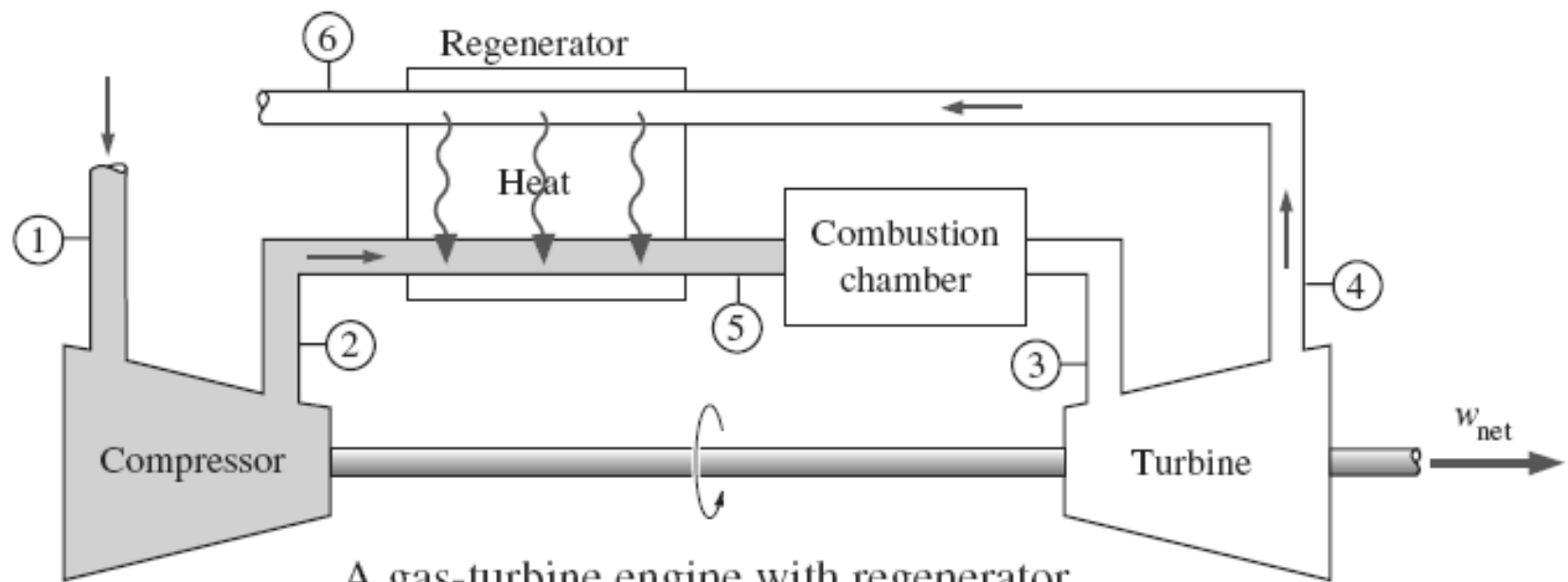


- Recap: Lecture 29, 25th March 2014, 0830-0930 hrs.
  - Stirling cycle
  - Operation of a Stirling cycle
  - Ericsson cycle
  - Stirling, Ericsson and Carnot cycles
  - Brayton cycle
  - Brayton cycle performance with pressure ratio and  $T_{\max}$
  - Performance improvement of the Brayton cycle

- Applications of gas turbine engines
  - Aircraft engines
  - Marine engines
  - Power generation
  - Racing cars
- Improve gas turbine efficiency
  - Increasing turbine inlet temperature
  - Limitation: materials that withstand high temperatures
  - Increasing efficiency of turbomachinery and other components
  - Adding modifications to the basic cycle
  - Regeneration, intercooling etc.

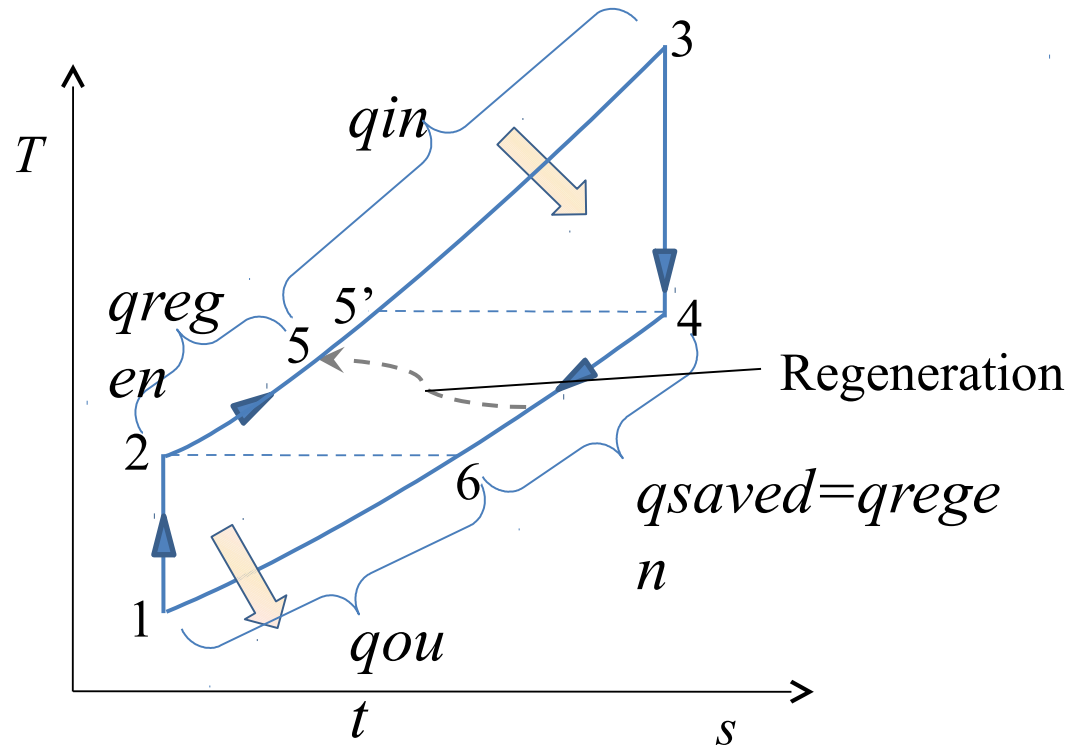
# Brayton cycle with regeneration

- Regeneration can be carried out by using the hot air exhausting from the turbine to heat up the compressor exit flow.
- The thermal efficiency of the Brayton cycle increases as a part of the heat rejected is re-used.
- Regeneration decreases the heat input (thus fuel) requirements for the same net work output.
- Regeneration is also sometimes referred to as recuperation.



A gas-turbine engine with regenerator.

# Brayton cycle with regeneration



T-s diagram of a Brayton cycle with regeneration

# Brayton cycle with regeneration

- The highest temperature occurring within the regenerator is  $T_4$ .
- Air normally leaves the regenerator at a lower temperature,  $T_5$ .
- In the limiting (ideal) case, the air exits the regenerator at the inlet temperature of the exhaust gases  $T_4$ .
- The actual and maximum heat transfers are:

$$q_{\text{regen},\text{act}} = h_5 - h_2 \quad \text{and} \quad q_{\text{regen},\text{max}} = h_{5'} - h_2 = h_4 - h_2$$

# Brayton cycle with regeneration

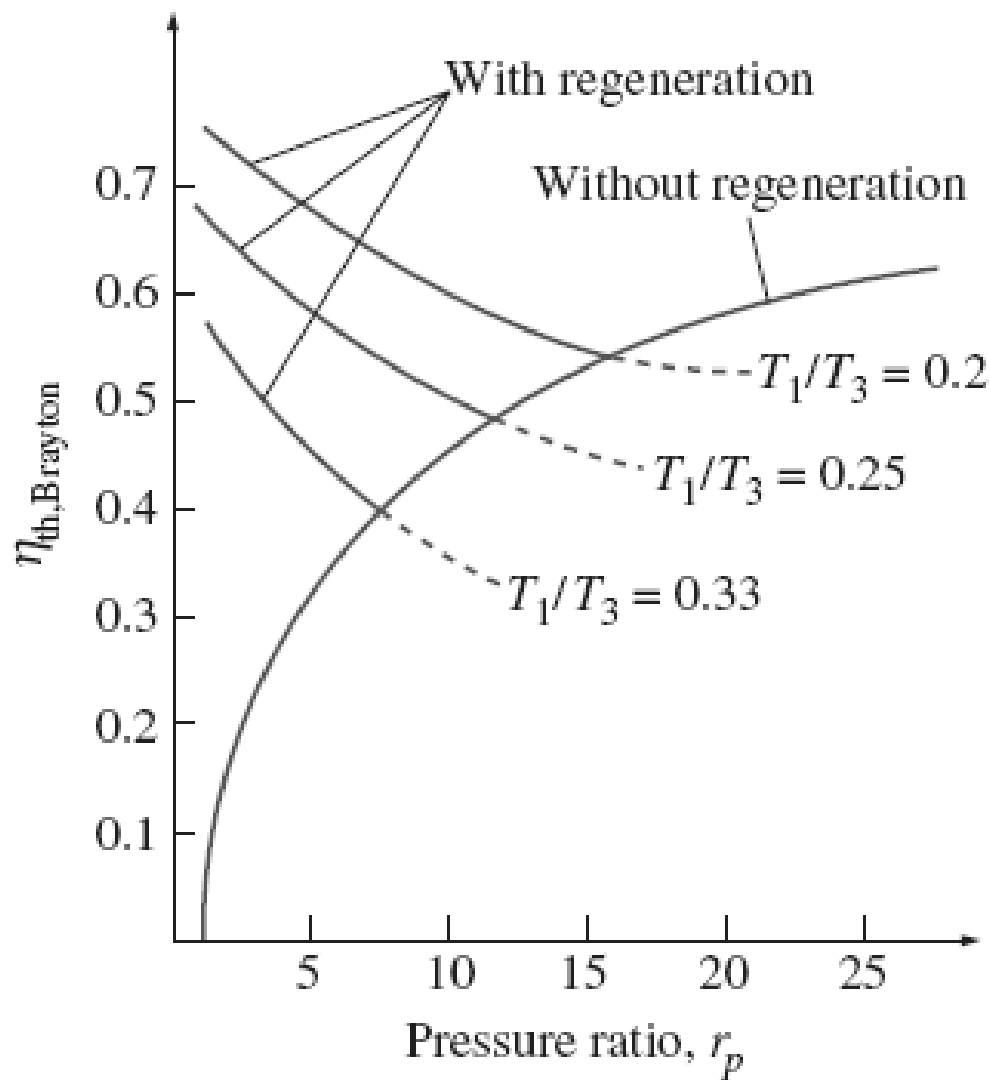
- The extent to which a regenerator approaches an ideal regenerator is called the **effectiveness,  $\varepsilon$**  and is defined as

$$\varepsilon = q_{regen,act} / q_{regen,max} = (h_5 - h_2)/(h_4 - h_2)$$

- Under the cold-air-standard assumptions, the thermal efficiency of an ideal Brayton cycle with regeneration is:

$$\eta_{th,regen} = 1 - \left( \frac{T_1}{T_3} \right) (r_p)^{(\gamma-1)/\gamma}$$

- The thermal efficiency depends upon the temperature as well as the pressure ratio.



Thermal efficiency of the ideal  
Brayton cycle with and without  
regeneration.



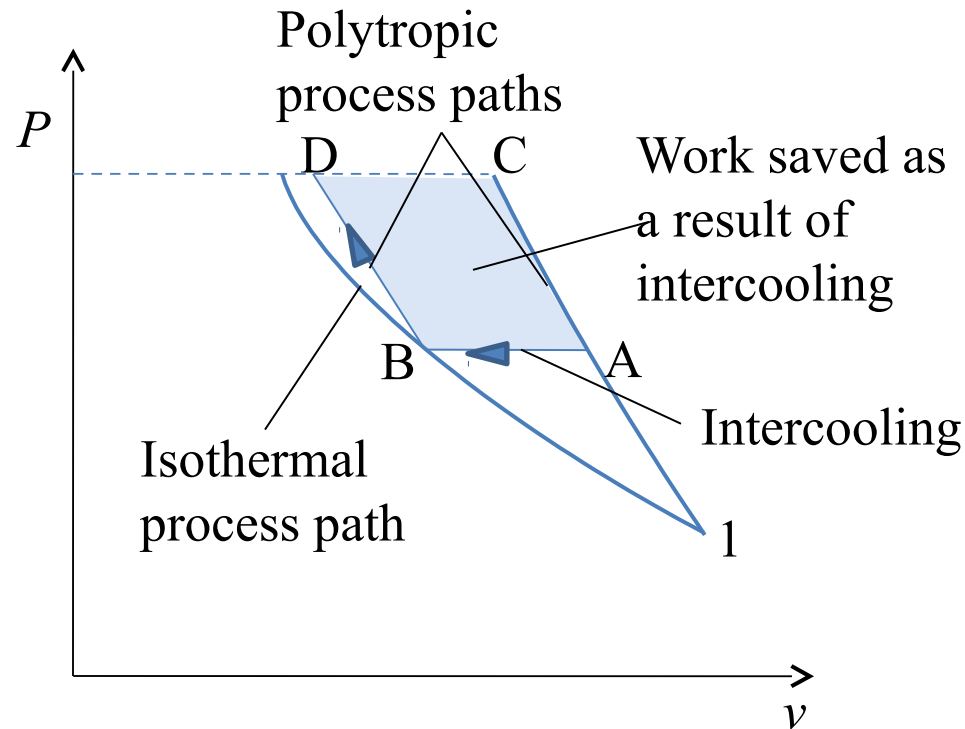
# Brayton cycle with intercooling, reheating and regeneration

- The net work of a gas-turbine cycle is the difference between the turbine work output and the compressor work input.
- It can be increased by either decreasing the compressor work or increasing the turbine work, or both.
- The work required to compress a gas between two specified pressures can be decreased by carrying out the compression process in stages and cooling the gas in between: multi-stage compression with intercooling.

# Brayton cycle with intercooling, reheating and regeneration

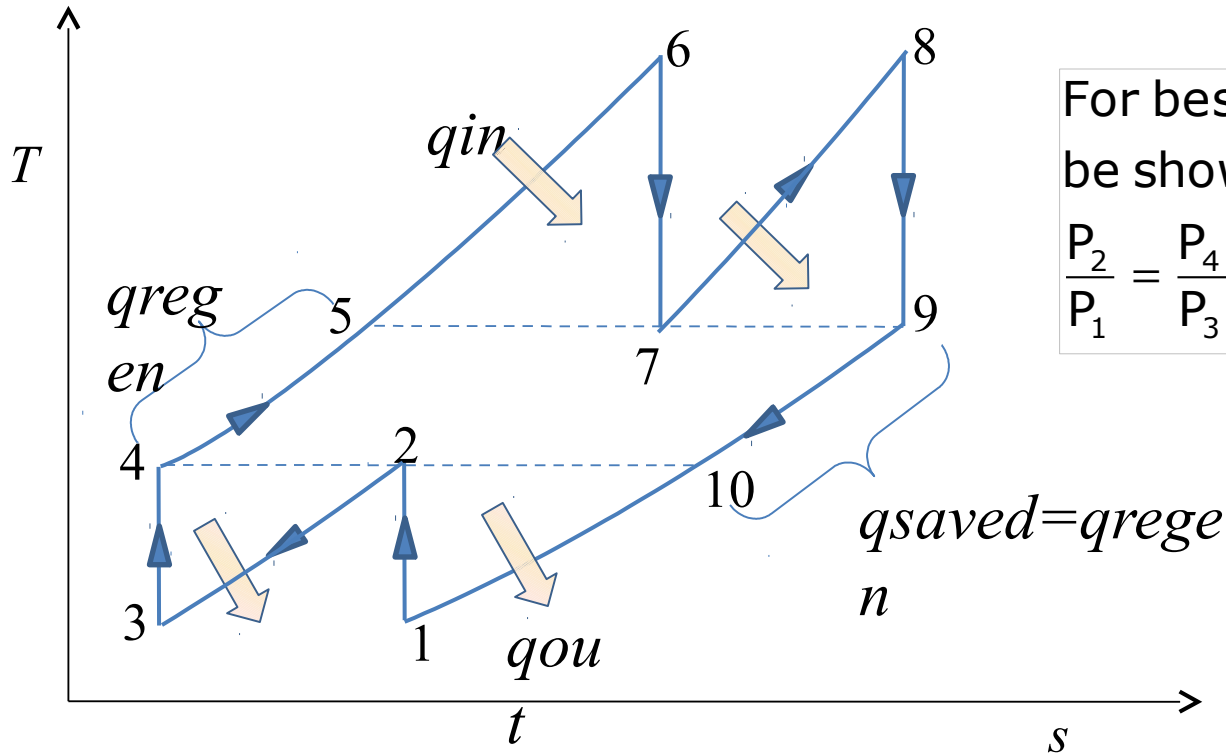
- Similarly the work output of a turbine can be increased by: **multi-stage expansion with reheating**.
- As the number of stages of compression and expansion are increased, the process approaches an isothermal process.
- A combination of intercooling and reheating can increase the net work output of a Brayton cycle significantly.

# Brayton cycle with intercooling, reheating and regeneration



Work inputs to a single-stage compressor (process: 1AC) and a two-stage compressor with intercooling (process: 1ABD).

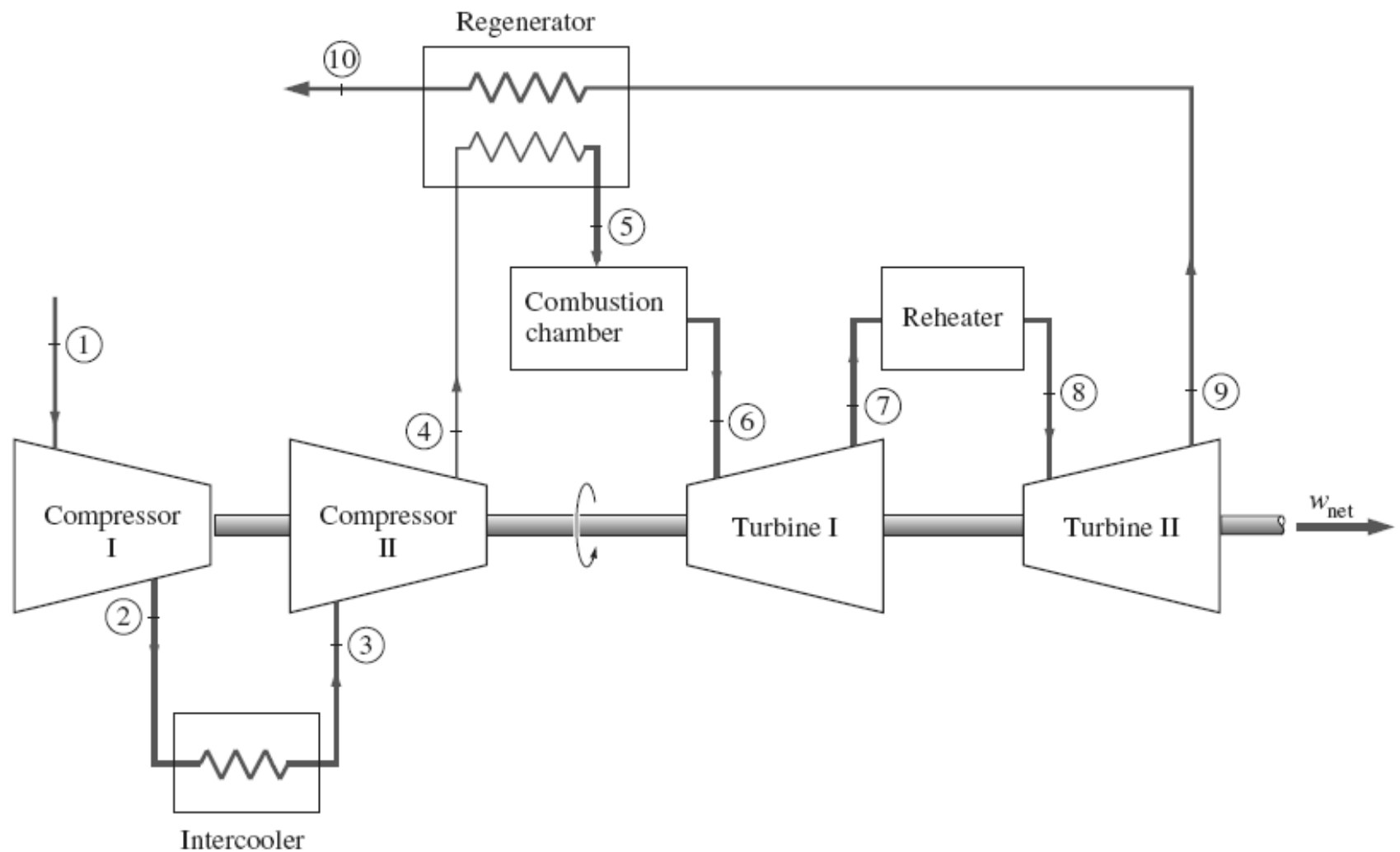
# Brayton cycle with intercooling, reheating and regeneration



For best performance, it can be shown that

$$\frac{P_2}{P_1} = \frac{P_4}{P_3} \text{ and } \frac{P_6}{P_7} = \frac{P_8}{P_9}$$

# T-s diagram of an ideal gas-turbine cycle with intercooling, reheating, and regeneration

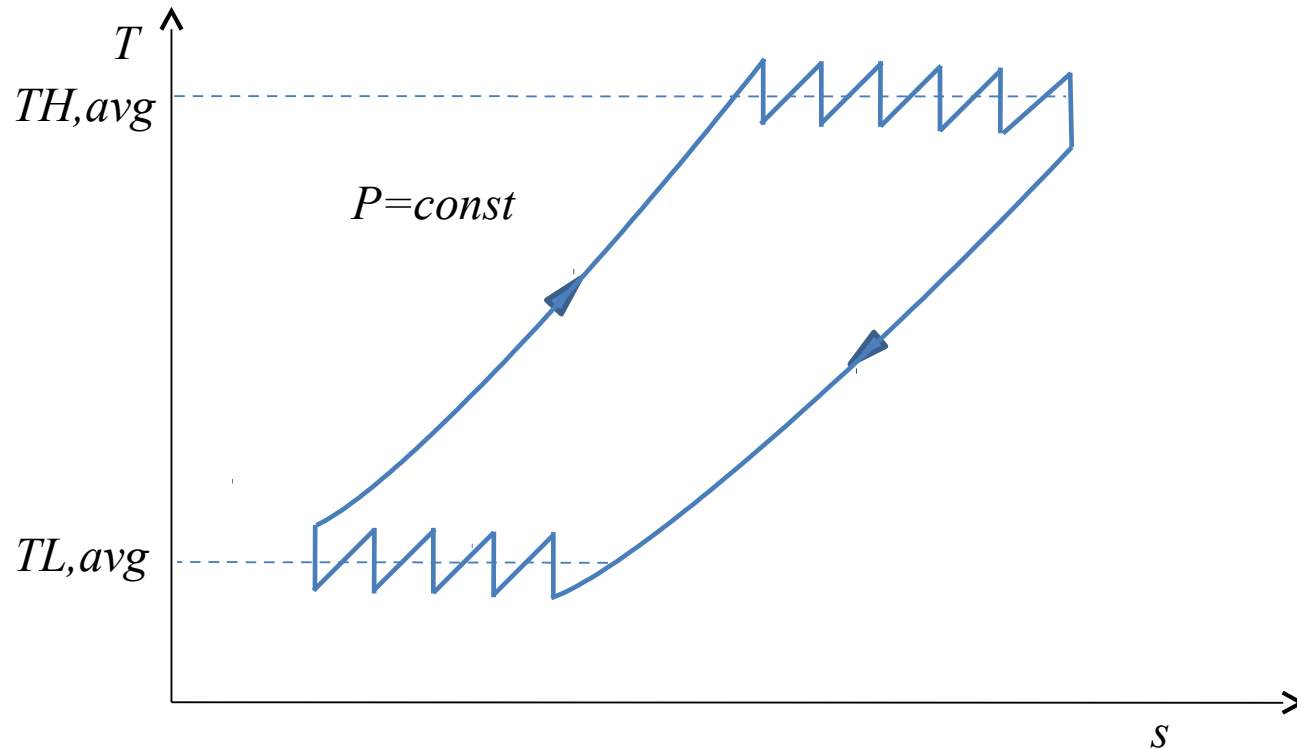


A gas-turbine engine with two-stage compression with intercooling, two-stage expansion with reheating, and regeneration.

# **Brayton cycle with intercooling, reheating and regeneration**

- The net work output of a gas-turbine cycle improves as a result of intercooling and reheating.
- However, intercooling and reheating reduces the thermal efficiency unless they are accompanied by regeneration.
- This is because
  - intercooling reduces the average temperature at which heat is added,
  - reheating increases the average temperature at which heat is rejected.

# Brayton cycle with intercooling, reheating and regeneration

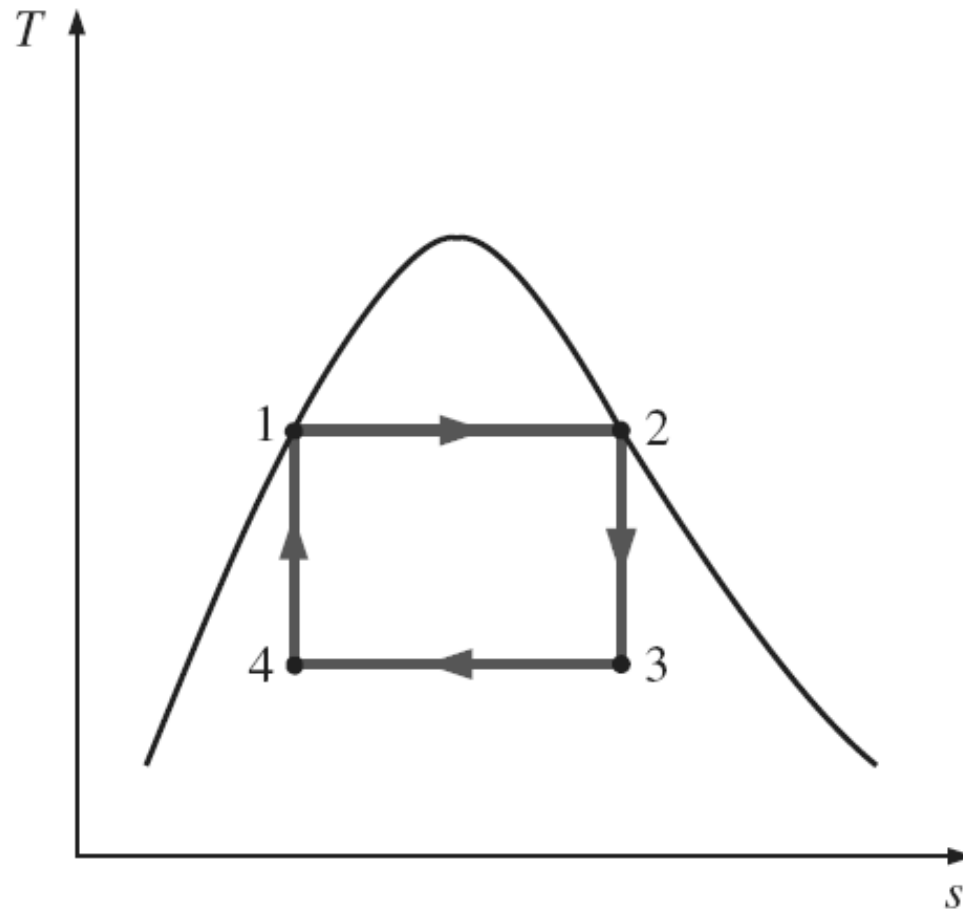


As the number of compression and expansion stages increases, the Brayton cycle with intercooling, reheating, and regeneration approaches the **Ericsson cycle**.

# Carnot Vapour Cycle

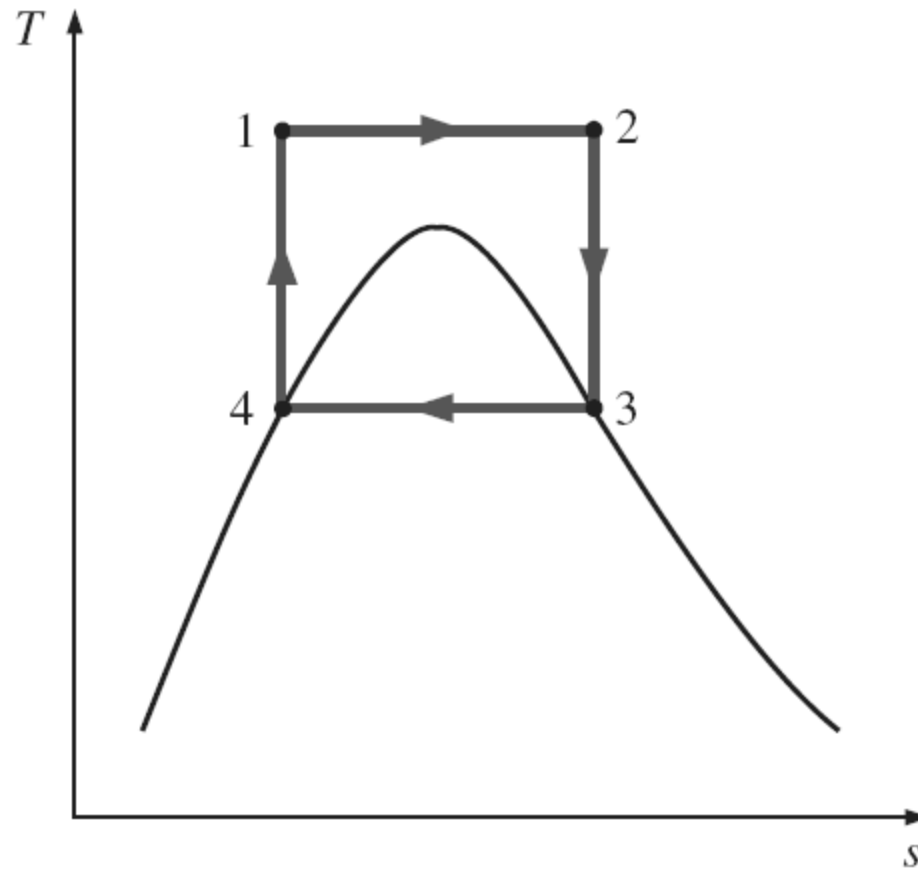
- Carnot cycle is the most efficient cycle between given temperature limits.
- Let us extend a Carnot cycle for a vapour cycle with steam as the working fluid.
- Though theoretically such a cycle seems feasible, there are severe practical limitations.
- Carnot vapour cycle needs to have the following processes
  - Isothermal heat addition
  - Isentropic expansion
  - Isothermal heat rejection
  - Isentropic compression





Case 1: Carnot cycle within the saturation dome

- Two-phase heat transfer, severely limits the maximum temperature that can be used in the cycle (it has to remain under the critical-point value) □ limits max efficiency.
- Raising the maximum temperature in the cycle involves heat transfer to the working fluid in a single phase, which is not easy to accomplish isothermally.
- Isentropic expansion: Leads to very low quality steam □ turbine operation is severely affected.
- Isentropic compression: Control the condensation process during 4-1, Compressor to handle two-phase flow.



Case 2: Carnot cycle partially outside the saturation dome

- Isentropic compression to extremely high pressures □ difficult to achieve
- Not possible to have isothermal heat transfer at variable pressures.
- Therefore Carnot cycle cannot form a realistic model for vapour power cycles.
- The ideal cycle for vapour is the Rankine cycle.