

Procedure:

- 1) Open MATLAB Command window.
- 2) Click on file/new/m file to open the MATLAB editor window.
In MATLAB editor window enter the program.
- 3) Save the program as .m file.
- 4) Execute the program by selecting run.
- 5) Obtain the specifications from the plot.

Theoretical Calculations:

$$T.F = \frac{100}{s^2 + 10s + 100}$$

$$\omega_n \approx 100$$

$$(\omega_n = 10)$$

$$2\zeta\omega_n = 10$$

$$(\zeta = 0.5)$$

$$\text{Resonance frequency } \omega_r = \omega_n \sqrt{1 - 2\zeta^2} = 10 \sqrt{1 - 2 \times 0.5^2} = 7.07106$$

$$M_R = \frac{1}{2\zeta \sqrt{1 - \zeta^2}} = \frac{1}{2 \times 0.5 \sqrt{1 - 0.5^2}} = 1.1547$$

$$\omega_b = \omega_n [1 - 2\zeta^2 + \sqrt{2 - 4\zeta^2 + 4\zeta^4}]^{1/2}$$

$$= 10 [1 - 2(0.5)^2 + \sqrt{2 - 4(0.5)^2 + 4 \times (0.5)^4}]^{1/2} = 12.7202$$

$$M_{LTF} = \frac{CLTF}{1 - CLTF} = \frac{100/s^2 + 10s + 100}{1 - \frac{100}{s^2 + 10s + 100}} = \frac{100}{s^2 + 10s} = \frac{100}{s(s+10)}$$

$$TF = \frac{100}{s(s+10)} = \frac{100}{j\omega(j\omega+10)}$$

$$= \frac{10 \angle 0^\circ}{\omega \angle 90^\circ (\sqrt{100 + \omega^2}) \angle \tan^{-1}(\frac{\omega}{10})}$$

$$= 0 - (90^\circ + \tan^{-1}(\frac{\omega}{10})) = -180^\circ$$

$$\tan^{-1}(\frac{\omega}{10}) = 90^\circ$$

gain cross over frequency

$$\frac{100}{\omega(\sqrt{100 + \omega^2})} = 1$$

$$\omega \sqrt{100 + \omega^2} = 100$$

$$\omega^2(100 + \omega^2) = 100 \times 100$$

$$\omega^4 + 100\omega^2 - 10^4 = 0$$

$$\omega^2 = t$$

$$t^2 + 100t - 10^4 = 0$$

$$t = -161.80 \times, t = 61.80$$

$$\omega_{gc} = \sqrt{61.80} = 7.861$$

$$PM (\text{Phase Margin}) = 180 + (-90 - \tan^{-1}(\frac{7.861}{10}))$$

$$\tan^{-1}(\frac{7.861}{10}) = 38.17^\circ$$

$$PM = 51.829.$$

Phase cross over frequency

$$0 + 90 + \tan^{-1} \left(\frac{\omega_{pc}}{10} \right) = +180$$

$$\tan^{-1} \left(\frac{\omega_{pc}}{10} \right) = 90$$

$$\boxed{\omega_{pc} = \infty}$$

$$|G(j\omega_{pc})| = \frac{100}{\omega \sqrt{100 + \omega^2}} = \frac{100}{\omega \sqrt{100 + \infty^2}}$$

$$GM = \frac{1}{|G(j\omega_{pc})|} = \frac{1}{0} = \infty$$

for $\xi = 1$

$$M_r = 1, \omega_r = 0, \omega_b = 6.4229$$

for $\xi = 0$

$$M_r = \infty, \omega_r = 10, \omega_b = 15.5323.$$

$1 < \xi < \infty \rightarrow$ over damped system.

$\xi = 1 \rightarrow$ critically damped.

$0 < \xi < 1 \rightarrow$ under damped

$\xi = 0 \rightarrow$ undamped system.

Time domain specifications

$$\text{Delay time } T_d = (1 + 0.7\xi) / \omega_n$$

$$\text{Rise time } T_r = (\pi - \theta) / \omega_d$$

$$\text{Peak overshoot time } = T_p = \pi / \omega_d$$

$$\% M_p = \exp(-\pi\xi / \sqrt{1-\xi^2})$$

$$\text{Settling time } T_s = 4 / \xi \omega_n \text{ (2\% tolerance).}$$

Theoretical calculation:

$$\omega_n = 30; \xi = 0.5 \Rightarrow \omega_d = \omega_n \sqrt{1-\xi^2} = 25.98 \text{ rad/sec}$$

$$t_d = \frac{1+0.7\xi}{\omega_n} = 0.045 \text{ sec}$$

$$t_r = \frac{\pi - \theta}{\omega_d} = \frac{\pi - \cos^{-1}(\xi)}{25.98} = \frac{\pi - \cos^{-1}(0.5)}{25.98} = 0.0806 \text{ sec}$$

$$t_p = \frac{\pi}{\omega_d} = \frac{\pi}{25.98} = 0.12 \text{ sec}$$

$$\% M_p = e^{-\frac{\pi\xi}{\sqrt{1-\xi^2}}} \times 100 = e^{-\frac{\pi(0.5)}{\sqrt{1-(0.5)^2}}} \times 100 = 16.30\%$$

$$\text{Settling time } (t_s) = \frac{4}{\xi \omega_n} = \frac{4}{(0.5)30} = 0.266 \text{ sec}$$

2) +ve feedback system:

Step into: Rise time: 5.8584, settling time: 10.6547, overshoot: 0
Peak: 0.9999, Peak time: 25.9983, steady state value = 1

→ The response took more than the previous system to settle down. So less stable. But steady state value = 1 (increased). So +ve feedback (unity) makes system less stable and increases the final value w.r.t DC gain.

3) Closed loop response $G = \frac{1}{(s+1)(s+2)}$, $H = \frac{1}{s+1}$ → pole at -1

+ve feedback:

$$CLTF = \frac{G}{1+GH} = \frac{1/(s+1)(s+2)}{1 + 1/(s+1)(s+2)} = \frac{s+1}{s^3 + 4s^2 + 5s + 3}$$

$$\text{So } s = -2.47, -0.767 \pm 0.793i$$

→ Based on location of poles we can say that the system is stable and as poles have imaginary component the response is oscillatory too.
(has overshoot) (i.e. $\xi = \cos \theta < 1$)

Step into:

Rise time: 1.2722

Settling time: 5.0078

Overshoot: 11.8450

Peaks: 0.3728

Peak Time: 2.2764

From response:

→ Due to addition of pole in feedback path the rise time got improved compared to unity feedback, But system became less stable (taking more settling time), became oscillatory.

3) +ve feedback:

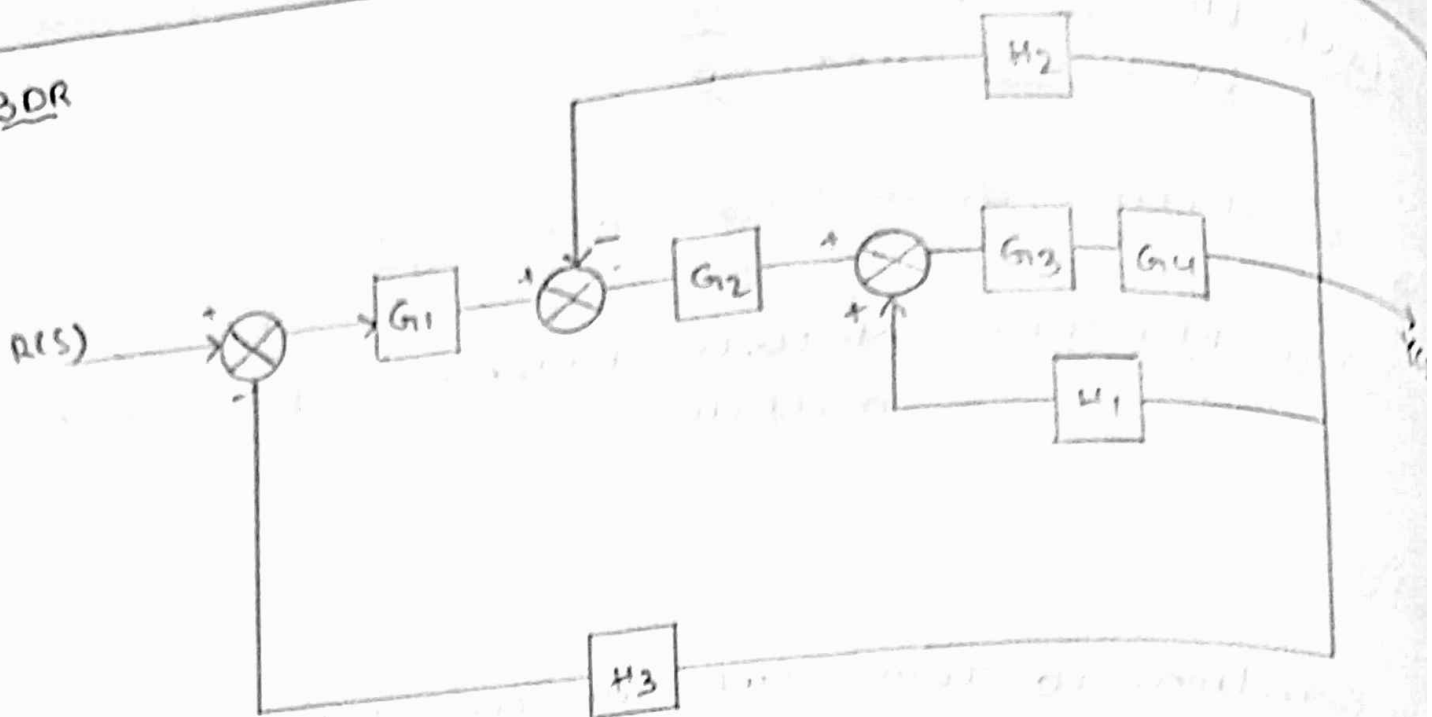
$$CLTF = \frac{G}{1-GH} = \frac{1/(s+1)(s+2)}{1 - \frac{1}{(s+1)(s+2)(s+1)}} = \frac{1}{s^3 + 4s^2 + 5s + 1}$$

Pole location:

$$s = -1.98 \pm j0.745, -0.245$$

→ Due to pole near to jw axis the system stability is reduced (relatively).

BDR



$$G_1(s) = G_2(s) = G_3(s) = G_4(s) = 1/s$$

$$H_1(s) = H_2(s) = H_3(s) = 1$$

Theoretical calculations:

Step 1: $\frac{1}{s} \cdot \frac{1}{s} = \frac{1}{s^2}$

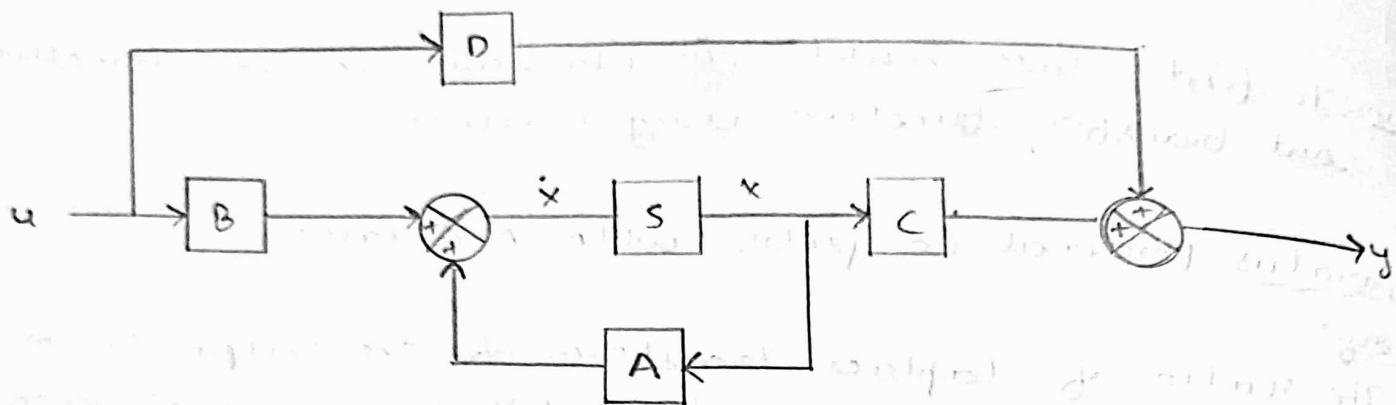
Step 2: $\frac{1/s^2}{1 - 1/s^2} = \frac{1}{s^2 - 1}$

Step 3: $\left(\frac{1}{s^2 - 1}\right) \left(\frac{1}{s}\right) = \frac{1}{s^3 - s}$

Step 4: $\frac{\frac{1}{s(s^2 - 1)}}{1 + \frac{1}{s(s^2 - 1)}} = \frac{1}{s^3 - s + 1}$

Step 5: $\left(\frac{1}{s^3 - s + 1}\right) \left(\frac{1}{s}\right) = \frac{1}{s^4 - s^2 + s}$

Step 6: $\frac{\frac{1}{s^4 - s^2 + s}}{1 + \frac{1}{s^4 - s^2 + s}} = \frac{1}{s^4 - s^2 + s + 1}$



where x = State Vector

\dot{x} = differential State Vector

U = input Vector

Y = Output Vector

$[A]_{n \times n}$ = System Matrix

$[B]_{n \times m}$ = Input Matrix

$[C]_{p \times n}$ = Output Matrix

$[D]_{p \times m}$ = feed forward matrix

(n = order of the System)

m = inputs to the System

p = Outputs to the System

Program 1:-

$$a = [0 \ 2; \ 1 \ -1]$$

$$b = [0; \ 1]$$

$$c = [1 \ 0]$$

$$d = [0]$$

$$[num, den] = ss2tf(a, b, c, d, 1);$$

$$Transfer - Function = tf(num, den);$$

Simulation Result:

$$T.F = \frac{2}{(s+2)(s-1)}$$

Theoretical Calculation:

$$YTF = C [sI - A]^{-1} B + D$$

$$= C [sI - A]^{-1} B + [0]$$

$$= [1 \ 0] \begin{bmatrix} s & -2 \\ -1 & s+1 \end{bmatrix} \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$= \frac{1}{(s+2)(s-1)} [1 \ 0] \begin{bmatrix} s+1 & 2 \\ 1 & s \end{bmatrix} \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$= \frac{1}{(s+2)(s-1)} [s+1 \ 2] \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$T.F = \frac{2}{(s+2)(s-1)}$$

Program 2:-

$$n = [0 \ 2]$$

$$d = [1 \ 1 \ -2]$$

$$H = tf(n, d);$$

$$[a, b, c, d] = tf2ss(n, d);$$

Simulation Result:

$$a = \begin{bmatrix} -1 & 2 \\ 1 & 0 \end{bmatrix}$$

$$b = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

$$c = [0 \ 2]$$

$$d = [0]$$

$$T.F = \frac{2}{s^2 + s - 2}$$

$$2) \frac{Y(s)}{U(s)} = \frac{2}{s^2 + s - 2}$$

$$(s^2 + s - 2) Y(s) = 2 U(s)$$

$$\frac{d^2 y(t)}{dt^2} + \frac{dy(t)}{dt} - 2y(t) = 2u(t)$$

$$\text{Let } y(t) = x_1; \quad u(t) = 4$$

$$\frac{dy(t)}{dt} = x_1 = x_2$$

$$x_2 + x_2 - 2x_1 = 2u$$

$$x_2 = 2x_1 - x_2 + 2u$$

Comparing with

$$\dot{x} = Ax + Bu$$

$$Y = Cx + Du$$

$$A = \begin{bmatrix} 0 & 1 \\ 2 & -1 \end{bmatrix}; \quad B = \begin{bmatrix} 0 \\ 2 \end{bmatrix}$$

$$C = [1 \ 0]; \quad D = [0]$$

Theoretical Calculations:

3) i) $Q_c = [B : AB : A^2B : \dots : A^{n-1}B]$

$$Q_c = \begin{bmatrix} 1 : [-3 & -1 & 0] \\ 0 : [2 & 0 & 0] \\ 1 : [0 & -1 & -1] \end{bmatrix} \begin{bmatrix} 1 \\ 0 \\ 1 \end{bmatrix} = \begin{bmatrix} -3 & -1 & 0 \\ 2 & 0 & 0 \\ 0 & -1 & -1 \end{bmatrix} \begin{bmatrix} -3 & -1 & 0 \\ 2 & 0 & 0 \\ 0 & -1 & -1 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \\ 1 \end{bmatrix}$$

$$Q_c = \begin{bmatrix} 1 & -3 & 7 \\ 0 & 2 & -6 \\ 1 & -1 & -1 \end{bmatrix} \quad |Q_c| = -4 \neq 0$$

\therefore Given system is Controllable.

ii) $Q_b = \begin{bmatrix} c \\ cA \\ cA^2 \end{bmatrix} = \begin{bmatrix} [1 \ 0 \ 1] \\ [1 \ 0 \ 1] [A]_{3 \times 3} \\ [1 \ 0 \ 1] [A]_{3 \times 3} [A]_{3 \times 3} \end{bmatrix} = \begin{bmatrix} 1 & 0 & 1 \\ -3 & -2 & -1 \\ 5 & 4 & 1 \end{bmatrix}$

$|Q_b| = 0 \therefore$ Given system is not observable.

Observation Table:

	Theoretical Calculations	Matlab
Program 1	$T.F = \frac{2}{(s+2)(s-1)}$	$T.F = \frac{2}{(s+2)(s-1)}$
Program 2	$A = \begin{bmatrix} 0 & 1 \\ 2 & -2 \end{bmatrix} \quad B = \begin{bmatrix} 0 \\ 2 \end{bmatrix}$ $C = [1 \ 0]$	$A = \begin{bmatrix} -1 & 2 \\ 1 & 0 \end{bmatrix} \quad B = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$ $C = [0 \ 2] \quad D = [0]$
Program 3	$Q_c = \begin{bmatrix} 1 & -3 & 7 \\ 0 & 2 & -6 \\ 1 & -1 & -1 \end{bmatrix} \quad Q_b = \begin{bmatrix} 1 & 0 & 1 \\ -3 & -2 & -1 \\ 5 & 4 & 1 \end{bmatrix}$	$Q_c = \begin{bmatrix} 1 & -3 & 7 \\ 0 & 2 & -6 \\ 1 & -1 & -1 \end{bmatrix}$ $Q_b = \begin{bmatrix} 1 & 0 & 1 \\ -3 & -2 & -1 \\ 5 & 4 & 1 \end{bmatrix}$

Theoretical Calculations

$$TF = \frac{36}{s^3 + 6s^2 + 11s + 6} = \frac{36}{(s+1)(s+2)(s+3)}$$

no. of poles = 3; no. of zeros = 0; poles = -1, -2, -3

no. of branches = $P = 3$

no. of asymptotes = $|P - Z| = 3$

angle of asymptotes: $\phi = \frac{(2k+1)\pi}{|P-Z|}$; $k = 0, 1, 2, \dots, (P-Z-1)$

for $k=0$; $\phi_1 = -\frac{\pi}{3} = 60^\circ$

for $k=1$; $\phi_2 = \frac{3}{3} \times \pi = 180^\circ$

$k=2$; $\phi_3 = \frac{5\pi}{3} = 300^\circ$

Centroid = $\frac{\sum \text{real part of pole} - \sum \text{real parts of zeros}}{|P-Z|}$

$$\sigma = \frac{(-1-2-3) - 0}{3} = -2$$

root locus exist in b/w (-2, -1) and $(-\infty, -3)$

Break away point: $\frac{dk}{ds} = 0$

$$1 + G(s)H(s) = 0$$

$$1 + \frac{K}{s^3 + 6s^2 + 11s + 6} = 0$$

$$s^3 + 6s^2 + 11s + (6+K) = 0$$

$$K = -(s^3 + 6s^2 + 11s + 6)$$

$$\frac{dK}{ds} = 0 \Rightarrow \frac{d}{ds} [-(s^3 + 6s^2 + 11s + 6)] = 0$$

$$3s^2 + 12s + 11 = 0$$

$$s = -1.42, -2.57$$

Breakaway points always exist in root locus path.

Valid breakaway point is $s = -1.42$.

Point of intersection with imaginary axis:

$$1 + G(s)H(s) = 0$$

$$s^3 + 6s^2 + 11s + (6+K) = 0$$

s^3	1	11	
s^2	6	$6+K$	
s^1	$\frac{66-6-K}{6}$	0	
s^0	$6+K$	0	

From rows

$$\frac{60-K}{6} = 0$$

$$K = 60$$

Root locus intersects the imaginary axis at $\pm j\sqrt{11}$.

from row s^2 :

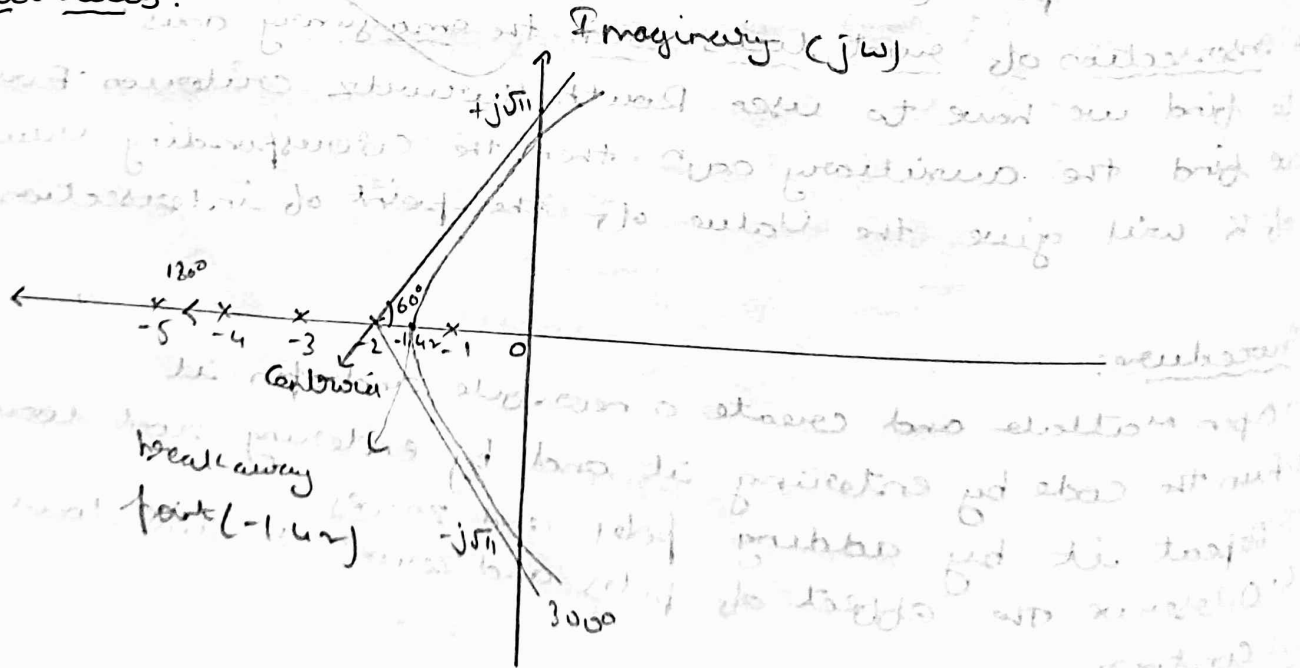
$$6s^2 + (6+K) = 0$$

$$6s^2 + 66 = 0$$

$$s^2 = -11$$

$$s = \pm j\sqrt{11} = \pm j3.316$$

Root locus:



Theoretical Calculations:

Bode plot:

$$TF = \frac{36}{s^3 + 6s^2 + 11s + 6} = \frac{36}{(s+1)(s+2)(s+3)} = \frac{6}{(1+s)(1+\frac{s}{2})(1+\frac{s}{3})}$$

Corner frequency: $\omega_1 = 1 \text{ rad/sec}$

$\omega_2 = 2 \text{ rad/sec}$

$\omega_3 = 3 \text{ rad/sec}$

Term

Corner frequency

Slope

$$\frac{1}{1+s}$$

-20 dB/Sec upto 1.

$$\frac{1}{1+s/2}$$

2

-40 dB/Sec

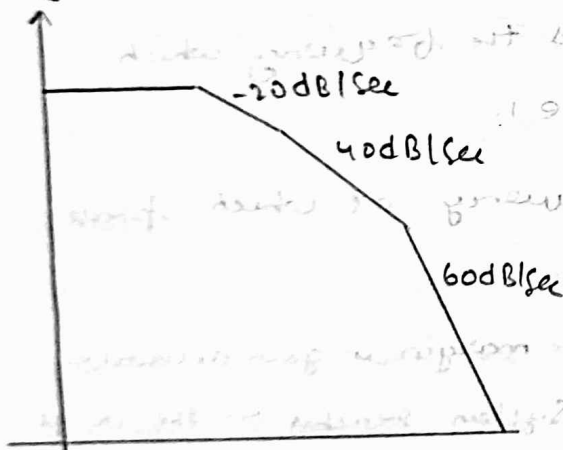
$$\frac{1}{1+s/3}$$

3

-60 dB/Sec

Slope = 0 upto 1.

$10 \log(j\omega) H(j\omega)$



$$\omega_{gco} = \sqrt{\frac{36}{1 \cdot 4 \cdot 9}} = 1 \text{ rad/sec}$$

$$\omega_{gco} = \sqrt{6.7} = 2.58 \text{ rad/sec}$$

$$PM = 180^\circ - \tan^{-1}(\omega) - \tan^{-1}(\omega/2) - \tan^{-1}(\omega/3)$$

$$PM = 19.09^\circ$$

$$\omega_{pc} = -\tan^{-1}(\omega) - \tan^{-1}(\omega/2) - \tan^{-1}(\omega/3) = -180^\circ \text{ at } \omega = \omega_{gco}$$

$$\omega_{pc} = 3.31 \text{ rad/sec}$$

$$\text{Gain Margin} = \left| \frac{36}{1 \cdot 4 \cdot 9} \right| = 4.436 \text{ dB}$$

Nyquist plot:

$$TF = \frac{36}{(s+1)(s+2)(s+3)}$$

$$T(j\omega) = \frac{36}{(1+j\omega)(2+j\omega)(3+j\omega)}$$

$$M = |G(j\omega)H(j\omega)| = \left| \frac{36}{\sqrt{1+\omega^2} \sqrt{4+\omega^2} \sqrt{9+\omega^2}} \right|$$

$$\phi = \angle G(j\omega)H(j\omega) = -\tan^{-1}(\omega) - \tan^{-1}(\omega/2) - \tan^{-1}(\omega/3)$$

G: polar plot:

at $\omega=0$; $M=6$, $\phi=0^\circ$

at $\omega=\infty$; $M=0$; $\phi=-270^\circ$

pole dominates \therefore clockwise direction
Meeting

End direction $= -270 - 0 = -270 = -ve$
 $\hookrightarrow C.W$

C2: Curve of zero magnitude

$$n-m = 3-0 = 3$$

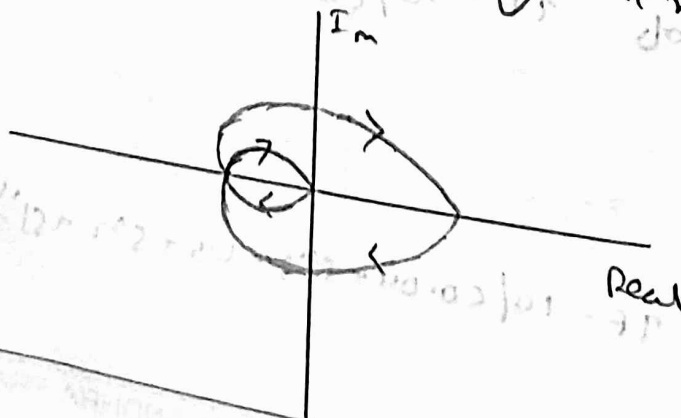
$$\therefore -\frac{3\pi}{2} \text{ to } \frac{3\pi}{2} = -270^\circ \text{ to } 270^\circ$$

C3: Sketch polar plot



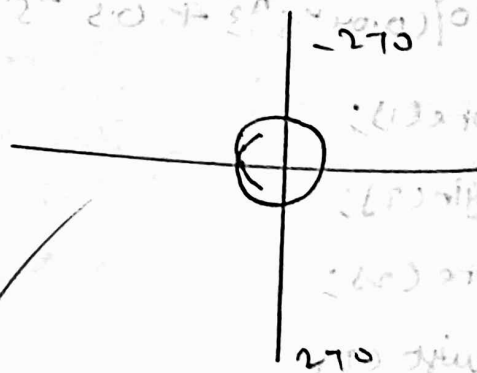
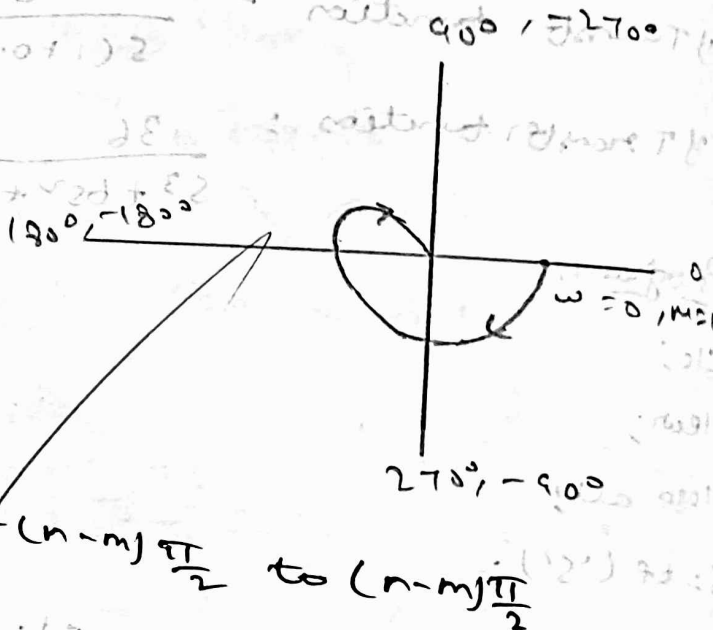
C4: Curve of infinite magnitude
Nyquist plot:

magnitude



from

$$+\frac{n\pi}{2} \text{ to } -\frac{n\pi}{2}$$



Theoretical calculations:-

1. Lead compensator:-

$$G_f(s) = \frac{k}{s(s+1)}, \text{ Given } k_v = 105', \phi_m = 35^\circ$$

$$\text{Taking } k=10, G_f(j\omega) = \frac{10}{j\omega(1+j\omega)}$$

phase margin of uncompensated system is

for lead compensator, phase lead required at new G_f is given by $\phi_c = 35 - 16 + 5 = 24$

$$\text{magnitude contribution} = 10 \log\left(\frac{1}{\alpha}\right) = -3.75 \text{ dB}$$

ie., frequency at which uncompensated system has magnitude of -3.75 dB

$$\omega_{g2} \text{ becomes } \omega_n, \omega_{g2} = 4 \text{ rad/sec}$$

$$\omega_1 = \frac{1}{T} = \omega_n \sqrt{\alpha} = 4 \sqrt{0.4217} = 6.16 \text{ rad/sec}$$

$$T.F = G_c(s) = \frac{14.5T}{1+0.5T} = \frac{14\left(\frac{1}{6.16}\right)s}{1+\left(\frac{1}{6.16}\right)s} = \frac{140.3855}{1+0.1623s}$$

$$A = \frac{1}{\alpha} = \frac{1}{0.4217} = 2.37$$

$$G_c(s) = G_f(s) G_{lc}(s) = \frac{10(140.3855)}{s(1+s)(1+0.1623s)}$$

$$\omega_{g2} = 4 \text{ rad/sec}$$

$$PM = 35^\circ$$

2. Lag compensator:-

$$G_f(s) = \frac{k}{s(1+s)(5+s)}, \quad \xi = 0.5, t_s = 12 \text{ sec}, \quad k_v \geq 85'$$

$$t_s = \frac{3}{\xi \omega_n} \Rightarrow \omega_n = 0.83 \text{ rad/sec}$$

$$\omega_b = \omega_n \sqrt{1-2\xi^2 + \sqrt{2-4\xi^2+4\xi^4}} = 1.217 \text{ rad/sec}$$

$$\phi_{PM} = \tan^{-1} \left(\frac{2\xi}{\sqrt{1+4\xi^2-2\xi^2}} \right)^{1/2} = 33^\circ$$

$$G_f(s) = \frac{k/5}{s(1+s)(1+0.2s)}$$

$$k_v = \lim_{s \rightarrow 0} s G_f(s) = \frac{k}{5} (5) \Rightarrow k_v = 8 \times 5 = 40$$

$$G_c(j\omega) = \frac{8}{(j\omega)(j\omega+1)(1+0.2j\omega)} \Rightarrow \omega_{gc} = 2.7 \text{ rad/sec}$$

$$PM = 8^\circ$$

$$\psi_2 = 33^\circ + 10^\circ = 43^\circ, \quad \omega_2 = \frac{1}{T} = \frac{\omega_{gc}}{2} = 0.2 \text{ rad/sec}$$

To bring log magnitude curve to zero dB, at which ω_{gc} attenuation must be 18dB

$$20 \log B = 18 \Rightarrow B = 8$$

$$\omega_1 = \frac{1}{BT} = \frac{1}{8 \times 5} = 0.025 \text{ rad/sec}, \quad BT = 40$$

$$G(s) = \frac{1}{s} \left[\frac{(s+0.2)}{(s+0.025)} \right] = \frac{1+5s}{1+40s}$$

$$\text{phase lag} = \angle \tan^{-1}(5\omega_{gc}) - \angle \tan^{-1}(40\omega_{gc}) = 75.96^\circ - 88^\circ = -12^\circ$$

$$G(s) = G_f(s) \cdot G_c(s) = \frac{8(1+5s)}{s(1+s)(1+0.2s)(1+40s)}$$

3 Lag lead compensator:

$$G_f(s) = \frac{k}{s(1+0.5s)(1+0.1s)}; \quad k_v \geq 25 \text{ s}^{-1}, \quad \psi_s \geq 60^\circ, \quad \omega_b = 10 \text{ rad/sec}$$

$$k_v = \lim_{s \rightarrow 0} sG(s) = k = k_v = 25 \Rightarrow G_f(j\omega) = \frac{25}{j\omega(1+0.5j\omega)(1+0.1j\omega)}$$

Let G_c lag compensator provide $PM = 36^\circ$, so new crossover frequency is 2 rad/sec.

For this to happen magnitude of bode plot must be 18dB

$$20 \log \beta = 18 \Rightarrow \beta = 8$$

$$\omega_2 = \frac{1}{T} = \frac{\omega_{cf}}{2} = 0.5, \quad T = 2 \text{ sec}$$

$$\omega_1 = \frac{1}{\beta T} = 0.0625 \text{ rad/sec}, \quad \beta T = 16$$

$$G_c(s) = \frac{1+Ts}{1+\beta Ts} = \frac{25(1+2s)}{s(1+0.5s)(1+0.1s)(1+16s)}$$

$$PM = 18^\circ, \quad \omega_{cf} = 2.25 \text{ rad/sec}$$

$$\alpha = \frac{1}{\beta} = \frac{1}{8} = 0.125, \quad \psi_m = \sin^{-1}\left(\frac{1-\alpha}{1+\alpha}\right) = 51.05^\circ$$

$$-10 \log\left(\frac{1}{\alpha}\right) = -9 \text{ dB}$$

$$\omega_1 = \frac{1}{T_2} = \omega_m \sqrt{\alpha} = 3.8 \sqrt{0.125} = 1.34 \text{ rad/sec}$$