1 Flow variables and units (SI)

Flow variables and their units:

- 1. Density ρ is kg/m³
- 2. Pressure p is Pa (is N/m² is J/m³)
- 3. Flow velocity v is m/s
- 4. Total energy per unit volume $E = \rho e = \rho \varepsilon_{\rm int} + \rho v^2/2$ is J/m³
- 5. Total specific energy e is J/kg
- 6. Total specific internal energy ε is J/kg
- 7. Mass of the constituent species m is kg

2 Euler equations

For a two-dimensional flow of a single-species inviscid gas the compressible Euler equations governing such a flow are given by

$$\frac{\partial}{\partial t}\mathbf{u} + \frac{\partial}{\partial x}\mathbf{f}_x + \frac{\partial}{\partial y}\mathbf{f}_y = 0. \tag{1}$$

The vector of conservative variables $\mathbf{u} \in \mathbb{R}^4$ is given by

$$\mathbf{u} = (\rho, \rho v_x, \rho v_y, E)^{\mathrm{T}}, \tag{2}$$

The inviscid fluxes are given by

$$\mathbf{f}_x = \left(\rho v_x, \rho v_x^2 + p, \rho v_x v_y, (E+p)v_x\right)^{\mathrm{T}},\tag{3}$$

$$\mathbf{f}_y = \left(\rho v_y, \rho v_x v_y, \rho v_y^2 + p, (E+p)v_y\right)^{\mathrm{T}}.$$
(4)

Instead of E one can also write ρe .

Re-writing via scaled variables and reference quantities, we obtain

$$\frac{\partial}{t_{ref}\partial\hat{t}}\mathbf{u} + \frac{\partial}{L_{ref}\partial\hat{x}}\mathbf{f}_x + \frac{\partial}{L_{ref}\partial\hat{y}}\mathbf{f}_y = 0, \tag{5}$$

$$\mathbf{u} = \rho_{ref} \left(\hat{\rho}, v_{ref} \hat{\rho} \hat{v}_x, v_{ref} \hat{\rho} \hat{v}_y, p_{ref} / \rho_{ref} \hat{\rho} \hat{e} \right), \tag{6}$$

$$\mathbf{f}_x = \left(\rho_{ref} v_{ref} \hat{\rho} \hat{v}_x, p_{ref} \hat{\rho} \hat{v}_x^2 + p_{ref} \hat{p}, p_{ref} \hat{\rho} \hat{v}_x \hat{v}_y, v_{ref} p_{ref} (\hat{\rho} \hat{e} + \hat{p}) \hat{v}_x \right), \tag{7}$$

$$\mathbf{f}_y = \left(\rho_{ref} v_{ref} \hat{\rho} \hat{v}_y, p_{ref} \hat{\rho} \hat{v}_x \hat{v}_y, p_{ref} \hat{\rho} \hat{v}_y^2 + p_{ref} \hat{p}, v_{ref} p_{ref} (\hat{\rho} \hat{e} + \hat{p}) \hat{v}_y \right). \tag{8}$$

Accounting for the fact that $t_{ref} = L_{ref}/v_{ref}$, we can write:

$$\frac{\partial}{\partial \hat{t}}\hat{\mathbf{u}} + \frac{\partial}{\partial \hat{x}}\hat{\mathbf{f}}_x + \frac{\partial}{\partial \hat{y}}\hat{\mathbf{f}}_y = 0, \tag{9}$$

$$\mathbf{u} = (\hat{\rho}, \hat{\rho}\hat{v}_x, \hat{\rho}\hat{v}_y, \hat{\rho}\hat{e},), \tag{10}$$

$$\mathbf{f}_x = (\hat{\rho}\hat{v}_x, \hat{\rho}\hat{v}_x^2 + \hat{p}, \hat{\rho}\hat{v}_x\hat{v}_y, (\hat{\rho}\hat{e} + \hat{p})\hat{v}_x), \tag{11}$$

$$\mathbf{f}_y = (\hat{\rho}\hat{v}_y, \hat{\rho}\hat{v}_x\hat{v}_y, \hat{\rho}\hat{v}_y^2 + \hat{p}, (\hat{\rho}\hat{e} + \hat{p})\hat{v}_y). \tag{12}$$

2.1 Scaling of specific heats

We have (for a single-component flow) $T=T_{ref}\hat{T}=m_{ref}p_{ref}/\rho_{ref}/k\hat{T},~\hat{T}=\hat{p}/\rho_{ref}$ (since at $T=T_{ref}$ and $n=n_{ref}$ the flow density is ρ_{ref} and the pressure is p_{ref}). Scaling of specific heats:

$$c_v(T) = \frac{\partial \varepsilon}{\partial T} = \frac{p_{ref}}{\rho_{ref} T_{ref}} \frac{\partial \hat{\varepsilon}}{\partial \hat{T}} = c_{v,ref} \hat{c}_v.$$
(13)

So $c_{v,ref} = p_{ref}/(\rho_{ref}T_{ref}) = k/m_{ref}$. Mayer's relation in scaled form then reads that $\hat{c}_p = \hat{c}_v + 1$.