

## I. Oscillations & Waves

types of oscillators:

SHO → simple

DHO → damped

FHO → forced.

### > Damped Harmonic Oscillator

Damping → decrease in amplitude.

#### Theory:

2-types of forces mainly

↓  
Restoring force ( $-\ell x$ )

Damping force ( $-\gamma \frac{dx}{dt}$ )

#### Force eq. of DHO:

$$m \frac{d^2x}{dt^2} = -\ell x - \gamma \frac{dx}{dt} \quad \begin{matrix} \gamma = \text{damping} \\ \text{coeff} \end{matrix}$$

$$m \frac{d^2x}{dt^2} + \gamma \frac{dx}{dt} + \ell x = 0 \quad \div m$$

$$\frac{d^2x}{dt^2} + \frac{\gamma}{m} \frac{dx}{dt} + \frac{\ell}{m} x = 0 \quad \frac{\gamma}{2m} = K$$

$$\sqrt{\frac{\ell}{m}} = \omega_0 \text{ const.}$$

↓  
natural angular freq.

→ force eq.

$$\therefore \frac{d^2x}{dt^2} + 2K \frac{dx}{dt} + \omega_0^2 x = 0 \quad - ①$$

Sol.  $x = A e^{\alpha t}$

$$\frac{dx}{dt} = A \alpha e^{\alpha t}$$

$$\frac{d^2x}{dt^2} = A \alpha^2 e^{\alpha t}$$

$$\therefore ① \rightarrow A \alpha^2 e^{\alpha t} + 2K A \alpha e^{\alpha t} + \omega_0^2 A e^{\alpha t} = 0$$

$$A e^{\alpha t} (\alpha^2 + 2K\alpha + \omega_0^2) = 0$$

$$\therefore \alpha^2 + 2K\alpha + \omega_0^2 = 0$$

$$\alpha = \frac{-2K \pm \sqrt{4K^2 - 4\omega_0^2}}{2}$$

$$\alpha_1 = -K + \sqrt{K^2 - \omega_0^2}$$

$$\alpha_2 = -K - \sqrt{K^2 - \omega_0^2}$$

gen. sol. of DHO

$$x = A_1 e^{\alpha_1 t} + A_2 e^{\alpha_2 t}$$

$$= A_1 e^{(-K + \sqrt{K^2 - \omega_0^2})t} + A_2 e^{(-K - \sqrt{K^2 - \omega_0^2})t}$$

$$x = e^{-Kt} \left[ A_1 e^{\sqrt{K^2 - \omega_0^2} t} + A_2 e^{-\sqrt{K^2 - \omega_0^2} t} \right] \quad - ②$$

Based on the values of  $K$  &  $\omega_0$ , DHO can be classified into 3.

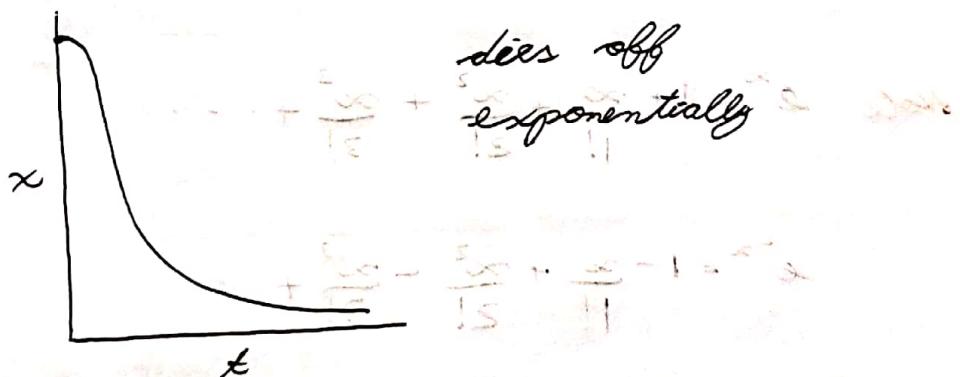
- >  $K > \omega_0$  → Over damped  $s = -\alpha \pm j\beta$
- >  $K = \omega_0$  → Critically damped  $s = -\alpha$
- >  $K < \omega_0$  → Under damped  $s = \alpha \pm j\beta$

Case 1;  $K > \omega_0$ ; Over damped.

Let  $\sqrt{K^2 - \omega_0^2} = \beta$ .

$$\begin{aligned}\therefore \textcircled{2} \rightarrow x &= e^{-Kt} [A_1 e^{\beta t} + A_2 e^{-\beta t}] \\ &= \underline{\underline{A_1 e^{(-K+\beta)t} + A_2 e^{(K-\beta)t}}}\end{aligned}$$

i.e. The disp. varies exponentially.



[Non-oscillatory  
Aperiodic / dead beat oscillators] → used in dead-beat voltmeter.

Case 2:  $K = \omega_0$ ; Critically damped

$$\text{Q} \Rightarrow x = e^{-Kt} [A_1 e^{\omega_0 t} + A_2 e^{-\omega_0 t}] \\ = e^{-Kt} [A_1 + A_2]$$

Let  $A_1 + A_2 = B$

$\therefore \underline{x = Be^{-Kt}}$   $\rightarrow$  Only one variable  
const.  $\cancel{\omega < K}$   
 $\therefore$  It doesn't form sol.  
of 2<sup>nd</sup> order diff. eq.

$\therefore$  Substitute  $\sqrt{K^2 - \omega^2} = h$ ;  $h$  is a small  
qty.

$$\Rightarrow x = e^{-Kt} [A_1 e^{ht} + A_2 e^{-ht}]$$

Note:  $e^x = 1 + \frac{x}{1!} + \frac{x^2}{2!} + \frac{x^3}{3!} + \dots$

$$e^{-x} = 1 - \frac{x}{1!} + \frac{x^2}{2!} - \frac{x^3}{3!} + \dots$$

$$\therefore e^{ht} = 1 + \frac{ht}{1!} + \frac{(ht)^2}{2!} + \dots \approx 1 + ht$$

$$e^{-ht} = 1 - ht$$

$$\therefore x = e^{-kt} [A_1(1+kt) + A_2(1-kt)]$$

$$= e^{-kt} [A_1 + A_2 + (A_1 - A_2)kt]$$

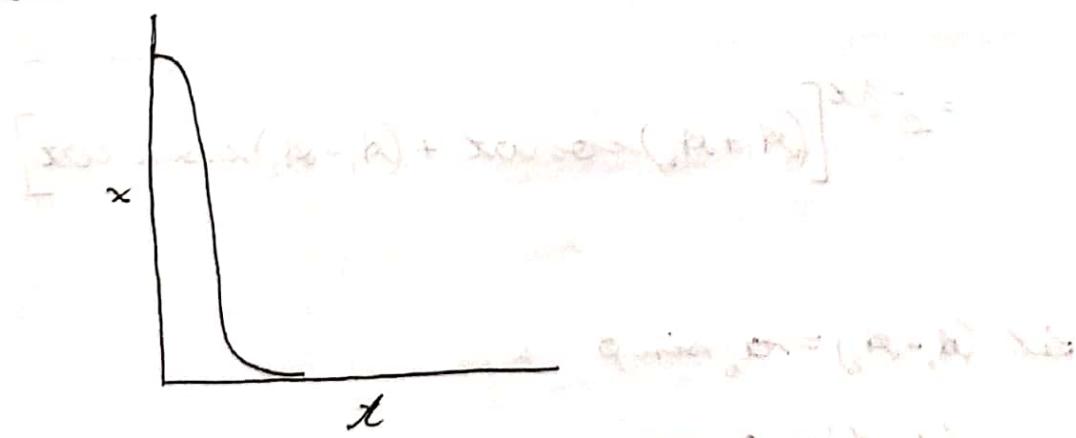
Let  $A_1 + A_2 = D$

$$(A_1 - A_2)k = \epsilon$$

$$\therefore x = e^{-kt} [D + \epsilon t]$$

$$\underline{x = e^{(D+\epsilon t)} e^{-kt}}$$

due to the  $(D + \epsilon t)$ , there will be a max. value, then it decreases, faster than the over damped case.



[Non-oscillatory]  
[Sub-aperiodic] → in galvanometers.

Case 3:  $K < \omega_0$ ; Under damped

$$\sqrt{K^2 - \omega_0^2} = \sqrt{-(\omega_0^2 - K^2)}$$
$$= i\omega \quad ; \quad \omega_0^2 - K^2 = \omega^2$$

$\omega$  = angular freq.

$$\therefore x = e^{-Kt} \left[ A_1 e^{\sqrt{K^2 - \omega_0^2} t} + A_2 e^{-\sqrt{K^2 - \omega_0^2} t} \right]$$
$$= e^{-Kt} \left[ A_1 e^{i\omega t} + A_2 e^{-i\omega t} \right]$$

$$e^{i\theta} = \cos \theta + i \sin \theta$$

$$\therefore x = e^{-Kt} \left[ A_1 (\cos \omega t + i \sin \omega t) + A_2 (\cos \omega t - i \sin \omega t) \right]$$

$$= e^{-Kt} \left[ (A_1 + A_2) \cos \omega t + (A_1 - A_2) i \sin \omega t \right]$$

$$\text{Let } (A_1 + A_2) = a_0 \sin \phi$$

$$i(A_1 - A_2) = a_0 \cos \phi$$

$$\therefore x = e^{-Kt} \left[ a_0 \sin \phi \cos \omega t + a_0 \cos \phi \sin \omega t \right]$$

$$\sin(x+y) = \sin x \cos y + \cos x \sin y$$

$$x = e^{-kt} \alpha_0 \sin(\phi + \omega t)$$

$$x = \alpha_0 e^{-kt} \sin(\omega t + \phi)$$

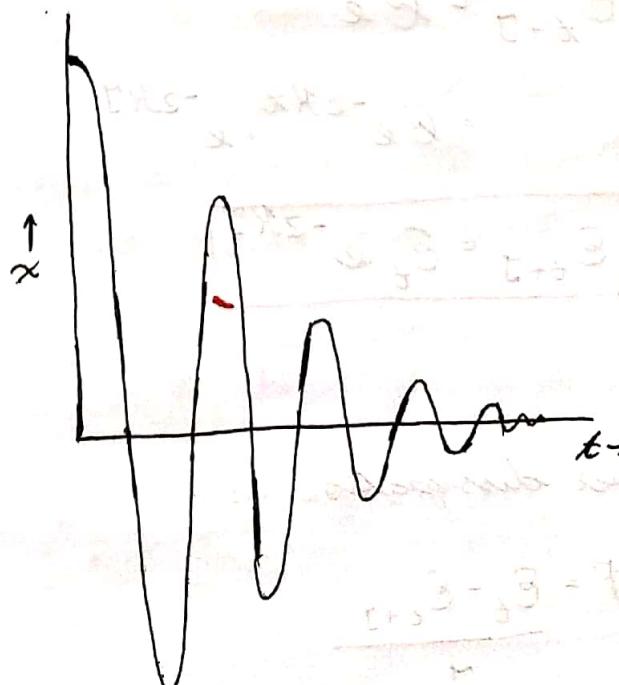
where,  $\alpha_0 e^{-kt}$  → Amplitude of damped harmonic oscillators

$$\omega = \sqrt{\omega_0^2 - k^2}$$

$$\zeta = \frac{2\pi}{\omega} = \frac{2\pi}{\sqrt{\omega_0^2 - k^2}}$$

In underdamped, two cases are possible.

- > Decrease in  $\omega$ ,  $\zeta$  increases
- > Amplitude term decreases exponentially with time



## Power dissipation of damped oscillator

It is the rate of dissipation of energy.

Defined as the ratio of energy loss in one period to the total time period

$$P = \frac{\text{energy loss in 1 period}}{\text{tot. time period}}$$

Tot. energy of oscillator

$$E_t \propto (a_0 e^{-Kt})^2$$

$$E_t = \rho e^{-2Kt} \rightarrow \text{Tot. energy}$$

Energy of oscillator after 1 cycle/period

$$E_{t+T} = \rho e^{-2K(t+T)}$$

$$= \rho e^{-2Kt} \cdot e^{-2KT}$$

$$E_{t+T} = E_t e^{-2KT}$$

$\therefore$  Power dissipation is:

$$P = \frac{E_t - E_{t+T}}{T}$$

$$= \frac{E_t - E_t e^{-2KT}}{J}$$

$$= \frac{E_t - E_t (1 - 2KT)}{J} \quad \text{) expn. of } e^{-2KT} \\ = 1 - \frac{e^{-2KT}}{1!} + \frac{e^{-2KT}}{2!} - \frac{e^{-2KT}}{3!} \dots$$

$$= \frac{E_t - E_t + 2KT E_t}{J}$$

$$\Rightarrow \underline{\underline{2KT}} \quad \underline{\underline{E_t}}$$

$$\therefore P = \frac{E_t}{\tau}$$

$$\tau = \frac{1}{2KT} \rightarrow \text{Relaxation time}$$

Time after which the energy is reduced to  $\frac{1}{e}$  of its initial value.

### Quality factor of damped oscillator (Q)

Quality factor is the ratio of  $2\pi$  times the energy stored in the system to the energy lost/Energy dissipated per unit time.

$$\text{i.e.: } Q = 2\pi \frac{E_t}{P\tau}$$

$$= 2\pi \frac{E_t}{\left(\frac{E_t}{\tau}\right)\tau}$$

$$Q = \frac{2\pi}{J} \tau = \omega \tau$$

## Forced/Driven harmonic oscillator

An oscillator that oscillates <sup>by force</sup> at a particular frequency, other than its natural frequency is said to be FHO.  
eg: Piano.

### Theory

3-types of forces are experienced.



Restoring force  $\rightarrow (-Cx)$

Damping force  $\rightarrow \left(-\frac{Rdx}{dt}\right)$

Ext. driving force  $\rightarrow (F_0 \sin pt)$

$\rightarrow \frac{R}{2m} \rightarrow \text{freq.}$

### Force eq. of FHO

$$m \frac{d^2x}{dt^2} = -Cx - \frac{Rdx}{dt} + F_0 \sin pt$$

$$m \frac{d^2x}{dt^2} + Cx + \frac{R}{m} \frac{dx}{dt} = F_0 \sin pt$$

$$\frac{d^2x}{dt^2} + \frac{C}{m}x + \frac{R}{m} \frac{dx}{dt} = \frac{F_0}{m} \sin pt$$

$$\frac{d^2x}{dt^2} + \omega_0^2 x + 2K \frac{dx}{dt} = \frac{F_0}{m} \sin pt$$

$$\sqrt{\frac{C}{m}} = \omega_0$$

$$\frac{R}{2m} = K$$

$$\boxed{\frac{d^2x}{dt^2} + 2K \frac{dx}{dt} + \omega_0^2 x = \frac{F_0}{m} \sin pt} \quad -① \quad \frac{F_0}{m} = f_0$$

### Solution

- It is a non-homogeneous diff. eq.  
 It contains  
 1) Complementary funct (CF)  
 2) Particular integral (PI)

$$\text{CF : } \ddot{x} + 2K \frac{dx}{dt} + \omega_0^2 x = 0$$

Its sol. is:

$$x = a_0 e^{-kt} \sin(\omega t + \phi)$$

PI : Set

$$x = A \sin(pt - \theta)$$

$$\frac{dx}{dt} = pA \cos(pt - \theta)$$

$$\frac{d^2x}{dt^2} = -p^2 A \sin(pt - \theta)$$

$$\therefore \textcircled{1} \rightarrow -p^2 A \sin(pt - \theta) + 2KpA \cos(pt - \theta)$$

$$+ \omega_0^2 A \sin(pt - \theta) = f_0 \sin pt$$

$$A \sin(pt - \theta) [\omega_0^2 - p^2] + 2KpA \cos(pt - \theta) = f_0 \sin pt$$

$$= f_0 \sin(pt - \theta + \phi)$$

$$= f_0 [ \sin(pt - \theta) \cos \phi + \cos(pt - \theta) \sin \phi ]$$

$$= f_0 \sin(pt - \theta) \cos \phi + f_0 \cos(pt - \theta) \sin \phi$$

$$= f_0 \sin(pt - \theta) \cos \phi + f_0 \cos(pt - \theta) \sin \phi$$

Equating the coeff. of  $\sin(pt - \Theta)$  &  $\cos(pt - \Theta)$

$$\text{ie: } f_0 \cos \Theta = A(\omega_0^2 - p^2) \quad \text{---(2)}$$

$$f_0 \sin \Theta = 2Kp \omega_0 t \quad \text{---(3)}$$

Sq + add, we get:  $①^2 + ③^2$

$$f_0^2 = A^2 (\omega_0^2 - p^2)^2 + (2Kp \omega_0 t)^2$$

$$\overline{f_0} = \sqrt{A^2 (\omega_0^2 - p^2)^2 + (2Kp \omega_0 t)^2}$$

$$A = \frac{f_0}{\sqrt{(\omega_0^2 - p^2)^2 + 4K^2 p^2}} \rightarrow \text{Amplitude}$$

Phase diff.

$$\frac{\Theta}{\Theta} \rightarrow \tan \Theta = \frac{2Kp}{(\omega_0^2 - p^2)}$$

$$\Theta = \tan^{-1} \frac{2Kp}{(\omega_0^2 - p^2)} \rightarrow \text{phase diff}$$

Substituting 'A' in PI,  $x = A \sin(pt - \Theta)$ :

$$x = \frac{f_0}{\sqrt{(\omega_0^2 - p^2)^2 + 4K^2 p^2}} \sin(pt - \Theta)$$

The gen. sol. of FHO is

$$\ddot{x} = CF + PI$$

$$x = a_0 e^{-\alpha t} \sin(\omega_0 t + \phi) + \frac{f_0}{\sqrt{(\omega_0^2 - \beta^2)^2 + 4K^2\beta^2}} \sin(\beta t - \Theta)$$

*damped*                            *forced*

Initially both vibrations are present in FHO, but with the passage of time, first term vanishes.

i.e.: The sol. of FHO is:

$$x = \frac{f_0}{\sqrt{(\omega_0^2 - \beta^2)^2 + 4K^2\beta^2}} \sin(\beta t - \Theta)$$

### Resonance condition for FHO

#### 1) Amplitude resonance

The amplitude of FHO is max. at a particular frequency, which is near to its natural frequency.

Resonant freq. is denoted by  $\nu_R$ .

We know that  $A = \frac{f_0}{\sqrt{(\omega_0^2 - \beta^2)^2 + 4K^2\beta^2}}$

For  $A = A_{\max}$ , denominator is min.

$\therefore$  For denominator to be min,

$$\frac{d}{d\theta} (\Delta_r) = 0$$

$$\text{ie: } \frac{d}{dp} [(w_0^2 - p^2)^2 + 4K^2 p^2] = 0$$

$$-4p(w_0^2 - p^2) + 8K^2 p = 0$$

~~$$\text{ie: } 8K^2 p = 4p(w_0^2 - p^2)$$~~

~~$$2K^2 = (w_0^2 - p^2)$$~~

~~$$\sqrt{2}K = w_0^2 - p^2$$~~

~~$$8K^2 p = 4p(w_0^2 - p^2)$$~~

~~$$p^2 = w_0^2 - 2K^2$$~~

When  $A = A_{\max}$ ,  $P = P_a$

$$\text{ie: } P_a = \sqrt{w_0^2 - 2K^2} - 0$$

Substituting  $P_a$  in  $A_{\max}$ ,

$$A_{\max} = \frac{b_o}{\sqrt{(w_0^2 - p^2)^2 + 4K^2 p^2}} \quad ; \quad P = P_a$$

~~$$= \frac{b_o}{\sqrt{(w_0^2 - w_0^2 + 2K^2)^2 + 4K(w_0^2 - 2K^2)}}$$~~

$\cong$

$$= \frac{f_0}{\sqrt{(\omega_0^2 - \rho_k^2)^2 + 4K^2\rho_k^2}}$$

$$\Rightarrow \rho_k^2 = \omega_0^2 - 2K^2 \rightarrow \omega_0^2 - \rho_k^2 = 2K^2$$

$$= \frac{f_0}{\sqrt{(2K^2)^2 + 4K^2\rho_k^2}}$$

$$A_{max} = \frac{f_0}{2K\sqrt{\mu^2 + \rho_k^2}}$$

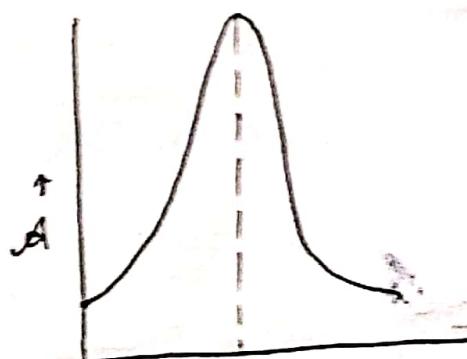
Case 1 (low damping)

Neglecting  $\mu^2$

$$\Rightarrow \rho_k \approx \omega_0 ; A_{max} = \frac{f_0}{2K\omega_0}$$

~~Case 2 (moderate damping)~~

$$\therefore A_{max} = \frac{f_0}{2K\omega_0}$$

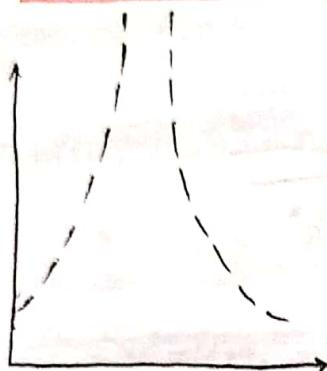


freq

### Case 2 (absence/0 damping)

$$\kappa = 0$$

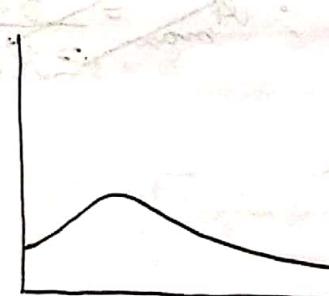
$$\Rightarrow A_{\text{max}} = \infty$$



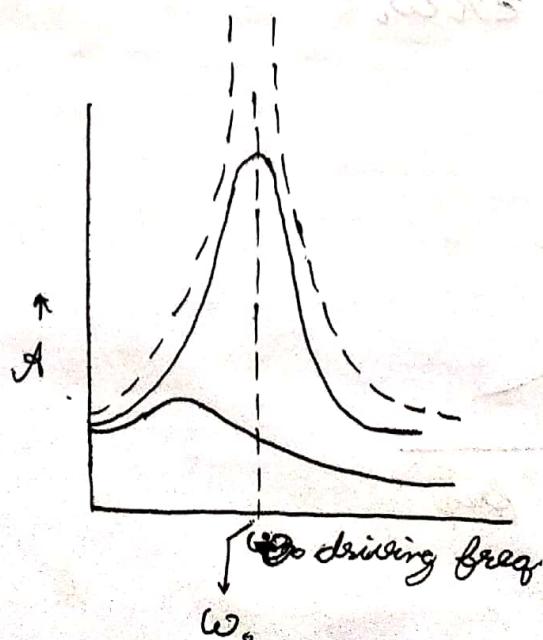
### Case 3 (High/finite damping)

A decreases

$$\text{i.e. } \rho_R < \omega_0$$



i.e.:



## Sharpness & flat of resonance

Amplitude of resonance decreases rapidly on both sides of the resonant freq.

This resonance cond. is called sharpness.

eg: sonometer

↓  
in the case of low damping

Amplitude decreases very slowly on either sides. This is called flat of resonance.

↓  
in the case of high damping

eg: air column exp

## Quality factor at resonance ( $\underline{\underline{Q=3740}}$ )

It is the ratio of amplitude at max to the amplitude at zero driving freq.

↓  
ie:  $p=0$ .

ie:

$$Q = \frac{A_{\max}}{A_{p=0}}$$

$$A_{\max} = \frac{f_0}{2K\omega_0}$$

$$= \frac{f_0}{2K\omega_0}$$
  
$$= \frac{f_0}{\omega_0}$$

$$A = \frac{f_0}{\sqrt{(\omega_0^2 - p^2)^2 + 4K^2p^2}}$$

;  $p=0$

$$Q = \frac{\omega_0}{2K} = \omega_0 \tau$$

$$\tau = \frac{1}{2K}$$

$$\text{Now: } Q = \frac{\omega_0}{2K} = \frac{\omega_0^2 \sqrt{\frac{R}{m}}}{2\pi}$$

$$= \frac{\sqrt{\frac{R}{m}}}{\frac{1}{2\pi}}$$

$$= \frac{\sqrt{Rm}}{\gamma}$$

$$K = \frac{1}{2\pi}$$

$$Q = \frac{\omega_0}{2K} = \omega_0 \tau = \frac{\sqrt{Rm}}{\gamma}$$

- Q. In the case of forced harmonic oscillator the amplitude increases from 0.02 mm at very low freq. to 5 mm at  $\nu = 100 \text{ Hz}$ . Find Q, F, damping const. & relaxation time.

$$\text{Ans. } A_0 = 0.02 \text{ mm}$$

$$A_{\max} = 5 \text{ mm}$$

$$Q = \frac{A_{\max}}{A_{p=0}} = \frac{5}{0.02} = \underline{\underline{250}}$$

$$\omega = 2\pi\nu = 200\pi \text{ rad/s}$$

$$\text{Relaxation time } \tau = \frac{Q}{\omega_0} = \frac{250}{200\pi} = \underline{\underline{3.92}}$$

$$\text{Damping const } K = \frac{\omega_0}{2Q} = \frac{200\pi}{2 \times 250} = \frac{200\pi}{500} = \underline{\underline{1.25}}$$

A damped vibrating sys. from rest reaches a 1<sup>st</sup> Amp. of 500 mm, which reduces to 50 mm after 100 osci, each of time period 2.3 sec. Find damp. const & relaxation time.

Ans.  $\alpha_0 = 2$

$$500 \times 10 = e^{\frac{-2t}{2.3}} \quad \text{--- (1)}$$

$$10 = \frac{e^{\frac{100 \times 2}{2.3}}}{e^{\frac{200}{2.3}}} = e^{\frac{99.9}{2.3}}$$

$$227.7 R$$

$$10 = e^{\frac{2.303}{2.3}} = e^{0.1}$$

$$\ln 10 = 227.7 R$$

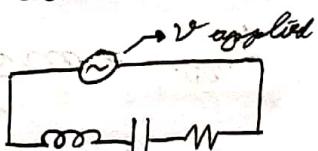
$$2.303 = 227.7 R$$

$$\underline{R = 0.01}$$

$$\tau = \frac{1}{2R} = \underline{50}$$

### Comparison of mechanical & electric oscillator

$$FHO \rightarrow m \frac{d^2x}{dt^2} + R \frac{dx}{dt} + Kx = F_{\text{applied}} \quad \text{--- (1)}$$



In an LCR circuit;

Pot. diff. across each component is:

Inductor :  $V_L = L \frac{di}{dt} = L \frac{dq}{dt}$

Resistor :  $V_R = R \frac{dq}{dt}$

$$\text{Capacitor: } V_c = \frac{Q}{C}$$

Consider the mechanical oscillator as FHO.  
Its force eqn. is given above.

For an LCR circuit, the voltage eqn. is given by the sum of P.d across each component.

$$V_{\text{applied}} = V_L + V_R + V_C$$

$$V_{\text{appl.}} = L \frac{d^2 q}{dt^2} + R \frac{dq}{dt} + \frac{1}{C} q \quad \text{--- (2)}$$

On comparison of eq. ① & ②, we can compare the terms.

$$\text{i.e. } V_{\text{appl.}} = L \frac{d^2 q}{dt^2} + R \frac{dq}{dt} + \frac{1}{C} q \quad \left. \right\}$$

$$F_{\text{appl.}} = m \frac{d^2 x}{dt^2} + R \frac{dx}{dt} + Kx$$

i.e. FHO force eq. is similar to voltage eq. of LCR circuit.

Mechanical  
Oscillator

Mass ( $m$ )

Displacement ( $x$ )

Velocity ( $dx/dt$ )

Damping const ( $K$ )

Damping coeff ( $\gamma$ )

tot. energy ( $\frac{1}{2}Kx^2$ )

K.E ( $\frac{1}{2}mv^2$ )

$$Q.F; Q = \frac{\sqrt{km}}{\gamma}$$

Electrical  
Oscillator

Inductance ( $L$ )

Charge ( $q$ )

Current ( $dq/dt$ )

$1/R$

Resistance ( $R$ )

$\frac{1}{2}R^2\omega^2$

$\frac{1}{2}L^2\omega^2$

$$Q = \frac{1}{R} \sqrt{\frac{L}{R}} = \frac{L\omega}{R}$$

Resonant freq.

$$\omega_r = \sqrt{\omega_0^2 - 2K^2}$$

$$\omega_r = \frac{1}{2\pi\sqrt{LC}}$$

- Q. Find the natural freq. of a circuit containing inductor of  $144 \mu H$  & capacitor of  $0.0025 \mu F$ . To which  $\lambda$  will its response will be max.

$$\text{Ans } L = 144 \mu H ; R = 0.0025 \mu F$$

$$\omega = \frac{1}{2\pi\sqrt{LC}} = \frac{1}{2\pi\sqrt{144 \times 0.0025 \times 10^{-12}}} = 2.65 \times 10^5 \text{ rad/s}$$

$$c = \nu \lambda \Rightarrow \lambda = \frac{c}{\nu}$$

$$= \frac{3 \times 10^8}{2.65 \times 10^5} \text{ m}$$

$$\underline{\lambda = 1.13 \times 10^{-12} \text{ m}}$$

Q. deduce freq. & Q.F for a circuit with

$$L = 2 \text{ mH}, Q = 5 \mu\text{F}, R = 0.2 \Omega$$

Ans.  $Q = \frac{1}{R} \sqrt{\frac{L}{C}}$

$$= \frac{1}{0.2} \sqrt{\frac{2 \times 10^{-3}}{5 \times 10^{-6}}} \text{ s}$$

$$= \frac{1}{0.2} \sqrt{\frac{2 \times 10^3}{5}} \text{ s} \quad \underline{\underline{\frac{20}{0.2} = 100 \text{ s}}}$$

$$f = \frac{1}{2\pi \sqrt{LC}} = \underline{\underline{1.59 \times 10^{-3} \text{ Hz}}}$$

Q. In LCR circuit,  $L = 10 \text{ mH}$ ,  $Q = 1 \mu\text{F}$  &  $R = 0.1 \Omega$

How long does the oscillation take to decay to  $\frac{1}{2}$  the amplitude Q.F?

Ans.  $A = A_0 e^{-Kt}$

~~$A = A_0 e^{-Kt_1}$~~ 

$$A_1 = \frac{A_0}{2} e^{-Kt_2}$$

$$\frac{A_1}{A} = \frac{A_0}{2} e^{-Kt_2}$$

~~$\therefore A_2 = A_0 e^{-Kt_2}$~~

$$Z = \frac{e^{-Kt_1}}{e^{-Kt_2}}$$

$$Q = \frac{1}{R} \sqrt{\frac{L}{C}} =$$

$$= \frac{1}{0.1} \sqrt{\frac{10 \times 10^{-3}}{1 \times 10^{-6}}} \text{ s}$$

$$= \frac{1}{0.1} \sqrt{10^4} = \frac{10}{0.1} \text{ s}$$

$$\underline{\underline{1000 \text{ s}}}$$

$$Z = e^{Kt}$$

$$Z = e^{\frac{Kt}{1000}}$$

$$0.693 =$$

$$\omega = \frac{1}{\sqrt{LC}}$$

$$Q = \frac{\omega}{2K}$$

$$K = 5$$

$$Z = L$$

$$0.693 =$$

Q. Rms volt &

$R = 10 \Omega$ ,  $L$   
calc natural  
resonance,

Ans.  $\omega = \frac{1}{\sqrt{LC}}$

$$V = \frac{1}{2\pi \sqrt{L}}$$

$$= \underline{\underline{1592}}$$

At resonan

$$Z = \sqrt{R^2 + (\dots)^2}$$

$$\therefore \Phi = \frac{100}{10}$$

$$Z = e^{\frac{K(E_2 - E_1)}{1 \times 10^{-6}}}$$

$$Z = e^{\frac{K(E_2 - E_1)}{10^{-6}}} \Rightarrow J = 0.693 \times 10^{-6} A$$

$$\omega = \frac{1}{\sqrt{LC}} = 10000$$

$$Q = \frac{L\omega}{R}$$

$$Q = \frac{\omega}{2K} \Rightarrow K = \frac{\omega}{2Q} = \frac{10000}{2 \times 1000} = 5$$

$$K = 5$$

and general expression for voltage across  $KJ$  will go that it would be  $5J$  as  $J$  is same as  $I$ .  
 $Z = L$  as no voltage and all resistance has  $0.693 = 5J$

$$\Rightarrow J = 0.138 A$$

- Q. Rms volt of  $100V$  is applied to  $LCR$  circuit.  $R = 10\Omega$ ,  $L = 10 \times 10^{-3} H$ ,  $C = 1 \times 10^{-6} F$ . Calc natural freq. of circuit, current at resonance, Q.F., band width.

$$\text{Ans. } \omega = \frac{1}{\sqrt{LC}}$$

$$V = \frac{1}{2\pi\sqrt{LC}} = \frac{1}{2\pi\sqrt{10^{-3} \times 10^{-6}}} = 1592 Hz$$

$$\approx \frac{1592}{200} = 7.96$$

At resonance,  $X_C = X_L$

$$Z = \sqrt{R^2 + (X_C - X_L)^2} = 10$$

$$\omega = 100 - 10 \cdot 4$$

$$Q.F = \frac{1}{R} \sqrt{\frac{L}{C}}$$

$$= \frac{1}{10} \sqrt{\frac{10 \times 10^{-3}}{10^{-3}}} = 10$$

$$= \frac{1}{10} \times 10^2 = \underline{\underline{10}}$$

Data

$$\boxed{\text{Bandwidth} = \frac{\omega_0}{Q}}$$

$$\therefore \omega = \frac{\omega_0}{Q} = \frac{2\pi D}{Q} = \frac{1000 \cdot 2}{1000} = \underline{\underline{20 \text{ rad/sec}}}$$

- Q. Amplitude of underdamped harmonic oscillator reduced to  $\frac{1}{10}$ th of initial value after 100 oscillations. Its time period after 100 osci is  $1.15 \text{ sec}$ . Calc. relaxation time & damping const.

After  $\frac{A}{A_0} = e^{-Kt}$

$$\frac{1}{10} = e^{-K \cdot 100t}$$

$$10 = e^{K(100t - 1)}$$

$$= e^{99tK}$$

$$= 113.85K$$

$$2.303 = 113.85K$$

$$\underline{\underline{K = 0.02}}$$

$$\tau = \frac{1}{2K} = \frac{1}{0.04} = \underline{\underline{25 \text{ s}}}$$

Q. Find natural freq. of a circuit containing  
 $L = 50 \times 10^{-3} \text{ H}$ ,  $R = 500 \times 10^{-12} \propto \lambda$ .

Ans.  $\omega = \frac{1}{\sqrt{25000 \times 10^{-12}}} = \frac{1}{5 \times 10^6}$

$= 0.2 \times 10^6$

$\nu = \frac{10^5}{16\pi} = 31830.8 \text{ Hz} = 3.18 \times 10^3 \text{ Hz}$

$$c = \nu \lambda \Rightarrow \lambda = \frac{c}{\nu} = \underline{\underline{9.424 \times 10^3 \text{ m}}}$$

$$\theta = \omega t + (\omega_0 + \omega) \frac{t}{2} = (\omega_0 + \omega) \frac{t}{2}$$

$$(\omega_0 + \omega) \frac{t}{2} = \underline{\underline{\omega_0 t}}$$

## Waves

### Differential eq. of 1-dime. wave

It is represented as a funct. of pos. & time.

Consider a 1-d wave with vel.  $v$ , moving in  $+x$  dir.

$$\Psi(x, t) = f(x - vt) \rightarrow +x \text{ dir. } -\textcircled{1}$$

$$\Psi(x, t) = f(x + vt) \rightarrow -x \text{ dir. } -\textcircled{2}$$

In gen;

$$\Psi(x, t) = f(x \pm vt)$$

Differentiating  $\textcircled{1}$  partially wrt.  $x$  twice,

$$\frac{\partial \Psi}{\partial x} = f'(x - vt)$$

$$\frac{\partial^2 \Psi}{\partial x^2} = f''(x - vt)$$

Now, wrt.  $t$ , twice

$$\frac{\partial^2 \Psi}{\partial t^2} = v^2 f''(x - vt)$$

$$\text{ie: } \frac{\partial^2 \Psi}{\partial t^2} = v^2 \frac{\partial^2 \Psi}{\partial x^2} \quad \left( \because \frac{\partial^2 \Psi}{\partial x^2} = f''(x-vt) \right)$$

$$\frac{\partial^2 \Psi}{\partial x^2} = \frac{1}{v^2} \frac{\partial^2 \Psi}{\partial t^2}$$

Its sol. is given by:

$$\Psi = A \sin[k(x-vt)]$$

$k$  = propagation vector / const.

### Differential eq. of 3-dim wave

$$\Psi(x, y, z, t)$$

$$\frac{\partial^2 \Psi}{\partial x^2} + \frac{\partial^2 \Psi}{\partial y^2} + \frac{\partial^2 \Psi}{\partial z^2} = \frac{1}{v^2} \frac{\partial^2 \Psi}{\partial t^2}$$

$$\nabla = \hat{i} \frac{\partial}{\partial x} + \hat{j} \frac{\partial}{\partial y} + \hat{k} \frac{\partial}{\partial z} \rightarrow \text{del}$$

∴ The eqn. becomes:

$$\nabla^2 \Psi = \frac{1}{v^2} \frac{\partial^2 \Psi}{\partial t^2}$$

Its sol. is given by:

$$\Psi = \alpha e^{\pm i(\vec{k} \cdot \vec{r} \pm \omega t)}$$

$$\begin{aligned} \vec{k} &= k_x \hat{i} + k_y \hat{j} \\ &\quad + k_z \hat{k} \\ \vec{r} &= x \hat{i} + y \hat{j} + z \hat{k} \end{aligned}$$

## Wavelength & timeperiod

wavelength is defined as the periodic dist. with space periodicity ( $\frac{2\pi}{k}$ )

$$\lambda = \frac{2\pi}{k}$$

Timeperiod is the periodic time with time periodicity ( $\frac{2\pi}{kv}$ )

$$T = \frac{2\pi}{kv}$$

## Relation b/w $\omega$ , $\theta$ , $\lambda$ , & $V$

$$\omega = \frac{2\pi}{T} \quad ; \quad \frac{1}{T} = \frac{\psi}{\lambda} + \frac{\psi}{\lambda} - \frac{\psi}{\lambda}$$

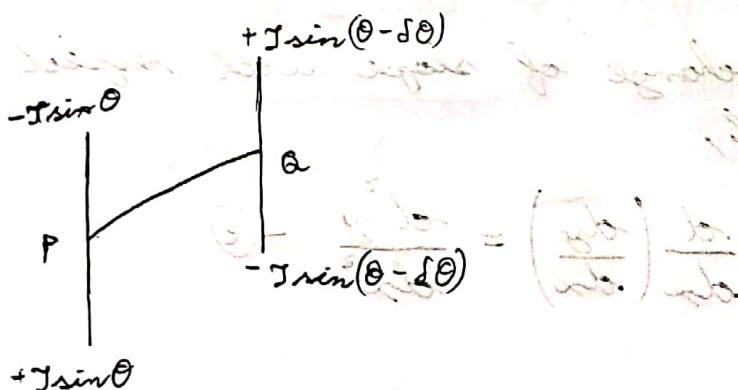
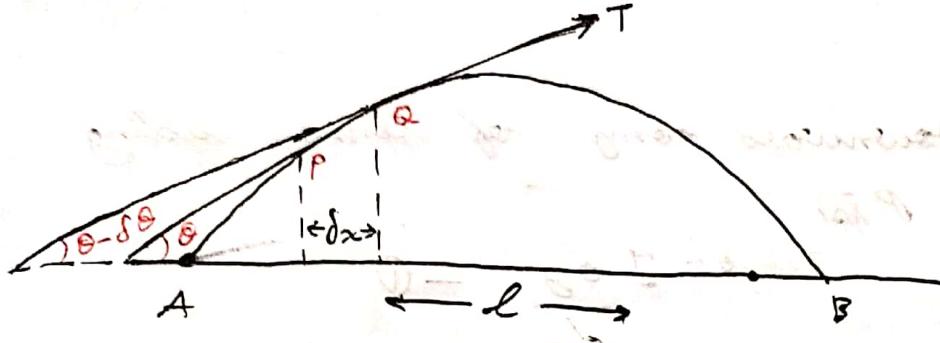
$$kv = \frac{2\pi}{T} \quad ; \quad \frac{1}{T} = \frac{\psi}{\lambda} + \frac{\psi}{\lambda} - \frac{\psi}{\lambda} = V$$

$$= 2\pi V$$

$$\omega = \frac{2\pi V}{\lambda} \quad ; \quad \lambda = \frac{2\pi}{\omega}$$

$$\text{i.e.: } \omega = V\lambda$$

## Transverse vibrations of a stretched string



The downward comp. of tension acting at P.

According to Taylor series expn;

$$\sin x = x - \frac{x^3}{3!} + \frac{x^5}{5!} - \dots$$

$$\tan x = x + \frac{x^3}{3} + \frac{2}{5}x^5 + \dots$$

When  $\theta$  is very small,

$$\sin \theta \approx \tan \theta$$

$$\text{i.e.: } T \sin \theta \approx T \tan \theta$$

slope at P,  $\tan \theta$ :

$$\tan \theta = \frac{dy}{dx}$$

$\therefore$  downward comp. of tension acting

at P is:

$$T \sin \theta = T \frac{dy}{dx} \quad \text{--- (1)}$$

rate of change of slope with respect  
to length;

$$\frac{d}{dx} \left( \frac{dy}{dx} \right) = \frac{d^2 y}{dx^2} \quad \text{--- (2)}$$

~~> Rate of change of slope at a dist.~~  
~~of the segment of string between P & Q is~~

$$\frac{d^2 y(\delta x)}{dx^2} = \frac{d^2 y \times \delta x}{dx^2} \quad \text{--- (3)}$$

Slope of Q  $\rightarrow$  (slope of P - change in slope at Q).

$$\tan(\theta - \delta \theta) = \frac{dy}{dx} - \frac{d^2 y}{dx^2} \delta x \quad \text{--- (4)}$$

Upward comp. of T acting at Q is;

$$T \sin(\theta - \delta \theta) \approx T \tan(\theta - \delta \theta)$$

$$= I \left( \frac{dy}{dx} - \frac{d^2y}{dx^2} \delta x \right) - \textcircled{5}$$

The resultant downward tension acting at PQ is:

$$\cancel{\frac{I dy}{dx}} = \cancel{\left[ I \frac{dy}{dx} \right]}$$

$$I \sin \theta - I \sin(\theta - \delta \theta)$$

$$= I \frac{dy}{dx} - \left[ I \left( \frac{dy}{dx} - \frac{d^2y}{dx^2} \delta x \right) \right]$$

$$= I \frac{d^2y}{dx^2} \delta x - \textcircled{6}$$

Consider  $m'$  is the mass per unit length of the string.

$\Rightarrow$  Mass of the small elem.  $\delta x$  is  $m' \delta x$ .

Force acting on the element.

$$= m' \delta x a$$

$$F = m' \delta x \frac{d^2y}{dt^2} - \textcircled{7}$$

Comparing  $\textcircled{6}$  &  $\textcircled{7}$ , we get;

$$m' \delta x \frac{d^2y}{dt^2} = I \frac{d^2y}{dx^2} \delta x$$

$$\frac{d^2y}{dt^2} = \frac{I}{m} \frac{d^2y}{dx^2} \quad \text{--- (8)}$$

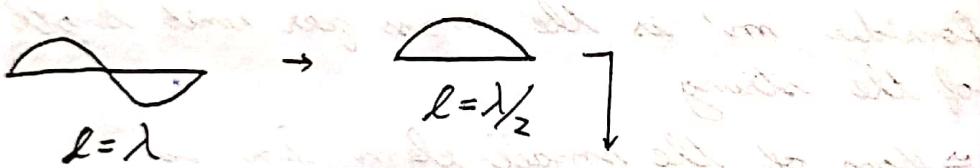
 similar to 1-D wave eqn;

$$\frac{d^2\psi}{dt^2} = v^2 \frac{d^2\psi}{dx^2} \quad \text{--- (9)}$$

Comparing (8) & (9)

$$v^2 = \frac{I}{m} \Rightarrow v = \sqrt{\frac{I}{m}} \rightarrow \text{vel. of propagation}$$

$$V = \frac{v}{\lambda} = \frac{1}{\lambda} \sqrt{\frac{I}{m}}$$



If the string is  
vibrating in one  
segment;

$$l = \lambda/2 \rightarrow 1 \text{ mode}$$

$$\Rightarrow V = \frac{1}{2l} \sqrt{\frac{I}{m}}$$

If no. of modes is given as  $P$ ;

$$V = \frac{P}{2l} \sqrt{\frac{I}{m}}$$

Q. A flexible string of  $l = 0.8 \text{ m}$  &  $M = 2.5 \text{ gm}$  is stretched to produce vibrations. It vibrates in 4 segments with  $\nu = 600 \text{ Hz}$ .

Soln. I.

$$\text{Ans} \quad l = 0.8 \text{ m}$$

$$M = 2.5 \text{ gm}$$

$$m = \frac{2.5}{0.8} = 3.125 \text{ kg/m} \quad \cancel{\text{kg/m}} = \underline{\underline{3.125 \times 10^{-3} \text{ kg/m}}}$$

$$p = 4 \text{ nodes.}$$

$$\nu = 600 \text{ Hz.}$$

$$\nu = \frac{p}{2l} \sqrt{\frac{I}{m}}$$

$$I = \frac{\nu^2 4l^2}{p^2} \times m = \underline{\underline{180 \text{ N}}}$$

Q. A string of violin ~~is~~  $l = 36 \text{ cm}$  &  $M = 0.2 \text{ gm}$ . With what tension it must be stretched to tune  $1000 \text{ Hz}$ .

$$\text{Ans} \quad l = 36 \times 10^{-2} \text{ m}$$

$$M = 0.2 \times 10^{-3} \text{ kg}$$

$$m = \frac{0.2 \times 10^{-3}}{36 \times 10^{-2}} = \frac{2}{36} \times 10^{-2} = \underline{\underline{0.055 \times 10^{-2} \text{ kg/m}}}$$

$$\nu = 1000 \text{ Hz.}$$

$$\text{Ans} \quad \nu = \frac{1}{2l} \sqrt{\frac{I}{m}} \Rightarrow \cancel{\text{Ans}}$$

$$I = \nu^2 4l^2 m = \underline{\underline{288.2 \text{ N}}}$$

a. Calc. freq. of fundamental note produced by string 1 m long & weighing = g m/m.  
stretched by load 400 kg.

$$Ans. \nu = ?$$

$$l = 1 \text{ m}$$

$$M = 2 \times 10^{-3} \text{ kg}$$

$$m = 2 \times 10^{-3} \text{ kg/m}$$

$$g = 400 \text{ kg} \times 9.8 \text{ m/s}^2$$

$$= \underline{\underline{3920 \text{ N}}}$$

$$\nu = \frac{1}{2l} \sqrt{\frac{I}{m}} = \frac{1}{2} \sqrt{\frac{3920}{2 \times 10^{-3}}}$$

$$= \underline{\underline{700 \text{ Hz}}}$$

b. Find  $\nu, T$  &  $\bar{D}$  of  $\lambda = 800 \text{ nm}$ .

~~$\nu = c/\lambda$~~

$$\nu = \frac{c}{\lambda} = \frac{3 \times 10^8}{800 \times 10^{-9}} = \underline{\underline{3750000 \text{ Hz}}}$$

$$= \underline{\underline{3.75 \times 10^{14} \text{ Hz}}}$$

$$\bar{D} = \frac{1}{\lambda} = \frac{1}{800 \times 10^{-9}} = \underline{\underline{1250000 \text{ cm}^{-1}}}$$

$$T = \frac{1}{\nu} = \underline{\underline{2.66 \times 10^{-15} \text{ s}}}$$

a. calc  
of  $\lambda = ?$   
Ans:  $\lambda = \frac{2\pi}{k}$

$$k = \frac{2\pi}{\lambda}$$

$$c = \nu \lambda$$

$$\omega =$$

$$=$$

Q. Calculate propagation const & angular freq.  
of  $\lambda = 700 \text{ nm}$ .

$$\text{Ans} \quad \lambda = \frac{2\pi}{k}$$

$$k = \frac{2\pi}{\lambda} = 8.97 \times 10^{-12}$$

$$c = V\lambda \Rightarrow V = \frac{c}{\lambda} = \frac{3 \times 10^8}{7 \times 10^{-7}} \times 10^{9 \times 10^2} \times \frac{0.42}{\frac{28}{20}}$$
$$= 0.42 \times 10^{15}$$
$$= \underline{\underline{4.2 \times 10^{14} \text{ m/s}}}$$

$$\omega = 2\pi V$$

$$= \underline{\underline{2.63 \times 10^{15}}}$$