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Matlab-Code Demonstrations

Subroutine PlaneTrussElementStiffness.m

```
function y = PlaneTrussElementStiffness(E,A,node_i,node_j)
%-compute element stiffnes matrix-----
L = sqrt((node_j(1) - node_i(1))^2 + (node_j(2) - node_i(2))^2);
if node_j(1) == node_i(1)
    if node_i(2) > node_j(2)
       theta = -pi/2;
   elseif node_i(2) < node_j(2)</pre>
       theta = pi/2;
   end
else
   theta = atan((node_j(2) - node_i(2))/(node_j(1) - node_i(1)));
end
C = cos(theta);
S = sin(theta);
y = E*A/L*[ C*C  C*S  -C*C  -C*S ;
           C*S S*S -C*S -S*S ;
          -C*C -C*S C*C C*S;
          -C*S -S*S C*S S*S];
```

Hauptdatei PlaneTruss.m

```
% MAIN PROGRAM
%-----
% FEM for simple plane struss structures
%-----
clear all
close all
%-read general properties-----
num_ele = input('give the total number of elements. num_ele = ');
num_nodes = input('give the total number of nodes. num_nodes = ');
disp(' ');
%-define nodes, coordinates and stiffnes (E and A) for elements-----
for i = 1:num_ele
   fprintf('element no.%g\n',i);
   connectivity(1,i) = input(' global node for 1st local node = ');
   connectivity(2,i) = input(' global node for 2nd local node = ');
   elementdata(i,:) = input(' element data
                                        [EA] = ');
```



```
disp(' ');
end
for i = 1:num_nodes
   fprintf('node no.%g\n',i);
   X(:,i) = input(' global node coordinates [ x y ] = ')';
   disp(' ');
end
%-compute number of equations-----
disp('give the total number of displacement boundary conditions')
num_bc = input('
                                                  num_bc = ');
num_eq = num_nodes*2 - num_bc;
%-assemble matrix------
assembleid = -ones(2,num_nodes);
        input of global dofs with zero boundary displacements')
                                    for i from 1 to num_bc')
disp('
for i = 1:num_bc
   node = \dots
            global node of constrained degree of freedom = ');
   input('
   dof = \dots
                  degree of freedom to constrained (1=x,2=y) = ');
   input('
   assembleid(dof,node) = 0;
end
idof = 0;
for i = 1:num_nodes
   for j = 1:2
       if assembleid(j,i) ~= 0
           idof = idof + 1;
          assembleid(j,i) = idof;
       end
   end
end
%-compute global stiffnes matrix-----
K = zeros(num_eq,num_eq);
for i = 1:num_ele
   k = PlaneTrussElementStiffness(elementdata(i,1), ...
   elementdata(i,2),X(:,connectivity(1,i)),X(:,connectivity(2,i)));
   I = assembleid(:,connectivity(1,i));
   J = assembleid(:,connectivity(2,i));
   IJ = find(([IJ]) \sim 0);
   K = PlaneTrussAssemble(K,k,I,J,IJ);
end
disp(' ');
%-global force vector------
```



```
R = zeros(num_eq, 1);
disp('give the total number of nodes with non-zero load vectors')
num_loads = ...
         input('
                                                 num_loads = ');
disp('
                 input of load vectors for i from 1 to num_loads')
for i = 1:num_loads
   lbc = input(' no. of global node with non-zero load vector = ');
   if assembleid(2*lbc-1) ~= 0 & assembleid(2*lbc) ~= 0
       R(assembleid(2*lbc-1)) = ...
       input('
                               force in global x-direction = ');
       R(assembleid(2*lbc)) = ...
       input('
                               force in global y-direction = ');
   elseif assembleid(2*lbc-1) ~= 0
       R(assembleid(2*lbc-1)) = ...
       input('
                              force in global x-direction = ');
   elseif assembleid(2*lbc) ~= 0
       R(assembleid(2*lbc)) = ...
       input('
                              force in global y-direction = ');
   elseif assembleid(2*lbc-1) == 0 & assembleid(2*lbc) == 0
       disp('all degrees of freedom in this node are constrained!');
       pause(1);
   end
end
%-displacement computed by Gaussian elimination-----
u = K \setminus R;
%-print the data-----
disp(' ');
disp('element connectivity table');
disp(' ');
disp(' ======;')
disp(' | e | Node i | Node j |')
disp(' =======,')
for i = 1:num_ele
                                    %g ∣', ...
   fprintf(' | %g |
                         %g
                            |
   i,connectivity(1,i),connectivity(2,i))
   disp(' ');
end
disp(' =======,')
disp(' ');disp('global stiffness matrix'); K
disp(' ');disp('global force vector '); R
disp(' ');disp('displacement vector
                                  '); u
```



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Subroutine PlaneTrussAssemble.m

```
function y = PlaneTrussAssemble(K,k,I,J,IJ)
%-----
%Input: element stiffnes matrix
%Output: updated global stiffnes matrix
%-----
                             _____
if size(find([ I' J'] ~= 0),2)
                          == 1
   K = Assemble1(K,k,I,J,IJ);
elseif size(find([ I' J'] \sim = 0),2) == 2
   K = Assemble2(K,k,I,J,IJ);
elseif size(find([ I' J'] = 0),2) == 3
   K = Assemble3(K,k,I,J,IJ);
elseif size(find([ I' J' ] ~= 0),2) == 4
   K = Assemble4(K,k,I,J,IJ);
end
y = K;
Assemble1.m
function y = Assemble1(K,k,I,J,IJ)
%-----
% Assembly routine for element with one active dof
i = find([ I' J'] ~= 0);
switch i
   case 1, a = I(1);
   case 2, a = I(2);
   case 3, a = J(1);
   case 4, a = J(2);
end
K(a,a) = K(a,a) + k(IJ,IJ);
y = K;
Assemble2.m
function y = Assemble2(K,k,I,J,IJ)
%-----
% Assembly routine for element with two active dofs
if size(find(I = 0),1) == 2
   a = I(1);
```



```
b = I(2);
elseif size(find(J = 0),1) == 2
    a = J(1);
    b = J(2);
elseif (size(find(I \sim= 0),1) == 1) & (size(find(J \sim= 0),1) == 1)
    a = I(find(I \sim 0));
    b = J(find(J = 0));
end
i = IJ(1);
j = IJ(2);
K(a,a) = K(a,a) + k(i,i);
K(a,b) = K(a,b) + k(i,j);
K(b,a) = K(b,a) + k(j,i);
K(b,b) = K(b,b) + k(j,j);
y = K;
Assemble3.m
function y = Assemble3(K,k,I,J,IJ)
% Assembly routine for element with three active dofs
if size(find(I = 0),1) == 1
    a = I(find(I \sim 0));
    b = J(1);
    c = J(2);
elseif size(find(J = 0),1) == 1
    a = I(1);
    b = I(2);
    c = J(find(J = 0));
end
K(a,a) = K(a,a) + k(IJ(1),IJ(1));
K(a,b) = K(a,b) + k(IJ(1),IJ(2));
K(a,c) = K(a,c) + k(IJ(1),IJ(3));
K(b,a) = K(b,a) + k(IJ(2),IJ(1));
K(b,b) = K(b,b) + k(IJ(2),IJ(2));
K(b,c) = K(b,c) + k(IJ(2),IJ(3));
K(c,a) = K(c,a) + k(IJ(3),IJ(1));
K(c,b) = K(c,b) + k(IJ(3),IJ(2));
K(c,c) = K(c,c) + k(IJ(3),IJ(3));
y = K;
```



Assemble4.m

```
function y = Assemble4(K,k,I,J,IJ)
%-----
% Assembly routine for element with four active dofs
a = I(1);
b = I(2);
c = J(1);
d = J(2);
K(a,a) = K(a,a) + k(1,1);
K(a,b) = K(a,b) + k(1,2);
K(a,c) = K(a,c) + k(1,3);
K(a,d) = K(a,d) + k(1,4);
K(b,a) = K(b,a) + k(2,1);
K(b,b) = K(b,b) + k(2,2);
K(b,c) = K(b,c) + k(2,3);
K(b,d) = K(b,d) + k(2,4);
K(c,a) = K(c,a) + k(3,1);
K(c,b) = K(c,b) + k(3,2);
K(c,c) = K(c,c) + k(3,3);
K(c,d) = K(c,d) + k(3,4);
K(d,a) = K(d,a) + k(4,1);
K(d,b) = K(d,b) + k(4,2);
K(d,c) = K(d,c) + k(4,3);
K(d,d) = K(d,d) + k(4,4);
y = K;
```