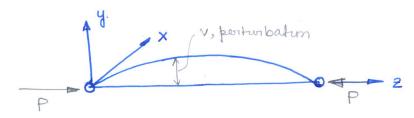
## Euler theory of buckling



### Moment balance equation:

$$\frac{\text{Ef } \frac{d^2v}{dz^2}}{dz^2} = -Pv \implies \frac{\text{Ef } \frac{d^2v}{dz^2}}{dz^2} + Pv = 0$$

$$\implies \frac{d^2v}{dz^2} + \mu^2v = 0 \quad \text{where } \mu^2 = \frac{P}{Ef}$$

$$V(z) = A \ln \mu z + B \sin \mu z$$

$$\Rightarrow \mu l = n\pi \Rightarrow \frac{P}{EI} = \frac{n^2 \pi^2}{L^2}$$

$$\Rightarrow P_{ev} = \frac{n^2 \pi^2 EI}{L^2}$$

$$P_{cr} = \frac{\pi^2 E t}{t^2}$$
 (first buckling)

Per =  $\frac{4\pi^2 E\Gamma}{\ell^2}$  (second buckling)

V(z) = A<sub>1</sub>Sin 
$$\overline{\Lambda}$$
2

A P) Per

bifurcation point

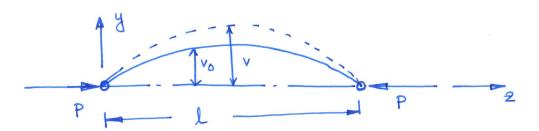
P

$$\nabla_{CY} = \frac{\pi^2 E}{(\ell/r)^2}$$
 ris lu radius of gyration.

For other support condons:

$$P_{cr} = \frac{4\pi^2 E^2}{l^2}$$
 (fixed-fixed suffert)

# Effect of rosition



Moment balance equation:

$$EI \frac{d^2v}{dz^2} - EI \frac{d^2v_0}{dz^2} = -Pv$$

$$\Rightarrow E \int \frac{d^2v}{dz^2} + Pv = E \int \frac{d^2v}{dz^2}$$

$$\Rightarrow \frac{d^2v}{dz^2} + \mu^2v = \frac{d^2v_0}{dz^2} \quad \text{where} \quad \mu^2 = \frac{P}{EL}$$

Substituting (2) in (1), we get,

$$\frac{d^2v}{dz^2} + \mu^2 v = -\sum \frac{Ann^2 \lambda^2}{L^2} \frac{Sin n \lambda^2}{L}$$

=> 
$$V(z) = A (ar \mu z + B Sin \mu z + \sum \frac{\sigma^2 A n}{n^2 - \alpha^2} Sin \frac{\sigma n \lambda z}{L}$$
  $\alpha^2 = \frac{\mu^2 L^2}{\Lambda^2} = \frac{P L^2}{\Lambda^2 E^2}$   
Imposing boundary conditions,  $\omega z = 0$ ,  $v = 0$  and  $\omega z = L$ ,  $v = 0$   $\frac{P}{P_{CT}}$ 

$$A = 0$$
 and  $B = 0$ 

$$\Rightarrow V(2) = \sum_{n=1}^{\infty} \frac{n^2 A n}{n^2 - \alpha^2} \frac{\sin n \pi 2}{0}$$

as P-Per, X->1

.. The first toron in the series corresponding to n=1 will dominate.

 $V_0(z) = \sum_{n=1}^{\infty} A_n S_{n} \frac{n \lambda z}{n}$ 

$$V(2) \simeq \frac{A_1}{1-\alpha^2} \frac{\sin \lambda 2}{\ell}$$
 as  $P \rightarrow P_{cr}$ 

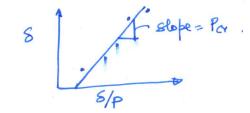
Deflection at the center of the column  $V(z=Y_2) = \frac{A_1}{1-\alpha^2}$ 

Denoting 
$$S = V(2=V_2) - A_1$$

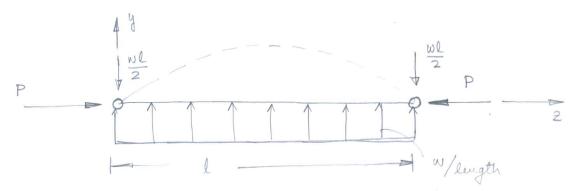
$$= \frac{A_1}{1-P/P_2} - A_1$$

$$= > 8 = \frac{8 P_{cY} - A_1}{P}$$
 (3)

Plotfing eqn (3) when P -> Per, we get Southwell plat



#### Stability of beauns under transverse and axial loads



## Moment balance egn:

$$V(z) = A \ln \mu z + B \sin \mu z + \frac{W}{2P} \left( z^2 - lz - \frac{2}{N^2} \right)$$

Imposing the BCs: @ 2=0, v=0 and @ 2=1, v=0, gives

$$A = \frac{W}{\mu^2 P}$$
 and  $B = \frac{W}{\mu^2 P \sin \mu i} (1 - \cos \mu i)$ 

$$V(z) = \frac{W}{\mu^2 P} \left[ \ln \mu z + \left( \frac{1 - \ln \mu}{\sin \mu} \right) \sin \mu z \right] + \frac{W}{2P} \left( z^2 - 1z - \frac{2}{\mu^2} \right).$$

$$Max^{20} \text{ dollarshim}$$

Maxim deflection

occurs at the mid- point of the beam

$$V_{\text{max}} = \frac{W}{\mu^2 P} \left( \text{Sec} \frac{\mu I}{2} - I \right) - \frac{WL^2}{RP}$$

$$M_{\text{max}} = \frac{W}{H^2} \left( 1 - \text{Sec} \frac{H^1}{2} \right)$$

$$= \frac{W\ell^2}{\Lambda^2} \frac{P_{cv}}{P} \left(1 - 8c_0 \frac{\Lambda}{2} \sqrt{\frac{P}{P_{cv}}}\right) P_{cv} = \frac{\Lambda^2 E \ell}{\ell^2}$$