



# DYNAMICS OF MACHINERY

## ME314

### COURSE PROJECT

**TITLE:** DYNAMIC ANALYSIS OF MULTI-STORY BUILDING  
DUE TO EARTHQUAKE EXCITATION

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# CAD MODEL



**Reference:** [https://www.researchgate.net/publication/312274411\\_Estimation\\_of\\_Storey\\_Stiffness\\_in\\_Multi-storey\\_Buildings](https://www.researchgate.net/publication/312274411_Estimation_of_Storey_Stiffness_in_Multi-storey_Buildings)

Matlab Code Defining the model-

%Input

N=5; %Number of stories

Ks=1.44e8; %Stiffness

Ms=2.0e8; %Mass of each floor

floor\_height = 3;

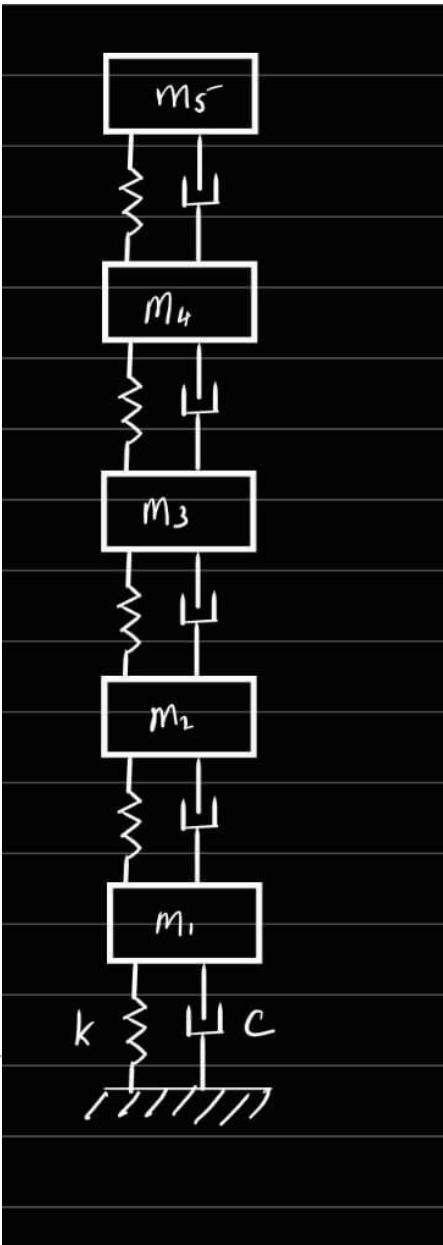
Dimensions:

Thickness of each Storey 0.2m;

Height = 0.3m;

Length \* breadth = 12m \* 10m

# SIMPLIFIED MODEL



## Key Features

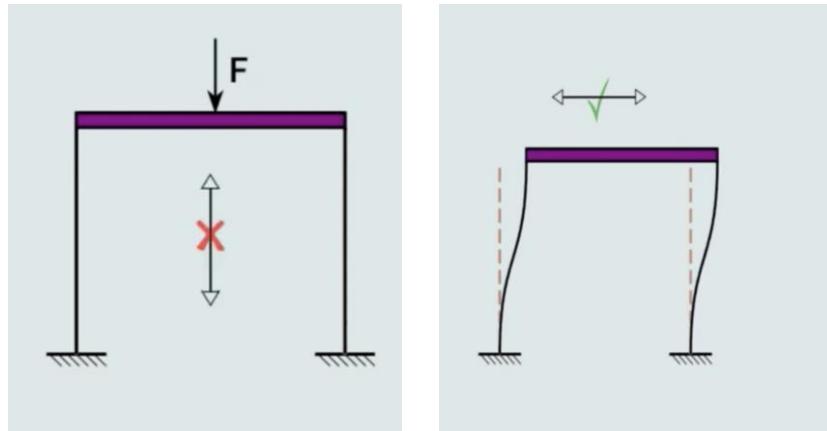
- Floors as Lumped Masses:
  - Each floor's mass is concentrated at a point (center of the slab), ignoring distributed mass.
- Vertical Members as Springs:
  - The columns or walls between floors are modeled as springs that only resist *horizontal* motion (shear deformation), not bending.
- Movement Type:
  - Only allows each floor to move back and forth (side to side), not rotate or bend—each floor acts as a rigid slab sliding horizontally.
- Equations of Motion:
  - The system is described by a set of coupled differential equations, one for each mass/floor, capturing how each interacts with the rest through the springs

Lumped-mass shear building model

Reference-**Seismic Analysis of Multistorey Reinforced Concrete Structures**

# *SOME IMPORTANT TERMS*

- Shear Frame- Special type of frame used to simplify the dynamic behavior of Single Story and multi story building.
  - Doesn't allow vertical deflections or span rotations, but allows horizontal sway.
  - Mode Shape- Each Natural Frequency is associated with a specific shape of vibration known as the mode shape



## References:-

<https://ijret.org/volumes/2018v07/i03/IJRET20180703009.pdf>

[https://www.iitk.ac.in/nicee/wcee/article/14\\_K001.pdf](https://www.iitk.ac.in/nicee/wcee/article/14_K001.pdf)

<https://ijettjournal.org/archive/ijett-v7i2p247>

## *Free Vibration of MDOF System*

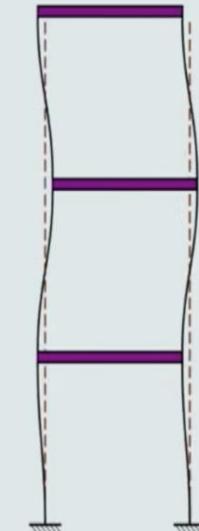
## Mode 1 - T1



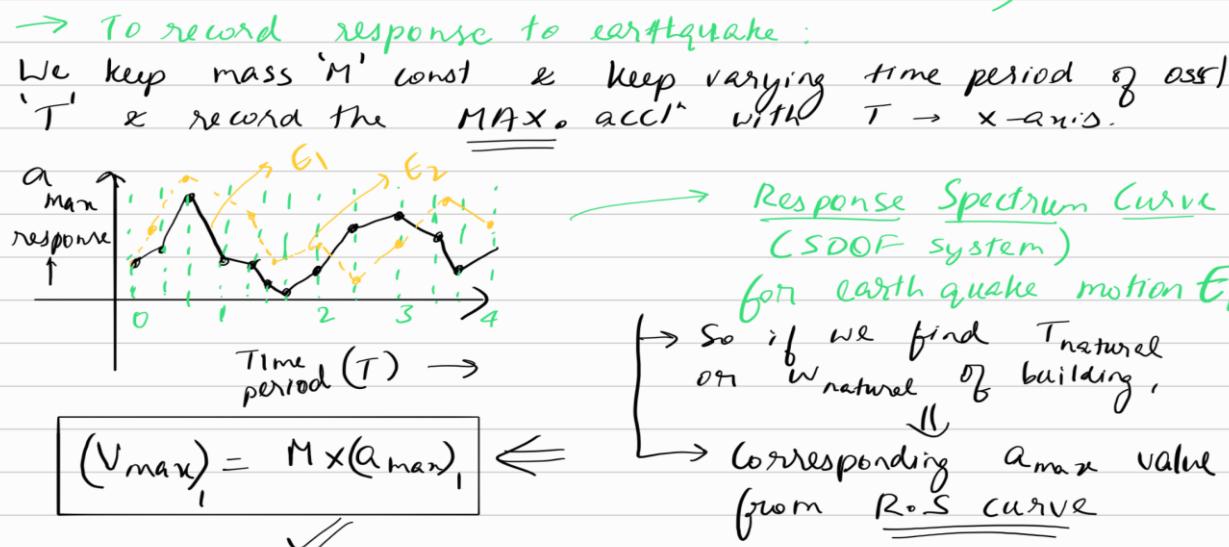
## Mode 2 - T2



### Mode 3 - T3

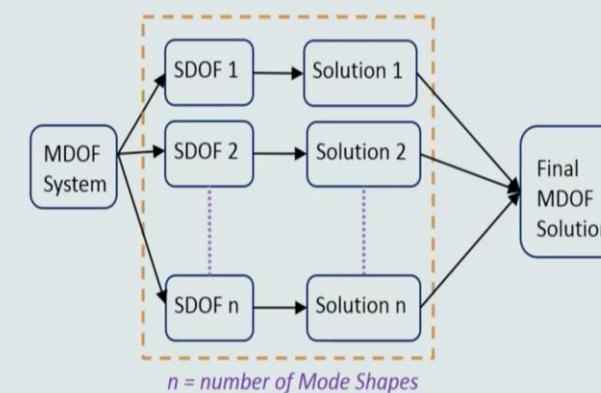


# THEORETICAL BACKGROUND



## Introduction to Dynamic Analysis

### Dynamic Analysis: Modal Analysis



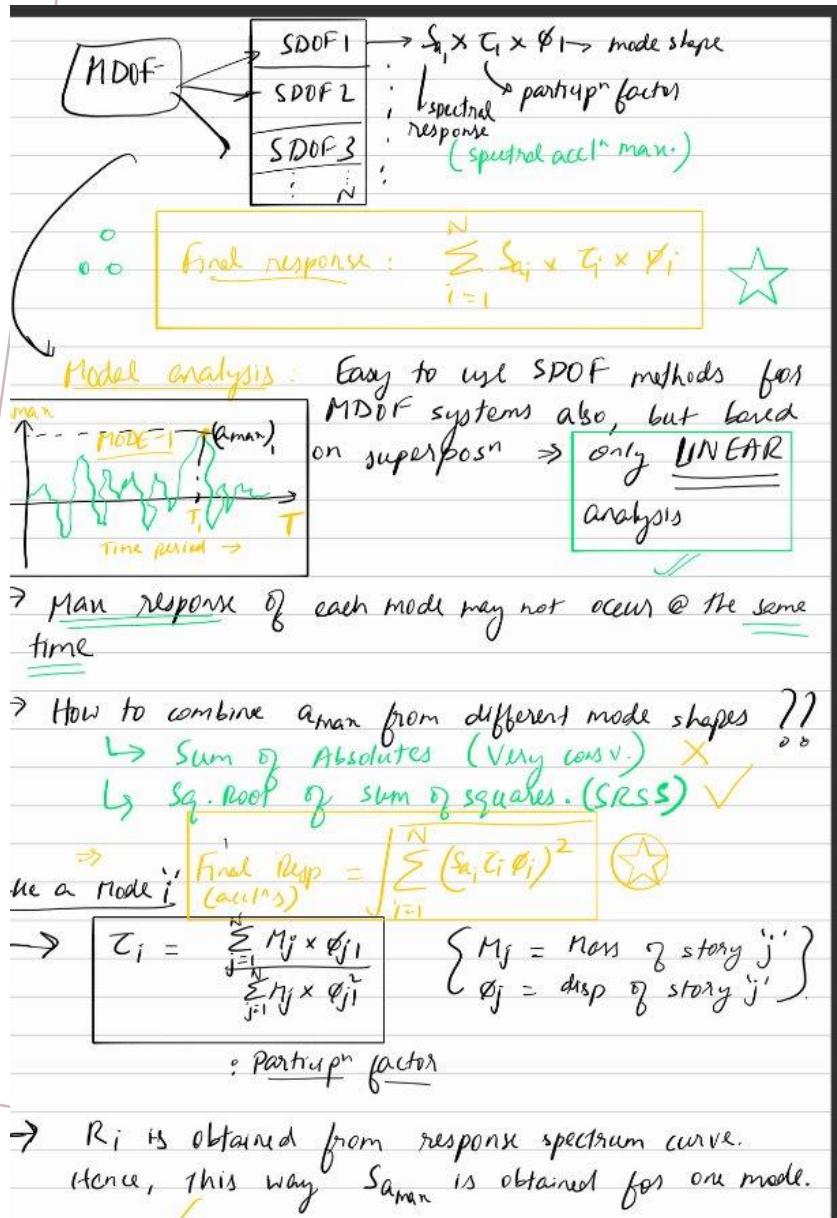
- The MDOF Structure is converted to number of SDOF systems according to Mode Shapes
- Each SDOF system is solved separately
- The Envelope of the Solutions of the SDOF systems are combined to obtain an Envelope Solution for the MDOF Problem.

We have to solve the problem for Multi Degree of freedom system

We will be using the method of Modal Analysis because –

- The solution is fast compared to Time history method
- Note- We will be getting the Approx. solution with max. and min. values , and our solution is limited to Linear problems.

## *MODAL ANALYSIS – OBTAINING UNCOUPLING MATRICES*



$R_i$  is obtained from response spectrum curve.  
Hence, this way  $S_{\text{max}}$  is obtained for one mode.

## Implementation

- Classical lumped mass shear building model.  
— 3 stories.

⇒ 3D → NDOF - 1D lateral system.

→ Mathematical model :  $M\ddot{x} + C\dot{x} + Kx = F(t)$

(\*) Imp. parameters : Interstory stiffness  $\rightarrow K_j$

& Rayleigh damping :  $\zeta = 0.05 \equiv 5\%$  {Assuming}  
 $\rightarrow \alpha_1 \beta = \underline{\quad}$

→  $f_{natural}, \tau_i, \beta_i, M_i, m_{Mi}, \phi_i$  {Mode shapes}

[ Rel. amplitude of displ. when building vibrates in  $i$ th mode  
(or) eigen vector corr. to eigen value  $w_j^2$  : Natural freq of mode  $j$ . ]

$\phi_i \rightarrow$  how str. deforms ;  $q_{ij} \rightarrow$  how much it deforms @ that instant

↳ Eigen vector  $\phi$   $\left[ \begin{matrix} [K] - \omega^2 [M] \\ [[K] - \omega^2 [M]] \cdot [\phi] = 0 \end{matrix} \right]$  ic:  $w_j = \sqrt{\lambda_j}$   
( $f_j = w_j \frac{2\pi}{T}$ ) (eigenvalues)

(\*) Uncoupling : To uncouple  $[M]$  &  $[K]$ , we use the orthogonality prop, with help of  $\phi$  :-

CONVERTING PHY space to MODAL coord.  
CODE :  $M_n = \text{diag}(\text{Vectors}^T * M * \text{Vectors})$ ; {uncoupled mass matrix}

$[\phi]^T [M] [\phi] = [M^*], [\phi]^T [K] [\phi] = [K^*]$

Coupled Model Mass matrix → Uncoupled = diagonalized

Modal stiffness matrix

Now, we also need to uncouple [c] ; but can't do it

# OBTAINING VELOCITY AND ACCELERATION FROM THE DISPLACEMENT

with  $\langle \phi \rangle$  for any random  $[C]$ : So use Rayleigh Damping  
 $\Rightarrow C = \alpha [M] + \beta [K]$   
 mass prop. stiffness prop.  
 damping weight ..  
 not physical law

&  $\zeta_i = \frac{\alpha}{2\omega_i} + \frac{\beta\omega_i}{2}$  : Find  $\alpha, \beta$  by setting  $\zeta_i$  values  
 $\alpha : \omega_i \uparrow \text{ses} \Rightarrow \zeta_i \downarrow \text{ses}$   
 $\beta : \omega_i \uparrow \text{ses} \Rightarrow \zeta_i \uparrow \text{ses}$

Now  $[C]$  can be uncoupled to  $[C^*]$  using  $\langle \phi \rangle$   
 $[C^*] = \alpha [M^*] + \beta [K^*]$  : Can be proved

∴ NDOF system  $\xrightarrow{\text{uncoupled}}$  N-SDOF systems.

For a mode "i":  $M_i^* \ddot{q}_i + C_i^* \dot{q}_i + K_i^* q_i = F_i^*(t)$

To obtain  $[F^*]$ ;  $[F^*] = \langle \phi \rangle^T [F]$

\* Computing the SDOF response from above eqn :-

Analytical solution by convolving:  $q_i(t) = (F_i^* * h_i)(t) \cdot \Delta t$

where  $h_i(t) = \frac{1}{M_i^* \omega_{d,i}} e^{-\xi_i \omega_i t} \sin(\omega_i t)$

called impulse response f^n -

## 2. Generalized Duhamel's integral

A linear SDOF system is governed by the following initial value problem

$$m\ddot{u}(t) + cu(t) + ku(t) = p(t), \quad u(0) = \dot{u}(0) = 0, \quad (2.1)$$

where  $m$ ,  $c$  and  $k$  are, respectively, mass, damping coefficient and stiffness of the system, while  $u(t)$  and  $p(t)$  denote the time-dependent displacement and force (load) functions with dot representing, as usual, differentiation with respect to time  $t > 0$ . The solution of the initial value problem defined by equation (2.1) is the convolution integral

$$u(t) = \int_0^t h(\tau)p(t-\tau) d\tau, \quad h(t) = e^{-\xi\omega t} \sin(\omega_d t) \omega_d^{-1} m^{-1}, \quad (2.2)$$

also known as Duhamel's integral [2], in which the parameters  $\omega$ ,  $\xi$  and  $\omega_d$  are defined by the relations

$$\omega^2 = m^{-1}k, \quad \xi\omega = \frac{1}{2}m^{-1}c, \quad \omega_d^2 = \omega^2 - (\xi\omega)^2, \quad (2.3)$$

denoting the natural frequency, the damping ratio and the damped frequency of the SDOF system, respectively.

$\left[ \text{CODE: conv}\left(f_n(i,:), h(i,:) * dt\right) \right]$

Similarly convolute for vel.  $\dot{q}_i(t)$  & accl<sup>n</sup>  $\ddot{q}_i(t)$ .

Total Response :-  $u(t) = \Phi \dot{q}_i(t); \quad v(t) = \Phi \ddot{q}_i(t);$   
 $\alpha(t) = \Phi \dddot{q}_i(t)$



Mode shape matrix  $\Phi$  (Transform modal resp. back to physical space)

$\left\{ \begin{array}{l} n = \text{vectors} * \xi_i \\ v = " * \omega_d \end{array} \right\} \rightarrow \text{Convolution:}$

$$q(t) = (h * f)(t) = \int_0^t h(\tau) f(t-\tau) d\tau$$

$q \rightarrow$  weighted sum of all past forces  
 Current displ. is affected by entire history of applied force, depending on how system weights past inputs.

## References:

<https://doi.org/10.1098/rspa.2021.0576>

<https://doi.org/10.3390/buildings14051286>

# MATLAB CODE

```

1 clear; clc; close all;
2
3 %Input
4 N=5; %Number of stories
5 Ks=1.44e8; %Stiffness
6 Ms=2.0e8; %Mass of each floor
7 floor_height = 3;
8
9 %Using Ground Motion Data (Acceleration(m/s2) vs Time)
10 data = readmatrix('Dataset.csv');
11 t = data(:,1);
12 ag = data(:,2);
13
14 dt = mean(diff(t)); % Time step (s)
15 fs = 1/dt; % Sample Frequency (Hz)
16
17 %Plot 1: Input Ground Acceleration (from CSV)
18 figure;
19 plot(t, ag);
20 xlabel('Time (s)'); ylabel('a_g (m/s^2)');
21 title('Input Ground Acceleration (from CSV)');
22 grid on;
23
24
25 M=Ms*diag(ones(N,1)); %Mass matrix
26 K=MultiStory_Stiffness(Ks,N); %Stiffness matrix
27
28 %Damping matrix
29 [V,D] = eig(K,M);
30 wn = sqrt(diag(D)); % natural frequencies (rad/s)
31 w1 = wn(1); w2 = wn(min(2,N));
32 zeta = 0.05; % 5% damping
33 A = [1/(2*w1), w1/2; 1/(2*w2), w2/2];
34 x = A\zeta; zeta;
35 alpha = x(1); beta = x(2);
36 C = alpha*M + beta*K;
37
38 % Force matrix
39 ag = ag(:)';
40 f = zeros(N, length(ag));
41 for i = 1:N
42     f(i,:) = -Ms * ag;
43 end

```

```

45 Result=MDOF_simulation(M,C,K,f,fs); %MDOF solver
46
47 %Acceleration, Velocity, Time Response Plot
48 t=[0:1/fs:(length(ag)-1)/fs];
49 figure;
50 for i=1:N
51     subplot(N,3,3*i-2)
52     plot(t,Result.Acceleration(i,:)); xlabel('Time(s)'); ylabel(strcat('Acc',num2str(i)));
53     subplot(N,3,3*i-1)
54     plot(t,Result.Velocity(i,:)); xlabel('Time(s)'); ylabel(strcat('Vel',num2str(i)));
55     subplot(N,3,3*i)
56     plot(t,Result.Displacement(i,:)); xlabel('Time(s)'); ylabel(strcat('Disp',num2str(i)));
57 end
58
59 %Modeshapes Plot
60 figure;
61 xscale = 10.0e5;
62 yscale = 25;
63 for i=1:N
64     plot([0 ;xscale*Result.Parameters.ModeShape(:,i)],yscale*(0:N),'*-');
65     hold on
66 end
67 hold off
68 xlabel('Amplitude');
69 ylabel('Floor');
70 grid on
71 daspect([1 1 1]);
72 title('Modeshapes');
73
74
75 %Interstory Drift
76 u = Result.Displacement;
77 interstory_drift = diff(u,1,1);
78 max_drift = max(abs(interstory_drift),[],2);
79 drift_ratio = max_drift / floor_height;
80
81
82 %Drift Ratios Plot
83 figure;
84 bar(1:length(drift_ratio), drift_ratio);
85 xlabel('Story Level');
86 ylabel('Max Drift Ratio');
87 title('Interstory Drift Ratio per Story');
88 grid on;
89
90
91
92 a = Result.Acceleration; % (N x time) floor accelerations
93 m = Ms * ones(N,1); % (N x 1) mass per floor
94 base_shear = (m' * a); % (1 x time) total shear at base
95 base_shear = base_shear(:); % ensure row vector for plotting
96 max_base_shear = max(abs(base_shear));
97
98 %Base Shear Time History Plot
99 figure;
100 plot(t(1:length(base_shear)), base_shear, 'LineWidth', 1.2);
101 xlabel('Time (s)');
102 ylabel('Base Shear (N)');
103 title('Base Shear Time History');
104 grid on;
105
106 %Base Shear vs Drift Ratio Plot
107 figure;
108 plot(drift_ratio, base_shear);
109 xlabel('Drift Ratio');
110 ylabel('Base Shear (N)');
111 title('Base Shear vs Drift Ratio');
112 grid on;

```

```

111 function Result=MDOF_simulation(M,C,K,f,fs)
112
113 n=size(f,1);
114 dt=1/fs; %sampling rate
115 [Vectors, Values]=eig(K,M);
116 Freq=sqrt(diag(Values))/(2*pi); % undamped natural frequency
117 steps=size(f,2);
118
119 Mn=diag(Vectors'*M*Vectors); % uncoupled mass
120 Cn=diag(Vectors'*C*Vectors); % uncoupled damping
121 Kn=diag(Vectors'*K*Vectors); % uncoupled stiffness
122 wn=sqrt(diag(Values));
123 zeta=Cn./(2*sqrt(Mn.*Kn)); % damping ratio
124 wd=wn.*sqrt(1-zeta.^2);
125
126 fn=Vectors'*f; % generalized input force matrix
127
128 t=[0:dt:dt*steps-dt];
129
130 %forced vibration
131
132 for i=1:1:n
133
134 h(i,:)=(1/(Mn(i)*wd(i))).*exp(-zeta(i)*wn(i)*t).*sin(wd(i)*t);
135 hd(i,:)=(1/(Mn(i)*wd(i))).*(-zeta(i).*wn(i).*exp(-zeta(i)*wn(i)*t).*sin(wd(i)*t));
136 hdd(i,:)=(1/(Mn(i)*wd(i))).*((zeta(i).*wn(i))^2.*exp(-zeta(i)*wn(i)*t));
137
138 qq=conv(fn(i,:),h(i,:))*dt;
139 qqd=conv(fn(i,:),hd(i,:))*dt;
140 qqdd=conv(fn(i,:),hdd(i,:))*dt;
141
142 q(i,:)=qq(1:steps); % modal displacement
143 qd(i,:)=qqd(1:steps); % modal velocity
144 qdd(i,:)=qqdd(1:steps); % modal acceleration
145
146
147 end
148
149 x=Vectors*q; %Displacement
150 v=Vectors*qd; %Vecloity
151 a=Vectors*qdd; %Acceleration

```

```

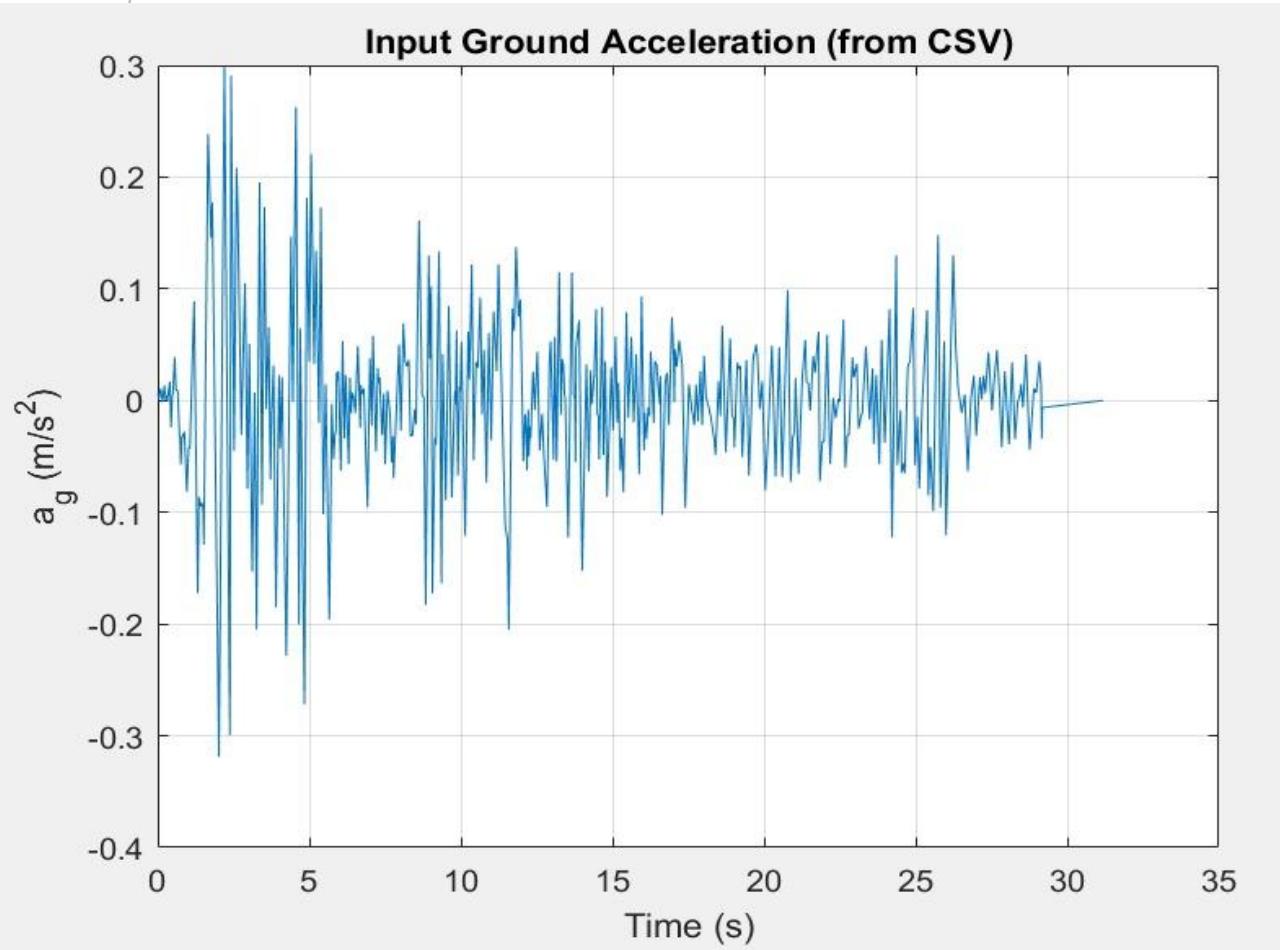
154 % Free vibration
155
156 xi=zeros(n,1); % Displacement initial condition
157 vi=zeros(n,1); % Velocity initial condition
158
159 xno=Vectors'*M*xi./Mn;
160 vno=Vectors'*M*vi./Mn;
161
162 for i=1:1:n
163
164 AA=(vno(i)+xno(i).*zeta(i).*wn(i))./wd(i);
165 BB=xno(i);
166
167 qf(i,:)=exp(-zeta(i)*wn(i)*t).*(AA.*sin(wd(i)*t)+BB.*cos(wd(i)*t));
168 qdf(i,:)=wd(i)*exp(-zeta(i)*wn(i)*t).*(AA.*cos(wd(i)*t)-BB.*sin(wd(i)*t))-zeta(i).*qf(i,:);
169 qddf(i,:)=wd(i)^2.*exp(-zeta(i)*wn(i)*t).*(-AA.*sin(wd(i)*t)-BB.*cos(wd(i)*t));
170
171
172 end
173
174 x=x+Vectors*qf;
175 v=v+Vectors*qdf;
176 a=a+Vectors*qddf;
177
178 Result.Displacement=x;
179 Result.Velocity=v;
180 Result.Acceleration=a;
181 Result.Parameters.Freq=Freq;
182 Result.Parameters.DampRatio=zeta*100;
183 Result.Parameters.ModeShape=Vectors;
184

```

```
189 [-]  
190  
191  
192  
193  
194  
195 [-]  
196 [-]  
197  
198  
199  
200  
201  
202  
203  
204  
205  
206  
207  
208  
209
```

```
function K=MultiStory_Stiffness(Ks,N)  
  
KK=Ks*ones(N+1,1);  
KK(N+1)=0;  
K=zeros(N,N);  
  
for i=1:1:N  
for j=1:1:N  
if i==j  
K(i,j)=KK(i)+KK(i+1);  
end  
if i==j+1  
K(i,j)=-KK(i);  
end  
if j==i+1  
K(i,j)=-KK(j);  
end  
end  
end  
end
```

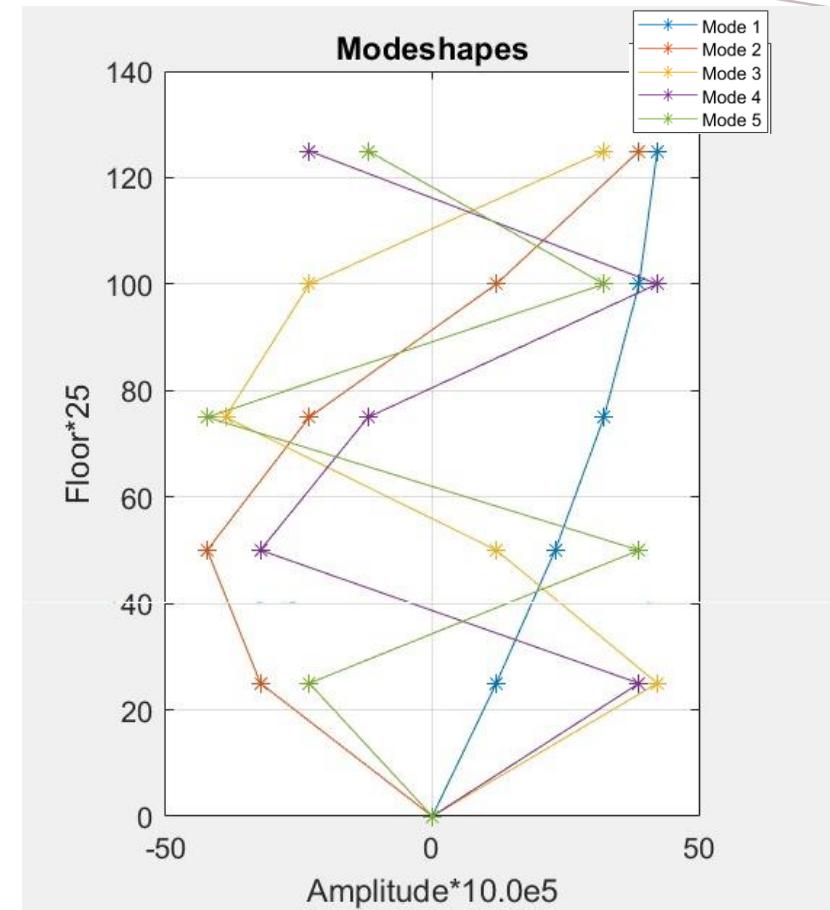
# *INPUT DATA*



## **El Centro Earthquake Dataset**

Reference:- <https://www.vibrationdata.com/elcentro.htm>

# *Mode Shapes-(Output )*

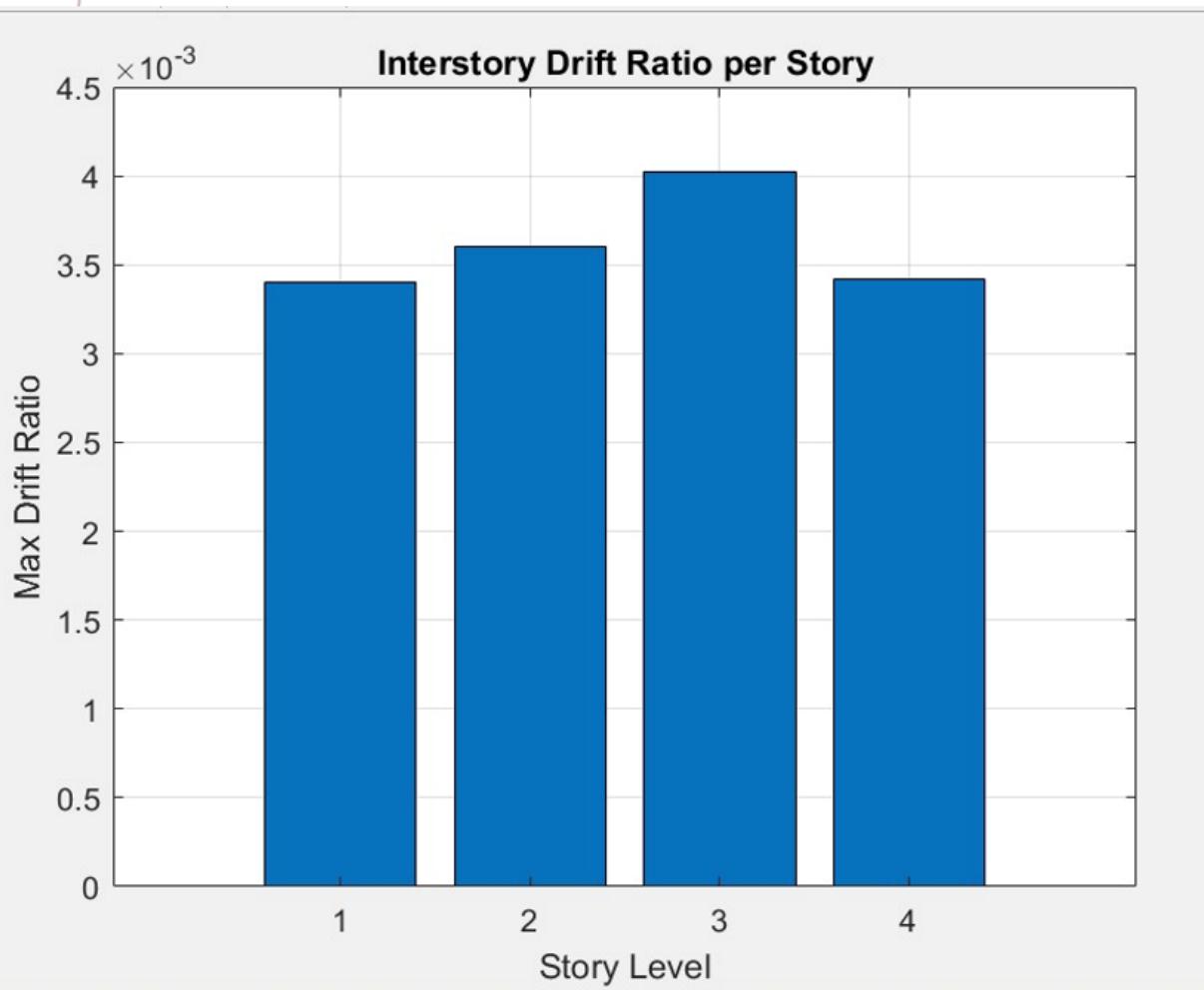


Displacements corresponding to different mode shapes

Mode shapes are crucial in seismic analysis:

- Lower modes usually control overall drift and base shear.
- Higher modes are essential for capturing forces and accelerations at upper floors, especially for short, impulsive ground motions

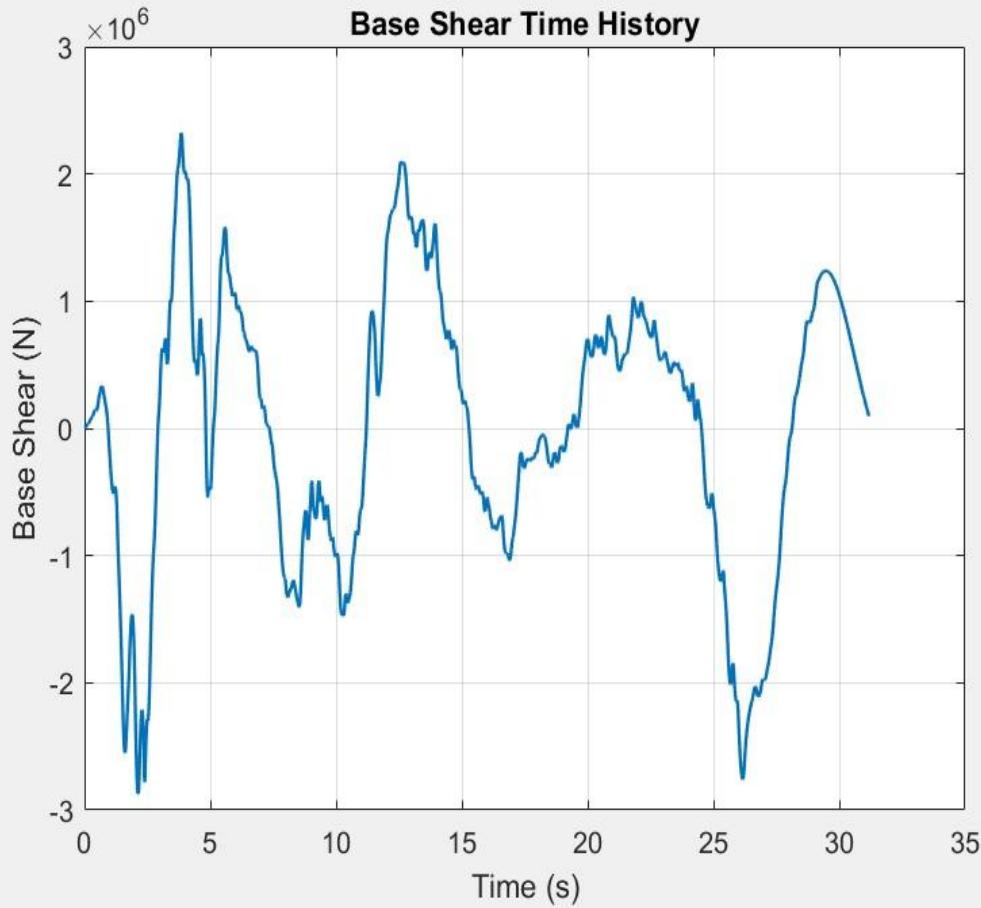
# RESULTS



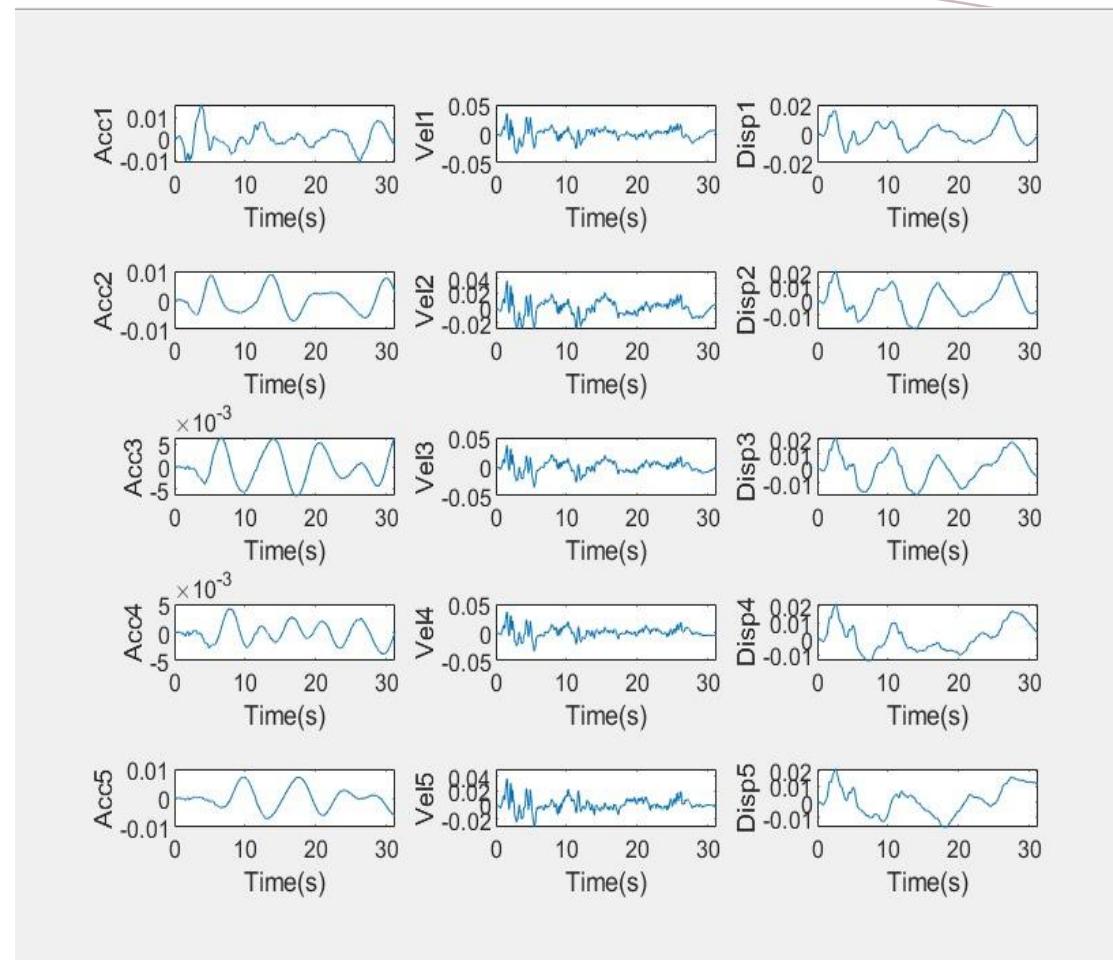
## **Max drift ratios:**

( $\approx 0.34\% - 0.4\%$ ) are well below code limits. For elastic response, most seismic codes (e.g., IS 1893, ASCE 7, Eurocode 8) allow up to 2%-3% drift for Life Safety and Collapse Prevention. Our results indicate an elastic response, with no excessive story drift

# RESULTS



The obtained base shear time-history and its maximum value are in strong agreement with published research and code-predicted results for similar MDOF systems, confirming accurate response capture.



Layered response plots (Acc, Vel, Disp) for each story exhibit typical dynamic behaviour under seismic input, with amplification at upper stories, aligning with benchmarks from academic and software-based studies

# *THANK YOU*

**GROUP MEMBERS:**

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