



# VEHICLE DESIGN PORTFOLIO

CONCEPTUAL DESIGN AND  
COMPUTATIONAL ANALYSIS  
OF AERODYNAMICS AND  
COOLING SYSTEMS

LARA MERICAN

# DESIGN SUMMARY

## INTRODUCTION

This portfolio presents the **aerodynamic analysis** of a **conceptual, urban MPV/minivan**. This minimalistic design is aimed at optimising aerodynamics by **reducing drag, improving airflow efficiency and minimising energy consumption**.

## KEY CFD OUTCOMES

**0.36**  
DRAG COEFFICIENT

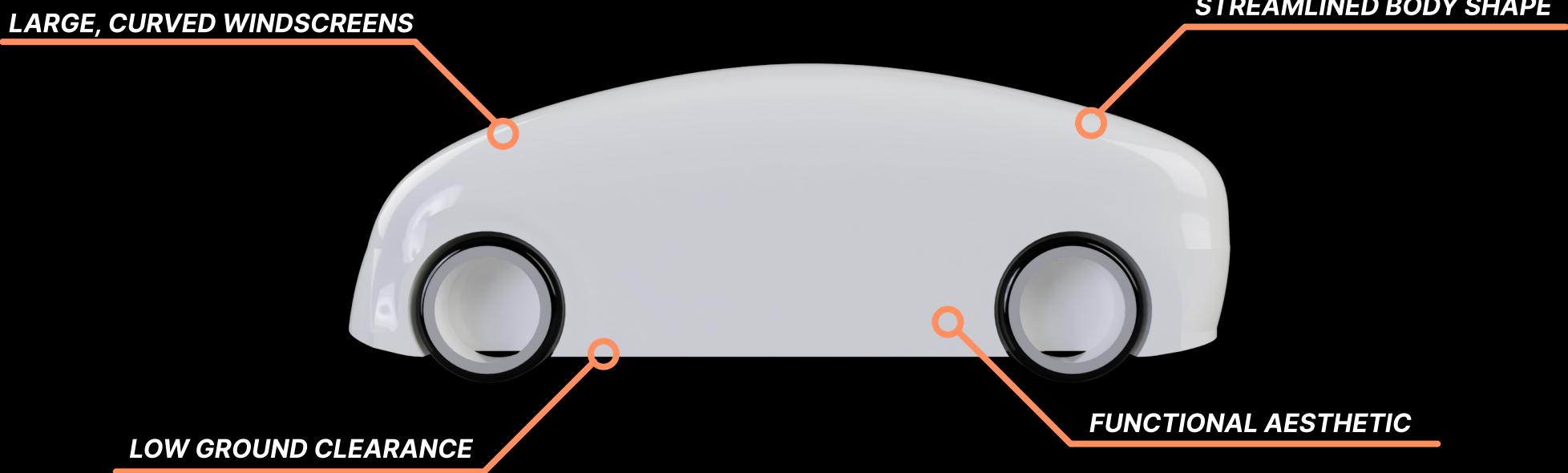
**2.94**  
DRAG FORCE (N)

**0.0285**  
FRONTAL AREA ( $m^2$ )

**0.50**  
LIFT COEFFICIENT

**4.08**  
LIFT FORCE (N)

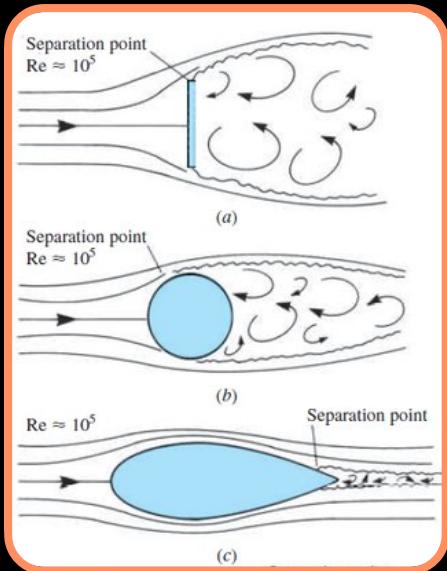
## KEY DESIGN FEATURES



# DESIGN RATIONALE: FEATURES

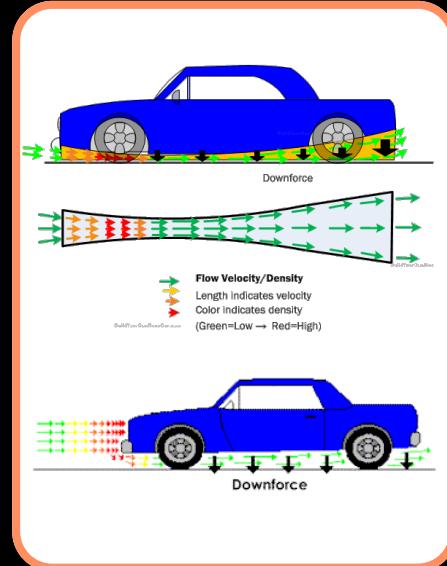
## STREAMLINED BODY

- Features inspired by “**the most aerodynamic shape in the world**” – the teardrop shape
- **Continuous curvature:** Aims to **reduce air resistance** by minimising flow separation
- **Gradual roofline taper:** Aims to **reduce wake turbulence** at the rear, improving aerodynamic efficiency



## GROUND CLEARANCE

- Low ground clearance reduces drag force
- **Reduced air:** Less air underneath the car reduces lift and turbulence
- **Lower centre of gravity:** Improves stability of the car, especially at higher speeds



## PANORAMIC WINDSCREENS

- **Large and curved:** Optimises airflow to move smoothly over the surface
- **Seamless connection:** No gaps between the windscreen and roof to prevent disruptions and promote laminar flow



## FUNCTIONAL AESTHETIC

- **Smooth surfaces:** Reduces energy consumption while aligning with urban electric vehicle trends
- **Spacious form:** A compromise between optimising aerodynamics and allowing for ample interior room



# DESIGN RATIONALE: EXISTING VEHICLES

These vehicles were chosen as **benchmarks** due to their size, type and purpose which results in a **similar aerodynamic performance**. They have **varying dimensions and drag coefficients**, giving valuable insight to optimising aerodynamics for an urban MPV.



**VOLKSWAGEN EUROVAN**

**0.45**

DRAG COEFFICIENT

**2.92**

FRONTAL AREA ( $m^2$ )

**$4.74 \times 1.84 \times 1.90$**

LENGTH X WIDTH X HEIGHT (m)



**MERCEDES BENZ VITO**

**0.34**

DRAG COEFFICIENT

**3.18**

FRONTAL AREA ( $m^2$ )

**$4.90 \times 1.93 \times 1.91$**

LENGTH X WIDTH X HEIGHT (m)



**CHEVROLET VENTURE**

**0.35**

DRAG COEFFICIENT

**2.60**

FRONTAL AREA ( $m^2$ )

**$5.10 \times 1.83 \times 1.73$**

LENGTH X WIDTH X HEIGHT (m)

## KEY TAKEAWAYS

- The Volkswagen Eurovan has the **highest drag coefficient**, probably due to its **boxier shape**.
- The Mercedes Benz Vito has the **lowest drag coefficient**, despite it having the **largest frontal area**. This reflects its streamlined body which contributes to aerodynamics.
- The Chevrolet Venture's **low height** contributes to a **lower frontal area** and hence a lower drag coefficient.

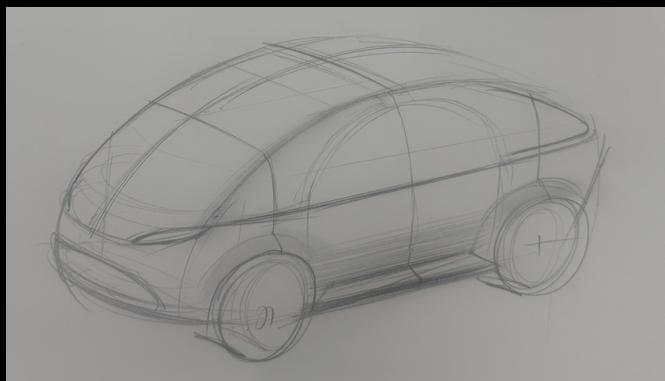
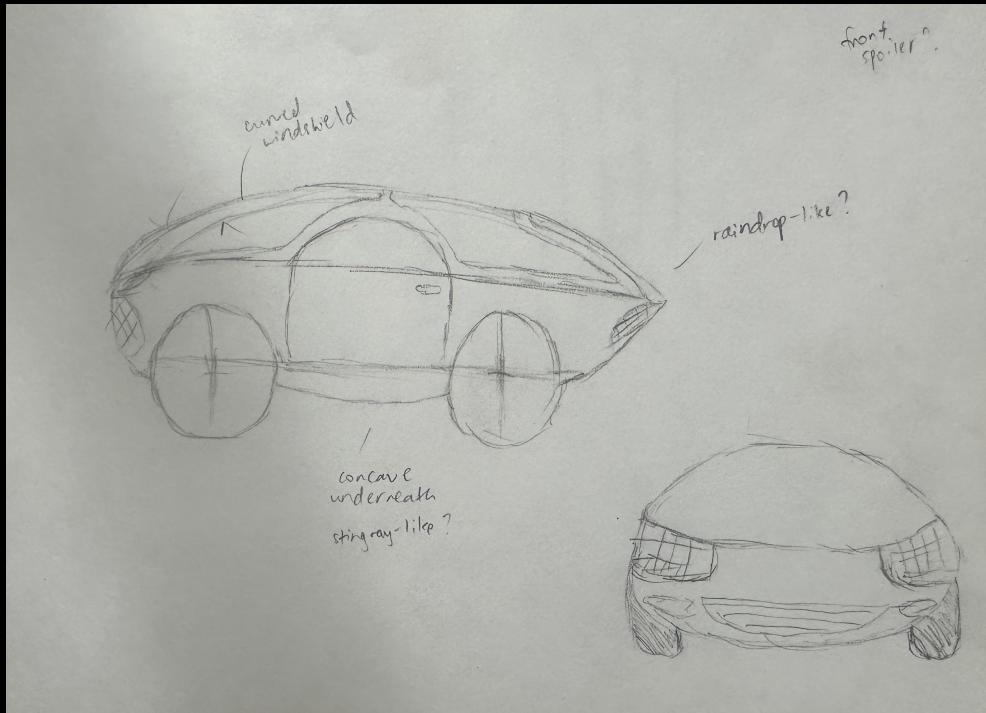
## SPECIFICATION AIMS

These real-world vehicles give an approximate idea of the dimensions and specifications of my design.

- Dimensions:**  $4.92 \times 1.88 \times 1.82$  approx.
- Frontal area:** 2.60 - 3.00
- Coefficient of drag:** Ideally lower than 0.34

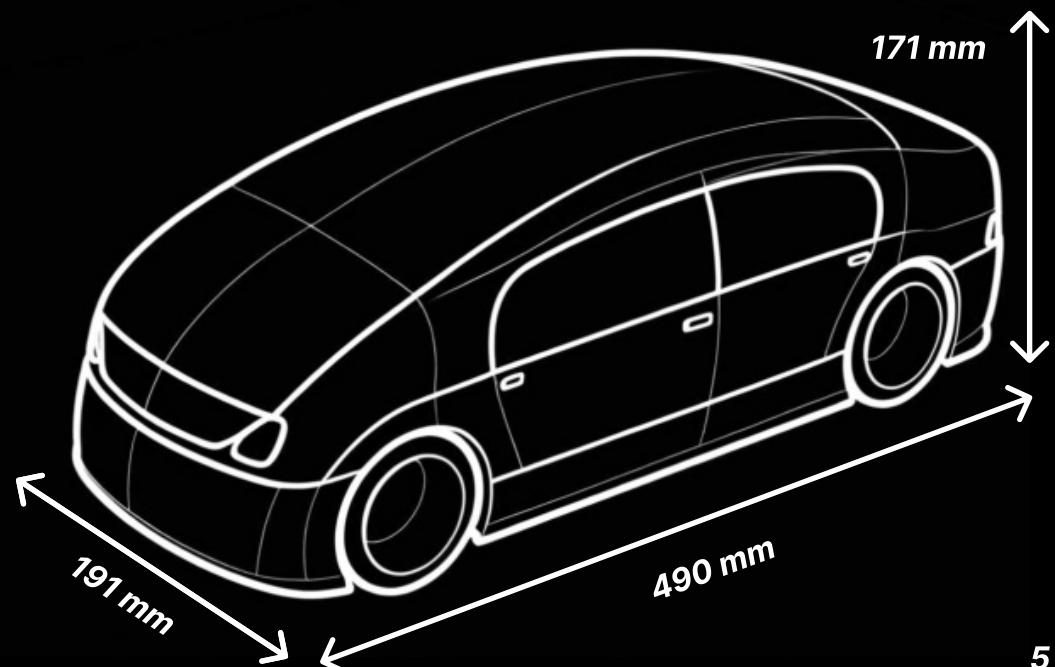
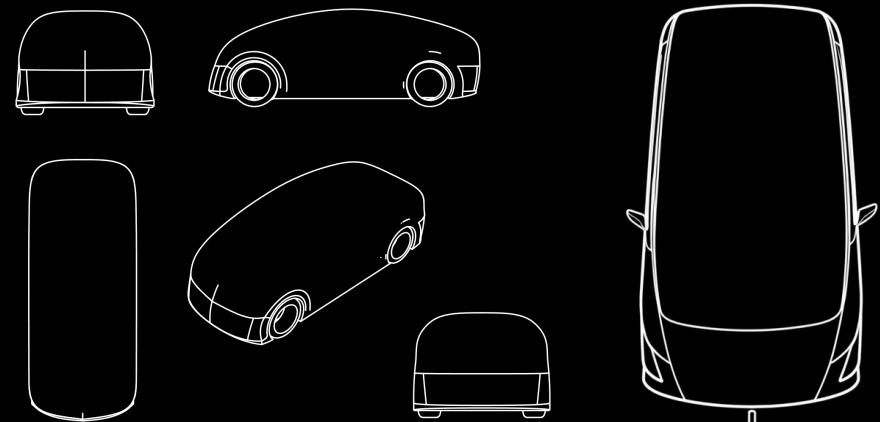
# CONCEPTUAL DESIGN DEVELOPMENT

## INITIAL SKETCHES



Sketched by  
professional  
Automotive &  
Transportation  
Designer, Boris Fabris

## REFINED SKETCHES



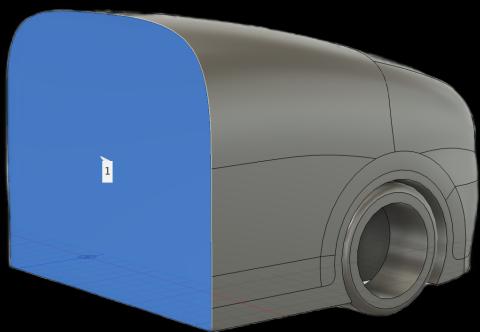
# LOW-FIDELITY 3D CAD MODEL



# FRONTAL AREA MEASUREMENT

## METHODOLOGY

Frontal area describes the exposure area to wind forces at the front of the vehicle. In order to find the frontal area of the car, **cross-sections were made and measured in Fusion 360**.

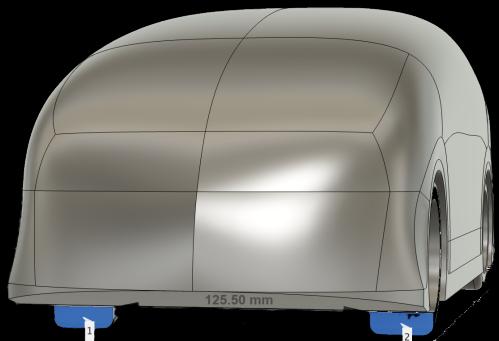


### 1 CAR BODY

- Cross-section was made at the tallest part of the car
- Cross-section area:  $27930 \text{ mm}^2 = 0.02793 \text{ m}^2$

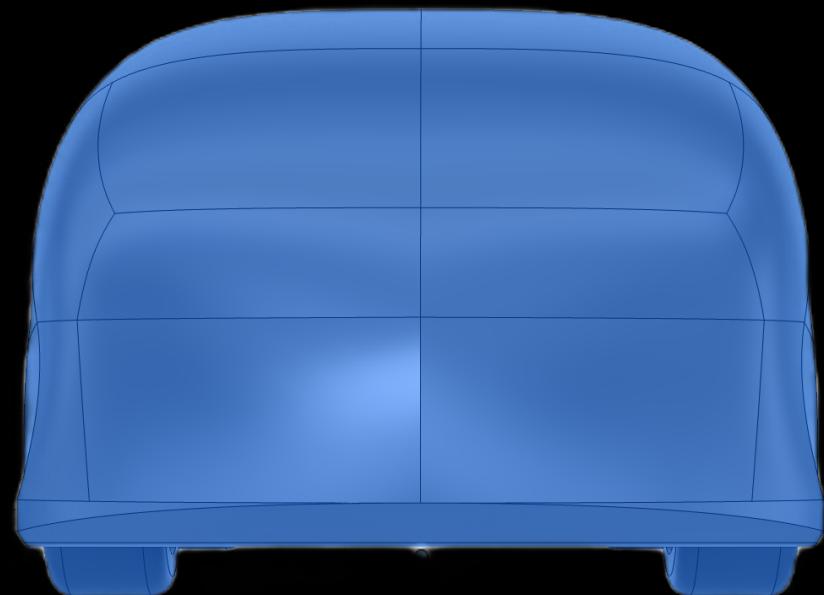
### 2 WHEELS

- Cross-section was made halfway through the wheels
- Cross-section area (1 wheel):  $286.94 \text{ mm}^2 = 2.8694 \times 10^{-4} \text{ m}^2$
- Area (2 wheels):  $2.8694 \times 10^{-4} \times 2 = 5.7388 \times 10^{-4} \text{ m}^2$



## TOTAL FRONTAL AREA

Generally, a **lower frontal area reduces drag**, however it also depends on many other factors. For a family car, there must be a **compromise between spaciousness and frontal area**.



$$0.02793 + 5.7388 \times 10^{-4} = 0.02850388 \text{ m}^2$$

**0.028**  
FRONTAL AREA ( $\text{m}^2$ )  
to 3 significant figures

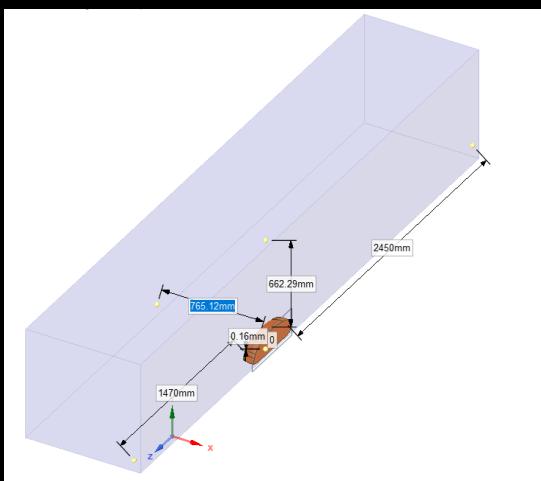
# CFD SETUP

## **COMPUTATIONAL DOMAIN**

The following dimensions were used for the enclosure:

- **Upstream:** 3 times the length of the car ( $490 \times 3 = 1470$  mm)
  - **Downstream:** 5 times the length ( $490 \times 5 = 2450$  mm)
  - **Width:** 4 times the width of the car ( $191.281 \times 4 = 765.12$  mm)

- Using the **standard blockage ratio of 5%**, height of the enclosure can be found by rearranging equation (1)
  - The cross-section of the enclosure can be found by product of the width and height, hence giving equation (2)
  - **Height:** Frontal area, divided by the blockage ratio multiplied by width of the enclosure ( $28500/(765.12 \times 0.05) = \text{mm}$ )

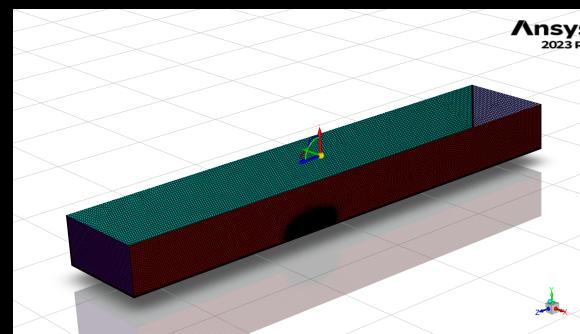


# INITIAL BOUNDARY CONDITIONS

- Enclosure material: Air
  - Car shear condition: No slip
  - Inlet velocity: 30.5 m/s, normal to boundary
  - Outlet gauge pressure: 0
  - Road shear condition: No slip
  - Wall shear condition: Specified shear of 0

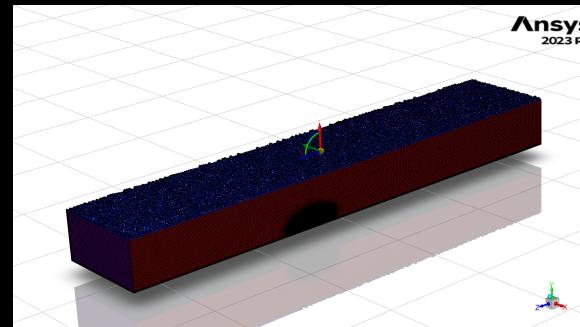
# MESHING

Meshing allows for an accurate simulation by **discretising** the vehicle into well-defined cells.



## 1 SURFACE MESH

- Nodes: 277420
  - Edges: 4577
  - Faces: 545714
  - Cells: 0

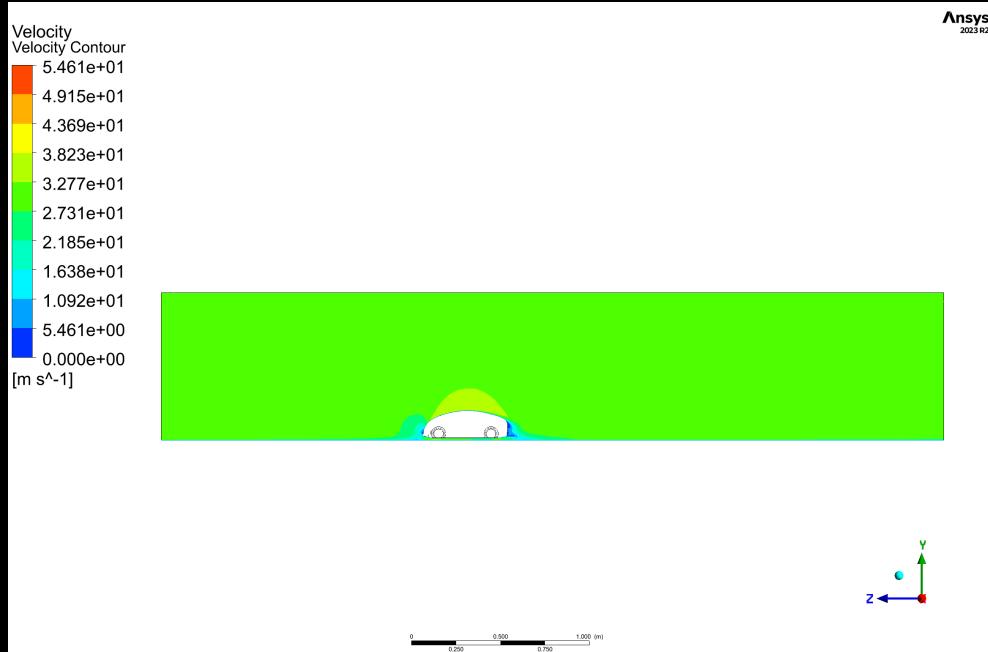


## 2 VOLUME MESH

- Nodes: 9367352
  - Edges: 4577
  - Faces: 11961247
  - Cells: 2091160

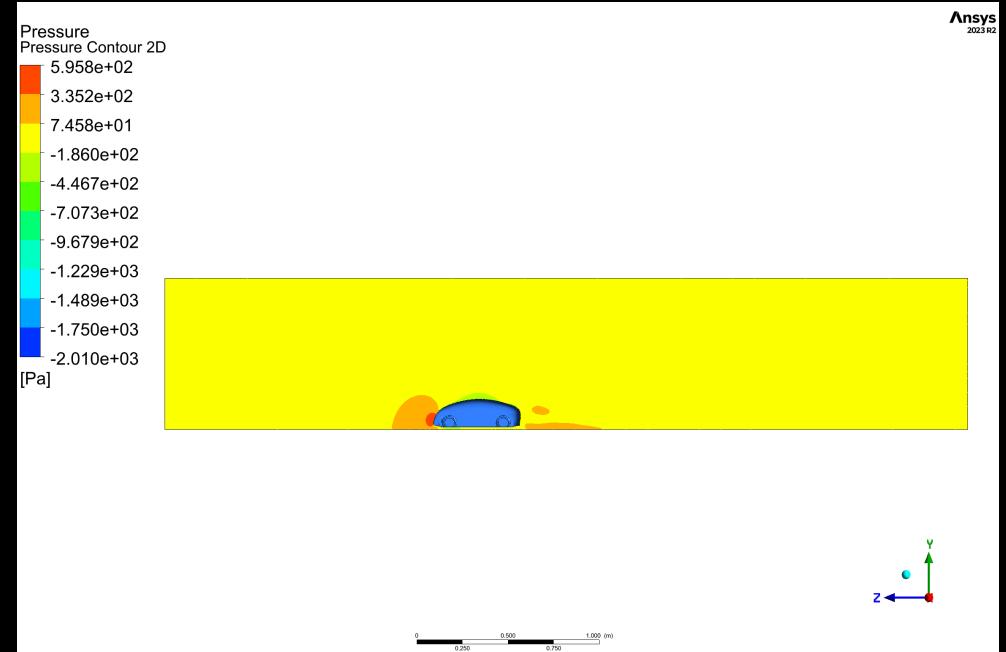
# CFD OUTPUTS

## 2D VELOCITY PLOTS



- Shows **airflow speed distribution** across vehicle
- Relatively **small recirculation zone**, suggesting good aerodynamics
- Majority of airflow is **attached to the vehicle body**, minimising turbulence
- Velocity gradients indicate **smooth airflow**
- Potential to reduce wake region** to enhance aerodynamic efficiency

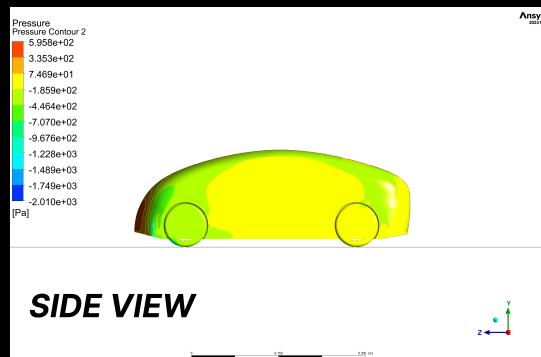
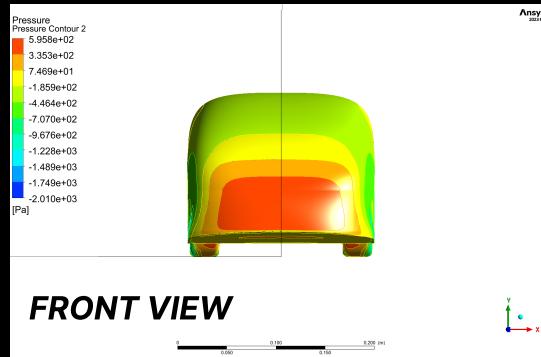
## 2D PRESSURE PLOTS



- Shows **pressure distribution** along vehicle's surface
- High pressure zone at the front** indicates aerodynamic resistance
- Low pressure zone at the back** indicates potential for drag reduction improvements
- Small pressure differentials suggest **minimal lift**
- Pressure distribution is generally uniform along the body, which contributes to **stability**

# CFD OUTPUTS

## 3D PRESSURE PLOTS

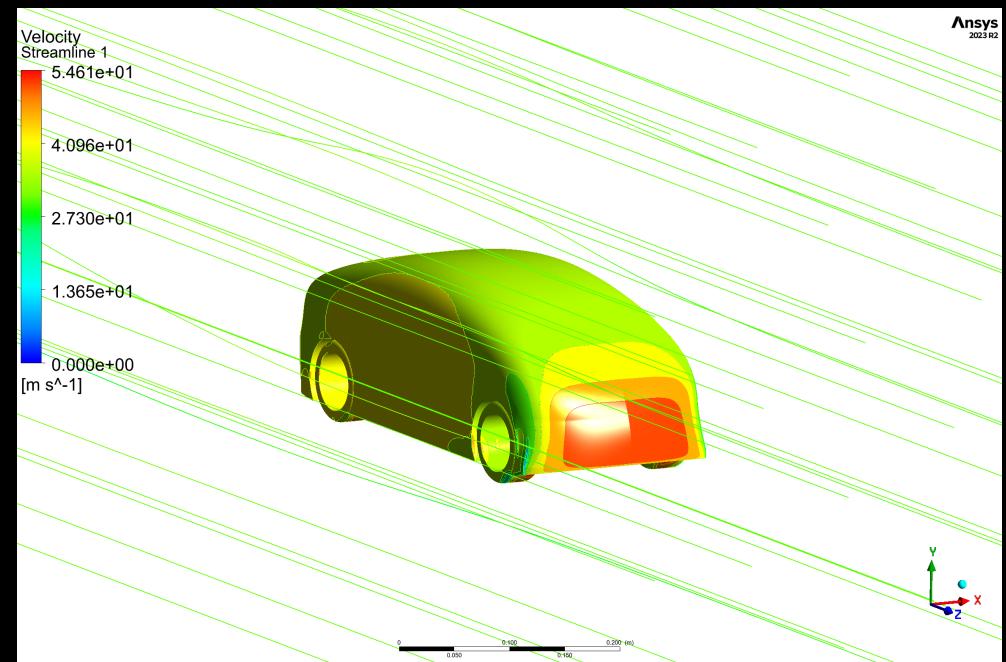


High pressure zone on the frontal surface **increases drag** but also ensures **stability**.

Gradual pressure drop along body **reduces flow separation**, enhancing aerodynamic efficiency.

Low pressure zone on the back indicates **wake formation**, a primary drag source (can be seen in 2D velocity plot).

## 3D FLOW TRAJECTORIES



- Shows **airflow streamlines** across vehicle
- Streamlines show **smooth airflow over vehicle body**, with minimal separation
- Green streamlines indicate **consistent laminar flow**
- Flow is well-guided around body** reducing turbulent wake

# RESULTS EVALUATION

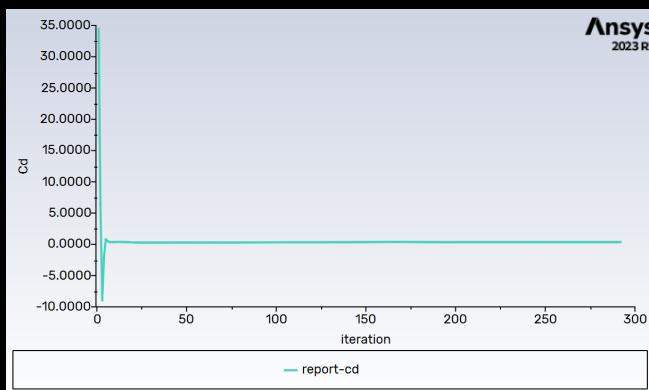
## DRAG COEFFICIENT: 0.36

Using the drag coefficient equation:

$$C_d = \frac{2F}{\rho v^2 A}$$

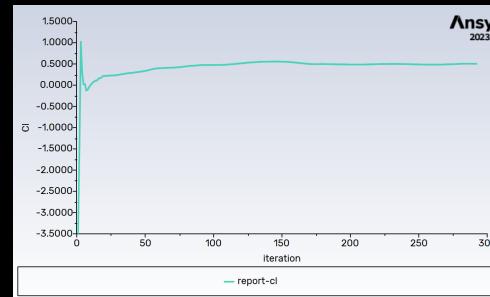
$F$  = Drag force (N)  
 $\rho$  = Air density ( $kg/m^3$ )  
 $v$  = Air velocity ( $m/s$ )  
 $A$  = Frontal area ( $m^2$ )

$$\begin{aligned} &= \frac{2 \times 2.94}{1.225 \times 30.5^2 \times 0.01425} \\ &= 0.362 \quad \text{(3 significant figures)} \end{aligned}$$



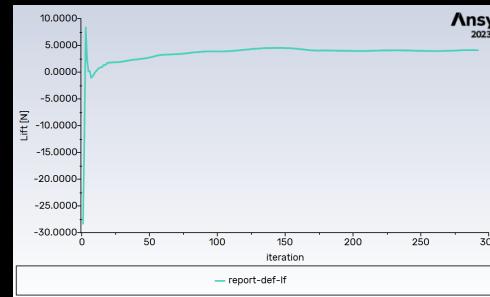
Moderate drag coefficient value suggesting a balance between aerodynamics and stability. Further improvements can be made to reduce the coefficient.

## LIFT COEFFICIENT: 0.50



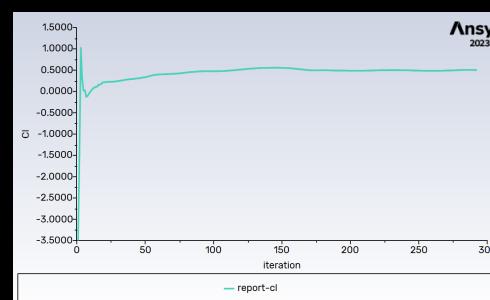
Positive lift coefficient may be undesirable and cause unwanted lift, possibly causing instability in the vehicle.

## LIFT FORCE: 4.08 N



Lift force is higher than drag force. This may cause instability. Modifying the shape may be necessary to reduce upward forces.

## DRAG FORCE: 2.94 N



Relatively low drag force, which indicates minimal resistance. Shows that the current design is effective, but with room for improvement.

# RESULTS & IMPROVEMENTS

## SUMMARY OF RESULTS

The aerodynamic analysis indicates that the design meets some of the specification aims while highlighting areas for improvement.

- The drag coefficient (0.36) is **slightly higher than the target of below 0.34**, suggesting that while the shape is relatively streamlined, further improvements could enhance efficiency.
- The lift coefficient (0.50) and lift force (4.08 N) indicate **notable lift generation**, which **could impact stability**, particularly at higher speeds.

Comparing to the benchmark vehicles:

- The drag coefficient is slightly higher than the Mercedes Benz Vito (0.34) but close to the Chevrolet Venture (0.35)
- The frontal area falls within the expected range, aligning with the 2.60 - 3.00 m<sup>2</sup> target
- Dimensions are within expected ranges

Overall, while the design is **aerodynamically competitive with existing MPVs**, **further refinements are needed** to optimise drag reduction and stability.



## FUTURE IMPROVEMENTS

### REFINING BODY SHAPE

Further streamlining could help reduce the drag coefficient to below 0.34. For example:

- Increased roof taper to reduce flow separation and vortex formation
- Reduce abrupt change in geometry at the back to minimise pressure

### REDUCING LIFT-INDUCED INSTABILITY

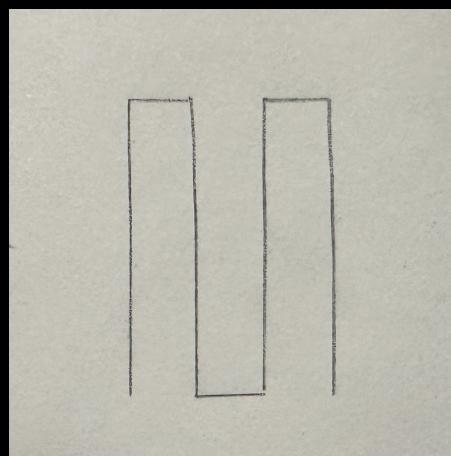
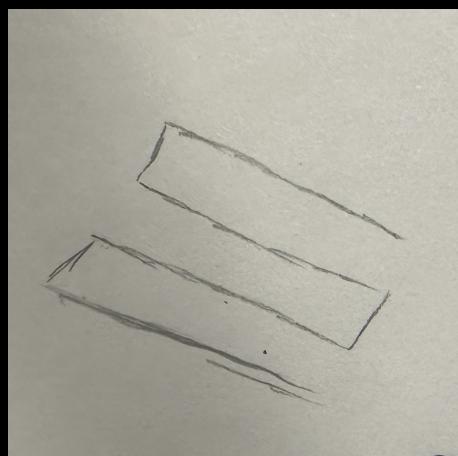
- A rear or roof-mounted spoiler at the back can prevent excessive pressure differences
- A front splitter could redirect airflow to reduce underbody airflow

### OTHER POSSIBLE FEATURES TO CONSIDER

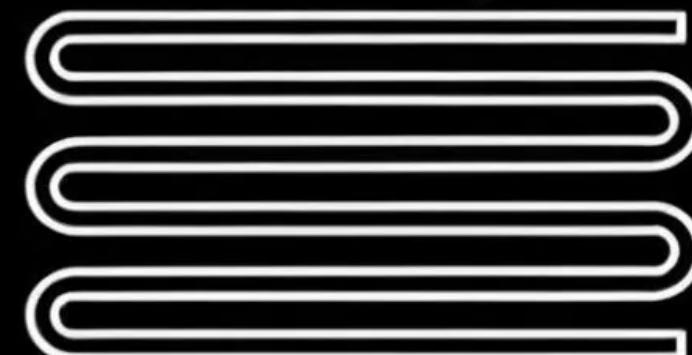
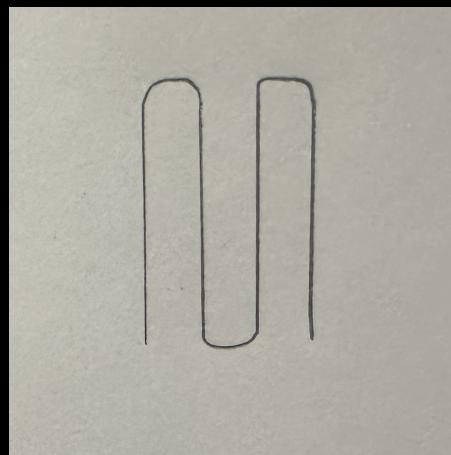
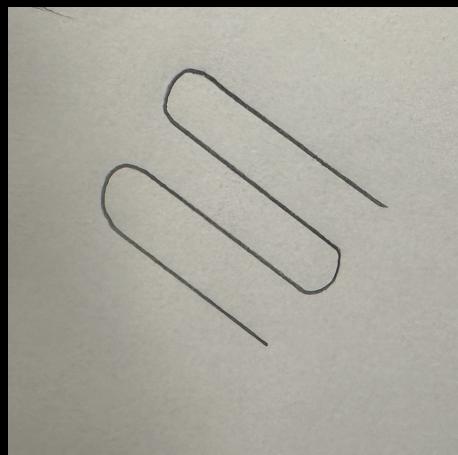
- Rear diffusers to guide airflow efficiently and reduce turbulence
- Aerodynamic wheel covers to reduce turbulence from rotating wheels

# CONCEPTUAL PIPELINE DESIGN

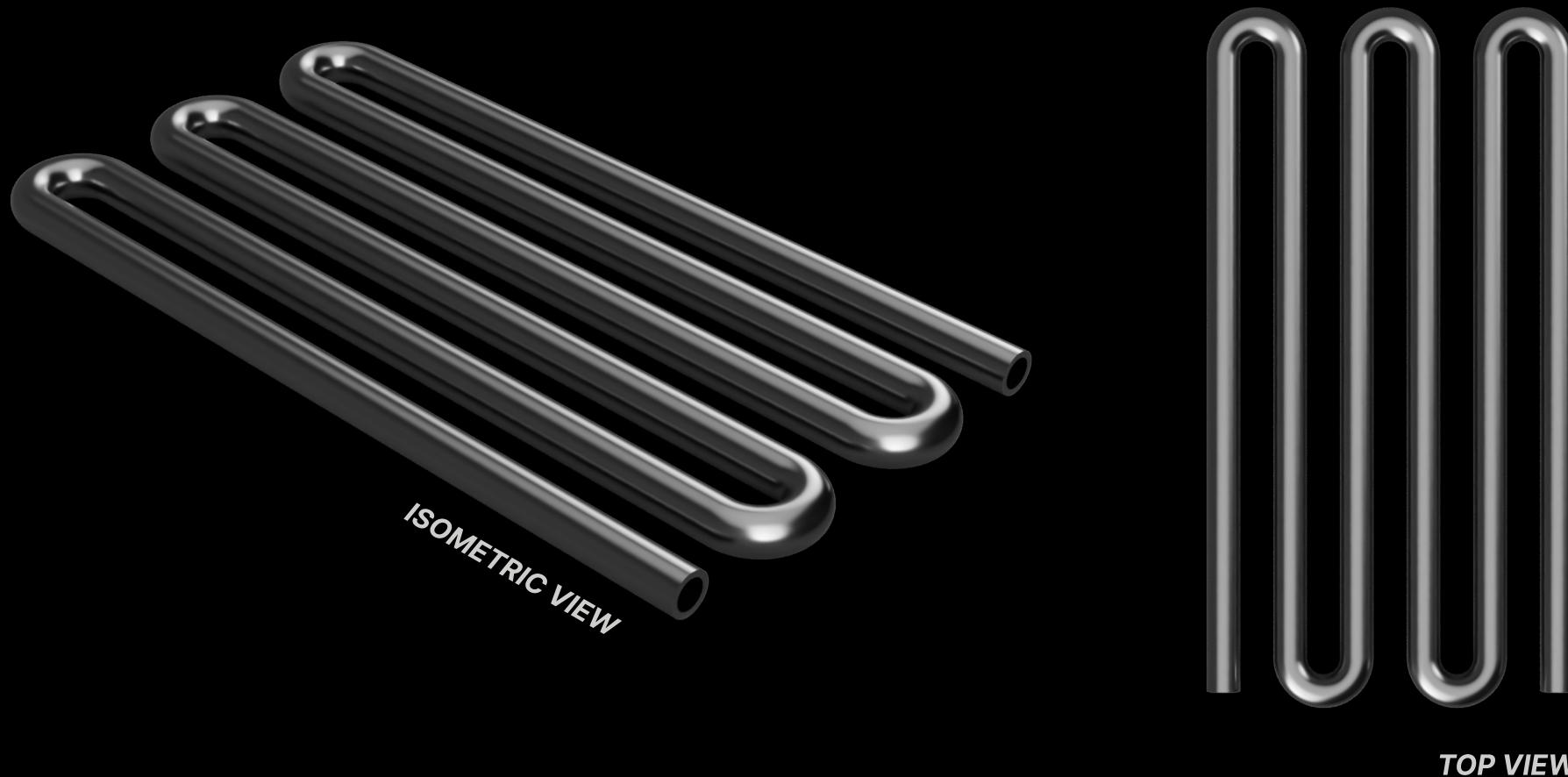
INITIAL SKETCHES



REFINED SKETCHES



# 3D CAD MODEL

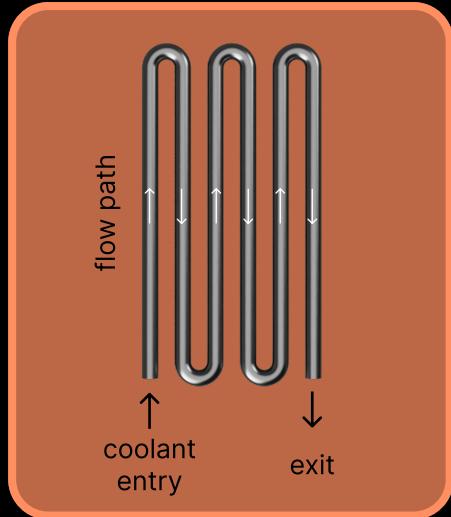


# DESIGN RATIONALE

To cool a simple battery pack, the pipeline design aims to focus on: **efficient heat transfer & minimising pressure loss**.

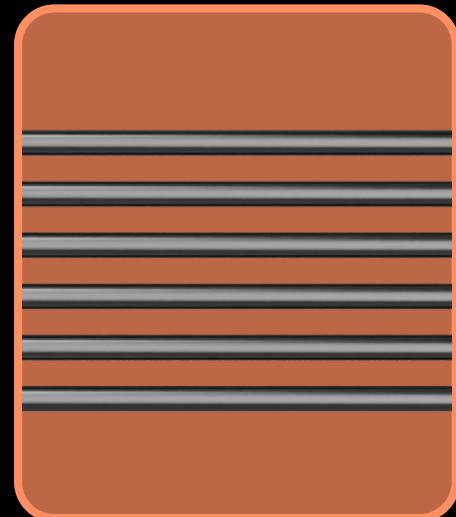
## SERPENTINE DESIGN

- Maximises surface contact for heat transfer
- Allows for a large portion of the coolant to stay in contact with the battery cooling plate, enhancing thermal exchange
- Parallel pipes allows for uniform cooling distribution, lower flow resistance and prevention of flow stagnation



## FLOW DISTRIBUTION

- Evenly spaced tubes ensures uniform cooling distribution across the battery
- A controlled moderate flow rate will ensure that the liquid absorbs enough heat before exiting, hence preventing inefficient heating from excessive velocities



## SMOOTH BENDS

- Rounded U-bends reduce sudden changes in velocity, minimising turbulence
- Pressure drop is reduced and pressure loss is minimised



## PIPE DIAMETER

- A uniform pipe diameter ensures steady flow – this avoids any unnecessary change to flow speed
- By optimising pipe dimensions and flow rate, a low Reynolds number can be maintained
- This prevents turbulence and reduces the power to pump the coolant



# CONCLUSION

## SUMMARY

The pipeline cooling system is designed with a focus on:

- ① EFFICIENT HEAT TRANSFER
- ② MINIMISING PRESSURE LOSS

The design has the following features:

- **Serpentine design shape:** To maximise coolant contact area
- **Smooth U-bends:** Reduces turbulence and pressure loss
- **Flow distribution considerations:** Ensures uniform cooling
- **Uniform pipe diameter:** Ensures steady flow

Additionally, the material should be a **high conductivity metal** such as aluminium or copper. This is to **enhance heat dissipation efficiency** – hence using a high conductivity metal means that it has **ideal thermal properties**.

## FINAL CONSIDERATIONS

In practice, this design aims to:

- Enhance thermal management
- Improve system efficiency
- Improve longevity
- Lower energy consumption
- Enhance sustainability

## NEXT STEPS

### DEFINE PIPELINE DIMENSIONS

The pipeline size is dependent on the size of the battery. As an initial guide, the bend radius of the pipe would be 2 or 3 times the diameter of the pipe and the space between pipes would be twice the bend radius.

### CALCULATE PRESSURE LOSS & HEAT TRANSFER

Determine the Reynolds number and pressure drop to validate the system's thermal efficiency.

### CFD ANALYSIS

Conduct CFD simulations to analyse the coolant flow, turbulence effects and heat transfer. This can then be compared with the theoretical calculations.

# REFERENCES

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