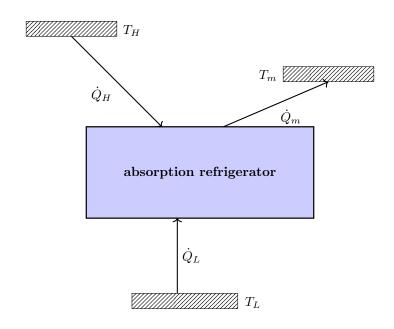
CHEG231 Homework 7

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1.



a

Let the system be the absorption refrigerator. Assume it operates at steady—state and is reversible. the energy balance is

$$0 = Q_H + Q_L - Q_m$$

and the entropy balance is

$$0 = \frac{Q_H}{T_H} + \frac{Q_L}{T_L} - \frac{Q_m}{T_m}$$

now rearranging the energy balance

$$Q_m = Q_H + Q_L$$

and slapping that into the entropy balance

$$0 = \frac{Q_H}{T_H} + \frac{Q_L}{T_L} - \frac{Q_H + Q_L}{T_m} = \frac{Q_H}{T_H} - \frac{Q_H}{T_m} + \frac{Q_L}{T_L} + \frac{Q_L}{T_m}$$
$$Q_H \left(\frac{1}{T_m} - \frac{1}{T_H}\right) = Q_L \left(\frac{1}{T_L} - \frac{1}{T_m}\right)$$
$$\frac{Q_L}{Q_H} = \frac{\left(\frac{1}{T_m} - \frac{1}{T_H}\right)}{\left(\frac{1}{T_L} - \frac{1}{T_m}\right)}$$

b.

First we need to convert the temperatures into Kelvin.

$$T_H = 1023 \,\mathrm{K}$$

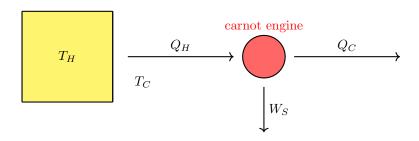
$$T_L = 270 \,\mathrm{K}$$

$$T_m = 300 \,\mathrm{K}$$

And now plugging these into the formula derived above

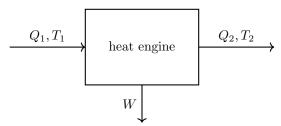
$$\frac{Q_L}{Q_H} = \frac{\left(\frac{1}{300} - \frac{1}{1023}\right)}{\left(\frac{1}{270} - \frac{1}{300}\right)} = \boxed{6.36}$$

2.



Carnot Efficienty Derivation

Consider a closed system of a heat engine where $T_1 > T_2$



energy balance:

$$\frac{dV}{dt} = \dot{Q} + \dot{W}_s + \sum \dot{m}_k()_k$$
$$0 = Q_1 + Q_2 + W$$

entropy balance:

$$\frac{dS'}{dt} = \frac{\dot{Q}_1}{T_1} + \frac{\dot{Q}_2}{T_2} + \sum_{k=1}^{K} \dot{m}_k \hat{S}_k + \dot{S}_{gen}$$

$$0 = \frac{\dot{Q}_1}{T_1} + \frac{\dot{Q}_2}{T_2} + \dot{S}_{gen}$$

integrating, and remembering that reversible processes are more efficient than irreversible processes

$$0 = \frac{Q_1}{T_1} + \frac{Q_2}{T_2}$$

$$\frac{Q_1}{T_1} = \frac{-Q_2}{T_2}$$

$$Q_2 = \frac{-Q_1 T_2}{T_1}$$

and taking this into the energy balance

$$-W_s = Q_1 + \frac{-Q_1 T_2}{T_1} = Q_1 \left(\frac{T_1 - T_2}{T_1}\right)$$
$$\eta = \frac{-W_s}{Q_1} = \left(\frac{T_1 - T_2}{T_1}\right)$$

back to the actual problem

Let the system be the block (the yellow thingy). The energy balance in difference form simplified (please don't crucify me for skipping to simplified balances)

$$\Delta U = Q_H$$

$$dU = dQ_H$$

$$\frac{dU}{dT_H} = C_v$$

$$dU = C_v dT_H$$

Now for the carnot engine we know this relationship will hold

$$\frac{-W_S}{Q_H} = 1 - \frac{T_c}{T_H}$$

which, in terms of infinitecimals

$$\frac{-dW_s}{dQ_h} = 1 - \frac{T_c}{T_H}$$
$$-dW_s = dQ_h \left[1 - \frac{T_c}{T_H} \right]$$

and applying the result of the energy balance

$$-dW_s = C_v dT + H \left[1 - \frac{T_c}{T_H} \right]$$

now integrating to sum up all the dQ

$$-W_{s} = \int_{T_{1}}^{T_{2}} C_{v} dT_{H} - \int_{T_{1}}^{T_{2}} C_{v} \frac{T_{c}}{T_{H}} dT_{H}$$
$$-W_{s} = C_{v} (T_{2} - T_{1}) - C_{v} T_{c} \ln \left(\frac{T_{2}}{T_{1}}\right)$$

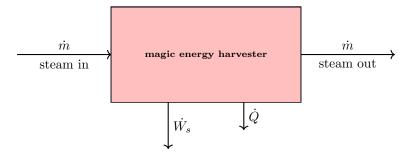
and just turning the absolute heat capacity into a not so absolute heat capacity

$$-W_s = m\hat{C}_v (T_2 - T_1) - m\hat{C}_v T_c \ln\left(\frac{T_2}{T_1}\right)$$

which now may be used to calculate out the two different exergies and compare them

Parameter	Case 1	Case 2
T_H	$500~\mathrm{K}$	$400~\mathrm{K}$
T_C	300 K	300 K
m	100 kg	200 kg
\hat{C}_p	500 J/kg K	500 J/kg K
$-\dot{W}_S$	$2.33~\mathrm{MJ}$	$1.37~\mathrm{MJ}$

The block that starts at 500 K has more exergy.



the system is the carnot engine. this system is at steady state. the mass balance is

$$\frac{dm}{dt} = 0 \to \dot{m}_{in} = \dot{m}_{out} = \dot{m}$$

energy balance

$$0 = \dot{Q} + \dot{W}_s + \sum_{k=1}^{K} \dot{m}_k \hat{H}_k = \dot{Q} + \dot{W}_s + \dot{m} \left(\hat{H}_{in} - \hat{H}_{out} \right)$$

entropy balance

$$0 = \dot{m} \left(\hat{S}_{in} - \hat{S}_{out} \right) + \frac{Q}{T} + \dot{S}_{gen}$$

but a reversible process is more efficient than an irreversible process, so,

$$0 = \dot{m} \left(\hat{S}_{in} - \hat{S}_{out} \right) + \frac{Q}{T}$$

and rearranging the entropy balance and slap—smacking it into the energy balance

$$\dot{Q} = T_{out} \dot{m} \left(\hat{S}_{out} - \hat{S}_{in} \right)$$

$$\dot{m} \left[\left(\hat{H}_{in} - T_{out} \hat{S}_{in} \right) - \left(\hat{H}_{out} - T_{out} \hat{S}_{out} \right) \right] + W_s = 0$$

$$-W_s = \dot{m} \left[\left(\hat{H}_{in} - T_{out} \hat{S}_{in} \right) - \left(\hat{H}_{out} - T_{out} \hat{S}_{out} \right) \right]$$

now going into the depths of the steam tables to look up the enthalpys and entropys

- \hat{H} (200C, 1 bar) = 2875.3
- \hat{H} (25C, 1 bar) = 104.89
- \hat{S} (200C, 1 bar) = 7.8343
- \hat{H} (25C, 1 bar) = 0.3674

also converting the flow rate

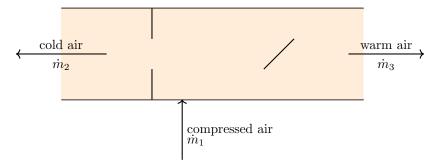
$$3.6\,\frac{\mathrm{tons}}{\mathrm{hr}}\times\frac{1000\,\mathrm{kg}}{1\,\mathrm{ton}}\times\frac{1\,\mathrm{hr}}{3600\,\mathrm{sec}}=1.0\,\mathrm{kg/s}$$

plugging everything into the derived equation

$$-W_s = 1 \left(2875.3 - 298 \times 7.8343 - (104.89 - 298 \times 0.3674)\right) \\ -W_s = 545.3 \, \mathrm{kJ/s}$$

4.

To tell if a process is thermodynamically possible it needs to satisfy all the equations and it needs to have non-negative \dot{S}_{gen} . Let's take a look at this hilsch tube I've been hearing about...



the mass balance looks like

$$0 = \dot{n}_1 - \dot{n}_2 - \dot{n}_3$$

and we know mass in = mass out and that its a 50-50 split

$$\dot{n}_1 = 2\dot{n}_2 = 2\dot{n}_3 = \dot{n}$$

energy balance

$$0 = \dot{n}_1 \underline{H}_1 - \dot{n}_2 \underline{H}_2 - \dot{n}_3 \underline{H}_3$$
$$0 = 2\underline{H}_1 - \underline{H}_2 - \underline{H}_3$$
$$0 = (\underline{H}_1 - \underline{H}_2) + (\underline{H}_1 - \underline{H}_3)$$

and remembering that $\Delta \underline{H} = C_p^* \Delta T$ for constant C_p^*

$$0 = \hat{C}_p (T_1 - T_2) + \hat{C}_p (T_1 - T_3)$$
$$0 = (T_1 - T_2) + (T_1 - T_3)$$
$$T_1 = \frac{1}{2} (T_2 + T_3) = 300 \,\mathrm{K}$$

now we toss this into the entropy balance to see if $\dot{S}_{gen} > 0$

$$0 = \sum n_k \underline{S}_k + \dot{S}_{gen} = n \left(\underline{S}_1 + \frac{1}{2} \underline{S}_2 + \frac{1}{2} \underline{S}_3 \right) + \dot{S}_{gen}$$
$$0 = \frac{1}{2} n \left((\underline{S}_1 - \underline{S}_2) + (\underline{S}_1 - \underline{S}_3) \right) + \dot{S}_{gen}$$

and now using that silly ideal gas adiabatic change in entropy equation derived from the gibbs equation

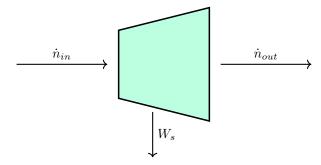
$$\begin{split} -\dot{S}_{gen} &= \frac{1}{2} n \left(\left(C_p \ln \left(\frac{T_2}{T_1} \right) - R \ln \left(\frac{P_2}{P_1} \right) \right) + \left(C_p \ln \left(\frac{T_3}{T_1} \right) - R \ln \left(\frac{P_3}{P_1} \right) \right) \right) \\ \dot{S}_{gen} &= -\frac{1}{2} n \left(\left(35.026 \ln \left(\frac{100}{300} \right) - 8.3145 \ln \left(\frac{1}{3.33} \right) \right) + \left(35.026 \ln \left(\frac{500}{300} \right) - 8.3145 \ln \left(\frac{1}{3.33} \right) \right) \right) \\ \dot{S}_{gen} &= -0.292 n \end{split}$$

since the entropy generation is negative, this process is impossible.

5.

Assume air is a diatomic gas in ideal state s.t. $C_p = \frac{7}{2}R$ and $C_v = \frac{5}{2}R$.

a. adiabatic, reversible turbine



Let the system be the contents of the turbine highlighted in mint green. Mass balance:

$$0 = \dot{n}_{in} - \dot{n}_{out}$$

energy balance

$$0 = -\dot{W}_s + \dot{n}_{in}\hat{H}_{in} - \dot{n}_{out}\hat{H}_{out}$$
$$-W_s = \dot{n}\left(\hat{H}_{out} - \hat{H}_{in}\right)$$

entropy balance

$$0 = \dot{n} \left(\underline{S}_{in} - \underline{S}_{out} \right)$$

 $\underline{S}_{in} = \underline{S}_{out}$

now bringing in that one fun equation derived from the gibbs equation

$$\Delta \underline{S} = 0 = C_p \ln \left(\frac{T_2}{T_1}\right) - R \ln \left(\frac{P_2}{P_1}\right)$$

$$\left(\frac{T_2}{T_1}\right)^{C_p} = \left(\frac{P_2}{P_1}\right)^R$$

$$T_2 = T_1 \left(\frac{P_2}{P_1}\right)^{R/C_p}$$

$$T_2 = 400 \left(\frac{1.01325}{10}\right)^{8.3145/(3.5*8.3145)}$$

$$T_2 = 208 \text{ K}$$

remembering $\Delta H = C_p \Delta T$ for constant C_p

$$-W_s = \dot{n}C_p \left(T_{out} - T_{in}\right)$$

and now because this question is annoyingly ideal i'll use the ideal gas law to find the molar volume

$$\underline{V} = \frac{RT}{P} = \frac{8.3145 \times 400}{10 \times 10^5} \times 1000 \frac{L}{m^2} = 3.33 \frac{L}{mol}$$
$$\dot{n} = \frac{10 L/s}{3.33 L/mol} = 3.01 \,\text{mol/s}$$

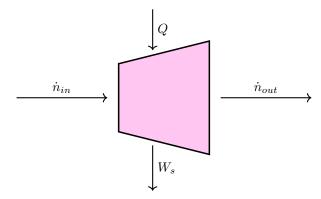
back to the equation from the energy and mass balance

$$-\dot{W}_s = 3.01 \times 29 (400 - 208)$$

 $-\dot{W}_s = 16782 \,\text{kJ/s}$

isentropic!

b. isothermal turbine



mass balance

$$\dot{m}_{in} = \dot{m}_{out} = \dot{m}$$

energy balance

$$0 = W_s + Q - m\left(\hat{H}_{out} - \hat{H}_{in}\right)$$

however, we know that for an ideal gas the enthalpy is only a function of temperature so $\Delta H = 0$

$$-W_s = Q$$

now looking at the entropy balance differential flavor

$$0 = \frac{\dot{Q}}{T} + \dot{n} \left(\underline{S}_{in} - \underline{S}_{out} \right) + \dot{S}_{gen}$$

since the problem statement said 100% efficient, no entropy is generated

$$0 = \frac{\dot{Q}}{T} + \dot{n} \left(\underline{S}_{in} - \underline{S}_{out} \right)$$

$$\dot{Q} = T\dot{n}\left(\underline{S}_{out} - \underline{S}_{in}\right)$$

remembering the ΔS equation that comes from the gibbs equation

$$\Delta \underline{S} = C_p \ln \left(\frac{T_2}{T_1}\right) - R \ln \left(\frac{P_2}{P_1}\right)$$

$$= -R \ln \left(\frac{P_2}{P_1}\right)$$

$$= -R \ln \left(\frac{1.01325 \text{ bar}}{10 \text{ bar}}\right)$$

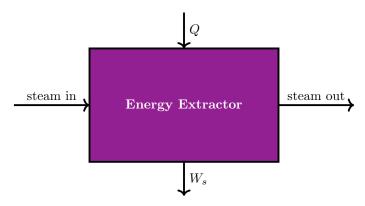
$$= 19.04 \frac{\text{kJ}}{\text{mol} \cdot \text{K}}$$

and going back to the energy balance

$$-W_s = Q = T\dot{n}\Delta S = 400 \times 3 \times 19.04 = 22865 \,\text{kJ/s}$$

c. maximum theoretical

consider the steam going through a magical black box that extracts all the energy from the steam until it is in the dead state.



the system is the contents of the energy extractor. mass balance

$$\frac{dn}{dt} = 0 = \dot{n}_{in} - \dot{n}_{out}$$
$$\dot{n}_{in} = \dot{n}_{out} = \dot{n}$$

energy balance

$$\frac{dU}{dt} = 0 = W_s + Q + \dot{n}_{in}\underline{H}_{in} - \dot{n}_{out}\underline{H}_{out}$$

entropy balance, noting that reversible processes are more efficient than irreversible processes

$$\begin{split} \frac{dS}{dt} &= 0 = \dot{n}_{in} \underline{S}_{in} - \dot{n}_{out} \underline{S}_{out} + \frac{Q}{T_{\text{ambient}}} \\ \dot{Q} &= \dot{n} \left(\underline{S}_{out} - \underline{S}_{in} \right) \end{split}$$

 $0 = W_s + \dot{n} \left(S_{out} - S_{in} \right) + \dot{n} H_{in} - \dot{n}_{out} H_{out}$

now substituting that into the energy balance

$$-W_s = \dot{n} \left[\left(\underline{H}_{in} - T_{\text{ambient}} \underline{S}_{in} \right) - \left(\underline{H}_{out} - T_{\text{ambient}} \underline{S}_{out} \right) \right]$$

$$-W_s = \dot{n} \left[\left(\underline{H}_{in} - \underline{H}_{out} \right) - T_{\text{ambient}} \left(\underline{S}_{in} - \underline{S}_{out} \right) \right]$$

$$-W_s = \dot{n} \left[\left(-C_p^* \Delta T \right) + T_{\text{ambient}} \left(C_p^* \ln \left(\frac{T_2}{T_1} \right) - R \ln \left(\frac{P_2}{P_1} \right) \right) \right]$$

$$-W_s = 3 \left[\left((29)(400 - 298) \right) + 298 \left(29 \ln \left(\frac{298}{400} \right) - 8.3145 \ln \left(\frac{1.013}{10} \right) \right) \right]$$

$$-W_s = 18266 \, \text{kW}$$

both Δ are negative

6.

a.

a.

$$d\underline{U} = \left(\frac{\partial \underline{U}}{\partial \underline{S}}\right)_{V} d\underline{S} + \left(\frac{\partial \underline{U}}{\partial \underline{V}}\right)_{S} d\underline{V}$$

b.

$$d\underline{H} = \left(\frac{\partial \underline{H}}{\partial \underline{S}}\right)_P d\underline{S} + \left(\frac{\partial \underline{H}}{\partial P}\right)_S dP$$

c.

$$d\underline{H} = \left(\frac{\partial \underline{H}}{\partial T}\right)_P dT + \left(\frac{\partial \underline{H}}{\partial P}\right)_T dP$$

b.

restating the fundemental relationship

$$d\underline{U} = Td\underline{S} - Pd\underline{V}$$

and now turning this into a complete/total differential

$$d\underline{U} = \left(\frac{\partial \underline{U}}{\partial \underline{S}}\right)_{V} d\underline{S} - \left(\frac{\partial \underline{U}}{\partial \underline{V}}\right)_{S} d\underline{V}$$

we see this

$$d\underline{U} = T d\underline{S} - P d\underline{V}$$

$$d\underline{U} = \left(\frac{\partial \underline{U}}{\partial \underline{S}}\right)_{V} d\underline{S} - \left(\frac{\partial \underline{U}}{\partial \underline{V}}\right)_{S} d\underline{V}$$

so now just putting this as an equation so it feels all formalized

$$\left(\frac{\partial \underline{U}}{\partial \underline{S}}\right)_{V} = T$$

$$\left(\frac{\partial \underline{U}}{\partial \underline{V}}\right)_{\underline{S}} = -P$$

7.

Starting off by writing the total differential

$$\begin{split} d\underline{H} &= \left(\frac{\partial \underline{H}}{\partial T}\right)_P dT + \left(\frac{\partial \underline{H}}{\partial P}\right)_T dP \\ \frac{\partial}{\partial T}\bigg|_{\underline{H}} d\underline{H} &= \left(\frac{\partial \underline{H}}{\partial T}\right)_{\underline{H}} = \left(\frac{\partial \underline{H}}{\partial T}\right)_P \left(\frac{\partial T}{\partial T}\right)_{\underline{H}} + \left(\frac{\partial \underline{H}}{\partial P}\right)_T \left(\frac{\partial P}{\partial T}\right)_{\underline{H}} \end{split}$$

remembering the rules tells us that the red term is 0 and the green term is 1

$$0 = \left(\frac{\partial \underline{H}}{\partial T}\right)_P + \left(\frac{\partial \underline{H}}{\partial P}\right)_T \left(\frac{\partial P}{\partial T}\right)_{\underline{H}}$$

$$\left(\frac{\partial \underline{H}}{\partial T}\right)_P = -\left(\frac{\partial \underline{H}}{\partial P}\right)_T \left(\frac{\partial P}{\partial T}\right)_{\underline{H}}$$

and now diving over the ones on the right since we can do that thanks to rule 3

$$\left(\frac{\partial \underline{H}}{\partial T}\right)_{P}\left(\frac{\partial P}{\partial \underline{H}}\right)_{T}\left(\frac{\partial T}{\partial P}\right)_{H} = -1$$