A hundred times every day I remind myself that my inner and outer life are based on the labors of other men, living and dead, and that I must exert myself in order to give in the same measure as I have received and am still receiving...



Kumaraguru Sivasankaran

MS in Aeronautics and Astronautics Engineering Candidate for Controls Engineer

 $\mathsf{K} \mathsf{I} \mathsf{T} \mathsf{T} \mathsf{Y} \mathsf{H} \mathsf{\Lambda} \mathsf{W} \mathsf{K}$









Journey so far..



Grounding work

Modern Control theory

- Linear systems
- Non linear Systems
- Distributed Network Control
- Optimal Control and Estimation
- Linear Algebra

Computational Science

- Optimization
- Computer Vision
- Parallel Computing
- Machine learning
- Deep learning

Applied Statistics

- Statistical methods
- Probability
- Design of Experiments
- Regression Analysis
- Computational Statistics I

Control design:

PID controller

LQR controller

Model predictive Controller

Sliding mode controllers

High Gain observers

Hybrid system control

Trajectory Optimization:

Dynamic Programming

Calculus of Variations

Sensor fusion / Estimation:

Kalman filtering, EKF, UKF

Least Squares Minimization

Levenberg Marquardt optimization

Sampling methods

Monte Carlo simulations

3D reconstruction from 2D images

Object detection and localization

Camera calibration

System modeling:

Physics based modeling

Markov Decision Process

Partially Observable Markov Decision Process

Deep Neural Networks

Gaussian Process models

My mentors @Purdue



With Dr. Martin Corless, Professor, Dynamics and Controls, Aeronautics and Astronautics Engineering Work: Consensus algorithms



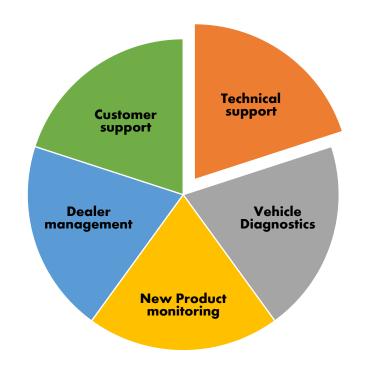
With Dr. Xiao Wang, Professor, Dept. of Statistics Work: Reinforcement learning



With Dr. Jan E Mansson, Distinguished Professor of Materials and Chemical Engineering and AAE (by Courtesy)
Work: Technical Cost Modeling

Life at Daimler (2014-16)

- Experienced with the lifecycle of a truck from production to full operation and finally to resale / scrap.
- Familiar with Maintenance cycle, Product failures,
 Failure analysis and Total Cost of Ownership calculations (TCO).
- Offered technical support when a trained technician was unable to troubleshoot a vehicle off road.
- Ensured customer satisfaction in terms of overall Aftersales service support to West Gujarat, India
- Collaborated with people at different levels
 - Technician, Dealer manager, Dealer Principal, Customer
 - Regional manager, Technical services, Warranty, PMG, VP
- Led field trials for establishing performance and responsible for New Product Monitoring
- Appreciated for quick learning, taking up responsibility and delivering with consistently high KPI in the region



Aftersales Service Manager





Coop at Volvo (2017)

- Verification and Validation of control systems for Production Vehicle Evaluation (PVE)
- Over 500 miles of road testing per week
- Involved Data collection, Analysis, Failure reporting and documentation
- More than 50 tests performed in a span of 4 months (Aug- Dec)
- Familiar with use of functional documentation for evaluation and troubleshooting



Hobbies







/ Blitz v







And more..

Decentralized controller for Multiagent systems – A motivation



Safety barrier Certificate: A review

Non-linear system: $\dot{x} = f(x) + g(x)u$

Control invariance set: $C = \{x \in R^n : h(x) \ge 0\}$

 $\partial C = \{ x \in R^n : h(x) = 0 \}$

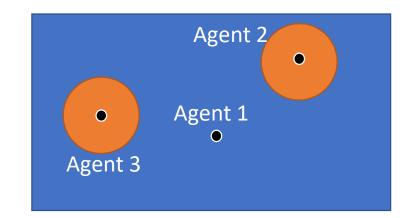
 $Int (C) = \{ x \in R^n : h(x) > 0 \}$

Barrier function: $\inf_{x \in Int(C)} B(x) \ge 0 \quad \lim_{x \to \partial C} B(x) = \infty$

Barrier function dynamics: $\dot{B} \leq \frac{\gamma}{B}$ Stronger condition: $\dot{B} \leq 0$

Control Barrier function: $\inf_{u \in U} \left[L_f B(x) + L_g B(x) u - \frac{\gamma}{B(x)} \right] \le 0$

$$K_{cbf}(x) = \{ u \in U : L_f B(x) + L_g B(x) u - \frac{\gamma}{B(x)} \le 0 \}$$



Review (contd.)

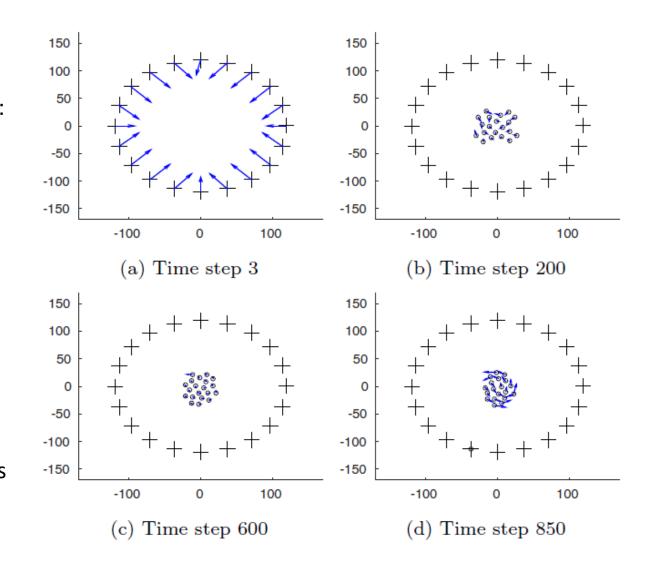
Solution: controller based on Quadratic programming:

$$u^* = argmin_u J(u) = \sum_{i} ||u_i - \hat{u}_i||^2$$
s.t. $A_{ij}u \le b_{ij}$, $\forall i \ne j$

$$||u_i||_{\infty} \le a_{max}$$
, $\forall i \in M$

Problems with this approach:

- Sluggish overall
- Conservative behavior in decentralized case
- Deadlocks due to infeasible optimization solutions



Reference: Borrmann, U., Wang, L., Ames, A. D. & Egerstedt, M. (2015). Control Barrier Certificates for Safe Swarm Behavior.. In M. Egerstedt & Y. Wardi (eds.), ADHS (p./pp. 68-73), : Elsevier

Problem setup and approach

Double integrator dynamics:

$$\begin{bmatrix} \dot{p}_i \\ \dot{v}_i \end{bmatrix} = \begin{bmatrix} 0 & I_{2*2} \\ 0 & 0 \end{bmatrix} \begin{bmatrix} p_i \\ v_i \end{bmatrix} + \begin{bmatrix} 0 \\ I_{2*2} \end{bmatrix} u_i$$

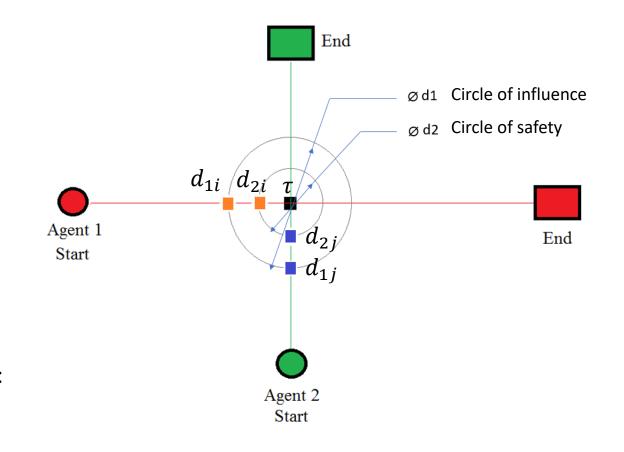
 $p_i \in R^2$, $v_i \in R^2$, $u_i \in R^2$ are position, velocity and acceleration of agent i

Reference point (final):

$$r_i = \begin{bmatrix} r_{ix} \\ r_{iy} \end{bmatrix}$$

Nominal controller to achieve end goal with zero velocity:

$$u_i = -k_1(p_i - r_i) - k_2 v_i$$



Reference: Mou, S., Cao, M., & Morse, A. S. (2015). Target-point formation control. Automatica, 61, 113-118.

Target point control

Position vectors:

$$A_{i} = \begin{bmatrix} p_{xi} \\ p_{yi} \end{bmatrix} + \mu \begin{bmatrix} p_{xi} - r_{xi} \\ p_{yi} - r_{yi} \end{bmatrix}$$

$$A_{i} = \begin{bmatrix} p_{xi} \\ p_{yi} \end{bmatrix} + \mu \begin{bmatrix} p_{xi} - r_{xi} \\ p_{yi} - r_{yi} \end{bmatrix} \qquad A_{j} = \begin{bmatrix} p_{xj} \\ p_{yj} \end{bmatrix} + t \begin{bmatrix} p_{xj} - r_{xj} \\ p_{yj} - r_{yj} \end{bmatrix}$$

At point of intersection:

$$A_i = A_j$$

Solve for intersection point:

$$\begin{bmatrix} \mu \\ t \end{bmatrix} = \begin{bmatrix} p_{xi} - r_{xi} & r_{xj} - p_{xj} \\ p_{yi} - r_{yi} & r_{yj} - p_{yj} \end{bmatrix}^{-1} \begin{bmatrix} p_{xj} - p_{xi} \\ p_{yj} - p_{yi} \end{bmatrix}$$

$$\tau = \begin{bmatrix} p_{xi} \\ p_{yi} \end{bmatrix} + \mu^* \begin{bmatrix} p_{xi} - r_{xi} \\ p_{yi} - r_{yi} \end{bmatrix} = \begin{bmatrix} p_{xj} \\ p_{yj} \end{bmatrix} + t^* \begin{bmatrix} p_{xj} - r_{xj} \\ p_{yj} - r_{yj} \end{bmatrix}$$

Target Points:

$$d_{1i} = p_i - \frac{||\tau - p_i|| - d_1}{||\tau - p_i||} (\tau - p_i)$$

$$d_{1i} = p_i - \frac{||\tau - p_i|| - d_1}{||\tau - p_i||} (\tau - p_i) \qquad d_{2i} = p_i - \frac{||\tau - p_i|| - d_2}{||\tau - p_i||} (\tau - p_i)$$

$$d_{1j} = p_j - \frac{||\tau - p_j|| - d_1}{||\tau - p_j||} (\tau - p_j) \qquad d_{2j} = p_j - \frac{||\tau - p_j|| - d_2}{||\tau - p_j||} (\tau - p_j)$$

Hybrid distributed control design

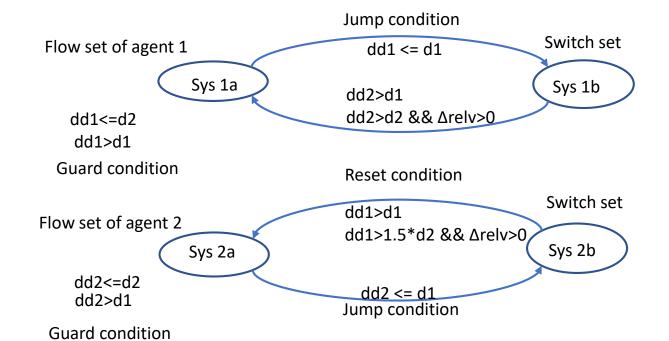
Flow set: Continuous ODE dynamic system where agent spends most time

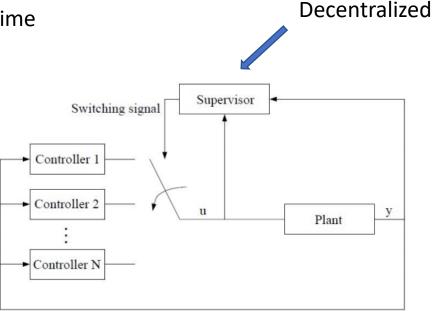
Switch set: Alternate system dynamics

Guard condition: Keeps system in flow set

Jump condition: Discrete transitions from flow set to switch set

Reset condition: Returns to flow set





Nominal

$$u_i = -k_1(p_i - r_i) - k_2 v_i$$

Switched

$$u_i = -k_1(p_i - d_{2i}) - k_2 v_i$$

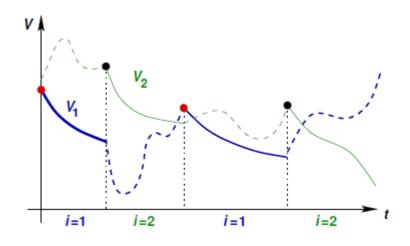
Analysis and Results

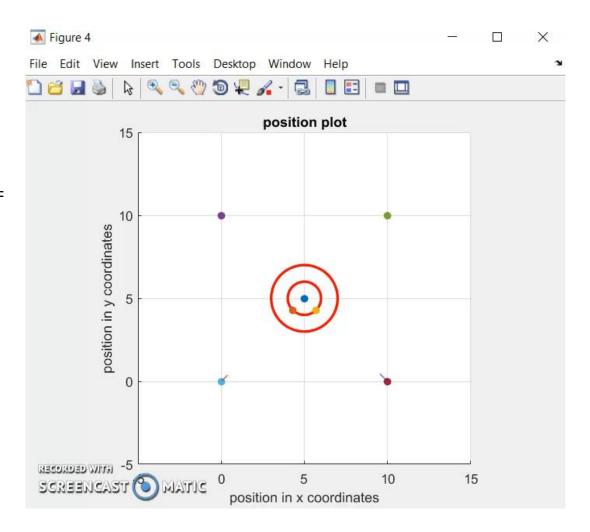
Theorem:

Consider switched system with all sub models $\dot{x} = f_i(x)$ as Globally asymptotically stable with corresponding Lyapunov function V_i .

Suppose for every pair of switching times (t_k, t_l) , t < l with $\sigma(t_k) = \sigma(t_l) = i$ and $\sigma(t_m) \neq i$ for $t_k < t_m < t_l$, we have

 $V_iig(x(t_l)ig) - V_iig(x(t_k)ig) \le hoig(ig||x(t_k)|ig|ig) < 0$, then the switched system is GAS

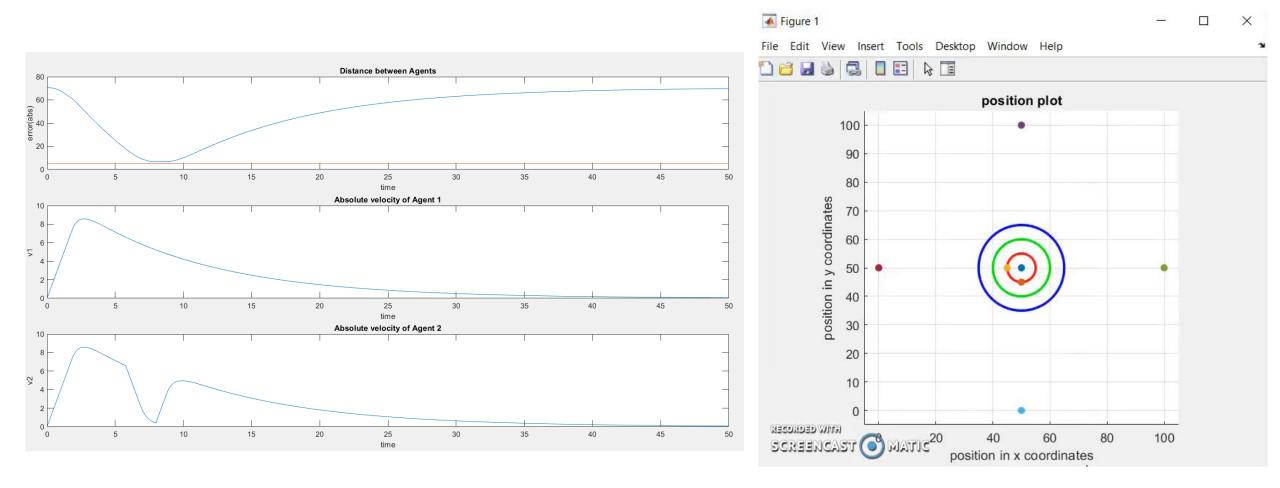




Agents having different max speed

Reference: Branicky, M. S. (1998). Multiple Lyapunov functions and other analysis tools for switched and hybrid systems. IEEE Transactions on automatic control, 43(4), 475-482.

Results:



Agents having same max speed

Advantages

- Combination of two ideas target point control and control barrier certificates (switching).
- Simpler than Quadratic programming controller.
- Eliminates unwanted deviation from path.
- Fast response and less use of computation.

Disadvantages:

- Assumes that the path travelled by both agents are always in straight lines.
- Requires additional control algorithm for scaling to multiple agents (e.g. consensus based ranking for resolving deadlocks)
- Symmetric cases are not handled.

High-gain observers

Non-linear system
$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = \phi(x, u, w)$$

Measurement
$$y = x_1$$

$$u = \gamma(x, w)$$

To stabilize the origin x=0

Observer
$$\dot{\hat{x}}_1 = \hat{x}_2 + h_1(y - \hat{x}_1)$$

$$\dot{\hat{x}}_2 = \phi_0(\hat{x}, u) + h_2(y - \hat{x}_1)$$

$$\dot{\tilde{x}}_1 = -h_1 \tilde{x}_1 + \tilde{x}_2$$

$$\dot{\tilde{x}}_2 = -h_2 \tilde{x}_1 + \delta(x, \tilde{x}, w)$$

$$\widetilde{x}_1 = x_1 - \widehat{x}_1
\widetilde{x}_2 = x_2 - \widehat{x}_2
\delta = \phi(x, \gamma(\widehat{x}), w) - \phi_0(\widehat{x}, \gamma(\widehat{x}))$$

Reference: Khalil, H. K., & Praly, L. (2014). High-gain observers in nonlinear feedback control. International Journal of Robust and Nonlinear Control. Lecture series: Forum on Robotics & Control Engineering http://force.eng.usf.edu/

Contd.

Change of variables

$$h_1 = \frac{\alpha_1}{\epsilon} \qquad h_2 = \frac{\alpha_2}{\epsilon^2}$$

$$\eta_1 = \frac{\tilde{x}_1}{\epsilon} \qquad \eta_2 = \tilde{x}_2$$

New Dynamics

$$\epsilon \, \dot{\eta}_1 = -\alpha_1 \eta_1 + \eta_2$$

$$\epsilon \, \dot{\eta}_2 = -\alpha_2 \eta_1 + \epsilon \, \delta$$

Peaking phenomenon

$$\eta_1(0) = \frac{\tilde{x}_1(0)}{\epsilon} = \frac{x_1(0) - \hat{x}_1(0)}{\epsilon}$$



$$\eta_1(0) = O\left(\frac{1}{\epsilon}\right)$$

Transient response could be impulsive in short time and could destabilize the system

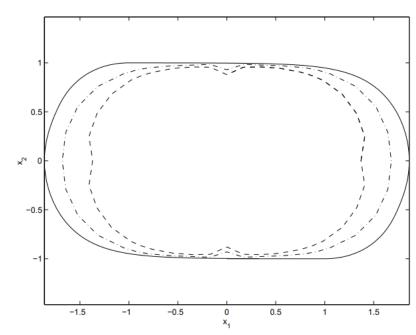
Solution: Control saturation

Advantages

- Asymptotic stability of origin as ϵ tends to zero
- Recovers region of attraction
- Recovers state trajectories

Disadvantages

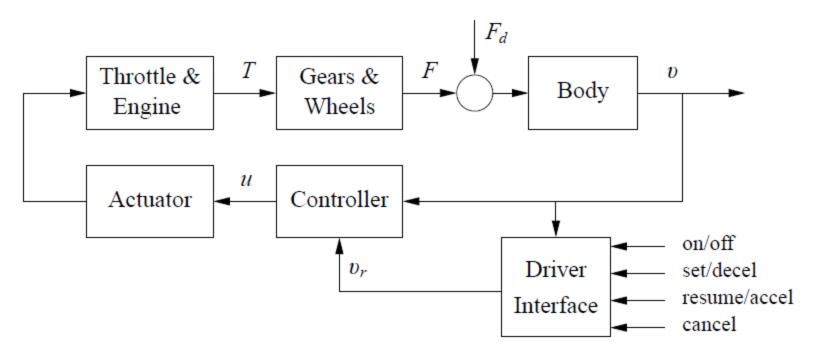
- High gain increases error due to noisy measurements
- Faster response at transient but poor steady-state performance
- No guarantee of asymptotic stability to origin. Converges within an error of ϵ



Region of attraction under state feedback and output feedback for different values of ϵ

Solution: Non-linear gain scheduling – high gains during transient phase and lower gain at steady state Alternate: Sliding mode controller

Extended High Gain observers for output feedback control of Non-linear systems





Motivating example: Adaptive Cruise control

System dynamics

Newton's law:

$$m\frac{dv}{dt} = F_{engine} - F_{drag}$$

Engine rpm:

$$\omega = -\frac{n}{r} \nu =: \alpha_n \nu$$

r = wheel radius, n = gear ratio, v = linear velocity

 $\alpha_1 = 40$ $\alpha_2 = 25$ $\alpha_3 = 16$

Typical values

Force due to engine:

$$F_{engine} = \frac{nu}{r} T(\omega) = \alpha_n u T(\alpha_n v)$$

u = Control input (rate of fuel injection)

$$\alpha_4 = 12$$

$$\alpha_5 = 10$$

$$T(\omega) = T_m (1 - \beta \left(\frac{\omega}{\omega_m} - 1\right)^2)$$

$$T_m = Max \ torque = 190 \ Nm$$

 $\omega_m = Max \ Engine \ rpm = 420 \ rads^{-1}$
 $\beta = 0.4$

Force due to friction:

$$F_r = mgC_r \, sign(v)$$

$$m = mass \ of \ car, g = gravity,$$

 $C_r = 0.01 = Coefficient \ of \ Rolling \ friction$

Aerodynamic drag:

$$F_a = \frac{1}{2}\rho C_d A v^2$$

$$\rho = Density \ of \ air = 1.3 \frac{kg}{m^3} \ , C_d = Drag \ coefficient = 0.32$$

$$A = Frontal \ area \ of \ car = 2.4m^2$$

Reference: Feedback Systems, K. J. Astrom & R. Murray

Contd.

Weight

$$F_w = mg\sin(\theta)$$

 $\theta = slope \ of \ road$

Total forces:

component:

$$m\frac{dv}{dt} = \alpha_n u T(\alpha_n v) - mgC_r \operatorname{sign}(v) - \frac{1}{2}\rho C_d A v^2 - mg \sin(\theta)$$

PID Controller:
$$u(t) = k_p e(t) + k_i \int_0^t e(\tau) d\tau + k_d \dot{e}(t)$$

Perfect state information available



State feedback controller works

Unmodelled dynamics/ non-linearities always exists. Finding perfect model is a challenging task than designing controller

Extended High-Gain Observer

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = a(x, w) + b(x, w)u$$

Measurement

$$y = x_1$$

Extended Dynamics

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = a_0(x) + b_0(x)u + \sigma$$

where
$$\sigma = a(x, w) - a_0(x) + (b(x, w) - b_0(x))u$$

$$\dot{\sigma} = \phi(x, w, \dot{w}, u, \dot{u})$$

$$y = x_1$$

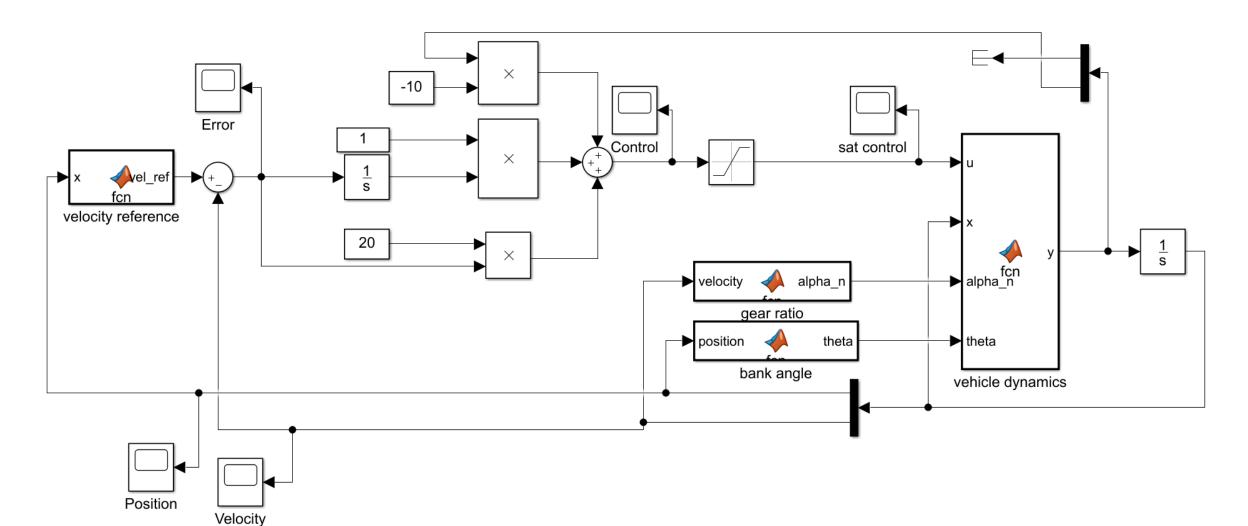
High-gain observer

$$\dot{\hat{x}}_1 = \hat{x}_2 + \frac{\alpha_1}{\epsilon} (y - \hat{x}_1)$$

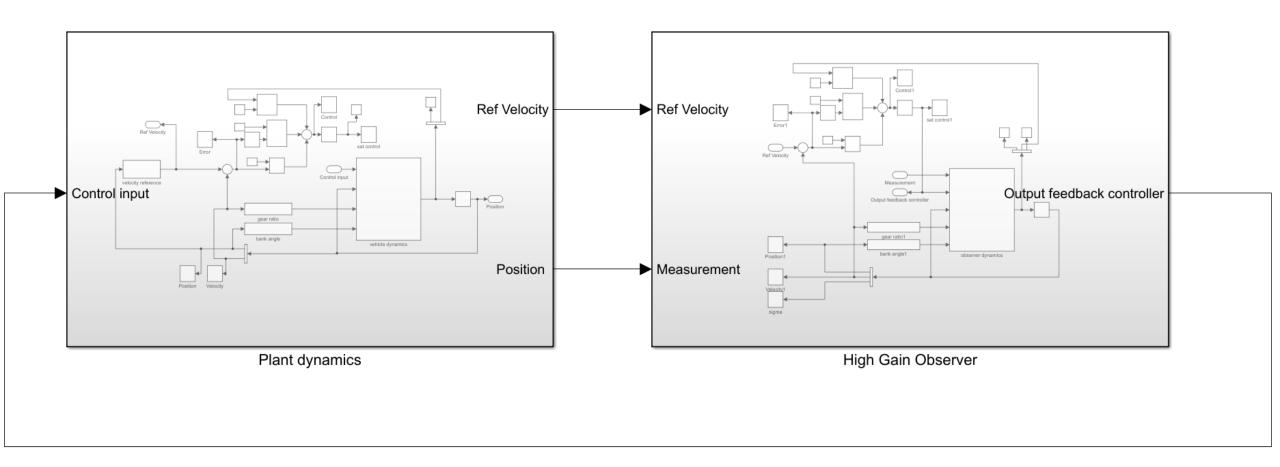
$$\dot{\hat{x}}_2 = a_0(\hat{x}) + b_0(\hat{x})u + \hat{\sigma} + \frac{\alpha_2}{\epsilon^2}(y - \hat{x}_1)$$

$$\dot{\hat{\sigma}} = \frac{\alpha_3}{\epsilon^3} (y - \hat{x}_1)$$

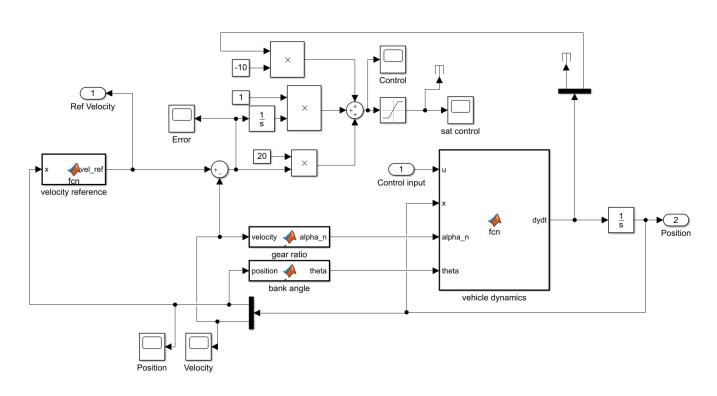
State feedback controller

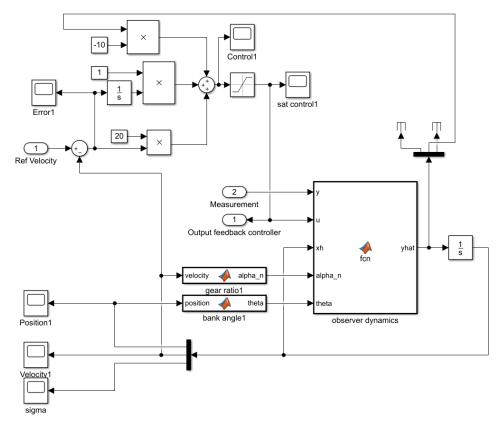


Output feedback controller



Simulation

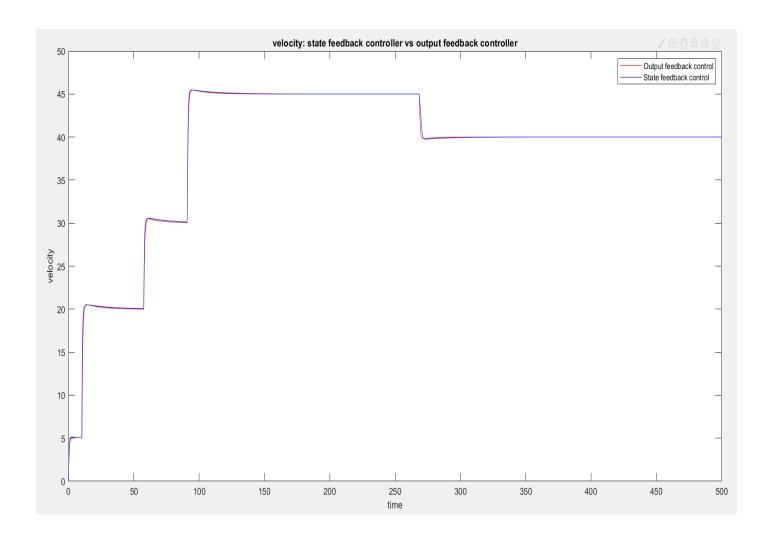




Plant dynamics

Observer dynamics

Results



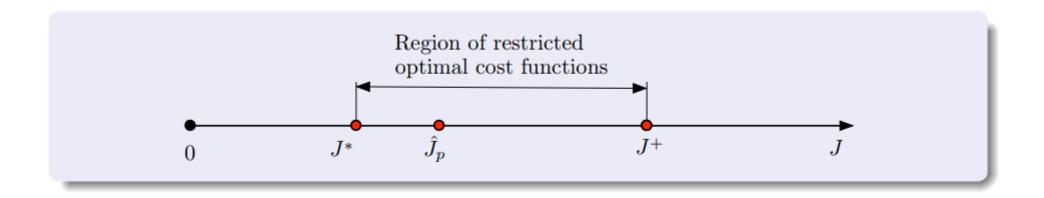
Parameters in Observer dynamics are modified to induce modeling inaccuracy.

For example: mass of the car was varied +/- 30%, frictional force set to zero, drag coefficient varied.

Yet, output feedback controller (using high gain observers) perform* at par with state feedback controller as seen here.

^{*}Requires tuning gain, epsilon for higher modeling errors

Thank you



Stability