrix} 1 & 0 & 0 & 0 \\ a_i & 1 & 0 & 0 \\ 0 & 0 & \cos(\alpha_i) & -\sin(\alpha_i) \\ d_i & 0 & \sin(\alpha_i) & \cos(\alpha_i) \end{pmatrix}. \quad (29)$$

$$\mathbf{M}_i = \begin{pmatrix} 1 & 0 & 0 & 0 \\ 0 & \cos(u_i) & -\sin(u_i) & 0 \\ 0 & \sin(u_i) & \cos(u_i) & 0 \\ 0 & 0 & 0 & 1 \end{pmatrix} \quad \text{or} \quad \mathbf{M}_i = \begin{pmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ u & 0 & 0 & 1 \end{pmatrix} \quad (30)$$

Following this sequence of transformations the endeffector will have the following pose:

$$\mathbf{D} = \mathbf{M}_1 \cdot \mathbf{G}_1 \cdot \mathbf{M}_2 \cdot \mathbf{G}_2 \cdots \mathbf{M}_n, \quad (31)$$

Parametric equation



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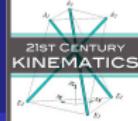
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## What do we gain?

- Using all features of algebraic geometry symbolic software (Maple, Mathematica, Singular, ....) e.g.:



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## What do we gain?

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with(PolynomialIdeals):

```
[`<`, `>`, Add, Contract, EliminationIdeal, EquidimensionalDecomposition, Generators, HilbertDimension,  
IdealContainment, IdealInfo, IdealMembership, Intersect, IsMaximal, IsPrimary, IsPrime, IsProper, IsRadical,  
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UnivariatePolynomial, VanishingIdeal, ZeroDimensionalDecomposition, in, subset]
```

with(Groebner);

```
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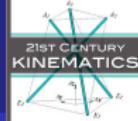
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```

- all solutions, sometimes a complete analytic description of a workspace.



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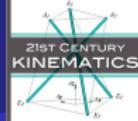
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```

- all solutions, sometimes a complete analytic description of a workspace.
- Singularities can be treated, pathologic cases (selfmotion) can be detected and degree of freedom computation (Hilbert dimension) can be performed



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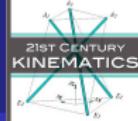
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Back to the parametric equations!



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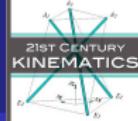
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Back to the parametric equations!

Half tangent substitution transforms the rotation angles  $u_i$  into algebraic parameters  $t_i$  and one ends up with eight parametric equations of the form:

$$\begin{aligned}x_0 &= f_0(t_1, \dots, t_n), \\x_1 &= f_1(t_1, \dots, t_n), \\&\vdots \\y_3 &= f_8(t_1, \dots, t_n).\end{aligned}\tag{32}$$

- Equations will be rational having a denominator of the form  $(1 + t_1^2) \cdot \dots \cdot (1 + t_n^2)$  which can be canceled because the Study parameters  $x_i, y_i$  are homogeneous.
- The same can be done with a possibly appearing common factor of all parametric expressions.



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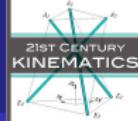
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- there exists a one-to-one correspondence from all spatial transformations to the Study quadric



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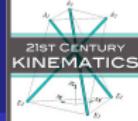
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- there exists a one-to-one correspondence from all spatial transformations to the Study quadric
- transformation parametrized by  $n$  parameters  $t_1, \dots, t_n$



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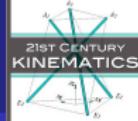
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- there exists a one-to-one correspondence from all spatial transformations to the Study quadric
- transformation parametrized by  $n$  parameters  $t_1, \dots, t_n$ 
  - → kinematic mapping a set of corresponding points in  $P^7$



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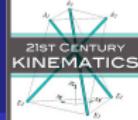
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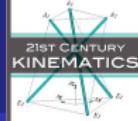
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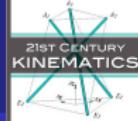
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- What do we know about this variety?
- Its ideal  $\mathcal{V}$  consists of homogeneous polynomials and contains  $x_0y_0 + x_1y_1 + x_2y_2 + x_3y_3$ , i.e. the equation for the Study quadric  $S_6^2$ .



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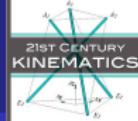
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- the minimum number of polynomials to describe  $\mathcal{V}$  corresponds to the degrees of freedom (dof) of the kinematic chain



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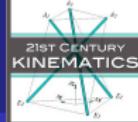
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- the minimum number of polynomials to describe  $\mathcal{V}$  corresponds to the degrees of freedom (dof) of the kinematic chain
- If the number of generic parameters is  $n$  then  $m = 6 - n$  polynomials are necessary to describe  $\mathcal{V}$



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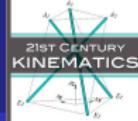
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General observation: **the parametric equations of a geometric object have to fulfill the implicit equations**



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Manfred L. Hustý,  
Hans-Peter  
Schröcker

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Quaternions

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mechanism freedom

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Parallel Manipulator

Synthesis of  
mechanisms

General observation: **the parametric equations of a geometric object have to fulfill the implicit equations**

- Make a general ansatz of a polynomial of degree  $n$ :

$$p = \sum_{\alpha, \beta} C_k x_i^\alpha y_j^\beta$$



# Implicitization Algorithm

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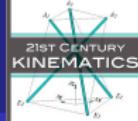
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- substitute the parametric equations into  $p$



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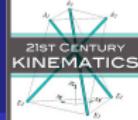
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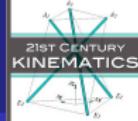
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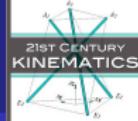
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  - collect with respect to the powerproducts of the  $t_i$  and extract their coefficients  $\rightarrow$

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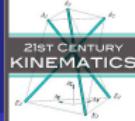
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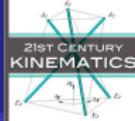
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- determine  $C_k$
- possibly increase the degree of the ansatz polynomial

Walter and H. , On Implicitization of Kinematic Constraint Equations, Chin.  
J. of Mech. Design, 2010.



## Remarks:

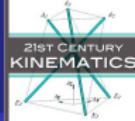
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## Remarks:

- The number of equations depends on the particular design of the chain

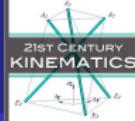
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- The number of equations depends on the particular design of the chain
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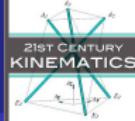
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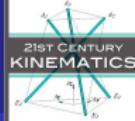
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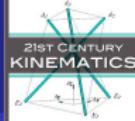
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- if these systems can be solved depends how complicated the chain is (we have solved up to degree 8)

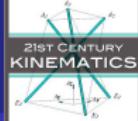
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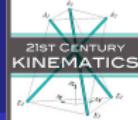
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- the algorithm could create more polynomials than needed; take out of the set the number needed (simplest!)

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# Constraint equations and mechanism freedom

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The degree of freedom of a mechanical system is the Hilbert dimension of the ideal generated by the constraint polynomials, the Study quadric and a normalizing condition



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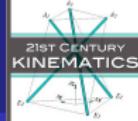
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Example: Self motions of Stewart Platforms

```
> with(Groebner):  
> F:=[U4,U2,U3,U8,U10,U7,h1,x0^2+x1^2+x2^2+x3^2-1];  
F:=[244x_1y_0 - 792x_3y_2 - 244x_0y_1 + 1370y_3x_1 - 1370x_3y_1 + 422x_0y_2 + 439323x_2^2 + 1465x_3^2 + 999x_0^2 -  
436859x_1^2 + 792x_2y_3 - 422x_2y_0 - 544796x_0x_3 + 505072x_2x_1, -101x_1y_0 - 156x_3y_2 + 101x_0y_1 + 156x_2y_3 +  
68081x_0x_3 - 101796x_2x_1 -  $\frac{4401}{4}x_1^2 - \frac{4401}{4}x_3^2 - \frac{4401}{4}x_2^2 - \frac{4401}{4}x_0^2$ , -61x_1y_0 + 198x_3y_2 + 61x_0y_1 - 198x_2y_3 -  
68203x_0x_3 - 126565x_2x_1 -  $\frac{6713}{2}x_1^2 - \frac{6713}{2}x_3^2 - \frac{6713}{2}x_2^2 - \frac{6713}{2}x_0^2$ , -204402x_0x_3 - 297x_2x_1 -  $\frac{3749}{2}x_1^2 -$   
 $\frac{3749}{2}x_3^2 - \frac{3749}{2}x_2^2 - \frac{3749}{2}x_0^2$ , -404x_1y_0 - 624x_3y_2 + 404x_0y_1 + 1082y_3x_1 - 1082x_3y_1 - 700x_0y_2 - 375644x_2^2 -  
22627x_3^2 - 22712x_0^2 + 330305x_1^2 + 624x_2y_3 + 700x_2y_0 - 545284x_0x_3 - 408372x_2x_1, x_0y_0 + x_1y_1 + x_2y_2 +  
x_3y_3, -640x_1y_0 - 5664x_3y_2 + 640x_0y_1 + 384y_3x_1 - 384x_3y_1 + 1496x_0y_2 + 4y_0^2 + 4y_3^2 + 4y_2^2 + 4y_1^2 + 1891923x_2^2 +  
1761263x_3^2 - 87533x_0^2 - 218193x_1^2 + 5664x_2y_3 - 1496x_2y_0 - 1089888x_0x_3 + 391552x_2x_1, x_0^2 + x_1^2 + x_2^2 + x_3^2 - 1]  
> HilbertDimension(F,tdeg(x0,x1,x2,x3,y0,y1,y2,y3));
```



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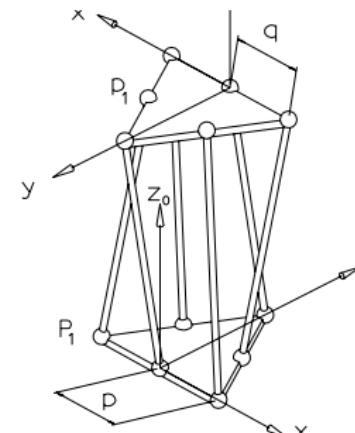
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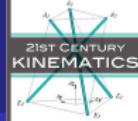
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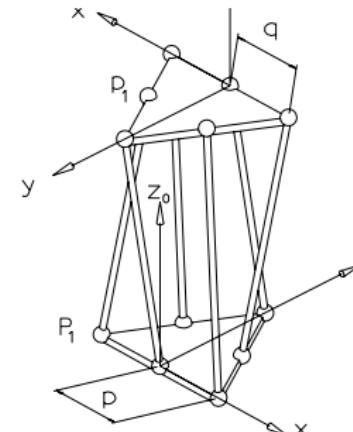
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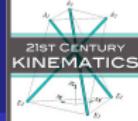
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$$G := [-4x_2y_3 + 12x_1y_0 + 4y_2x_3 + 4\sqrt{3}x_1x_2, 8\sqrt{3}x_2y_0, -4x_2y_3 - 12x_1y_0 + 4y_2x_3 + 4\sqrt{3}x_1x_2, -\frac{2}{3}\sqrt{3}(\sqrt{3}x_2y_3 - 3\sqrt{3}x_1y_0 - \sqrt{3}y_2x_3 + \sqrt{3}x_2^2 + \sqrt{3}x_3^2 - 3x_1x_2 + 3x_2y_0 - 3y_3x_1 + 3x_3y_1), 4\sqrt{3}y_0(\sqrt{3}x_1 + x_2), 4y_0^2 + 4y_1^2 + 4y_3^2 + 4y_2^2 + x_3^2(2 - R) + x_1^2(2 - R) + x_2^2(2 - R) + 2\sqrt{3}x_3y_1 + 6x_1y_0 + 2y_2x_3 + 2\sqrt{3}x_2y_0 - 2x_2y_3 - 2\sqrt{3}y_3x_1 + x_1^2 - x_3^2 + 2\sqrt{3}x_1x_2 - x_2^2, x_1y_1 + x_2y_2 + x_3y_3, -1 + x_1^2 + x_2^2 + x_3^2]  
> HilbertDimension(F,tdeg(x1,x2,x3,y0,y1,y2,y3));$$

```



# Schatz Mechanism - Bricard's overconstrained 6R chain

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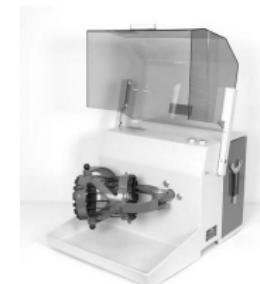
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Turbula T2F Heavy-Duty Shaker-Mixer (Willy A. Bachofen AG, <http://www.wab.ch/ie/e/turbula1.htm>)

The DH parameters of Bricard's orthogonal chain

i	$a_i$	$d_i$	$\alpha_i$
1	$a_1$	0	$\pi/2$
2	$a_2$	0	$\pi/2$
3	$a_3$	0	$\pi/2$
4	$a_4$	0	$\pi/2$
5	$a_5$	0	$\pi/2$
6	$a_6$	0	$\pm\pi/2$

Table: DH parameters of Bricard's orthogonal chain

with the additional condition that  $a_1^2 - a_2^2 + a_3^2 - a_4^2 + a_5^2 - a_6^2 = 0$ .



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$$F := [-2z_0 + z_1 + z_2 v_1 - 2z_3 v_1 - 2s_1 - 2s_2 v_1, -z_0 + 2z_1 + 2z_2 v_1 - z_3 v_1 - 2s_0 - 2s_3 v_1, -z_1 v_1 + z_2 - 2s_1 v_1 + 2s_2, -z_0 v_1 + z_3 + 2s_0 v_1 - 2s_3, z_0 v_6 q + z_1 + 2z_1 v_6 q - z_2 + 2z_2 v_6 q + z_3 v_6 q - 2s_0 v_6 q - 2s_1 + 2s_2 - 2s_3 v_6 q, -z_0 + 2z_0 v_6 q + z_1 v_6 q + z_2 v_6 q + z_3 + 2z_3 v_6 q - 2s_0 + 2s_1 v_6 q + 2s_2 v_6 q + 2s_3, z_0 + 2z_0 v_6 q - z_1 v_6 q + z_2 v_6 q + z_3 - 2z_3 v_6 q + 2s_0 - 2s_1 v_6 q + 2s_2 v_6 q + 2s_3, -z_0 v_6 q - z_1 + 2z_1 v_6 q - z_2 - 2z_2 v_6 q + z_3 v_6 q + 2s_0 v_6 q + 2s_1 + 2s_2 - 2s_3 v_6 q, s_0 z_0 + s_1 z_1 + s_2 z_2 + s_3 z_3, z_0^2 + z_1^2 + z_2^2 + z_3^2 - 1] \\ > \text{HilbertDimension}(F);$$

1



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$$F := [-2z_0 + z_1 + z_2 v_1 - 2z_3 v_1 - 2s_1 - 2s_2 v_1, -z_0 + 2z_1 + 2z_2 v_1 - z_3 v_1 - 2s_0 - 2s_3 v_1, -z_1 v_1 + z_2 - 2s_1 v_1 + 2s_2, -z_0 v_1 + z_3 + 2s_0 v_1 - 2s_3, z_0 v_{6q} + z_1 + 2z_1 v_{6q} - z_2 + 2z_2 v_{6q} + z_3 v_{6q} - 2s_0 v_{6q} - 2s_1 + 2s_2 - 2s_3 v_{6q}, -z_0 + 2z_0 v_{6q} + z_1 v_{6q} + z_2 v_{6q} + z_3 + 2z_3 v_{6q} - 2s_0 + 2s_1 v_{6q} + 2s_2 v_{6q} + 2s_3, z_0 + 2z_0 v_{6q} - z_1 v_{6q} + z_2 v_{6q} + z_3 - 2z_3 v_{6q} + 2s_0 - 2s_1 v_{6q} + 2s_2 v_{6q} + 2s_3, -z_0 v_{6q} - z_1 + 2z_1 v_{6q} - z_2 - 2z_2 v_{6q} + z_3 v_{6q} + 2s_0 v_{6q} + 2s_1 + 2s_2 - 2s_3 v_{6q}, s_0 z_0 + s_1 z_1 + s_2 z_2 + s_3 z_3, z_0^2 + z_1^2 + z_2^2 + z_3^2 - 1]$$

> HilbertDimension(F);

1

$$> \text{Basis}(F, \text{tdeg}[z_0, z_1, z_2, z_3, s_0, s_1, s_2, s_3, v_{6q}, v_1]);$$

$$F := [z_2 - z_1 - z_0 + z_3, 2s_1 - 2s_0 + z_1 + z_0, 2s_2 + 2s_0 - z_1 + z_3, 2s_3 + 2s_0 - 2z_1 - z_0 + z_3, 2s_0 z_0 - 2s_0 z_3 + 2z_1 z_3 + z_0 z_3 - z_3^2, z_0 v_1 - z_3 v_1 - 2s_0 + 2z_1 - z_3, 2z_1^2 - 1 + 2z_1 z_0 + 2z_0^2 - 2z_1 z_3 - 2z_0 z_3 + 2z_3^2, z_1 v_{6q} + z_0 v_{6q} - 2s_0 + z_1, 4s_0^2 z_1 - z_0^2 + 2z_1 z_3 + 2z_0 z_3 - 2z_3^2, 2s_0 v_1 - z_3 v_1 - z_0 + z_3, 2s_0 v_{6q} - z_0 v_{6q} + 2z_3 v_{6q} - 2s_0 - z_0, 8v_{6q} z_3^2 - 2 - 4v_1 z_3^2 + v_{6q} v_1 + 12s_0 z_1 + 2z_1 z_0 + 2z_0^2 - 12s_0 z_3 - 10z_0 z_3 + 6z_3^2 - 4v_6 q + v_1, 8v_{6q} z_0 z_3 - 1 + 4s_0 z_1 + 2z_1 z_0 - 12s_0 z_3 + 8z_1 z_3 + 2z_0 z_3 - v_{6q}, 2v_1 z_1 z_3 + z_0^2 + 4s_0 z_3 - 4z_1 z_3 - 2z_0 z_3 + 3z_3^2, v_{6q} v_1 z_3 - z_0 v_{6q} - z_3 v_1 - z_0, z_0^3 + 4s_0 z_1 z_3 + z_0^2 z_3 - 2z_1 z_3^2 - z_0 z_3^2 + 3z_3^3 - 2z_3, 8v_{6q} z_0^2 - 2 + 4v_1 z_3^2 - v_{6q} v_1 + 4s_0 z_1 + 6z_1 z_0 + 6z_0^2 - 4s_0 z_3 + 2z_0 z_3 + 2z_3^2 - v_1, 2v_1 z_3^3 - z_1 z_0^2 - 2z_1 z_0 z_3 + 4s_0 z_3^2 - 5z_1 z_3^2 + 2z_3^3 - z_3 v_1, 4v_1 z_3^2 + 1 - v_{6q} v_1^2 + 4v_{6q} v_1 - v_1^2 - 4z_1 z_0 + 16s_0 z_3 - 12z_1 z_3 - 4z_0 z_3 + 8z_3^2 - v_{6q} + 2v_1, v_{6q}^2 v_1^2 - 1 - 4v_{6q}^2 v_1 + v_{6q}^2 - v_1^2]$$



# Schatz Mechanism - Bricard's overconstrained 6R chain

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$$F := [-2z_0 + z_1 + z_2 v_1 - 2z_3 v_1 - 2s_1 - 2s_2 v_1, -z_0 + 2z_1 + 2z_2 v_1 - z_3 v_1 - 2s_0 - 2s_3 v_1, -z_1 v_1 + z_2 - 2s_1 v_1 + 2s_2, -z_0 v_1 + z_3 + 2s_0 v_1 - 2s_3, z_0 v_{6q} + z_1 + 2z_1 v_{6q} - z_2 + 2z_2 v_{6q} + z_3 v_{6q} - 2s_0 v_{6q} - 2s_1 + 2s_2 - 2s_3 v_{6q}, -z_0 + 2z_0 v_{6q} + z_1 v_{6q} + z_2 v_{6q} + z_3 + 2z_3 v_{6q} - 2s_0 + 2s_1 v_{6q} + 2s_2 v_{6q} + 2s_3, z_0 + 2z_0 v_{6q} - z_1 v_{6q} + z_2 v_{6q} + z_3 - 2z_3 v_{6q} + 2s_0 - 2s_1 v_{6q} + 2s_2 v_{6q} + 2s_3, -z_0 v_{6q} - z_1 + 2z_1 v_{6q} - z_2 - 2z_2 v_{6q} + z_3 v_{6q} + 2s_0 v_{6q} + 2s_1 + 2s_2 - 2s_3 v_{6q}, s_0 z_0 + s_1 z_1 + s_2 z_2 + s_3 z_3, z_0^2 + z_1^2 + z_2^2 + z_3^2 - 1]$$

> HilbertDimension(F);

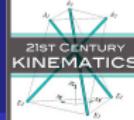
1

> Basis(F, tdeg[z0, z1, z2, z3, s0, s1, s2, s2, v6q, v1]);

$$F := [z_2 - z_1 - z_0 + z_3, 2s_1 - 2s_0 + z_1 + z_0, 2s_2 + 2s_0 - z_1 + z_3, 2s_3 + 2s_0 - 2z_1 - z_0 + z_3, 2s_0 z_0 - 2s_0 z_3 + 2z_1 z_3 + z_0 z_3 - z_3^2, z_0 v_1 - z_3 v_1 - 2s_0 + 2z_1 - z_3, 2z_1^2 - 1 + 2z_1 z_0 + 2z_0^2 - 2z_1 z_3 - 2z_0 z_3 + 2z_3^2, z_1 v_{6q} + z_0 v_{6q} - 2s_0 + z_1, 4s_0^2 - 4s_0 z_1 - z_0^2 + 2z_1 z_3 + 2z_0 z_3 - 2z_3^2, 2s_0 v_1 - z_3 v_1 - z_0 + z_3, 2s_0 v_{6q} - z_0 v_{6q} + 2z_3 v_{6q} - 2s_0 - z_0, 8v_{6q} z_3^2 - 2 - 4v_1 z_3^2 + v_{6q} v_1 + 12s_0 z_1 + 2z_1 z_0 + 2z_0^2 - 12s_0 z_3 - 10z_0 z_3 + 6z_3^2 - 4v_6 q + v_1, 8v_{6q} z_0 z_3 - 1 + 4s_0 z_1 + 2z_1 z_0 - 12s_0 z_3 + 8z_1 z_3 + 2z_0 z_3 - v_{6q}, 2v_1 z_1 z_3 + z_0^2 + 4s_0 z_3 - 4z_1 z_3 - 2z_0 z_3 + 3z_3^2, v_{6q} v_1 z_3 - z_0 v_{6q} - z_3 v_1 - z_0, z_0^3 + 4s_0 z_1 z_3 + z_0^2 z_3 - 2z_1 z_3^2 - z_0 z_3^2 + 3z_3^3 - 2z_3, 8v_{6q} z_0^2 - 2 + 4v_1 z_3^2 - v_{6q} v_1 + 4s_0 z_1 + 6z_1 z_0 + 6z_0^2 - 4s_0 z_3 + 2z_0 z_3 + 2z_3^2 - v_1, 2v_1 z_3^3 - z_1 z_0^2 - 2z_1 z_0 z_3 + 4s_0 z_3^2 - 5z_1 z_3^2 + 2z_3^3 - z_3 v_1, 4v_1 z_3^2 + 1 - v_{6q} v_1^2 + 4v_{6q} v_1 - v_1^2 - 4z_1 z_0 + 16s_0 z_3 - 12z_1 z_3 - 4z_0 z_3 + 8z_3^2 - v_{6q} + 2v_1, \frac{v_{6q}^2 v_1^2 - 1 - 4v_{6q}^2 v_1 + v_{6q}^2 - v_1^2}{2}]$$

M. Pfurner, PhD thesis, Innsbruck, 2007

<http://repository.uibk.ac.at/viewer.alo?viewmode=overview&objid=1015078&page=>



# The TSAI-UPU Parallel Manipulator

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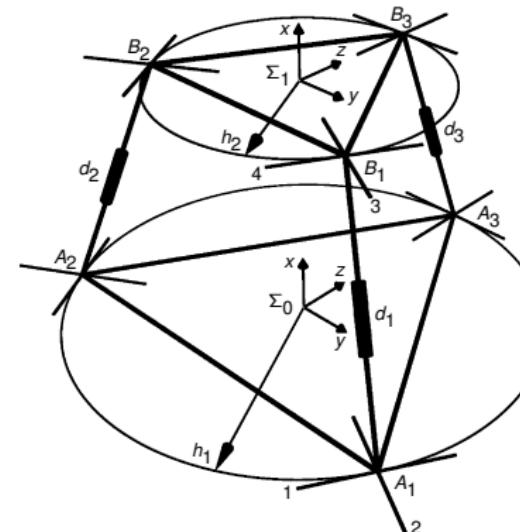
Solving the system of  
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Operation modes

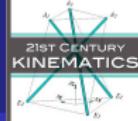
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- difference to the SNU-3UPU manipulator:  
legs are rotated by 90 degrees before assembly



# The algebraic constraint equations

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$$g_1 : x_0 y_0 + x_1 y_1 + x_2 y_2 + x_3 y_3 = 0$$

$$g_2 : (h_1 - h_2) x_0 x_2 + (h_1 + h_2) x_1 x_3 - x_2 y_3 - x_3 y_2 = 0$$

$$g_3 : (h_1 - h_2) x_0 x_3 - (h_1 + h_2) x_1 x_2 - 4 x_1 y_1 - 3 x_2 y_2 - x_3 y_3 = 0$$

$$g_4 : (h_1 - h_2) x_0 x_3 - (h_1 + h_2) x_1 x_2 + 2 x_1 y_1 + 2 x_3 y_3 = 0$$

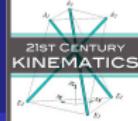
$$\begin{aligned} g_5 : & (h_1^2 - 2h_1 h_2 + h_2^2 - d_1^2) x_0^2 + 2\sqrt{3}(h_1 - h_2) x_0 y_2 - 2(h_1 - h_2) x_0 y_3 + (h_1^2 + 2h_1 h_2 + h_2^2 - d_1^2) x_1^2 - \\ & - 2(h_1 + h_2) x_1 y_2 - 2\sqrt{3}(h_1 + h_2) x_1 y_3 + (h_1^2 - h_1 h_2 + h_2^2 - d_1^2) x_2^2 + 2\sqrt{3}h_1 h_2 x_2 x_3 - \\ & - 2\sqrt{3}(h_1 - h_2) x_2 y_0 + 2(h_1 + h_2) x_2 y_1 + (h_1^2 + h_1 h_2 + h_2^2 - d_1^2) x_3^2 + 2(h_1 - h_2) x_3 y_0 + \\ & + 2\sqrt{3}(h_1 + h_2) x_3 y_1 + 4(y_0^2 + y_1^2 + y_2^2 + y_3^2) = 0 \end{aligned}$$

$$\begin{aligned} g_6 : & (h_1^2 - 2h_1 h_2 + h_2^2 - d_2^2) x_0^2 - 2\sqrt{3}(h_1 - h_2) x_0 y_2 - 2(h_1 - h_2) x_0 y_3 + (h_1^2 + 2h_1 h_2 + h_2^2 - d_2^2) x_1^2 - \\ & - 2(h_1 + h_2) x_1 y_2 + 2\sqrt{3}(h_1 + h_2) x_1 y_3 + (h_1^2 - h_1 h_2 + h_2^2 - d_2^2) x_2^2 - 2\sqrt{3}h_1 h_2 x_2 x_3 + \\ & + 2\sqrt{3}(h_1 - h_2) x_2 y_0 + 2(h_1 + h_2) x_2 y_1 + (h_1^2 + h_1 h_2 + h_2^2 - d_2^2) x_3^2 + 2(h_1 - h_2) x_3 y_0 - \\ & - 2\sqrt{3}(h_1 + h_2) x_3 y_1 + 4(y_0^2 + y_1^2 + y_2^2 + y_3^2) = 0 \end{aligned}$$

$$\begin{aligned} g_7 : & (h_1^2 - 2h_1 h_2 + h_2^2 - d_3^2) x_0^2 + 4(h_1 - h_2) x_0 y_3 + (h_1^2 + 2h_1 h_2 + h_2^2 - d_3^2) x_1^2 + 4(h_1 + h_2) x_1 y_2 + \\ & + (h_1^2 + 2h_1 h_2 + h_2^2 - d_3^2) x_2^2 - 4(h_1 + h_2) x_2 y_1 + (h_1^2 - 2h_1 h_2 + h_2^2 - d_3^2) x_3^2 - 4(h_1 - h_2) x_3 y_0 + \\ & + 4(y_0^2 + y_1^2 + y_2^2 + y_3^2) = 0 \end{aligned}$$

normalization equation is added:

$$g_8 : x_0^2 + x_1^2 + x_2^2 + x_3^2 - 1 = 0 \quad (33)$$



# Solving the system of equations

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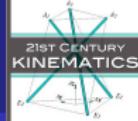
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mechanisms

- polynomial ideal over the ring  $\mathbb{R}[h_1, h_2, d_1, d_2, d_3][x_0, x_1, x_2, x_3, y_0, y_1, y_2, y_3]$

$$\mathcal{I} = \langle g_1, g_2, g_3, g_4, g_5, g_6, g_7, g_8 \rangle$$



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$$\mathcal{J} = \langle g_1, g_2, g_3, g_4, g_5, g_6, g_7, g_8 \rangle$$

- primary decomposition

$$\langle g_1, g_2, g_3, g_4 \rangle = \bigcap_{i=1}^6 \mathcal{J}_i$$

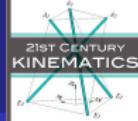
$$\mathcal{J}_1 = \langle y_0, x_1, x_2, x_3 \rangle, \quad \mathcal{J}_2 = \langle x_0, y_1, x_2, x_3 \rangle, \quad \mathcal{J}_3 = \langle y_0, y_1, x_2, x_3 \rangle, \quad \mathcal{J}_4 = \langle x_0, x_1, y_2, y_3 \rangle,$$

$$\begin{aligned}\mathcal{J}_5 = & \langle (h_1 - h_2)x_0 x_2 + (h_1 + h_2)x_1 x_3 - x_2 y_3 - x_3 y_2, \\ & (h_1 - h_2)x_0 x_3 - (h_1 + h_2)x_1 x_2 - x_2 y_2 + x_3 y_3,\end{aligned}$$

$$2x_1 y_1 + x_2 y_2 + x_3 y_3, x_0 y_0 - x_1 y_1, (h_1 - h_2)^2 x_0^2 + (h_1 + h_2)^2 x_1^2 - y_2^2 - y_3^2,$$

$$\begin{aligned}& (h_1 + h_2)x_2^3 y_0 - 3(h_1 - h_2)x_2^2 x_3 y_1 - 2x_2^2 y_0 y_1 - \\ & - 3(h_1 - h_2)x_2 x_3^2 y_0 + (h_1 - h_2)x_3^3 y_1 - 2x_3^2 y_0 y_1\rangle\end{aligned}$$

$$\mathcal{J}_6 = \langle x_0, x_1, x_2, x_3 \rangle.$$



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$$\mathcal{I} = \langle g_1, g_2, g_3, g_4, g_5, g_6, g_7, g_8 \rangle$$

- primary decomposition

$$\langle g_1, g_2, g_3, g_4 \rangle = \bigcap_{i=1}^6 \mathcal{J}_i$$

$$\mathcal{J}_1 = \langle y_0, x_1, x_2, x_3 \rangle, \quad \mathcal{J}_2 = \langle x_0, y_1, x_2, x_3 \rangle, \quad \mathcal{J}_3 = \langle y_0, y_1, x_2, x_3 \rangle, \quad \mathcal{J}_4 = \langle x_0, x_1, y_2, y_3 \rangle,$$

$$\begin{aligned}\mathcal{J}_5 = & \langle (h_1 - h_2)x_0 x_2 + (h_1 + h_2)x_1 x_3 - x_2 y_3 - x_3 y_2, \\ & (h_1 - h_2)x_0 x_3 - (h_1 + h_2)x_1 x_2 - x_2 y_2 + x_3 y_3,\end{aligned}$$

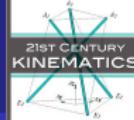
$$2x_1 y_1 + x_2 y_2 + x_3 y_3, x_0 y_0 - x_1 y_1, (h_1 - h_2)^2 x_0^2 + (h_1 + h_2)^2 x_1^2 - y_2^2 - y_3^2,$$

$$\begin{aligned}& (h_1 + h_2)x_2^3 y_0 - 3(h_1 - h_2)x_2^2 x_3 y_1 - 2x_2^2 y_0 y_1 - \\ & - 3(h_1 - h_2)x_2 x_3^2 y_0 + (h_1 - h_2)x_3^3 y_1 - 2x_3^2 y_0 y_1\rangle\end{aligned}$$

$$\mathcal{J}_6 = \langle x_0, x_1, x_2, x_3 \rangle.$$

- decomposition of the vanishing set of  $\mathcal{I}$

$$\mathcal{V}(\mathcal{I}) = \bigcup_{i=1}^5 \mathcal{V}(\mathcal{J}_i \cup \langle g_5, g_6, g_7, g_8 \rangle) = \bigcup_{i=1}^5 \mathcal{V}(\mathcal{K}_i)$$



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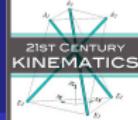
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modes

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mechanisms

- solutions for generic parameters  $h_1, h_2$  and  $d_1, d_2, d_3$ :

$$|\mathcal{V}(\mathcal{K}_1)| = |\mathcal{V}(\mathcal{K}_2)| = 2, |\mathcal{V}(\mathcal{K}_3)| = 4,$$

$$|\mathcal{V}(\mathcal{K}_4)| = 6, |\mathcal{V}(\mathcal{K}_5)| = 64.$$



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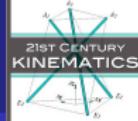
$$|\mathcal{V}(\mathcal{K}_1)| = |\mathcal{V}(\mathcal{K}_2)| = 2, |\mathcal{V}(\mathcal{K}_3)| = 4,$$

$$|\mathcal{V}(\mathcal{K}_4)| = 6, |\mathcal{V}(\mathcal{K}_5)| = 64.$$

- solutions for parameters with  $d_1 = d_2 = d_3$ :

$$|\mathcal{V}(\mathcal{K}_1)| = |\mathcal{V}(\mathcal{K}_2)| = |\mathcal{V}(\mathcal{K}_3)| = 2,$$

$$|\mathcal{V}(\mathcal{K}_4)| = 6, |\mathcal{V}(\mathcal{K}_5)| = 60$$



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- solutions for generic parameters  $h_1, h_2$  and  $d_1, d_2, d_3$ :

$$|\mathcal{V}(\mathcal{K}_1)| = |\mathcal{V}(\mathcal{K}_2)| = 2, |\mathcal{V}(\mathcal{K}_3)| = 4,$$

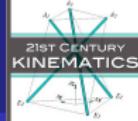
$$|\mathcal{V}(\mathcal{K}_4)| = 6, |\mathcal{V}(\mathcal{K}_5)| = 64.$$

- solutions for parameters with  $d_1 = d_2 = d_3$ :

$$|\mathcal{V}(\mathcal{K}_1)| = |\mathcal{V}(\mathcal{K}_2)| = |\mathcal{V}(\mathcal{K}_3)| = 2,$$

$$|\mathcal{V}(\mathcal{K}_4)| = 6, |\mathcal{V}(\mathcal{K}_5)| = 60$$

- “home pose” is solution of multiplicity 1 (SNU-3UPU → multiplicity 4)



# Operation modes

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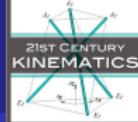
■ partial system  $\mathcal{J}_i \cup \langle g_5, g_6, g_7, g_8 \rangle \longleftrightarrow$  operation mode

■ five different modes:

- translational mode,  $\mathcal{J}_1 = \langle y_0, x_1, x_2, x_3 \rangle$
- twisted translational mode,  $\mathcal{J}_2 = \langle x_0, y_1, x_2, x_3 \rangle$
- planar mode,  $\mathcal{J}_3 = \langle y_0, y_1, x_2, x_3 \rangle$
- upside-down planar mode,  $\mathcal{J}_4 = \langle x_0, x_1, y_2, y_3 \rangle$
- general mode,  $\mathcal{J}_5 = \langle \dots \rangle$

Transformation matrix for translational mode

$$\begin{pmatrix} 1 & 0 & 0 & 0 \\ -2y_1 & 1 & 0 & 0 \\ -2y_2 & 0 & 1 & 0 \\ -2y_3 & 0 & 0 & 1 \end{pmatrix}$$



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- conditions on  $h_1, h_2, d_1, d_2, d_3$  for singular poses are computable

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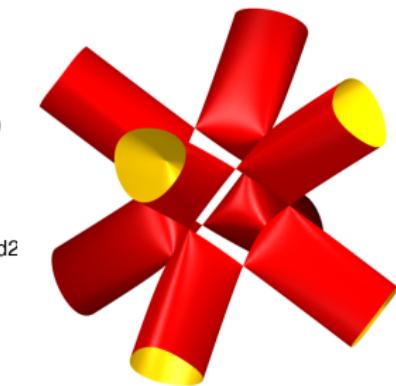
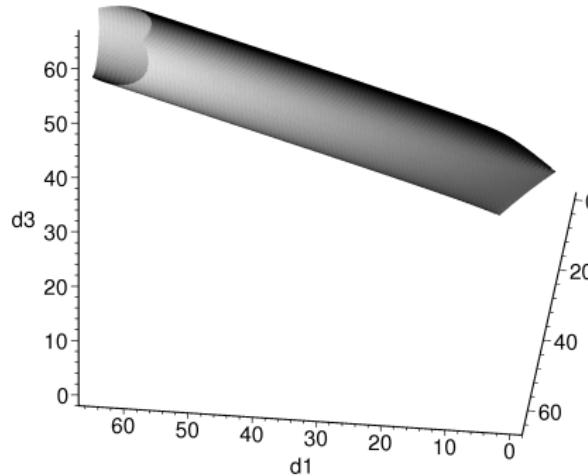
- conditions on  $h_1, h_2, d_1, d_2, d_3$  for singular poses are computable
- Example: translational mode

$$d_1^4 + d_2^4 + d_3^4 - d_1^2 d_2^2 - d_1^2 d_3^2 - d_2^2 d_3^2 - \\ - 3(h_1 - h_2)^2(d_1^2 + d_2^2 + d_3^2) + 9(h_1 - h_2)^4 = 0$$

# Singular poses

- conditions on  $h_1, h_2, d_1, d_2, d_3$  for singular poses are computable
- Example: translational mode

$$d_1^4 + d_2^4 + d_3^4 - d_1^2 d_2^2 - d_1^2 d_3^2 - d_2^2 d_3^2 - \\ - 3(h_1 - h_2)^2(d_1^2 + d_2^2 + d_3^2) + 9(h_1 - h_2)^4 = 0$$



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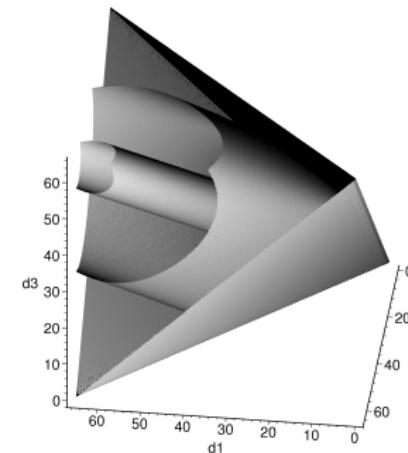
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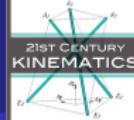
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## ■ Example: planar mode



$$F_1 F_2 (d_1 + d_2 - d_3) (d_1 + d_3 - d_2) (d_2 + d_3 - d_1) F_3 = 0$$



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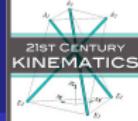
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- change of operation mode only at special poses possible



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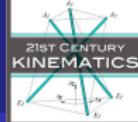
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- change of operation mode only at special poses possible
- dimensions of ideal intersections

	$\mathcal{K}_1$	$\mathcal{K}_2$	$\mathcal{K}_3$	$\mathcal{K}_4$	$\mathcal{K}_5$
$\mathcal{K}_1$	3	-1	2	-1	2
$\mathcal{K}_2$	-1	3	2	-1	2
$\mathcal{K}_3$	2	2	3	-1	2
$\mathcal{K}_4$	-1	-1	-1	3	2
$\mathcal{K}_5$	2	2	2	2	3



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- mode change poses are also singular poses
- conditions on  $h_1, h_2, d_1, d_2, d_3$  for such poses are computable

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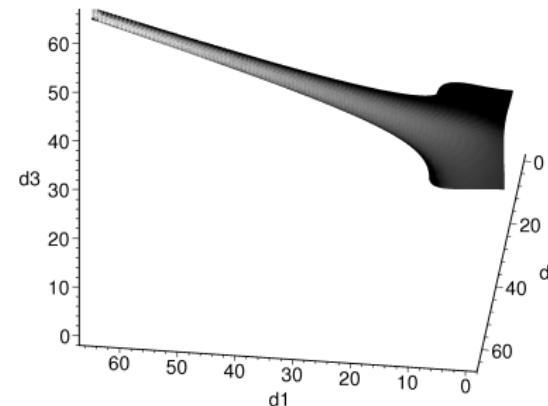
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- mode change poses are also singular poses
- conditions on  $h_1, h_2, d_1, d_2, d_3$  for such poses are computable
- Example: translational mode  $\longleftrightarrow$  general mode

$$d_1^4 + d_2^4 + d_3^4 - d_1^2 d_2^2 - d_1^2 d_3^2 - d_2^2 d_3^2 - 36(h_1 - h_2)^4 = 0$$



$$h_1 = 12, h_2 = 7$$

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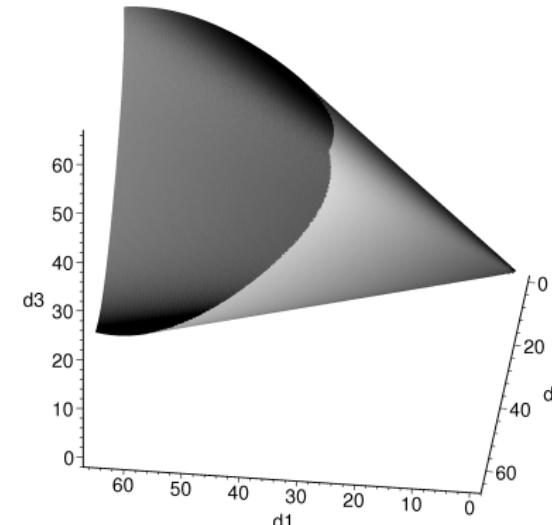
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- Example: planar mode  $\longleftrightarrow$  general mode

$$7(d_1^4 + d_2^4 + d_3^4) - 11(d_1^2 d_2^2 - d_1^2 d_3^2 - d_2^2 d_3^2) = 0$$



$$h_1 = 12, h_2 = 7$$



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# Most complicated transition are transitions to general mode

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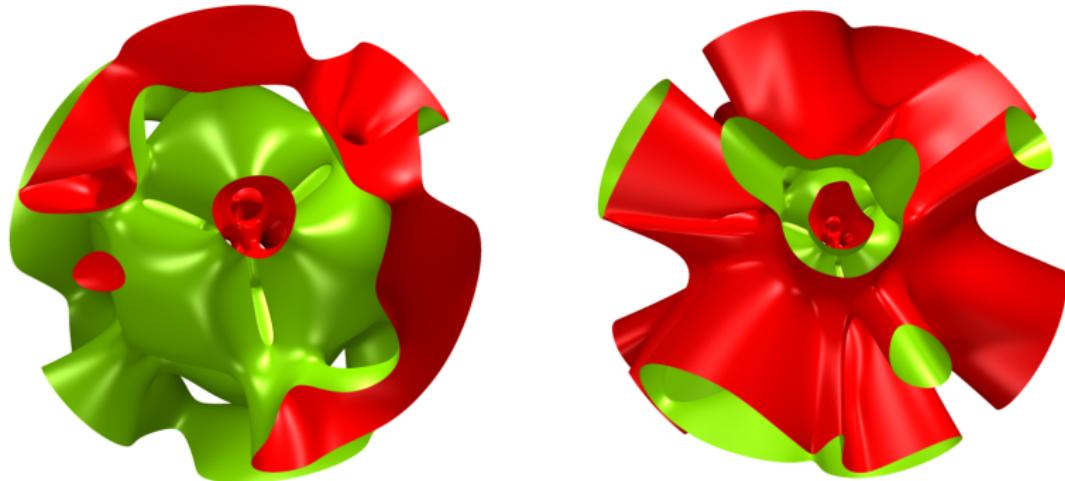
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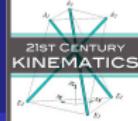
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Most complicated transition are transitions to general mode



Transition surfaces of degree 24



# Synthesis of mechanisms

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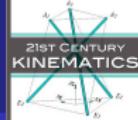
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Changing the point of view the same constraint equations can be used for mechanism synthesis



# Synthesis of mechanisms

Changing the point of view the same constraint equations can be used for mechanism synthesis

- Function synthesis
- Trajectory synthesis
- Motion synthesis

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Changing the point of view the same constraint equations can be used for mechanism synthesis

- Function synthesis
- Trajectory synthesis
- Motion synthesis

Planar Burmester problem:

Given five poses of a planar system, construct a fourbar mechanism whose endeffector passes through all five poses

BURMESTER L. (19th century)

It is well known that the solution of this problem yields four dyads that can be combined to six four-bar mechanisms

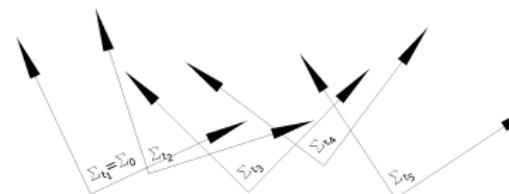
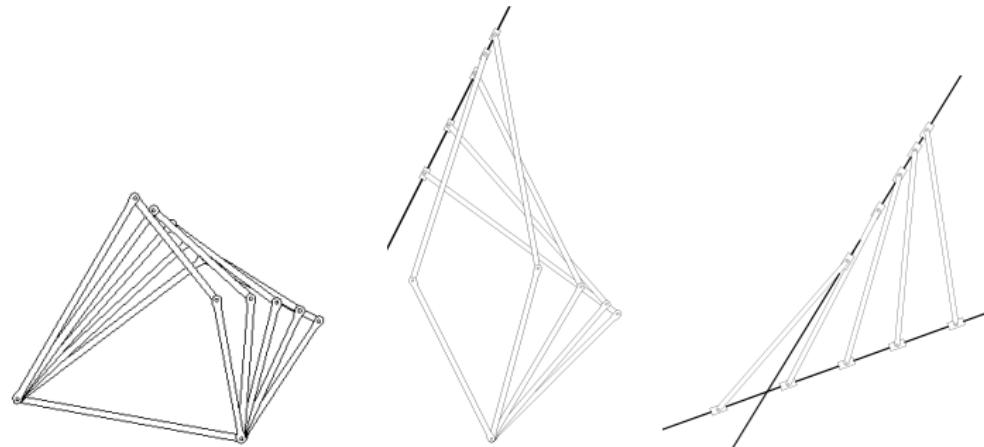


Figure: Five given poses



**Figure:** All possible four-bar mechanisms: a general one, a slider crank and a double slider mechanism

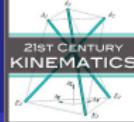
Here the expanded version of the constraint equation has to be used

$$\begin{aligned}
 & (R^2 - C_1^2 - C_2^2 - C_0(x^2 + y^2) + 2C_1x + 2C_2y)X_0^2 \\
 & + (R^2 - C_1^2 - C_2^2 - C_0(x^2 + y^2) - 2C_1x - 2C_2y)X_1^2 \\
 & + ((4C_2x - 4C_1y)X_1 + (4C_0y - 4C_2)X_2 + (-4C_0x + 4C_1)X_3)X_0 \\
 & + ((4C_1 + 4C_0x)X_2 + (4C_0y + 4C_2)X_3)X_1 - 4C_0X_3^2 - 4C_0X_2^2 = 0.
 \end{aligned} \tag{34}$$

$X_i$  image space coordinates

$C_i$  centers of the fixed pivots

$x, y$  centers of the moving pivots



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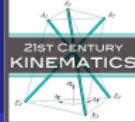
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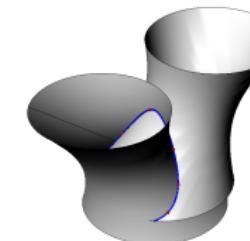
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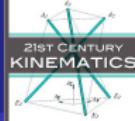


One of those points can be considered to be the point corresponding to the identity

$$(X_0 : X_1 : X_2 : X_3) = (1 : 0 : 0 : 0) \quad (35)$$

this simplifies the constraint equation

$$(-X_0 X_3 x + X_0 X_2 y + X_1 X_2 x + X_3 X_1 y - X_2^2 - X_3^2) C_0 - X_0 X_2 C_2 + X_0 X_3 C_1 + X_0 X_1 x C_2 - X_1^2 x C_1 + X_1 X_2 C_1 - X_0 X_1 y C_1 - X_1^2 y C_2 + X_1 X_3 C_2 = 0 \quad (36)$$



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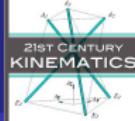
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Now the four remaining poses are given via their image space coordinates:  
 $X_{ij}, j = 1 \dots 4.$



Now the four remaining poses are given via their image space coordinates:  
 $X_{ij}, j = 1 \dots 4.$

It would be important for the designer to know in advance if among the synthesized mechanisms is a slider crank. This is the case if the following two conditions are fulfilled:

$$E1 : \left( -\frac{x_{13}(-x_{11}^2 x_{02} x_{22} + x_{01} x_{21} x_{12}^2 - x_{11} x_{31} x_{12}^2 + x_{11}^2 x_{12} x_{32})}{x_{11} x_{12} (x_{01} x_{12} - x_{11} x_{02})} + x_{23} \right) x_{03} - \frac{(x_{11} x_{31} x_{02} x_{12} - x_{01} x_{11} x_{12} x_{32} - x_{01} x_{21} x_{02} x_{12} + x_{01} x_{11} x_{02} x_{22}) x_{13}^2}{x_{12} x_{11} (x_{01} x_{12} - x_{11} x_{02})} - x_{13} x_{33} = 0 \quad , \quad (37)$$

$$E2 : \left( -\frac{x_{14}(-x_{11}^2 x_{02} x_{22} + x_{01} x_{21} x_{12}^2 - x_{11} x_{31} x_{12}^2 + x_{11}^2 x_{12} x_{32})}{x_{11} x_{12} (x_{01} x_{12} - x_{11} x_{02})} + x_{24} \right) x_{04} - \frac{(x_{11} x_{31} x_{02} x_{12} - x_{01} x_{11} x_{12} x_{32} - x_{01} x_{21} x_{02} x_{12} + x_{01} x_{11} x_{02} x_{22}) x_{14}^2}{x_{12} x_{11} (x_{01} x_{12} - x_{11} x_{02})} - x_{34} x_{14} = 0 \quad . \quad (38)$$

If a double slider is among the synthesized mechanisms then a third (more complicated compatibility condition has to be fulfilled



# Some examples

## General four-bar mechanism

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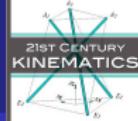
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$C_0$	1	1
$C_1$	2	6
$C_2$	2	1
$x$	7,3821	9,1605
$y$	4,2434	1,1070

Table: Design parameter of mechanism 1

	pose 1	pose 2	pose 3	pose 4
$a$	-0,245005	-0,914683	-2,056744	-3,054058
$b$	0,523260	1,240571	2,235073	3,179009
$\phi$	0,101061	0,116316	0,072202	-0,013746

Table: Given relative poses



# Some examples

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$\phi$	0,101061	0,116316	0,072202	-0,013746

Table: Given relative poses

	solution 1	solution 2	solution 3	solution 4
$C_0$	1	1	1	1
$C_1$	-34.640483	1.999996	6.000008	-4.402381
$C_2$	-29.947423	2.000000	0.999996	16.136008
$x$	18.091483	7.382096	9.160473	-3.697626
$y$	17.844191	4.243444	1.106973	13.877304
$\Rightarrow R$	71.166696	5.830956	3.162275	2.366097

Table: Obtained results

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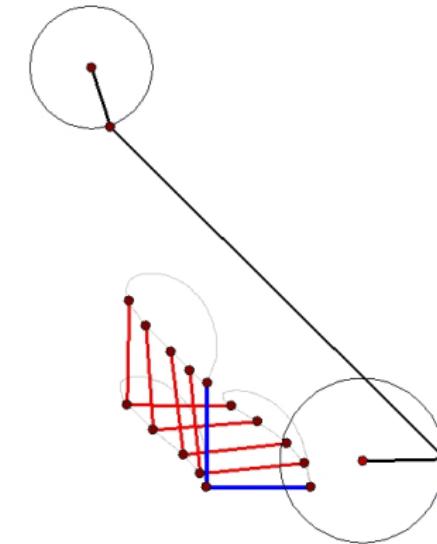
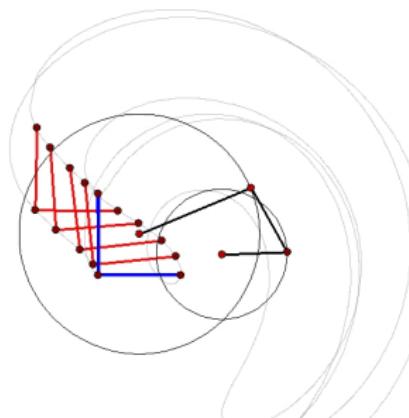
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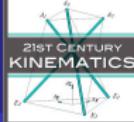
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show animation



## Kinematics and Algebraic Geometry

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## Example: spherical Burmester problem

*Given five poses of a spherical system, construct a four-bar mechanism whose endeffector passes through all five poses.*

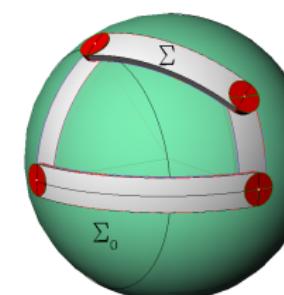


## Example: spherical Burmester problem

*Given five poses of a spherical system, construct a four-bar mechanism whose endeffector passes through all five poses.*

Spherical circle constraint equation:

$$\begin{aligned} SCS : & 4Acx_0x_2 - 4Abx_0x_3 + 4Bax_0x_3 - 4Bcx_3x_2 - 4Cax_0x_2 - 4Cbx_3x_2 \\ & - 2Aa - 2Bb - 2Cc + 4Bbx_3^2 + 4Ccx_2^2 + 4Aax_3^2 + 4Aax_2^2 + 4x_2^2 Cc + \\ & 4x_1^2 Bb - 4x_1 Bcx_0 + 4x_1 Cbx_0 - 4x_1 Abx_2 - 4x_1 Bax_2 - 4x_1 Acx_3 \\ & - 4x_1 Cax_3 + B^2 + A^2 + C^2 + a^2 + b^2 + c^2 - r^2 = 0. \end{aligned} \quad (39)$$





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$$DCS : \mathbf{w}^T \begin{pmatrix} \mathbf{I} & -2\mathbf{B} & \mathbf{0} \\ -2\mathbf{B} & \mathbf{I} & \mathbf{0} \\ \mathbf{0}^T & \mathbf{0}^T & -1 \end{pmatrix} \mathbf{w} = 0 \quad (40)$$

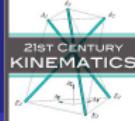
Without loss of generality we can assume that the fixed system  $\Sigma_0$  coincides with one of the five given orientations.

$$DCS_1 := -2Bb - 2Cc - 2Aa + A^2 + C^2 + B^2 + a^2 + b^2 + c^2 - r^2 = 0. \quad (41)$$

Now four simple equations are built by subtracting  $DCS_1$  from the other four constraint equations:

$$M_{1j} = DCS_j - DCS_1, \quad j = 2, \dots, 5.$$

The four difference equations are bilinear in the unknowns  $A, B, C, a, b, c$  and do not contain  $r$ .



## Solution algorithm:

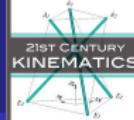
- Two of these equations, say  $M_{1,2}$  and  $M_{1,3}$  are used to solve linearly for two of the unknowns, say  $a, b$ .
- The solutions are substituted into  $M_{1,4}$  and  $M_{1,5}$ . This yields two cubic equations  $C_1, C_2$ .
- The resultant of  $C_1, C_2$  with respect to one of the remaining unknowns, say  $B$  yields a univariate polynomial  $Q^9$  of degree nine in the unknown  $A$ .
- $Q^9$  factors into the solution polynomial  $Q^6$  of degree six and in three linear factors.

## Remarks:

- the univariate can be computed without specifying the pose parameters!
- Branch defect can also be easily detected with this approach!

Brunnhaler, Schröcker, and H., Synthesis of spherical four-bar mechanisms using spherical kinematic mapping. Advances in Robot Kinematics, 2006.

Schröcker and H., Kinematic mapping based assembly mode evaluation of spherical four-bar mechanisms. Proceedings of IFToMM 2007, Besançon, 2007.



# Example

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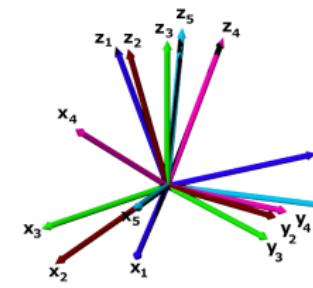
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	$x_0$	$x_1$	$x_2$	$x_3$
Pose1	1	0	0	0
Pose2	0.37721	0.82336	0.38967	0.16722
Pose3	0.0078934	-0.041131	0.085164	-0.99549
Pose4	0.039457	0.77456	-0.60494	-0.18041
Pose5	-0.30301	-0.36492	0.85697	0.20157

Table: Input data for the example

This example yields six real dyads that can be combined to 15 real spherical four-bars.



Five input poses

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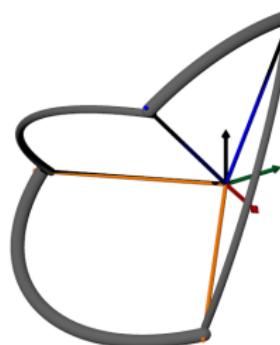
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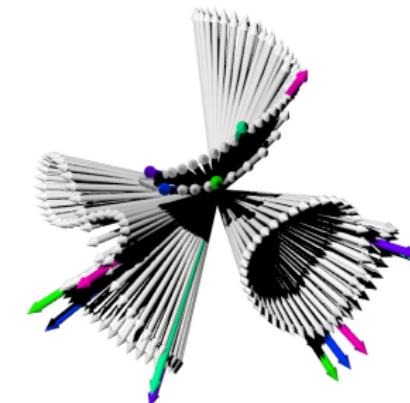
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One solution four-bar



Motion of the coordinate frame



Motion of a rigid body