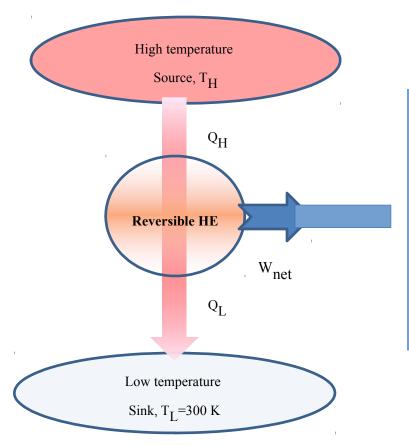
- Recap: Lecture 12: 6<sup>th</sup> Feb 2014, 0930-1030 hrs.
  - Carnot principles
  - Proof of the Carnot principles
  - Thermodynamic temperature scale

# **Quality of energy**



T <sub>H</sub> , K	η <sub>th</sub> , %
1000	70
700	57.1
500	40
400	25
350	14.3

## **Quality of energy**

- Energy has quality as well as quantity.
- More of the high-temperature thermal energy can be converted to work.
- The higher the temperature, the higher the quality of the energy.
- Work is a high quality form of energy than heat since 100 percent of work can be converted to heat, but only a fraction of heat can be converted to work.

#### Carnot refrigerator and heat pump

- Operates on a reversed Carnot cycle.
- The coefficients of performance are:

$$COP_{R} = \frac{1}{Q_{H}/Q_{L}-1}$$
  $COP_{HP} = \frac{1}{1-Q_{L}/Q_{H}}$   
 $or, COP_{R} = \frac{1}{T_{H}/T_{L}-1}$   $COP_{HP} = \frac{1}{1-T_{L}/T_{H}}$ 

• These are the highest coefficients of performance that a refrigerator or a heat pump operating between the temperature limits of  $T_L$  and  $T_H$  can have.

#### Carnot refrigerator and heat pump

$$COP_{R/HP} \begin{cases} < COP_{R/HP, reversible} & Irreversible \\ = COP_{R/HP, reversible} & Reversible \\ > COP_{R/HP, reversible} & Impossible \end{cases}$$

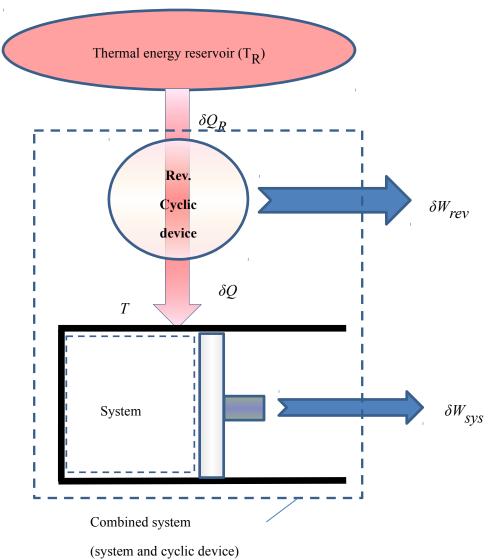
# From thermodynamic temperature scale

• Lord Kelvin proposed thermodynamic scale as

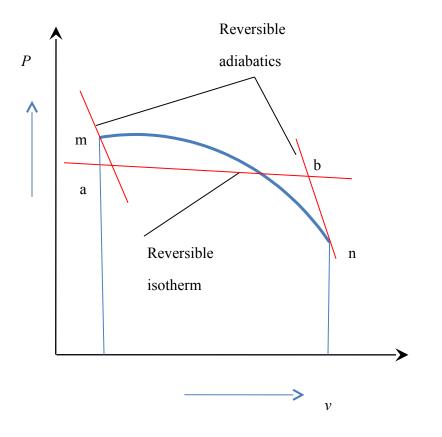
to defi 
$$\phi(T) = T$$

$$\left(\frac{Q_H}{Q_L}\right)_{rev} = \frac{T_H}{T_L}$$

$$\frac{Q_H}{T_H} = \frac{Q_L}{T_L}$$



#### **Reversible adiabatics**



Process m-n

$$Q_{m-n} = U_n - U_m + W_{mn}$$

Process m-a-b-n

$$Q_{m-a-b-n} = U_n - U_m + W_{m-a-b-n}$$

Since, 
$$W_{m-a-b-n} = W_{mn}$$

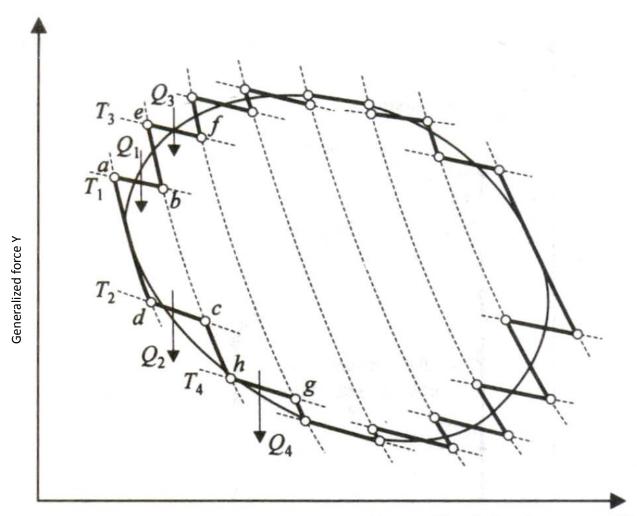
$$Q_{m-n} = Q_{m-a-b-n}$$

$$= Q_{m-a} + Q_{a-b} + Q_{b-n}$$

Since 
$$Q_{m-a} = 0$$
,  $Q_{b-n} = 0$ 

$$Q_{m-n} = Q_{a-b}$$

Reversible path can be substituted by two reversible adiabatics and a reversible isotherm



Generalized displacement X

$$\frac{|Q_2|}{T_2} = \frac{|Q_1|}{T_1}$$

$$\frac{Q_1}{T_1} + \frac{Q_2}{T_2} = 0$$
or
$$\frac{Q_1}{T_1} + \frac{Q_2}{T_2} + \frac{Q_3}{T_3} + \frac{Q_4}{T_4} + \dots = 0$$

$$\sum_{j} \frac{Q_j}{T_j} = 0$$

- Applying the energy balance to the combined system identified by dashed lines yields:  $\delta W_C = \delta Q_R dE_C$
- where  $\delta W_C$  is the total work of the combined system ( $\delta W_{rev} + \delta W_{sys}$ ) and  $dE_C$  is the change in the total energy of the combined system.
- Considering that the cyclic device is a reversible one

$$\frac{\delta Q_R}{T_R} = \frac{\delta Q}{T}$$

From the above equations:

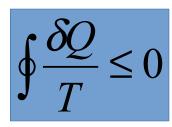
$$\delta W_C = T_R \frac{\delta Q}{T} - dE_C$$

Let the system undergo a cycle while the cyclic device undergoes an integral number of cycles

Since the cyclic integral of ene 
$$W_C = T_R \oint \frac{\delta Q}{T}$$

- The combined system is exchanging heat with a single thermal energy reservoir while involving (producing or consuming) work  $W_{\mathbb{C}}$  during a cycle. Hence  $W_{\mathbb{C}}$  cannot be a work output, and thus it cannot be a positive quantity.
- Considering T<sub>R</sub> to be a positive quantity,

• This is the Clausius inequality.



- Clausius inequality is valid for all thermodynamic cycles, reversible or irreversible, including the refrigeration cycles.
- If no irreversibilities occur within the system as well as the reversible cyclic device, then the cycle undergone by the combined system is internally reversible.

$$\oint \left(\frac{\delta Q}{T}\right)_{\text{int.rev}} = 0$$

• Clausius inequality provides the criterion for the irreversibility of a process.

$$\oint \frac{\delta Q}{T} = 0, \text{ the process is reversible.}$$

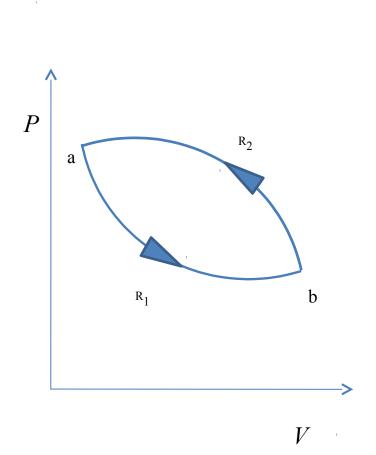
$$\oint \frac{\delta Q}{T} < 0, \text{ the process is irreversible and possible.}$$

$$\oint \frac{\delta Q}{T} > 0, \text{ the process is impossible.}$$

#### Clausius inequality and entropy

- The cyclic integral of work and heat are not zero.
- However, the cyclic integral of volume (or any other property) is zero.
- Conversely, a quantity whose cyclic integral is zero depends on the state only and not the process path, and thus it is a property
- Clausius realized in 1865 that he had discovered a new thermodynamic property, and he chose to name this property entropy.

# The property of entropy



$$\oint_{R_1 R_2} \frac{dQ}{T} = 0$$

$$\int_a^b \frac{\delta Q}{T} + \int_b^a \frac{\delta Q}{T} = 0$$

$$\text{or, } \int_a^b \frac{\delta Q}{T} = -\int_b^a \frac{\delta Q}{T}$$
Since  $R_2$  is a reversible path,
$$\int_a^b \frac{\delta Q}{T} = \int_T^b \frac{\delta Q}{T}$$

# The property of entropy

- $\int_{R} \frac{\partial Q}{T}$  idependent of the reversible path connecting a and b.
- This property whose value at the final state minus the initial state is equal to is called entropy, denoted by *S*.

• When the two equilibrium state  $\int_{a}^{b} \frac{\delta Q}{T} = S_b - S_a$   $\frac{\delta Q_R}{T} = dS$ 

# **Entropy**

- Entropy is an extensive property of a system and sometimes is referred to as total entropy. Entropy per unit mass, designated *s*, is an intensive property and has the unit kJ/kg·K
- The entropy change of a system during a process can be determined by

$$\Delta S = S_2 - S_1 = \int_{1}^{2} \left( \frac{\delta Q}{T} \right)_{\text{int. rev.}} \text{ (kJ/kg)}$$

# **Entropy**

• Entropy is a property, and like all other properties, it has fixed values at fixed states.

Therefore, the entropy change dS between two specified states is the same no matter what path, reversible or irreversible.

# **Temperature-entropy plot**

$$dS = \frac{\delta Q_{rev}}{T}$$

If the process is reversible and adiabatic,  $\delta Q_{rev} = 0$ 

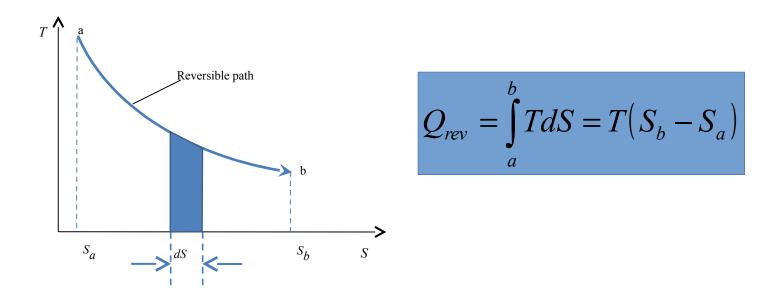
$$\therefore dS = 0$$
 or  $S = \text{constant}$ 

• A reversible adiabatic process is, therefore, and isentropic process.

$$\delta Q_{rev} = TdS$$

$$or, Q_{rev} = \int TdS$$

# **Temperature-entropy plot**



• The area under the reversible path on the T-S plot represents heat transfer during that process.

#### **Isentropic processes**

- A process where,  $\Delta s=0$
- An isentropic process can serve as an appropriate model for actual processes.
- Isentropic processes enable us to define efficiencies for processes to compare the actual performance of these devices to the performance under idealized conditions.
- A reversible adiabatic process is necessarily isentropic, but an isentropic process is not necessarily a reversible adiabatic process.